

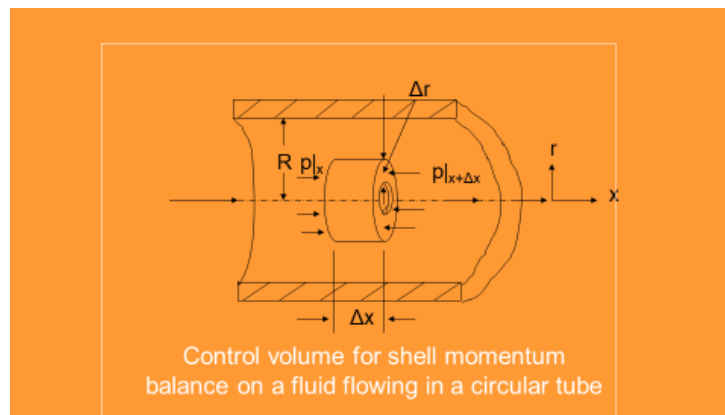
Cooling Technology: Why and How utilized in Food Processing and allied Industries

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Module No 03

Lecture 11 Basics of Thermodynamics Contd.

Good afternoon my dear students. If you remember in the previous class we started with basics of thermodynamics and in that we saw that heat transfer, fluid flow, things are also coming. A little of heat transfer, I had already said and maybe at the end of this basics of thermodynamics may take some classes. So before that let us look at the flow through the conduit or flow through pipe.



Now, this, I had given you yesterday I mean in the last class and I told that there are two types of force transfer, one is by bulk movement of the molecules and the other is by the molecular transport, that also I had explained to you, right. So assuming that we know molecular transport and we know bulk flow or bulk transfer then let us take this as a sample that is section of a pipe in which we are taking a volume element of radius Δr thickness rather and the length we are taking as Δx right and as you see this r and x this is the r and this is x right.

So I can show you that this is the r direction and this is the x direction right and the thickness we have taken as Δr , we have taken the length as Δx . So, this means there, this is two dimension and the third dimension is taken as unit right. So if we do the

momentum balance, let me clear up, if we do the momentum balance then what we get, again we have to come back, if you do momentum balance, then what we get we said that, we will be ge sorry, we will be getting, if you are doing shell momentum balance which is valid for incompressible fluid, this also I said that incompressible fluid means that, the fluid which does not change its density with time or rather with pressure, right that is incompressible. Compressible fluids do change their density with pressure, but incompressible does not and the flow is one dimensional, it is a steady state laminar flow fully developed that also we said, and there is no end effects and velocity profile does not vary along the x axis or x direction, right.

So, the governing equation that we can write, as governing equation that we can write as momentum, rate of momentum in, rate of momentum in minus rate of momentum out plus sum of the forces acting on the volume element must be equal to rate of accumulation of momentum. This is the general equation for solving the flow through pipe equation. Now, we have already said that it is a steady state. So, momentum in by convection is equal to momentum out by convection. So, they are cancelling out each other, right.

So there is no then convective momentum left over because whatever was coming is also going out. So momentum in by convection equal to momentum out by convection that is why, because of steady state. Now momentum in by molecular transport, we can write that, it is equal to tau r x into 2 pi r into delta x at the position r. If we go back to the previous slide, where we had, this slide, you see how we are finding out the area. We are finding out that 2 pi r, right into delta x, right.

$$\tau_{rx} 2\pi r \Delta x |_r$$

This we are getting for the momentum flux ok. So, we are getting 2 pi r into delta x at the face r, right. So at the face r it is so. So what about at the face r plus del r? If you remember, I am drawing it here, it was our thing, right and this was our thing right, and we had taken delta r like this, and this was delta x right. So, 2 pi r delta x that is the area at the face r into the tau r x right.

$$\tau_{rx} \cdot 2\pi r \Delta x |_{r+\Delta r}$$

So, this of course, you know tau r x, what is that tau r x is? It is not going out, let it be. So, the other one is momentum out by molecular transport that is tau r x into 2 pi r into delta x at r is equal to r plus delta r. This is the momentum out by molecular transport

right. This τ_{rx} is nothing, but shear stress, right this is nothing, but shear stress and of course, it has a definition, and we define it after words, we will come it that momentum out and momentum in due to molecular transport, we have seen one is τ_{rx} rather, into $2\pi r$ into Δx at r is equal to r and τ_{rx} into $2\pi r$ into Δx at r plus Δr . So, this is the force for molecular transport.

Now, there are other forces like pressure force. So, pressure force is also similarly working at p at x is equal to x and that is equal to p times the area, what is the area? Area is $2\pi r$ into Δr at the face x right, that is the pressure force. Similarly, we can say pressure force at the outlet that is p at x plus Δx . so that we write p times the area that is $2\pi r$ Δr at x plus Δx . So, if we know this then pressure forces are known, molecular transport forces are known and there is also gravity force, but since we have taken one horizontal tube right.

$$\text{Pressure force in} = P|_x = p 2\pi r \Delta r|_x$$

$$\text{Pressure force out} = P|_{x+\Delta x} = p 2\pi r \Delta r|_{x+\Delta x}$$

So, we can neglect the gravitational force, gravity. So, neglecting gravity because it is not there, we can then write in the governing equation that τ_{rx} into $2\pi r$ into Δx at r is equal to r minus τ_{rx} into $2\pi r$ Δx at r plus Δr plus p into $2\pi r$ into Δr at x minus $p 2\pi r$ into Δr at x plus Δx , right this is equal to 0 right. So, this is equal to 0. So, if we rearrange, it you know by the definition of derivative, derivative is, how it is made? It is out, sorry, it is out minus in over the distance, right that is, what is, any derivative, out minus in over the distance. Now, if we say that, dividing both side by Δx into Δr , if we divide both sides by Δx and Δr , then we can write this $r \tau_{rx}$ at r is equal to r plus Δr minus $r \tau_{rx}$ at r is equal to r over Δr .

$$\therefore r \tau_{rx} 2\pi r \Delta x|_r - r \tau_{rx} 2\pi r \Delta x|_{r+\Delta r} + p 2\pi r \Delta r|_x - p 2\pi r \Delta r|_{x+\Delta x} = 0$$

$$\text{or, } (r \tau_{rx}|_{r+\Delta r} - r \tau_{rx}|_r) / \Delta r = r (p|_x - p|_{x+\Delta x}) / \Delta x$$

$$\text{or, } \partial(r \tau_{rx}) / \partial r = -r \partial p / \partial x = (\Delta p / L) r$$

$$\text{or, } r \tau_{rx} = (\Delta p / L) r^2 / 2 + c$$

$$\text{or, } \tau_{rx} = (\Delta p / L) r / 2 + c / r$$

This is equal to p at x minus p at $x + \Delta x$ over Δx right. So, we can write, from the definition of derivative, that $\frac{d}{dr} \tau_{rx}$ is equal to $-\frac{dp}{dx}$, equal to $\frac{\Delta p}{L}$, that p at $x + \Delta x$, right, and p at x , this difference, we can write, to be $\frac{\Delta p}{L}$, right. So, we can write that $\frac{d}{dr} \tau_{rx}$ is equal to $-\frac{dp}{dx}$ and that is equal to $\frac{\Delta p}{L}$ into r , right. So, this on rearrangement, we can write, τ_{rx} is equal to $\frac{\Delta p}{L} \int r \, dr$ plus integration constant, as c right. So we have to find out what is the value of c that is the integration constant.

We have not done. It is not a definite integration. It is indefinite integration. So we need a constant c to be evaluated. So, one unknown so we need one equation, as boundary condition, right.

So this on rearrangement we can write τ_{rx} is equal to $\frac{\Delta p}{L} \int r \, dr$ plus c by r this is the τ_{rx} that we are looking for. Now the boundary, we need to know the boundary, and that boundary is at r is equal to 0 since τ_{rx} is not equal to infinity, therefore, c is 0 . We go back to the previous slide, here we had said that τ_{rx} , right is equal to $\frac{\Delta p}{L} \int r \, dr$ plus c by r and we need to find out the constant c and that is only possible by knowing the boundary condition and boundary condition is what? Boundary condition is at r is equal to 0 since τ_{rx} that is the tangential force or what we call it to be the shear force, which we are coming across with layer by layer or molecules by molecules. So, that is the τ_{rx} and since it is not infinity, therefore c has to be 0 at r is equal to 0 , otherwise, if it is infinite, then obviously it would be very very difficult to solve, right, it becomes a mathematical problem. So, to avoid that the best boundary is taken that at r is equal to 0 , since τ_{rx} is not infinity therefore c is 0 .

$$\text{at } r = 0, \text{ since } \tau_{rx} \neq \infty, \quad c = 0$$

$$\therefore \tau_{rx} = \left(\frac{\Delta p}{L}\right) \frac{r^2}{2},$$

$$\therefore \text{ and since } \tau_{rx} = -\mu \frac{dv_x}{dr}$$

$$\mu \frac{dv_x}{dr} = -\left(\frac{\Delta p}{L}\right) \frac{r^2}{2}$$

$$\text{or, } \frac{dv_x}{dr} = -\left(\frac{\Delta p}{2\mu L}\right) r$$

$$\text{or, } v_x = -\left(\frac{\Delta p}{2\mu L}\right) \frac{r^2}{2} + c_1$$

Then we can write τ_{rx} we can write τ_{rx} is equal to $\frac{\Delta p}{L} \int r \, dr$. This we remember. In future, not only here, but in any other, maybe, whenever you are doing any other fluid flow, this in many cases, you start with this, right, if the other conditions

remain similar. So, τr_x is equal to Δp by L into r by 2 right. Now we define τr_x , right.

Here you see τ is associated with 2, one is r another is x , right, and first one is r with τ second one is x , right. So, when it is defined, τ is defined, it is said that it is equal to μ , that is viscosity, μ into dv_x/dr . So the second one is the velocity and the first one is the direction, second one is dv_x , right and this is the direction, dr , sorry dr . So, dv_x/dr is the τr_x . So if we substitute in place of τr_x , if we substitute this, $\mu dv_x/dr$ then we can write $\mu dv_x/dr$ is equal to this minus we have taken to this side.

So minus ΔP by L into r by 2 right. So we get by rearranging, dv_x/dr is equal to minus ΔP by $2 \mu L$ into r right. So on integration of this, we get v_x is equal to minus Δp by $2 \mu L$. Now this r becomes r square and divided by 2 plus an integration constant called C_1 , other was C and this one is C_1 , right. So, we need another boundary to find out this integration constant C_1 , we need another boundary.

$$\text{Now, at } r = R, V_x = 0$$

$$\therefore C_1 = (\Delta p/4\mu L) R^2$$

$$V_x = (\Delta p/4\mu L) (R^2 - r^2)$$

So, what could be the boundary our pipe was like this, right and fluid was flowing like this, right, and it is laminar, etc. all things were said. So, the velocity here, it is v_x . So in the direction of x , velocity is v_x . So this velocity which is in the pipe wall, so that is not moving because it is clinging to the surface, right, this is called clinging to the surface.

So, the fluid which is clinging to the surface is not moving and therefore, the velocity at r is equal to say, capital R is equal to 0, sorry, v_x is 0 at r is equal to capital R , v_x is 0 right. So, that is what we have taken and by solving we get the value of C_1 equal to ΔP by $4 \mu L$ into R square, right. So, ΔP by $4 \mu L$ into R square, we get and substituting the value of C_1 to the value of v_x , if we substitute, then we get v_x is equal to ΔP by $4 \mu L$ into R square, capital R square minus small r square right. Here again we have taken that volume element small, here, and the radius we have taken to be equal to r right this is r and if it is the axis right.

$$v_x = \frac{P_{in} - P_{out}}{4\mu L} R^2 \left[1 - \left(\frac{r}{R} \right)^2 \right]$$

So, we have gotten the velocity in this, is called instantaneous velocity at any position because x we have not said, x can be anything, like x can be anything, like here.

So that is $v_x 1$ or here that is $v_x 2$ or here $v_x 3$, right. We have not said. So, that is why it is called instantaneous velocity and the velocity is then determined as the maximum velocity or average velocity. So that instantaneous velocity v_x we have gotten ΔP by $4\mu L$ into R^2 minus r^2 capital R^2 minus small r^2 , right. Since, we have gotten it, we can find out the average velocity.

The velocity which normally we talk about is the average velocity, that is, you have this pipe here you have one velocity as we said $v_x 1$ here we have another velocity $v_x 2$ here we have another velocity $v_x 3$ dot dot dot as many points we get right. So we can say that the average velocity is the velocity which we talk about. So how can, we find out average velocity. So $v_{average}$ is $1/A$ into integration of the area, right. So, that is 0 to 2π and the other one that is 0 to L , that is the x , right.

So if this two we take then it is v_x into dA . Now this dA is nothing but $r d\theta$ and $d r$, right, $r d\theta d r$ into v_x . So, this we substitute as 1 by r 1 by A integral of, double integration of $v_x dA$ and this $v_x dA$, we are substituting as v_x 1 by πR^2 right and v_x into $r d\theta$, $r d\theta$ is that area right, that this is $r d\theta$ is that angle between this and this. So this is the $d\theta$. So $r d\theta$ into $d r$ right into v_x is the expression.

Now, on simplification, on integration, we can write that ΔP is P_{in} minus P_{out} obviously, that ΔP this P_{in} is much more than p_{out} right, that is P_{in} is higher than p_{out} right. So that we can take as P_{in} minus P_{out} over $4\mu L$ times R^2 into 1 minus small y capital R whole square right. This we are getting and we get that this $v_{average}$ that becomes equal to P_{in} minus P_{out} into capital D square by $32\mu L$ right. This is known as Hagen-Poiseuille equation. Most of the cases, you are asked, how can you find out the pressure drop in a pipe.

$$\begin{aligned} V_{av} &= 1/A \cdot \iint v_x dA = 1/\pi R^2 \cdot \iint v_x r d\theta dr \\ &= 1/\pi R^2 \int 2\pi r dr v_x \\ &= (2\pi/\pi R^2) \int r (\Delta p/4\mu L)(R^2-r^2) dr \\ &= (P_0-P_L)/2\mu L (R^2/2 - R^2/4) \end{aligned}$$

$$= (P_0 - P_L) / 8\mu L R^2 = (P_0 - P_L) / 32\mu L D^2$$

So that you can say through the Hagen-Poiseuille's equation. If you know pressure drop, or if you do not know pressure drop then, if you know average velocity, if you know the viscosity, length, and diameter, then you can find out what is the pressure drop in the pipe, right. This is called theoretical and from there we get v_{\max} , which is how much v is max at, this was the pipe, v is max at r is equal to 0. So that r is equal to 0 v is equal to v_{\max} right. So by substituting that we get P_{in} minus P_{out} into R^2 by $4\mu L$ this is the v_{\max} and from the relation we get v_{average} is equal to v_{\max} by 2.

$$v_{av} = \frac{(P_{in} - P_{out}) D^2}{32 \mu L}$$

$$V_{\max} = (P_0 - P_L) R^2 / 4\mu L, \text{ at } r = 0$$

$$\therefore V_{av} = V_{\max} / 2$$

v_{average} equal to v_{\max} by 2 ok. Hope you have understood how the pressure, how the pressure drop in the pipe can be calculated with the help of Hagen-Poiseuille's equation ok. Thank you.