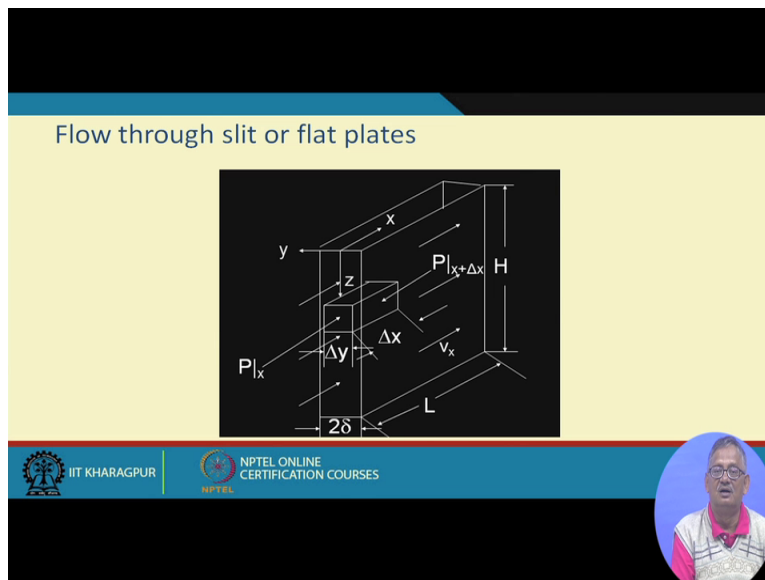


**Course on Momentum Transfer in Process Engineering**  
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**Lecture 24**  
**Module 5**  
**Flow through flat plates or slits**

Yeah, today we will do on flow through flat planes, right? Flow through slit or flat plates those as we mentioned in the previous class previous class that the flow through the homogenizing valve this homogenizing valve can be assumed to be say slit, right? A very small hole through which it is moving and during that process as we said earlier that the fat globules get homogenized or the bigger fat particles becomes smaller according to the size which you have given according to the space through which the flow is occurring in then that will indicate that at what ultimate size of the fat globules you will get, right?

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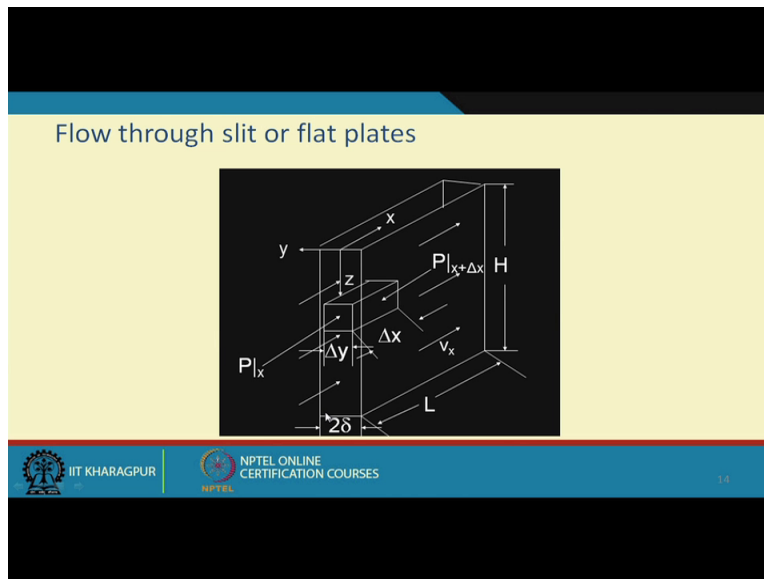


So let us look into that today, so now we have drawn one similar thing as it is here flow through slit or flat plates, right? So we have taken this to be enlarged (value) figure of the slit, right? This is the enlarge view and in which we have taken a small volume element, right? We have taken a small volume element and this volume element has the dimensions of delta x, delta y and delta z, right? Or say if the third dimension be delta z or H and this width or one dimension could be 2

delta so that the center is at plus minus del, right? 0 plus del minus del and this is say L and this is I, right?

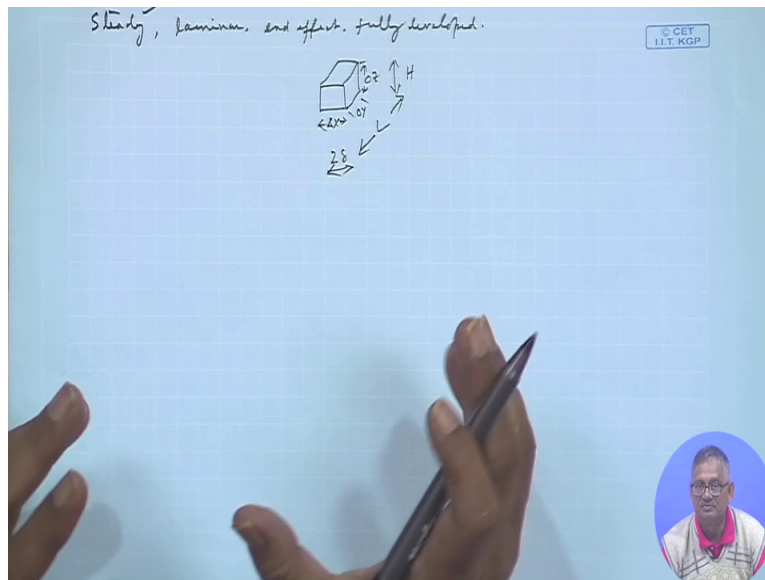
So we will do the similar force balances as we have done earlier and we will find out what is the velocity profile and what is the stress profile in the volume element and then integrate over the entire range right?

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So again this we had taken a  $2\delta$ , so the axis is at the center so this is plus  $\delta$  and minus  $\delta$  this is  $L$  length and height  $H$ , right? So what is acting  $P$  at the (fo) pressure inside our inlet at  $P$  at  $x$  and at the outlet at the phase  $x$  plus  $\delta x$  it is  $P$  at  $x$  plus  $\delta x$ , right? So let us look into that so similar to the flow problems earlier we have done, right?

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Here, also we assume some of the things that one is the flow is steady flow is steady then the flow is laminar there is no end effect rather it is fully developed, right? Fully developed flow and we will take a small volume and that small volume is having  $\Delta x \Delta y \Delta z$ , right? This  $\Delta x \Delta y$  and  $\Delta z$ , right? Where we have shown this is the (vol) small volume and the entire slit we had shown to be equals to  $2 \Delta y$  then  $L$  to  $\Delta x$  in this way  $L$  in this way and  $H$  in this way, right?

That was the entire slit, right? Now, since we are saying that it is a steady state, then then the from the general equation which we start with that the general equation or governing equation that some of the forces acting on to it is equals to 0, right? And that also we will do here, but since we said that the momentum transfer takes place in two ways, one by the bulk molecule and one by the molecular transport so this bulk force bulk fluid that term goes off, right?

Then we are neglecting that so unnecessarily we again writing everything momentum in minus momentum out for the bulk fluid that we are skipping.

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Steady, laminar, end effect, fully developed.

momentum flux (out-in) due to molecular transport =  $(\tau_{yx}|_{y+\Delta y} - \tau_{yx}|_y) \Delta x w$

Pressure forces =  $(p|_{x+\Delta x} - p|_x) \Delta y w$

$\Delta x w (\tau_{yx}|_{y+\Delta y} - \tau_{yx}|_y) + (p|_{x+\Delta x} - p|_x) \Delta y w = 0$  dividing by  $\Delta x \Delta y w$

$w, \frac{\tau_{yx}|_{y+\Delta y} - \tau_{yx}|_y}{\Delta y} + \frac{p|_{x+\Delta x} - p|_x}{\Delta x} = 0$  from the definition of derivative.

$\lim_{\Delta y \rightarrow 0} \frac{\partial(\tau_{yx})}{\partial y} + \frac{\partial(p)}{\partial x} = 0$

$\frac{\partial \tau_{yx}}{\partial y} = -\frac{\partial p}{\partial x} = -\frac{\Delta p}{L}$

So we are starting with that the momentum flux, right? Due to molecular transport momentum flux due to molecular transport that we take as  $\tau_{yx}$  at the phase  $y$  plus  $\Delta y$  minus this is okay instead of flux we call it to be out minus in, right?  $\tau_{yx}$  at the phase  $\tau_{yx}$  plus  $\Delta y$  times the area that or rather  $\tau_{yx}$  at the phase  $y$  times the area that is  $\Delta x \Delta y$  at the phase  $\Delta x$  into  $w$ , right?

So this is area  $\Delta x$  into  $w$  this is acting flow is occurring through this, right? We said the flow is occurring like that, okay. So there is the area  $\Delta x$  into width that is  $w$  or  $H$  ultimately it will come, right? This is the momentum flux, so pressure drop or pressure force acting on this volume element net is  $P$  at the phase  $x$  plus  $\Delta x$ , right? And minus  $P$  at the phase  $x$ , right? Times the area that is  $\Delta y$  times  $w$ , right? So if this be true, then some of the forces acting we can write  $\tau_{yx}$  at  $y$  plus  $\Delta y$  minus  $\tau_{yx}$  at  $y$ , right? Plus  $P$  at  $x$  plus  $\Delta x$  minus  $P$  at  $x$  this times  $\Delta y$  times  $w$  this times  $\Delta x$ , right?

So now if we divide this is equals to 0, if we divide both sides with  $\Delta x w$  and  $y$ , then next stage we can write  $\tau_{yx}$  at  $y$  plus  $\Delta y$  minus  $\tau_{yx}$  at  $y$  over over this we can say  $\Delta y$  minus  $P$  at  $x$  plus  $\Delta x$  minus  $P$  at  $x$  over or this is of course plus divided by  $\Delta x$  is equals to 0 dividing dividing by  $\Delta x \Delta y w$ , right? Then we get this, and now if put that limit condition limit  $\Delta y$  tends to 0, right? And the limit  $\Delta x$  tends to 0 if we put and then from the

definition of derivative we can write that  $\frac{\partial \tau_{yx}}{\partial y}$  of  $\tau_{yx}$  plus  $\frac{\partial \tau_{xy}}{\partial x}$  of  $\tau_{xy}$  rather is equals to 0, right?

So we can write  $\frac{\partial \tau_{yx}}{\partial y}$  is equals to minus  $\frac{\partial \tau_{xy}}{\partial x}$ , right? So this  $\frac{\partial \tau_{xy}}{\partial x}$  you can write  $\frac{\Delta P}{L}$  that is  $\frac{\Delta P}{L}$  actually actually this should have been written  $\frac{\partial \tau_{xy}}{\partial x}$  this should have been written  $\frac{\partial \tau_{xy}}{\partial x}$ , right? So this is  $\frac{\partial \tau_{xy}}{\partial x}$  this is  $\frac{\partial \tau_{xy}}{\partial x}$  exactly, so yeah this is  $\frac{\partial \tau_{xy}}{\partial x}$  this is  $\frac{\partial \tau_{xy}}{\partial x}$  otherwise this is not coming, okay  $\frac{\partial \tau_{xy}}{\partial x}$  this area okay. So this is  $\frac{\partial \tau_{xy}}{\partial x}$  which we ultimately said, so  $\frac{\partial \tau_{xy}}{\partial x}$  is  $\frac{\Delta P}{L}$  if it is written like that so minus  $\frac{\Delta P}{L}$  is  $\frac{\partial \tau_{yx}}{\partial y}$ , right?

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$$\frac{\partial \tau_{yx}}{\partial y} = -\frac{\partial P}{\partial x} = -\frac{\Delta P}{L}$$

$$\frac{\partial \tau_{yx}}{\partial y} = \frac{\Delta P}{L} = \frac{(P_{in} - P_{out})}{L}$$

$$\tau_{yx} = \frac{\Delta P}{L} y + C_1$$
 B.C. at  $y=0$ ;  $\tau_{yx}=0$ ;  $C_1=0$   

$$\tau_{yx} = \frac{\Delta P}{L} y \rightarrow \text{stress profile}$$

$$\tau_{yx} = -\mu \frac{\partial v_x}{\partial y} = \frac{\Delta P}{L} y \quad \text{or,} \quad \frac{\partial v_x}{\partial y} = -\frac{\Delta P}{L} y$$
 B.C.  $v_x = -\left(\frac{\Delta P}{2\mu}\right) \frac{y^2}{2} + C$   

$$0 = -\frac{\Delta P}{2\mu} \frac{h^2}{2} + C, \quad C = \frac{\Delta P}{2\mu} \frac{h^2}{2}$$

$$v_x = \frac{\Delta P}{2\mu L} \left[ h^2 - y^2 \right] = \frac{\Delta P}{2\mu L} \left[ 1 - \left(\frac{y}{h}\right)^2 \right]$$

Then, we can now say that this negative sign or  $\frac{\partial \tau_{yx}}{\partial y}$  of  $\tau_{yx}$  this is equals to minus  $\frac{\partial \tau_{xy}}{\partial x}$  is equals to minus  $\frac{\Delta P}{L}$  this indicates that as the as  $x$  increasing, right? As ( $\Delta P$ ) at  $x$  increasing value of  $\Delta P$  is decreasing the negative sign says that, alright as  $L$  is increasing  $\Delta P$  is decreasing, right? So we can write that  $\frac{\partial \tau_{yx}}{\partial y}$  again that  $\frac{\partial \tau_{yx}}{\partial y}$  then we can say that  $\frac{\partial \tau_{yx}}{\partial y}$  or  $\frac{\partial \tau_{yx}}{\partial y}$  is equals to  $\frac{\Delta P}{L}$  in this  $\Delta P$  takes care of is equals to  $P$  in minus  $P$  out, right? Over  $L$ .

So this negative sign is taken into  $\frac{\Delta P}{L}$ , right? Now if we integrate then what we get that  $\tau_{yx}$  is equals to, right?  $\frac{\Delta P}{L}$  into  $y$  plus  $C_1$ , right? So what we can write at boundary conditions if we put boundary condition that is what at  $y$  is equals to 0  $y$  is equals to 0 if you

remember our thing like this, right? And this was like this, right? We said that this is  $2 \Delta$  okay  $\Delta y$  it was this was  $\Delta x$  and this was  $H$  or  $\Delta z$  whatever you call, right?

So at  $y$  is equals to 0, that means this point at  $y$  is equals to 0  $\tau_{yx}$  that is the  $(0)$ (14:57), right? Not the one the center. So  $\tau$  at  $y$  is equals to 0  $\tau_{yx}$  is equals to 0, right? Then we can write  $C_1$  is also equal to 0 or  $\tau_{yx}$  is equals to  $\Delta P$  over  $L$  into  $y$  that is the this is the stress profile, right? If you plot this stress profile that as  $y$  is increasing  $\tau_{yx}$  is also increasing, right? So under limiting condition that is when  $y$  is  $\Delta$ , then  $\tau_{yx}$  will have one plus  $\Delta$  when  $y$  is equals to minus  $\Delta$   $\tau_{yx}$  will have another because this we have told ultimately the bigger one is  $2 \Delta$  that is the thickness or width or whatever we call, right?

Where the axis is at the center, so it is minus  $\Delta$  plus  $\Delta$  say this point and minus  $\Delta$  this point, right? So here it is 0 so we write this as  $y$  is increasing  $\tau_{yx}$  is increasing true but it is (lim) putting to the limit when it is plus  $\Delta$  then  $\tau_{yx}$  is plus and where is minus  $\Delta$   $\tau_{yx}$  is also some minus, that means some if this is the point and if this is the point this is 0, so one will have one and another will have this, right? This is the stress profile if we take this to be the stress profile, okay fine.

Then substituting  $\tau_{yx}$  as minus  $\tau_{yx}$  is equals to minus  $\mu \Delta v_x \Delta y$ , right? We can write that this is equal to  $\Delta P$  over  $L$  into  $y$ , right? And if this is true, then again putting the second boundary or or now if we if re-write this  $\Delta v_x \Delta y$  is equals to minus  $\Delta P$  over  $L$ , right? Or  $L$  into  $y$  rather, okay or we on integration you can write that  $v_x$  is equals to this minus  $\Delta P$  over  $L$  into  $C$  or rather  $y$  square by 2, right?  $y$  square by 2 plus  $C$   $y$  square by 2 plus  $C$ .

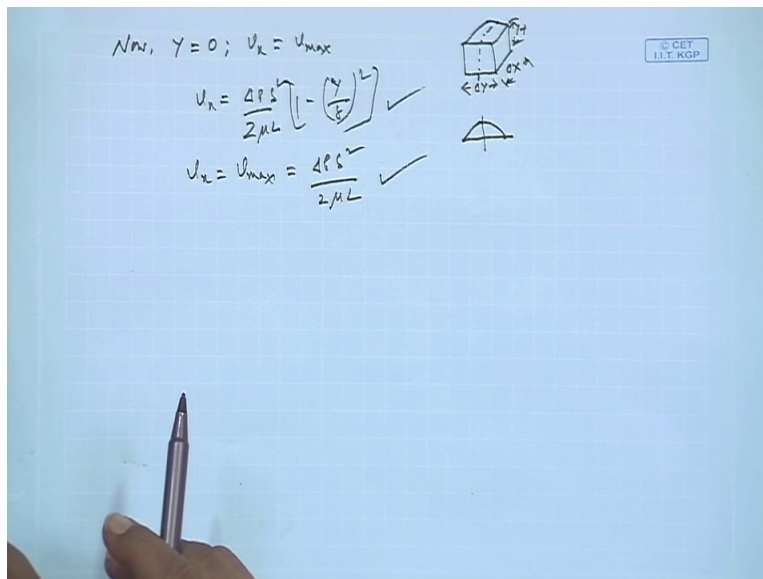
So if we put that boundary, what is the boundary? Boundary is at plus  $\Delta$  or minus  $\Delta$   $v$  is equals to 0 or  $v_x$  is equals to 0 at plus  $\Delta$  or minus  $\Delta$  because plus  $\Delta$  is at the valve and this minus  $\Delta$  is also at the valve and this minus  $\Delta$  is also at the valve, right? So in that case we can say that this plus  $\Delta$  or minus  $\Delta$  at plus minus  $\Delta$  this is equals to 0  $v_x$ . So we can write 0 is equals to minus 0 is equals to minus say plus minus  $\Delta$  we have taken, so we can write  $\Delta P$  over  $2 \mu L$  this  $\mu$  comes here or here it should have been  $2 \mu$ .

So  $2 \mu L$ , right? 0 is equals to is equals to this is plus  $C$  so if we take minus then  $C$  is equals to  $\Delta P$  over  $2 \mu L$ , right? And and we can write  $v_x$  is equals to  $\Delta P$  by  $2 \mu L$  if we take it to be common, right? Then we have said  $\Delta P$  this was  $y C$  and of course  $y$  square. So it is  $\Delta$

square, right? This was  $\Delta^2$  so  $C$  is equal to this into  $\Delta^2$ , right? So if we take  $\Delta^2$  common or  $\Delta^2$  minus  $y^2$  is the  $v_x$ , right? So we can also write this is also equal to  $\frac{\Delta P \Delta^2}{2 \mu L} \left(1 - \frac{y^2}{\Delta^2}\right)$ , right?

So this is the  $v_x$  at any instant velocity is like that at any instant velocity  $v_x$  that is  $\frac{\Delta P \Delta^2}{2 \mu L} \left(1 - \frac{y^2}{\Delta^2}\right)$ , okay  $y$  by  $\Delta^2$ .

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Now to (understa) for the next one that is now at  $y$  is equal to 0, what is the value of  $v_x$ ?  $v_x$  should become  $v_{max}$ , right? I again for understanding put this we had this, we had this  $\Delta x$  we had this  $\Delta y$ , right? We had this as  $H$  or height or  $\Delta z$  whatever we call, right?

Then  $v$  at  $y$  is equal to 0 that is this, right? At  $y$  is equal to 0 we can say  $v_x$  is  $v_{max}$ , right? What we had this one equation  $v_x$  is equal to  $\frac{\Delta P \Delta^2}{2 \mu L} \left(1 - \frac{y^2}{\Delta^2}\right)$ , right?  $\Delta^2$ , right? This is nothing but parabolic equation, right? That is why the the profile we always see to be parabolic like that, right? If we see the profile of velocity it is always parabolic like this, right? So this is also parabolic in nature, so at  $y$  is equal to 0, then  $v_{max}$   $v_x$  becomes  $v_{max}$  and that it becomes equal to  $\frac{\Delta P \Delta^2}{2 \mu L}$  because this becomes  $0$   $1$  minus this so that is  $1$ .

So  $\frac{\Delta P \Delta^2}{2 \mu L}$  is the  $v_{max}$ , right? Next we go to the other one that is the average velocity.

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$N/m^2, \gamma = 0; v_x = v_{max}$   
 $v_x = \frac{\Delta P \delta^2}{2\mu L} \left(1 - \left(\frac{y}{\delta}\right)^2\right)$   
 $v_x = v_{max} = \frac{\Delta P \delta^2}{2\mu L}$   
 Average velocity  $v_{av} = \frac{1}{A} \int_0^{\delta} \int_0^{\delta} v_x dx dy = \frac{L}{\delta L} \int_0^{\delta} v_x dy$   
 $= \frac{L}{\delta L} \int_0^{\delta} \left[ \frac{\Delta P \delta^2}{2\mu L} \left(1 - \left(\frac{y}{\delta}\right)^2\right) \right] dy$   
 $= \frac{\Delta P \delta}{2\mu L} \left( \delta - \frac{\delta^3}{3\delta^2} \right) = \frac{\Delta P \delta^2}{3\mu L}$   
 $v_{av} = v_{max}?$   
 $v_{av} = \frac{2}{3} v_{max}$

So  $v_{max}$  you have found out  $v$  we have found out, then from there we have found out  $v_{max}$  you have also seen the  $\tau$  and now  $\tau$  is that is the momentum flux, now let us go to find out average velocity. So average velocity  $v_{average}$  that can be said to be equals to integration of 1 by  $A$  0 to  $\delta$  0 to  $\delta$  of  $v_x dx dy$ , right? So this is nothing but  $L$  over  $\delta L$ , right? Because one integration of  $dx$  so that that goes into this  $\delta$ .

So equal to  $L$ , right? 0 to  $L$  0 to  $\delta$  of  $v_x dy$  we have  $L$  by  $\delta L$ , right? 0 to  $\delta$  of  $v_x dy$  we have and this we can write  $L$  by  $\delta L$ , right? And  $v_x$  if we substitute with this  $\frac{\Delta P \delta^2}{2\mu L} \left(1 - \left(\frac{y}{\delta}\right)^2\right)$ , right? This was between 0 to  $\delta$ , right? This into  $dy$ , right? So this on simplification and integration and simplification we can write to be  $\frac{\Delta P \delta^2}{2\mu L} \left( \delta - \frac{\delta^3}{3\delta^2} \right)$ , right? So that comes out  $\frac{\Delta P \delta^2}{3\mu L}$ , right?  $\frac{\Delta P \delta^2}{3\mu L}$  goes out  $2\mu L$  this comes there and on integration  $dy$  is  $y$  putting the limit is  $\delta$  minus this becomes  $\delta^3$  by  $3\delta^2$ , right?

$\frac{\Delta P \delta^2}{3\mu L}$  or this is becoming nothing but equals to  $\frac{\Delta P \delta^2}{3\mu L}$  because this  $\delta^3$  this  $\delta^2$  goes out it becomes  $\delta$  by  $3$ , so  $3\delta$  minus  $\delta$  that is  $2\delta$  by  $3$ , right?  $2\delta$  this  $2$  and this  $2$  goes out this  $\delta$  becomes  $\delta^2$  and this  $3$  remains, right? So this is  $\frac{\Delta P \delta^2}{3\mu L}$ , right? And this if you



remember we had made  $v_{max}$  was  $\frac{\Delta P \delta^2}{2 \mu L}$ . Then what is the relation between  $v_{average}$  and  $v_{max}$ ?

$v_{average}$  and  $v_{max}$  that relation becomes equals to  $v_{average}$  is equals to 2 third of  $v_{max}$ , right? 2 third of  $v_{max}$  is the  $v_{average}$ , right? So this way if we can find out that what is the average velocity in terms of maximum velocity it is 2 third.

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Now,  $y=0; v_x = v_{max}$

$$v_x = \frac{\Delta P \delta}{2 \mu L} \left[ 1 - \left( \frac{y}{\delta} \right)^2 \right]$$

$$v_x = v_{max} = \frac{\Delta P \delta}{2 \mu L}$$

Average velocity  $v_{av} = \frac{1}{A} \int_0^{\delta} v_x dx dy = \frac{L}{\delta L} \int_0^{\delta} v_x dy$

$$= \frac{L}{\delta L} \int_0^{\delta} \left[ \frac{\Delta P \delta}{2 \mu L} \left( 1 - \frac{y^2}{\delta^2} \right) \right] dy$$

$$= \frac{\Delta P \delta}{2 \mu L} \left( \delta - \frac{\delta^3}{3 \delta^2} \right) = \frac{\Delta P \delta}{3 \mu L}$$

$v_{av} = v_{max}?$

$$\underline{v_{av} = \frac{2}{3} v_{max}}$$

- ① Pipe
- ② Film
- ③ Annular
- ④ Slit flow.

$v_{av} = v_{max}$  ✓

Now if you if you remember what it was for the pipe flow average velocity with maximum velocity what it was for for the flow through or or for film flow and this is the slit flow then this 3 you put together and and see and see that there will be a distinct difference between the three, right?

In one case it is pipe flow, in other case it is film flow, in third case it is annular flow and in the fourth case it is that slit flow. So all these four cases you just see what is the relation between  $v_{average}$  and  $v_{max}$  in all 4 cases you will see that they are distinctly different and I tell I tell you that you find out and make a tabular form so that you remember them for future activities, right? For future work or for in future so that you can memorize it, if required that what is the velocity and maximum velocity relation in pipe or in annulus or in film or maybe in case of slit, right?

Then you will see they are not same, they are different, okay.

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$N_{Re} = \frac{4 v_{avg} \delta \rho}{\mu}$   
 $f = \frac{\tau_w}{\frac{\rho v_{avg}^2}{2}}, \tau_w = \frac{\Delta P \delta}{L} = \frac{3 \mu L v_{avg} \delta}{L^2} = \frac{3 \mu v_{avg}}{\delta}$   
 $\therefore f = \frac{\Delta P \delta}{L} = \frac{2 \Delta P \delta}{L \rho v_{avg}^2}$   
 $\Delta P = f \rho \frac{L}{2 \delta} v_{avg}^2 = \frac{2 f \rho L}{2 \delta} \cdot \frac{v_{avg}^2}{2}$   
 $\underline{\underline{4 f \rho \frac{L}{\delta} \frac{v_{avg}^2}{2}}}$

Now let us look into that H we have said to be the height again we draw that H to be equals to the height, right? This was our H and this H is quite quite bigger, right? H is much much greater than to delta, right? So in that case we can say that the hydraulic diameter for the slit for the slit that becomes equals to 4 del, right? So so we can say NRe this becomes 4 because we have taken hydraulic diameter to be 4 del because H is much much greater than 2 del, so hydraulic diameter becomes equal to 4 del.

So we can write 4 v average del rho, sorry del rho divided by mu to be the Reynolds number 4 v average del rho by mu, right? And fanning friction factor f we can write that is tau at valve w over rho into v by 2, right? Or tau at valve is equals to delta P del by L, right? That is equals to 3 mu L v average by del square into del by L, right? That is equals to 3 mu v average by del, right? Therefore, f we can write delta P del over L over rho v average square by 2, so this is 2 delta P del by L rho v average square.

Therefore, delta P can be written 4 rho L by 2 del v average square, right? So this can be also written 2 f rho L by 2 del into v average square by 2, right? So if you here it is 2 del here it is 2, right? So so if we remember for the pipe flow what was delta P, right? That with f that was 4 f rho L by D v square by 2, right? So that was for the pipe flow so this is also analogous, so that is why these four as I said that if you find out all 4 you will find it to be so different and then you

should have all these together in one page and then it will be easy for you to remember, okay  
thank you.