

MARINE ENGINEERING

By

Prof. Abdus Samad

IIT Madras

Lecture52

Numerical Problems

Now, we will try to solve some problem. these problems are basically from gate papers. Let us see the problem. In a simple Brayton cycle, the pressure ratio is given 8. R_p is given 8 Brayton cycle or gas training cycle and temperature at the entrance of compressor.

T_1 is given 300, 1400 temperature at the entrance and, okay, this is T_1 and so that means I have to draw my T-S diagram, T-S 1, 2, 3, 4. T_1 is given and T_3 also given 1400 K. Both compression and gas turbine have isentropic efficiency η is in 0.8, η is in for both turbine compressor both the for the gas assume constant volume constant value of C_p so C_p value 1 kg per kg given specific heat ratio is given 1.4 neglect changes for kinetic potential energy calculate power required and calculate thermal efficiency it a thermal and WC theta thermal maybe I will put TH.

Now, I have to calculate these two parameters. Now, R_p is given 8 that means P_2 by P_1 is given and γ also given 1.4. process 1 to 2, 1 to 2 process. So, what is happening? Actual compression happening, 1 to 2 isentropic compression.

2 dash 4 dash. 1 to 2 actual process, 1 to 2 irreversible process or energy will be lost. for reversible adiabatic compression process T_2 by T_1 , T_2 by T_1 equals T_2 by P_1 gamma minus 1 by gamma. T_2 equals 300 T_1 equals 300.

T_2 equals T_1 into 8 divided by 1.4 minus 1, 1.4. this value is coming 543.43 K. Now, η isentropic efficiency, η isentropic. isentropic compression work, isentropic compression for compressor. for compressor, compression divide by actual compression. So, C_p T_2 minus T_1 W actual.

Now, this gives like η isentropic given 0.8 and C_p given in the problem is 1 and T_2 value we just calculated 543.43 and T_1 value we know 300, W actual. therefore, W actual equals

303.75. This is the answer we are asking, unit kilowatt. Now, to calculate efficiency first Q in equals $C_p T_3$ minus T_2 bracket η isentropic W actual by W isentropic turbine so H_2 minus H_1 H_2 dash minus H_1 okay so 0.8 the value is given so this will be giving 0.8 equals five four three point four three minus 300 divided by T_2 dash minus 300

Therefore, T_2 equals 604.3 k. So, we got the T_2 value. We got T_2 dash value. Q in equals 1 and T_3 you can we are calculating here T_3 value we know 1400 given and T_2 value we just calculated 604.3. So, this is giving 795.7 kg per kg and

Problem-1

GATE 2013

In a simple Brayton cycle, the pressure ratio is 8, and temperatures at the entrance of the compressor and turbine are 300 K and 1400 K, respectively. Both compressor and gas turbine have isentropic efficiencies equal to 0.8. For the gas, assume a constant value of c_p (specific heat at constant pressure) equal to 1 kJ/kg-K and a specific heat ratio of 1.4. Neglect changes in kinetic and potential energies. Calculate: Power required in the compressor.
Calculate Thermal efficiency.

$p_2/p_1 = 8$
 $T_1 = 300 \text{ K}$
 $T_3 = 1400 \text{ K}$
 $\eta_{isen} = 0.8$
 $\gamma = 1.4$
 $c_p = 1 \text{ kJ/kg-K}$

$T_2 = T_1 \left(\frac{p_2}{p_1}\right)^{\frac{\gamma-1}{\gamma}} = 300 \times 8^{\frac{1.4-1}{1.4}} = 604.3 \text{ K}$

$T_4 = T_3 \left(\frac{p_4}{p_3}\right)^{\frac{\gamma-1}{\gamma}} = 1400 \times \left(\frac{1}{8}\right)^{\frac{1.4-1}{1.4}} = 898.288 \text{ K}$

$Q_{in} = c_p (T_3 - T_2) = 1 \times (1400 - 604.3) = 795.7 \text{ kJ/kg}$

$W_{turbine} = c_p (T_3 - T_4) = 1 \times (1400 - 898.288) = 501.712 \text{ kJ/kg}$

$W_{compressor} = c_p (T_2 - T_1) = 1 \times (604.3 - 300) = 304.3 \text{ kJ/kg}$

$\eta_{th} = \frac{W_{turbine} - W_{compressor}}{Q_{in}} = \frac{501.712 - 304.3}{795.7} = 0.248 = 24.8\%$

Process 3 to 4, T_3 by T_4 equals P_3 by P_4 gamma minus 1 by gamma. Therefore, T_4 equals T_3 by P_3 by P_4 gamma minus 1 by gamma equals 1400. divided by 8, 1.4, minus 1, 1.4. this value is coming 772.86 k. Now, η isentropic, η isentropic equals W actual by W isentropic. implies 0.8 h_3 minus h_4 dash h_3 minus h_4 equals t_3 minus t_4 dash t_3 minus t_4 so therefore t_4 dash equals 898.288 k so

Now, W actual equals $C_p T_3$ minus T_4 equals 1400 minus 898.288. So, this is giving 501.712. What is η isen, η thermal equals W actual minus W actual turbine by W actual compressor divided by Q in. this is giving 501.712 minus 304.3 divided by 795.7 into 100.

So, this is giving 24.8 percent. So, this is your answer. Another problem is an ideal Brayden cycle operating between two pressure limits 1 bar and 6 bar. That means T - s diagram showing this is 1, this is 6, within that pressure limit is working. Temperature also given, minimum temperature means here minimum temperature will be there 300, maximum will be 1500.

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The ratio of specific heat, gamma is giving 1.4. it will be like this the approximate final temperature in kelvin at the end of compression expansion process respectively so t 1 2 3 4 so t 2 t compression expansion process so t 4 we have to find so work is very simple actually here cp by cv given so adiabatic condition if i apply t 2 by t 1 equals P 2 by P 1 gamma minus 1 by gamma equals imply imply or 300 divided by T 2 1 by 6 0.286 so this will be giving T 2 equals 500 K Now, again for the process 3-4, 3-4 process if we consider then T3 by T4 very simple problem this one equals P4 by P3 gamma minus 1 by gamma. from there you are getting T4 equals 900 k, this is your answer.

Problem 2
 An ideal Brayton cycle, operating between the pressure limits of 1 bar and 6 bar, has minimum and maximum temperatures of 300 K and 1500 K. The ratio of specific heat of the working fluid is 1.4. The approximate final temperatures in Kelvin at the end of compression and expansion processes are respectively ____.

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again similar problem in a gas turbine cycle hot combustion product with the specific case Cp is given 98 Cp also given. Cp by Cp can calculate from here and enters turbine 20 bar and 1500 k exits at 1 bar the isentropic efficiency given work developed by the turbine per kg of gas flow is. you have to calculate work per kg of gas. first you draw TS diagram with two pressure line 1 to 2, 3 to 4. 1, 2 dash, 3, 4 dash.

This is given 20 bar, this is given 1 bar, lower one 1 bar. Cp is given 0.98, H isentropic is given 0.94, eta isentropic 0.94 Cv, Cp given, T3 also given T3 equals 1500, this is T3, P2, this is P2 equals P3, P1 equals P4 equals 1 bar and gamma you can calculate Cp by Cv equals 0.98 by 0.7538 these values are given.

it is coming 1.3. Now, applying general equation for the reversible adiabatic process in 3 and 4. T3 by T4 equals P3 by P4 gamma minus 1 by gamma. this value will be giving T4 equals 751.37 you see the previous calculations so same way you can calculate eta isentropic actual output divided by ideal so T3 minus T4 dash T3 minus T4 so this will be giving 0.94 equals 1500 minus T4 dash

1500 minus 751.37 so this will be giving t4 dash equals 796.3 k so turbine work wt equals cp t3 minus t4 equals We can write 0.98. The value is given already. T3 1500 given 796.3. It is 793.796.3.

this is giving 698.64 kg per kg. This is your answer. you see another problem in a gas domain plant working on a Brayton cycle with regenerator. TS diagram we draw TS 1, 2, 3, 4 Brayton cycle with regenerators efficiency 75 percent 75% regenerative efficiency is there, effectiveness is there, air inlets, air inlet 0.1 MPa, 0.1 MPa, 30 degree centigrade, so T1 equals 30 degree centigrade, pressure ratio 6, maximum cycle temperature is given is equals to 900 T3.

Problem 2
 In a gas turbine, hot combustion products with the specific heats $c_p = 0.98 \text{ kJ/kgK}$, and $c_v = 0.7538 \text{ kJ/kgK}$ enter the turbine at 20 bar, and $1500 = T_3$ K exit at 1 bar. The isentropic efficiency of the turbine is 0.94. The work developed by the turbine per kg of gas flow is _____?

Handwritten calculations:
 $\gamma = \frac{c_p}{c_v} = \frac{0.98}{0.7538} = 1.4$
 $\frac{T_3}{T_4} = \left(\frac{P_3}{P_4}\right)^{\frac{\gamma-1}{\gamma}} = 6^{\frac{1.4-1}{1.4}} = 2.5137$
 $\therefore T_4 = \frac{1500}{2.5137} = 596.7 \text{ K}$
 $\eta_{\text{turb}} = \frac{\text{actual}}{\text{ideal}} = \frac{T_3 - T_4'}{T_3 - T_4}$
 $0.94 = \frac{1500 - T_4'}{1500 - 596.7}$
 $T_4' = 796.3 \text{ K}$
 Turbine work = $WT = c_p(T_3 - T_4') = 0.98(1500 - 796.3)$

TS diagram showing a cycle with points 1, 2, 3, 4. $P_1 = P_4 = 1 \text{ bar}$, $P_2 = P_3 = 20 \text{ bar}$. $T_3 = 1500 \text{ K}$. $\eta_{\text{turb}} = 0.94$.

If the turbine and compressor have efficiency 80%, both are having efficiency 80%, find increase in cycle efficiency due to regeneration. again T2 dash sorry T4 dash. Now, T2 minus by T1 equals P2 by P1 gamma minus 1 by gamma equals T3 by T4, 4s or 4 equals 60.4 by 1.4, 1.668.

T_2 equals 303 into 1.668, 505. T_4 equals 1173 by 1.668 equals 705K. T_2 minus T_1 , T_2 . better I will write this is 2SS. T_4 SS.

T_2 minus T_1 T_2 s minus T_1 eta c equals 505 minus 303 divided 0.8 equals 252.k. So, W_t equals first T_3 minus T_4 eta t t_3 minus t_4 equals 0.8 1173 minus 705 375 k now t_w equals h_3 minus h_4 equals c_p t_3 minus t_4 is 1.005 this c_p value t_3 minus t_4 375 equals 376.88 kj per kg so w_c equals c_p t_2 minus t_1 1.005 into 252 253.26 kj per kg okay so t_2 equals 252 plus 303 five five five k q_1 equals h_3 minus h_2 equals c_p t_3 minus t_2 so this value is giving six to one point zero nine kg per kg so eta equals w

Problem-3
 In a gas turbine plant, working on the Brayton cycle with a regenerator of 75% effectiveness, the air at the inlet to the compressor is at 0.1MPa, 30 C, the pressure ratio is 6, and the max cycle temp is 900 C. If the turbine and the compressor have an efficiency of 80%, find ~~the~~ increase in cycle efficiency due to regeneration.

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w_c divided by q_1 376.88 minus 253.26 6 to 1.09 equals 19.9 percent with regenerator now we have added regenerator T_4 equals T_3 minus 375 equals 1173 minus 375 798 okay regenerative effectiveness effectiveness because T_6 minus T_2 divided by T_4 minus T_2 equals 0.75 so T_6 minus 555 because 0.75 798 minus 555 therefore T_6 equals 737 point 3 K or Q_1 equals H_3 minus H_6 equals C_p T_3 minus T_6 1 point 0 0 6 1 1 7 3 minus 7 3 7 1 3 plus 4 3 7 point 8 8 kg per kg now the blue net remaining the same so eta equals w_{net} q_1 equals one two three point six two four three seven point nine equals one zero point two eight three seven equals twenty eight point three seven percent increase

due to regeneration equals 0.2837 minus 0.199 0.199 equals 0.4256 42.56 percent. this is your answer. thank you very much for today's lecture. Next day, we will start new topic, we will discuss turbine thing. Thank you very much.

with reagent.

$$T_s = T_3 - 375 = 1173 - 375 = 798 \text{ K}$$

$$\text{Reagan efficiency} = \frac{T_s - T_4}{T_s - T_3} = 0.25$$

$$T_s - 515 = 0.25 (798 - 515)$$

$$\therefore T_4 = 732.3 \text{ K}$$

$$a) \dot{Q}_1 = h_s - h_c = C_p (T_3 - T_4) = 1 \text{ mol} (1173 - 732.3) = 440.7 \text{ kJ/kg}$$

Work remaining the same.

$$\eta = \frac{W_{\text{net}}}{\dot{Q}_1} = \frac{123.62}{440.75} = \frac{0.2837 - 0.199}{0.199} = 0.4258 = 42.58\%$$

η increases due to reagent.



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