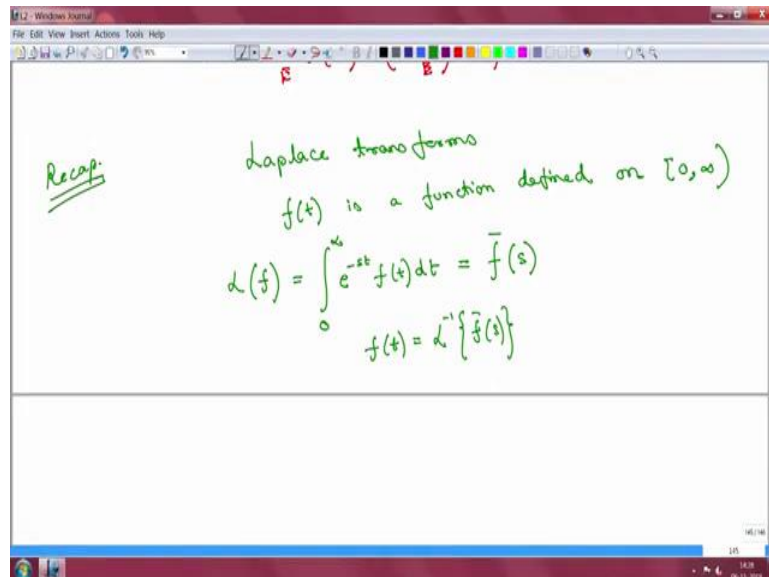


Introduction to Soft Matter
Professor Alope Kumar
Department of Mechanical Engineering
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Lecture 28
Three Parameter Model Cont

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So, last class, we were looking at we just started looking at Laplace transforms with the intention of applying them to solve the ordinary differential equation that we had derived the constitutive relationship. So let us, we have not finished with the Laplace recap of the Laplace transform. So, I will just quickly go ahead and recap for a few other things. So, with this definition, you can apply Laplace to the Laplace operator to lot of different functions and you can show what the Laplace looks like.

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Handwritten notes on a digital whiteboard:

$$\mathcal{L}\{f(t)\} = \int_0^{\infty} e^{-st} f(t) dt = f(s)$$

$$f(t) = \mathcal{L}^{-1}\{f(s)\}$$

$f(t)$	$\mathcal{L}\{f(t)\}$
1	$\frac{1}{s}$
t	$\frac{1}{s^2}$
t^2	$\frac{2!}{s^3}$
e^{at}	$\frac{1}{s-a}$

Handwritten notes on a digital whiteboard:

$f(t)$	$\mathcal{L}\{f(t)\}$
1	$\frac{1}{s}$
t	$\frac{1}{s^2}$
t^2	$\frac{2!}{s^3}$
e^{at}	$\frac{1}{s-a}$
$\cos \omega t$	$\frac{s}{s^2 + \omega^2}$
$\sin(\omega t)$	$\frac{\omega}{s^2 + \omega^2}$

So, some commonly used Laplace transforms are, so if we have a function $f(t)$, this is I am just going to list the common Laplacian and if you want you can just go over them and you can derive them for yourself. So, for example, Laplace of 1 comes out as $\frac{1}{s}$. Laplace of T comes out as $\frac{1}{s^2}$, Laplace of T^2 comes out as $\frac{2!}{s^3}$.

The Laplace of e^{at} comes out as $\frac{1}{s-a}$. The couple of more. So, for example, Laplace of $\cos \omega t$ comes out as $\frac{s}{s^2 + \omega^2}$ and the Laplace of $\sin \omega t$ comes out as $\frac{\omega}{s^2 + \omega^2}$.

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Handwritten notes on Laplace transforms in a Windows Journal window. The window title is "112 - Windows Journal". The notes are as follows:

$$\mathcal{L}\{f\} = \int_0^\infty e^{-st} f(t) dt = f(s)$$

$$f(t) = \mathcal{L}^{-1}\{f(s)\}$$

$f(t)$	$\mathcal{L}\{f\}$
1	$1/s$
t	$1/s^2$
t^2	$2!/s^3$
e^{at}	$1/(s-a)$

Other properties listed:

$$\mathcal{L}\{\dot{f}\} = s \mathcal{L}\{f\} - f(0)$$

$$\mathcal{L}\{\ddot{f}\} = s^2 \mathcal{L}\{f\} - s f(0) - \dot{f}(0)$$

$$\mathcal{L}\{f(t-a)u(t-a)\} = e^{-as} \bar{f}(s)$$

$$\mathcal{L}\{\delta(t-a)\} = e^{-as}$$

Some other properties that are important, let me just divide this space here. So, Laplace of the derivative of f comes out as S times Laplace of f minus f of at 0 , evaluated at 0 , the Laplacian of f double dot comes out as S square $1 f$, Laplace of f minus S times f of 0 minus f dot 0 . Laplace of $f t$ minus a multiplied by the Heaviside function translocated at t minus a or the Heaviside function which is moved to a , this comes out as e to the power minus $a s$ f bar of s . The Laplace of the Dirac delta function at a is simply e to the power minus $a s$.

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Handwritten notes on Laplace transforms in a Windows Journal window. The window title is "112 - Windows Journal". The notes are as follows:

$f(t)$	$\mathcal{L}\{f\}$
1	$1/s$
t	$1/s^2$
t^2	$2!/s^3$
e^{at}	$1/(s-a)$
$\cos \omega t$	$s/(s^2 + \omega^2)$
$\sin(\omega t)$	$\omega/(s^2 + \omega^2)$

Other properties listed:

$$\mathcal{L}\{\dot{f}\} = s \mathcal{L}\{f\} - f(0)$$

$$\mathcal{L}\{\ddot{f}\} = s^2 \mathcal{L}\{f\} - s f(0) - \dot{f}(0)$$

$$\mathcal{L}\{f(t-a)u(t-a)\} = e^{-as} \bar{f}(s)$$

$$\mathcal{L}\{\delta(t-a)\} = e^{-as}$$

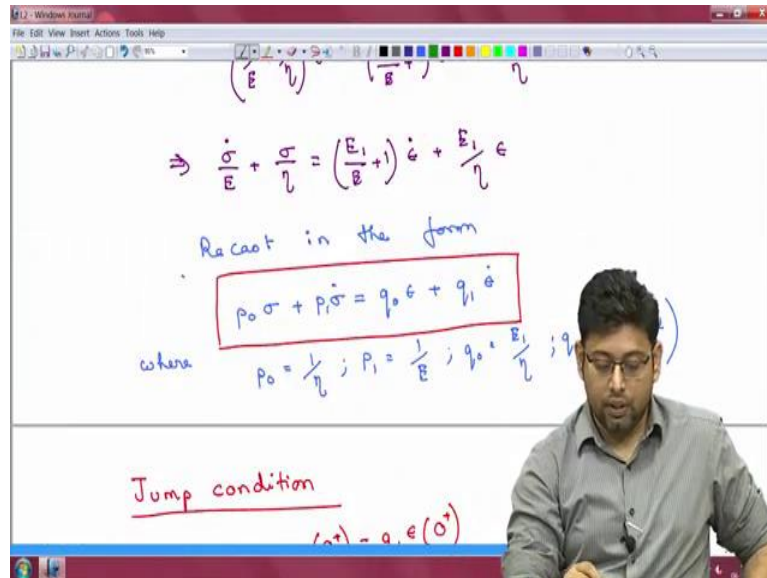
$$h(t) = (f * g)(t) = \int_0^t f(\tau) g(t-\tau) d\tau$$

$$\Rightarrow \bar{h}(s) = \bar{f}(s) \bar{g}(s)$$

And finally, if you have a function h , which is defined as the convolution of two functions, so let us say you have function h which is defined as convolution of two functions here this star

is the convolution operator defined as this particular integral f of sorry, f of tau, g of t minus tau delta, then the Laplace of h is the Laplace of the two functions simply multiplied with each other.

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$$\Rightarrow \frac{\dot{\sigma}}{\epsilon} + \frac{\sigma}{\eta} = \left(\frac{\epsilon_1}{\epsilon} + 1\right) \dot{\epsilon} + \frac{\epsilon_1}{\eta} \epsilon$$

Recast in the form

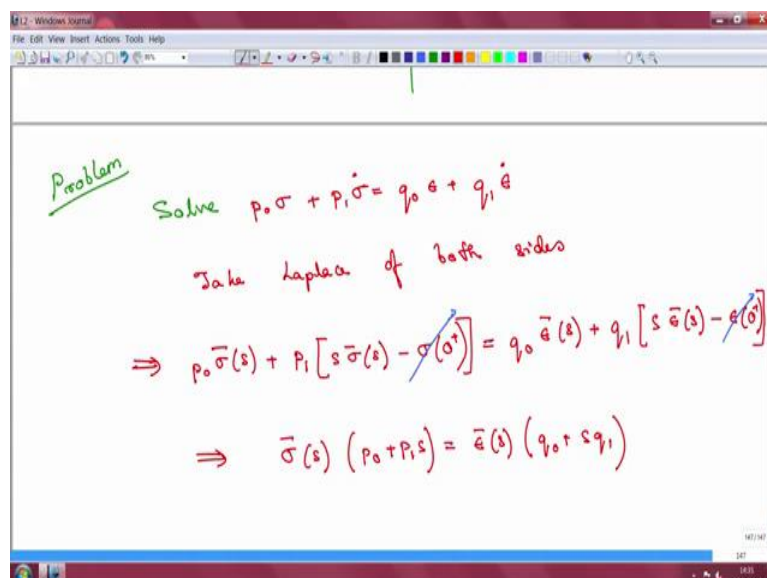
$$p_0 \sigma + p_1 \dot{\sigma} = q_0 \epsilon + q_1 \dot{\epsilon}$$

where $p_0 = \frac{1}{\eta}$; $p_1 = \frac{1}{\epsilon}$; $q_0 = \frac{\epsilon_1}{\eta}$; $q_1 = \epsilon_1$

Jump condition
 $\sigma(0^+) = \sigma(0^-)$

So, now with this quick recap we are ready to solve the particular ODE that we wanted to, what is that ODE in question? It was this, we have $p_0 \sigma + p_1 \dot{\sigma} = q_0 \epsilon + q_1 \dot{\epsilon}$, we want to solve this and at the beginning we will not care what p_0 , p_1 , et cetera. are. Once we solve this, then we will insert these values to get our result.

(Refer Slide Time: 5:39)



Problem

Solve $p_0 \sigma + p_1 \dot{\sigma} = q_0 \epsilon + q_1 \dot{\epsilon}$

Take Laplace of both sides

$$\Rightarrow p_0 \bar{\sigma}(s) + p_1 [s \bar{\sigma}(s) - \sigma(0^+)] = q_0 \bar{\epsilon}(s) + q_1 [s \bar{\epsilon}(s) - \epsilon(0^+)]$$

$$\Rightarrow \bar{\sigma}(s) (p_0 + p_1 s) = \bar{\epsilon}(s) (q_0 + s q_1)$$

So, if you take the Laplace of this, you will. So maybe I will rewrite that equation we have. So, our problem is problem is solve this equation, $P_0 q + P_1 \dot{q}$ equal to ϵ dot. So, let us take Laplace of both sides and if you do that, you will end up with $P_0 \bar{q}(s) + P_1 s \bar{q}(s)$ plus P_1 , please note that this is now a derivative, so you are going to apply that particular form that we had just see, equal to $q_0 \bar{\epsilon}(s) + q_1 s \bar{\epsilon}(s) - \epsilon(0)$.

Now, I had already said that this is from the jump condition which we did not derive and that was left as a homework for you. We can take this quantity out. So, we have left with a simpler situation, we have $\bar{q}(s)$ into $P_0 + P_1 s$ equal to $\bar{\epsilon}(s)$ multiplied by $q_0 + q_1 s$. So, now, we have to take, so now we have to decide what we are solving for. So, this is very general Laplace one, the Laplace equivalent for that above equation. So, let us say we are solving the stress for the stress relaxation response.

(Refer Slide Time: 7:59)

$$\Rightarrow \bar{\sigma}(s) (p_0 + p_1 s) = \bar{\epsilon}(s) (q_0 + s q_1)$$

Case 1 Stress relaxation response
 $\epsilon(t) = \epsilon_0 H(t) \Rightarrow \bar{\epsilon}(s) = \frac{\epsilon_0}{s}$

$$\Rightarrow \bar{\sigma}(s) = \frac{\epsilon_0}{s} \frac{q_0 + s q_1}{p_0 + p_1 s}$$

$$= \frac{\epsilon_0}{s} \left[\frac{s + q_0/p_1}{s + p_0/p_1} \right] \cdot \frac{q_1}{p_1}$$

So, case one stress relaxation response. So, in this case, we know that $\epsilon(t)$ is equal to some ϵ_0 naught into the Heaviside function. So, if you take the Laplace of this, we know that $\bar{\epsilon}(s)$ is equal to ϵ_0 naught by s . Now, we can replace, so here we did not know what $\bar{\epsilon}(s)$ was and now we have a functional form for that. So, what we will do? Is just we will replace it, second.

So, let us go ahead and replace, so you have $\bar{q}(s)$ equal to $\bar{\epsilon}(s)$ naught s , you have $q_0 + q_1 s$ naught by s , q_1 , $P_0 + P_1 s$. So here we have, now, if you take the Laplace inverse, you will be able to compute what the stress is, but this is a sort of rational fraction format, if we

have to simplify this a little bit before we can easily take the Laplace. So, let us go ahead and try to simplify your right hand side.

So, left hand side you do not have to do anything anymore, the left hand side you have to, the right hand side has to be simplified. So, we will try to write it as s plus q naught by q_1 and. So, now I am going to try and get to a simple rational function format for this and I just divided here by q_1 , so I multiplied by q_1 , here we did that divided by P_1 .

(Refer Slide Time: 10:13)

$$\begin{aligned}
 \Rightarrow \bar{\sigma}(s) &= \frac{\epsilon_0}{s} \frac{q_0 \cdot s + q_1}{p_0 + p_1 s} \\
 &= \frac{\epsilon_0}{s} \left[\frac{s + q_1/q_0}{s + p_0/p_1} \right] \cdot \frac{q_1}{p_1} \\
 &= \epsilon_0 \left[\frac{q_1}{p_1} \cdot \frac{1}{s + p_0/p_1} + \frac{q_1}{p_1} \cdot \frac{q_0}{q_1} \cdot \frac{1}{s} \cdot \frac{1}{s + p_0/p_1} \right] \\
 &= \epsilon_0 \left[\frac{q_1}{p_1} \cdot \frac{1}{s + p_0/p_1} + \frac{q_0}{q_1} \cdot \frac{p_1}{p_0} \left(\frac{1}{s} - \frac{1}{s + p_0/p_1} \right) \right]
 \end{aligned}$$

So, now this is equivalent to ϵ_0 , this s I am going to take it inside and I am going to open the bracket here and I will see what I get. We have q_1 , P_1 , 1 by s plus P naught by P_1 , this s and this s it gets cancelled for this first case. In the second one we have q_1 on P_1 multiplied by q naught, q_1 into 1 by s , s plus, so this simplifies. So, this part I have already got into a form which I can easily take the Laplace inverse of.

If you look at the tables of the Laplace function the general Laplace functions that you know that we can easily take the Laplace inverse of this. So, we have, so this first part, we will leave it as it is, we still have to simplify the last part a little bit before we can take the Laplace inverse. So, what we will do is? This part, we are going to write it as two other functions q naught by q_1 , sorry I have to this will be P_1 by P_0 .

(Refer Slide Time: 12:48)

The image shows a handwritten derivation in a software window. At the top, there is a small expression: $\frac{1}{s + \frac{p_0}{p_1}}$. Below it, the first equation is:

$$= \epsilon_0 \left[\frac{q_1}{p_1} \cdot \frac{1}{s + \frac{p_0}{p_1}} + \frac{q_0}{p_1} \cdot \frac{p_1}{p_0} \left(\frac{1}{s} - \frac{1}{s + \frac{p_0}{p_1}} \right) \right]$$

The second equation is:

$$= \epsilon_0 \left[\frac{q_0}{p_0} \cdot \frac{1}{s} + \frac{1}{s + \frac{p_0}{p_1}} \left(\frac{q_1}{p_1} - \frac{q_0}{p_0} \right) \right]$$

The final result is:

$$\Rightarrow \sigma(t) = \epsilon_0 \left[\frac{q_0}{p_0} + \left(\frac{q_1}{p_1} - \frac{q_0}{p_0} \right) e^{-\frac{p_0}{p_1} t} \right]$$

So, now if you take the two, so now you have these two are common. So you can combine these two, and then you have rational function of the form 1 by s. So, you have basically two separate rational functions, one is q naught by P naught, 1 by s, this, so this is not q 1 this is P 1. So, then you have the other one part which is s naught plus P naught by P 1 and that is q 1 by P 1 minus q naught by P naught.

So, this implies if you now take the Laplace inverse of both sides, you will get on the left hand side you will get the stress sorry on the right hand side you on the left hand side you will get the stress on the right hand side you will recover these terms, so q 0. So, we know that the Laplace of 1 was 1 by s. So, this is just the fraction and on here you have q 1 by P 1 minus q 0 by P naught, e to the power minus P 0 by P 1 t.

(Refer Slide Time: 14:40)

$$= \epsilon_0 \left[\frac{q_0}{p_0} \cdot \frac{1}{s} + \frac{1}{s + p_0/p_1} \left(\frac{q_1}{p_1} - \frac{q_0}{p_0} \right) \right]$$

$$\Rightarrow \sigma(t) = \epsilon_0 \left[\frac{q_0}{p_0} + \left(\frac{q_1}{p_1} - \frac{q_0}{p_0} \right) e^{-p_0/p_1 t} \right]$$

$$\Rightarrow \frac{\sigma(t)}{\epsilon_0} = G(t) = G_\infty + (G_0 - G_\infty) e^{-p_0/p_1 t}$$

So, if I just divide these two, you have sigma t by epsilon 0, which is a stress relaxation function, right. So, we are seeking that anyway that we get and now that will have two parts, so this part which is a constant and then you have this part minus again the same fraction here. So, if I decide to call this G infinity, then you have the function can be written as some G naught minus G infinity e to the power minus P 0 by P 1 t.

(Refer Slide Time: 15:43)

$$\Rightarrow \frac{\dot{\sigma}}{E} + \frac{\sigma}{\eta} = \left(\frac{E_1}{E} + 1 \right) \dot{\epsilon} + \frac{E_1}{\eta} \epsilon$$

Recast in the form

$$p_0 \sigma + p_1 \dot{\sigma} = q_0 \epsilon + q_1 \dot{\epsilon}$$

where

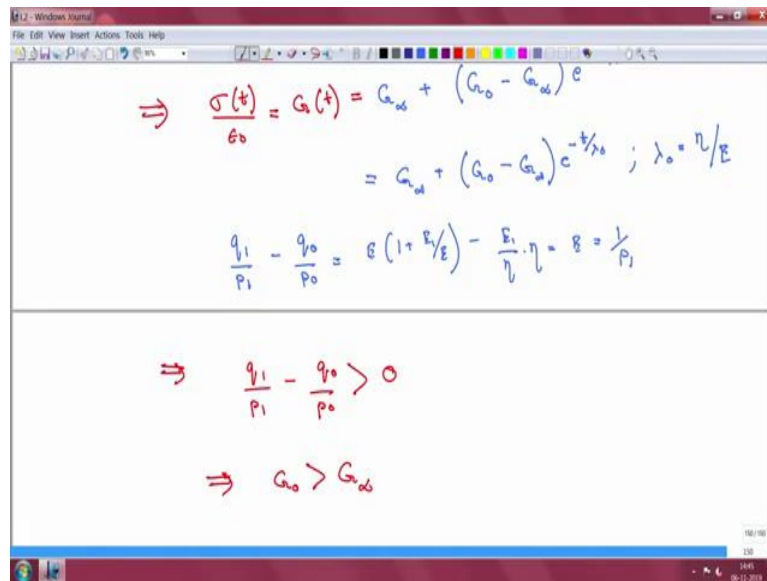
$$p_0 = \frac{1}{\eta} ; p_1 = \frac{1}{E} ; q_0 = \frac{E_1}{\eta} ; q_1 = \left(1 + \frac{E_1}{E} \right)$$

Jump condition

$$p_1 \sigma(0^+) = q_1 \epsilon(0^+)$$

What is P 0 by p 1? Let us quickly go back and check. So, P 0 was 1 by eta and P 1 is 1 by E. So, this is back to our lambda, eta by E.

(Refer Slide Time: 15:55)



Handwritten mathematical derivations in a software window:

$$\Rightarrow \frac{\sigma(t)}{\epsilon_0} = G(t) = G_\infty + (G_0 - G_\infty)e^{-t/\lambda_0}$$

$$= G_\infty + (G_0 - G_\infty)e^{-t/\lambda_0}; \lambda_0 = \eta/\epsilon$$

$$\frac{q_1}{P_1} - \frac{q_0}{P_0} = \epsilon \left(1 + \frac{\epsilon_1}{\epsilon}\right) - \frac{\epsilon_1}{\eta} \cdot \eta = \epsilon = \frac{1}{P_1}$$

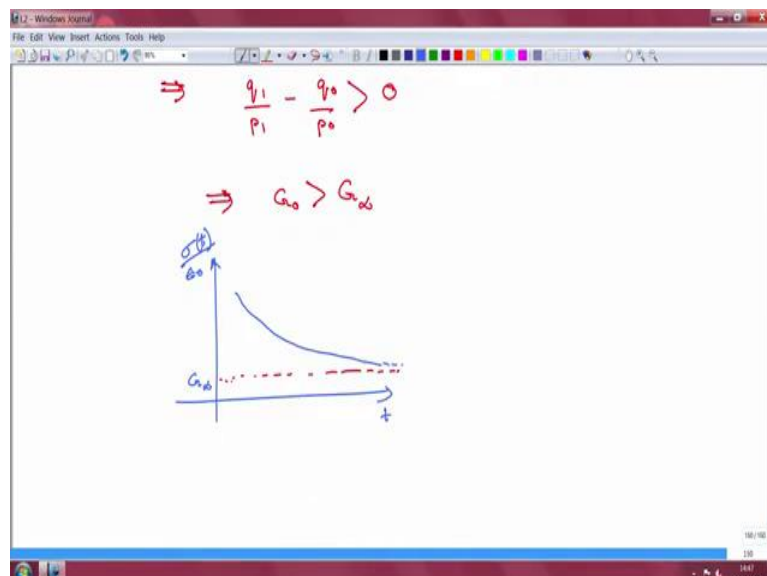
$$\Rightarrow \frac{q_1}{P_1} - \frac{q_0}{P_0} > 0$$

$$\Rightarrow G_0 > G_\infty$$

So, this quantity I can replace and I can or I can just say that is equal to this G infinity plus G_0 minus G infinity e to the power minus t by λ_0 . Actually, what we will do is? We will call this λ_0 and we will see why that has to be there, where λ_0 is now η . Now, convince yourself that this quantity is in fact positive.

So, what is q_1 by P_1 minus q_0 by P_0 ? Can we quickly calculate this? So, if you calculate this, this should come out to your ϵ times of 1 plus ϵ_1 by ϵ minus ϵ_1 , η into η equal to ϵ equal to 1 by P_1 . So, this fraction is greater than 0 . So, is actually a positive number, which implies at your G_0 is greater than G infinity.

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So, if you wanted to plot this curve, what it will look like? It will be some exponential function, some constant, plus an exponential decay will there, so the exponential decay will be will not go down to 0, because you always have this constant G infinity that will be left. So, this is your σ by ϵ , then this will be G t , maybe it is not legible, let me rewrite this sorry.

So, this was your plotting that this is your G infinity. So, G infinity is a nonzero value, so, your stress relaxation function does not decay down to 0, which is a viscoelastic solid type of response. And that is what we had said in the very-very beginning itself that this by intuition are simple physics based arguments.

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Consider the model on L.H.S.

Diagram: A Maxwell model consisting of a spring with displacement $\Delta x_s, E, F_s(t)$ and a dashpot with displacement $\Delta x_m, \eta, F_m(t)$ in parallel. The total displacement is Δx and the total force is $F(t)$.

Recall that for Maxwell model

$$\frac{1}{E} \frac{d\sigma}{dt} + \frac{1}{\eta} \sigma = \frac{d\epsilon}{dt}$$

$$\left(\frac{D}{E} + \frac{1}{\eta} \right) \sigma = D \epsilon$$

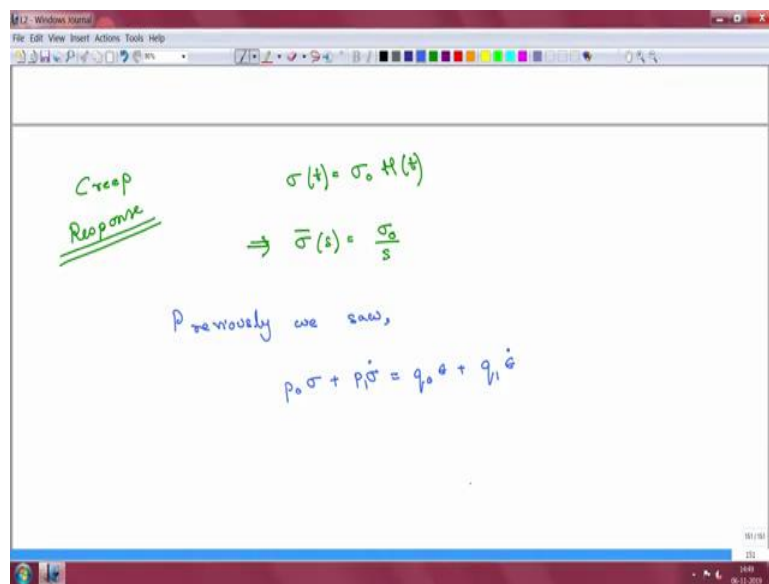
where $D \equiv$ differential operator is $\frac{d}{dt}$

From Geometry

$$\Delta x = \Delta x_m = \Delta x_s$$

We had said that this model has to behave like a solid body, because the displacement and the system, the springs displacement is arresting the total displacement of the system. So, this has to be a solid like situation and that is what exactly what we have found out that this behaviour does satisfy that of a simple solid or a viscoelastic that this response is in line with what we would expect from viscoelastic solid.

(Refer Slide Time: 19:56)



The slide is titled "Creep Response" in green. It contains the following equations and text:

$$\sigma(t) = \sigma_0 H(t)$$
$$\Rightarrow \bar{\sigma}(s) = \frac{\sigma_0}{s}$$

Previously we saw,

$$p_0 \sigma + p_1 \dot{\sigma} = q_0 \epsilon + q_1 \dot{\epsilon}$$

So, similarly, to what we have done, we can now also look at Creep response. So, in Creep response you have your $\sigma(t)$ is equal to some σ_0 into our Heaviside function and this implies again that if you take the Laplace of both sides end up with σ_0/s . So, previously we saw and we will just rewrite that equation because otherwise it becomes difficult to solve it without looking at it.

So, what we saw was, $p_0 \sigma + p_1 \dot{\sigma}$ was equal to some $q_0 \epsilon + q_1 \dot{\epsilon}$. There is one more thing I want to point out that there are four coefficients here in the ordinary differential equation, but your original model only has three constants E_1 , E and η . So, the 4 coefficients here are not independent, there are actually in reality only 3 coefficients and the fourth one is a dependent one.

(Refer Slide Time: 21:25)

Creep Response

$$\sigma(t) = \sigma_0 H(t)$$

$$\Rightarrow \bar{\sigma}(s) = \frac{\sigma_0}{s}$$

Previously we saw,

$$p_0 \sigma + p_1 \dot{\sigma} = q_0 \epsilon + q_1 \dot{\epsilon}$$

$$\Rightarrow \bar{\sigma}(s) (p_0 + p_1 s) = \bar{\epsilon}(s) (q_0 + q_1 s)$$

Now, replace $\bar{\sigma}(s)$ with $\frac{\sigma_0}{s}$.

Take Laplace of both sides

$$\Rightarrow p_0 \bar{\sigma}(s) + p_1 [s \bar{\sigma}(s) - \sigma(0^+)] = q_0 \bar{\epsilon}(s) + q_1 [s \bar{\epsilon}(s) - \epsilon(0^+)]$$

$$\Rightarrow \bar{\sigma}(s) (p_0 + p_1 s) = \bar{\epsilon}(s) (q_0 + q_1 s)$$

Case 1 Stress relaxation response

$$\epsilon(t) = \epsilon_0 H(t) \Rightarrow \bar{\epsilon}(s) = \frac{\epsilon_0}{s}$$

$$\Rightarrow \bar{\sigma}(s) = \frac{\epsilon_0}{s} \frac{q_0 + s q_1}{p_0 + s p_1}$$

So, now, we again we are this part is something that we can still reuse. So, we will just write that, sigma s, so, from the constitutive equation you have, this is generally true, for this ordinary differential equation, we have not applied anything till now. So, now apply now, replace sigma bar s with sigma naught by s, and what I want you to do for the next class? Is I would like you to give it a try, where you replace this and you try to solve the equation all over again.

And now, you are instead of solving for the stress relaxation function, you are going to solve for the Creep function. So, I would like you to do that and I would like you to convince yourself that the system response is still in line with the idea of a viscoelastic solid, if it is not, then we have a problem. So for today, what we will do is we will stop here, and I would

like you to do this and next class I will work it out by myself. So we will stop here today, thanks.