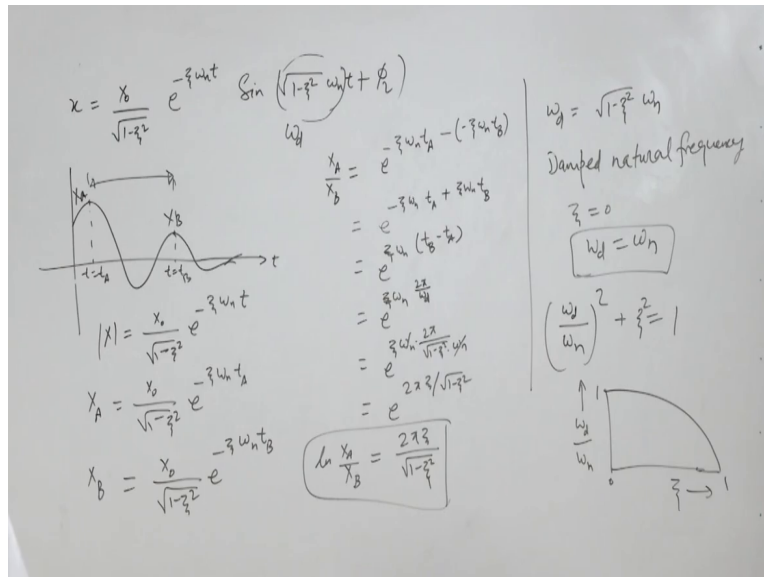


Introduction to Mechanical Vibration
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Lecture - 10
Coulomb Damping

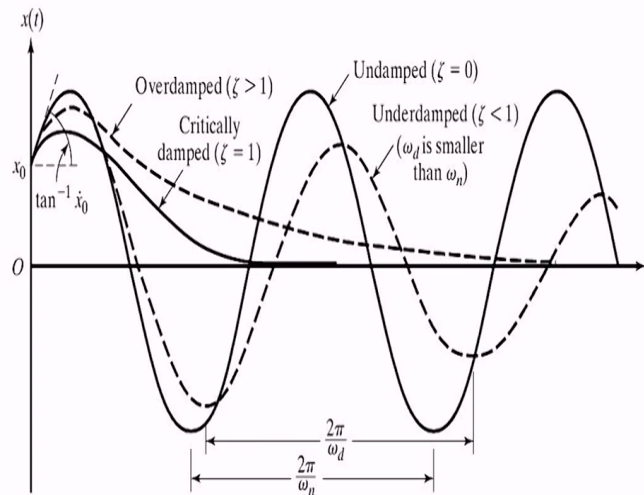
Welcome to the lecture on logarithmic decrement and free vibration under coulomb damping. So we have already discussed the viscous damping case and we discussed the over damping, critical damping and under damping.

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And in case of under damping we find the response that is $x = X_0$ by root $1 - \zeta^2$ square exponential e power $-\zeta \omega_n t$ $\sin$ root $1 - \zeta^2$ square $\omega_n t + \phi$ here ϕ is the phase, so and this - this all the three responses we can compare on in the figure shown here so we can see here we have this over damped system $\zeta > 1$.

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And we can see that okay through all the initial conditions they start and it is reaching to the equilibrium or 0 for this amount of time, then the critical damping $\zeta = 1$ and this is reaching to 0 here and we can see both are aperiodic motion they do not do any oscillations and we see that the critical damping is faster than the over damping and we said that when we increase damping the system becomes the motion becomes slower.

And critical damping is the least amount of damping that will make the system to reach to 0 or equilibrium. now comes to the under damping system so here is the under damping system this is undamped system $\zeta = 0$ means there is no damping so system is following the harmonic motion and we have the under damped $\zeta < 1$ and we see that this system is the amplitude is going to decay with time.

Now comes to we come to discuss logarithmic decrement that is relevant to the under damped system and logarithmic decrement is the, it shows the rate of decay in an under damped system because underdamped system has oscillation - oscillations that decay with time in the exponential manner so logarithmic decrement it is the ratio of the two successive amplitudes on natural log so we say $\delta = \ln X_1 \text{ by } X_2$, where X_1 is the first amplitude and X_2 is the next amplitude.

So we can find out let us say this is our system so let us say this is A this is B, $X_A X_B$ the amplitude here, so because our amplitude - amplitude is given as X_0 by $\sqrt{1 - \zeta^2}$ in to $e^{-\zeta \omega_n t}$, so our amplitude is given like this. now if we say X_A that is a time $t = t_A$ and this is time $t = t_B$ this is time, so X_0 by $\sqrt{1 - \zeta^2}$ $e^{-\zeta \omega_n t_A}$ and $e^{-\zeta \omega_n t_B}$.

And similarly $X_B = X_0$ by $\sqrt{1 - \zeta^2}$ $e^{-\zeta \omega_n t_B}$, so they are X_A and X_B they are two consecutive amplitudes of the under damped system motion, so we got X_A and X_B . now we put X_A by X_B - X_A by $X_B =$ so this will be cancelled out we will have $e^{-\zeta \omega_n t_A} / e^{-\zeta \omega_n t_B}$ so this is equal to $e^{\zeta \omega_n t_A - \zeta \omega_n t_B}$.

So here we can write $e^{\zeta \omega_n t_B - \zeta \omega_n t_A}$, $e^{\zeta \omega_n t_B - \zeta \omega_n t_A}$, this is $t_B - t_A$ and this is nothing but the time - the time period because this is the time between two period and what is the time period we can see the time period is coming from this because this is the ω_d we call it ω_d , that is $\omega_d = \sqrt{1 - \zeta^2} \omega_n$, so this is called damped natural frequency - damped natural frequency.

When $\zeta = 0$ $\omega_d = \omega_n$, so the system will vibrate with this frequency when we have a damping ζ while the undamped system was vibrating with its own natural frequency that was ω_n , so ω_d is $\sqrt{\omega_n^2 - \zeta^2}$ here we can write also ω_d by $\omega_n \sqrt{1 - \zeta^2}$, so this is showing some ellipse and we can plot this ω_d by ω_n here and this is ζ .

So this is showing an ellipse so here $1 / \omega_d$, now here we can write $t_B - t_A$ that is one time period, so it is $2\pi / \omega_d$ and $= e^{\zeta \omega_n (2\pi / \omega_d)}$ $= e^{2\pi \zeta \omega_n / \omega_d}$ $= e^{2\pi \zeta / \sqrt{1 - \zeta^2}}$, so ω_n and cancelled out we find $e^{2\pi \zeta / \sqrt{1 - \zeta^2}}$, now we take the natural log $\ln X_A / X_B = 2\pi \zeta / \sqrt{1 - \zeta^2}$. So here we see that this logarithmic decrement is depending nothing.

But one - only on the damping factor zeta, when we have a zeta much less than 1, so we can write this as $\delta = \ln X_A \text{ by } X_B = 2\pi \zeta$, so this is an approximate relation for the logarithmic decrement, okay.

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The image shows a handwritten derivation of the logarithmic decrement formula. On the left, a graph of displacement x versus time t is shown, with peaks labeled x_A and x_B at times t_A and t_B respectively. The equation for x is given as $x = \frac{x_0}{\sqrt{1-\zeta^2}} e^{-\zeta \omega_n t} \sin(\sqrt{1-\zeta^2} \omega_n t + \phi)$. The amplitude at time t_A is $x_A = \frac{x_0}{\sqrt{1-\zeta^2}} e^{-\zeta \omega_n t_A}$ and at time t_B is $x_B = \frac{x_0}{\sqrt{1-\zeta^2}} e^{-\zeta \omega_n t_B}$. The ratio of these amplitudes is $\frac{x_A}{x_B} = e^{-\zeta \omega_n (t_A - t_B)}$. The time difference $t_A - t_B$ is the period $T_d = \frac{2\pi}{\omega_d} = \frac{2\pi}{\omega_n \sqrt{1-\zeta^2}}$. Substituting this into the ratio gives $\ln \frac{x_A}{x_B} = \frac{2\pi \zeta}{\sqrt{1-\zeta^2}}$. On the right, the logarithmic decrement δ is defined as $\delta = \ln \frac{x_0}{x_1} = \ln \frac{x_1}{x_2} = \dots = \ln \frac{x_{n-1}}{x_n}$. Summing these n terms gives $n\delta = \ln \frac{x_0}{x_1} + \ln \frac{x_1}{x_2} + \dots + \ln \frac{x_{n-1}}{x_n} = \ln \frac{x_0}{x_n}$. Therefore, $\delta = \frac{1}{n} \ln \frac{x_0}{x_n}$. For $\zeta \ll 1$, the formula simplifies to $\delta = \ln \frac{x_A}{x_B} = 2\pi \zeta$.

Now we can also write $\delta = \ln X_0 \text{ by } X_1$ or $\ln X_1 \text{ by } X_2$ or $\ln X_{n-1} \text{ by } X_n$, because δ is the log of ratio of any two successive amplitudes, so it could be between X_0 and X_1 or X_1 and X_2 or X_{n-1} , now we - we do like we add this terms so we add so this is n terms so it is n into δ and when we add they will come into multiplication so $X_0 \text{ by } X_1 \text{ into } X_1 X_2 \text{ into } X_{n-1} \text{ by } X_n$. So all these intermediate terms simply cancelled out and we will get $\ln X_0 \text{ by } X_n$,

So here $\delta = \frac{1}{n} \ln X_0 \text{ by } X_n$, so this is another expression for the logarithmic decrement. So now here we complete the viscous damping free vibration under viscous damping. Now we start the another type of damping and free vibration of a system that as the coulomb damping. So we already discussed the coulomb damping, a damping due to friction between the two surfaces and there is the dry friction and coefficient of friction μ is there.

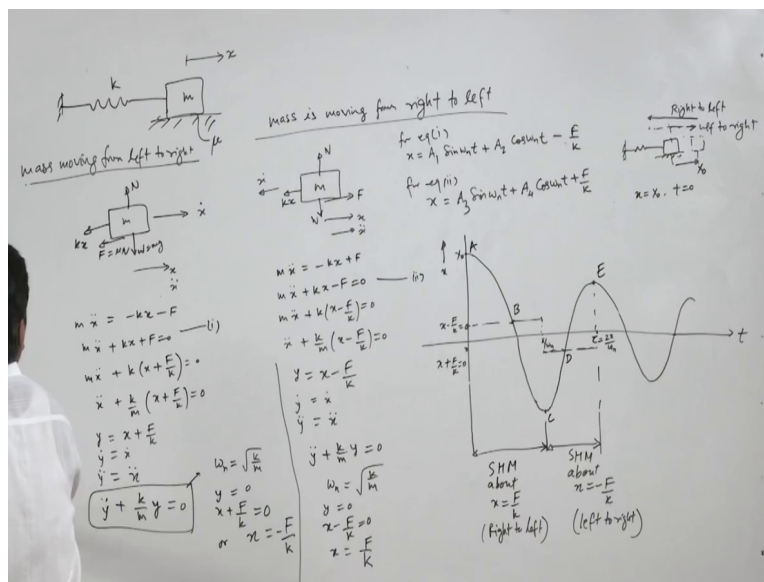
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COULOMB DAMPING

- Here the damping force is constant in magnitude but opposite in direction to that of the motion of the vibrating body.
- It is caused by friction between rubbing surfaces that either are dry or have insufficient lubrication.

And due to this coefficient of friction and due to the moment relative moment between the two bodies there is this kind of damping. So we have now this system so we represent the system.

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So we have a system, let us say this is our spring and this is our mass, this mass is moving on a surface that has a coefficient of friction μ , so this is m and k and here it is moving in this direction let us assume and there is the friction coefficient μ between the two surfaces on the it is moving on horizontal surface, so there is a coefficient of friction μ between the horizontal surface and the block m .

Now because the direction of the frictional force depends on the relative velocity, so in half cycle the mass will move from left to right and another half cycle it will move from right to left so we will have two cases of this moment, so let us say this is the mass, so mass moving from left to right, so we have this mass and it is moving from left to right, so it is moving this - this is the direction of $\dot{x}$ and this is x and $\ddot{x}$.

So the in this direction we assume x and $\ddot{x}$ as positive, so when it is moving we will have the spring force kx and because it is moving this side so the friction force will be working in this direction, so we will have a friction force here F that is equal to μN and N is the reaction and here W is the weight and N is the reaction here $W = mg$, similarly so we can write a equation of motion so this is a free body diagram.

Now we write $m\ddot{x} = \text{the forces}$ so $\ddot{x}$ direction is this and force $-kx$ and $-F$, so we have $m\ddot{x} + kx + F = 0$, now we can have this we can readjust these terms so $\ddot{x} + \frac{k}{m}x + \frac{F}{m} = 0$, so we can write $\ddot{x} + \frac{k}{m}x = -\frac{F}{m}$ by $k = 0$. Now we assume another axis $y = x + \frac{F}{k}$, so $\dot{y} = \dot{x}$ and $\ddot{y} = \ddot{x}$.

Now we put these terms here and we change the co-ordinate so $\ddot{y}$ we can write $\ddot{x} = \ddot{y}$ and here $x + \frac{F}{k} = y$ so here it is $y = 0$, so this is nothing but showing differential equation showing similar to the undamped free vibration and so from here we see that the system is vibrating with the natural frequency $\omega_n = \sqrt{\frac{k}{m}}$, okay.

So in this case the mass will oscillate with the natural frequency that is $\omega_n = \sqrt{\frac{k}{m}}$ but it will oscillate not about $x = 0$, but it will oscillate about $y = 0$, so $y = 0$ means $x + \frac{F}{k} = 0$ or $x = -\frac{F}{k}$, so it will oscillate about this line rather than $x = 0$. Now we take the another case, the second case which will mass is moving from right to left, so here mass is moving from right to left.

So we can make the free body so this is mass and it is moving in this direction so this is $\dot{x}$, however x is this and $\ddot{x}$ we assume in this direction positive and so spring force will

be working here, because mass is still stretch to this direction to this side but moving towards the left side, so kx is the spring force this time the friction force F will work in this direction and here will be W and here the N .

Now again we write the equation of motion so here $m \ddot{x} = -kx + F$, so here $m \ddot{x} + kx - F = 0$. so this is the equation of motion of this system for this cycle, because in the half cycle the system, when the system is moving from left to right it will follow this equation, when it is moving from right to left it will follow this equation. Now again we do like $m \ddot{x} + kx - F = 0$ and $m \ddot{x} + kx + F = 0$.

So again we assume $y = x - F/k$ here and $\dot{y} = \dot{x}$ and $\ddot{y} = \ddot{x}$, okay. So $d^2y/dt^2 = d^2x/dt^2$. Now we will put these values here so we will have y co-ordinate $\ddot{y} + k/m$ into $y = 0$, from here we find that the natural frequency ω_n is $\sqrt{k/m}$, but the system is vibrating about $y = 0$ that is $x - F/k = 0$ or $x = +F/k$. now what is this is the system, now we can plot and see the response of the system although if we want to see the response of this differential equation.

So here the response of the differential equation this one will be $x = A_1 \sin \omega_n t + A_2 \cos \omega_n t + F/k$, while for equation this is for equation for equation one, for equation two $x = A_3 \sin \omega_n t + A_4 \cos \omega_n t + F/k$. so now if we want to see the response of the system, so we can plot it with for this we, let us assume some initial conditions, so $x = X_0$ at $t = 0$.

So for such initial conditions we will start, so it means that we have this mass say equilibrium position and we have moved pushed it this mass here, so we pushed it with X_0 and then we release it, so it will move start moving from right to left, so it is moving from right - right to left and then in the next cycle it will move left to - left to right, okay. So when we have pulled it with X_0 and leaving it is moving right to left.

So here this is X_0 so and this is x , so it is moving, so the system here at X_0 when $t = 0$ this is time and till here, because here is the one cycle completed and so it is $\tau - \tau - \tau = 2\pi$ upon

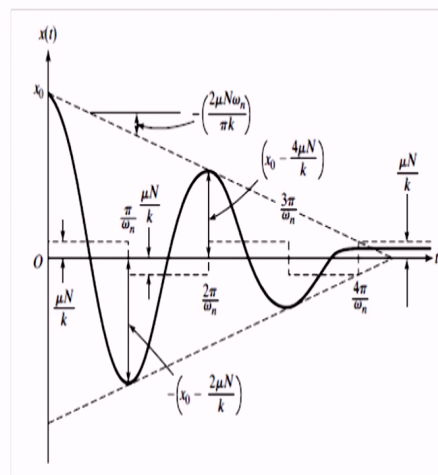
ω_n and this is π upon ω_n , so this is half cycle - half cycle this system is moving so this system is in this cycle doing SHM about because this in this part it is moving right to left. So here we have the equation right to left and it is moving about $x = F$ by k or $x - F$ by $k = 0$.

So this is $x - F$ by $k = 0$ line, so this is the line for this cycle, so SHM about $x = F$ by k , so in this half cycle it will vibrate about this axis. In the next half cycle when it is moving so here it is moving right to left - right to left, now in this cycle till here it will do SHM about $x = -F$ by k . Because here it is moving left to right - left to right, so left to right when it is moving it will oscillate about $x = -F$ by k or $x + F$ by $k = 0$.

So this is the line here, so this is the line $x + F$ by $k = 0$, so this is this line and so in this it will vibrate about this axis, so this response we can of this system having coulomb damping we can see also from this curve.

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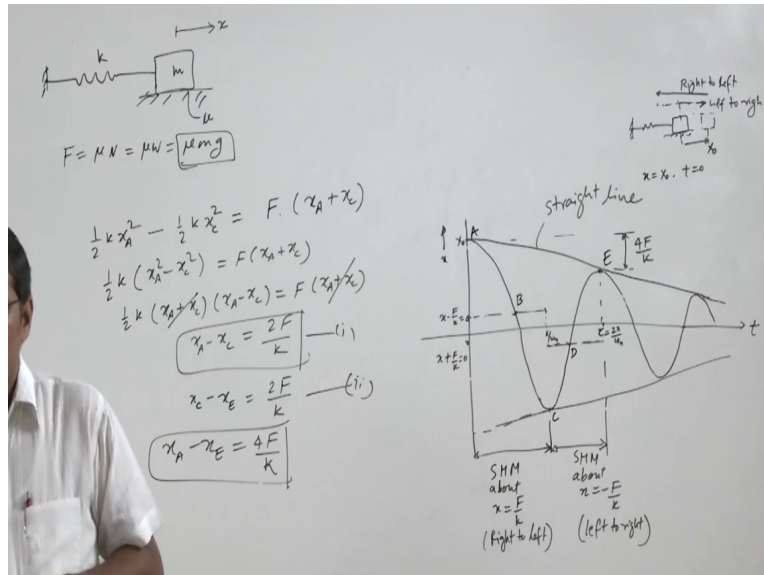
RESPONSE OF A SDOF SYSTEM HAVING COULOMB DAMPING



Now we want to know that what is the - what is the oscillations so the rate of decay, because we want to know that what is the kind of decay here because although the amplitude is going to decrease but how much so this - this information we want to know, so for this let us we say this is a point A, this is point B, and this is point C, point D and point E, so it starts its motion at A and passing through B, C, D and E, so this is the half cycle.

So we can - we are interested to know that what is the decay in one half decay of the amplitude in one cycle this is what we want to know and how we can know this, because we have the response we can see here the response, now we can see that when we have this system k and m and this system is vibrating, there is a constant friction force F .

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That is $F = \mu$ times n and $N = W$ so μ times m g , here all these things are constants, because μ is constant m is constant g , so our force is constant so our mass is moving against a constant force, so it is doing some work against this constant force F and in doing this force the systems energy is dissipated and systems energy in terms of potential energy in the spring or kinetic. So this potential energy of the system.

So we have at A we have a potential energy half $k x_A^2$ square, when the system is reaching to point C it has potential energy half $k x_C^2$ square, now the change in potential energy will be used to do the work against the friction force, so change in potential energy is this half $k x_A^2$ square - half $k x_C^2$ square and this is friction force is F , and the net moment of this system is so this is x_A and this is x_C , so $x_A + x_C$.

So total this is the moment of the system, now we can write half k , so $x_A^2 - x_C^2 = F x_A + x_C$, so here we can write half $k x_A + x_C$ and $x_A - x_C = F x_A + x_C$, so $x_A + x_C$ this will cancelled out and so we will get $x_A - x_C = \frac{2F}{k}$ by k , so we see that the amplitude there is the

decay in the amplitude from A to C and that decay $x_A - x_C = \text{constant } 2F \text{ by } k$. Because the F is constant and k is the stiffness of the spring that is constant so it is a constant quantity.

So this is what we calculated for half cycle because from A to C it is half cycle, similarly for this half cycle we will find that $x_C - x_E = 2F \text{ by } k$, so let us say this is an equation one and this is equation two, now we add this so we will have $x_A - x_E = 4F \text{ by } k$. So here we see that from A to E there is a reduction in the amplitude $4F \text{ by } k$ a constant quantity, so here if we have this and this is this so we have this quantity that is $4F \text{ by } k$, and so this is constant.

So this relation is linear, similarly here this relation will be coming as linear and same here, so we will have a linear relation, so this is a straight line. So we see that in case of coulomb damping the reduction in the amplitude the two consecutive amplitude is for $4F \text{ by } k$ that is a constant quantity and the envelope that is passing through the maxima of this amplitudes maxima is a straight line, if we compare with respect to the viscous under damped viscous damping case where the envelope was exponential curve.

So the reduction envelope was the amplitude was reducing in a - in an exponential way and the rate of decay we defined in terms of logarithmic decrement and logarithmic decrement we defined that was the ratio of the two consecutive amplitudes and taking the natural log of that, but here we can directly define the difference between the two amplitude $x_A - x_C$ that is equal to $4F \text{ by } k$.

And another important point is that in case of viscous damping the vibration was occurring about the $x = 0$, but here in the half cycle it is occurring about $x - F \text{ by } k = 0$, in the next half cycle it is occurring about $x + F \text{ by } k = 0$. Here the frequency of vibration is the same as the undamped vibration, undamped natural frequency. So here the system is vibrating with the ω_n that is the natural frequency of the system $\omega_n = \sqrt{k \text{ by } m}$.

In case of viscous damping the system was vibrating with not ω_n but a changed the frequency that was ω_d , the damped natural frequency we defined ω_d that was equal to $\omega_n \sqrt{1 - \zeta^2}$. So there are the differences between the two the physics of the

two damping's and we understand basically the two things we were interested the frequency of vibration and the rate of decay and both things.

We have discussed for viscous damping as well as the coulomb damping. So thank you for your attention for this lecture and see you in the next lecture. Thank you.