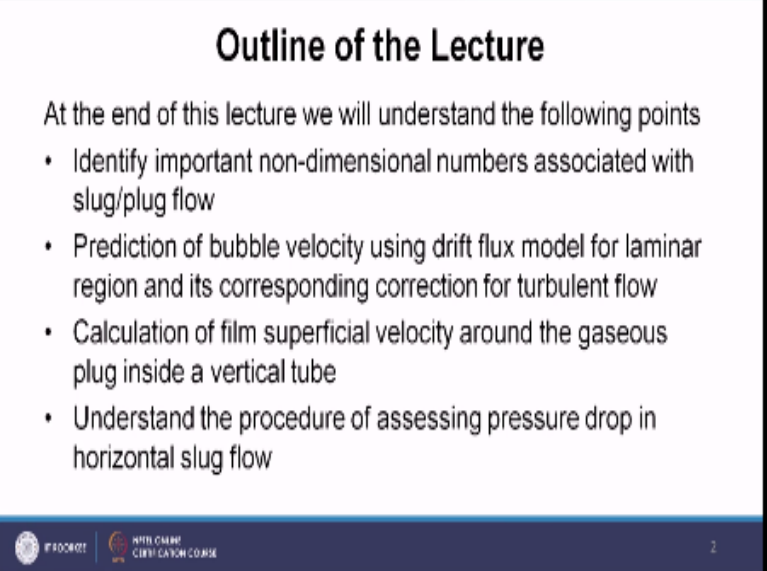


Two Phase Flow and Heat Transfer
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Lecture No: 07
Slug Flow Model

Hello welcome to the seventh lecture of Two Phase Flow and Heat Transfer. So, today's topic is slug flow model. So, at the end of this lecture you will be understanding the following points, you will identify the important non-dimensional numbers to specify the velocity of the slug bubble.

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Outline of the Lecture

At the end of this lecture we will understand the following points

- Identify important non-dimensional numbers associated with slug/plug flow
- Prediction of bubble velocity using drift flux model for laminar region and its corresponding correction for turbulent flow
- Calculation of film superficial velocity around the gaseous plug inside a vertical tube
- Understand the procedure of assessing pressure drop in horizontal slug flow

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We will be predicting the velocity of the slug bubble using drift flux model and laminar regime, we will be finding out how the velocity is changing for turbulent flow. We will be calculating the film superficial velocity around the gaseous plug inside a vertical tube. And we will be understanding the assessment of pressure drop inside the horizontal slug flow.

Now to give you better understanding what is stuck slug flow, I have shown you here a schematic. So you can find out that in this schematic we are having a Taylor bubble as gaseous plug over here.

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Slug Flow

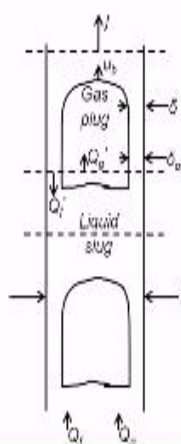
Correlation for single bubble rise velocity:



$$K_1 = 0.345 \left(1 - \exp \left(\frac{-0.01 N_f}{0.345} \right) \right) \left\{ 1 - \exp \left(\frac{3.37 - Eo}{m} \right) \right\}$$

Where, $K_1 = \left[\frac{\rho_f u_{\infty}^2}{D g (\rho_f - \rho_g)} \right]^{1/2}$ $N_f = \frac{[D^3 g (\rho_f - \rho_g) \rho_f]^{1/2}}{\mu_f}$

$$Eo = \frac{D^2 g (\rho_f - \rho_g)}{\sigma}$$

and, $m = 10 \forall N_f > 250$
 $m = 69 N_f^{-0.35} \forall 18 < N_f < 250$
 $m = 25 \text{ for } N_f < 18$





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And in between 2 gaseous plugs, we will be having liquid slug. So in real slug bubble regime you will be finding out lots of satellite bubbles are here proceeding this gaseous plug but for simplicity and purpose of analysis we have eliminated those satellite bubbles. We are only dealing with the gaseous plug over here and followed/ A liquid slug.

So it will be some sort repetitive pattern of gaseous slug and gaseous plug and liquid slug. So what we have considered over here. We have considered that this is a unit cell where we will be finding out a gaseous plug is over here. At the top of that we are having a portion of the liquid slug from the previous gaseous plug and at the bottom of that we are having another portion of the liquid slug for this gaseous plug. So this cell will be actually continuing in the bottom side as well as top side right.

So our point of concern is over here, this unit cell okay now let us consider this unit cell if you see minutely this unit cell, we are having gaseous plug but around that we are having liquid film also okay. So here we are having liquid film, here also we are having liquid film. Now this gaseous plug will be moving up with the velocity u_b and to accommodate that movement of the gaseous plug, the liquid needs to come down okay.

So here we will be finding out the liquid velocity volumetric flow rate of the liquid. We have considered over here q_f right. And to recognize the film thickness over here, we have

considered the film thickness is δ and in the vertical portion of the slug bubble we have considered that the film thickness is becoming uniform and that thickness is actually δ infinite right.

And obviously we know that pipe diameter, we always represent with D and the overall flow rates for the liquid and gas that is q_f and q_g . Remember though the liquid is actually coming down over here with a volumetric flow rate of the q_f , actually the liquid overall is movement moving up because this liquid slug is being pushed by the bottom Taylor, bottom gaseous plug and overall movement of the liquid is also in the upward direction along with the gas.

So you can find out q_f is also in the upward direction along with Q_g right. So at first let us try to find out that how this rise velocity of this gaseous plug can be calculated. So for that there are lots of correlations available, experimentally observed correlations available. The best one is this

$$K_1 = 0.345(1 - e^{-0.01 N_f / 0.345}) \{1 - e^{-(3.37 - Eotvos \text{ numbers by } m)}\}$$

right.

Let us find out what are these terms first. K_1 , so K_1 is actually $\rho_f u_{\infty}^2$. So where ρ_f is actually the liquid density, u_{∞} is the terminal velocity of the bubble of the gaseous plug. And we are having D pipe diameter $\cdot g (\rho_f - \rho_g)^{1/2}$ okay. So we can put this whole expression over here which will be getting as a function of u_{∞} .

Then in the expression if you go further, we are having another non-dimensional number which is N_f , this is actually inverse non-dimensional viscosity. So you can find out this N_f is actually represent as $[D^3 g (\rho_f - \rho_g) \rho_f]^{1/2} / \mu_f$. So as μ_f is in denominator. So we call this one as inverse viscosity non-dimensional viscosity. On the other hand if you go further, we are having Eotvos numbers.

So the Eotvos numbers all of we know once again to represent that the Eotvos numbers will be $D^2 g (\rho_f - \rho_g) / \sigma$. So here we get the ratio between the buoyancy force and surface tension force. Here we get the ratio between your buoyancy force and viscous force and here inertia and your buoyancy force right. Now only empirical constant over here left is m .

So m will be 10, if your non-dimensional inverse viscosity is higher than 250. m can be represent as a 69 Nf to the power -.35. If Nf varies in between 18 to 250 and if $Nf < 18$ that means for very viscous flow you will be finding out $m = 25$ right. So if I put all these non -dimensional numbers over here, so all other parameters will be known. Only unknown will be u infinity. So one can find out the velocity of the gaseous plug right.

Next once we know the gaseous plug velocity then we have to find out what is the superficial velocity and how this can be found out for some different situations like laminar and turbulent zones. So lets us try to see that how in a slug plug flow bubble velocity varies.

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Bubble velocity in slug-plug flow:

Average liquid slug velocity: $j = \frac{Q}{A} = \frac{Q_g + Q_f}{A}$

For free rise bubble: $u_b = u_g$ & $u_{jg} = u_{\infty}$

Therefore, $u_b = u_g = j + u_{jg} = j + u_{\infty}$

$u_b = j + u_{\infty}$ Valid for laminar bubble movement only

Using this void fraction can be written as:

$$\alpha = \frac{j_g}{u_g} = \frac{j_g}{u_b}$$

So first we have shown you over here average liquid slug velocity will be nothing but j okay which= all of we know that $j = Q/A$ and Q can be written as $Q_g + Q_f$ okay. So Q_g and Q_f already I have shown you in the figure here, this Q_f and this Q_g okay. So you will be finding out that j is actually $Q_g + Q_f / A$. Now for a free rising bubble, so let us say the bubble is moving freely. So we know that u_b will be actually symbolizing the gas velocity.

So bubble velocity eventually will become the gas velocity. So we can right down $u_b = u_g$ right. And if we see that what is the velocity in comparison to the average. Overall velocity that means the velocity is sleep out in comparison to the average overall velocity that will be u_{jg} that is

actually equivalent u_{∞} which we have calculated in the previous slide okay. Next over here you see in this equation if we put u_b which is equivalent to your u_g , so here we will be finding out $j + u_g$ right.

Now u_g just now we have shown that is actually u_{∞} . So we will be finding out $u_b = j + u_{\infty}$ okay. So this new expression, we get $u_b = j + u_{\infty}$ this is valid for a laminar bubble movement in a slug plug flow right. So this u_{∞} is the unconstrained bubble velocity. So if you do not consider the walls inside the slug plug flow then this u_{∞} can be found out from the previous expression over here.

And a bubble velocity in slug plug flow can be found out by adding j along with the u_{∞} okay. Now already we know that if we have to find out that what the average void fraction is, average void fraction α can be written as j_g / u_g okay. Now we know that u_g will be u_b . So actually, the α in this slug plug flow case we can write this one as j_g / u_b okay.

So ultimately, we get for laminar flow $u_b = j + u_{\infty}$ and α average void fraction = j_g / u_b . Remember in this type of slug plug flow case the void fraction at different cross section will be changing with respect to time of the flow progresses. Because here, whenever we are seeing through this liquid slug obviously void fraction will be 0. And here if you see over here you will be having a finite amount of void fraction.

So we are always interested to find out the average fraction in the cell okay. Now here we have talked about the laminar flow. Let us see what happens if we are having fully developed turbulent flow okay. So if we are having fully developed turbulent flow, we will be finding that the bubble velocity is changing little bit. So what we can do empirically we can add 1 constants over here, 2 constants over here.

C_1 and C_2 along with the expressions whatever we have found out for u_b in the laminar regime. So already we have shown in the laminar regime u_b is $j + u_{\infty}$ for turbulent regime.
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For fully developed turbulent flow: $u_b = C_1 j + C_2 u_{\infty}$

Where, $C_1 = 1.2$ & $C_2 = 1 \forall Re_j > 8000$

Using this mean void fraction can be written as:

$$\alpha = \frac{Q_g}{C_1(Q_g + Q_f) + C_2 A u_{\infty}}$$

Pressure drop along the vertical pipe having intermittent slug/plug flow:

$$-\frac{dP}{dz} = g[\rho_f(1 - \alpha) + \rho_g \alpha] + (1 - \alpha)f_f \frac{2\rho_f j^2}{D}$$

We are writing down $u_b = C_1 j + C_2 u_{\infty}$ right. Now various researchers they have proposed the values of C_1 and C_2 . Wallis has proposed that you can take for slug plug flow $C_1 = 1.2$ and $C_2 = 1$ if your overall Reynolds number is actually greater than 8000 okay. If your Reynolds number is in between 2000 to 8000, this expression will not be valid. If it is more than 8000 then only you can apply $C_2 = 1.2$ and $C_1 = 1.2$ and $C_2 = 1$ right.

Now using this mean void fraction what we can write down. Alpha what I have earlier showed you over here which is nothing but j_g / u_b . So same thing we can write down over here. Alpha = Q_g now this j_g can be converted to Q_g / A okay. Q_g / A . So that can be written as Q_g and A will be absorbing over here. So in place of your u_b , we are writing down $C_1 j + C_2 u_{\infty}$.

So $C_1 j$, now j can be once again written as $j_g + j_f$ and j_g can be written as Q_g / A and Q_g / A and j_f can be written as Q_f / A . So you can find out here we are having Q_g / A . Here we are having Q_g / A and here we are having Q_f / A . So A can be cancelled out and A can be absorbed in the last term which was $C_2 u_{\infty}$. So $C_2 u_{\infty} \cdot a$. So this becomes the expression for alpha for turbulent flow right.

Now as we know that not only the value of alpha and the velocities of bubble will be important. Important will be to know what is the pressure drop. So here I have given you the expression for pressure drop in slug plug flow also. You see $-dp/dz$ can be written as the buoyancy pressure

drop. So this is $g^* \rho_f (1-\alpha)$. Now this α will be actually used from the previous calculation.

The average α whatever we have found out so $\rho_f (1-\alpha) + \rho_g \alpha$. This is from the buoyancy and then $(1-\alpha) f_f$. So we are using fluid part only assumption okay. So $f_f^2 \rho_f j^2 / D$ okay. As you are dealing with the slug flow finding out the liquid superficial velocity and gas superficial velocity will be difficult. So what we do, we find out the overall superficial velocity j and we express the frictional pressure drop in terms of fluid part okay.

So f_f we are using over here this frictional portion we have already discussed in a previous lecture okay. Now after finding out the friction factor it is also very important to know that what will be the liquid flow rate okay. What it is coming out in downward side okay. So to assess that first let us get what is the gaseous flow rate. So the gaseous flow rate will be flow of the gaseous plug.

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Gas flow rate: $Q'_g = \frac{\pi}{4} (D - 2\delta_\alpha)^2 u_g$

Liquid flow rate, Q'_l (downward)

$$Q = Q'_g - Q'_l \quad \frac{Q'_l}{A} = j'_l = \frac{Q'_l}{A} - \frac{Q}{A}$$

$$j'_l = \frac{\frac{\pi}{4} (D - 2\delta_\alpha)^2 u_g}{\frac{\pi}{4} D^2} - j$$

$$j'_l = \left(1 - 2\frac{\delta_\alpha}{D}\right)^2 u_g - j$$

$$\frac{j'_l}{u_{gs}} = \left(1.2 \frac{j}{u_{gs}} + 1\right) \left(1 - 2\frac{\delta_\alpha}{D}\right)^2 - \frac{j}{u_{gs}}$$

So this is Q'_g and liquid flow rate would be Q'_l . So Q'_g can be written as the area occupied by the liquid sorry, gaseous plug multiplied by the velocity of the gas. So here what we have considered the pipe diameter was D and we have considered that this thickness of the film uniform thickness of the film around the gaseous plug is D infinity. So overall we will be finding out the diameter of this gaseous plug is $D - 2\delta_\alpha$.

So we will be finding out the area of this gaseous plug, if you consider a perfect cylinder as this gaseous plug. So it will be $\pi/4 (D - 2\delta)^2$ okay. So this is the area of the gaseous plug considering it as a cylinder multiplied by its velocity u_g . So will be finding out this is the gas flow rate in this cell right. Now let us right, to find out what is a liquid flow rate. So what will be doing? As we know the overall flow rate we know from summation of $Q_f + Q_g$.

So I can write down $Q = Q_g - Q_f$. Now this Q_g is will be never equivalent to this Q_g okay because they are different because in this case we are having no gaseous phase right. In the liquid slug we are having no gaseous phase. So here we can find out this Q as $Q_f + Q_g$ and that can be written as $Q_g - Q_f$. Why this Q_f - because in this cross section if you see this cross section, this dotted line here Q_f is in the now negative direction right.

So we can write down $Q_f/A = Q_g/A - Q/A$. So from this expression I can write down $Q_f = Q - Q_g$ and if you divide it by A , we get this expression. The left hand side can be written as j_f okay because Q_f/A is nothing but j_f okay. So we get $j_f = Q_g/A$. Now what is Q_g once again we have written over here or assessed found out the value of Q_g gaseous flow rate. So this will be written over here and A which is nothing but the area of the pipe π .

So $\pi/4 * D^2$. So you can write down over here in terms of a $\pi/4 D^2$ and obviously we know that Q/A is actually j right. So if we simply, this first term in the right hand side we will be getting something around $(1 - 2\delta/D)^2 u_b$ now this u_g we are converting to u_b because we know here this gaseous velocity a gaseous plug velocity will be same as the bubble velocity okay.

So we get $j_f = (1 - 2\delta/D)^2 u_b - j$ okay. So already we have found out what is u_b in the previous slide. For turbulent cases so same formula will be using over here $(1.2 * j/u_\infty + 1)$. So that was actually $(1.2 * j + u_\infty)$. If you will take u_∞ common and then divide the whole expression by u_∞ . So this left side, so left hand side will become j_f/u_∞ , right hand side will become $(1.2*j/u_\infty + 1)$ and then this expression multiplier will be this is coming from the area ratios.

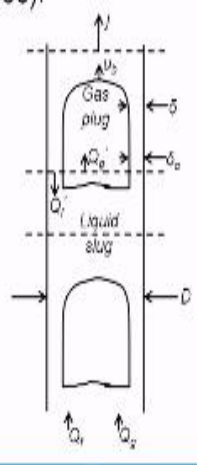
So this will be staying over here and the last term will remain as $-j/u$ infinity. So in this way we can find out the liquid flow rate. Liquid superficial velocity in the in the cross section where gaseous plug is present right. Next if we consider the falling film theory then this j_f / u infinity can be predicted from falling film theory. So that has been discussed by Miosis and Griffith. I will not going into detail of this one.

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From falling film theory (Miosis and Griffith, 1960):

$$Re_f = \frac{Q'_f}{Au_\alpha} (0.345 N_f)$$

$$\frac{j'_f}{u_\alpha} = 3.85 N_f \left(\frac{\delta_\alpha}{D} \right)^3 \quad \forall Re_f < 3500$$

$$\frac{j'_f}{u_\alpha} = 183 \left(\frac{\delta_\alpha}{D} \right)^{3/2} \quad \forall Re_f > 3500$$


But I will be showing that how empirically can be found out using falling film theory how the liquid velocity superficial velocity can be found out. So here you see this velocity will be nothing but j_f / u infinity will be $3.85 N_f$. N_f already we have discussed that is non-dimensional inverse viscosity into delta infinity, uniform film thickness / D whole cube. This expression valid is valid for Reynolds number, film Reynolds number less than 3500.

The definition of film Reynolds number, I have given over here. So film Reynolds number is nothing but $Q_f / A * u$ infinity $(0.345 N_f)$. And if this film Reynolds number is larger than this 3500 then you can use this expression j_f / u infinity = $183 (d \text{ infinity} / D)$ to the power $3/2$ right. Next let us shift to horizontal slug flow from the vertical one. So here I have shown once again the schematic of horizontal slug flow.

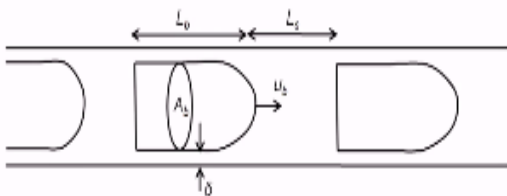
So you can find out that in case of horizontal slug flow this is once again like a Taylor bubble. So this is actually a gaseous plug in between we are having in between 2 gaseous plug, we are having a liquid slug.

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Horizontal slug flow

From continuity:



$$A_b u_b = A j$$



$$u_b = D^2 D^{-2} \left(1 - 2 \frac{\delta}{D}\right)^{-2} j$$

$$\frac{\pi}{4} \left(\frac{D}{2} - \delta\right)^2 u_b = \frac{\pi}{4} D^2 j$$

Bubble velocity: $u_b = \left(1 + \frac{4\delta}{D}\right) j$ When, $\frac{\delta}{D} \ll 1$



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Once again the same situation this is having bubble velocity. The film thickness we have represented as delta okay. And here I have shown the length of the bubble as L_b and the length of liquid slug as L_s . Now if you start from the very basic continuity equation then I can write down A_b to into $u_b = A * j$. Why because you see in this horizontal slug flow actually the movement of the bubble will be causing the overall flow because otherwise it is not assisted by buoyancy.

The flow is not assisted by buoyancy. So you will be finding out when the bubble is moving that will be only causing the flow. So it you see from the continuity side if you find out what is the volumetric flow rate for this bubble so which is nothing but $A_b * u_b$ that will be responsible for the overall flow $A * j$ right. Now from here if we try to find out that what is the value of u_b , so will be $A_b / A * j$ right.

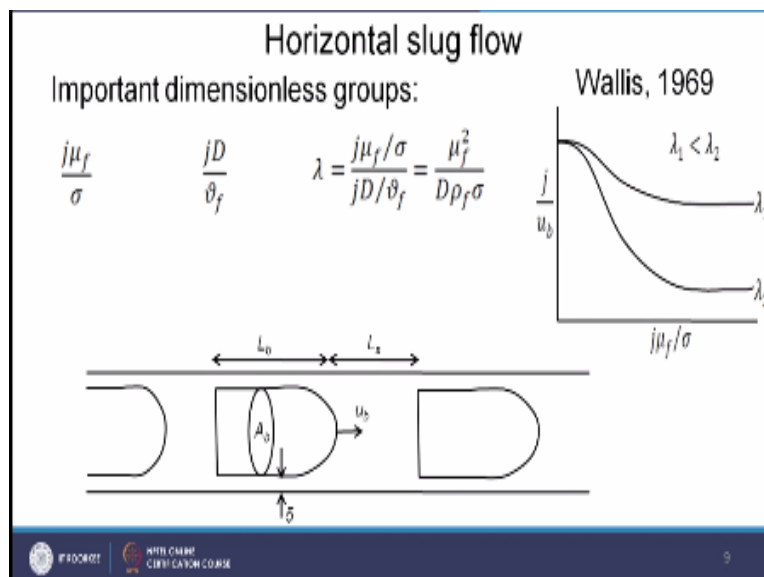
Now A once again we know that will be $\pi / 4 D^2$, D is the tube diameter and A_b , I can write down as $\pi / 4 (1 - 2 \delta / D)^2$ which I have already shown you in the vertical case okay into j will be remaining from here. So here we are

having j so the same j will be remaining over here okay. So already I have shown you the expression over here. So you can find out so this from this we are getting this expressions.

So u_b finally after cancelling this D square and D to the power -2 comes out as $(1 + 4 \delta/D) j$ okay. Now this we can write down whenever we considered $\delta/d < 2$. Why because this is a polynomial. So if we find out this δ/d is very small than 1 then only I can write down this is actually $(1 - 2)(-2 * \delta/d)$. So this is actually $1 + (4 \delta/d)$ okay. So this expression bubble velocity is valid only for very small film thickness compared to the pipe diameter.

If it is not small then you have to go with the overall polynomial and we have to evaluate the value okay. Now let us see just like our vertical case what are the important non-dimensional numbers in case of horizontal slug flow.

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So important non-dimensional numbers first one is $j\mu_f/\sigma$ okay. Second one we can take as jD/μ_f . So this is having the viscous effect this is having the surface tension effect compare to viscous effect. And another one we can derive as λ which will be the ratio between these 2 okay. So here you will be finding out that in this we are having 3 parameters λ , $j\mu_f/\sigma$ and jD/μ_f .

If you try to find out that how this parameter are actually interlinked then we can plot something like this. So in the abscissa we are having the first non-dimensional number $j\mu_f/\sigma$. And in the ordinate we are having the second one which is actually j/u_b . So here this is actually non-dimensional bubble velocity, non-dimensionalised by the overall superficial velocity and here will be finding out several curves having different λ value.

So λ is actually the ratio between these 2 non-dimensional numbers and its values $\mu_f^2 / D \rho_f \sigma$ okay. So here we can find out that the curve will be varying like this. For a low viscous fluid will be finding out that bubble velocity is high okay. And we will be finding out that for different λ . As λ increases we will be finding out that the curve is falling. Actually, in the downward side right this has been proposed by Wallis.

Next let us try to find out what is the bubble velocity in this case and the overall void fraction. So as we know already that u_b is nothing but $C_1 j + C_2 u_\infty$ for turbulence situations.

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We know that, $u_b = C_1 j + C_2 u_\infty$

For horizontal flow, $u_\infty \approx 0$ Therefore, $u_b = C_1 j$

For large λ , C_1 can be given as:

$$C_1 = 1 + 1.27 \left\{ 1 - \exp \left(-3.8 \left(\frac{u_f j}{\alpha} \right)^{0.8} \right) \right\}$$

Void fraction: $\alpha = \frac{j_g}{u_g} = \frac{j_g}{u_b} = \frac{j_g}{C_1 j} = \frac{1}{C_1} \frac{j_g}{j_g + j_f}$

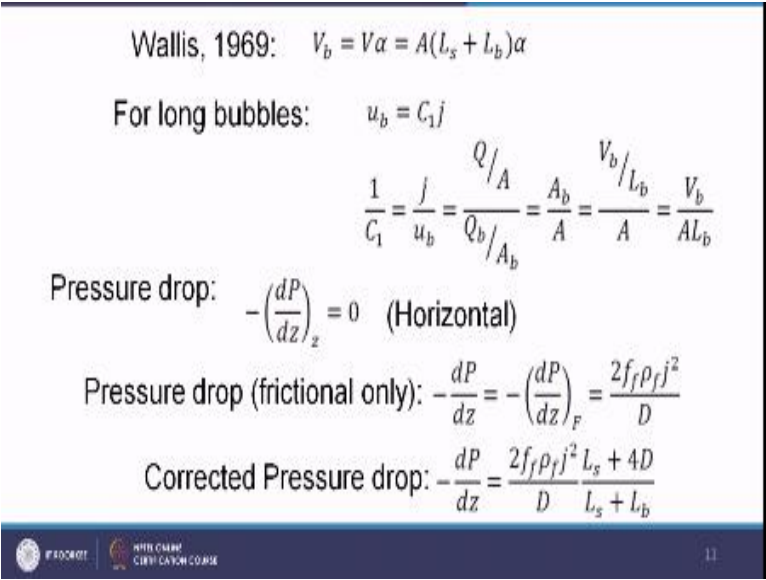
$$\alpha = \frac{1}{C_1} \frac{Q_g}{Q_g + Q_f}$$

So here what we can do, u_∞ we can make 0 because it is in horizontal pipeline. So there will be no velocity, unconstrained velocity of the bubble. So you can write down $u_\infty = 0$. So we get $u_b = C_1 j$, over here $C_1 j$ right. Now once again various correlations are there. 1 very important correlation is for finding out the C_1 for horizontal slug flow C_1 will be $1 + 1.27 \{ 1 - e \text{ to the power } (- 3.8 (u_f j / \sigma) \text{ to the power } 0.8) \}$ right.

Okay now if we want to calculate the void fraction also, you see void fraction alpha will be j_g / u_g . So j_g / u_g , we know that u_g will be actually your u_b because gaseous phase is only limited in bubble. So we will be finding out u_g / u_b okay. And already I have shown $u_b = C_1 j$. So what we can write down $j_g / C_1 * j$. So u_b has been replaced by $C_1 j$. So you get $1 / C_1$ and then j_g / j . j can be written as $j_g + j_f$ okay. In the first lecture I have shown you this part right.

And once again j_s can be converted into corresponding Q_s . So you can write down that $\alpha = 1 / C_1 * Q_g / (Q_g + Q_f)$. So over here $Q_g / (Q_g + Q_f)$ right. Now let us try to assess that what will be the pressure drop across this slug plug flow. So here we are writing first or we assessing first the volume of the bubble.

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Wallis, 1969: $V_b = V\alpha = A(L_s + L_b)\alpha$

For long bubbles: $u_b = C_1 j$

$$\frac{1}{C_1} = \frac{j}{u_b} = \frac{Q/A}{Q_b/A_b} = \frac{A_b}{A} = \frac{V_b/L_b}{A} = \frac{V_b}{AL_b}$$

Pressure drop: $-\left(\frac{dP}{dz}\right)_z = 0$ (Horizontal)

Pressure drop (frictional only): $-\frac{dP}{dz} = -\left(\frac{dP}{dz}\right)_F = \frac{2f_f \rho_f j^2}{D}$

Corrected Pressure drop: $-\frac{dP}{dz} = \frac{2f_f \rho_f j^2}{D} \frac{L_s + 4D}{L_s + L_b}$

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So we can find out that volume of the bubble will be $V * \alpha$ and this volume of bubble, this volume can be written as $A * (L_s + L_b)$. Now this $A * (L_s + L_b)$ is the volume for the (()) 25:26 width of the cell. Whatever we have shown over here that we are having a gaseous plug and liquid slug. So if you consider a unit, so this unit volume is this $1/A$ is the pipe diameter and L_s is the slug length and L_b is the bubble length.

So we can find out that V_b turns out to be $A * (L_s + L_b) * \alpha$. Now already we have shown that for long bubble. We will be finding out $u_b = C_1 j$ in the previous slide we have shown $u_b = C_1 j$.

So what we can do we can write down C_{11}/C_1 is actually equals to j/u_b okay. So now let us try to find out what is this j and u in terms of the volumetric flow rates. So j can be written as Q/A and u_b can be written as Q_b/A_b okay. And as we know that Q and Q_b both will be same because the slug bubble is actually causing the flow inside the pipeline. So we can find out that $1/C_1$ comes out to the ratio of A_b/A okay.

Now what is A_b , A_b once again can be written as V_b/L_b because of bubble volume will be actually your area of the bubble and length of the bubble okay. So ultimately, I get $1/C_1 = V_b/A L_b$ right. Now related to pressure drop already we know as it is a horizontal pipe. So for horizontal cases your gravitational pressure drop will be 0. So $-dp/dz$ for z gravitational head is actual buoyancy head is actually 0. But there will be frictional head.

So frictional head I can write down $-(dp/dz) f$ will be $2 f f$. Here I am considering fluid part okay. So $2 f f \rho f j^2 / D$ okay. Now if we are considering that inside dynamics of the Taylor bubble that means the gaseous dynamics inside the Taylor bubble then we will be finding out 1 multiplier is necessary. So this multiplier has been proposed by Wallis over here in 1969. So this multiplier is $L_s + 4D / L_s + L_b$.

So dependent on the slug length and the bubble length and the tube diameter okay. So next let us try to find out that this pressure drop how that can be simplify and found out in the form of volumetric qualities.

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$$\text{Corrected Pressure drop: } -\frac{dP}{dz} = \frac{2f_f \rho_f j^2 L_s + 4D}{D L_s + L_b}$$

$$-\frac{dP}{dz} = \frac{2f_f \rho_f j^2}{D} \left[1 - \frac{L_b}{L_s + L_b} + \frac{4D}{L_s + L_b} \right]$$

$$-\frac{dP}{dz} = \frac{2f_f \rho_f j^2}{D} \left[1 - \frac{V_b C_1 A \alpha}{A V_b} + \frac{4DA}{V} \right]$$

$$-\frac{dP}{dz} = \frac{2f_f \rho_f j^2}{D} \left[1 - C_1 \alpha + \frac{4DA}{V} \right]$$

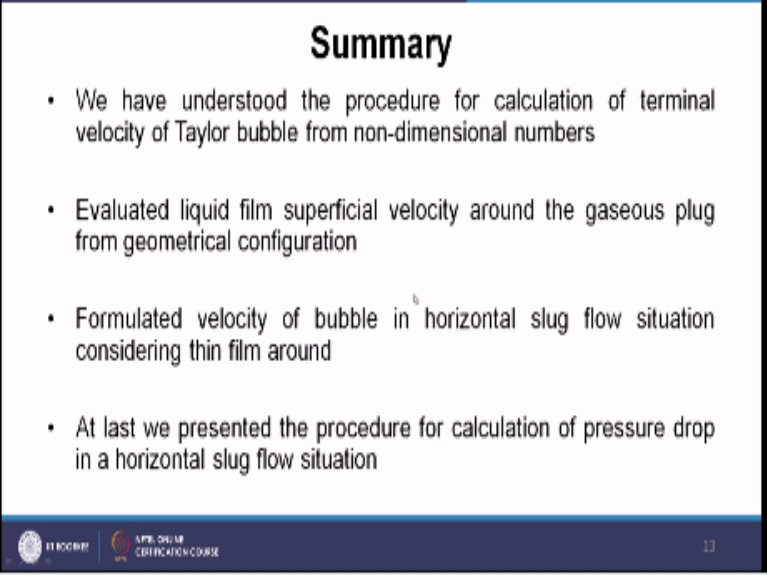
So already we have shown that $-dp/dz$ will be $2f_f \rho_f j^2 / D L_s + 4D / L_s + L_b$. Now let me simplify this part. So this, I can write down easily $1 - L_b / L_s + L_b + 4D / L_s + L_b$. Basically this first 2 terms gives me $L_s / L_s + L_b$ okay. Then this L_b and $L_s + L_b$, I can replace so $L_s + L_b$ quickly I can write as $A * V_b$ because already I have shown $V_b = A * L_s + L_b * \alpha$ right. So $L_s + L_b$ will be $V_b / A \alpha$.

So what I can do here, I have written $A b / \alpha$ right and for L_b , I have written $b b C_1 A$ okay $B b C_1 ; A$ why because this L_b is actually the length of the bubble. So length of the bubble will be actually merged with the volume of the bubble over here. So volume of the bubble into the area of the bubble will be coming out as the length of the bubble okay. So here we can find out that both this L_b and $L_s + L_b$ has been replaced by this factor and here once again this $L_s + L_b$ has been done by same $(())$ 29:15.

So this will be V / A once again V / A . So if you simplified further ultimately, I will be getting $-dp/dz = w f f \rho_f j^2 / D (1 - C_1) \alpha + 4 D A / V$. Now here in this expression you see C_1 is actually empirical parameter. So using this can be found out from Wallis correlation whatever I have explained you earlier okay. And the rest things we can find out from the slug plug flow and can be evaluated okay.

To summarize in this lecture, we have understood the procedure of calculation for terminal velocity of a Taylor bubble from non-dimensional numbers.

(Refer Slide Time: 29:58)



The slide is titled "Summary" in a bold, black font. It contains four bullet points, each preceded by a small black square. The text is centered on the slide. At the bottom of the slide, there is a dark blue footer bar containing logos on the left and the number "13" on the right.

Summary

- We have understood the procedure for calculation of terminal velocity of Taylor bubble from non-dimensional numbers
- Evaluated liquid film superficial velocity around the gaseous plug from geometrical configuration
- Formulated velocity of bubble in horizontal slug flow situation considering thin film around
- At last we presented the procedure for calculation of pressure drop in a horizontal slug flow situation

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We have evaluated the liquid film superficial velocity around the gaseous plug from geometrical configurations. We are formulated velocity of bubble in horizontal slug flow situation okay. And in that we have considered that we are having thin film around the bubble and at the end we have presented the procedure for calculation of pressure drop inside a horizontal slug situation okay.

Next let us test how you have gone through this lecture. So we are having once again 3 questions.

(Refer Slide Time: 30:33)

Test your understanding ?

1. In slug flow around the gaseous plug liquid velocity is
 - a. Upward
 - b. Oscillating
 - c. Downward
 - d. Horizontal
2. Which force is considered for calculation of rise velocity of a single bubble
 - a. Buoyancy
 - b. Inertia
 - c. Viscous
 - d. Surface tension
3. For horizontal slug flow which relationship is not correct
 - a. $-\left(\frac{dP}{dz}\right)_g = 0$
 - b. $-\left(\frac{dP}{dz}\right)_f = \frac{2f_f \rho_l j^2}{D}$
 - c. $-\left(\frac{dP}{dz}\right)_g = \frac{2f_f \rho_l j^2}{D}$
 - d. $u_b = C_1 j$


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First, in slug flow around the gaseous plug liquid velocity is 4 options we are having upward, oscillating, downward and horizontal. So mostly you have understood what is the answer correct. Answer is downward because liquid will be having downward velocity around the slug. Okay second question which force is considered for calculation of rise velocity of a single bubble okay.

We are having 4 options buoyancy, inertia, viscous and surface tension. Which force is important for calculation of terminal velocity rise velocity of a single bubble answer all? So all the forces are equally important for calculation of right rise velocity okay. Last question for horizontal slug flow which relationship is not correct.

So we are having 4 equations, gravitational pressure drop= 0, frictional pressure drop= $2f_f \rho_l j^2 / D$, once again gravitational pressure drop= $2f_f \rho_l j^2 / D$ and $u_b = C_1 j$. Probably you have understood which one is the correct one, obviously part c is not the correct answer. Hope you have like this lecture. Thank you.