

Gas Dynamics and Propulsion
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Lecture - 35
Calculations for Thrust and Fuel Consumption

So in the last class, we derived the final or we calculated the value for the total thrust under static condition.

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The image shows handwritten calculations on a chalkboard. The first line is $T = 166 \text{ kN} + 63 \text{ kN} = 229 \text{ kN}$. Below this, '73%' is written under '166 kN' and '(233.6 kN)' is written to the right. Below that, 'medium bypass' is written. The next line is the fuel mass flow rate formula: $\dot{m}_{\text{fuel}} = \frac{\dot{m}_H (C_{p9} T_{04} - C_p T_{03})}{H_{\text{fuel}} - C_{p9} T_{04}}$. Below this, the value $= 2.64 \text{ kg/s}$ is written. The final line is the Thrust Specific Fuel Consumption (TSFC) formula: $TSFC = \frac{\dot{m}_f}{T} = 0.0414 \text{ kg/hr.N}$.

So, we said that the total thrust was composed of 166 kilo Newton of thrust from the fan and 63 kilo Newton of thrust from the core engine which worked out to 229 kilo Newton of thrust and we compared with this value with the value quoted by the manufacturer which was about 233.6 kilo Newton. So the comparison appears to be reasonably good, so we come to the conclusion that the calculation process is probably okay.

One point that you should notice is that the fan contributes to about 73% of the overall thrust which tells you that this is medium to high bypass ratio turbo fan engine. So medium to high bypass will have numbers in the range from 75 to 80 so this tells you that it is a medium bypass ratio engine. The next thing that we wish to calculate is the mass flow rate of fuel so let us go ahead and calculate the \dot{m}_{fuel} based on an energy balance for the combustor.

So we did energy balance for the combustor and we showed that the mass flow rate of fuel is given by this expression $\dot{m} \frac{H}{C_p(T_4 - T_3)}$ divided by the calorific value of the fuel $C_p(T_4)$, and if you go ahead and substitute the numbers that we have calculated earlier then we get this value to be 2.64 kg per second as we suspected earlier this mass flow rate of 2.64 kg per second is much less compared with the total amount of mass flow rate that goes through the engine.

Remember the air mass flow rate is around 600 kg per second or so. So this is indeed much less than that. So we were correct in neglecting \dot{m} fuel when we add the mass flow rate of air to this so it is indeed a small number and if I calculate the thrust specific fuel consumption TSFC, TSFC is nothing but the total thrust divided by \dot{m} fuel. In this particular problem, the combustion efficiency is not given.

So the ideal value that we calculate using this formula for mass flow rate of fuel also happens to be the actual value since combustion efficiency is not given. So T divided by \dot{m} f comes out to be 0.0414 in units of kg per hour Newton is what we calculated for this and the manufacturer quotes value of.

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The image shows a chalkboard with the following handwritten equations:

$$\text{book value} = 0.03781 \frac{\text{kg}}{\text{hr N}}$$

$$\dot{m}_c = \frac{B}{B+1} \dot{m} = 543.823 \text{ kg/s}$$

$$= \rho_{ec} A_{ec} V_{ec}$$

$$= \frac{\rho_{ec}}{RT_{ec}} A_{ec} V_{ec}$$

$$A_{ec} = 1.5 \text{ m}^2$$

So, book value for TSFC 0.03781 kg per hour Newton. It seems to be reasonably okay for the kind of assumptions that we have made. This seems to be reasonably alright. It is the other way

around. So this is \dot{m} divided by T thank you. So the comparison seems to be alright. We are also asked to calculate in addition to thrust and TSFC we are also asked to calculate the area of the cold nozzle and the hot nozzle.

So let us go ahead and do that so the mass flow rate through the cold nozzle \dot{m}_c is nothing but $\frac{1}{b+1}$ * the total mass flow rate and if you substitute the numbers this comes out to be about 543.823 kg per second and we can also use the following expression \dot{m}_c is also going to be $= \rho_e$ at the exit for the cold nozzle * A_{ec} * V_{ec} right. \dot{m} is also $= \rho_e v$ and in this expression I can write this ρ_e as p/RT * A_{ec} * V_{ec} .

So I know all the quantities in this expression except A_{ec} and I can calculate the cold nozzle area to be if you substitute the numbers you get the cold nozzle area to be 1.5-meter square. Remember the cold nozzle is an annular nozzle. So this is the area of the annular nozzle and in the same manner I can calculate the mass flow rate through the hot nozzle.

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The image shows a chalkboard with the following handwritten equations:

$$\dot{m}_H = \frac{1}{b+1} \dot{m} = 126.177 \text{ kg/s}$$

$$= \rho_{eH} A_{eH} V_{eH}$$

$$= \frac{p_{eH}}{R_g T_{eH}} A_{eH} V_{eH}$$

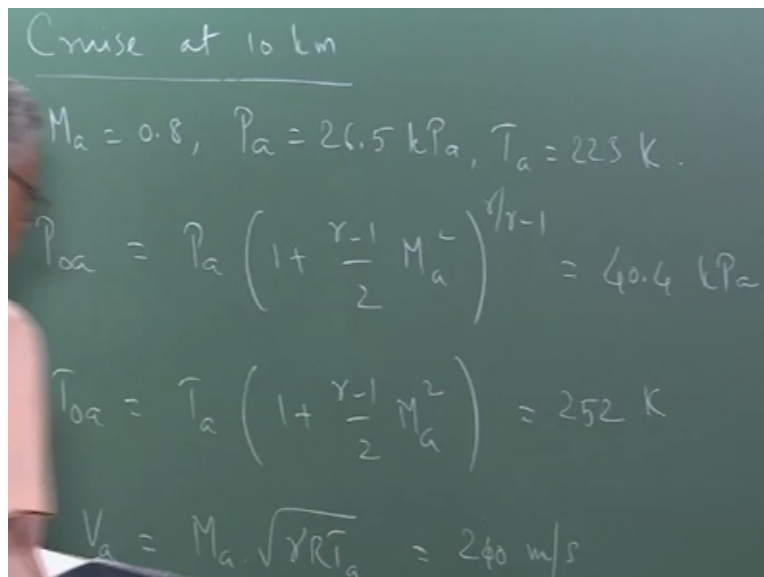
$$\therefore A_{eH} = 0.511 \text{ m}^2$$

So \dot{m}_H is going to be $= \frac{1}{b+1}$ * \dot{m} . So the mass flow rate through the hot nozzle is about 126.177 kg per second and this is also $= \rho_{eH}$ * A_{eH} * V_{eH} or this is nothing but p_{eH} divided by R_g * T_{eH} . Remember these are combustion gases. So we use R_g here and R here for the cold air, here this is cold air and this is combustion gases so we use R_g for this * A_{eH} * V_{eH} and so this gives me A_{eH} to be the hot nozzle area to be 0.511-meter square.

Now since the calculated value of thrust and the TSFC agree reasonably well with the manufacturer quotes we believe that these areas are correct. This is what the actual values are so we have faith in these areas and we will assume that these are correct. This is important as you place a role later on in what we are going to do. So this completes the calculation procedure for static conditions.

What we will do next is calculate the same quantities for cruise at the given altitude that is what we are going to do. We have been asked to do the same thing for cruising at an altitude of 10 kilometers.

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Cruise at 10 km

$$M_a = 0.8, P_a = 26.5 \text{ kPa}, T_a = 223 \text{ K}$$
$$P_{0a} = P_a \left(1 + \frac{\gamma-1}{2} M_a^2\right)^{\frac{\gamma}{\gamma-1}} = 40.4 \text{ kPa}$$
$$T_{0a} = T_a \left(1 + \frac{\gamma-1}{2} M_a^2\right) = 252 \text{ K}$$
$$V_a = M_a \sqrt{\gamma R T_a} = 240 \text{ m/s}$$

And it is given that $Ma = 0.8$, the ambient pressure at this altitude is given to be 26.5 kilo Pascal and the ambient temperature is given to be 223 Kelvin. Now we know from experience that when the engine or when the air craft cruises at this kind of an altitude the thrust that the engine will produce or thrust the engine is required to produce is much less than the take off thrust. Remember the static thrust or the take off thrust which are almost the same is quite high. So the same engine will produce much less thrust when it is actually cruising.

The throttle settings will be changed mass flow rate of fuel will change and other things may also change, but in the problems statement none of these things are given. We are not given we are

only given that the altitudes cruises at this kind of an altitude. I am sorry the aircraft cruises at this kind of an altitude. So we have to figure out how the conditions have to change. We will do that to some extent as we go along.

For now, we will simply use the same values and go through the calculations to see if there is anything wrong with the predictions. That is our strategy now. We will start with this and let us go through in the same manner. So I can calculate P_{0a} to be $P_a \cdot (1 + \frac{\gamma - 1}{2} Ma^2)^{\frac{\gamma}{\gamma - 1}}$ and if I substitute the numbers I get this to be 40.4 kilo Pascal and T_{0a} can also be calculated in the same manner.

$T_{0a} = T_a \cdot (1 + \frac{\gamma - 1}{2} Ma^2)$ and this comes out to be 252 Kelvin and the free stream speed V_a can be calculated as $Ma \cdot \sqrt{\gamma R T_a}$ and this comes out to be 240 meter per second. So starting from this free stream conditions, we will go through each and every state as we did before. So we go to the end of the inlet, end of the fan, end of the fan nozzle which completes the fan stream then we will go to the compressor core stream and then finish the calculation.

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At entry to fan, $T_{01} = T_{0a} = 252 \text{ K}$.

$$T_{01s} = T_a \left[1 + \eta_{\text{inlet}} \left(\frac{T_{01}}{T_a} - 1 \right) \right]$$

$$= 247 \text{ K}$$

$$P_{01s} = P_a \cdot \left(\frac{T_{01s}}{T_a} \right)^{\frac{\gamma}{\gamma - 1}} = 38.04 \text{ kPa}$$

$$= P_{01}$$

So at the end of the inlet or at entry to fan since there is no energy exchange, heat or work interaction in the inlet $T_{01} = T_{0a} = 252$ Kelvin. So I need to calculate P_{01} also remember we want T_{01} and P_{01} so I calculate P_{01} in the manner that we did before. So I calculate my T_{01s}

from the definition of the isentropic efficiency of the inlet. So $T_{01s} = T_a \cdot 1 + \eta_{\text{inlet}} \cdot T_{01}$ divided by $T_a - 1$.

This is the definition of the isentropic efficiency for the inlet. So I substitute the numbers I get my T_{01} is to be 247 Kelvin. Once I have T_{01} is I can use the isentropic relationship to calculate P_{01s} . $P_{01s} = P_a \cdot T_{01s}$ divided by T_a to the power $\gamma/\gamma-1$. So if I substitute the numbers, I get this to be 38.04 kilo Pascal and if you remember from our earlier lecture $P_{01} = P_{01s}$ because both of them lie on the same isobar. That is how we define the efficiency so this is $= P_{01}$. So now I have T_{01} and P_{01} .

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At the fan exit,

$$P_{02} = P_{01} \cdot \text{FPR} = 64.67 \text{ kPa}$$

$$P_{02s} = P_{01} \cdot \left(\frac{T_{02s}}{T_{01}} \right)^{\gamma/\gamma-1}$$

$$\Rightarrow T_{02s} = T_{01} \cdot \left(\frac{P_{02s}}{P_{01}} \right)^{\gamma-1/\gamma}$$

$$= 293 \text{ K}$$

Now at the fan exit we move to the next state. At the fan exit $P_{02} = P_{01} \cdot \text{fan pressure ratio}$ and fan pressure ratio is given to be 1.7 so we change P_{02} is 64.67 kilo Pascal. So I have P_{02} now. I have to calculate T_{02} . Earlier I had the stagnation temperature and I needed to calculate the corresponding stagnation pressure. Now I have the stagnation pressure and I need to calculate the corresponding stagnation temperature which is what we are going to do next.

So remember $P_{02s} = P_{02}$. So I use that fact and do the following. Remember $P_{02s} = P_{01} \cdot T_{02s}$ divided by T_{01} raise to the power γ over $\gamma-1$ which tells me that I can calculate my T_{02s} as $T_{01} \cdot P_{02s}$ divided by P_{01} to the power $\gamma-1/\gamma$ and $P_{02s} = P_{02}$ and both the states lie on the same isobar so I can substitute the numbers and If I do that I get T_{02} is to be 293

Kelvin. So once I have T_{02s} by using the definition of the isentropic efficiency of the fan I can calculate T_{02} .

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Handwritten equations on a chalkboard:

$$T_{02} = T_{01} \left[1 + \frac{1}{\eta_{fan}} \left(\frac{T_{02s}}{T_{01}} - 1 \right) \right]$$

$$= 296 \text{ K}$$

For the fan nozzle,

$$P_{ec, crit} = P_{02} \left(1 - \frac{1}{\eta_{nozzle}} \frac{\gamma - 1}{\gamma + 1} \right)^{\frac{\gamma}{\gamma - 1}}$$

$$= 33.43 \text{ kPa}$$

Since $P_a < P_{ec, crit}$, fan nozzle is choked.

So T_{02} is going to be $15:38 = T_{01} * 1 + 1/\eta_{fan} * T_{02s}/T_{01} - 1$. So this is the definition of the isentropic efficiency of the fan and if you substitute the numbers I get T_{02} to be 296 Kelvin so now we have P_{02} and T_{02} . So the next component in the fan stream is going to be the fan nozzle. So let us go ahead and complete the fan stream. For the fan nozzle since I know P_{02} I can calculate the critical pressure P for the cold nozzle.

Critical pressure = $P_{02} * 1 - 1/\eta_{nozzle} * \gamma - 1/\gamma + 1$ raise to the power $\gamma/\gamma - 1$ and if you substitute the values we get the critical pressure to be 33.43 kilo Pascal and contrary to the static condition where the ambient pressure was 100 kilo Pascal. Now the ambient pressure is 26.5 kilo Pascal and the critical pressure is 33.43 kilo Pascal which tells me that the fan nozzle is going to be choked now in the previous case it was not choked now fan nozzle is choked since $P_a < P_{ec, crit}$, fan nozzle is choked.

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$$P_{ec} = P_{ec, \text{crit}} = 33.43 \text{ kPa}$$

$$\frac{P_{ec, s}}{P_{02}} = \left(\frac{T_{ec, s}}{T_{02}} \right)^{\gamma/\gamma-1}$$

$$T_{ec, s} = T_{02} \cdot \left(\frac{P_{ec, s}}{P_{02}} \right)^{\gamma-1/\gamma} = 245 \text{ K}$$

$$T_{ec} = T_{02} - \eta_{\text{nozzle}} (T_{02} - T_{ec, s})$$

$$= 247 \text{ K}$$

So that means that the exit pressure $P_{ec} = P_{ec}$ critical since the nozzle is choked and that is = 33.43 kilo Pascal. So I know the exit static pressure. I need to calculate the exit velocity. I calculate the exit velocity by making use of the fact that P_{ec} isentropic = P_{ec} both of those states lie on the same isobar so which means that I can do the following. I can use the following relationship P_{02} let me arrive it like this.

P_{ec} for an isentropic process divided by $P_{02} = T_{ec, s}$ divided by T_{02} raise to the power $\gamma/\gamma-1$. So from this I can calculate $T_{ec, s}$. So, $T_{ec, s} = T_{02} \cdot P_{ec, s}$ divided by P_{02} raise to the power $\gamma-1/\gamma$ and this value is = P_{ec} . So remember both this and P_{ec} lie on the same isobar. So I can substitute the value and get this temperature to be = 245 Kelvin.

Now the actual exit static pressure T_{ec} can be calculated from the definition of the isentropic efficiency of the nozzle in this way so this = $T_{02} - \eta_{\text{nozzle}} \cdot T_{02} - T_{ec, s}$. So this gives me the actual static temperature at the exit of the cold nozzle. Substitute the numbers we get this to be 247 K. So I have the static pressure I have the exit static temperature from which I can calculate the exit velocity. V_{ec} can be calculated very easily. $V_{ec} =$ let us do that.

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$$V_{ec} = \sqrt{2 C_p (T_{02} - T_{ec})} = 315 \text{ m/s}$$

$$T_{fan} = \frac{B}{B+1} \dot{m} \left[(V_{ec} - V_a) + \frac{R T_{ec}}{P_{ec} V_{ec}} (P_{ec} - P_a) \right]$$

$$= 41.075 \text{ kN} + 25.365 \text{ kN}$$

$$= 66.44 \text{ kN}$$

V_{ec} = square root of $2 C_p * T_{02} - T_{ec}$ and this comes out to be 315 meter per second. Working with this number also gives you an idea of what values or magnitudes these numbers have what kind of velocities do you have at the exit of the nozzle, fan nozzle and the exit of the core engine nozzle and so on. Now the thrust from the fan stream can be written as so this is the momentum thrust plus the pressure thrust which is nothing but so the second term is the pressure thrust.

There was an area term which normally multiplies the pressure thrust. I have taken out the mass flow rate. I have divided by $\dot{m} \cdot c$. So that area divided by $\dot{m} \cdot c$ is what gives rise to this expression here. $\dot{m} \cdot c$ is nothing but $\rho_{ec} * A_{ec} * V_{ec}$ so that is what I have used here to write this expression. This is the momentum thrust, this is the pressure thrust and if you substitute the numbers we get this to be 41.075 kilo Newton + 25.365 kilo Newton which adds up to 66.44 kilo Newton.

Notice that we have not touched or changed this value of \dot{M} . We are still assuming \dot{M} dot to be whatever was given for sea level static condition. Remember \dot{m} dot was given to be 670 kg per second. We are using the same value here and as I said earlier it is very likely that when the engine is cruising at 10 kilometers altitude the mass flow rate is likely to be less than what it takes in under sea level conditions.

So now the mass flow rate is less due to 2 reasons number 1 the density of the air itself changes at the higher altitude and number 2 the throttle setting may also change to produce a certain amount of thrust. So we will see what changes need to be made. For now, we will use this and then go ahead with this. Now notice that the momentum thrust is about roughly 2 times the pressure thrust in this case.

Under sea level condition, this was completely 0 only this momentum thrust was present. So that completes the fan stream. We now move on to the core engine stream.

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Core engine stream:

At the exit of the HP compressor,

$$P_{03} = PR \cdot P_{01} = 1156.4 \text{ kPa.}$$

$$T_{03s} = T_{02} \cdot \left(\frac{P_{03s}}{P_{02}} \right)^{\gamma-1/\gamma} = 675 \text{ K.}$$

$$T_{03} = T_{02} \left[1 + \frac{1}{\eta_{\text{comp}}} \left(\frac{T_{03s}}{T_{02}} - 1 \right) \right]$$

$$= 708 \text{ K}$$

So as you said earlier part of the air and amount = \dot{m} goes through the fan and then it goes through the core engine. So we are going to track the \dot{m} amount of here which now goes through the core engine. So we start from the fan outlet and then we go to the exit of the high pressure compressor. So at the exit of the HP compressor $P_{03} = \text{pressure ratio } PR \cdot P_{01}$ so if you plug in the numbers you get this to be 1156.4 kilo Pascal.

So I know P_{03} . I need to calculate T_{03} now. So what we do? same procedure as earlier. We evaluate since I know P_{03} I know P_{03s} so I evaluate T_{03s} . So $T_{03s} = T_{02} \cdot P_{03s}$ divided by P_{02} raise to the power $\gamma-1/\gamma$ and if I substitute the values $P_{03s} = P_{03}$ and if I substitute the values I get this to be 675 Kelvin. Now since I know T_{03s} I can use the definition of the

isentropic efficiency of the turbine to calculate my T_{03} and so $T_{03} = T_{02} * 1 + 1/\eta_{\text{compressor}}$ * T_{03s} divided by $T_{02} - 1$.

And if I plug in the numbers I get this to be 708 Kelvin. So this is consistent with what we were telling earlier that the temperature at the end of the compression process is around 700 Kelvin or so. So this confirms what we are saying earlier. so the air at the end of the compression process comes out at a temperature of around 700 Kelvin then we add fuel, burn fuel, to raise the temperature to some value around 1500 or 1600 Kelvin so that is how much energy is being added in the combustor and that is what we are going to look at next.

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At the combustor exit,
 $T_{04} = 1500 \text{ K}$ and
 $P_{04} = 0.95 P_{03} = 1099 \text{ kPa}$

At the HP turbine exit,

$$T_{05} = T_{04} - \frac{1}{\eta_{\text{mech}}} \cdot \frac{C_p}{C_{p9}} (T_{03} - T_{02})$$

$$= 1120 \text{ K}$$

$$T_{055} = T_{04} - \frac{1}{\eta_{\text{turbine}}} (T_{04} - T_{05})$$

So at the combustor exit $T_{04} = 1500$ Kelvin and $P_{04} =$ it is given that there is a 5% loss of stagnation pressure in the combustor so P_{04} is $0.95 * P_{03}$ and this comes out to be 1099 kilo Pascal. Once again just like the total mass flow rate we are not going to change this quantity also for now. but when we are cruising at an altitude of 10 kilometers for in order to produce less thrust the engine will take in less amount of air plus the amount of fuel burned will also be less,

Little bit less than what it is going to be for takeoff condition, but we do not know that value. You have not been given that value in the problem description. So we will proceed with this and then try to correct this or rectify this later on. So let us proceed. We will remember that this value

most likely needs to be changed and the $m \dot{v}$ value also most likely needs to be changed. So this value also most likely needs to be changed.

We will keep that in mind and proceed. So the next component is the high pressure turbine. So at the HP turbine exit, from an energy balance remember the HP turbine produces the amount of work that is required to run the HP compressor. So T_{05} can be calculated based on an energy balance so $T_{05} = T_{04} - 1/\text{the mechanical efficiency} * C_p/C_{pg} * T_{03} - T_{02}$, and if you substitute the numbers we get this to be 1120 Kelvin.

So I have T_{05} now. I need to evaluate P_{05} . So to do this what I do is I calculate my T_{05s} using the definition of the isentropic efficiency of the turbine so I can calculate $T_{05s} = T_{04} - 1/\eta_{\text{turbine}} * T_{04} - T_{05}$. So this is the definition of the isentropic efficiency of the turbine so I can calculate T_{05s} .

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The image shows a chalkboard with handwritten equations. At the top, it says $= 1078 \text{ K}$. Below that, the equation for T_{05s} is written as $T_{05s} = T_{04} - \left(\frac{T_{05s}}{T_{04}} \right)^{\frac{\gamma}{\gamma-1}} = 293 \text{ kPa}$. Below this, it says $= P_{05}$. Then, it says "At the exit of the HP turbine," followed by the equation $T_{05} = T_{04} - \frac{1}{\eta_{\text{turbine}}} \left(\frac{\gamma}{\gamma-1} \right) (T_{04} - T_{05})$. Below that, it says $= 1120 \text{ K}$. Finally, it says $T_{05s} = T_{04} - \frac{1}{\eta_{\text{turbine}}} (T_{04} - T_{05}) = 1078 \text{ K}$.

So if substitute the numbers you get T_{05s} to be 1078 Kelvin and once I have T_{05s} I can actually calculate P_{05s} using the isentropic relationship. So $P_{05s} = P_{04} * T_{05s}$ divided by T_{04} raise to the power. Now we are using γ/g please bear that in mind these are combustion gases so this is $\gamma/g/\gamma/g - 1$ and γ/g itself was given to be 1.333. So we substitute these numbers and we get P_{05s} to be 293 kilo Pascal and this is also = P_{05} .

So now we have both T05 and P05. The next component is the low pressure turbine. Let us go ahead and do that. So at the exit at the LP turbine so the LP turbine produces enough power to run the fan so energy balance for this gives me $T_{06} = T_{05} - 1/\eta_{\text{mechanical}} * B+1 * C_p/C_{pg} * T_{02} - T_{01}$. Notice that the expression for the exit stagnation temperature of the LP turbine is little bit different from the expression for this exit temperature for the HP turbine mainly.

Because there is no mass flow rate here, but there is a B+1 here. Mainly because the HP compressor handles an amount of air = $m \cdot H$. The HP turbine also handles an amount of air = $m \cdot H$. So that $m \cdot H$ cancels out whereas in this case, the LP turbine handles an amount of air = $m \cdot H$ whereas the fan handles an amount of air = $m \cdot C + m \cdot H$ which is why we are getting this additional B+1 here.

So you substitute the numbers and we get this to be 901 Kelvin and once I have T06 I need to evaluate T06s and we will use the same procedure. So we calculate T06s maybe I can write it over here so T06s looks like this. So $T_{06s} = T_{05} - 1/\eta_{\text{turbine}} * T_{05} - T_{06}$. So this is the definition of the isentropic efficiency of the LP turbine and if we substitute the numbers you get this to be 877 Kelvin. So once I have T06s I can calculate P06s from the isentropic relationship.

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The image shows a chalkboard with the following handwritten equations and text:

$$P_{06s} = P_{05} \left(\frac{T_{06s}}{T_{05}} \right)^{\frac{\gamma_g}{\gamma_g - 1}} = 110 \text{ kPa}$$

$$= P_{06}$$

$$P_{eH, \text{crit}} = P_{06} \left(1 - \frac{1}{\eta_{\text{nozzle}}} \cdot \frac{\gamma_g - 1}{\gamma_g + 1} \right)^{\frac{\gamma_g}{\gamma_g - 1}}$$

$$= 58 \text{ kPa}$$

Since $P_a < P_{eH, \text{crit}}$, hot nozzle is choked.

So $P_{06s} = P_{05} * T_{06s}$ divided by T_{05} raised to the power γ_g divided by $\gamma_g - 1$ and this value comes out to be 110 kilo Pascal. So I have now this is also = P06. So I have T06 and P06.

The next component is the hot nozzle. So what we do is we first evaluate the critical pressure. So P_{eH} for the hot nozzle exit pressure hot nozzle critical value is given by.

$P_{06} \cdot \left(\frac{1}{\gamma} \right)^{\frac{\gamma}{\gamma-1}}$ divided by γ raised to the power $\frac{\gamma}{\gamma-1}$ and if you plug in the numbers we get this to be 58 kilo Pascal and the ambient pressure is let us see. The ambient pressure is given to be 26.5 kilo Pascal in this case so since $P_a < P_{eH}$ critical hot nozzle is choked, which means that the exit static pressure is = the critical pressure.

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$$\begin{aligned} \therefore P_{eH} &= P_{eH, \text{crit}} = 58 \text{ kPa} \\ T_{eH,s} &= T_{06} \cdot \left(\frac{P_{eH,s}}{P_{06}} \right)^{\frac{\gamma-1}{\gamma}} = 767 \text{ K} \\ T_{eH} &= T_{06} - \eta_{\text{nozzle}} (T_{06} - T_{eH,s}) \\ &= 771 \text{ K} \\ V_{eH} &= \sqrt{2 C_p (T_{06} - T_{eH})} = 543 \text{ m/s} \end{aligned}$$

Therefore, the exit static pressure $P_{eH} = P_{eH}$ critical which is = 58 kilo Pascal. So I have the static pressure. Now I can calculate the I have to calculate the static temperature so what I do is I calculate $T_{eH,s}$. So T_{eH} for isentropic process is going to be $T_{06} \cdot P_{eH}$ isentropic process divided by P_{06} raised to the power $\frac{\gamma-1}{\gamma}$ at I know P_{eH} for the isentropic process because this is = P_{eH} .

Both the state points lie on the same isobar. I know this so I can calculate $T_{eH,s}$ which comes out to be 767 Kelvin. Once I have this I can evaluate T_{eH} by using the definition of the isentropic efficiency of the nozzle so T_{eH} is = $T_{06} - \eta_{\text{nozzle}} \cdot (T_{06} - T_{eH,s})$. so the exit static temperature we substitute the numbers comes out to be 771 Kelvin and now I can calculate the exit velocity V_{eH} is = square root of $2 \cdot C_p \cdot (T_{06} - T_{eH})$ and this comes out to be 543 meter per second. So now I can go ahead and calculate the thrust from the core engine.

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$$\begin{aligned} T_{\text{core}} &= \frac{\dot{m}}{B+1} \left[(V_{eH} - V_a) \right. \\ &\quad \left. + \frac{R_g T_{eH}}{P_{eH} V_{eH}} (P_{eH} - P_a) \right] \\ &= 38.35 \text{ kN} + 28 \text{ kN} \\ &= 66.35 \text{ kN} \\ T &= T_{\text{fan}} + T_{\text{core}} = 132.79 \text{ kN} \\ &\quad (57.91 \text{ lbf}) \end{aligned}$$

So T from the core engine is $= \dot{m} / (B+1) * (V_{eH} - V_a)$ which is the momentum thrust $+ R_g * T_{eH}$ divided by $P_{eH} * V_{eH} * (P_{eH} - P_a)$ which is the pressure thrust and once again I am not going to touch this mass flow rate I will use the same value as before and if I do that I get this to be 38.3 kilo Newton + 28 kilo Newton which gives a total value of 66.35 kilo Newton from the core engine.

So the total thrust T is $= \text{fan thrust} + \text{core engine thrust}$ which works out to the problem is the fan thrust if you check back your numbers you will see that the fan thrust was about 66. something. So the core engine thrust is also 66. something. Now they appear to be the same okay, but the overall thrust itself reduces to about 132.79 kilo Newton. So the thrust has gone down from 229 kilo Newton to 132.79 kilo Newton. Let us calculate the mass flow rate of fuel and then we will compare our numbers with the book values.

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$$\dot{m}_f = \frac{\dot{m}_H (C_{p3} T_{04} - C_p T_{03})}{H_{fuel} - C_{p3} T_{04}}$$

$$= 2.94 \text{ kg/s}$$

$$TSFC = \frac{\dot{m}_f}{\gamma} = 0.08 \text{ kg/hr}\cdot\text{N}$$

$$A_{ec} = 3.661 \text{ m}^2 \quad (0.0642 \text{ kg/hr}\cdot\text{N})$$

$$A_{eH} = 0.8855 \text{ m}^2 \quad (1.5 \text{ m}^2)$$

$$\quad \quad \quad (0.511 \text{ m}^2)$$

So the mass flow rate of fuel \dot{m}_f can be evaluated as $\dot{m}_H \cdot C_{p3} T_{04} - C_p T_{03}$ divided by $H_{fuel} - C_{p3} T_{04}$ and if I substitute the numbers I get this to be 2.94 kg per second and the TSFC works out to be $\dot{m}_f / \text{total thrust}$ works out to be about 0.08 kilogram per hour Newton. So now we have to see what we are seeing at the end of the tunnel a sunlight or the head light of the oncoming train that we do not know.

That we will know only when we compare these values with the actual book values now the quoted value by the manufacturer for cruise thrust at this altitude is about 51.41 kilo Newton and the TSFC is the manufacturer quotes is about 0.0642 kg per hour Newton. So it appears that this not light at the end of the tunnel, but the head light of an oncoming train. So we need to fix do something.

We need to do something to make this compare better. What do we do? We can figure out what to do if we if you proceed in the same way as before. What if I calculate the area of the core engine nozzle and hot nozzle for these conditions So if I calculate the area of the cold nozzle and the hot nozzle for this conditions I get this to be 3.661 meter square and the hot nozzle area to be 0.8855 meter square and if you remember earlier.

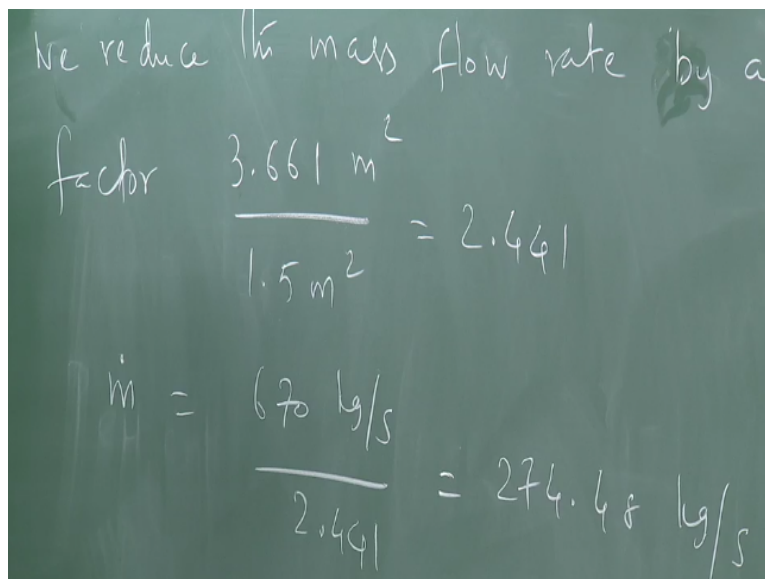
We calculated the same values to be 1.5 meter square and this was calculated to be 0.511-meter square. So now we can see what the problem is. We had kept \dot{m}_f and T_{04} the same. If you do

that and go ahead with the calculation, we can see that there is a poor agreement with the value quoted by the manufacturer and the reason for that lies in this to pass the same mass flow rate and have the same T04 we need these kinds of areas for the cold and the hot nozzle.

But remember this is the turbo fan engine with fixed nozzle. It does not have adjustable area nozzle which means that I need to redo my mass flow rate so that the nozzle area will come out to be same. So the same nozzle must pass that mass flow rate so let us start with the fan nozzle we can simply scale down the mass flow rate to agree with this remember our exit velocity and other things that we calculated for example if you look at VeH, mass flow rate was not involved.

In any in these kinds of expression so the VeH is not going to change everything was taken into account properly everything was a ratio. So if I scale down the mass flow rate by a factor of half Vec will remain the same and I should be able to get a better agreement with manufacturer quoted values so that I can do for 1 nozzle then what do I do the other nozzle that is the question that we must answer next. Let us see what happens with this. So let us do this part and then we will pick it up in the next class.

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We reduce the mass flow rate by a factor

$$\frac{3.661 \text{ m}^2}{1.5 \text{ m}^2} = 2.441$$
$$\dot{m} = \frac{670 \text{ kg/s}}{2.441} = 274.48 \text{ kg/s}$$

So we reduce the mass flow rate by the factor 3.661-meter square divided by 1.5 meter square which gives me about 2.441 which means $\dot{m} = 670 \text{ kg per second divided by } 2.441$ so the

mass flow rate at this altitude should be 274.48 kg per second. So this directly tells me how much the mass flow rate should be changed.

So now if I do this and if I recalculate my A_{ec} for corresponding to this mass flow rate and the same V_{ec} , V_{ec} is not going to change if I do that then I will notice that A_{ec} comes out to be 1.5-meter square perfect what happens to A_{eH} and how do we fix that is what we are going to discuss in the next class.