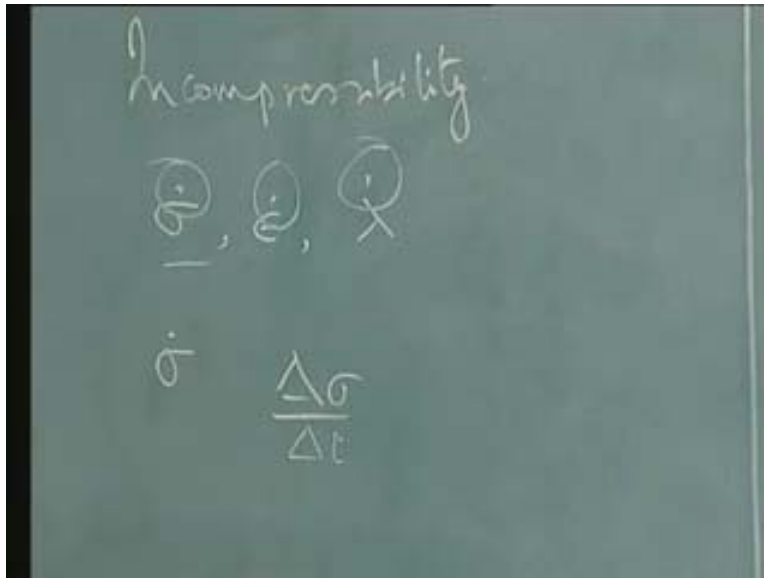


Advanced Finite Element Analysis
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Lecture - 9

In the last class, we were looking at incompressibility.

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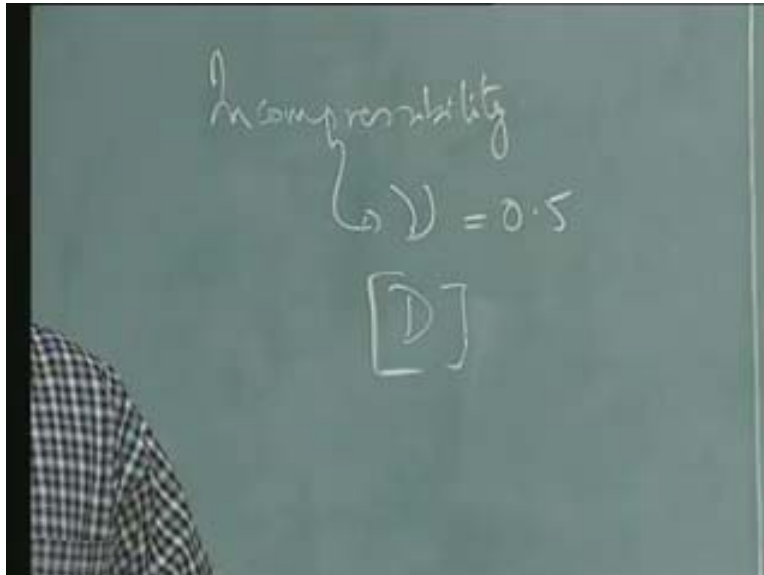
Before that, there was a question on what we mean by dot? Say, sigma dot, epsilon dot or lambda dot, what this dot really means? There is a small confusion. I had said that, said this before and I am just repeating that; we are not into what is called as rate dependent phenomena when we talk about plasticity. These rates are pseudo rates. So, sigma dot] is perfectly equivalent to writing delta sigma, both of them are the same. That is in other words, delta sigma when divided just by delta t, what is this? Time; time has nothing to do with your application of load or loading time or whatever it is, it has nothing to do with that. Just it is a pseudo time to only capture that the loading has to be applied in an incremental fashion. That is all. So, this lambda dot, sigma dot, epsilon dot in one sense conveys that the approach is incremental in nature. But we are going to see, there are

some other things that come into picture later in the course, but right now you can say that $\dot{\sigma}$ and $\Delta \sigma$, they are, both of them are equal.

So, the loading carries itself in time. In one sense the rate of loading is what manifests as or comes out as $\dot{\sigma}$ and so on. But, this rate of loading does not mean that we are taking into account the inertial effects or rate dependent effects or strain rate effects and so on. Once you are looking at strain rate effects, then the time should be the correct time. It cannot be pseudo time. Is that clear? We will, we will give more explanation to it as we go along in the course, but let us now look at a very important thing called incompressibility.

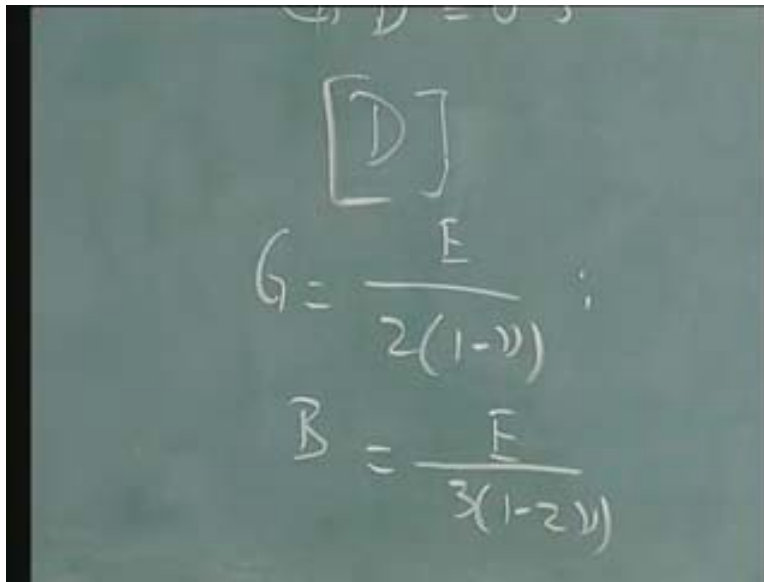
Now, what is incompressibility? There was a question again in the last class, when we finished that what is incompressibility? Incompressibility simply means that there is no volume change. Incompressibility does not mean that we cannot compress it. We can compress it, but when you compress it, the compressive strains that are developed are in such a fashion that there is no change in volume. Is that clear? That is what it means.

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Now, we know that when it is compressible, we will prove it again in a more rigorous setting later in the course that when incompressibility is the condition then ν equal to 0.5. Now, let us see, look at the difficulties that happens that take place when we have ν is equal to 0.5. Since this is a material behavior, the culprit is going to be the D matrix. All of you know that this is the elastic matrix. Now, let us split this D matrix into a part which is going to be my problem or problematic part and a part which is not going to be so very problematic part.

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The image shows a chalkboard with handwritten equations. At the top, there is a boxed letter 'D'. Below it, the shear modulus G is defined as $G = \frac{E}{2(1-\nu)}$. Below that, the bulk modulus B is defined as $B = \frac{E}{3(1-2\nu)}$.

In order to do that let me define the shear modulus μ or G to be E divided by 2 into 1 minus ν and the bulk modulus B to be E divided by 3 into 1 minus 2 ν and say that this D now can be split into two parts, one which goes along with the shear modulus, the other part which goes along with the bulk modulus. Let us see how we can split this D .

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The image shows a chalkboard with the following handwritten equation:

$$[D] = G \begin{bmatrix} 4/3 & -2/3 & -2/3 & 0 & 0 & 0 \\ -2/3 & 4/3 & -2/3 & 0 & 0 & 0 \\ -2/3 & -2/3 & 4/3 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 1 & 1 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} + B \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

This is regular D, we are not talking about dt now; just we are going to look at regular D. This D can now be written as, say for example, G into 4 by 3 minus 2 by 3 minus 2 by 3 minus 0 0 0, minus 2 by 3 4 by 3 minus 2 by 3 0 0 0 and so on, as one part plus the other part consists of B into, B is my bulk modulus into and so on. Of course, the last rows are zeros. Then it is, this becomes 0 0. Yeah, this part, this is zero; this is, no, third is minus 2 by 3 minus 2 by 3 0 0 0; the rest of it, 0 4 by 3 or sorry 1 1 1 and so on. Yeah, this is 6 by 6 matrix; this is a 6 by 6 matrix which of course we are talking about.

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The chalkboard shows the following equations and matrices:

$$\{\sigma\} = [D] \{\epsilon\}$$
$$D_S = \begin{bmatrix} -2/3 & 4/3 & -2/3 & 0 & 0 & 0 \\ -2/3 & -2/3 & 4/3 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \end{bmatrix}$$
$$+ \underline{\underline{B}} \begin{bmatrix} 1 & 1 & 1 & 0 & 0 & 0 \\ 1 & 1 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$
$$D_B \begin{bmatrix} 1 & 1 & 1 & 0 & 0 & 0 \\ 1 & 1 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

This is the relationship between sigma is equal to D epsilon. So, we are splitting this D into two parts, what goes into B transpose dB? Now, if you look at these two parts, the culprit obviously is here, because you have 1 minus 2 nu term here. Now substituting, let me call this as say D_S, first part as D_S and second part say as D_B to indicate that one part is formed by shear modulus and the other part formed by bulk modulus, so that my total K consists of B transpose, instead of D, I am going to say that D shear modulus and bulk modulus B d omega.

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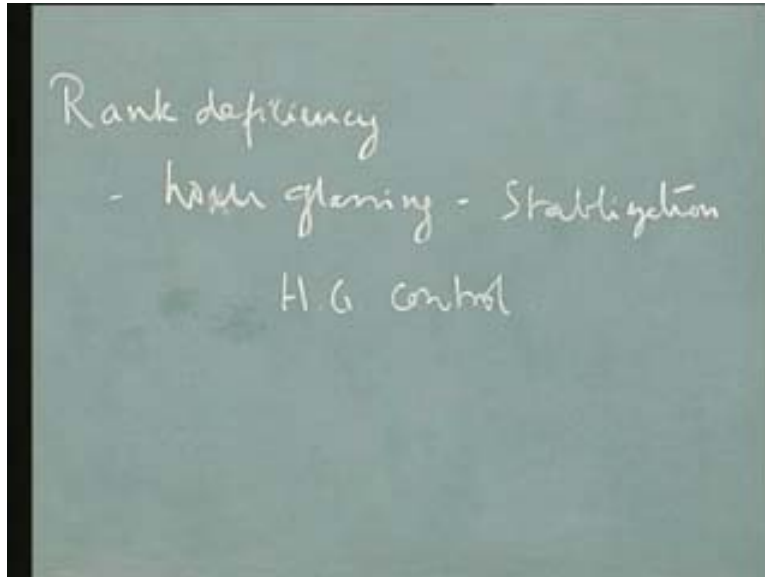
$$K = \int_{\Omega} B^T [D_S + D_B] B d\Omega$$
$$= \int_{\Omega} B^T D_S B d\Omega + \int_{\Omega} B^T D_B B d\Omega$$
$$[K] = [K_S + K_B]$$

This is D_S and D_B is what I am going to write, D_S . So, I have two terms now in K . So, let me call that then B transpose $D_S B$ d Ω plus B transpose $D_B B$ d Ω . So, in other words, my K can be split into two parts say K_S plus K_B . Is it clear? Of course, D_S is complete thing. So, this B goes here. You have a B here, I mean of course it goes into D_B . What happens when we get ν is equal to 0.5? B goes to infinity. No, no; please note that in G , I have only ν . So, when ν is equal to 0.5, this G does not go to infinity, but when I have B which is there in this K_B , second part where I have a B , then you will see that E by $1 - 2\nu$, ν is equal to 0.5, so this goes to infinity.

As I approach ν to be 0.5, all my trouble starts; even at 0.49 or 95, 499 and so on, this will make this K matrix, the situation will make the K matrix to be highly ill conditioned and inverse here of this matrix is going to be difficult, because once it becomes ill conditioned that means that it is going to a situation where it is, where it is almost you know infinity, then I will have problems. At 0.5, completely incompressible, then it is totally out. May be you can work up to 0.45, 44; it may not be a problem. Now what do I do? What do I do to alleviate this problem?

In a crude sense, I mean this is not, I am not going to talk total theory; we will come to that a bit later. In a crude sense you may think that there is a solution. What is the solution? Solution comes from a possible mistake I may make, use the integration rule.

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Probably you would remember or studied, what is called as rank deficiency or hour glassing. In our earlier class, we said that there is a particular integration rule I have to use in order that I can integrate this accurately; polynomial of $2n$ minus 1 integrated accurately with an order of n . In other words, we said that for a quadrilateral, I have to use a 2 by 2 integration. What happens when I reduce this integration, when I go to 1 by 1 rule? I will get into what is called as rank deficiency, what is called as hour glassing.

The rank of the matrix is not sufficient enough for me to invert and make it stable. I will get into singularity problems and so on. So, there is what is called rank deficiency when I reduce the numerical integration and so I avoided it. So, I will go for full integration. If it all I do reduced integration, I do other procedures called stabilization procedures. So, that is called as hour glassing and in order to avoid hour glassing, I did some what is called as stabilization procedure or in other words, in other words, you call this as hour glass control.

In fact, if you had used many of the explicit finite element code, you would see that most of them use reduced integration and they will have a proper hour glass control. In other words, by making a small, if I may call it a mistake, mistake within quotes, I got into some trouble and I got out also by doing some damage exercises. So, major thing is that that matrix is becoming singular when I did those things. Now, what I am going to do now is very similar thing; I am going to make the mistake wantonly. How I am going to do that? Having known that this kind of problems will exist, what I will do is simply take this and perform an under integration or reduced integration.

In other words, I selectively integrate a part of the K matrix in a reduced fashion. So, instead of 2 by 2, I may use 1; 1 by 1 row. If I need 3 by 3, I will use 2 by 2 row; I can reduce the integration, order of integration. So, once I make that decision my problem to a great extent is solved, because this K_B matrix will become now singular. This will not have, so this, the whole, though I will still have K from here. So, this contribution, this, this problem of inverting this and going to infinity will not happen. That is called as selective reduced integration. This problem of inverting this, I will not have difficulty. I will have difficulty in the sense that that would be automatically taken care of by reduced integration.

Now, this looks like a crime. People call this as crimes and so on; variational crimes, common word that is used many times. This looks like a crime, but a beautiful paper by Semo and Hughes showed that after all what you are doing is very well within the framework of variational approach. So, though it looks as if we can do the whole thing by learning from a mistake, it interestingly has a very good theoretical background as well. It comes from some of the variational principles which you might not have heard and we will see that in this course. So, by doing this selective reduced integration, I can in one word alleviate many of the problems. Is that clear?

How do you find out the order? But, you know the full order of integration that is required, this depends upon the powers or the the polynomials that are there. That is why I said $2n$ minus 1 polynomial is integrated exactly by end point integration rule. So,

depending upon what is that you have, you can find out the exact and then you can reduce it accordingly. Now, this is fine as far as this kind of simple elastic problem are concerned and the approach seems to be sort of though not very correct, looks logical.

Now, how do we do a problem like plasticity or some other aspect where I cannot write this very clearly like D_S and D_B and so on. Now, various methods have been proposed and one of the methods which is very popular is called as B-bar method of Hughes. We will see more theory after we do this. We are going to do the same selective reduced integration, extend this concept what we have given here. I have not introduced, there is lot more again simple things that you can talk about this. You will see that this B here works like a penalty constant, but since we have not done lot more on penalty approach and all this, I am not commenting on that, but suffice it for me to take me to the next group of algorithms which come under what are called as B-bar method.

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The image shows a chalkboard with the following handwritten equations:

$$K = \int_{\Omega} B^T [D_S + D_B] B \, d\Omega$$

$\underbrace{\hspace{10em}}_D$

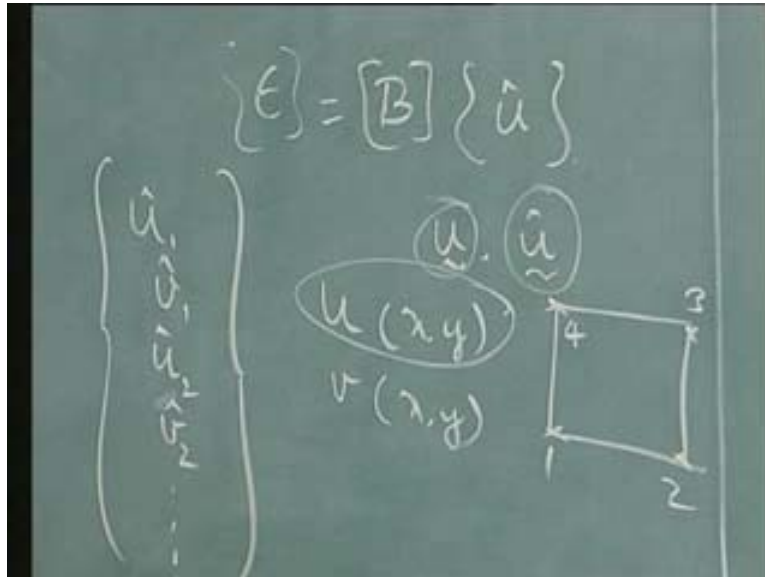
$$= \int_{\Omega} B^T D_S B \, d\Omega + \int_{\Omega} B^T D_B B \, d\Omega$$

$$[K] = [K_S + K_B]$$

What essentially B-bar method does is to now alter not this D, but to alter B. After all you can see that the culprit seems to be in the volumetric part and so can I now split B into two parts - a deviatoric and a volumetric part and do something to the volumetric part of K, which results from splitting this B, that B volume. Let us proceed instead of D

splitting B. Yes, what is this B? Correct, the B is the strain displacement relationship; B is the strain displacement relationship which is epsilon.

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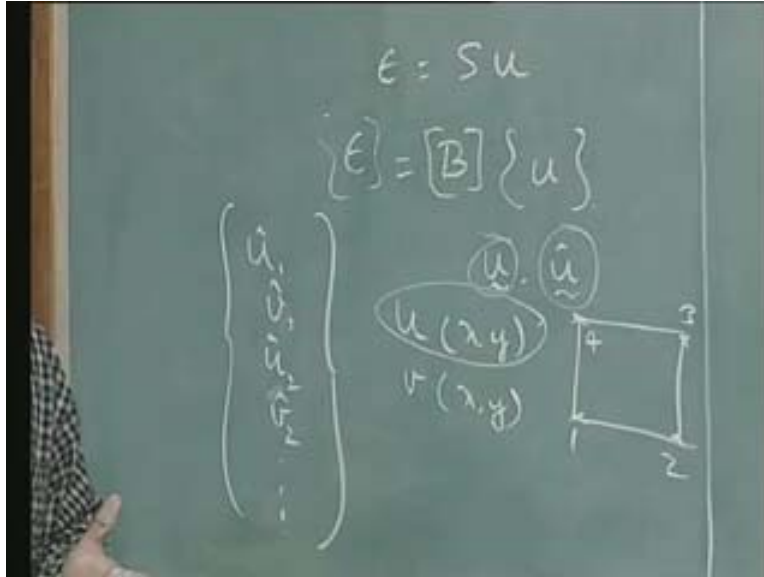


In this case epsilon happens to be B into the degrees of freedom, degrees of freedom, which you can call this as u hat. Now, there is a difference between u and u hat. Note this subtle difference between them. u is the displacement of a continuum. **Now I am going to use this because, I need some other, you know, alleviate to define these two separately.** So, u is a displacement of a continuum. In other words, u is a vector which can be written as say this u is u x comma y and v x comma y. This u hat is the u at the say, nodal locations; it is at the nodal locations. Suppose I have 1 2 3 4, then $u_1 u_2 v_1, u_1 u_2 u_3 u_4 v_1 v_2 v_3 v_4$ that come under u hat. So, B is of course the strain displacement matrix. See, this is the continuum u. This u and v are the continuum u. This u hat is after this continuum is discretized and suppose I call this as 1 2 3 and 4, then u hat gives me a vector of the form say $u_1 v_1 u_2$ sorry v_2 and so on up to $u_4 v_4$. This vector is what I call by this u hat.

Yeah, without hat; yes, that is because in the next formulation, I am going to use certain things with u as well. So, that is why we are looking at this more closely. As you raise

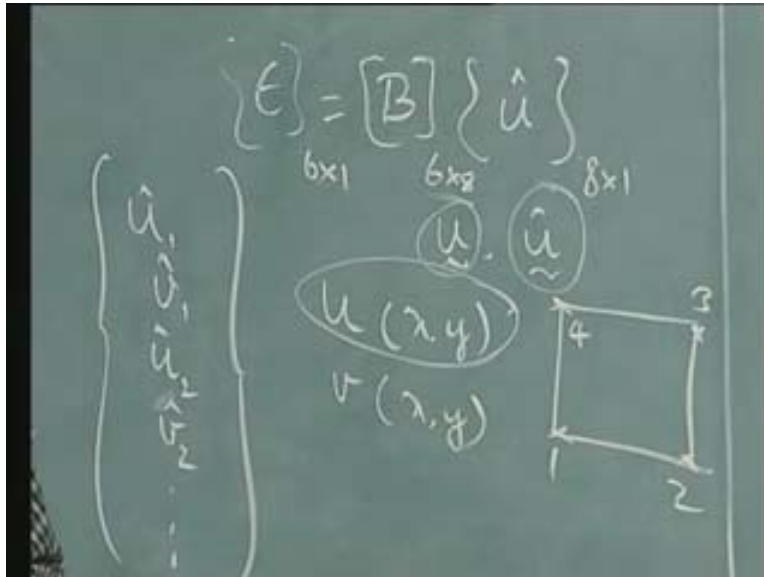
your level of understanding, you have to distinguish between u and \hat{u} or in other words continuum u and discretized u .

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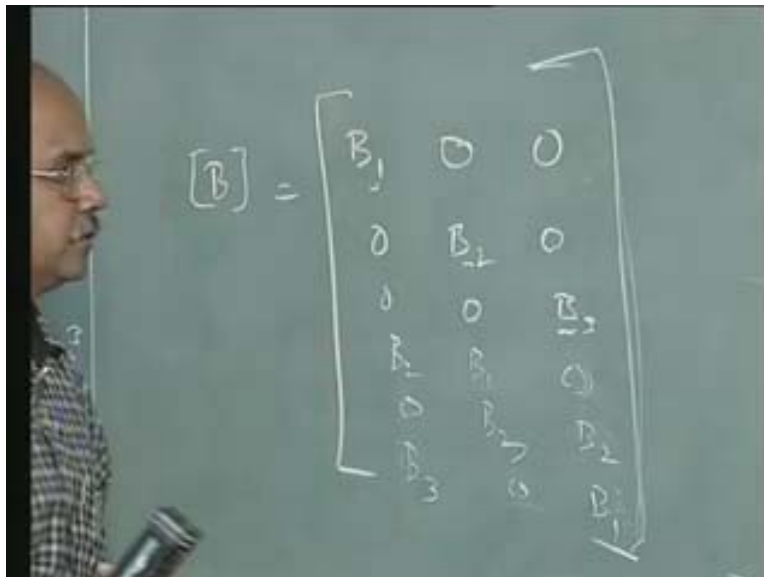
Say, maybe in earlier classes we had put this as $B u$ and understood that this u is this. We never worried about the continuum u . Sometimes it may be necessary to have that also, so that strain ϵ we will define it as Su . S is my operator. We will define that again bit later. So, let us now define it like this. Now of course, B is a matrix. B is a matrix which consists of say $B_1 B_2 B_3$ and so on up to B into number of elements.

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So suppose this is, what is it? You know what is the size? This is say 6; 6 by 1 and this happens to be, is it 6 by 1? Yes, for we are talking about this quad, 8 by 1 and so 6 by 8. Is that clear?

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So, this B matrix has a sub matrix as well with two in each can be written as B_1 , B_1 is itself is a sub matrix, $0 \ 0$, $0 \ B_2 \ 0$, $0 \ 0 \ B_3$, all of them are matrices; $0 \ B_2$ or $B_1 \ B_2$. So,

depending upon whether you write ϵ_{23} or ϵ_{13} and then B_3 0 B_1 and B_2 B_1 B_3 B_2 or B_3 , yeah B_2 B_3 . Let us check that out. So, suppose I have ϵ_{11} , this will become, so that should be B_1 0, sorry, it should be B_1 B_2 yes, just we stick on to the same thing sorry B_2 B_1 ; B_2 B_1 0 B_3 B_2 B_3 B_1 . So, let us put it like this. Anyway you know what this B's are.

Now, what we are going to do essentially is to split this B into two parts, a dilatational part and a deviatoric part.

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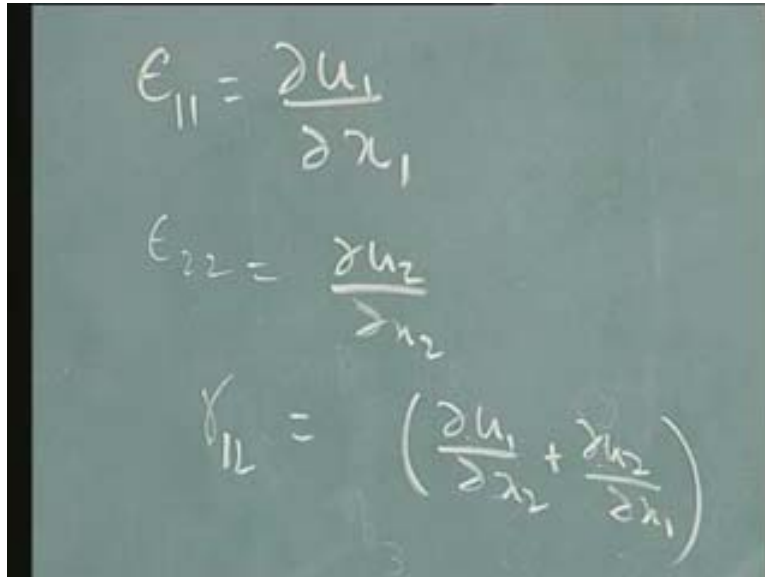
$$B_{dil} = \frac{1}{3} \begin{bmatrix} B_1 & B_2 & B_3 \\ B_1 & B_2 & B_3 \\ B_1 & B_2 & B_3 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$\epsilon_{11} + \epsilon_{22} + \epsilon_{33}$

Let me call this B dilatational part to be say, one-third of, this comes out very easily; B_1 , B_1 comes from that, this one B_1 B_2 B_3 , B_1 B_2 B_3 , B_1 B_2 B_3 , 0 0 0, 0 0 0, 0 0 0. Now, how did I get that? One minute, how did I get that? I got this from the fact that the dilatational part is given by ϵ_{11} plus ϵ_{22} plus ϵ_{33} and similar to what we did for **pressure** by 3, so, ϵ_{11} ϵ_{22} ϵ_{33} consists of sum of B_1 and corresponding to u_1 , B_2 corresponding to u_2 , B_3 corresponding to u_3 , so, B_1 B_2 B_3 , together forms ϵ_{11} ϵ_{22} ϵ_{33} . So, that is why we had that by 3 and so that gives the dilatational part.

This B, let, let us, that, that was covered in the last class or last course, but it was, it is quite simple. How did we get epsilon? You know, just look at how we derived epsilon.

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$$\epsilon_{11} = \frac{\partial u_1}{\partial x_1}$$

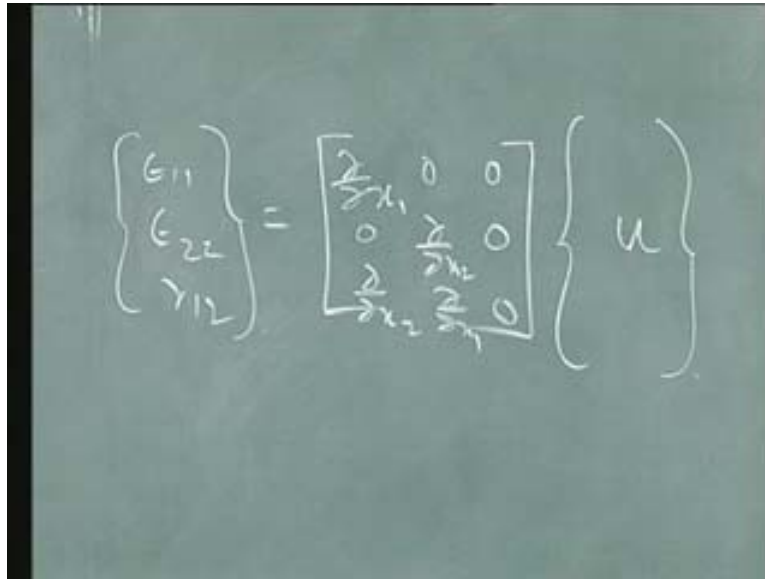
$$\epsilon_{22} = \frac{\partial u_2}{\partial x_2}$$

$$\gamma_{12} = \left(\frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right)$$

So, epsilon, say epsilon₁₁, what is epsilon₁₁? Dow u, yeah, dow u by dow x; so, in other words, dow u by dow x₁ if you want to keep u as u₁ u₂ u₃ and so on. So, you say that dow u₁ by dow x₁; epsilon₂₂ dow u₂ by dow x₂ and so on. So, what is epsilon₁₂? Half of u_i comma j plus u_j comma i. Dow u₁ by dow x₂ plus dow u₂ by dow x₁ and similarly you can write. Usually like, I said in the last course we will not use epsilon, most of the time we will use only gamma₁₂, the engineering shear stress.

Yeah, 1 by 2 will not be there. That is why we have removed that. So, this actually can be written in a matrix form. Let me remove this, we will come back to this in a minute; we will be repeating some of the things.

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The image shows a chalkboard with a handwritten matrix equation. On the left, a column vector is enclosed in large curly braces and contains the terms ϵ_{11} , ϵ_{22} , and γ_{12} . This is followed by an equals sign. In the center, a square matrix is enclosed in large square brackets. The matrix is upper triangular with the following elements: the top-left element is $\frac{\partial}{\partial x_1}$, the top-right element is 0, the middle-left element is 0, the middle-right element is $\frac{\partial}{\partial x_2}$, the bottom-left element is $\frac{\partial}{\partial x_2}$, the bottom-middle element is $\frac{\partial}{\partial x_1}$, and the bottom-right element is 0. To the right of the matrix is another column vector enclosed in large curly braces, containing the letter u .

This will, can be written in the matrix form as epsilon is equal to u, with first entry being ϵ_{11} and second entry being ϵ_{22} . Suppose, I am only looking at a two dimensional case, you can extend it very easily to three dimensional case, so that let me see that is ϵ_{11} , next is ϵ_{22} , say γ_{12} ; only these three I am going to see. Then, this can be written as $\epsilon_{11} = \frac{\partial u}{\partial x_1}$, $\epsilon_{22} = \frac{\partial u}{\partial x_2}$, $\gamma_{12} = \frac{\partial u}{\partial x_2} + \frac{\partial u}{\partial x_1}$. This would be the, for these three. So, I will have actually, because 33 was not there, this was all zero. In other words, you can look it at even as just like that.

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A chalkboard showing a handwritten equation. On the left, a vector of parameters is written as $\begin{Bmatrix} G_{11} \\ G_{22} \\ \gamma_{12} \end{Bmatrix}$. This is followed by an equals sign and a matrix labeled 'S'. The matrix S is a 3x2 matrix with elements $\begin{bmatrix} \frac{2}{\gamma_{11}} & 0 \\ 0 & \frac{2}{\gamma_{12}} \\ \frac{2}{\gamma_{22}} & \frac{2}{\gamma_{21}} \end{bmatrix}$. To the right of the matrix is a vector $\begin{Bmatrix} u \end{Bmatrix}$. Below the matrix S, there is another equals sign followed by an empty bracketed box $\begin{bmatrix} \end{bmatrix}$.

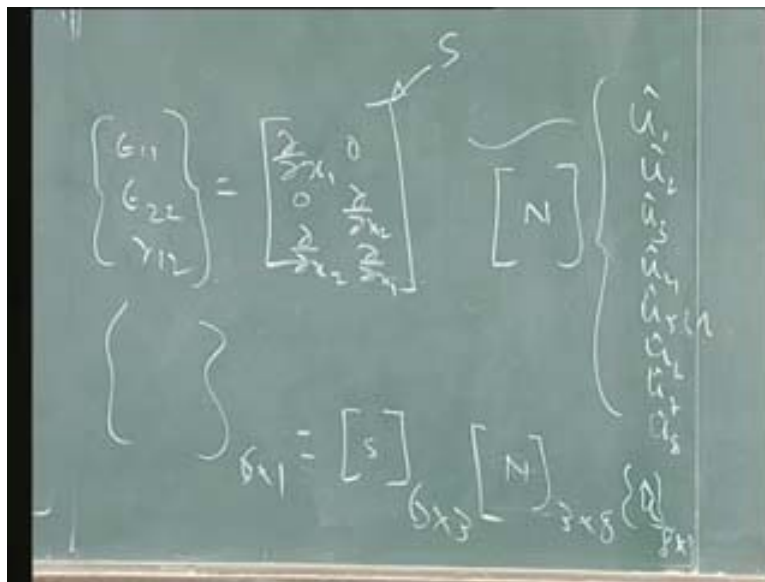
Of course, when I want to have a complete set then, I may, I may have to extend this into 3 rows or rather 3 columns and 6 rows. So, this matrix is what I would call as a S matrix. Note that this u is my continuum u. This is what now is replaced by N into discretized u. That is what I called as u hat.

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A chalkboard showing a handwritten equation. On the left, the same vector of parameters $\begin{Bmatrix} G_{11} \\ G_{22} \\ \gamma_{12} \end{Bmatrix}$ is written. This is followed by an equals sign and the same matrix S as in the previous slide. To the right of the matrix S is a vector $\begin{Bmatrix} \hat{u} \end{Bmatrix}$. Above the vector \hat{u} is a horizontal brace with the letter 'N' written above it. Below the matrix S, there is another equals sign followed by the expression $S N \hat{u}$. Below that, there is a third equals sign followed by the expression $B \hat{u}$.

So S into N into u . Now, let us not worry about the isoparametric; N is our shape function, obviously so, u hat this is, this is what is called as my u hat and $N u$. Now, this multiplied by this $S N$, this can be written as $S N u$. So, this $S N$ is what is called as B , sorry, u hat; that is u hat. So, $S N$ is what is called as B into u hat. So, you can down write B very easily. Is that clear? Yes; this is for, this for one u . So, if you, if I now put down all the nodes correspondingly I will have in u , I will have corresponding to all the nodes, so it will, I will have u_1 .

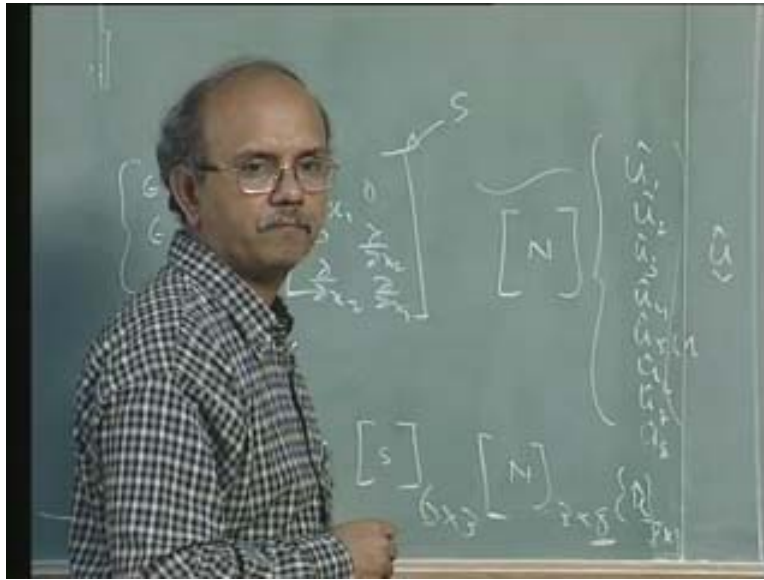
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Let me call this as from u_1 to u_8 to make things clear again. So, $u_1 u_2 u_3 u_4 u_5 u_6 u_7$ and u_8 . So, u will be defined in terms of $N_1 N_2 N_3 N_4$. So, how do you write N now? N will be, now will be 3. See, this will be say, let, let us come back to dimensions of this. Suppose I am looking at 6; so, 6 here. So, 6 by 1 is equal to, S will be 3, 6 by 3. What will be N ? N will be 3 by 8, sorry and u will be 8 by 1, so that 6, 6 by 3 into 3 by 8 6 by 8 into 8 by 1. Now, my B matrix I said is 6 by 8. That is what I wrote this as in a form which is, you know, combining this I wrote it like that, $B_1 B_2$. That is why I said B_1 is a sub matrix. Is that clear?

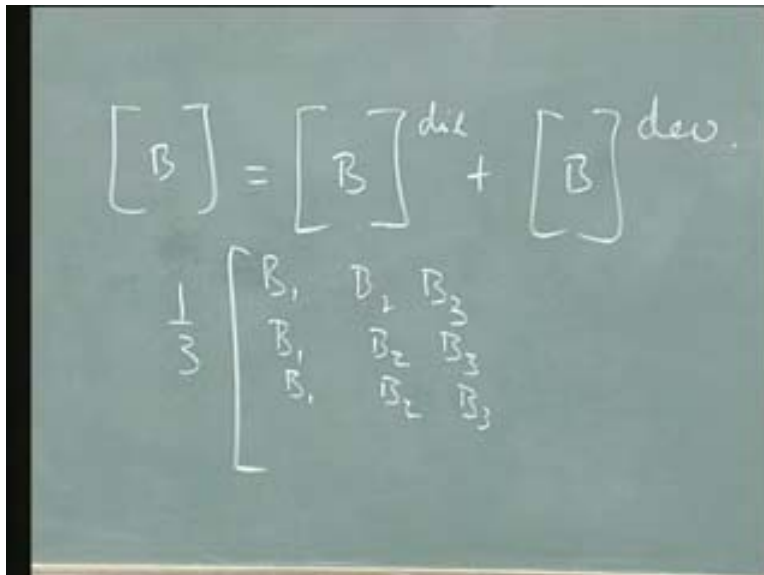
So, symbolically, I mean I am, I am not expanding all this.

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Symbolically I am writing it so that you will, I mean you know here after I will not write u_1 u_2 , I will just say that this is the u vector. That is all. Fine; so please note that the number of degrees of freedom also entered into this B matrix.

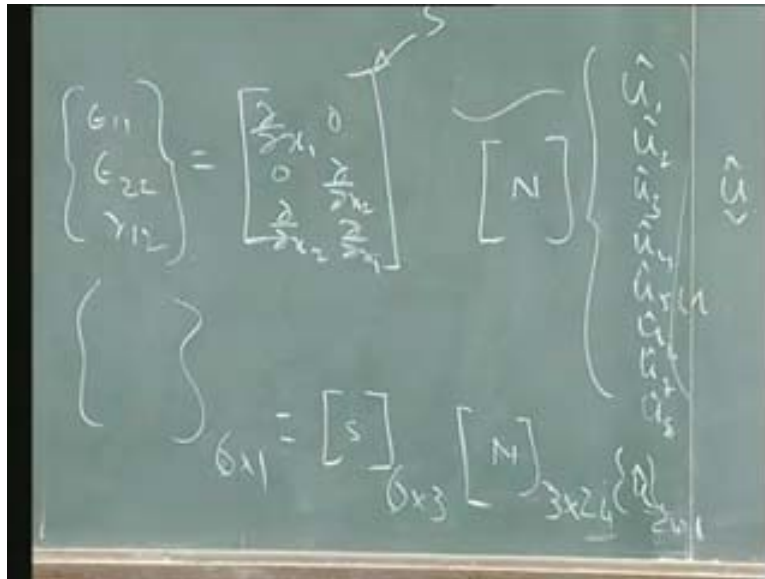
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So, this B matrix is the one which I say can be split up into two parts. One is B dilatational part plus B deviatoric part; B dilatational part and B deviatoric part. The

dilatational part consists of one-third of B's, one-third of say, the first part of B is or in other words, the dilatational part comes from the fact that it is epsilon 11 plus 22 plus 33. So, I can write this say, as $B_1 B_2 B_3$; it is the same $B_1 B_2 B_3$, $B_1 B_2 B_3$. This is for, actually what we did here note that this is for 2D case. If I have to actually, this a small, I think we did not do this correctly; sorry, one minute.

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We did not do that correctly, because we are looking at 3D case. So 3D case is, actually we have extended it to 3D case. So, in 3D case this is equal to 6 by 1 and S is, there are how many rows? 3; so how many, how many columns? 3 columns and 6 rows, so, 6 into 3. Then here it is 3 into, say for example, you are looking at a brick element now. Brick element, 8; 8 into, yes, 8 into 2, number of degrees of freedom, 16. So this should be 3 into 16; so, 16 into 1. Which one? Yeah, 8 into 3, 24 right, not 62, not 2; 3, so, 3 into 24 and 24 into 1, 24 into 1; correct. So, 6 into 3, 3 into 24, 24 into 1, so that now B matrix is actually S into N, which means 6 into 24; 6 into 24 and so each of this B_1 are

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The image shows a chalkboard with handwritten mathematical equations. At the top, the equation is
$$[B] = [B]^{dil} + [B]^{dev}$$
. Below this, the dilatational part is expanded as
$$[B]_a^{dil} = \frac{1}{3} \begin{bmatrix} B_1 & & & \\ & B_2 & & \\ & & B_3 & \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} + \dots$$
 Above the matrix, the text "6x24" is written. The matrix elements are arranged in three columns: the first column contains B_1 , B_1 , B_1 , and three zeros; the second column contains B_2 , B_2 , B_2 , and three zeros; the third column contains B_3 , B_3 , B_3 , and three zeros. The rest of the matrix is empty, and a plus sign is followed by another empty matrix structure.

Yeah, so, that is why now B dilatational one-third is written as B₁ B₂ B₃. These are matrices, now you know what they are? B₁ B₂ B₃, B₁ B₂ B₃, 0 0 0, 0 0 0, 0 0 0 ; one-third. Yes; wait a minute. Let me finish it. So, plus I have a deviatoric part such that the sum of this deviatoric plus dilatational part will give me B. Is that clear?

No; I am splitting this B into a dilatational part in the same fashion that I can split strains, because after all these are strain displacement relationship. So, in the same fashion I can split strains I am splitting B into the dilatational part and deviatoric part. So, the dilatational part is this and deviatoric part is the other one. So, in fact if I am going to write down these two before we write down the equation and substitute this into my K matrix, you will see that I can write that down as my K matrix or K_T matrix, can be written as integral omega say B deviatoric plus B dilatational.

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$$k_T = \int_{\Omega} (B^{dev} + B^{dil})^T D_T d\Omega$$

Diagram of a square element with four nodes (x) and a central node (+).

Vertical list of terms: $\epsilon_1, \epsilon_2, \epsilon_3, \epsilon_4, \epsilon_5, \epsilon_6, \epsilon_7, \epsilon_8, \epsilon_9, \epsilon_{10}, \epsilon_{11}, \epsilon_{12}$

I am going to introduce lot more theory, but quick glance of what we are going to do, transpose D into, same way B is again split B deviatoric plus B dilatational or D_T into d omega. So, wherever I get the dilatational part, I am going to under integrate and deviatoric part is full integration. So, in other words, this $B_1 B_2$ what you see here will be replaced by new B 's called B bars which is the result of under integration. There are various ways of getting B bars we will see that in the next class. One is by under, under integration and you will see that we can get the same formulation by going over to another formulation called mixed formulation. It is possible to go to a formulation called mixed formulation; I will just explain it in a minute and come back to this again. But, I just want to say here that now I am splitting B .

The essence of the story is that I am splitting B . I am splitting this into a deviatoric part and a dilatational part and I am concentrating on the dilatational part and under integrating it and the deviatoric part I am doing full integration. In other words, if I just look at the implementation, what will happen is that in an element I will have 1 2 3 4, 4 Gauss points for a 2D element, correspondingly 2 by 2 by 2 in a 3D element for not, it is not a higher order element, it is a 4 noded element and these 4 nodes or 4 Gauss points

rather are used to integrate the deviatoric part and I will use one Gauss point at the center to integrate the dilatational part. So, this is called as selective reduced integration.

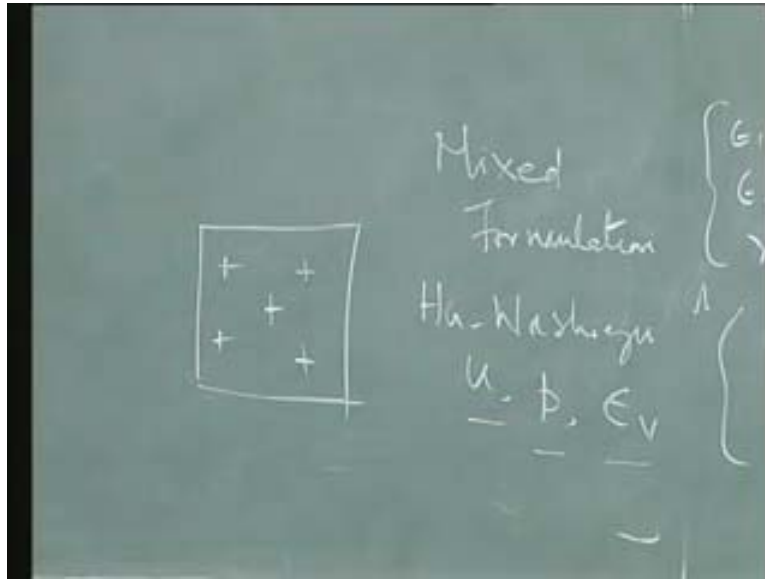
Yes. Yeah, usually you do not consider them, but you take that separately as deviatoric D_T . That is why we would call that as a B -bar. You will see how it comes up in the next part of the derivation in the mixed formulation, but you will, usually you take deviatoric part separately into D_T , dilatational part separately. So, you will have deviatoric D_T deviatoric, dilatational D_T dilatational. These are the two things that you will take them up separately. In other words, it is not very correct to split that. In fact, I should have written this as B consisting of, actually this comes from my P matrix, from the B transpose sigma.

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$$K_r = \int B^T \sigma d\omega + \int (B_{dev} + B_{dil}) \sigma d\omega$$

Actually I should have, I should not have written it like this. Actually we should have written it as B transpose sigma $d\omega$ and from here I should have written this as B deviatoric plus B dilatational sigma $d\omega$ and so on. In other words, you will have only two parts **B deviatoric $d B$ deviatoric plus dilatational** part. So, that is what you under integrate. So, you will have, actually if you really look at it, you will have 5 Gauss points.

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These four are the ones which are used for deviatoric part and the center Gauss point is what is used for the dilatational part. So, you will have, in other words, the history variables are stored in all these 5 Gauss points as well. Now, what is the theoretical background? We will, we will do that in the next class to this. The theoretical background for this comes from what is called as mixed formulation and going over to a variational approach called as Hu-Washizu variational principle.

In the mixed formulation, in the mixed formulation you have instead of displacement alone as a variable, you also have sometimes p as well as volumetric strain ϵ_v . When I have u p and v , ϵ_v as a variable, then we call this as, there are three things, so, these are your three mixed formulations or sometimes you will have only u and stress. These are two point mixed formulation. We will talk about mixed formulation and Hu-Washizu rule or variational formulation in the next class and tie that up and go into the details of this B and how that comes about Hu-Washizu principle in the next class.

Yes, because in the plasticity problem, can we decompose D_T clearly? It is not possible as simple as what we did for an elastic case, because elastic case we had the shear modulus and the bulk modulus clearly defined and quite easy to demarcate it. We will have

problems when we do that in the plastic, in the plasticity part of it. So, let us look at the mixed formulation and then talk about the rest of it.