

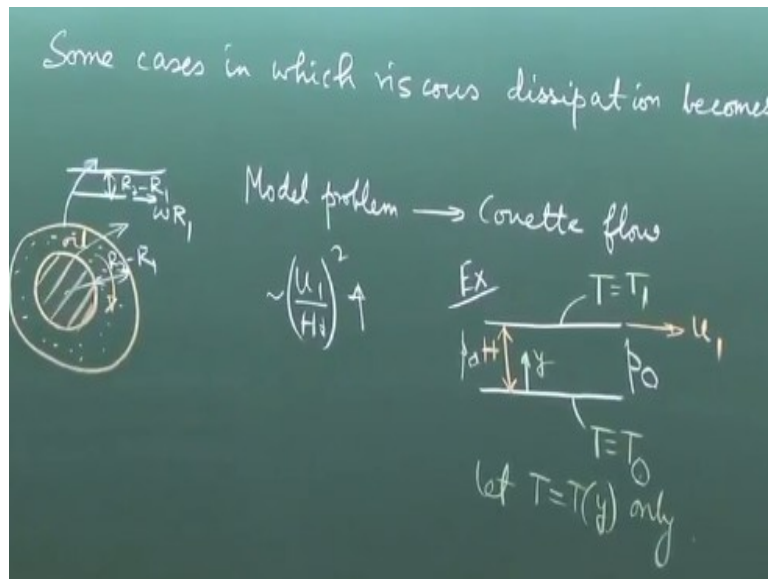
Conduction and Convection Heat Transfer
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Lecture 51
Viscous Dissipation - I

So far, we have discussed the cases in which the viscous dissipation is not important. We have discussed forced convection over surfaces and we have discussed forced convection within a duct or a pipe but in all those cases we have not considered the viscous dissipation to be important. Now in Engineering, there are many practical situations when the viscous dissipation is important.

So, we will now try to analyze one or two example scenarios where the viscous dissipation becomes important.

(Refer Slide Time: 01:12)



So, our agenda now is, some cases in which viscous dissipation becomes important. To analyze this scenario, we will consider a model problem. For every physical scenario, we consider certain representative problems like for internal flow we consider flow through a parallel plate channel. So here we will consider a model problem which is called as Couette flow. So, this kind of fluid flow is often discussed in fluid mechanics in this particular fashion.

Let us say you have two parallel plates. This is a very simplistic representation of the scenario. You have two parallel plates separated by a narrow gap of H . This gap is narrow and there is a relative translational motion between the two plates. That means, let us say the top plate is moving towards the right with a velocity u_1 and the bottom plate is having zero velocity. Okay?

So that means there is a relative velocity, so it does not matter whether this is moving towards the right, left or whether the bottom plate is moving towards right or left. Important message is that there is a relative motion between the two plates. So, two important things. One is a narrow gap between the two plates. The other is there is a relative motion between the two plates.

Now, many times when things are introduced through lectures often we do not ask ourselves a question that why this situation? Why we are studying this problem? When Couette flow is first introduced in say fluid mechanics, have you ever thought that have you ever seen somebody cooling up late in a fluid. I have never seen. I do not know whether you have. I mean, there are situations which are similar to this.

But exactly this situation where you have two parallel plates separated by a narrow gap, someone is cooling one plate relative to the other. I mean, this kind of scenario is not very commonly seen, but then why are we studying this. So, it is not that this kind of scenario is absolutely not there in practice. I mean this kind of scenario is there but I want to mean is that it is not a very common industrial scenario like flow through a pipe is a very common industrial scenario.

Like if you go to any process industry, power industry you will see that there are lots of pipelines and fluid is flowing through a pipe. But in those industries, you will not see that somebody is pulling a plate relative to the other within a fluid. So, what is the motivation. So, I would like to give you a motivation behind this. So, I will give you one fundamental scientific motivation and one practical engineering motivation.

And we will merge those motivations together to see that why such a flow is very important in the context of fluid mechanics and heat transfer. Now as mechanical engineers you must be familiar with bearings. Or at least you have heard of the term bearing. So, if you have a shaft,

at least you know what is a shaft, right? So, if you have a shaft which is rotating and transmitting power, now to support that shaft you require a bearing.

So, let us say this is a shaft which is rotating, say clockwise, anticlockwise whatever, with a particular angular velocity. Now, this may be the rotor but this may be a stator just an outer casing to support the shaft. Now we have actually magnified this figure. This gap is very small. But at the same time this gap is finite. Why this gap is finite because during the rotation of the shaft.

I mean, it will acquire certain eccentricity at certain times and there is always a tendency that the shaft may be in contact with the bearing, then there will be metal to metal contact. So that should be avoided. So in between to avoid that high friction through metal to metal contact a lubricating oil is kept. See, this is practical engineering. You must understand this. Always, the issue would be that, at the end in heat transfer.

We will boil down the situation to mathematical equations, solution of the mathematical equations and all, but we should case a model practical problem towards that direction not that we are just as mathematicians interested to solve those problems arbitrarily. So, you have oil here, separated by this gap. Now this gap is very small. When this gap is very small, then what happens? Then effect of curvature of these two is not important.

Effect of this curvature will be important when this gap is relatively large, but when the gap is relatively small then the effect of curvature is not important. When the effect of curvature is not important we can model this by almost like a flat surface, this by another flat surface separated by, so if this distance is r_2 minus r_1 where r_2 is the radius of the outer and r_1 is the radius of the inner.

Then this is r_2 minus r_1 and the inner one is having a velocity which is what ω multiplied by r_1 , where ω is the angular velocity. So, this is having a translational velocity ω multiplied by r_1 . The upper one is having no velocity and there is a narrow gap. So, you can see that this problem which appears to be hypothetical is actually a very practical engineering problem.

One has to have the mindset towards understanding the problem in this way. Now this is of course the practical way of looking in to this problem, but this problem also has a very fundamental scientific insight because of which this problem needs to be studied carefully. So far you have seen that, can you tell whether this is an internal flow or an external flow? This is an internal flow, because eventually these two plates form a channel. How is the flow driven?

When you have two plates in a channel or two plates forming a channel and there is a fluid which is flowing between the two plates, then how is the flow actuated? In the previous examples we have seen that the flow is actuated by a driving pressure gradient, that is called as a pressure gradient driven flow or pressure driven flow. Here there is not pressure gradient, of course you can create a pressure gradient.

But we will consider a simple Couette flow when this pressure is P_0 and this pressure is also P_0 . That means there is no pressure gradient. But still there is a fluid flow in between the two plates, how? This is driven by shear. So, this is a classical example of a shear driven flow. So just like we have a classical example of a pressure driven flow, this is a classical model problem of a shear driven flow.

And in many physical scenarios shear is a very important dominating mechanism. So, if shear creates a flow, what is the signature of that flow that can be studied by this very simple problem. So, one fundamental scientific motivation is studying shear driven flow. So, in any practical scenarios starting from engineering to biological applications, wherever there is a shear driven flow you can consider this model problem and start with that.

On the other hand, as a mechanical engineer, the problem of lubrication in bearings is very important and for such problems you can first start with the fluid mechanics and then of course heat transfer because, see, in one way this narrow gap, how this narrow gap manifest in analysis of the problem? Of course, here we have discussed that if the gap is narrow then this curvature effect may be neglected.

But that is not all because even if the curvature effect is not negligible you can analyze this problem taking the curvature into account, that is not a very big thing. But the narrow gap means, see, what is the velocity gradient? What is the order of magnitude of the velocity

gradient? Order of magnitude of the velocity gradient is u_1 by H and if you recall the viscous dissipation terms the viscous dissipation terms scale will square of the velocity gradients.

So, your viscous dissipation term will scale with this. So, if this is small then this will be large and then viscous dissipation will have a big role to play. So, what will viscous dissipation do? It will convert the viscous effects or the shear between the work done due to shear between various fluid layers irreversibly into intermolecular form or energy or internal energy that will rise the temperature.

We have shown during our derivations that it will trivially rise the temperature. It is trial heating and not cooling. So just like if you rub your palms you do not expect that your palms will get cooler, right? You expect that these will become hotter. So, the same mechanism works here. And when it becomes hotter than that means that will act like a heat source in this oil. Okay.

And when there is a heat source in the oil, the oil temperature may rise and that might have several practical consequences in the performance of the bearing. So, it is important to understand that what is the temperature distribution within the system. So that is our motivation. So, as we are seeing that in convection when we are interested about the temperature distribution.

We first start with the fluid mechanics analysis because we need to get the velocity field and we will use the velocity field to get the temperature field. So, we will consider this model problem and we will set up the coordinate axis, y axis like this. We will assume a fully developed flow so that x dependence is not there. So, the momentum equation, so it is just like flow in a parallel plate channel. Only thing is that the flow actuation mechanism is different.

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g.d.e.

$$0 = -\frac{dp}{dx} + \mu \frac{d^2 u}{dy^2}$$

$$\frac{dp}{dx} = \mu \frac{d^2 u}{dy^2} = C = \frac{p_{x=L} - p_{x=0}}{L}$$

$$\frac{d^2 u}{dy^2} = 0$$

$$\Rightarrow \frac{du}{dy} = C_1$$

$$\Rightarrow u = C_1 y + C_2$$

So, what is the governing differential equation? So, if you recall, for fully developed flow, $\frac{du}{dx}$ is equal to zero and if there is no penetration at the wall that is there are no holes at the wall, you have v equal to zero also. So, left hand side, $\frac{du}{dx}$ plus $v \frac{du}{dy}$ that term will be zero and because of steady flow the unsteady term will also be zero. So, total acceleration is zero.

So fully developed flow is a flow where all the forces are balanced, so that there is no net acceleration in the flow. So therefore, you will have zero is equal to... So I will write $\frac{dp}{dx}$ instead of $\frac{dp}{dx}$, okay? This is the equation that we had when we were having pressure driven flow through parallel plate channels, that equation is applicable here. So why I have written, I have actually jumped one step.

I mean here in the first step you write partial derivative and you write partial derivative because this is a function of y only and because this is a function of x only, that means each equal to a constant. So, I have jumped all those steps because we have already done that in the previous examples. So, you have $\frac{dp}{dx}$ is equal to $\mu \frac{d^2 u}{dy^2}$, is equal to a constant, right? Now, what is $\frac{dp}{dx}$? Because $\frac{dp}{dx}$ is a constant that means pressure versus x is linear.

So, what is this? This is p at x is equal to L minus p at x equal to zero divided by L , that is $\frac{dp}{dx}$ because $\frac{dp}{dx}$ is a constant, p versus x is linear, so $\frac{dp}{dx}$ is $\frac{\Delta p}{\Delta x}$, right? So, what is this? This is P_0 and this is also P_0 . This is called as simple Couette flow. But it is not necessary that you have a Couette flow where the pressures at the inlet and the outlet are always the same. If not, it is a combined Poiseuille and Couette flow.

So, then you can solve this problem very easily by noting that this is a linear differential equation, right? So, you can consider it as a super position of two problems. In problem one you have only pressure gradient but no shear driven flow. In problem two, no pressure gradient and only shear driven flow and the combination of these two is the solution of the problem.

But just to get the implication of pure shear driven flow we are considering that there is no pressure gradient. So that means what is the value of c , c is equal to zero. So, u is equal to $c_1 y$ plus c_2 .

(Refer Slide Time: 19:27)

bc (1) At $y=0, u=0 \Rightarrow c_2=0$
 (2) At $y=H, u=u_1 \Rightarrow u_1=c_1 H \Rightarrow c_1 = \frac{u_1}{H}$
 $u = u_1 \frac{y}{H}$
 $\frac{du}{dy} = \frac{u_1}{H}$

So, you apply the boundary condition at y is equal to zero, u equal to zero. That means c_2 is equal to zero and boundary condition two at y equal to H and u equal to u_1 . What boundary condition is this?

This is no slip boundary condition, right? So, no slip does not mean zero velocity of the fluid. No slip means zero relative tangential component of velocity between the fluid and the solid boundary and that originates from viscous effects. So, at y equal to H u equal to u_1 , that means u_1 is equal to $c_1 H$, that means c_1 is equal to u_1 by H . So, the velocity profile is u equal to $u_1 y$ by H . So, what is du dy ?

What is the rate of deformation of fluid in a plane like this, that is $\frac{\partial v}{\partial x}$ plus $\frac{\partial u}{\partial y}$. So here there is no v , so it is basically du dy . So, this is the rate of deformation. So, this is the

rate of shear. So, rate of deformation is u_1 by H which is a constant, that means if somebody prescribes a rate of deformation, then you can use that rate of deformation to model this problem because if the rate of deformation is given.

And the gap is given you can find out what is u_1 , so the rate of deformation the value is given. If the value is given, given a particular gap you can find out what is u_1 . So, with that boundary condition you can stimulate a practical situation where the flow is purely driven by shear of a given magnitude, okay? So, this is the magnitude of that shear. So that is why this is a pure shear driven flow. Now next what you will do.

This is the fluid mechanics part of the problem, next we will analyze the heat transfer part of the problem. So, let us say that this is isothermal T is equal to T_0 , is the bottom plate and T equal to T_1 is the top plate. And let us assume that T is a function of y only. Eventually if you have long parallel plates, infinitely long and two plate, top and bottom plates are isothermal.

Then it will come to a situation where temperature ceases to be a function of x , if you have such infinitely long plate. So that is the situation that we are studying. See, all these idealizations are there to simplify the mathematics and bring out the essential physics. So always in mathematical modeling, good physicist what they do is that, see, the actual practical problem is very complex.

But what good physicist try to do is they try to simplify the problem to an extent that the mathematics becomes simple enough to bring out the essential physics and that physics is utilized for design and other analysis. So that is what we are attempting here.

(Refer Slide Time: 24:03)

Energy eq. (steady) $T = T(x)$

$$\rho c_p \left[\frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} \right] = k \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) + \mu \Phi$$

$$k \frac{dT}{dy^2} + \mu \Phi = 0 \quad \left(\frac{du}{dy} \right)^2 = \frac{u^2}{H^2}$$

$$\frac{dT}{dy^2} + \frac{\mu u^2}{k H^2} = 0$$

Now, for the heat transfer part of the problem we need to solve the energy equation. See, we have considered the viscous dissipation term but we will analyze later on whether this term is at all important or not. This term we have kept in the analysis because at least it has the potential of being important because H being small the square of this maybe large. So that gives the motivation of keeping this term in the analysis.

So, this ϕ is the viscous dissipation function which let me write the expression of this in an index notation. This we have derived earlier but let me just write it.