

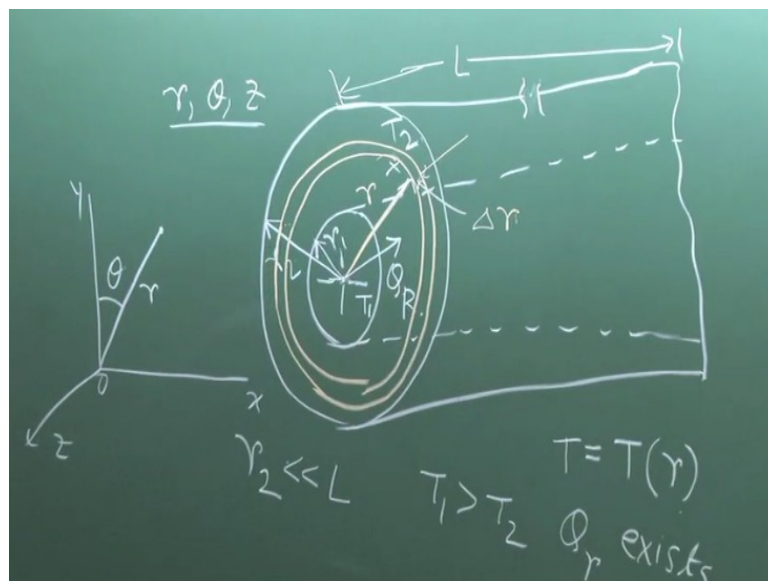
**Conduction and Convection Heat Transfer**  
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**Lecture 14**

Good afternoon to all of you and I welcome you all to the session on Conduction and Convection heat transfer. In last classes, we discussed the one-dimensional steady state heat conduction through plane surfaces. We also discussed the plane surface where the area is cross sectional area, that is area normal to the heat flow is varying, how to deal with the problems and we have solved two interesting problems in the last class.

Now, today we will discuss one-dimensional steady heat conduction in cylindrical geometry. Now, in a plane wall, as we have seen the application where we can use the sufficient coordinate system, but in cylindrical geometry, we have to use cylindrical polar coordinate system. Now the cylindrical geometry comes mostly in case of pipe. When a hot fluid flows through a pipe.

then the heat is being transferred from the fluid by convection to the inner wall of the pipe and then by conduction from inner wall to outer wall of the pipe and in many occasions to reduce the heat loss, we have to provide insulation, insulating material and provide another thickness to the cylindrical pipe, so those are the application of steady one-dimensional heat conduction in cylindrical coordinate system. Let us see that.

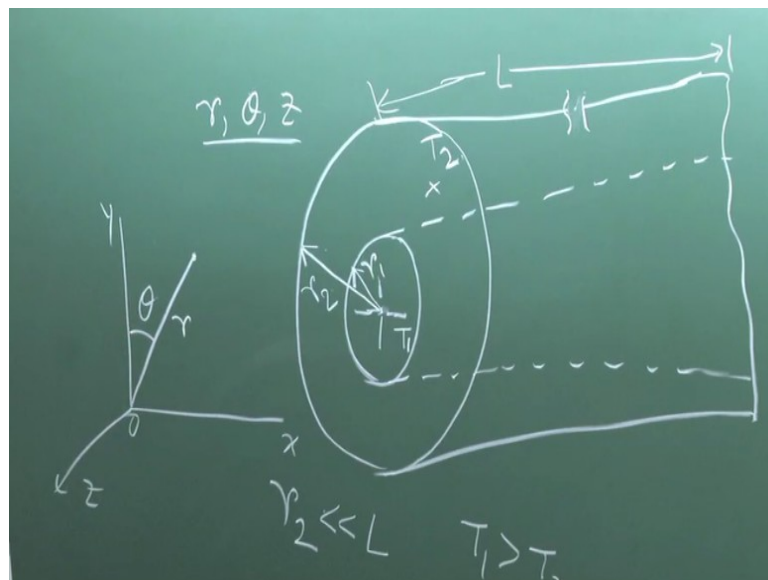
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Let us consider a cylinder like this and the problem is specified like this, which we will be discussing. Let this be the center. The cylinder with inside radius  $r_1$  is cylindrical tube, outside radius  $r_2$ , that means this portion is the solid cylindrical wall type of thing and the length of this cylinder is  $L$  and the problem is like this,  $r_2$  the outside radius of the tube, outer radius is very less than  $L$ , that means length is much more than the radius.

And the inner surface is maintained at a temperature  $T_1$  constant temperature throughout the surface. So, the inner cylindrical surface is maintained at  $T_1$  while the outer surface is maintained at a temperature  $T_2$  and  $T_1$  is greater than  $T_2$ . Now, in this case since it is maintained at a constant temperature, there is no variation in the azimuthal direction, theta and also the temperature is same along the length of the cylinder.

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This can be expressed in terms of cylindrical coordinate system if it is sufficient coordinate the  $r$ , this is the polar azimuthal angle theta and  $z$ ,  $r$ , theta and  $z$ . The coordinate system, in which any point here in the cylindrical wall,  $r_1$  to  $r_2$  can be specified, where  $z$  is the coordinate along the length and  $r$  and theta the radial and azimuthal coordinate.

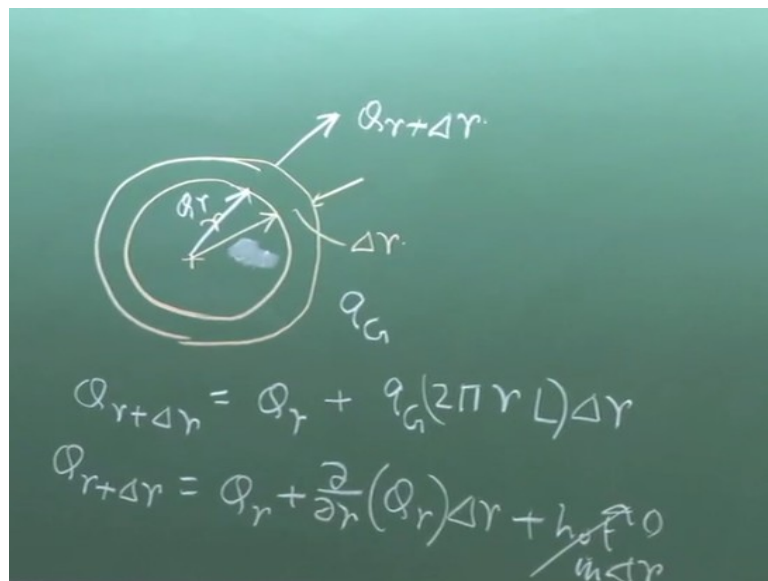
Now, for this type of boundary condition, constant temperature both azimuthally and axially along the length, that is along  $z$  direction and  $r_2$  being less than  $L$ , this assumed one-dimensional heat conduction and we will consider a steady state, that means heat is going in such a way that the temperature is invariant with time, that is the situation when we are going to analyse the problem.

This problem in fact is assumed as state when the boundary condition are steady that we have to understand. The problem becomes generally unsteady continuously when the boundary conditions are unsteady, but the boundary conditions are steady after a transient, the system always attain a steady state, so we consider a steady state.

In this case, T is function of r only and only the heat flux in r direction exists, only  $Q_r$  exists that means there is heat flux in the r direction,  $Q_r$ , now our job is to find out the temperature distribution, heat flux distribution, what is the amount of heat flux across any section, all these, which we did for plane surface. The analysis is exactly the same as we did for plane surface, only difference is the mathematical state, how?

We proceed with that the x and element, annular element at an orbit value radius r. We take an annular element like this, a cylindrical ring at an orbit value radius r with the thickness delta r.

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For clarity, I write here the annular ring. This is at an orbit value radius r, with this one delta r, then our same analysis what we do is that if  $Q_r$  is the heat flux incident on this inner area, then this is the dimension, I am sorry. This  $Q_r$  is the heat and heat going out is  $Q_r$  plus delta r. The equation is very simple. If we consider a heat generation per unit volume  $q_G$  within the cylindrical wall with extent from  $r_1$  to  $r_2$ .

Then for this elemental ring, I can write  $Q_r$  plus delta r at steady state is  $Q_r$  plus  $q_G$  times volume of this element. The volume of the element is the cross-sectional area is twice pi r

into length where  $L$  is the length, that means it is a cylindrical ring, length is perpendicular to the direction of the board. This is the area times  $\Delta r$ , that means this is the total thermal energy generation, so  $Q_r$  plus  $\Delta r$  going out.

At steady state, there is no other alternative, this is same equation, which we wrote for plane area and I have always told that, my suggestion that for steady one-dimensional heat conduction is greater to derive the equation from fundamental. This is okay. Now, if you write series expansion  $Q_r$  plus  $\Delta r$  is  $Q_r$  plus  $\frac{dQ_r}{dr} \Delta r$  plus higher order term, which mean  $\Delta r$ , which we neglect, higher order term in  $\Delta r$ , because  $\Delta r$  is very small.

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$$\frac{\partial}{\partial r}(Q_r)\Delta r = q_G(2\pi r L)\Delta r$$

$$Q_r = -K A_r \frac{dT}{dr}$$

$$\frac{d}{dr}\left(K A_r \frac{dT}{dr}\right) + q_G A_r = 0$$

$$A_r = 2\pi r L$$

So, if we write this,  $Q_r$  plus then, we can write  $\frac{dQ_r}{dr} \Delta r$  is equal to  $q_G$  into twice  $\pi r L \Delta r$ . So, this equation is the same as we did for plane wall. Now, we have to replace  $Q_r$  by food-air conduction equation. What is  $Q_r$ ? At any radius, at an orbit steady location  $r$ ,  $Q_r$  is minus  $K$  into the cross-sectional area at that  $r$  into  $dT/dr$ , so therefore if we put that here, now here also twice  $\pi r L$ , actually it is the cross-sectional area.

I can write in terms of the cross-sectional area,  $A_r$ . This is nothing but the cross-sectional area, twice  $\pi r L$ , so that we can write. Now this  $\frac{dQ_r}{dr}$ , I can write as  $d/dr$  because it is a one-dimensional heat conduction  $Q_r$  exists and it is a function of  $r$  only,  $d/dr$  of, take this here  $A$  this side,  $A_r dT/dr$  plus  $q_G A_r$  is equal to 0.  $\Delta r$  gets cancelled. My sole intention to write in this fashion, again this right to  $A_r$ .

So, I wrote earlier twice  $\pi r$  is very simple, that this is the same equation, which we have used in the plane surface with varying area, that  $d/dx$  in  $x$  and  $y$  coordinate system when  $T$  is a function of  $x$  in sufficient coordinate system  $KAx \, dT/dx + qGAx = 0$ , but the difference is that in plane area, this  $A$  may be constant under substance equation. It will come out and equation becomes  $d/dx \, K \, dT/dx + qG = 0$ .

In some cases, it is varying and the integration depends upon the type of variation of  $A$  with  $x$  even if  $K$  remains constant, but here  $A$  is fixed.  $A$  is  $2\pi r L$ .

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$$\frac{d}{dr} \left( k r \frac{dT}{dr} \right) + q_G r = 0$$

$$\rho C \frac{\partial T}{\partial t} = \frac{\partial}{\partial x} \left( k \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial y} \left( k \frac{\partial T}{\partial y} \right) + \frac{\partial}{\partial z} \left( k \frac{\partial T}{\partial z} \right) + q_G$$

$$\rho C \frac{\partial T}{\partial t} = \nabla \cdot (k \nabla T) + q_G$$

$$\nabla T = \vec{i} \frac{\partial T}{\partial x} + \vec{j} \frac{\partial T}{\partial y} + \vec{k} \frac{\partial T}{\partial z}$$

$$\nabla = \vec{i} \frac{\partial}{\partial x} + \vec{j} \frac{\partial}{\partial y} + \vec{k} \frac{\partial}{\partial z}$$

So, therefore, if you just substitute this, then you will get  $d/dr$  and if you take  $2\pi L$  to be constant and  $L$  cannot be 0, so you can cancel that,  $d/dr$  of  $Kr \, dT/dr + qG$  into  $r$  equal to 0 and this is our basic equation for temperature distribution in differential form. If  $K$  is constant, then  $K$  will come out. It may be going here by  $K$ . Now this same equation, I again tell you can be derived from the general conduction equation.

The way I told the plane surfaces also, that we can generate the same thing by it is greater always to maintain energy balance for a steady one dimensional and derive your own equation, but from the general equation also, we can come to this. In a cylindrical coordinate system, the same thing will appear, just for your interest I tell you, if you recollect the general energy equation, which was derived in the class in terms of Cartesian  $A$ .

Cartesian frame of reference  $\rho C \, \partial T / \partial t$  is equal to  $\partial / \partial x$  of  $K \, \partial T / \partial x$  plus if you remember  $\partial / \partial y$  of  $K \, \partial T / \partial y$  plus  $\partial / \partial z$  of  $K \, \partial T / \partial z$  plus  $qG$ . So, this was the

general energy equation where  $T$  was defined as a specific heat, which if you find in such a way that mass times a specific heat times the rate of change of temperature equals to the rate of change of internal energy of the material

And in a solid, it is the change of the internal energy because there is no flow, no other energy process to the boundary, so therefore it is the change. Within the control volume, it is the change of the internal energy, so this is the left-hand term. So, therefore, this equation, you know, you are familiar with, general energy equation or heat conduction equation, but now the question is that this is in frame of Cartesian coordinate system,  $x, y, z$ .

How do I get the same equation in cylindrical coordinate system or cylindrical polar coordinate system. There are two ways of doing that. The most simple way of doing this, that is just transform this equation in vector form. You write this in general vector form  $\text{del } T / \text{del } t$  is equal to what is these  $\text{del} / \text{del } x$  of  $K \text{ del } T / \text{del } x, \text{ del } y$ , this is divergent of  $K \text{ grad } T$  plus  $qG$ .

And you know in Cartesian coordinate system,  $\text{grad } T$  is what,  $i, j, k$  being the unit vector along  $x, y$ , and  $z$  direction,  $i \text{ del } T / \text{del } x$  plus  $j \text{ del } T / \text{del } y$  plus  $k \text{ del } T / \text{del } z$  and divergent is a vector operated, which  $i \text{ del} / \text{del } x$  plus  $j \text{ del} / \text{del } y$  plus  $k \text{ del} / \text{del } z$ , this is again a school level thing and if you make this scalar product, which the operator and the exact  $K \text{ grad } T$ , so if you convert this.

Then your job will be only to expand this term in different coordinate system, not only cylindrical, even the spherical polar coordinate system, what is the expression of divergent  $K \text{ grad}$ . This is only the spatial derivation, which change with respect to the coordinate system. That is all.

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$$\begin{aligned}
 & \rho c \frac{\partial T}{\partial t} = \frac{1}{r} \frac{\partial}{\partial r} \left( k r \frac{\partial T}{\partial r} \right) + \frac{1}{r^2} \frac{\partial}{\partial \theta} \left( k \frac{\partial T}{\partial \theta} \right) \\
 & + \frac{\partial}{\partial z} \left( k \frac{\partial T}{\partial z} \right) + q_c
 \end{aligned}$$

So, in a cylindrical polar coordinate system,  $r$   $\theta$   $z$ . If we write this divergent  $K \text{ grad } T$ , then it will be like this,  $\rho c \frac{\partial T}{\partial t}$  is equal to  $\frac{1}{r} \frac{\partial}{\partial r} \left( k r \frac{\partial T}{\partial r} \right) + \frac{1}{r^2} \frac{\partial}{\partial \theta} \left( k \frac{\partial T}{\partial \theta} \right) + \frac{\partial}{\partial z} \left( k \frac{\partial T}{\partial z} \right) + q_c$ . That means, in a cylindrical polar coordinate system, if we define a cylindrical polar coordinate system like this, in x-y plane.

If I define this the point  $r$  and its azimuth of this and this is the location in three-dimensional cylindrical polar coordinate system, from the origin by the radial coordinate arc, azimuthal coordinate and the axial  $z$  coordinate. So, in that  $r$   $\theta$   $z$  if you expand this divergent  $K \text{ grad } T$  you get this. That means this is the counter part of this and here, for one dimensional definitely you just ignore this and steady state, steady state one dimensionally  $\frac{1}{r} \frac{\partial}{\partial r} \left( k r \frac{\partial T}{\partial r} \right) + \frac{\partial}{\partial z} \left( k \frac{\partial T}{\partial z} \right) + q_c$ , that means if you take out this becomes the same expression.

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$$\frac{1}{r} \frac{d}{dr} \left( k r \frac{dT}{dr} \right) + q_G = 0$$

That means this equation becomes  $\frac{1}{r} \frac{d}{dr}$ , you write  $\frac{d}{dr}$ , that means from the general heat conduction equation we can also get the same expression that is  $\frac{1}{r} \frac{d}{dr}$  of  $Kr \frac{dT}{dr}$  plus  $q_G$  is 0, so this is my one-dimensional steady heat conduction in a cylindrical wall. It is as simple as that. Another way of deriving this equation is the same. This is the most simple and intelligent way.

Another way of deriving this equation is the same the way you derived the heat conduction equation in Cartesian coordination. What you did? You took an element or controlled volume in the conducting medium whose surfaces are parallel to the coordinate planes or edges are same thing parallel to the coordinate axis and which becomes a parallel with pipe  $x$  for a Cartesian coordinate system.

Similarly, you have consider an element of controlled volume whose planes are parallel to cylindrical coordinate system. That means parallel to  $r$  theta,  $r$   $z$ ,  $z$  theta plane and then recognize the heat flux flowing across this plane and take a balance of the total heat flux coming into the control volume, which is change in energy, the way it has been done for Cartesian coordinate system.

And it is a routine matter and it is done in any book you can see that it is not being done in the class, but most easy way is to convert this into cylindrical polar coordinate system by changing, expanding the divergent  $K \text{ grad } T$  in the respective cylindrical coordinate. Now, after this my job becomes much more simple mathematics at school level to integrate this equation and before that, I consider another simple case that  $K$  is constant.



We consider that K is constant. Now when K is constant, this expression comes.

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$$\frac{1}{r} \frac{d}{dr} \left( r \frac{dT}{dr} \right) + \frac{q_G}{K} = 0$$

$$\frac{d}{dr} \left( r \frac{dT}{dr} \right) = 0$$

$$T = C_1 \ln r + C_2$$

at  $r = r_1$ ,  $T = T_1$   
 $r = r_2$ ,  $T = T_2$

1 by r d/dr of r dT/dr plus K is constant, so if you take K, plus qG by K is 0. Now next you consider without heat generation, that means q 0, that means without heat generation steady state constant thermal conductivity, the expression is 1 by r d/dr of r dT/dr, which means d/dr. Now this becomes so simple that it is a mental problem without paper, we can solve this. The solution of this is T is equal to some constant ln r plus C2.

r dT/dr is C1, it is constant and then again integrate C1 ln r dr/r plus C2, okay. What are the boundary conditions? Boundary conditions are this and there are two constants, two boundary conditions also given in the problem at r is equal to r1 and T is equal to T1 and r is equal to r2 and T is equal to T2.

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$$\frac{T_1 - T}{T_1 - T_2} = \frac{\ln\left(\frac{r}{r_1}\right)}{\ln\left(\frac{r_2}{r_1}\right)}$$

If you use these two boundary conditions, you get a profile temperature distribution  $T_1$  minus  $T$  divided by  $T_1$  minus  $T_2$ .  $T_1$  is the inner surface temperature, which is higher than  $T_2$  is equal to  $\ln r$  by  $r_1$  divided by  $\ln r_2$  by  $r_1$ . It is very simple that you substitute this to find out  $T_1$  and  $T_2$ .  $T$  is equal to  $T_1 \ln r_1$  plus  $C_2$  and  $T_2$  is  $T_1 \ln r_2$  plus  $C_2$  and you can find out  $T_1$ ,  $T_2$  and finally you get this as the logarithmic distribution.

This is the temperature variation with  $r$ . It cannot be linear, because the area is parallel. In a way that it is directly proportional to the  $r$ . Now, if you find out the  $Q_r$  at any section, at any arbitrary location that already we wrote the expression, minus  $K$  into  $ar$ , that is twice  $\pi rL$  into  $dT/dr$ .

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$$\frac{dT}{dr} = -\frac{(T_1 - T_2)}{\ln\left(\frac{r_2}{r_1}\right)} \cdot \frac{1}{r}$$

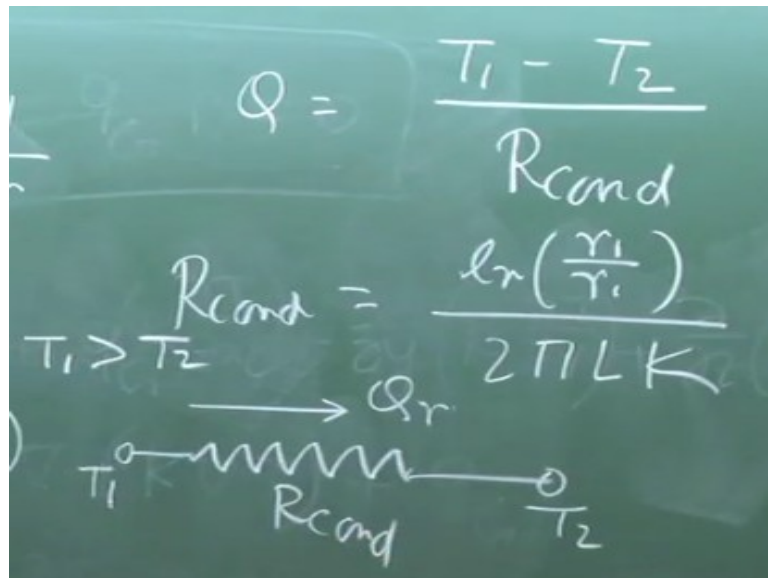
$$Q_r = \frac{2\pi KL}{\ln\left(\frac{r_2}{r_1}\right)} (T_1 - T_2)$$

$$Q_r = \frac{(T_1 - T_2)}{\ln\left(\frac{r_2}{r_1}\right) / 2\pi KL}$$

What is  $dT/dr$ ?  $dT/dr$  is minus  $dT/dr$  minus  $T_1$  minus  $T_2$  divided by this is constant  $\ln r_2$  by  $r_1$  and this differentiation  $r_1$  by  $r$  and again  $1$  by  $r_1$ , that means  $1$  by  $r$  and if you multiply here that  $r$  cancels. That means  $Q_r$  becomes, and minus minus is plus, so therefore twice  $\pi$   $KL$  divided by  $\ln r_2$  by  $r_1$  into  $T_1$  minus  $T_2$ . So, it is found that with this temperature distribution, heat flux at any radial location  $r$  is independent of  $r$ .

Obviously because I have found the temperature distribution from the steady state constant without heat generation,  $Q_r$  plus  $\Delta r$  has to be equal to  $Q_r$ , we have made  $Q_g = 0$ . So, therefore it has to be like this. It is proved that it is okay. So, this is the heat flux and this heat flux in the similar way can be expressed as  $T_1$  minus  $T_2$  in the numerator,  $Q_r$ , divided by  $\ln r_2$  by  $r_1$  divided by twice  $\pi$   $KL$  and this acts as the conduction resistance.

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So, therefore if I write  $Q$  is equal to  $T_1$  minus  $T_2$  divided by conduction resistance. This conduction resistance is equal to  $\ln r_2$  by  $r_1$  divided by twice  $\pi$   $LK$ , that means this can be represented by electrical analogous circuit like this, that this is the  $Q_r$  radial direction, this is  $T_1$ , this is  $T_2$ , 2 potential  $T_1$  greater than  $T_2$ ,  $T_1$  greater than  $T_2$  and this is the  $R$  conduction, which is in case of plane surface it was  $L$  by  $K$ .

In case, cylindrical wall it is  $\ln$ , or  $Q$  by  $r_1$  or what nomenclature you use or  $O$  by  $r_i$  from outer radius to inner radius, that means  $\ln$  times the ratio, that means  $\ln$  of the ratio of the  $Q$  radius and then,  $r_2$  by  $r_1$ , twice  $\pi$   $LK$ , okay. Now this thing can be also be deduced in a very simpler way, how? The same thing  $Q$  is twice  $\pi$   $KL$ ,  $T_1$  minus  $T_2$  divided by  $\ln r_2$  by  $r_1$  twice  $\pi$   $KL$ .

I am telling you another method, which you see often in a book, because this is a very generalised approach to find out the temperature distribution and like this, but another method is sometimes used.

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$$Q_r = -K 2\pi r L \frac{dT}{dr}$$

$$\frac{Q}{2\pi L K} = r \frac{dT}{dr}$$

That has any radial location  $Q_r$ , I know the temperature distribution is written like that, heat flux is  $K$  into twice  $\pi r L dT/dx$ . There is no assumption,  $Q_r$  is minus  $K 2 \pi r L dT/dx$ , this is an analog of heat conduction. Now if I neglect heat generation, steady state, we did so many things, but it is a quick approach, that  $Q_r$ , that is, across any cross section, at any  $R$  is constant, that means  $Q_r$  itself is constant.

Then you can write as  $Q$  itself. That means I can write  $Q$ , and I can make like this twice  $\pi LK$  equals to,  $\pi R$ ,  $\pi LK$  equals to  $R dT/dr$ , and next we can write  $Q$  by  $2 \pi LK dr$  by  $r$  is  $dT$ .

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$$\frac{Q}{2\pi r L K} = r \frac{dT}{dr}$$

$$\int_{r_1}^{r_2} \frac{Q}{2\pi L K} \frac{dr}{r} = \int_{T_1}^{T_2} dT$$

Now if I integrate this from  $r_1$  to  $r_2$ , inner to outer, and a variability  $T_1$  to  $T_2$ , and with this consideration, that  $Q$  is constant, which is not varying in the radial direction, because for a steady state heat transfer without heat generation same heat has to pass through each and every perception. In the similar way, I told for plane wall also it can be done like that, in many books it is done like that.

If you integrate this taking this out, this will be simply  $T_2$  minus  $T_1$ , you straight get the equation.  $Q$  is equal to  $Qr$  or simply  $Q$  whatever you call this now you can think of as  $Q$ , so  $Q$  is equal to that heat flux already in the radial direction there is no point of writing in the radial direction  $Qr$ ,  $Q$  is equal to this, automatically you get this.

Now the question comes here, by this I arrived an expression of heat transfer in terms of the terminal temperature difference, but where is the temperature distribution, very good. Then you can take the arbitrary  $r$  not the  $r_2$  the outer radius, then you take the  $T$ , then you get an equation  $Q$ , heat flux in terms of  $T_1$  minus  $T$ , and here it is  $\ln r$  by  $r_1$ , becomes a mental problem. And divide one by another, you get the expression like this.

So, this is the most easy way, also to find out the temperature distribution. That you write the heat flux, connect from one to two integrate, you get the heat flux in terms of the total potential difference, or you take any arbitrary location  $r$ , which a temperature  $T$  there. Then you get an expression  $Q$  in terms of  $T_1$  minus the temperature  $T$ , where this will be  $r$  instead of  $r_2$ , and divide one by other you can get.

This is the easiest way of finding out that, but always it is better to derive an equation under certain special or complicated cases, you must have the practice of generating the basic equation from an energy value for small element, of the heat conducting medium, which is always good. So, that you arrive at and solve the temperature distribution and then write the periodic conduction equation to get that thing.

And this is one of the easiest way most of the book write that taking  $Q$  to the constant, that of all section to get this.