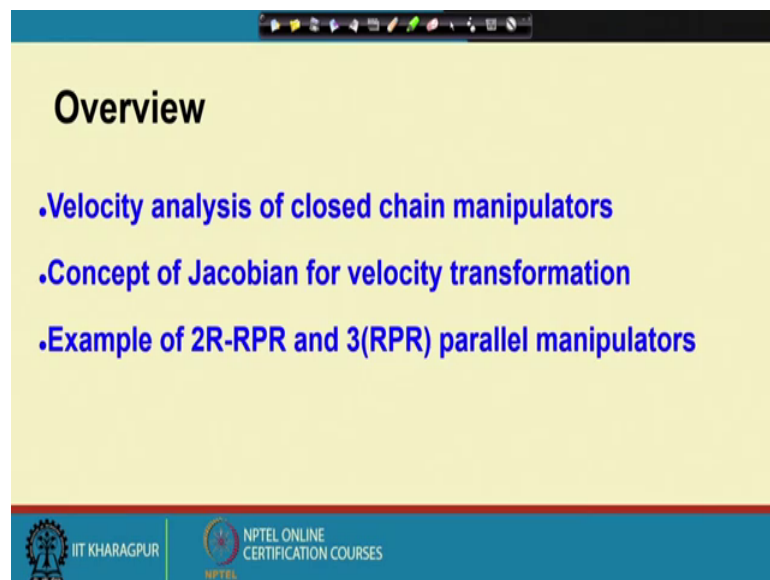


Kinematics of Mechanisms and Machines
Prof. Anirvan Dasgupta
Department of Mechanical Engineering
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Lecture – 30
Robot Velocity Analysis – III

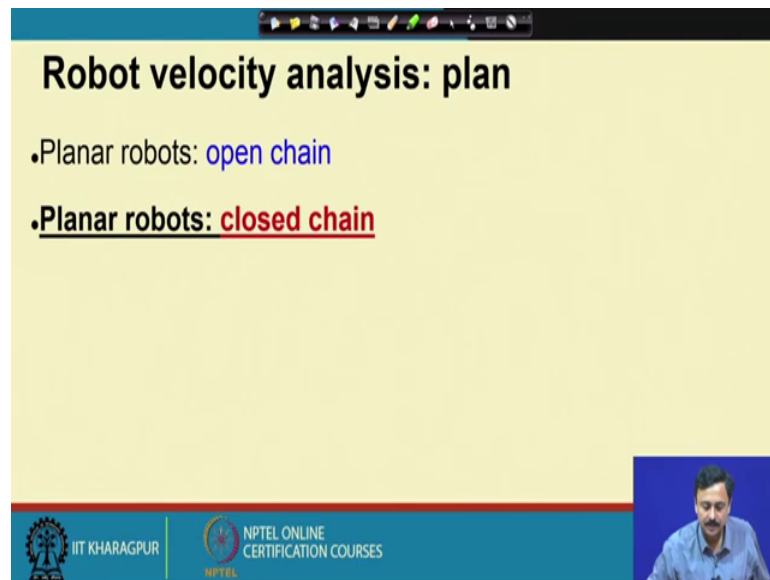
We have been discussing about the Velocity Analysis of Robot manipulators. We have already discussed open chain robot manipulators, we have looked at the Jacobian; it's a interpretation and singularity of the Jacobian. We have looked at the consequences of singularity of the Jacobian.

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In this lecture we are going to embark upon discussions on closed chain manipulators. So, we are going to look at velocity analysis of closed chain manipulators and I will take 2 examples the 2 R-RPR and 3 RPR parallel manipulators; these are planar manipulators. We have already looked at the displacement analysis of both these parallel manipulator chains.

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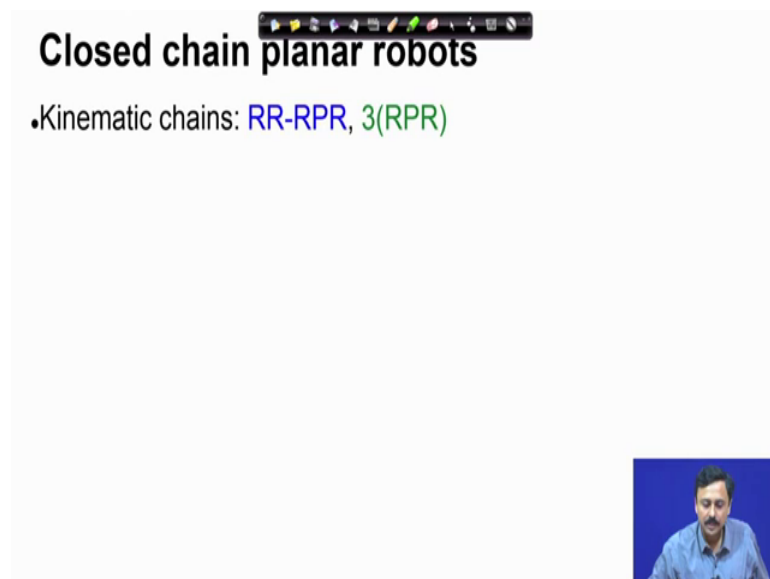
Robot velocity analysis: plan

- Planar robots: **open chain**
- Planar robots: **closed chain**

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So, in this lecture we are going to look at closed chain planar manipulators.

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Closed chain planar robots

- Kinematic chains: **RR-RPR**, **3(RPR)**

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Going to discuss this RR-RPR and 3 RPR chain.

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Closed chain RR-RPR robot: velocity analysis

- Given configuration (θ, s_4)
- Determine relation between $(\dot{\theta}, \dot{s}_4)$ and (\dot{x}_E, \dot{y}_E)

?

So, let us begin with the RR RPR planar closed chain robot. So, in this manipulator the configuration is specified by this angle theta and the extension of this prismatic pair which I have called s 4. So, theta and s 4 they specify the configuration of the manipulator; now the problem is to determine the relation between their velocities. So, we have theta dot and s 4 dot as the joint velocities or the actuator velocities and x E dot and y E dot as the end effector velocities; the velocity vector, so we would like to find out a relation between them.

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$$x_E = l_2 \cos \theta + (l_3 + d) \cos \phi$$

$$y_E = l_2 \sin \theta + (l_3 + d) \sin \phi$$

$$(x_E - l_2 \cos \theta)^2 + (y_E - l_2 \sin \theta)^2 = (l_3 + d)^2$$

where

$$\Rightarrow \underline{A} \sin \theta + \underline{B} \cos \theta = \underline{C}$$

$$A = \underline{y_E}, \quad B = \underline{x_E}$$

$$C = \frac{x_E^2 + y_E^2 + l_2^2 - (l_3 + d)^2}{2l_2}$$

Here I have written out the displacement relations which we have already discussed in a previous lecture. The angle ϕ ; the angle ϕ is this configuration, this orientation of the end effector; we have also seen in the displacement analysis that given θ and s_4 we can find out x_E , y_E and ϕ .

So, we can find out everything; now if you eliminate ϕ between these 2 relations, we ultimately arrive at relation involving x_E , y_E and θ . Here this A , B and C they are functions of x_E and y_E ; you can interpret this as given x_E and y_E ; its a relation for θ its an equation from where we can solve for θ .

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where $\frac{d}{dt} [A \sin \theta + B \cos \theta = C]$

where $A = y_E, B = x_E$

$$C = \frac{x_E^2 + y_E^2 + l_2^2 - (l_3 + d)^2}{2l_2}$$

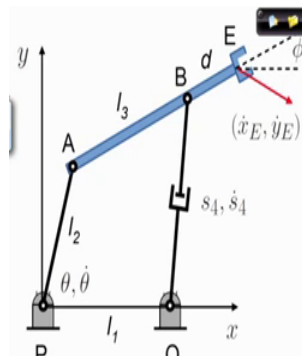
$$\dot{A} \sin \theta + A \dot{\theta} \cos \theta + \dot{B} \cos \theta - B \dot{\theta} \sin \theta = \dot{C}$$

$$\dot{y}_E \sin \theta + y_E \dot{\theta} \cos \theta + \dot{x}_E \cos \theta - x_E \dot{\theta} \sin \theta = \frac{1}{2l_2} (2x_E \dot{x}_E + 2y_E \dot{y}_E)$$

So, given x_E and y_E we can solve for θ . Now what we are going to do with this relation is that we are going to differentiate this with respect to time. So, this relation we are going to differentiate with respect to time. Then what we have is $\dot{A} \sin \theta + A \dot{\theta} \cos \theta + \dot{B} \cos \theta - B \dot{\theta} \sin \theta = \dot{C}$. $\dot{A} \sin \theta$ represents $\sin \theta$ plus $A \dot{\theta} \cos \theta$ plus $B \dot{\theta} \cos \theta$ minus $B \dot{\theta} \sin \theta$ is equal to \dot{C} .

Now, if you use these expressions of A , B and C ; then you have these relation this is equal to, now I can collect terms of $x_E \dot{x}_E$, $y_E \dot{y}_E$ and $\dot{\theta}$. From here I can collect terms of $x_E \dot{x}_E$, $y_E \dot{y}_E$ and $\dot{\theta}$ and write in a simplified form which I will show you directly.

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$$A \sin \theta + B \cos \theta = C$$

where

$$A = y_E, \quad B = x_E$$

$$C = \frac{x_E^2 + y_E^2 + l_2^2 - (l_3 + d)^2}{2l_2}$$

Time differentiating both sides

$$P_1 \dot{x}_E + Q_1 \dot{y}_E = R_1 \dot{\theta}$$

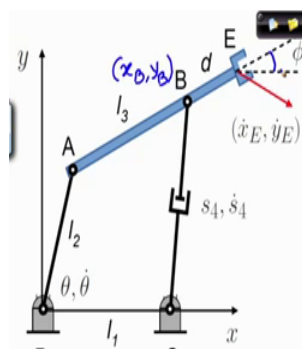
where

$$P_1 = \left(\cos \theta - \frac{x_E}{l_2} \right), \quad Q_1 = \left(\sin \theta - \frac{y_E}{l_2} \right)$$

$$R_1 = x_E \sin \theta - y_E \cos \theta$$

So, I obtain a relation like this by collecting terms of \dot{x}_E , \dot{y}_E and $\dot{\theta}$. So, here P_1 is this expression Q_1 and R_1 which you can very easily derive by simplifying the steps that I have shown you. So, this equation relates \dot{x}_E , \dot{y}_E and $\dot{\theta}$, but this is not enough I need one more relation, I must also relate \dot{s}_4 to \dot{x}_E and \dot{y}_E . So, let us move on. So, here I have one relation next I must bring in \dot{s}_4 .

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$$(x_B - l_1)^2 + y_B^2 = s_4^2$$

where

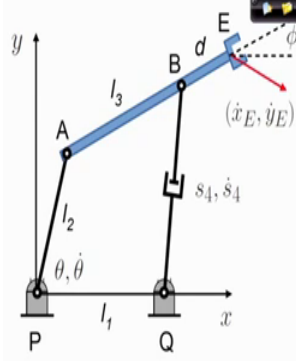
$$x_B = x_E - d \cos \phi, \quad y_B = y_E - d \sin \phi$$

$$\Rightarrow s_4 = \sqrt{(x_E - d \cos \phi - l_1)^2 + (y_E - d \sin \phi)^2}$$

So, that is what we are going to look at. Now from the previous expressions; so, if you look at the coordinates of point B. So, coordinates of point B; x_B and y_B and the

distance between B and Q is nothing, but s_4 . So, therefore, the distance square this is the distance square. So, this is QB square is equal to s_4 square and x_B can be related to x_E and ϕ ; similarly y_B can be related to y_E and ϕ . So, if you use these expressions s_4 can be related to x_E y_E , but you also have ϕ . Now, ϕ remember is the orientation angle of the end effector.

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$$(x_B - l_1)^2 + y_B^2 = s_4^2$$

where

$$x_B = x_E - d \cos \phi, \quad y_B = y_E - d \sin \phi$$

$$\Rightarrow s_4 = \sqrt{(x_E - d \cos \phi - l_1)^2 + (y_E - d \sin \phi)^2}$$

Time differentiating both sides

$$\dot{s}_4 = P_2 \dot{x}_E + Q_2 \dot{y}_E + R_2 \dot{\phi}$$

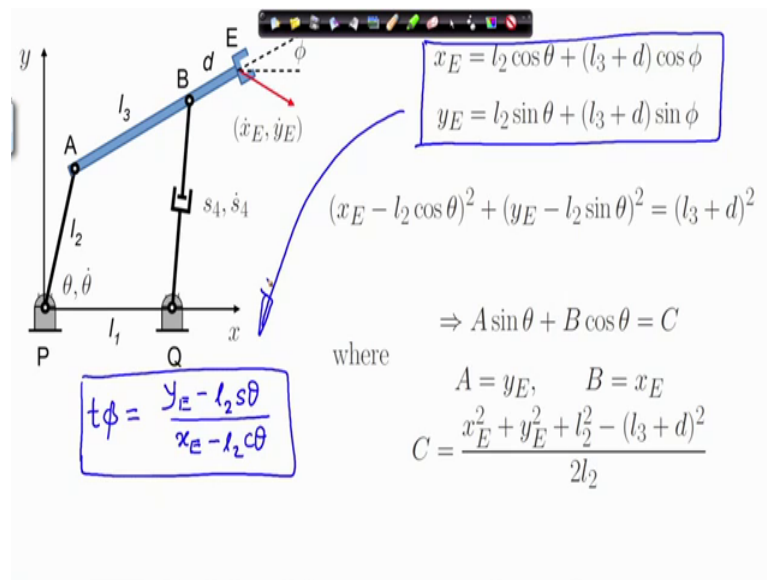
$$\tan \phi = \frac{y_E - l_2 \sin \theta}{x_E - l_2 \cos \theta}$$

Now, if you differentiate with respect to time both sides of this equation, then you can imagine that you will have derivatives of x_E , you will have derivatives of ϕ and you will have derivatives of y_E . And the coefficients corresponding coefficients are P_2 Q_2 and R_2 which can be very easily found by differentiating this.

And this is going to be little complicated, but it is otherwise straightforward; I mean there will be a little bit of algebra involved. So, the steps are straightforward if we just differentiate the expression of s_4 with respect to time; now we have involved ϕ . So, let us see how we can find out ϕ and we have also involved remember this cosine ϕ and sine ϕ ; we also have this cosine ϕ and sine ϕ as well as ϕ dot.

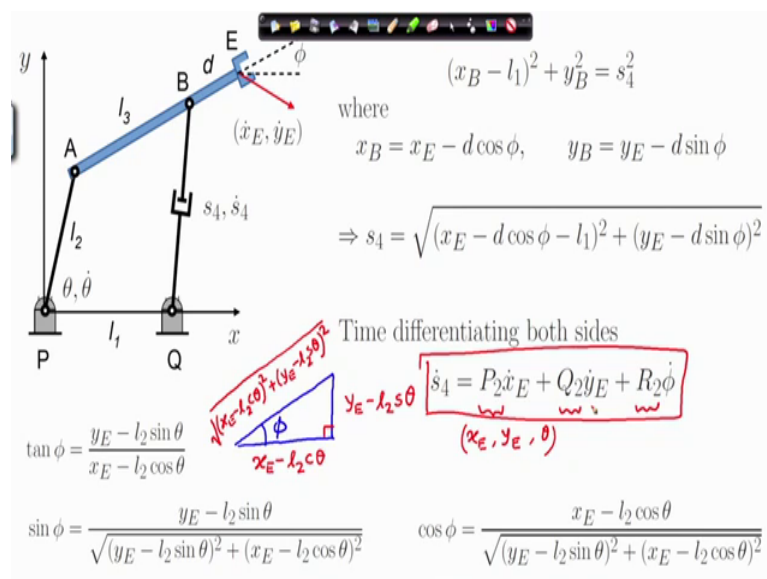
So, this P_2 Q_2 R_2 these will be functions of x_E y_E and ϕ which are the configuration which specify the configuration of the manipulator ok. Now this $\tan \phi$ you can very easily write $\tan \phi$ in terms of θ . So, tangent of this angle comes from the displacement relations that we have looked at before.

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So, these are the relations; you can find out tangent phi. So, tan phi is as equal to y E minus l 2 sine theta by x E minus l 2 cosine theta right; from here we can obtain tangent phi and this is what we are going to use.

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Here, so this is the expression of tan phi. Now once I have tan phi I can also find out cosine phi and sine phi. So, sine phi is this expression, so, how do I find this? Suppose this angle is phi then tan phi is this ratio.

So, once I have done that these terms P^2 , Q^2 and R^2 they are only functions of $x \in y \in$ and θ . P^2 , Q^2 and R^2 are functions of $x \in y \in$ and θ and this relation is between the velocities s^4 . So, $s^4 \cdot x \in y \in$ and $\dot{\phi}$; I have eliminated ϕ . The next step is to eliminate $\dot{\phi}$, then I would have a relation between $x \in y \in$ and $s^4 \cdot \dot{\phi}$; now how to determine $\dot{\phi}$.

Diagram of a slider block mechanism. A horizontal bar of length l_1 is pivoted at point P on the left and point Q on the right. A vertical rod of length l_2 is pivoted at P and has a slider block at point A. A vertical rod of length l_3 is pivoted at Q and has a slider block at point B. A horizontal rod of length d is pivoted at B and has a slider block at point E. The slider blocks move along a horizontal guide. The angle of the vertical rod at P is θ , and the angle of the horizontal rod at B is ϕ . The velocity of the slider block at E is denoted by s_4 .

$$\dot{s}_4 = P_2 \dot{x}_E + Q_2 \dot{y}_E + R_2 \dot{\phi}$$

$$x_E = l_2 \cos \theta + (l_3 + d) \cos \phi$$

$$y_E = l_2 \sin \theta + (l_3 + d) \sin \phi$$

$$\frac{d}{dt} \left[\tan \phi = \frac{y_E - l_2 \sin \theta}{x_E - l_2 \cos \theta} \right]$$

$$\sec^2 \phi \dot{\phi} = (\quad) \dot{x}_E + (\quad) \dot{y}_E + (\quad) \dot{\theta}$$

$$(1 + \tan^2 \phi) \dot{\phi} = (\quad) \dot{x}_E + (\quad) \dot{y}_E + (\quad) \dot{\theta}$$

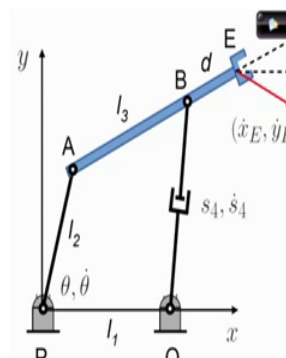
$$\dot{\phi} = \underline{P_3 \dot{x}_E + Q_3 \dot{y}_E + R_3 \dot{\theta}} \quad (x_E, y_E, \theta)$$

Now, if you think a little bit, it will be absolutely clear that the time derivative of this ratio will have a form or can be expressed in the form something times \dot{x} plus something times \dot{y} plus something times $\dot{\theta}$. Now this time derivative is straightforward, but there will be a little bit of algebra in this. Now second square phi can

be again written in terms of tangent phi. So, I have this relation; now tangent phi is already known in terms of x_E y_E and theta.

So, therefore, finally, I must have let me call it some $P_3 \dot{x}_E + Q_3 \dot{y}_E + R_3 \dot{\theta}$; where remember that P_3 Q_3 and R_3 are solely functions of x_E y_E and theta. So, these are solely functions of x_E y_E and theta. So, I have found phi dot in terms of \dot{x}_E \dot{y}_E $\dot{\theta}$; where this P_3 Q_3 R_3 are complicated expressions which you can easily find out by a little bit of algebra.

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$$\dot{s}_4 = P_2 \dot{x}_E + Q_2 \dot{y}_E + R_2 \dot{\phi}$$

$$x_E = l_2 \cos \theta + (l_3 + d) \cos \phi$$

$$y_E = l_2 \sin \theta + (l_3 + d) \sin \phi$$

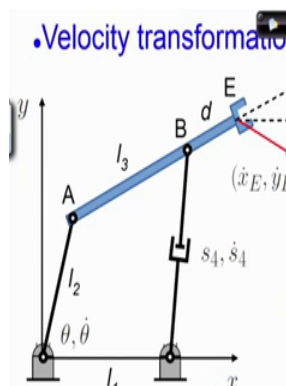
$$\tan \phi = \frac{y_E - l_2 \sin \theta}{x_E - l_2 \cos \theta}$$

Time differentiating both sides

$$\dot{\phi} = P_3 \dot{x}_E + Q_3 \dot{y}_E + R_3 \dot{\theta}$$

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• Velocity transformation: Jacobian



$$P_1 \dot{x}_E + Q_1 \dot{y}_E = R_1 \dot{\theta}$$

$$\dot{\phi} = P_3 \dot{x}_E + Q_3 \dot{y}_E + R_3 \dot{\theta}$$

$$\dot{s}_4 = P_2 \dot{x}_E + Q_2 \dot{y}_E + R_2 \dot{\phi}$$

$$\left[\begin{array}{c} \dot{s}_4 \\ \dot{\phi} \end{array} \right] = \left[\begin{array}{cc} P_2 + R_2 P_3 & Q_2 + R_2 Q_3 \\ P_3 & Q_3 \end{array} \right] \left[\begin{array}{c} \dot{x}_E \\ \dot{y}_E \end{array} \right] + \left[\begin{array}{c} R_2 R_3 \\ R_3 \end{array} \right] \dot{\theta}$$

$$P_1 \dot{x}_E + Q_1 \dot{y}_E = R_1 \dot{\theta}$$

$$(P_2 + R_2 P_3) \dot{x}_E + (Q_2 + R_2 Q_3) \dot{y}_E = -R_2 R_3 \dot{\theta} + \dot{s}_4$$

$$[A] \dot{\vec{x}}_E = [B] \dot{\vec{Y}}$$

$$\dot{\vec{x}}_E = \begin{Bmatrix} \dot{x}_E \\ \dot{y}_E \end{Bmatrix}, \quad \dot{\vec{Y}} = \begin{Bmatrix} \dot{\theta} \\ \dot{s}_4 \end{Bmatrix}$$

$$\dot{\vec{x}}_E = [A]^{-1} [B] \dot{\vec{Y}}$$

$$\dot{\vec{Y}} = [B]^{-1} [A] \dot{\vec{x}}_E$$

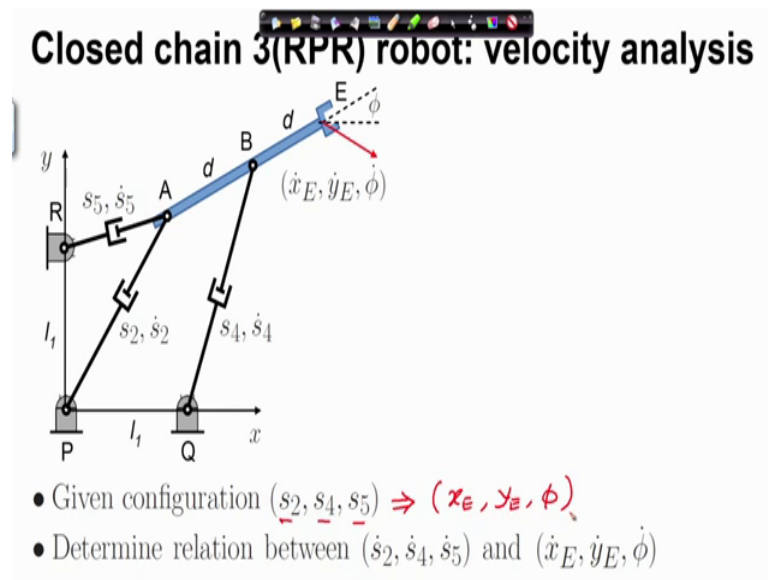
So, here I have collated all the expressions that we had derived; we have derived this first and then these two expressions were derived. Now here I have $\dot{\phi}$ which appears in the expression of \dot{s}_4 . So, therefore, I can eliminate $\dot{\phi}$ in the expression of \dot{s}_4 and write this as; so, when I substitute $\dot{\phi}$ here I will get this times \dot{x}_E plus this term times \dot{y}_E plus $R_2 R_3 \dot{\theta}$.

Now I can rewrite this as; so let me write out the first equation here. So, $P_1 \dot{x}_E + Q_1 \dot{y}_E$ is $R_1 \dot{\theta}$. And now I will write this equation after a little bit of manipulation as that is equal to minus of $R_2 R_3 \dot{\theta}$ plus \dot{s}_4 . Now this I can combine and write in a compact form as let us say some matrix A times $\begin{bmatrix} \dot{x}_E \\ \dot{y}_E \end{bmatrix}$ is equal to matrix B times the vector \dot{Y} , where $\begin{bmatrix} \dot{x}_E \\ \dot{y}_E \end{bmatrix}$ of course, this vector $\begin{bmatrix} \dot{x}_E \\ \dot{y}_E \end{bmatrix}$ is nothing, but $\begin{bmatrix} \dot{x}_E \\ \dot{y}_E \end{bmatrix}$ the vector \dot{Y} is $\begin{bmatrix} \dot{\theta} \\ \dot{s}_4 \end{bmatrix}$.

So, the joint velocity vector and the matrices A and B ; you can very easily read out from these two equations; so, this is our velocity relation. Now it looks a little different than what we had derived earlier, but you can always invert A and rewrite this as; such that this is our Jacobian for the manipulator. So, $A^{-1} B$ is the Jacobian for this manipulator; if you invert B then you have the inverse Jacobian. So, if you write \dot{Y} as $B^{-1} A \dot{X}$, then you have the inverse Jacobian.

So, we have finally, determined the velocity relations. So, our joint velocity vector is $\begin{bmatrix} \dot{\theta} \\ \dot{s}_4 \end{bmatrix}$. So, from here I have related I found a relation to \dot{x}_E and \dot{y}_E which is the end effector velocity.

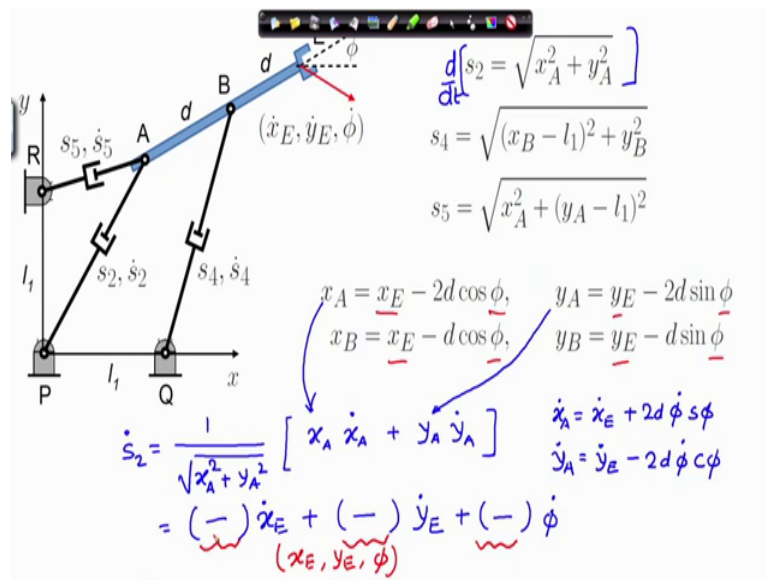
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So, let us move forward and discuss the 3 RPR planar closed chain robot manipulator. Here we need to recall that we had already discussed the displacement analysis the configurations s_2, s_4, s_5 . So, given s_2, s_4, s_5 ; the configuration of the manipulator is specified and we have we had found x_E, y_E and ϕ .

So, given s_2, s_4 and s_5 you can find out x_E, y_E and ϕ ; so, these are 3 degree of freedom manipulator. So, you have 3 inputs s_2, s_4 and s_5 and we can find out x_E, y_E and ϕ . Now the velocity analysis requires the specification of the relation between $\dot{s}_2, \dot{s}_4, \dot{s}_5$ and \dot{x}_E, \dot{y}_E and $\dot{\phi}$. So, that is the velocity analysis problem.

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Now, we have derived these relations; these are very easy to see s_2 is the throw of this actuator. Similarly s_4 and s_5 and they are related in terms of the coordinates of point A and B and the coordinates of P Q and R.

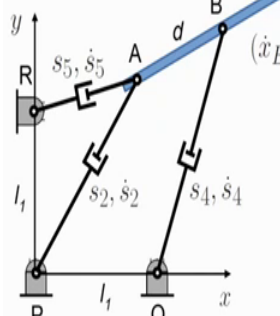
So, these relations we have already looked at. So, here you can relate x_A y_A ; so coordinates of point A and similarly coordinates of point B. In terms of the coordinates of the end effector and the orientation of the end effector ϕ ; so therefore, you can just substitute them here and start differentiating. So, for example, if you differentiate the first equation, which I have written out after some simplification; so this is what you have.

And now you can replace x_A y_A in terms of x_E y_E and ϕ and you can find out \dot{x}_A dot as \dot{x}_E dot plus $2d \dot{\phi} \sin \phi$. Similarly \dot{y}_A dot is \dot{y}_E dot minus $2d \dot{\phi} \cos \phi$; so these also you can replace. So, finally, \dot{s}_2 will be something times \dot{x}_E dot plus something times \dot{y}_E dot plus something times $\dot{\phi}$. Now these bracketed quantities can be easily determined after some algebra; you substitute these expressions of x_A y_A \dot{x}_A dot and \dot{y}_A dot and you can find out this relation.

Now this bracketed quantities are functions of only x_E y_E and ϕ . So, this bracketed quantities are functions of x_E y_E and ϕ . So, if you know the configuration of the manipulator; these things are already known to you. So, which means these bracketed quantities are known to you; in that case I have found a relation between the expansion

rate of s_2 which is \dot{s}_2 and the velocity of the end effector which is \dot{x}_E , \dot{y}_E and $\dot{\phi}$.

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$$s_2 = \sqrt{x_A^2 + y_A^2}$$

$$s_4 = \sqrt{(x_B - l_1)^2 + y_B^2}$$

$$s_5 = \sqrt{x_A^2 + (y_A - l_1)^2}$$

$$x_A = x_E - 2d \cos \phi, \quad y_A = y_E - 2d \sin \phi$$

$$x_B = x_E - d \cos \phi, \quad y_B = y_E - d \sin \phi$$

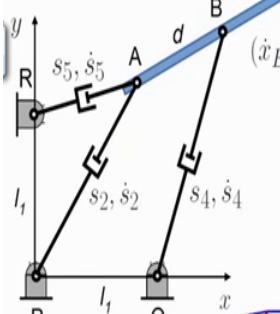
Time differentiating both sides

$$\dot{x}_A = \dot{x}_E + (2d \sin \phi) \dot{\phi}, \quad \dot{y}_A = \dot{y}_E - (2d \cos \phi) \dot{\phi}$$

$$\dot{x}_B = \dot{x}_E + (d \sin \phi) \dot{\phi}, \quad \dot{y}_B = \dot{y}_E - (d \cos \phi) \dot{\phi}$$

Now, if you continue. So, this is the time derivative.

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$$s_2 = \sqrt{x_A^2 + y_A^2}$$

$$s_4 = \sqrt{(x_B - l_1)^2 + y_B^2}$$

$$s_5 = \sqrt{x_A^2 + (y_A - l_1)^2}$$

Time differentiating both sides

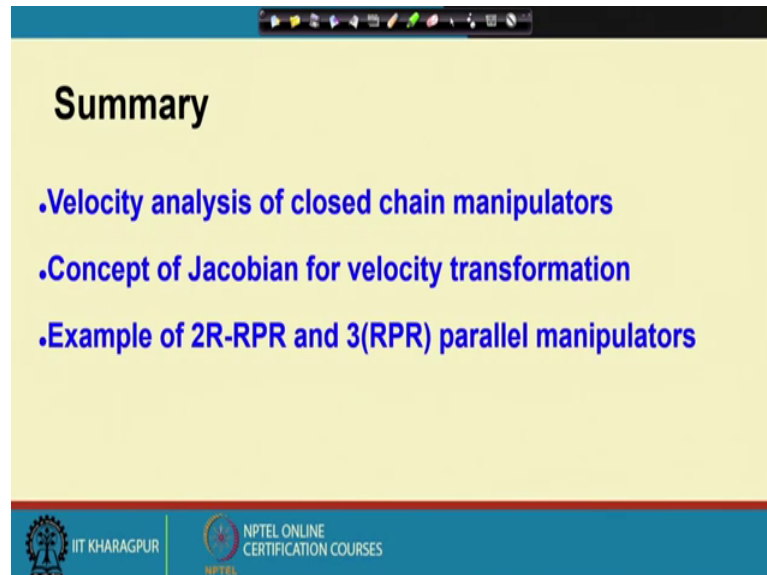
$$\begin{bmatrix} \dot{s}_2 \\ \dot{s}_4 \\ \dot{s}_5 \end{bmatrix} = [J] \begin{bmatrix} \dot{x}_E \\ \dot{y}_E \\ \dot{\phi} \end{bmatrix}$$

$$\begin{aligned} \dot{s}_2 &= P_1 \dot{x}_E + Q_1 \dot{y}_E + R_1 \dot{\phi} \\ \dot{s}_4 &= P_2 \dot{x}_E + Q_2 \dot{y}_E + R_2 \dot{\phi} \\ \dot{s}_5 &= P_3 \dot{x}_E + Q_3 \dot{y}_E + R_3 \dot{\phi} \end{aligned}$$

So, if you substitute all these expressions; then finally, you obtain these relations between \dot{s}_2 , \dot{s}_4 , \dot{s}_5 and \dot{x}_E , \dot{y}_E and $\dot{\phi}$. So, which means I have \dot{s}_2 , \dot{s}_4 , \dot{s}_5 is equal to a Jacobian matrix times \dot{x}_E , \dot{y}_E and $\dot{\phi}$ and

that is the velocity relation that we are seeking; the Jacobian can be easily read out from these detailed relations.

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So, to summarize what we have discussed in this lecture; we have looked at the closed chain parallel planar manipulators. We looked at the velocity analysis problem, we have found the Jacobian of the Jacobian matrix for these manipulators. I have outlined methods for determining this Jacobian. We have looked at these 2 examples of 2 R RPR and 3 RPR closed chain planar a parallel manipulators.

So, with this we have now discussed the velocity analysis of both open chain and closed chain manipulators. Now this paves way for discussions on motion planning or path planning; path generation problem for these manipulators. So, this lecture I will close here.