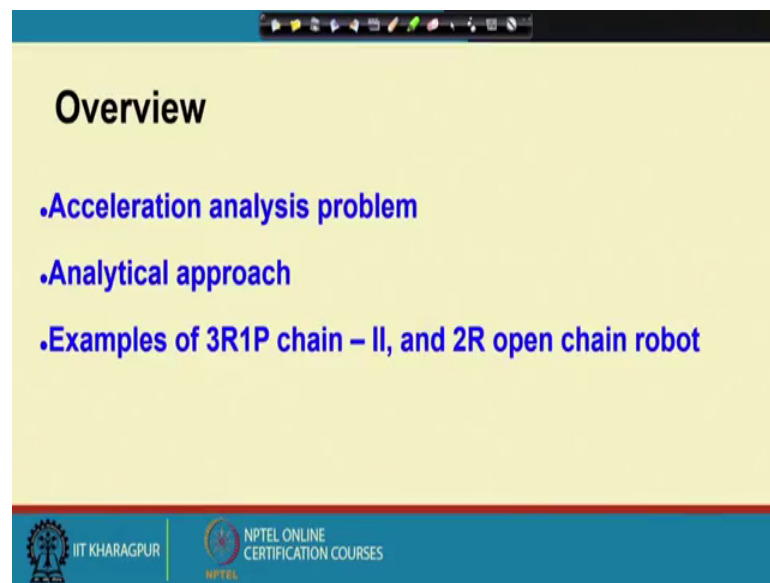


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**Lecture – 35**  
**Acceleration Analysis – II**

In this lecture we are going to continue our discussion on acceleration analysis. So, in the last lecture; we had looked at two examples of constraint mechanisms we are going to discuss on this further and take examples of constraint mechanisms as well as robots.

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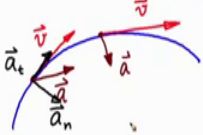


To give you an overview of today's lecture we are going to discuss the analytical acceleration analysis problem with examples of 3R1P chain of type 2 and 2 our open chain planar manipulator.

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## Acceleration analysis

- Mechanism transforms actuator motion input(s) to motion of output link
- Motion: characterized by displacement, velocity and acceleration
- Requires completion of displacement and velocity analysis
- Acceleration analysis is non-intuitive!



Let us review the things we had discussed before we know that a mechanism transforms the actuator input motion to the motion of the output link. Now motion as we know is characterized by displacement velocity and acceleration and in these lectures, we are now discussing the acceleration part. We have already discussed the displacement and velocity analysis of mechanisms and robots.

This acceleration analysis will now require the displacement and velocity analysis as a starting point, now acceleration analysis is non-intuitive. So, we have discussed this point let me reiterate once more you know that when a particle is moving on a path the velocity of the particle at any point on the path is tangential to the path the velocity vector is always tangential to the path.

However, the acceleration vector does not have any such restriction. So, I am showing the acceleration at 2 points at 2 instance of a particle moving on a curved path. As you know that this has 2 components the acceleration has 2 components one is tangent to the path the other is normal to the path. The tangent acceleration if you call it  $a_t$  the tangential component of acceleration goes on to change the magnitude of the velocity which means it changes the speed.

And the normal component is because of the path curvature. So, the normal component of acceleration in this case is because of the path curvature. So, acceleration can have an

arbitrary direction for a particle moving on a curved path, there was another example that I discussed.

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**Acceleration analysis**

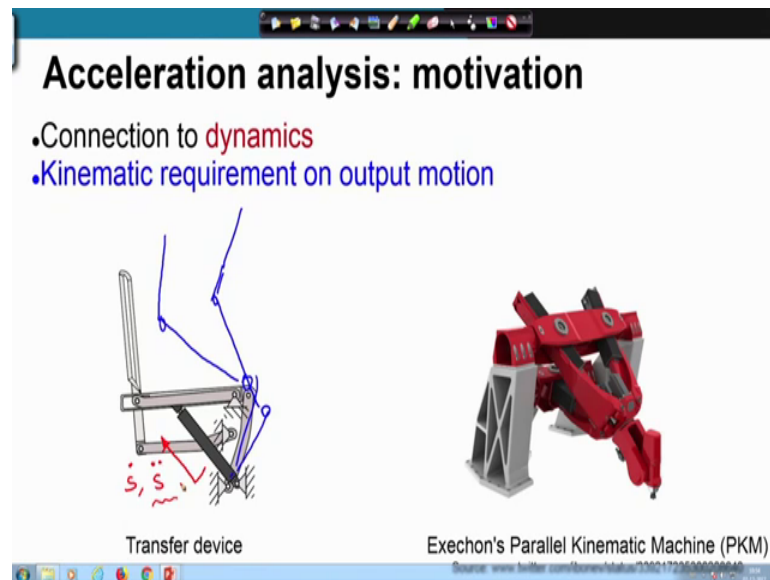
- Mechanism transforms actuator motion input(s) to motion of output link
- Motion: characterized by displacement, velocity and acceleration
- Requires completion of displacement and velocity analysis
- Acceleration analysis is non-intuitive!

The diagram shows a circle representing a disc. A curved arrow on the left indicates angular velocity  $\omega$ . Inside the circle, a red arrow points downwards, labeled 'Coriolis acceleration'.

And that is of a disc that is rotating at a constant speed and a particle that is moving with respect to the disc also at a constant speed let us say.

So, even though speeds here are all constant the angular speed is constant the linear speed of the particle with respect to the disc is constant the particle still suffers acceleration and this is the Coriolis acceleration as you know. So, the particle suffers Coriolis acceleration. So, therefore, the acceleration analysis is non-intuitive.

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Now, the motivation for; that means, doing x acceleration analysis is that it is connected to dynamics. For example, Newton's second law relates the acceleration of a particle or a rigid body center of mass of a rigid body to the forces acting on the particle of the body. There is another requirement that we have and that is the kinematic requirement on the input output motion.

So, here we have this transfer device, which I might require to have constant speed while it goes from the sitting to the standing position ; in that case not only the velocity of expansion of the actuator is important, but also the acceleration of expansion is also important.

So, if I want to produce a certain acceleration or possibly 0 acceleration at the output, I may require a non-0 acceleration at the input. So, this is what we need to find out. So, we want to find out a relate the output acceleration with the input acceleration.

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**Acceleration analysis problem**

- Relating the actuator acceleration(s) with output link acceleration
- Given: analytical displacement and velocity relations

Exechon's Parallel Kinematic Machine (PKM)

Excavator

Source: [www.twitter.com/ibonev/status/339217235300208640](http://www.twitter.com/ibonev/status/339217235300208640)

So, the acceleration analysis problem is about relating the actuator acceleration in input with the acceleration of the output link. And we are given the displacement and velocity relations. So, we will start with these as inputs.

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**Acceleration analysis: plan**

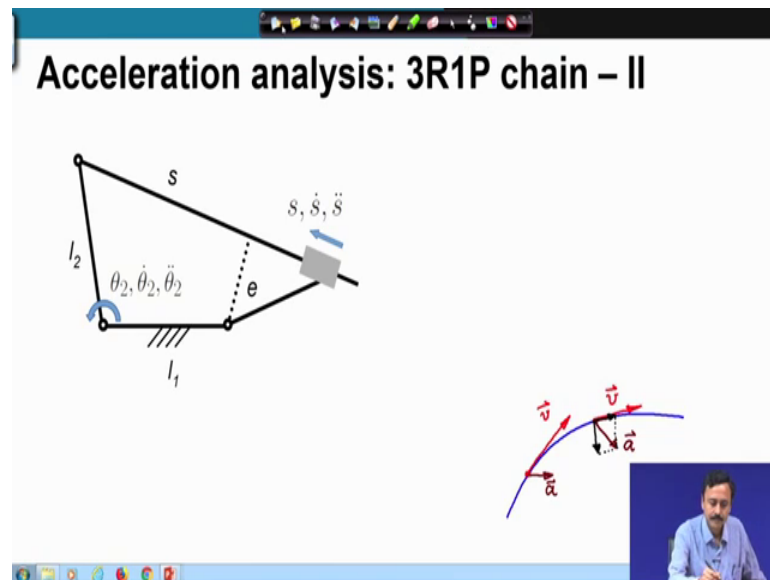
- Constrained mechanisms: 4R and 3R1P chains
- Robots: 2R open chain planar manipulator

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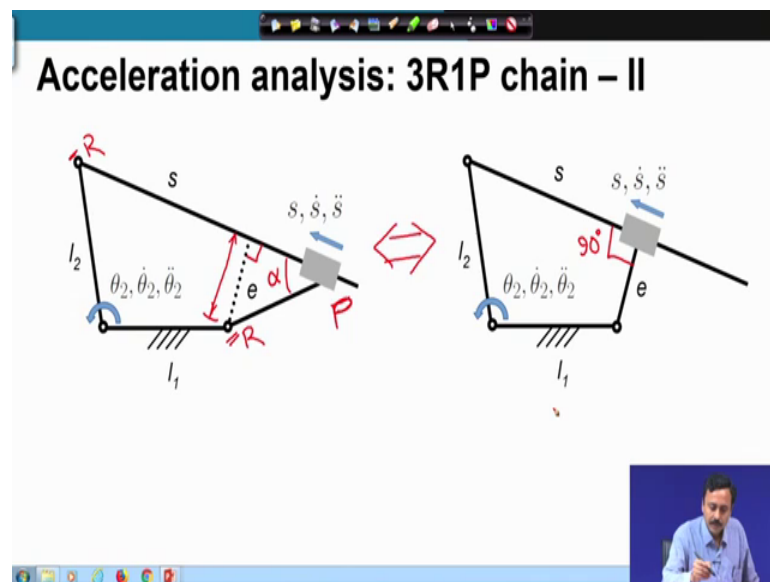
So, as we have discussed before our plan is to first discuss the constraint mechanism and subsequently go over to robotic manipulators. So, we will discuss the example of a 2R open chain planar manipulator.

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We will start with the acceleration analysis of a 3R1P chain of type 2, we have discussed this point before that for a 3R1P chain of type 2 here we have the offset.

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So, this is the distance between the 2 revolute pairs connected to the ends of the links with the prismatic pair. So, here we have the prismatic pair and these are the 2 revolute pairs connected to the links which are connected through the prismatic pair. So, the distance between these 2 revolute pairs measured in a direction perpendicular to the direction of the P PR is the offset now this angle let us say alpha is a constant.

So, therefore, I can consider an equivalent 3R1P chain with this angle being 90 degree. The reason we are doing this is that it simplifies certain expressions, and we can always go from one chain to the other. So, if I know the velocity acceleration analysis of the chain on the right I can always re-compute the velocity and acceleration of the chain on the left.

(Refer Slide Time: 09:32)

### Acceleration analysis: 3R1P chain – II

- Configuration  $(\theta_2, s)$ : displacement analysis ✓
- To determine expression relating  $\ddot{\theta}_2$  and  $\ddot{s}$  — ?

So, therefore, we are going to analyze the chain shown on the right we are given the displacement relations. So, we already know the displacement relations we know the angle theta 2 and s. So, if I if you are given theta 2 we can find out s and vice versa. We also know the velocity relations so essentially, we know this relation, we know the velocity relation what we have to find out is the acceleration relation. So, this is what is; our objective in this lecture to determine the relation between theta 2 double dot and s double dot.

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The diagram shows a slider-crank mechanism. Link 1 is the ground, link 2 is the crank of length  $l_2$ , and link 3 is the slider of length  $s$ . The angle of link 2 is  $\theta_2$ . The slider moves along a vertical guide. The velocity of the slider is  $\dot{s}$  and its acceleration is  $\ddot{s}$ . The angular velocity of link 2 is  $\dot{\theta}_2$ . The analytical velocity relation is given as:

$$\dot{\theta}_2 = \left[ \frac{s}{l_1 l_2 \sin \theta_2} \right] \dot{s}$$

$$\Rightarrow \dot{\theta}_2 = J \dot{s}$$

A small inset video shows a presenter.

So, we will start off with the analytical velocity relation which we have derived earlier further for this chain. I have mentioned that this is the angular speed at the output link and the angle and the linear speed at the input of the prismatic pair.

So, essentially this distance which is  $s$  we know the rate at which this link is sliding out of the prismatic pair. So, this is  $\dot{s}$  so I know the rate at which this link is sliding out of the P pair and that relation is given here this relation we have derived earlier when we discussed the velocity analysis.

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The diagram is identical to the previous slide, showing the slider-crank mechanism. The analytical velocity relation is repeated:

$$\dot{\theta}_2 = \left[ \frac{s}{l_1 l_2 \sin \theta_2} \right] \dot{s}$$

$$\Rightarrow \dot{\theta}_2 = J \dot{s}$$

Time differentiating both sides

$$\ddot{\theta}_2 = \dot{J} \dot{s} + J \ddot{s}$$

$$J = J(s, \theta)$$

$$\dot{J} = \frac{\partial J}{\partial s} \dot{s} + \frac{\partial J}{\partial \theta} \dot{\theta}$$

$$= \left( \frac{\partial J}{\partial s} + \frac{\partial J}{\partial \theta} J \right) \dot{s}$$

$$\ddot{\theta}_2 = \left( \frac{\partial J}{\partial s} + \frac{\partial J}{\partial \theta} J \right) \dot{s}^2 + J \ddot{s}$$

A small inset video shows a presenter.

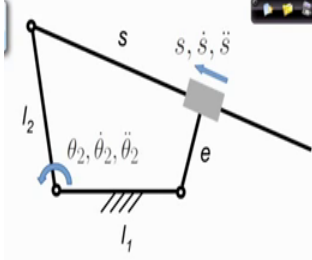


So, if you differentiate with respect to time both sides of this velocity relation you will obtain this acceleration relation. So, on the left I have the acceleration at the output and on the right, I have a term where I have the time derivative of the Jacobian and the rate of expansion of the actuator; and a term which involves the acceleration of expansion of the actuator.

Now, once again this Jacobian is a function of  $s$  and  $\theta$ ; therefore, the time derivative of Jacobian can be obtained easily using chain rule. So,  $\frac{dJ}{dt} = \frac{\partial J}{\partial s} \dot{s} + \frac{\partial J}{\partial \theta} \dot{\theta}$  now you will observe that  $\dot{\theta}$  is nothing but  $J \dot{s}$ . So, therefore, this expression can be rewritten by taking  $\dot{s}$  out in this form.

Finally, if you substitute this expression of  $\frac{dJ}{dt}$  in here, you will obtain the expression of  $\ddot{\theta}$ . So, this expression relates the angular acceleration of the output link with the input acceleration and velocity of the prismatic pair expansion.

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Analytical velocity relation

$$\dot{\theta}_2 = \left[ \frac{s}{l_1 l_2 \sin \theta_2} \right] \dot{s}$$

$$\Rightarrow \dot{\theta}_2 = J \dot{s}$$

Time differentiating both sides

$$\ddot{\theta}_2 = \dot{J} \dot{s} + J \ddot{s}$$

$$\Rightarrow \ddot{\theta}_2 = \left( \frac{\partial J}{\partial \theta_2} J + \frac{\partial J}{\partial s} \right) \dot{s}^2 + J \ddot{s}$$

So, let me show this formally. So, this is our final expression that we have.

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The diagram shows a slider-crank mechanism. A horizontal link of length  $l_1$  is pivoted at the origin. A connecting link of length  $l_2$  is pivoted at the other end of  $l_1$ . A slider of length  $e$  is attached to the end of  $l_2$  and moves vertically along a guide. The distance from the origin to the slider is  $s$ . The angle between  $l_1$  and  $l_2$  is  $\theta_2$ . The slider's displacement is  $s$ , velocity is  $\dot{s}$ , and acceleration is  $\ddot{s}$ . The angle's velocity is  $\dot{\theta}_2$  and acceleration is  $\ddot{\theta}_2$ .

Handwritten notes and equations:

- Linear relation of acceleration
- Inhomogeneous
- If  $\ddot{\theta}_2 = 0$
- $\ddot{s} = J^{-1} \left[ \frac{\partial J}{\partial \theta_2} J + \frac{\partial J}{\partial s} \right] \dot{s}^2$
- $\dot{\theta}_2 = \left[ \frac{s}{l_1 l_2 \sin \theta_2} \right] \dot{s}$
- $\Rightarrow \dot{\theta}_2 = J \dot{s}$
- $\ddot{\theta}_2 = \left( \frac{\partial J}{\partial \theta_2} J + \frac{\partial J}{\partial s} \right) \dot{s}^2 + J \ddot{s}$
- $\frac{\partial J}{\partial s} = \frac{1}{l_1 l_2 \sin \theta_2}$
- $\frac{\partial J}{\partial \theta_2} = -\frac{s \cos \theta_2}{l_1 l_2 \sin^2 \theta_2}$

So, let me rewrite these expressions and determine what is  $\frac{\partial J}{\partial \theta_2}$  and  $\frac{\partial J}{\partial s}$ . So, these are the relations which can be easily obtained from here by differentiating with respect to  $s$ , I have  $\frac{\partial J}{\partial s}$  and with respect to  $\theta_2$  I have this expression of  $\frac{\partial J}{\partial \theta_2}$ .

Now, what we observe here is that the acceleration relations are linear. So, between  $\ddot{\theta}_2$  and  $\ddot{s}$  I have a linear relation; however, this relation is inhomogeneous. Because of the presence of this term which is configuration and velocity dependent. So, this is the inhomogeneous term which depends on the configuration and the rate of expansion of the actuator the velocity of expansion of the prismatic actuator.

Furthermore, if I want to move the output at a constant speed; in other words, if I want to have  $\ddot{\theta}_2 = 0$  which means the output moves at a constant speed, I must have acceleration at the input and that is obtained in this form. So, this is the acceleration that I must have at the input in order to produce a constant output acceleration.

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$\dot{\theta}_2 = \left[ \frac{s}{l_1 l_2 \sin \theta_2} \right] \dot{s}$   
 $\Rightarrow \dot{\theta}_2 = J \dot{s}$   
 $\ddot{\theta}_2 = \left( \frac{\partial J}{\partial \theta_2} J + \frac{\partial J}{\partial s} \right) \dot{s}^2 + J \ddot{s}$   
 $\frac{\partial J}{\partial s} = \frac{1}{l_1 l_2 \sin \theta_2}$   
 $\frac{\partial J}{\partial \theta_2} = -\frac{s \cos \theta_2}{l_1 l_2 \sin^2 \theta_2}$

Handwritten notes:  
 $\int \ddot{s} = 0$   
 $\ddot{\theta}_2 = \left( \frac{\partial J}{\partial \theta_2} J + \frac{\partial J}{\partial s} \right) \dot{s}^2$   
 $\dot{s} = \text{constant}$

Similarly, if I have the input as a constant which means if  $s$  double dot is 0; that means, that prismatic actuator is expanding at a constant rate; in that case I have acceleration at the output. If I am moving at a constant if I have the prismatic actuator expand at a constant rate which means  $s$  dot is constant, then the output has a angular acceleration.

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### Acceleration analysis: planar 2R manipulator

- Configuration  $(\theta_1, \theta_2)$ : specified ✓
- Velocity  $(\dot{\theta}_1, \dot{\theta}_2)$ : specified ✓
- To determine expression relating  $(\ddot{\theta}_1, \ddot{\theta}_2)$  and  $(\ddot{x}_E, \ddot{y}_E)$  ?

Next let us look at the acceleration analysis of a planar 2R manipulator. So, here I have shown a 2R manipulator which is moving at a certain configuration it is moving with

velocity  $\dot{\theta}_1$   $\dot{\theta}_2$  and acceleration  $\ddot{\theta}_1$  and  $\ddot{\theta}_2$  double dot.

So, this has a velocity of the end effector as  $\dot{x}_E$   $\dot{y}_E$  and an acceleration  $\ddot{x}_E$   $\ddot{y}_E$ . Now the configuration of the manipulator is specified the velocity is specified we need to relate the actuator accelerations the joint accelerations and the end effector accelerations.

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$$\dot{x}_E = [-l_1 \sin \theta_1 - l_2 \sin(\theta_1 + \theta_2)]\dot{\theta}_1 + [-l_2 \sin(\theta_1 + \theta_2)]\dot{\theta}_2$$

$$\dot{y}_E = [l_1 \cos \theta_1 + l_2 \cos(\theta_1 + \theta_2)]\dot{\theta}_1 + [l_2 \cos(\theta_1 + \theta_2)]\dot{\theta}_2$$

where

$$\begin{Bmatrix} \dot{x}_E \\ \dot{y}_E \end{Bmatrix} = \begin{bmatrix} J_{11} & J_{12} \\ J_{21} & J_{22} \end{bmatrix} \begin{Bmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \end{Bmatrix}$$

$$J_{11} = -l_1 \sin \theta_1 - l_2 \sin(\theta_1 + \theta_2)$$

$$J_{12} = -l_2 \sin(\theta_1 + \theta_2)$$

$$J_{21} = l_1 \cos \theta_1 + l_2 \cos(\theta_1 + \theta_2)$$

$$J_{22} = l_2 \cos(\theta_1 + \theta_2)$$

So, here I have written out the velocity relations which we have already derived earlier. Now if you assemble them in this form we have the relation of the end effector velocity and the joint velocities through the Jacobian which we have discussed.

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$$\begin{Bmatrix} \dot{x}_E \\ \dot{y}_E \end{Bmatrix} = \begin{bmatrix} J_{11} & J_{12} \\ J_{21} & J_{22} \end{bmatrix} \begin{Bmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \end{Bmatrix}$$

where

$$\begin{aligned} J_{11} &= -l_1 \sin \theta_1 - l_2 \sin(\theta_1 + \theta_2) \\ J_{12} &= -l_2 \sin(\theta_1 + \theta_2) \\ J_{21} &= l_1 \cos \theta_1 + l_2 \cos(\theta_1 + \theta_2) \\ J_{22} &= l_2 \cos(\theta_1 + \theta_2) \end{aligned}$$

where  $\{\dot{\mathbf{X}}_E\} = [\mathbf{J}]\{\dot{\boldsymbol{\theta}}\}$

$$\{\dot{\mathbf{X}}_E\} = \begin{Bmatrix} \dot{x}_E \\ \dot{y}_E \end{Bmatrix}, \quad \{\dot{\boldsymbol{\theta}}\} = \begin{Bmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \end{Bmatrix}$$

$$[\mathbf{J}] = \begin{bmatrix} J_{11} & J_{12} \\ J_{21} & J_{22} \end{bmatrix}$$

This can be written in a compact form as shown here, where  $\dot{x}_E$  this is the end effector velocity vector and this  $\dot{\theta}$  vector is the joint velocity vector.

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$$\{\mathbf{X}_E\} = [\mathbf{J}]\{\boldsymbol{\theta}\}$$

where

$$\{\dot{\mathbf{X}}_E\} = \begin{Bmatrix} \dot{x}_E \\ \dot{y}_E \end{Bmatrix}, \quad \{\dot{\boldsymbol{\theta}}\} = \begin{Bmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \end{Bmatrix}$$

$$[\mathbf{J}] = \begin{bmatrix} J_{11} & J_{12} \\ J_{21} & J_{22} \end{bmatrix}$$

$$\ddot{\mathbf{X}}_E = \underbrace{[\dot{\mathbf{J}}]}_{\text{Jacobian derivative}} \dot{\boldsymbol{\theta}} + [\mathbf{J}] \ddot{\boldsymbol{\theta}}$$

Now, if you differentiate this velocity relation with respect to time as we have been doing. So, we will obtain  $\ddot{x}_E$  is equal to the time derivative of the Jacobian remember that the Jacobian is a function of the configuration. So, time derivative of the Jacobian times the joint velocity vector plus Jacobian matrix times the joint acceleration vector. Now this Jacobian derivative is now a little more complicated.

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where

$$\{\dot{\mathbf{X}}_E\} = \begin{Bmatrix} \dot{x}_E \\ \dot{y}_E \end{Bmatrix}, \quad \{\dot{\boldsymbol{\theta}}\} = \begin{Bmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \end{Bmatrix}$$

$$[\mathbf{J}] = \begin{bmatrix} J_{11} & J_{12} \\ J_{21} & J_{22} \end{bmatrix}$$

Time differentiating both sides

$$\{\ddot{\mathbf{X}}_E\} = [\dot{\mathbf{J}}]\{\dot{\boldsymbol{\theta}}\} + [\mathbf{J}]\{\ddot{\boldsymbol{\theta}}\}$$

So, we are going to look at that, here I have written out this the derivative of the end effector velocity which gives us the end effector acceleration.

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where

$$[\mathbf{J}] = \begin{bmatrix} J_{11} & J_{12} \\ J_{21} & J_{22} \end{bmatrix}, \quad [\dot{\mathbf{J}}] = \begin{bmatrix} \dot{J}_{11} & \dot{J}_{12} \\ \dot{J}_{21} & \dot{J}_{22} \end{bmatrix}$$

$$J_{11} = -l_1 \sin \theta_1 - l_2 \sin(\theta_1 + \theta_2)$$

$$J_{12} = -l_2 \sin(\theta_1 + \theta_2)$$

$$J_{21} = l_1 \cos \theta_1 + l_2 \cos(\theta_1 + \theta_2)$$

$$J_{22} = l_2 \cos(\theta_1 + \theta_2)$$

Handwritten derivative equations:

$$\dot{J}_{11} = -l_1 \dot{\theta}_1 c \theta_1 - l_2 (\dot{\theta}_1 + \dot{\theta}_2) c(\theta_1 + \theta_2)$$

$$\dot{J}_{12} = -l_2 (\dot{\theta}_1 + \dot{\theta}_2) c(\theta_1 + \theta_2)$$

$$\dot{J}_{21} = l_1 \dot{\theta}_1 s \theta_1 + l_2 (\dot{\theta}_1 + \dot{\theta}_2) s(\theta_1 + \theta_2)$$

$$\dot{J}_{22} = -l_2 (\dot{\theta}_1 + \dot{\theta}_2) s(\theta_1 + \theta_2)$$

And the derivative of the Jacobian is the derivative time derivative of individual entries of the Jacobian; remember that the Jacobian has these entries.

So, if you calculate for example, J 11 dot then you have minus l 1 theta one dot cosine theta one I am writing c theta one in place of cosine theta one and minus l 2 theta one dot plus theta 2 dot times cosine theta 1 plus theta 2. So, that is j 11 dot similarly you can

write  $\ddot{J}_{12}$  which is nothing but  $-\dot{\theta}_2 \sin \theta_1 + \dot{\theta}_1 \cos \theta_1 \dot{\theta}_2$ .

In this manner you can find out all the time derivatives you can find out  $\ddot{J}_{21}$ ,  $\ddot{J}_{22}$ ,  $\ddot{J}_{21}$  dot and  $\ddot{J}_{22}$  dot etcetera. So, these are straightforward, but now the expressions are little complicated a little lengthy. So, these expressions go into this  $\ddot{J}$  dot and then we have the relation between the input or the joint accelerations and the end effector acceleration.

So, here again we find that in terms of acceleration these relations are linear, but there is this term  $\ddot{J} \dot{\theta}$  which makes this relation inhomogeneous. Now in this case of robot manipulators the acceleration analysis has one more significance. We have discussed this path generation problem and we have found that in the path generation problem when the manipulator moves close to the singular singularity or the singular configuration, then the input velocities can have very high values the reason being the inverse of the Jacobian becomes goes close to singularity; which means the entries of the inverse of the Jacobian becomes very large the entries become very large.

There in the inverse expression in the expression of the inverse of the Jacobian we have the determinant of the Jacobian sitting in the denominator. So, when this denominator goes to 0 that is when the Jacobian becomes singular the terms in the inverse of the Jacobian they become very large, because the denominator is going to 0. And hence, the required input velocity to generate the finite output velocity becomes very large the required input velocities or the joint velocities become very large.

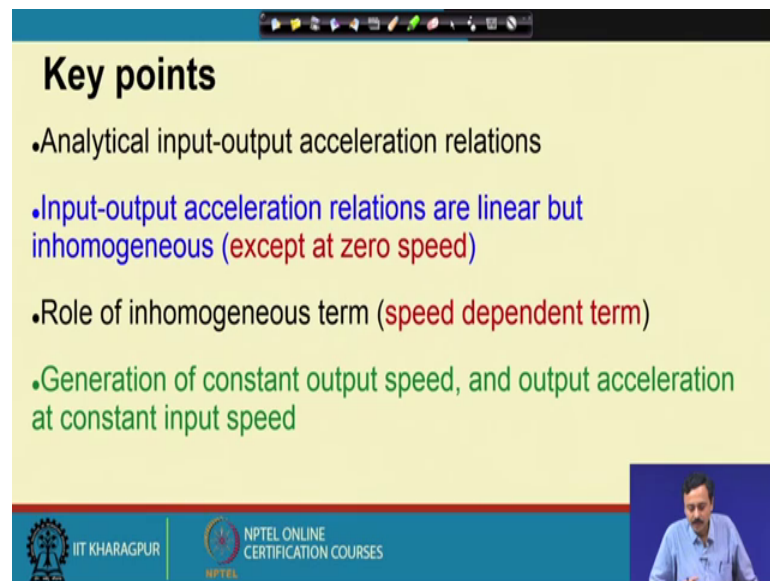
Now, if the joint velocities become very large; that means, this will has have very high accelerations. Now very high joint accelerations would require very high torques on the part of the motor this needs to be restricted. So, any motor will have a finite torque producing capacity or torque specification it cannot produce talked more than a certain value.

Furthermore, if you do not want to damage the motors you would like to have the accelerations restricted to certain values. So, therefore, from the acceleration expressions we can now find out what will be the required joint acceleration near singularity and we can possibly taper the acceleration. So, in order to understand how we should restrict

accelerations and what effect it is going to produce on our output motion; we need this acceleration expression which we have derived here.

So, if you want to restrict the joint accelerations to within certain range, then you can correspondingly calculate what will be the acceleration on the end effector. So, this will accrue certain errors and you can estimate those errors.

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**Key points**

- Analytical input-output acceleration relations
- Input-output acceleration relations are linear but inhomogeneous (except at zero speed)
- Role of inhomogeneous term (speed dependent term)
- Generation of constant output speed, and output acceleration at constant input speed

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So, the key points of our discussion in this lecture we have derived the analytical input output acceleration relations, they are found to be linear, but in homogeneous we have looked at the role of the inhomogeneous term which is speed dependent.

And we have also discussed the generation of constant output speed constant and the corresponding acceleration at the input that is required to produce constant output speed. And we have also seen that if I have a constant input speed then we have acceleration at the output and we can calculate all these things from our acceleration relations.



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**Summary**

- Acceleration analysis problem
- Analytical approach
- Examples of 3R1P chain – II, and 2R open chain robot

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So, to summarize we have looked at the acceleration analysis problem using the analytical approach we have looked at this samples of 3R1P chain of type 2 and a 2R open chain planar robot manipulator.

So, with that I will conclude this lecture.