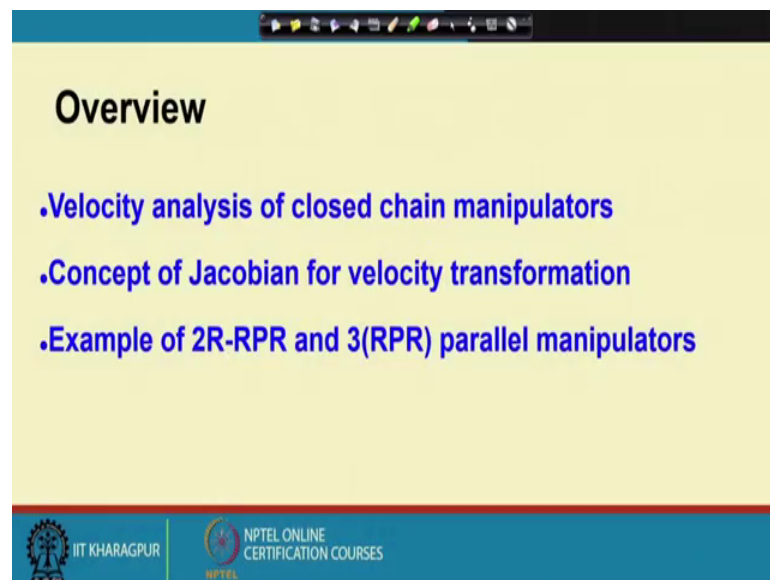


Mechanism and Robot Kinematics
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Lecture – 32
Parallel Manipulator Velocity Analysis

We have been discussing about the velocity analysis of robot manipulators. We have already discussed open chain robot manipulators. We have looked at the Jacobean, its interpretation and singularity of the Jacobean. We have looked at the consequences of singularity of the Jacobean.

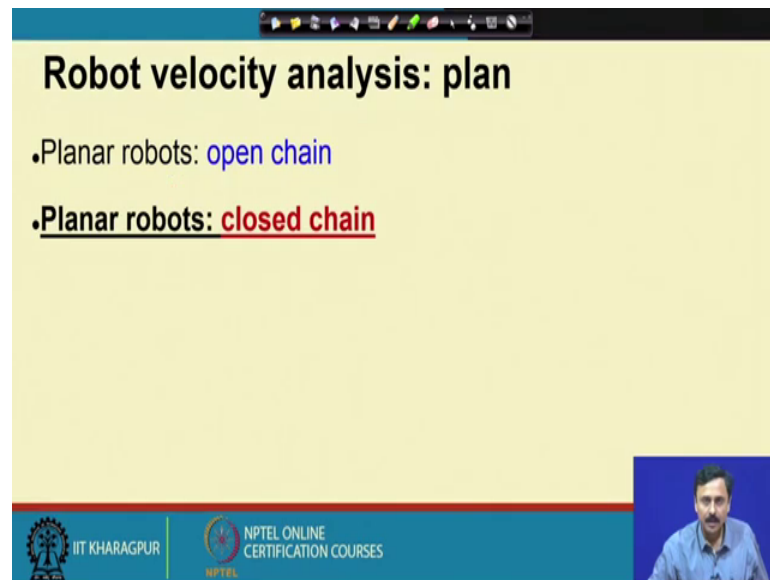
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In this lecture, we are going to embark upon discussions on closed chain manipulators. We are going to look at velocity analysis of closed chain manipulators and I will take two examples.

The 2R-RPR plus 3 RPR parallel manipulators these are planer manipulators. We have already looked at the displacement analysis of both these parallel manipulator chains.

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Robot velocity analysis: plan

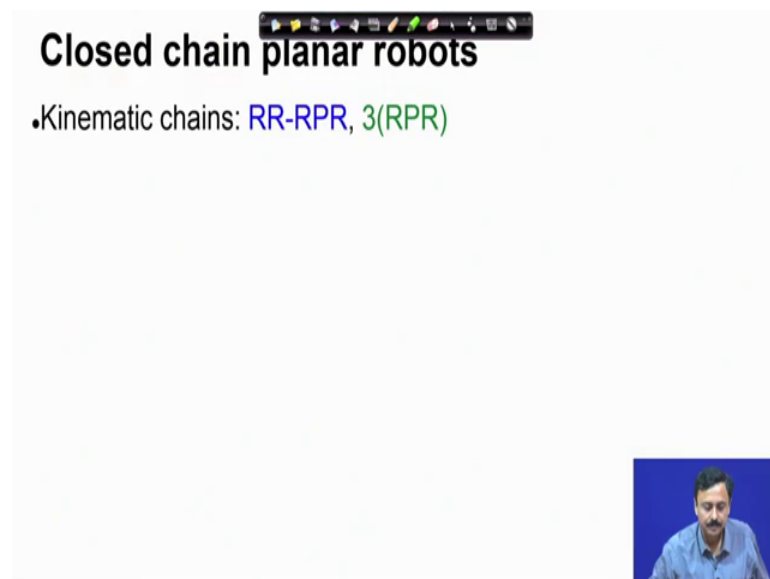
- Planar robots: **open chain**
- **Planar robots: closed chain**

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So, in this lecture we are going to look at closed chain planar manipulators.

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Closed chain planar robots

- Kinematic chains: **RR-RPR, 3(RPR)**

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Going to discuss this RR-RPR and 3 RPR chain.

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Closed chain RR-RPR robot: velocity analysis

- Given configuration (θ, s_4)
- Determine relation between $(\dot{\theta}, \dot{s}_4)$ and (\dot{x}_E, \dot{y}_E)

So, let us begin with the RRR PR planar closed chain robot. So, in this manipulator the configuration is specified by this angle theta and the extension of this prismatic pair which I have called S4. So, theta and x S4 they specify the configuration of the manipulator now the problem is to determine the relation between their velocities. So, we have theta dot and S4 dot as the joint velocities are the actuator velocities and xE dot and yE dot as the end effector velocities the velocity vector. So, we would like to find out a relation between them.

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$$x_E = l_2 \cos \theta + (l_3 + d) \cos \phi$$

$$y_E = l_2 \sin \theta + (l_3 + d) \sin \phi$$

$$(x_E - l_2 \cos \theta)^2 + (y_E - l_2 \sin \theta)^2 = (l_3 + d)^2$$

where

$$\Rightarrow \underline{A \sin \theta} + \underline{B \cos \theta} = \underline{C}$$

$$A = \underline{y_E}, \quad B = \underline{x_E}$$

$$C = \frac{x_E^2 + y_E^2 + l_2^2 - (l_3 + d)^2}{2l_2}$$

Here I have written out the displacement relations which we have already discussed in a previous lecture the angle phi. The angle phi is this configuration this orientation of the end effector we have also seen in the displacement analysis that given theta and S4 we can find out xE yE and phi. So, we can find out everything.

Now, if you eliminate phi between these two relations we ultimately arrive at a relation involving xE yE and theta here this a b and c they are functions of xE and yE you can interpret this as given xE and yE. It is a relation for theta it is an equation from where we can solve for theta. So, given xE and yE we can solve for theta.

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$$\frac{d}{dt} [A \sin \theta + B \cos \theta = C]$$
 where

$$A = y_E, \quad B = x_E$$

$$C = \frac{x_E^2 + y_E^2 + l_2^2 - (l_3 + d)^2}{2l_2}$$

$$\dot{A} s \theta + A \dot{\theta} c \theta + \dot{B} c \theta - B \dot{\theta} s \theta = \dot{C}$$

$$\dot{y}_E s \theta + y_E \dot{\theta} c \theta + \dot{x}_E c \theta - x_E \dot{\theta} s \theta = \frac{1}{2l_2} (2x_E \dot{x}_E + 2y_E \dot{y}_E)$$

Now, what we are going to do with this relation is that we are going to differentiate this with respect to time. So, this relation we are going to differentiate with respect to time then, what we have is A dot sin theta S theta represents sin theta. This A theta dot cosine theta is B dot cosine theta minus B theta dot sine theta is equal to C dot.

Now, if you use these expressions of A B and C, then you have these relations; this is equal to now I can collect terms of xC dot yE dot and theta dot from here. I can collect terms of xE dot yE dot and theta dot and write in a simplified form which I will show you directly.

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$A \sin \theta + B \cos \theta = C$

where

$$A = y_E, \quad B = x_E$$

$$C = \frac{x_E^2 + y_E^2 + l_2^2 - (l_3 + d)^2}{2l_2}$$

Time differentiating both sides

$$P_1 \dot{x}_E + Q_1 \dot{y}_E = R_1 \dot{\theta}$$

where

$$P_1 = \left(\cos \theta - \frac{x_E}{l_2} \right), \quad Q_1 = \left(\sin \theta - \frac{y_E}{l_2} \right)$$

$$R_1 = x_E \sin \theta - y_E \cos \theta$$

So, I obtain a relation like this by collecting terms of \dot{x}_E , \dot{y}_E and $\dot{\theta}$. So, here P_1 is this expression Q_1 and R_1 which you can very easily derive by simplifying the steps that I have shown you. So, this equation relates \dot{x}_E , \dot{y}_E and $\dot{\theta}$, but this is not enough I need one more relation I must also relate \dot{s}_4 to \dot{x}_E and \dot{y}_E .

So, let us move on so, here I have one relation next, I must bring in \dot{s}_4 dot.

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$(x_B - l_1)^2 + y_B^2 = s_4^2$

where

$$x_B = x_E - d \cos \phi, \quad y_B = y_E - d \sin \phi$$

$$\Rightarrow s_4 = \sqrt{(x_E - d \cos \phi - l_1)^2 + (y_E - d \sin \phi)^2}$$

So, that is what we are going to look at now from the previous expressions. So, if you look at the coordinates of point B. So, coordinates of point B x_B and y_B and the distance

between B and Q is nothing but S4. So, therefore, the distance square this is the distance square. So, this is Q B square is equal to S4 square and x b can be related to xE and phi similarly y B can be related to yE and phi.

So, if you use these expressions S4 can be related to xE yE, but you also have phi now phi remember is the orientation angle of the end effector.

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$(x_B - l_1)^2 + y_B^2 = s_4^2$
 where
 $x_B = x_E - d \cos \phi, \quad y_B = y_E - d \sin \phi$
 $\Rightarrow s_4 = \sqrt{(x_E - d \cos \phi - l_1)^2 + (y_E - d \sin \phi)^2}$

Time differentiating both sides

$$\dot{s}_4 = P_2 \dot{x}_E + Q_2 \dot{y}_E + R_2 \dot{\phi}$$

(x_E, y_E, θ)

$$\tan \phi = \frac{y_E - l_2 \sin \theta}{x_E - l_2 \cos \theta}$$

$$\sin \phi = \frac{y_E - l_2 \sin \theta}{\sqrt{(y_E - l_2 \sin \theta)^2 + (x_E - l_2 \cos \theta)^2}}$$

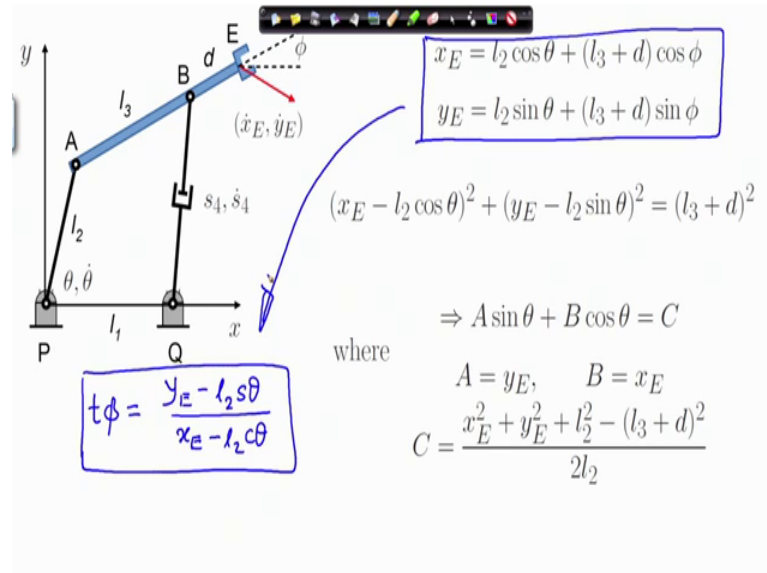
$$\cos \phi = \frac{x_E - l_2 \cos \theta}{\sqrt{(y_E - l_2 \sin \theta)^2 + (x_E - l_2 \cos \theta)^2}}$$

Now, if you differentiate with respect to time both sides of this equation then you can imagine that you will have derivatives of xE you will have derivatives of phi and you will have derivatives of yE. And the coefficients corresponding coefficients are P2 Q2 and R2 which can be very Easily found by differentiating this and this is going to be little complicated.

But it is otherwise straightforward I mean there will be a little bit of algebra involved. So, the steps are straightforward it will just differentiate the expression of S4 with respect to time. Now, we have involved phi. So, let us see how we can find out phi and we have also involved remember this cosine phi and sine phi we also have this cosine phi and sine phi as well as phi dot. So, this P2 Q2 R2 these will be functions of xE yE and phi which are the configuration which specify the configuration of the manipulator ok. Now, this tan phi you can very easily write tan phi in terms of theta.

So, tangent of this angle comes from the displacement relations that we have looked at before.

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So, these are the relations you can find out tangent phi. So, tan phi is as equal to yE minus l2 sin theta by xE minus l2 cosine theta right from here we can obtain tangent phi and this is what we are going to use here. So, this is the expression of tan phi.

Now, once I have tan phi I can also find out cosine phi and sine phi. So, sine phi is this expression. So, how do I find this suppose this angle is phi then tan phi is this ratio? So, if I say that this is yE minus l2 sin theta and this base of the triangle is xE minus l2 cosine theta then you can very easily see that tangent phi is this ratio.

So, this hypotenuse of this right-angle triangle is nothing but square root of the base square and the height square. So, therefore, you can relate sine phi and cosine phi which I have written out now in these relations what I have done therefore, is I have replaced sine phi and cosine phi in terms of xE yE and theta.

So, once I have done that these terms P minus Q2 and R2 they are only functions of xE yE and theta P2 Q2 and R2 are functions of xE yE and theta and this relation is between the velocities S4. So, S4 dot xE dot ye dot and phi dot I have eliminated phi the next step is to eliminate phi dot then I would have a relation between xE dot yE dot and S4 dot now how to determine phi dot.

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Diagram showing a mechanism with points P, A, B, Q, E. Lengths are labeled l_1, l_2, l_3, d . Angles are θ, ϕ . Velocity components are \dot{x}_E, \dot{y}_E . A right angle is shown at B with s_4, \dot{s}_4 .

$$\dot{s}_4 = P_2 \dot{x}_E + Q_2 \dot{y}_E + R_2 \dot{\phi}$$

$$x_E = l_2 \cos \theta + (l_3 + d) \cos \phi$$

$$y_E = l_2 \sin \theta + (l_3 + d) \sin \phi$$

$$\frac{d}{dt} \left[\tan \phi = \frac{y_E - l_2 \sin \theta}{x_E - l_2 \cos \theta} \right]$$

$$\sec^2 \phi \dot{\phi} = () \dot{x}_E + () \dot{y}_E + () \dot{\theta}$$

$$(1 + \tan^2 \phi) \dot{\phi} = () \dot{x}_E + () \dot{y}_E + () \dot{\theta}$$

$$\dot{\phi} = P_3 \dot{x}_E + Q_3 \dot{y}_E + R_3 \dot{\theta} \quad (x_E, y_E, \theta)$$

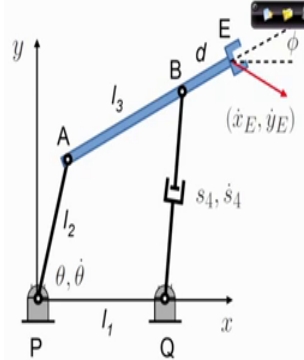
So, here I have rewritten that expression and I have also rewritten the displacement relation and this expression of tan phi. Now, if I differentiate with respect to time I have second square phi into phi dot is equal to the time derivative of the right-hand side of this ratio.

Now, if you think a little bit it will be absolutely clear that the time derivative of this ratio will have a form or can be expressed in the form something times xE dot plus something times yE dot the something times theta dot. Now, this time derivative is straightforward, but there will be a little bit of algebra in this.

Now, second square phi can be again written in terms of tangent phi. So, I have this relation now 10 tan phi is already known in terms of xE yE and theta. So, therefore, finally, I must have let me call it some P3 xE dot plus Q3 yE dot R3 theta dot where remember that P3 Q3 R3 are solely functions of xE yE theta.

So, these are solely functions of xE yE and theta. So, I have found phi dot in terms of xE dot yE dot theta dot xE yE theta.

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$\dot{s}_4 = P_2 \dot{x}_E + Q_2 \dot{y}_E + R_2 \dot{\phi}$
 $x_E = l_2 \cos \theta + (l_3 + d) \cos \phi$
 $y_E = l_2 \sin \theta + (l_3 + d) \sin \phi$
 $\tan \phi = \frac{y_E - l_2 \sin \theta}{x_E - l_2 \cos \theta}$

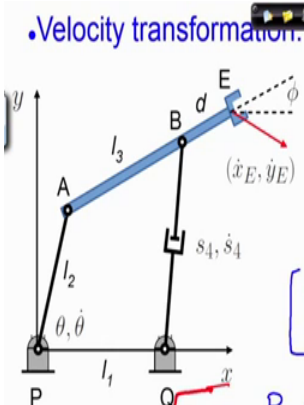
Time differentiating both sides

$$\dot{\phi} = P_3 \dot{x}_E + Q_3 \dot{y}_E + R_3 \dot{\theta}$$

Where this; P3 Q3 R3 are complicated expressions which you can easily find out by a little bit of algebra.

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• Velocity transformation. Jacobian



$P_1 \dot{x}_E + Q_1 \dot{y}_E = R_1 \dot{\theta}$
 $\dot{\phi} = P_3 \dot{x}_E + Q_3 \dot{y}_E + R_3 \dot{\theta}$
 $\dot{s}_4 = P_2 \dot{x}_E + Q_2 \dot{y}_E + R_2 \dot{\phi}$

$$\dot{s}_4 = (P_2 + R_2 P_3) \dot{x}_E + (Q_2 + R_2 Q_3) \dot{y}_E + R_2 R_3 \dot{\theta}$$

$P_1 \dot{x}_E + Q_1 \dot{y}_E = R_1 \dot{\theta}$
 $(P_2 + R_2 P_3) \dot{x}_E + (Q_2 + R_2 Q_3) \dot{y}_E = -R_2 R_3 \dot{\theta} + \dot{s}_4$

$\dot{\vec{x}}_E = [A]^{-1} [B] \dot{\vec{Y}}$
 $\dot{\vec{Y}} = [B]^{-1} [A] \dot{\vec{x}}_E$
 $[A] \dot{\vec{x}}_E = [B] \dot{\vec{Y}}$
 $\dot{\vec{x}}_E = \begin{Bmatrix} \dot{x}_E \\ \dot{y}_E \end{Bmatrix}, \dot{\vec{Y}} = \begin{Bmatrix} \dot{\theta} \\ \dot{s}_4 \end{Bmatrix}$

So, here I have collated all the expressions that we had derived we are derived this first and then these 2 expressions were derived.

Now, here I have phi dot which appears in the expression of S4 dot. So, therefore, I can eliminate phi dot in the expression of S4 dot and write this as. So, when I substitute phi

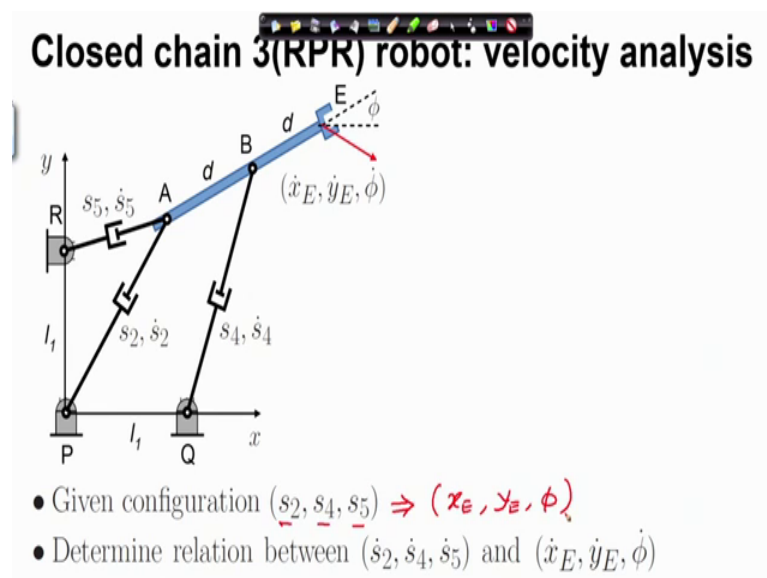
dot here I will get this times x_E dot plus this terms times y_E dot plus $R_2 R_3$ times theta dot.

Now, I can rewrite this as. So, let me write out the first equation here. So, $P_1 x_E$ dot plus $Q_1 y_E$ dot is R_1 theta dot and now I will write this equation after a little bit of manipulation as that is equal to minus of $R_2 R_3$ theta dot plus S_4 dot. Now, this I can combine and write in a compact form as let us say some matrix a times x_E dot is equal to matrix B times the vector y dot where x_E dot of course, this vector x_E dot is nothing but x_E dot y dot the vector y dot is theta dot S_4 dot.

So, the joint velocity vector and the matrices A and B you can very easily read out from these 2 equations. So, this is our velocity relation now it looks a little different than what we had derived earlier, but you can always invert a and rewrite this as such that this is our Jacobean for the manipulator. So, $a^{-1} b$ is the Jacobean for this manipulator.

If you invert b then you have the inverse Jacobean. So, if you write y dot as B^{-1} times a x_E dot then you have the inverse Jacobean. So, we have finally, determined the velocity relations. So, our joint velocity vector is theta dot S_4 dot. So, from here I have related I found a relation to x_E dot and y_E dot which is the end effector velocity.

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So, let us move forward and discuss the 3 RPR planar closed chain robot manipulator here we need to recall that we had already discussed the displacement analysis the

configurations s_2, s_4, s_5 . So, given s_2, s_4, s_5 the configuration of the manipulator is specified and we have we had found x_E, y_E and ϕ .

So, given s_2, s_4, s_5 you can find out x_E, y_E and ϕ . So, this is a 3 degree of freedom manipulator. So, f 3 inputs s_2, s_4 and s_5 and we can find out s_E, y_E, x_E, y_e and ϕ .

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Closed chain 3(RPR) robot: velocity analysis

- Given configuration (s_2, s_4, s_5)
- Determine relation between $(\dot{s}_2, \dot{s}_4, \dot{s}_5)$ and $(\dot{x}_E, \dot{y}_E, \dot{\phi})$

Now, the velocity analysis requires the specification of the relation between s_2 dot s_4 dot s_5 dot and x_E dot y_E dot and ϕ dot. So, that is the velocity analysis problem.

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$$\frac{d}{dt} [s_2 = \sqrt{x_A^2 + y_A^2}]$$

$$s_4 = \sqrt{(x_B - l_1)^2 + y_B^2}$$

$$s_5 = \sqrt{x_A^2 + (y_A - l_1)^2}$$

$$x_A = x_E - 2d \cos \phi, \quad y_A = y_E - 2d \sin \phi$$

$$x_B = x_E - d \cos \phi, \quad y_B = y_E - d \sin \phi$$

$$\dot{s}_2 = \frac{1}{\sqrt{x_A^2 + y_A^2}} [x_A \dot{x}_A + y_A \dot{y}_A]$$

$$= (-) \dot{x}_E + (-) \dot{y}_E + (-) \dot{\phi}$$

(x_E, y_E, ϕ)

$$\dot{x}_A = \dot{x}_E + 2d \dot{\phi} \sin \phi$$

$$\dot{y}_A = \dot{y}_E - 2d \dot{\phi} \cos \phi$$

Now, we have derived these relations these are very Easy to see S_2 is the throw of this actuator. Similarly, S_4 and S_5 and they are related in terms of the coordinates of point A and B and the coordinates of P Q and R. So, these relations we have already looked at. So, here you can relate x_A y_A . So, coordinates of point A and similarly coordinates of point B.

In terms of the coordinates of the end effector and the orientation of the end effector ϕ so, therefore, you can just substitute them here and start differentiating. So, for example; if you differentiate the first equation which I have written out after some simplification so, this is what you have and now you can replace x_A y_A in terms of x_E y_E and ϕ and you can find out \dot{x}_A as \dot{x}_E dot plus $2 d \phi$ dot sine ϕ .

Similarly, \dot{y}_A is \dot{y}_E dot minus $2 d \phi$ dot cosine ϕ . So, these also you can replace. So, finally, \dot{S}_2 dot will be something times \dot{x}_E dot plus something times \dot{y}_E dot something times ϕ dot. Now, these bracketed quantities can be easily determined after some algebra you substitute these expressions of x_A y_A \dot{x}_A dot and \dot{y}_A dot and you can find out this relation.

Now, this bracketed quantities are functions of only x_E y_E and ϕ . So, this bracketed quantities are functions of x_E y_E and ϕ . So, if you know the configuration of the manipulator these things are already known to you. So, which means this bracketed quantities are known to you in that case I have found a relation between the expansion rate of s^2 which is \dot{s}^2 and the velocity of the end effector which is \dot{x}_E dot \dot{y}_E dot and ϕ dot.

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$s_2 = \sqrt{x_A^2 + y_A^2}$
 $s_4 = \sqrt{(x_B - l_1)^2 + y_B^2}$
 $s_5 = \sqrt{x_A^2 + (y_A - l_1)^2}$

$x_A = x_E - 2d \cos \phi, \quad y_A = y_E - 2d \sin \phi$
 $x_B = x_E - d \cos \phi, \quad y_B = y_E - d \sin \phi$

Time differentiating both sides

$\dot{x}_A = \dot{x}_E + (2d \sin \phi) \dot{\phi} \quad \dot{y}_A = \dot{y}_E - (2d \cos \phi) \dot{\phi}$
 $\dot{x}_B = \dot{x}_E + (d \sin \phi) \dot{\phi} \quad \dot{y}_B = \dot{y}_E - (d \cos \phi) \dot{\phi}$

Now, if you continue. So, this is the time derivative.

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$s_2 = \sqrt{x_A^2 + y_A^2}$
 $s_4 = \sqrt{(x_B - l_1)^2 + y_B^2}$
 $s_5 = \sqrt{x_A^2 + (y_A - l_1)^2}$

Time differentiating both sides

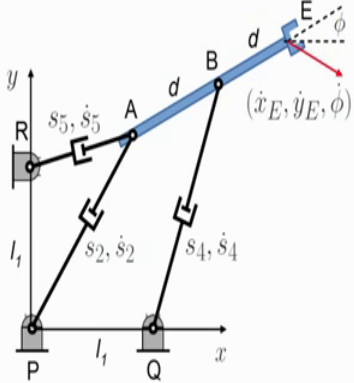
$\dot{s}_2 = P_1 \dot{x}_E + Q_1 \dot{y}_E + R_1 \dot{\phi}$
 $\dot{s}_4 = P_2 \dot{x}_E + Q_2 \dot{y}_E + R_2 \dot{\phi}$
 $\dot{s}_5 = P_3 \dot{x}_E + Q_3 \dot{y}_E + R_3 \dot{\phi}$

$$\begin{Bmatrix} \dot{s}_2 \\ \dot{s}_4 \\ \dot{s}_5 \end{Bmatrix} = [J] \begin{Bmatrix} \dot{x}_E \\ \dot{y}_E \\ \dot{\phi} \end{Bmatrix}$$

So, if you substitute all these expressions and finally, you obtain these relations between \dot{s}_2 , \dot{s}_4 , \dot{s}_5 and \dot{x}_E , \dot{y}_E and $\dot{\phi}$. So, which means I have \dot{s}_2 , \dot{s}_4 , \dot{s}_5 is equal to Jacobean matrix times \dot{x}_E , \dot{y}_E and $\dot{\phi}$ and that is the velocity relation that we are seeking the Jacobean can be easily read out from these detailed relations.

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•Velocity transformation. Jacobian


$$\begin{aligned}\dot{s}_2 &= P_1\dot{x}_E + Q_1\dot{y}_E + R_1\dot{\phi} \\ \dot{s}_4 &= P_2\dot{x}_E + Q_2\dot{y}_E + R_2\dot{\phi} \\ \dot{s}_5 &= P_3\dot{x}_E + Q_3\dot{y}_E + R_3\dot{\phi}\end{aligned}$$

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Summary

- Velocity analysis of closed chain manipulators
- Concept of Jacobian for velocity transformation
- Example of 2R-RPR and 3(RPR) parallel manipulators



So, to summarize what we have discussed in this lecture we have looked at the closed chain parallel planar manipulators we looked at the velocity analysis problem. We have found the Jacobean of the Jacobean matrix for these manipulators I have outlined methods for determining this Jacobean. We have looked at these 2 examples of 2 R RPR and 3 RP are closed chain planar parallel manipulators.

So, with this we have now discussed the velocity analysis of both open chain and closed chain manipulators now this space wave for discussions on motion planning or path

planning a path generation problem for these manipulators. So, this lecture I will close here.