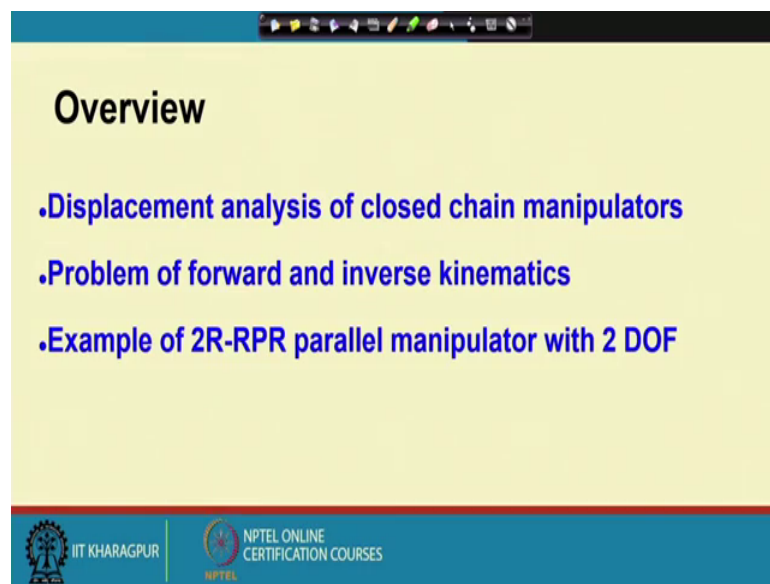


Mechanism and Robot Kinematics
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Lecture – 18
Displacement Analysis: Closed Chain Robot – I

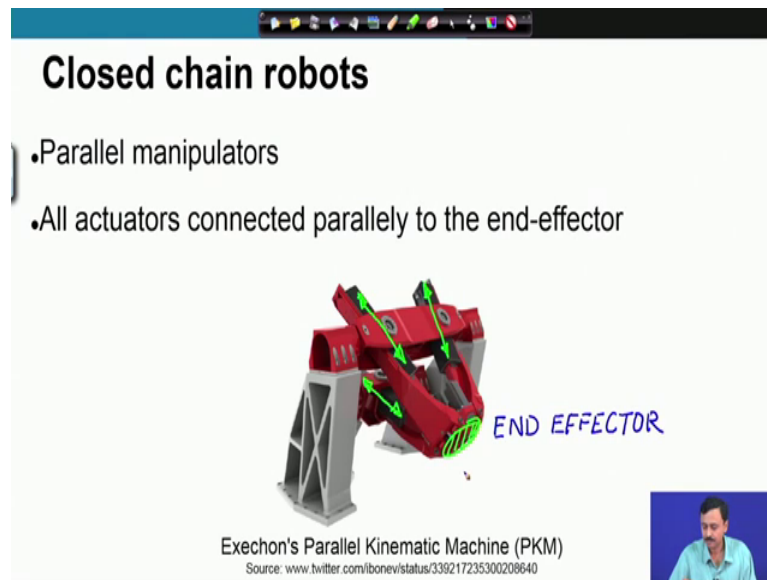
In this lecture, we are going to look at the displacement analysis problem of closed loop parallel manipulators.

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So, to give you the overview of what we are going to discuss in this lecture, we are going to look at the displacement analysis problem of closed chain manipulators, problem of forward and inverse kinematics of a 2R-RPR parallel manipulator with 2 degree of freedom. So, here we have a nomenclature which I am going to explain.

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So, what are closed chain robots? They are also known as parallel manipulators. So, what are these? Closed chain robots. In normal robots as we have an idea of we talk of serial chain manipulators. So, for example, my hand you can consider is a serial chain why do we consider this as serial chain? Because the actuators and the joints, they appear serially in the chain. So, this is the chain of my hand; so, here there is one actuator joint which is actuated, here is another joint which is actuated and they come serially. In a serial manipulator therefore, the end-effector which is my hand is connected with the links through these joints in a serial manner.

As opposed to this in a parallel manipulator, we have all the links which are actuated; connected to the end-effector directly parallelly. So, that is why you also use this term parallel manipulator. So, all actuators are connected parallelly to the end-effector. So, here I have this example of exechons parallel kinematic machine which is actually used for machining operations. So, let us understand why this is a parallel manipulator; here you can see this is one actuator. This is the second actuator and underneath; this is the third actuator and all these actuators are connected to the end-effector, so this is the end-effector.

This is the end-effector where the machining tool or the gripper will be connected. So, all these actuators parallelly connect to the end-effector and as you can very easily see that there are no singular links as expected in a closed chain closed kinematic chain. So, there

is no singular link no link with only one kinematic pair. So, we have a closed chain robot in which all actuators connect parallelly to the end-effector.

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Robot displacement analysis: plan

- Planar robots: open chain
- Planar robots: closed chain

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So, as per our plan we have discussed open chain thinner robots previously. So, in this lecture, we are going to start with closed chain planar robots.

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Closed chain planar robots: 2 DOF

• Kinematic chains: $2R-3R$, $RP-RPR$, $2R-RPR$

$n_L = 5$ $n_J = 5$ $\sum f_i = 5$
 $F = 3(5-1) - 3(5) + 5 = 2$

So, there can be various kinds of chains, let me explain this nomenclature and draw out. So, we have one link which is ground and the other link which is the end-effector. Now in this nomenclature like $2R$ dash $3R$; this $2R$ stands for one of the legs of this parallel

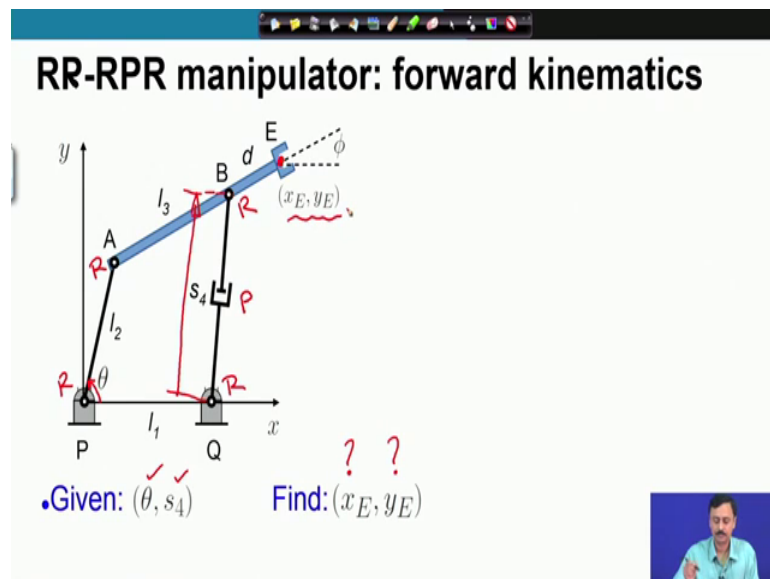
manipulator. So, therefore, this leg the 2R leg is like this. So, RR. So, you have R here and R here and the other leg is a 3R leg and this one is your end-effector.

So, if you want to calculate the degree of freedom. So, this ground is 1, 2, 3, 4, 5. So, number of links 5 number of joints 1, 2, 3, 4, 5 and summation of degree of freedom of each joint, since, they are all revolute; there are 5 revolute. So, summation of degree of freedom is 5. So, therefore, degree of freedom is 3 times number of links minus one minus 3 times number of joints plus summation of degree of freedom of each joint.

So, this turns out to be 2. So, this has 2 degrees of freedom. So, that is why this is a robot there is no longer a constraint mechanism. So, it will require 2 joints to be actuated. So, possibly this joint and this joint. So, the 2 ground revolute pairs can be actuated. So, 2 joints will be required to be actuated.

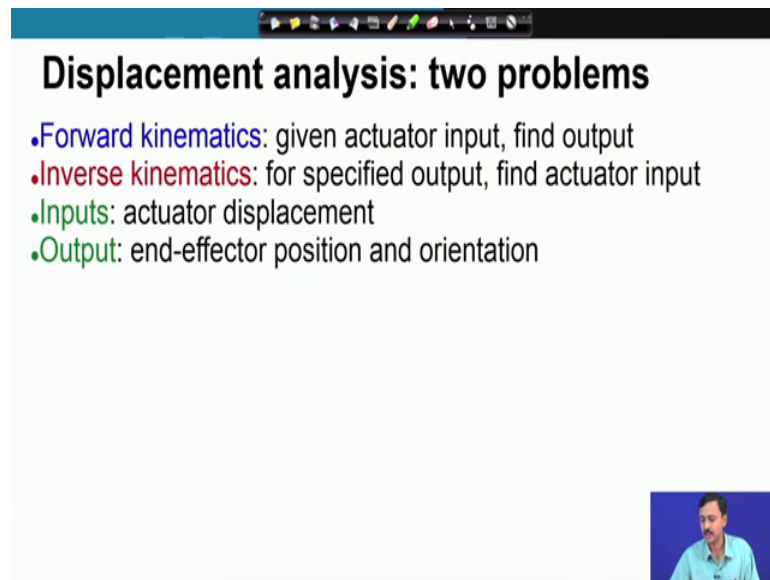
RPRPR; the next chain. So, once again we have a ground and an end-effector link. So, we have R and P which is here it is welded. So, RP and the other leg is RPR. So, this is the RPR leg here also you can calculate the degree of freedom it will turn out to be 2 the next chain is 2R-RPR which we are going to study.

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So, this is the 2R-RPR which also has 2 degrees of freedom as you can easily check.

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Displacement analysis: two problems

- **Forward kinematics:** given actuator input, find output
- **Inverse kinematics:** for specified output, find actuator input
- **Inputs:** actuator displacement
- **Output:** end-effector position and orientation

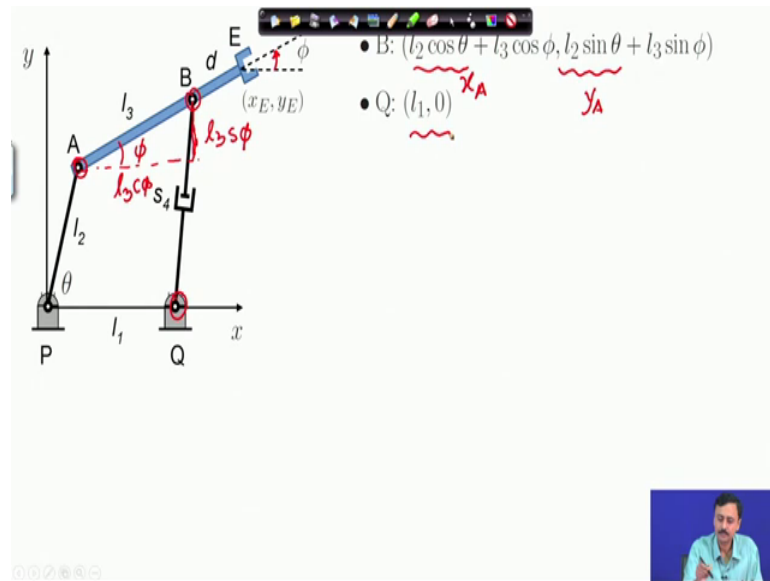
Video inset: A man in a light blue shirt speaking.

Now we have two kinds of problems as you know the forward kinematics problem in which the actuator inputs are given we have to find out the output; output is the end-effector position or position and orientation depending on degree of freedom of the chain.

So, an inverse kinematics problem for a specified output; that means, the position and orientation of the end-effector adjust the position of the end-effector, we have to find out the actuator input or inputs. So, in this RPR PR; so, here we have this R R RPR, so this actually is RR RPR chain. So, we are going to discuss the forward kinematics problem of this RR RPR chain.

So, here you have RR RPR and the forward kinematics problem we are specified theta which is this angle and the throw of the prismatic actuator which is this length which is S₄. So, we are given theta and S₄ these are to be actuated we have to find out x_E and y_E which are the coordinates of the end-effector point we have to find this in the forward kinematics problem.

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So, let us look at how we go about doing this. So, the point B here you see we have this point B whose coordinates you can now very easily find out is $l_2 \cos \theta$ which is the coordinate of point A. So, this is the x coordinate of point A plus $l_3 \cos \phi$ of this angle ϕ is an orientational coordinate. So, this gives the orientation of the end-effector link with the datum; the x axis. So, that is ϕ ; so, I relate the coordinates of point B in terms of θ and this ϕ I have brought in this additionally which I will show you how to calculate. So, the first term $l_2 \cos \theta$ is the x coordinate of point A plus $l_3 \cos \phi$ is a this projection. So, that is the x coordinate of point B the y coordinate of point B; this is the y coordinate of point A and to that I add the y projection of a B that is $l_3 \sin \phi$ this is $l_3 \sin \phi$ and this is $l_3 \cos \phi$.

And we have this coordinates of point Q as $(l_1, 0)$.

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• B: $(l_2 \cos \theta + l_3 \cos \phi, l_2 \sin \theta + l_3 \sin \phi)$
 • Q: $(l_1, 0)$

$$\rightarrow (l_2 \cos \theta + l_3 \cos \phi - l_1)^2 + (l_2 \sin \theta + l_3 \sin \phi)^2 = s_4^2$$

$$s_4^2 = (x_B - x_Q)^2 + (y_B - y_Q)^2$$

Therefore the length s_4 I can express. So, s_4 square is nothing, but x_B minus x_Q square plus y_B minus y_Q square. Now if you substitute these expressions the coordinates of point B and Q, then you come to this expression and when you open, this up and arrange the terms then you can simplify this equation remembering that we are given θ and s_4 and the unknown here in this equation is ϕ we are given θ and this s_4 the only thing that is unknown is ϕ therefore, I can assemble this equation I can simplify this equation and assemble it in the form.

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• B: $(l_2 \cos \theta + l_3 \cos \phi, l_2 \sin \theta + l_3 \sin \phi)$
 • Q: $(l_1, 0)$

$$(l_2 \cos \theta + l_3 \cos \phi - l_1)^2 + (l_2 \sin \theta + l_3 \sin \phi)^2 = s_4^2$$

$$\Rightarrow A \sin \phi + B \cos \phi = C$$

where

$$A = -\sin \theta, \quad B = \left(\frac{l_1}{l_2} - \cos \theta\right)$$

$$C = \frac{l_1^2 + l_2^2 + l_3^2 - s_4^2 - 2l_1 l_2 \cos \theta}{2l_2 l_3}$$

Some $A \sin \phi + B \cos \phi = C$ which you can easily do where you will find that this A , B and C are completely known because I know θ and I know $S 4$. So, therefore, A , B and C are completely known to me; so, what is unknown is ϕ which I need to solve from this equation.

So, I need to solve this equation in order to find ϕ as discussed previously.

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Solution of $A \sin \phi + B \cos \phi = C$

Let $x = \tan \frac{\phi}{2}$. Then

$$\sin \phi = \frac{2x}{1+x^2} \quad \cos \phi = \frac{1-x^2}{1+x^2}$$

Substituting in the equation yields

$$A \left(\frac{2x}{1+x^2} \right) + B \left(\frac{1-x^2}{1+x^2} \right) = C$$
$$\Rightarrow (B+C)x^2 - 2Ax + (C-B) = 0$$

We will take this approach which can be very easily programmed on a computer and you can get all the solutions of this equation $A \sin \phi + B \cos \phi = C$. So, in that we make a definition x equal to $\tan \phi$ by 2 and represent $\sin \phi$ and $\cos \phi$ in terms of x which when substituted into our master equation finally, gives us this quadratic equation in x whose roots, we can now easily find out and hence we can find out $\tan \phi$ by 2 and that is what we are going to do.

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$$(B + C)x^2 - 2Ax + (C - B) = 0$$

Solutions are

$$x = \tan \frac{\phi}{2} = \frac{A \pm \sqrt{A^2 + B^2 - C^2}}{B + C}$$

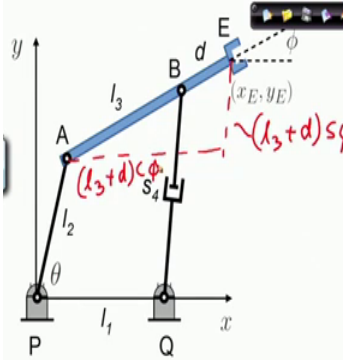
where

$$A = -\sin \theta, \quad B = \left(\frac{l_1}{l_2} - \cos \theta\right)$$

$$C = \frac{l_1^2 + l_2^2 + l_3^2 - s_4^2 - 2l_1 l_2 \cos \theta}{2l_2 l_3}$$


So, the solution solutions of this quadratic equation you have these two solutions given by these 2 signs positive and negative. So, we get two solutions of x and hence two solutions of phi A, B and C are completely known.

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The diagram shows a mechanism with joints P, Q, A, B, E. Lengths are labeled l_1, l_2, l_3, d . Angles are θ and ϕ . A point (x_E, y_E) is marked. Handwritten notes include $(l_3 + d) \cos \phi$ and $(l_3 + d) \sin \phi$.

$$\tan \frac{\phi}{2} = \frac{A \pm \sqrt{A^2 + B^2 - C^2}}{B + C}$$

$$\phi_1 = 2 \tan^{-1} \left[\frac{A - \sqrt{A^2 + B^2 - C^2}}{B + C} \right]$$

$$\phi_2 = 2 \tan^{-1} \left[\frac{A + \sqrt{A^2 + B^2 - C^2}}{B + C} \right]$$

$$x_E = \underbrace{l_2 \cos \theta}_{x_A} + \underbrace{(l_3 + d) \cos \phi}_{x_{AE}}$$

$$y_E = \underbrace{l_2 \sin \theta}_{y_A} + \underbrace{(l_3 + d) \sin \phi}_{y_{AE}}$$

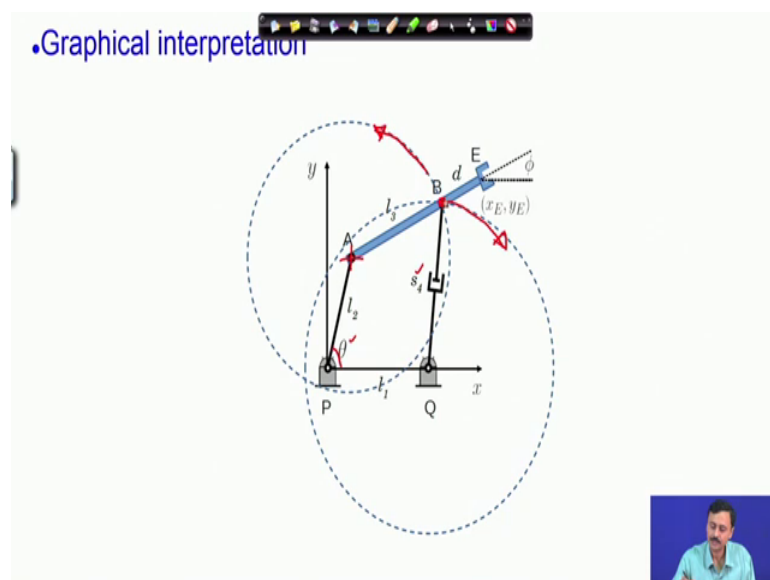
•Use atan2(y,x) function for correct quadrant

So, therefore, we have this tan phi by 2 expression in terms of A, B, C, these are the 2 solutions once again you need to use this atan2 function. So, that you get the correct quadrant of phi one and phi 2.

And finally, what we set out to calculate was the coordinates of this end-effector. So, x_E and y_E . So, x_E coordinate of the end-effector is $l_3 \cos \theta$ which is nothing, but x coordinate of point A and this part the second term in the expression of x_E which is $l_3 \cos \theta + d \cos \phi$ is nothing, but the vector a_E the x coordinate of the vector a_E . So, I will be I will write it like this that this is the x coordinate of a_E . Similarly in the expression of y_E you have $l_2 \sin \theta$ which is the y coordinate of a and the second term is nothing, but the y projection of this a_E which is $l_3 \sin \theta + d \sin \phi$.

So, this is $l_3 \cos \theta + d \cos \phi$ and this is $l_3 \sin \theta + d \sin \phi$. So, that is x_E and y_E . So, we have obtained the coordinates of the end-effector point E let us understand.

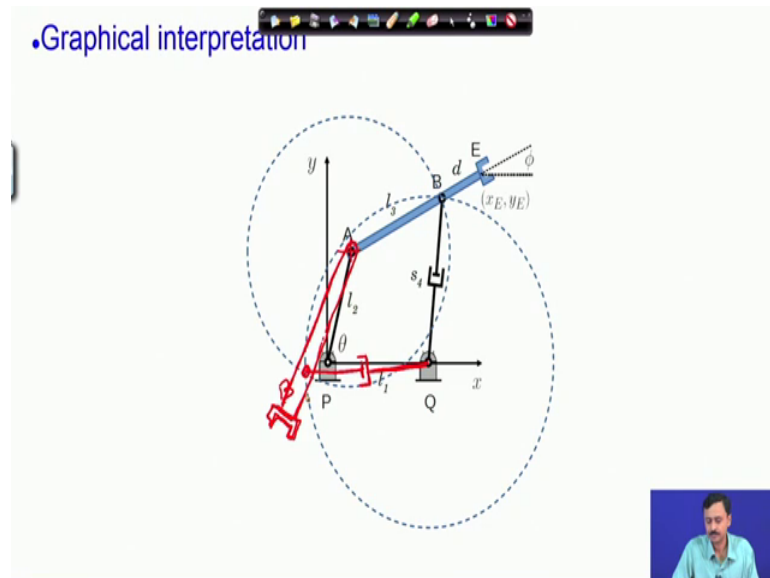
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The solution graphically remember that we are given θ and s_4 . So, therefore, θ and s_4 are given. Now if you see when θ is given then this point A gets fixed what is not fixed is ϕ because this hinge be on the end-effector link can rotate on this circle while the hinge be on the actuator arm on this on this other leg can rotate on this circle.

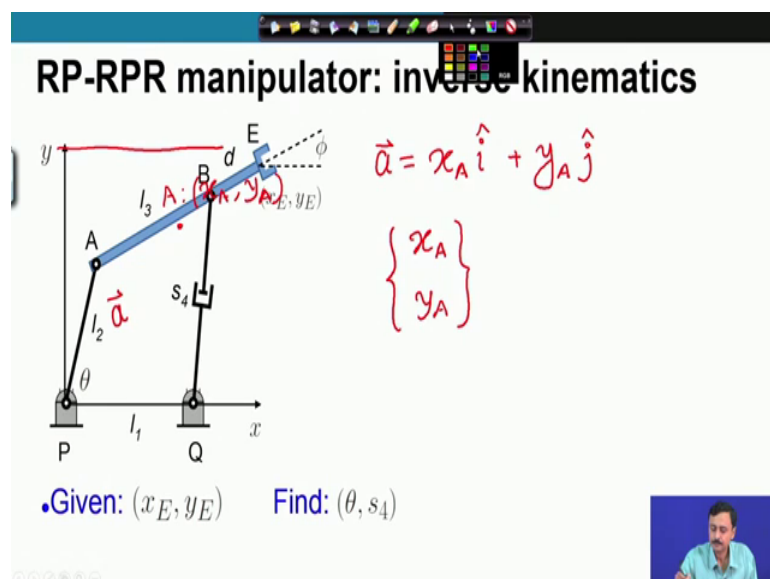
So, the way to assemble the mechanism is where these 2 circles intersect for example, this is one intersection point.

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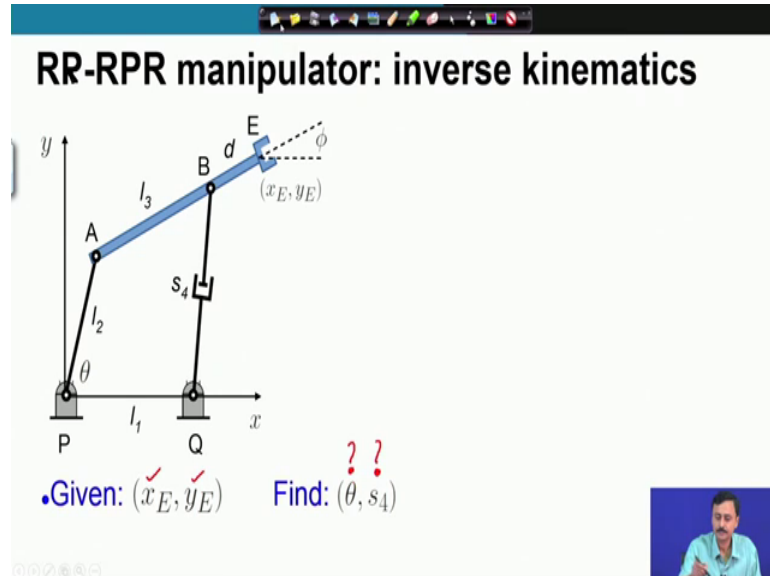
So, you have one configuration that is already shown there is another solution, this is given by this your hinge B can also lie here. So, therefore, the mechanism in this configuration will look like this. So, in the red configuration because point A is fixed remember because theta is given since theta is specified a gets fixed and hence you have another assembly mode of this mechanism as shown by this red configuration. So, these are the 2 solutions.

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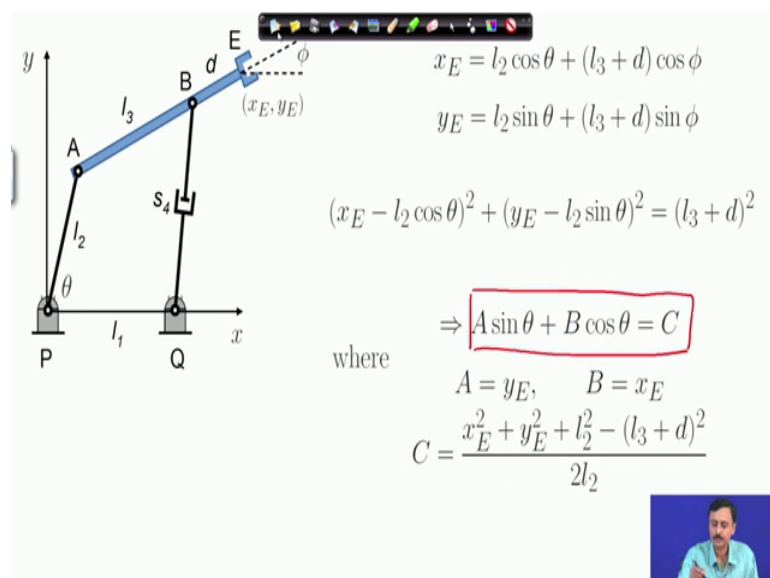
Now, let us move on with the. So, this is the RR RPR manipulator and we are going to study the inverse kinematics of this chain.

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Now here we are given the coordinates of the end-effector and we are to find out the inputs; the actuator inputs which are given by theta and S 4. So, here I have written out.

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The forward kinematic solution you remember we are derived these expressions of x E and y e. So, we start with the forward kinematic solution or relations if I take this term.

So, so what I am given I am given this x and y . So, these are known to me x and y are known to me what I have to find out is θ let us say the first thing is θ .

So, from these 2, I can eliminate ϕ and this is what I have done in the next step. So, I have taken these terms to the left hand side and squared and added them to eliminate ϕ . So, ϕ is completely eliminated in this equation. So, what I am left with we have in this equation x and y which are completely known and what is not known is θ .

Now if you open up this expression on the left hand side and simplify then you can very easily arrive at this form. So, remember we have to find out θ and these terms A and C they are completely known because y and x these are given to us. So, we need to solve this equation in order to solve for θ .

So, this is a standard equation which we have been solving.


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Solution of $A \sin \theta + B \cos \theta = C$

Let $x = \tan \frac{\theta}{2}$. Then

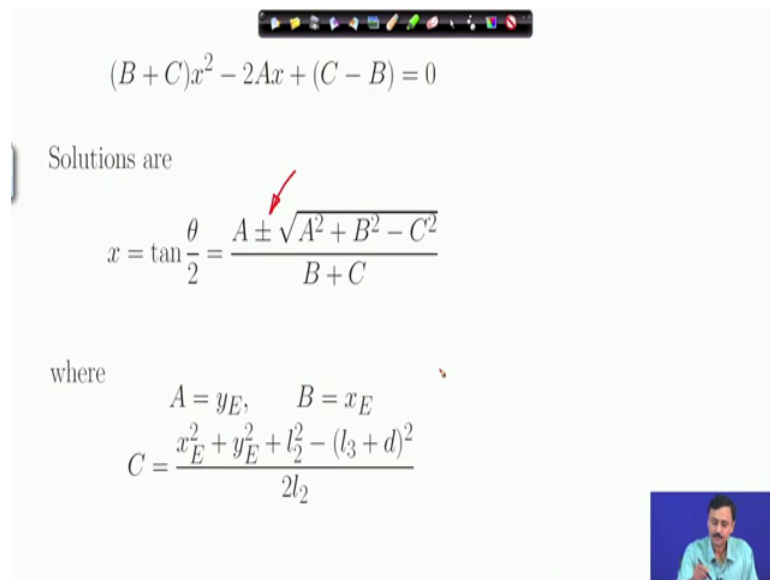
$$\sin \theta = \frac{2x}{1+x^2} \quad \cos \theta = \frac{1-x^2}{1+x^2}$$

Substituting in the equation yields

$$A \left(\frac{2x}{1+x^2} \right) + B \left(\frac{1-x^2}{1+x^2} \right) = C$$
$$\Rightarrow (B+C)x^2 - 2Ax + (C-B) = 0$$


So, once again just to reiterate what we have done we have defined this x in terms of $\tan \theta$ by 2 express sine θ and cosine θ in terms of x substituted into the equation that we want to solve and finally, obtain this quadratic equation which has solutions in terms of A , B , C which are completely known to us we have 2 solutions as you can see again here.

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$$(B + C)x^2 - 2Ax + (C - B) = 0$$

Solutions are

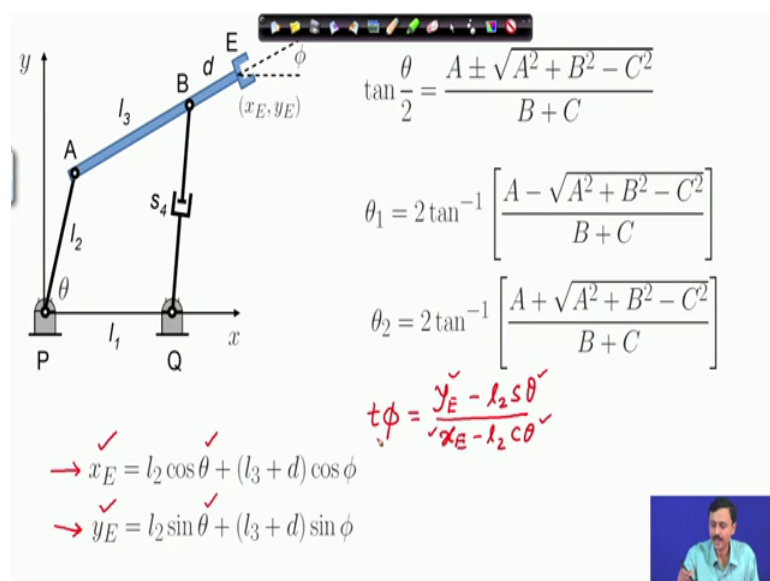
$$x = \tan \frac{\theta}{2} = \frac{A \pm \sqrt{A^2 + B^2 - C^2}}{B + C}$$

where

$$A = y_E, \quad B = x_E$$

$$C = \frac{x_E^2 + y_E^2 + l_2^2 - (l_3 + d)^2}{2l_2}$$

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$$\tan \frac{\theta}{2} = \frac{A \pm \sqrt{A^2 + B^2 - C^2}}{B + C}$$

$$\theta_1 = 2 \tan^{-1} \left[\frac{A - \sqrt{A^2 + B^2 - C^2}}{B + C} \right]$$

$$\theta_2 = 2 \tan^{-1} \left[\frac{A + \sqrt{A^2 + B^2 - C^2}}{B + C} \right]$$

$$t\phi = \frac{y_E - l_2 s\theta}{x_E - l_2 c\theta}$$

$$\rightarrow x_E = l_2 \cos \theta + (l_3 + d) \cos \phi$$

$$\rightarrow y_E = l_2 \sin \theta + (l_3 + d) \sin \phi$$

So, once we have these solutions we can obtain theta the 2 solutions of theta, theta 1 and theta 2 in terms of tangent inverse of this expression. So, for that again we need to use the A tan 2 function.

Now, once I have found theta I need to find out S 4. So, to find out S 4, we take recourse to these steps first I will again look at these relations the forward kinematics relations which we have used now. Now we know theta x E and y E are of course, given we are now solved for theta from these 2 equations we can now solve for phi. So, we find out

tangent phi. So, tan phi is nothing, but y E minus l 2 sine theta by x E minus l 2 cosine theta.

Now, since I know theta and know x E and y e. So, I can calculate phi. So, formally. So, this is the expression for tan phi.

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$$\tan \frac{\theta}{2} = \frac{A \pm \sqrt{A^2 + B^2 - C^2}}{B + C}$$

$$\theta_1 = 2 \tan^{-1} \left[\frac{A - \sqrt{A^2 + B^2 - C^2}}{B + C} \right]$$

$$\theta_2 = 2 \tan^{-1} \left[\frac{A + \sqrt{A^2 + B^2 - C^2}}{B + C} \right]$$

$$x_E = l_2 \cos \theta + (l_3 + d) \cos \phi$$

$$y_E = l_2 \sin \theta + (l_3 + d) \sin \phi$$

$$\Rightarrow \tan \phi = \frac{y_E - l_2 \sin \theta}{x_E - l_2 \cos \theta}$$

So, from here I can solve for phi; again using the A tan 2 function because I need to get the quadrant right.

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$$\theta_1 = 2 \tan^{-1} \left[\frac{A - \sqrt{A^2 + B^2 - C^2}}{B + C} \right]$$

$$\theta_2 = 2 \tan^{-1} \left[\frac{A + \sqrt{A^2 + B^2 - C^2}}{B + C} \right]$$

$$\tan \phi = \frac{y_E - l_2 \sin \theta}{x_E - l_2 \cos \theta}$$

$$\underline{x_B} = \underline{x_E} - \underline{d \cos \phi}, \quad \underline{y_B} = \underline{y_E} - \underline{d \sin \phi}$$

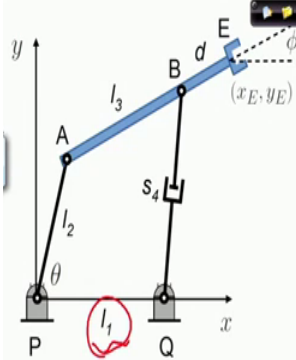
$$s_4^2 = (x_B - x_Q)^2 + (y_B - y_Q)^2$$



So, I have collected these expressions now. So, we know theta 1, theta 2 in terms of x E y E, then I calculate phi, once I have phi; I can define the coordinates of point B. So, coordinates of point B is nothing, but coordinates of point e which is the end-effector point which is given to me which is known minus this D cosine phi which is the projection of B e this is projection of B E along the x axis. So, this is d cosine phi similarly y B is equals to y E which is known to me minus d sine phi this is d sine phi the vertical projection of B E..

So, I know the coordinates of point B once I know coordinates of point B I also know coordinates of point Q therefore, I can now find out this length S 4 because S 4 square is equal to x B minus x Q whole square plus y B minus y Q whole square that is S 4 square.

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$$\theta_1 = 2 \tan^{-1} \left[\frac{A - \sqrt{A^2 + B^2 - C^2}}{B + C} \right]$$

$$\theta_2 = 2 \tan^{-1} \left[\frac{A + \sqrt{A^2 + B^2 - C^2}}{B + C} \right]$$

$$\tan \phi = \frac{y_E - l_2 \sin \theta}{x_E - l_2 \cos \theta}$$

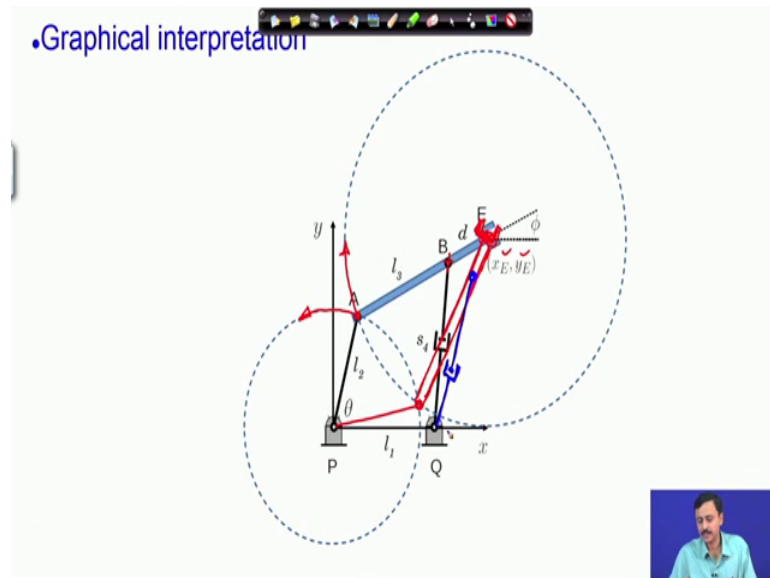
$$x_B = x_E - d \cos \phi, \quad y_B = y_E - d \sin \phi$$

$$(x_B - l_1)^2 + y_B^2 = s_4^2$$

$$\Rightarrow s_4 = \sqrt{(x_B - l_1)^2 + y_B^2}$$

So, from here I can find out the throw of this prismatic actuator. So, this is what I have written out. So, S 4 is square root of x B minus l 1. So, you have l one here the length P Q. So, x B minus l one whole square plus y B square because y Q is 0 the y coordinate of point Q is 0.

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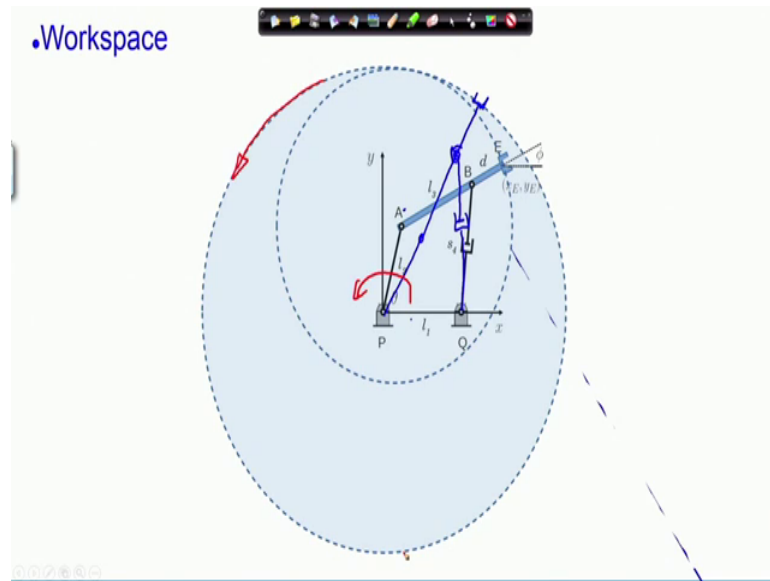


So, let us understand this solution graphically we have been given x_E and y_E . So, this point E is fixed what is not fixed is this hinge A on the end-effector link A can move on this circle on the hinge on the link 1 2 can move on this circle. Therefore, if I want to assemble the mechanism then it can happen only at these intersection points of the two circles.

Now once A is fixed since E is also fixed therefore, B gets fixed and therefore, you can find out B Q as we have done there is another configuration which looks like this. So, this is the end-effector link.

And let me draw the prismatic actuator the other leg. So, here I have drawn it in blue. So, this red blue configuration that I have drawn is the second configuration of the second solution for the inverse kinematics problem; this is the workspace of the manipulator.

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So, if you completely extend this link and then move it in the circle; you generate the outer circle which defines the workspace of this manipulator. Of course, with joint limits or actuator limits this workspace is going to get more complicated and will be reduced which you can find out based on geometry.

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Summary

- Displacement analysis of closed chain manipulators
- Problem of forward and inverse kinematics
- Example of 2R-RPR parallel manipulator with 2 DOF

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So, finally, let me summarize we have looked at the displacement analysis problem of closed chain manipulators with the example of a 2R-RPR kinematic chain. So, with that I will close this lecture.