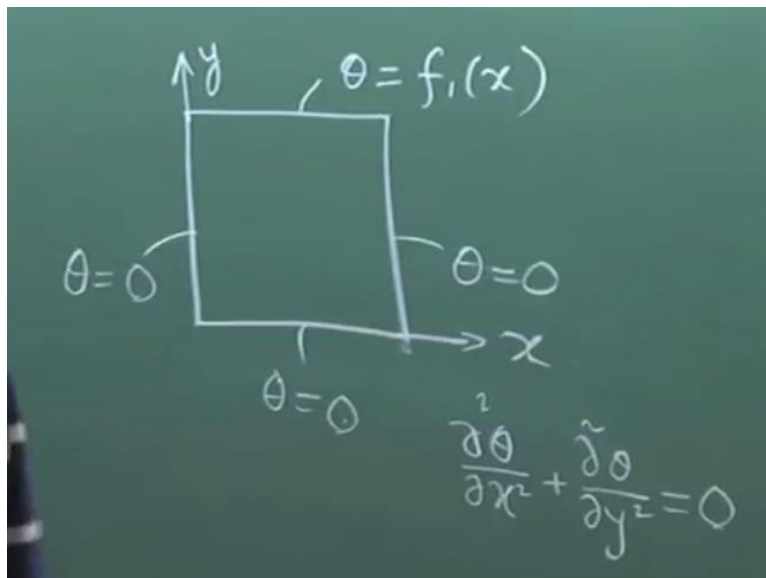


Conduction and Convection Heat Transfer
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Lecture - 14
Unsteady State Heat Conduction I

We discussed about 2-dimensional steady state heat conduction and we will take it up from there. We discussed about one simple problem that you have a domain like this with these as the boundary conditions and this is the equation which we solved.

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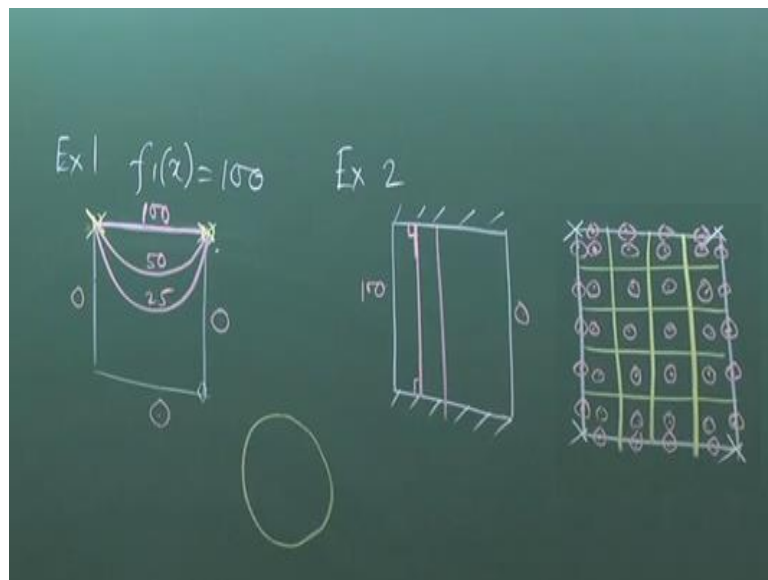
Now, we will try to develop some physical insight about this problem. So, to develop the physical insight let us take an example. Let us say $f_1(x)$ equal to 100 a constant, so in that case, if you consider this domain then how will the constant temperature lines are isotherms look like that is what we want to schematically show. So, these kind of physical insight is important because many times you can solve the problem and then find that graphically plot the solution.

But if you have a good physical intuition, good physical insight you can schematically draw the graphical representation without even solving the problem and that kind of skill is important for solving practical problems. So, now here you see that the temperature here is throughout 100, but here it is 0 in this boundary it is 0, so what is the predominant direction in which temperature gradient is created.

The predominant direction in which the temperature gradient is created is the Y direction because it is changing from 0 to 100 here. Along x also there is a gradient, but the gradient gives the periodic nature of the solution, ok. It is not a nonperiodic solution. It is a periodic solution along x and a nonperiodic solution along y. So that will mean that you will get constant temperature lines like this. What is the value of this? this is 100.

Let us say this is 50, this is 25 like that, ok. So, you will see that constant temperature lines will be like this. Now, there are typical situations when the boundaries are adiabatic and how will the constant temperature lines be at the boundary when the boundary is adiabatic. Let us consider a second example, this is example 1.

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Example 2, let us say these 2 boundaries are adiabatic. When you have adiabatic boundaries, you will have constant temperature line perpendicular to the boundary and the reason is obvious. If it is adiabatic then what does it mean it means that the heat flux is 0 that means the normal temperature gradient is 0 that means along the normal direction temperature is not changing. Therefore, the normal direction is the isotherm or constant temperature line, ok.

So, this kind of a situation for example you will get isotherms like this. Let us say this is 100 and this is 0, so you will get isotherms like this. Now, all these examples give rise to some ambiguity and let us try to discuss about that ambiguity. Now let us look into this problem. Here the temperature is 100, here the temperature is 0, the shifted temperature that is theta is $t - t_0$. Let us say this is 0, this is 0, this is 0 and this is 100.

And this is the question which we always like to avoid is that what is the temperature here or what is the temperature here right. This is the problem we always try to avoid and why we try to avoid this is because when the theories of differential equations are discussed over boundaries. The boundaries normally had not having any corner. They are smoothly connected boundaries.

Now, here you are having boundaries with corners and these are practical problems like in engineering you will often get boundaries with corners. So, the question is what is the temperature here. So, in mathematical analysis if you are solving the problem numerically then this kind of question is tackled in a different way.

So, if you have a domain, if you are solving this problem numerically, I told you that this belongs to the CFD for like solving these problems numerically but just to get some idea that what is done if you want to solve this problem numerically. What you basically do, you divide the domain into a number of subdomains, discrete subdomains and you may mark each subdomain with a grid point may be the centroid of each subdomain.

These subdomains are called sometimes elements, elements in finite element method, control volume, in finite volume method and so on. So, there are different names of the subdomains. So, if you have these points these are discrete points and then you also have discrete points on the boundaries because boundary condition needs to be incorporated, ok. There are some here also, but you can see that very purposefully.

These points are not considered to be parts of the discretization, right. So, what you are doing, you are trying to write algebraic equation instead of your differential equation. So, you convert your differential equation into a system of algebraic equations. Each equation for each of these circle points. So, as many number of circle points you have, so many number of algebraic equations you have.

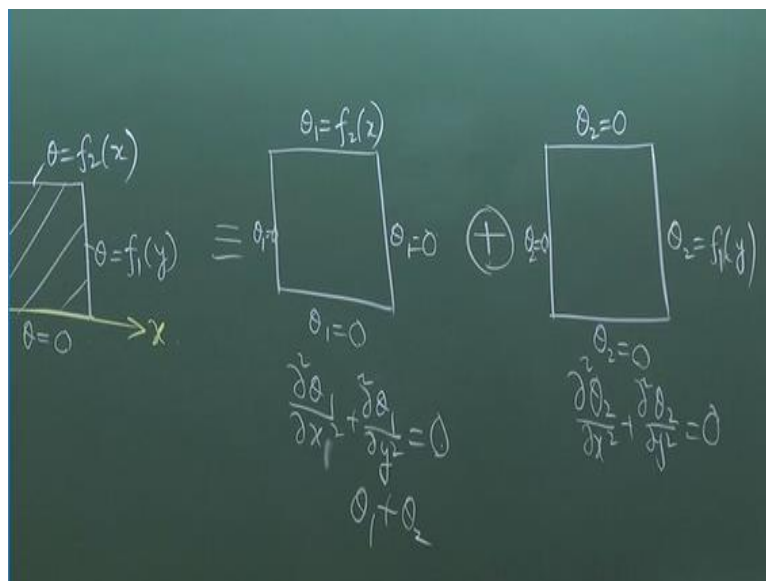
Then you solve a system of algebraic equation by standard methods in linear algebra that is what you do but this corner points are not considered. So, why these corner points are not considered is because these are points of singularity. So, at this point the temperature is

neither 100 nor 0, sometimes some people do it like this they say that ok let us make it 100 plus 0 by 2. There is everything in that except science.

So, that is not the proper way of handling it. So, these points are usually avoided because these are points of singularity and because these belong to 2 boundaries and these 2 boundaries have different conditions. Therefore, it is impossible to impose the condition at that common point. ok. So, typically this point is just excluded. So, excluding these points when you draw the isotherm it shows that this point is included.

But actually, this is slightly Epsilon distance away from the corner. Because in the limit that Epsilon tends to 0, so in drawing the figure we cannot show it, but conceptually it is a small Epsilon from the corner where the Epsilon tends to 0, but is not exactly equal to 0 because those are points of singularity.

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There is another example which I want to consider where we relax the boundary condition and consider a problem like this. Ok, you have a domain like this. This is your domain, you are interested to find out the solution for theta within the domain. Now, this clearly does not satisfy the requirement that 3 of the 4 boundary conditions are homogeneous. Here, only 2 boundary conditions are homogeneous and 2 are not.

So, these kind of problem can be solved by using the principle of superposition, so how this can be done. This is equivalent to 2 problems. Theta 1 is equal to 0, theta 1 is equal to 0, theta 1 is equal to 0, and theta 1 is equal to f 2 x plus theta 2 is equal to 0, theta 2 is equal to 0,

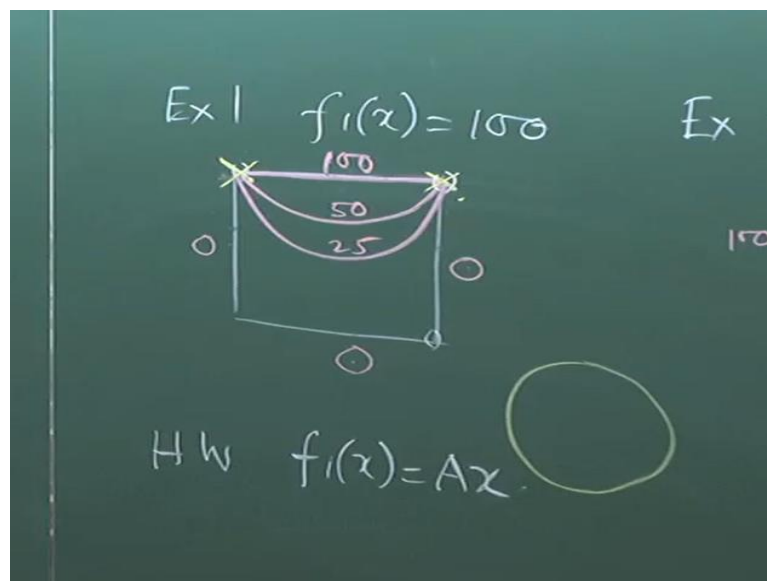
θ_2 is equal to 0, and θ_2 is equal to $f_1 y$, so this will be governed by $\nabla^2 \theta_1 = \nabla^2 \theta_2 = 0$.

This is $\nabla^2 \theta_1 = 0$ and $\nabla^2 \theta_2 = 0$ and the general solution is $\theta = \theta_1 + \theta_2$. The reason is that because the governing differential equation is linear if $\theta = \theta_1$ is a solution and $\theta = \theta_2$ is a solution $\theta = \theta_1 + \theta_2$ is also a solution to the governing differential equation which is shown here, ok.

So, this kind of superposition technique we can use for linear problems, we cannot use for nonlinear problems so in many problems where fluid flow is present then sudden nonlinearity may come or may not come, so if nonlinearity comes then we cannot use this technique anymore, ok. So, this is about the 2-dimensional steady state problems. Now, we will give you some homework assignments and homework assignment will be uploaded.

And I would like to ensure that we send the homework assignments prior to you over email and like for the mock course it will be uploaded in some sites and then there will be a separate tutorial when we discuss about the solutions of the specific problems. So, the way in which we go about this course is that we work out some problems in the class, we give you some homework problems, out of which some of them have might have already been worked out in the class or somewhat straight forward extensions.

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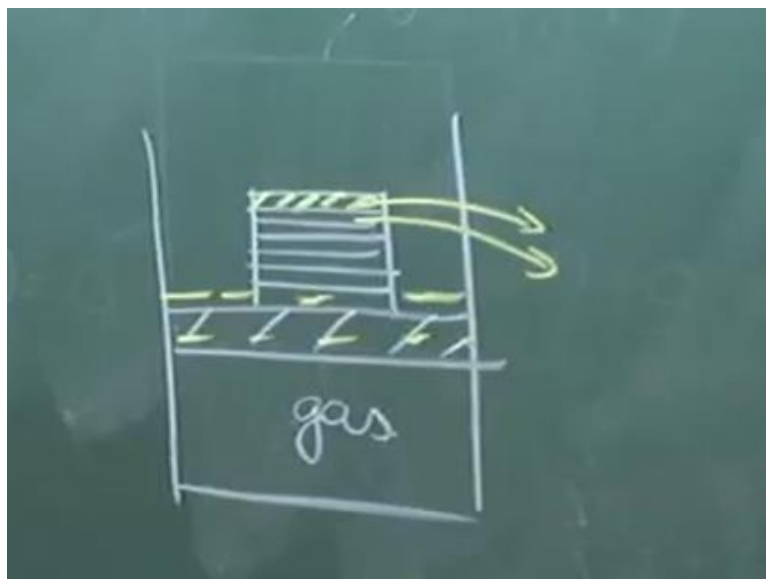
For example, I can give you a homework problem where $f 1 x$ is equal to a x . This is the problem from the exercise of the textbook of (()) (16:03). So, $f 1 x$ is equal to a x , now it is only a simple integration that you have to do because the entire frame work I have developed already in the class, so in heat transfer remember sometimes students say where is the problem because it is very difficult to understand what is theory and what is the problem.

In heat transfer, everything is theory and everything is problem. So, there is no distinction between it is not like a junior school level thing, so that you have a formula you monitor and then you put a value in the formula it becomes a problem, so that kind of education in heat transfer we are not going to provide you.

So, the problem means it is basically from the scratch, from the first principles you have to derive and if there are numerical values you just have to plug in the numerical values but we do not expect that you remember any formula to solve any problem that is the kind of philosophy that we will adopt in the course of heat transfer, so no formula based study please, ok. Now, we will move on to our next issue in the conduction heat transfer, which is unsteady.

Of course, we have done steady state problems, so the next obvious extension is unsteady, but there is something in between which is neither steady nor unsteady and that is called as quasi steady and I will discuss some such problem before coming into the unsteady problem.

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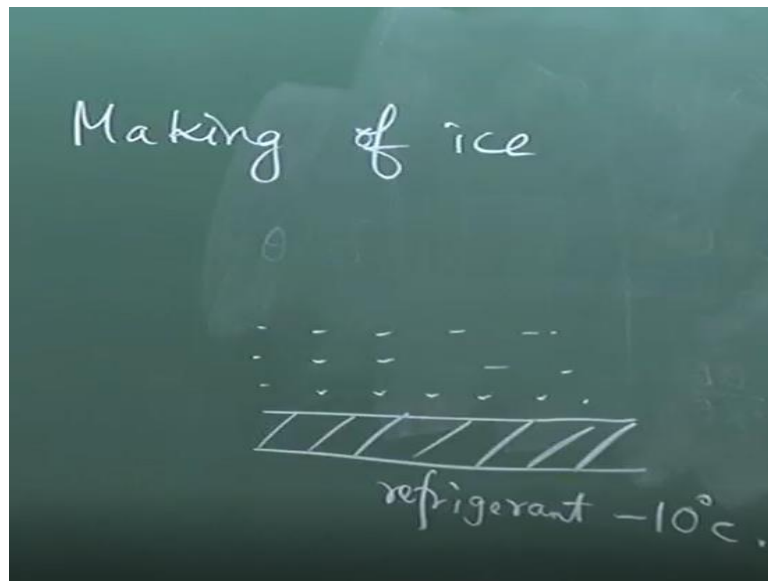
So, quasi steady has some resemblance with quasi-static or quasi-equilibrium process that you must have studied in thermodynamics, so again the only distinction is that there is a difference between steady and equilibrium, so that much of difference is there but notionally if you think of a problem in thermodynamics, a classical situation when there is a piston moving in a cylinder, very classical problems in thermodynamics involve a piston moving in a cylinder may be there is some gas in between the piston and the cylinder.

Lets us say that you have some waves on the top of the piston, the waves are such that these are very thin slices so what will happen you remove this top slice this piston will go up by a little bit, right. Then, you remove the next slice the piston will go further up by a little bit because for a given mass the pressure is decreasing, so the volume is increasing right.

So, in this way it will undergo a very slow expansion such that for the entire expansion process all the in between states are almost in thermodynamic equilibrium and the deviation from thermodynamic equilibrium is very little. So, the keyword is that this is the very slow process because it is a very slow process the deviation from equilibrium is almost nil, so that kind of process is called as quasi-static or quasi-equilibrium process in thermodynamics.

So, similar analogy in heat transfer not exactly the same again I am giving you a caution there is a difference between steady state and equilibrium which we have discussed, so similar analogy in heat transfer is a quasi steady. Quasi steady means in the long run it is unsteady because things are changing with time, but the change is so slow that the entire change can be thought of as a collection of a change taking place through a large number of intermediate steady states that kind of process is called as a quasi steady process.

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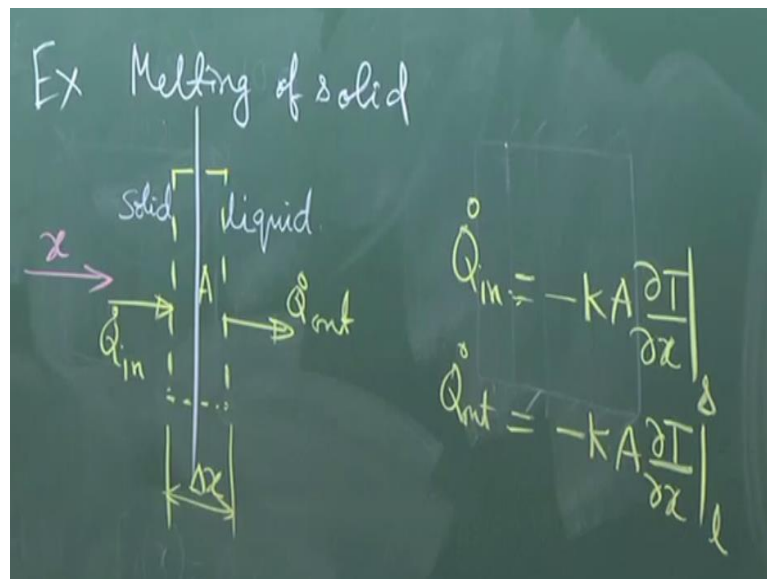
So, now I will discuss about an engineering problem as an example you may think of it as a problem or as an example or whatever, let us say making of ice. Ice making is a big industry and there are many ways in which ice can be made. Now, there is a situation something like this you have a metal plate and there is a refrigerant which is flowing below the metal plate with a particular velocity and there is a heat loss, so over this there is water.

Let us say this refrigerant is at minus 10 degree centigrade, ok. So, this water because it is at a higher temperature as compared to this, it will lose heat through this metal plate by conduction and once the heat is lost it will come to a state when the temperature comes at the freezing point of water and then some ice will be formed. The water converted into ice its layer will thicken and beyond the critical thickness that is crept off and new layer of ice starts forming.

So, this is a simple old-style technology of making ice, ok. So, now let us make a heat transfer analysis for this particular problem. So, we can make a quasi steady type of analysis but this is a very interesting problem where we are involving a change of phase. So far in the theoretical description of heat transfer, we have purposefully avoided problems with change of phase. Now, let us bring into an example where we have a change of phase.

So, let us say that this is an interface, ok. On this side there is solid, on this side there is liquid. I am not discussing about this problem first, but I am discussing about a different problem just to know that what kind of boundary condition you should use at the interface between the ice and the water.

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So, we are taking an example which is little bit different this is melting of a solid. So, let us say there is a solid, there is a heat transfer here, let us say it is a 1-dimensional situation. There is a heat transfer along x , so some heat is utilized for melting the solid to liquid. Let us say that we take a small control volume like this surrounding the interface. This control volume is of thickness Δx in the limit as Δx tends to 0 we can recover a sharp interface.

In reality, in the molecular world no interface is sharp because there are molecules in one phase and molecules in the other phase and there is a gradual transition when you go from the molecular arrangement of one phase to the molecular arrangement of another phase. So, it is not sharp but macroscopically if you see it will appear to be a sharp interface and that sharp interface is recovered in the limit as Δx tends to 0.

But we take a control volume like this and let us say that there is a rate of heat transfer \dot{Q} in and there is a rate of heat transfer \dot{Q} out. Let us say A is the area of the interface perpendicular to the direction of heat transfer, so what is \dot{Q} in this is minus KA by Fourier's law. What is \dot{Q} out similar term. Now, which one is more \dot{Q} in or \dot{Q} out, some heat has been transferred to the control volume.

And something is leaving now in between what has happened in between the solid has converted into liquid, so what has been the case when the solid has converted into liquid some heat has been taken by the solid to get converted into liquid in the form of latent heat,

so whatever is going out must be less than whatever has come in because whatever is going out and whatever has gone in a part of that is utilized for melting.

So, whatever is going out is less than whatever has gone in.

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The image shows handwritten equations on a chalkboard. At the top, there is a term $\frac{\rho A \Delta x}{\Delta t}$ with an arrow pointing to it. Below this, the energy balance equation is written as $\dot{Q}_{in} - \dot{Q}_{latent} = \dot{Q}_{out}$. In the middle, the mass flux is given as $\frac{\dot{m}}{A} = \ddot{m}$. Below that, the Stefan-Boltzmann law is written as $-k \frac{\partial T}{\partial x} \Big|_s - \dot{m} h_{sf} = -k \frac{\partial T}{\partial x} \Big|_l$, with the text "Stefan bc" written to the right. There are also some faint background notes like $\theta = f(x)$ and $\theta = f(y)$.

So, we can say $\dot{Q}_{in} - \dot{Q}_{latent}$ is equal to \dot{Q}_{out} , right. So, what is \dot{Q}_{latent} , so \dot{m} into the latent heat which is let us say h_{sf} , ok. We will see what is \dot{m} but I will just symbol wise this is the rate of mass being converted from solid to liquid and that multiplied by the latent heat is the total latent energy transfer, ok. Now, if you divide all the terms by the cross-sectional area then \dot{m} by A let us say that is mass flux \ddot{m} then we can write minus K .

Of course, what is \ddot{m} not actually the row into A into Δx this is the mass divided by the time Δt over which the space change process is being studied. So, \ddot{m} is that divided by A , so this becomes row into limit as Δt tends to 0 Δx by Δt right because why we make the limit as Δt tends to 0 because in the limit of Δt tends to 0 Δx will also tend to 0.

Then only this interface will become a sharp interface macroscopically, so this will become ok. Now, there is something which is more interesting at this undergraduate level I do not want to bring this issue, but just an open question which you think about this is row of which? row of liquid or row of solid. Yes, row of liquid or row of solid. So, sometimes you know when you given an answer in the answer book you write something here.

Then if I say that it is liquid you say I have written liquid, you cannot see it and because as professors we are old with spectacles and all then it is expected that we cannot see it, so you can keep it fuzzy but heat transfer is not a topic of fuzzy logic it is different. Fuzzy logic is a different subject and heat transfer is a different subject, so there is no fuzzy answer to it. I will complicate the question by even one more standard what is that let us say that the density of the solid and liquid is different and that is practical.

Usually if a substance melts either it will expand or it will contract, so the densities are different then how is this differential density taken into account in this, ok. These are 2 questions which I keep open for this particular case we will consider the density of the solid and the liquid phase are equal so that that ambiguity is not there, so you can consider this is as either row S or row L.

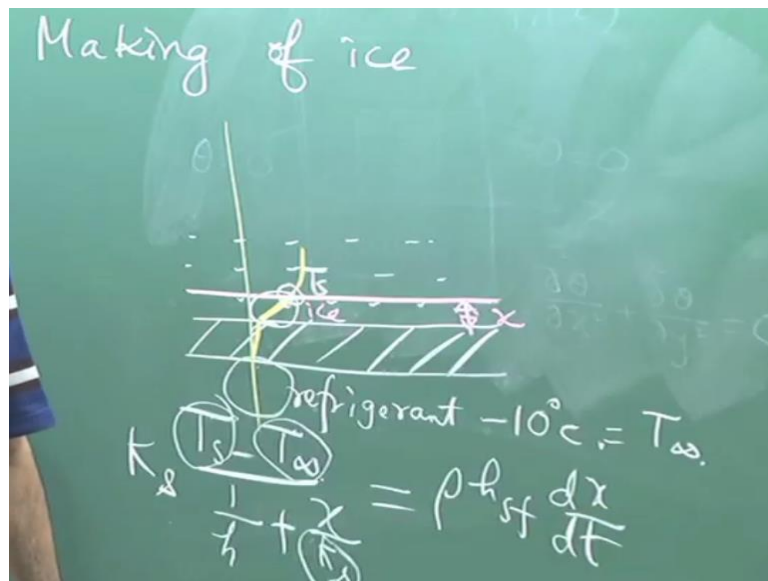
But I will ask you the answer please think about it, it is a question for thinking. Now, this type of boundary condition in heat transfer is an interfacial boundary condition known as Stefan boundary condition, ok. So, you can use this Stefan boundary condition for heat transfer for a phase change, any phase change. If it is evaporation, condensation just these things will change.

For evaporation, this will become h_{fg} . From saturated liquid to saturated vapor whatever is the latent heat. If there is no phase change then these term will be 0, right. So, at the interface what is the boundary condition that you have $-K \frac{dT}{dx}$ is continuous. Why because interface cannot store thermal energy, so whatever thermal energy has come to the interface the same thermal energy should be transferred to the other side of the interface.

So $-K$ into temperature gradient normal to the interface that is continuous. What is the analogy in fluid mechanics, if you have flat interface and let us say that you have a velocity profile along Y then $\mu \frac{du}{dy}$ is continuous across the interface. So, if you have liquid and vapor, so μ_{liquid} into $\frac{du}{dy}$ at the liquid is same as μ_{vapor} into $\frac{du}{dy}$ at the vapor. Of course, this will be little bit disturbed if you have a curved interface and so on.

But I am not bringing that complication at this level. I am just trying to give you an analogy between heat transfer and fluid mechanics.

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Now, how do you apply that formula to this particular problem. This problem is just inverse of this problem where instead of melting it is freezy, ok. Now let us try to assess the problem by assuming that this is the layer of ice that has been formed and let us say the thickness is x . So, now can you draw the temperature profile within the system, so first of all this is a metallic plate, why do we use the metallic plate in this case in engineering?

See, metallic plate will have a very high thermal conductivity, so the temperature drop within the plate is very small or temperature rise rather in this case not drop. Temperature rise within that is small, so that means virtually the water is exposed to a temperature which is close to minus 10 without any substantial increase within the metallic plate, ok. So, if you want to draw a temperature profile.

So virtually there will be some little change because it will have some conductivity then within the ice, it is pure 1-dimensional conduction, so there will be a linear temperature profile and then within water the temperature gradient will not be as sharp as that in the ice. So, you can safely neglect this turn. The temperature gradient in the liquid phase is much much less than the temperature gradient which is there because the temperature gradient is virtually is primarily imposed across this, ok.

So, you are left with these 2 terms, this term and this term. So, let us say that the temperature here is T_s and the temperature this minus 10 degree centigrade is T_∞ . So, how do you write this term you write K of solid into the temperature gradient, so basically you write T_s

minus T_{∞} . Now when you write $T_s - T_{\infty}$ you have to take the resistance of this phase plus the resistance due to convection here.

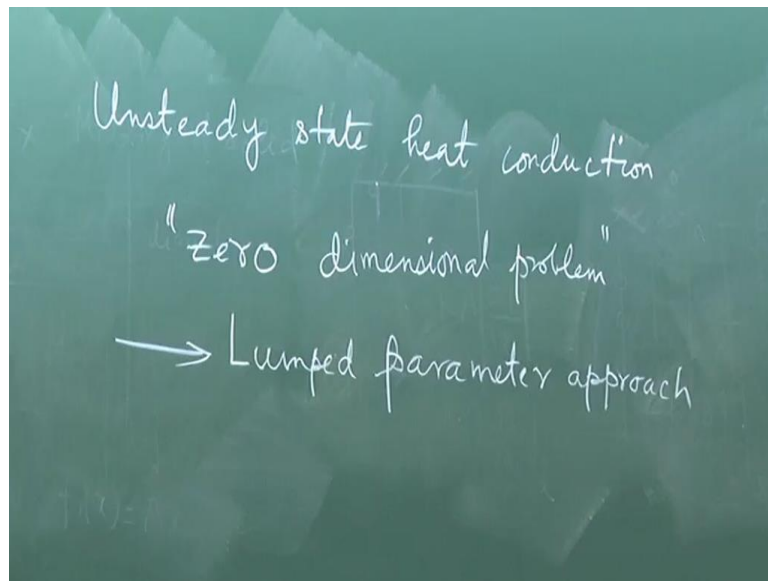
The resistance due to conduction here is very small because the conductivity of the plate is very high. So, you have basically 3 resistances in series, one resistance is because of the flow of refrigerant, this refrigerant is flowing so what kind of resistance it is creating it is a convective resistance. So, let us say h is the convective heat transfer coefficient here and this is the conductive resistance due to of course the area is observed here.

Because it is already divided by area that is why the area is not put there. This term balances the other term that is row into h s f into $d x$, $d t$, right. So now you can integrate this with respect to x to find out x as the function of time, ok. So that is a simple state forward integration, I am not putting that here. So, this will tell you that how do you design the system.

Because if you want to design the system you have to design that what is the thickness of the ice that is formed at a given time because that is your productivity so you must keep a proper design and you will see that what are the parameters which are defining these. What is the temperature here, what is T_{∞} and what is the conductivity of the solid and what is the heat transfer coefficient?

Heat transfer coefficient will be higher if you flow the refrigerant at a higher speed, so that is where convection will come into the picture, ok.

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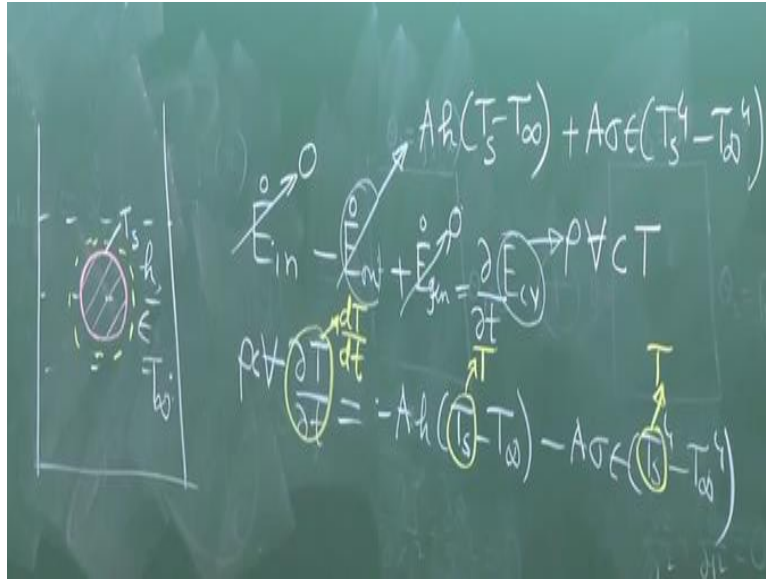


Now, we will move on to unsteady state heat conduction. When we think of unsteady state heat conduction, the situation is that we are considering that temperature is also a function of time. So, as I told you earlier that all problems are fundamentally unsteady so we are doing unsteady state analysis when our time domain of interest is such that over that time domain temperature is changing with time.

Now, temperature is changing with time, but temperature is also changing with respect to space. It is not just change with respect to time, but also change with respect to space. So, with respect to space, it can be 1 dimensional, it can be 2 dimensional, it can be 3 dimensional, but we will start with something, which is easier than all these that is zero dimensional.

That means with respect to space there is no variation and with respect to time only there is variation. So, we will consider something these terminology is not used in books, but in a loose sense it is like a zero-dimensional problem and the corresponding mathematical modeling is known as lumped parameter approach. In lumped parameter approach, what we are trying to do we are trying to neglect the temperature variation as a function of position and only consider temperature variation as a function of time.

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Let me give you an example, let us start with an engineering example, in manufacturing or in processing of materials, there is a process which is called as quenching. Quenching is a heat treatment process, so in quenching what happens let us say you have a metal block which you had heated, so heat treatments are done in materials industry to give sudden material property to impart certain properties to the material.

Now, once the heat treatment is done that it is heated, it may be heated in air, it may be heated in furnace or whatever then the material is cooled and sometimes it is cooled very rapidly by immersing it in some fluid. So, the material is hot and then it is suddenly put in a cold fluid and the material will cool down almost instantaneously. This process is called as quenching.

Now, this process from material science point of view is very interesting, because quenching will have very little time for heat transfer given and very little time for change to take place and over this little time the grains in the material will not be able to grow by a long way so there will be fine grain microstructure and that can give rise to a good amount of hardness to the material.

So, see this is how an engineering problem is, when you go to an industry nobody will tell you this is the problem of mechanical engineering, this is the problem of chemical engineering, this is the problem of material science, this is you have to use a holistic approach. So, this is the problem that you want to solve so let us say you want to predict the microstructure.

Why do you want to predict the microstructure because the microstructure will dictate what will be the mechanical property of the material say that is a steel ball and that steel ball will be used for making some structures, so depending on the microstructure the property of the steel ball will be different and it will behave mechanically different. So how do you know that what will be the microstructure.

The microstructure is the function of the cooling rate that means the rate at which the temperature of the steel ball decreases with the time in this case. In some other case, it might also increase but for a quenching process of course it decreases. The rate at which it decreases accordingly the microstructure develops, so the microstructure has a very important relationship with the heat transfer in general and the temperature gradient with respect to time in particular.

So, our objective will be to figure out how temperature of the steel ball is changing with time during the quenching process. So, let us draw a schematic of the problem. Say this is the bath and this is the ball, ok. So, for zero-dimensional analysis, we consider this ball to be a lumped mass. What is the analogy in mechanics, it is a particle. In mechanics, when we say particle what we assume even rigid bodies we can assume as particles if there is no rotation.

So, what does it mean? so let us say that we write Newton second law of motion for a particle. Now sometimes a card is also modeled as a particle. As practical engineers, we can argue that is card a point mass, card is not a point mass, but for all practical purposes the entire motion of that may be conceptualize as a translatory motion may be say center of mass which is the point mass.

So, similarly for under certain conditions, we will answer this question that under what conditions, under certain conditions we can study the heat transfer here by neglecting the temperature variation within the ball. That is, we can assume that the entire ball is at a spatial uniform temperature. I will discuss that under what condition that is valid and under what conditions that is not valid, but let us first assume that is valid.

So, if that is valid then let us say that you have a system which is the block, so energy in we can write an energy balance, energy in minus energy out plus energy generator, ok. This is

the energy balance that in the very first lecture which I delivered I talked about this the energy balance of the system. Now this is losing heat fluid surroundings by say convection and radiation.

So, let us say h is the convecting heat transfer coefficient and ϵ is the emissivity of the surface, so what is the rate at which energy is entering the control volume. There is no energy it is only losing energy because of cooling, so energy in is zero. What is energy out, let us say that temperature of the bulk fluid is T_∞ , so $A h (T_s - T_\infty)$, right. This is heat loss by convection.

This is heat loss by radiation. Of course, the minus sign is there before both of these. This T_s is what T_s is the temperature at the surface of the ball. What is energy generation? There is no energy generation in this case, but I will work out a problem in the class where there is energy generation. Now, what is energy of control volume. This is the solid one, so it is internal energy that is what row into the volume this is the mass into C into T .

So, we can write. Now if we assume that this is like a lumped mass then the temperature everywhere within the mass is same. So, this T_s is as good as temperature anywhere within the body. This is the lumped parameter analysis and then because temperature is everywhere uniform spatially temperature variation with time is the only derivative of temperature, so you can write this as ordinary derivative instead of partial derivative.

So, this is an ordinary differential equation governing the temperature as a function of time. Now, if you consider all these terms together, the integration may be a little bit cumbersome, but you can consider 2 limiting cases. In one case, convection is dominating, in another case radiation is dominating.

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Limiting cases:

Case 1 Convection dominating over radiation

$$\rho C V \frac{dT}{dt} = -A h (T - T_{\infty})$$

let $\theta = T - T_{\infty}$

$$\frac{d\theta}{dt}$$

So, limiting cases: Convection dominating over radiation. Now, you can solve this equation. Let us say that theta is equal to T minus T infinity. So, you can write d theta by theta is equal to minus A h by row CV d t. So, if you integrate this ln theta is equal to minus A h by row CV t plus ln C1 where ln C1 is a constant of integration. So, theta is equal to C1 into e to the power minus A h by row CV t.

How to get C1, you can use the initial condition at time equal to 0. T is equal to T i which is the initial temperature, so theta is equal to T i minus T infinity that is theta i. So, you will get theta is equal to theta i e to the power minus A h by row CV t, ok.

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$$\frac{d\theta}{\theta} = -\frac{Ah}{\rho C V} dt$$

$$\ln \theta = -\frac{Ah}{\rho C V} t + \ln C_1$$

$$\theta = C_1 e^{-\frac{Ah}{\rho C V} t}$$

ic \rightarrow At $t=0$, $T=T_i \Rightarrow \theta = T_i - T_{\infty} = \theta_i$

$$\Rightarrow \theta = \theta_i e^{-\frac{Ah}{\rho C V} t}$$

$\frac{Ah}{\rho C V} t = \frac{t}{\tau}$ $\tau \rightarrow$ time constant $= \frac{\rho C V}{Ah}$

Now, let us try to develop some physical insight on this. First of all, this term in the box must be non-dimensional because when you write e to the power something that something must

be non-dimensional. It is quite clear when you write e to the power x , $1 + x$ by factorial 1 plus x square by factorial 2 like that. So, if x as a dimension then dimensionally it will never match. So, e to the power something that something has to be nondimensional.

So, this is something which is a non-dimensional number, so $A h$ by row CV t that can be written as t by some τ . What is τ , is called as a time constant of the problem. Now this can also be written with some electrical analogy. What is this, this is like a capacitance, this is a storage ability of the system to store thermal energy. So, this is like capacitance and this is what this is one by resistance. So, this is like RC in a RC circuit in electrical circuit theory.

So, you will get similar type of characteristics for the current voltage type of relationship in an RC circuit in circuit theory in electrical engineering, so that is an analogy with electrical thing, but what is left for us to understand is what is the physical significance of this time constant. How can you use it for a practical problem? Not only that under what conditions this solution will remain valid.

Because we have assumed that the lumped model is valid that the entire solid is at a uniform temperature, but where is the guarantee that the entire solid is at a uniform temperature. So, we will figure that out that under what conditions the entire solid is at a uniform temperature and under what conditions it may not be consider to be so. We will take a short break of about five minutes and then we will continue with this discussion.