

Solar Energy Technology
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Lecture - 32
The phi bar-f chart Method, Tank
Losses and Finite Heat Exchanger

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Lecture 32 The $\bar{\phi}, f$ – Chart Method
Tank Losses and Finite Heat Exchanger

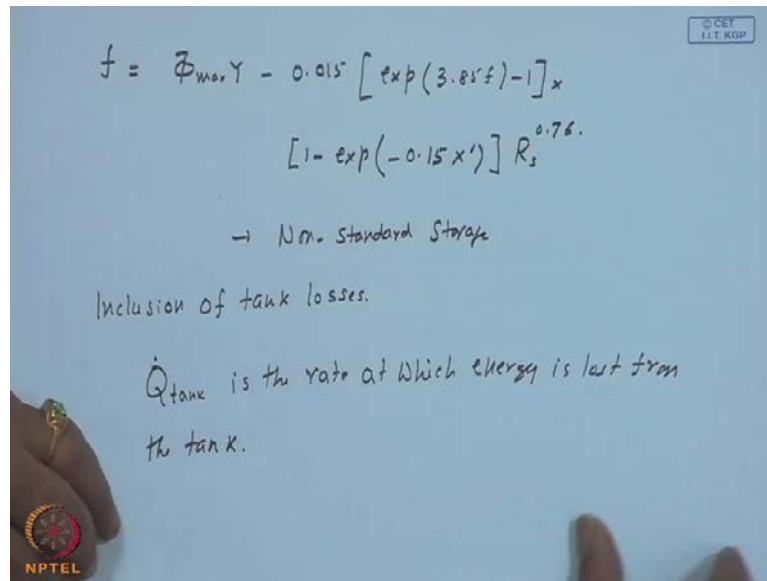
It has been noted that the $\bar{\phi}, f$ – Chart
correlation,

$$f = \bar{\phi}_{max} Y - 0.015 [\exp(3.85f) - 1] [1 - \exp(-0.15X')] \times R_s^{0.76}$$
takes into account, the non-standard storage,



So, today in this lecture, we shall proceed how to account for solar tank losses in the solar energy system, which is being calculated by the phi bar f chart method as well as the finite heat exchanger.

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The image shows a handwritten equation and text on a blue background. The equation is $f = \bar{\Phi}_{max,y} - 0.015 \left[\exp(3.85f) - 1 \right] x \left[1 - \exp(-0.15x') \right] R_s^{0.76}$. Below the equation, it says "→ Non-standard Storage". Further down, it says "Inclusion of tank losses." and then " \dot{Q}_{tank} is the rate at which energy is lost from the tank." There is a small logo in the top right corner that says "© CEET I.I.T. KGP" and an NPTEL logo in the bottom left corner.

$$f = \bar{\Phi}_{max,y} - 0.015 \left[\exp(3.85f) - 1 \right] x \left[1 - \exp(-0.15x') \right] R_s^{0.76}$$

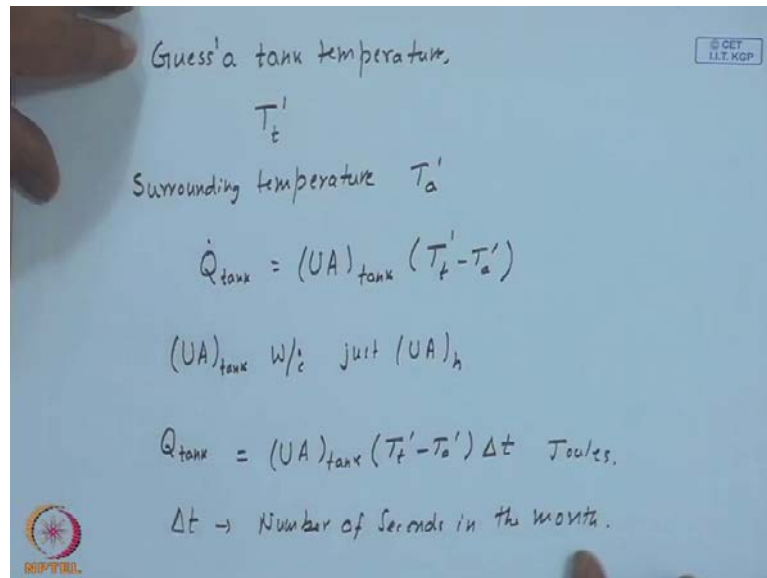
→ Non-standard Storage

Inclusion of tank losses.

\dot{Q}_{tank} is the rate at which energy is lost from the tank.

So, we shall just recall the correlation f in phi bar chart method is given by $\bar{\Phi}_{max,y}$ minus 0.015 exponential 3.85 f minus 1 times 1 minus exponential minus 0.15 x dash times stranded storage to actual storage ratio raise to the power 0.76. So, this takes into account as the non-stranded storage. So, now how do we include the tank losses? If the tank is at an average temperature of T , it will be losing heat at a particular rate depending upon the overall loss coefficient of the tank multiplied by the surface area of the tank, times temperature difference of the type T tank minus surrounding T a dash. Initially, we do not know what is the tank temperature? But let us say \dot{Q}_{tank} is the rate at which energy is lost from the tank.

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Since, I do not know the tank temperature, I will guess a tanks temperature which may require iteration. So, this I will call it guess 1 and let this be T_t^1 and the surrounding temperature may be T_a^1 . Please note that this may be different from the ambient temperature since many times, the storage tank can be in the basement and the basement temperature produce heat could be quite different from the ambient temperature. So, we will call it to keep it general T_a^1 . So, \dot{Q}_{tank} is $U A$ of the tank times T_t^1 minus T_a^1 . So, this $U A_{\text{tank}}$ is the overall loss coefficient watts per meter square degree c multiplied by the surface area of the tank. So, it will be in watts per degree c just like which we have introduced in calculating in space heating, the building heat loss coefficient $U A_h$.

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to $(UA)_h$ for a dwelling, used in calculating
the space heating load

The tank loss over the month is

$$Q_{\text{tank}} = (UA)_{\text{tank}}(T_t^1 - T_a^1)\Delta t$$

Note, it is Q_{tank} and not \dot{Q}_{tank}

Δt is the number of seconds in the month



If I integrate this \dot{Q}_{tank} rate over the month and assume this T_t and T_a remains constant or some average value, I will get UA_{tank} . It should not differ so much with the temperature $T_t^1 - T_a^1$ times Δt , where Δt is the number of seconds. This makes it joules. If U is watts per meter square degree c, this will be so many joules number of seconds in the month.

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Now, the solar energy system needs to meet the load on the system and the tank loss. Thus the load fraction f_{TL} inclusive of the tank losses is,

$$f_{TL} = \frac{Q_s + Q_{\text{tank}}}{L + Q_{\text{tank}}}$$

By definition the solar load fraction is defined as the fraction of the load on the



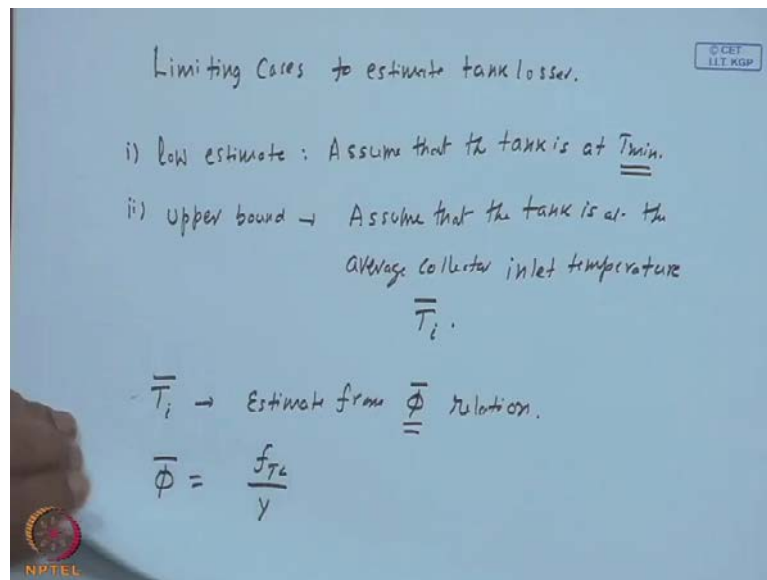
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Assume
 $L' = L + Q_{\text{tank}}$
Y and X' with $L + Q_{\text{tank}}$
 $f_{TL} = \frac{Q_s + Q_{\text{tank}}}{L + Q_{\text{tank}}}$
 $f = \frac{Q_s}{L}$
 $f = \frac{Q_s}{L} = f_{TL} \left(1 + \frac{Q_{\text{tank}}}{L} \right) - \frac{Q_{\text{tank}}}{L}$

So, what I shall do is assume load now will be equal to, let me say L_1 equals to L plus Q_{tank} . The affect of the tank loss as far as the solar energy system is concerned; it looks like an additional load. So, if I calculate my Y and X' with L plus Q_{tank} , I do not want to complicate things unnecessary calling L_1 , L_2 , L_3 etcetera. So, it is a actual load on the system plus the last one tank. I will get the solar load fraction, but I shall qualify it as f_{TL} , that is inclusive of the tank losses. So, what I get this number is Q_{system} by Q_{tank} by L plus Q_{tank} .

We have not deviated from our original definition except we realize that Q_{tank} has been fully met and has to be fully met the load on the system is L , and Q is the amount of energy that is going to meet the load on the system. So, consequently the slow load fraction that I obtain will be Q_s plus Q_{tank} by the total load now being L plus Q_{tank} , but actually my f_{solar} is Q_s by L . So, from this I can find out f is equal to Q_s by L . Just divide throughput with L , then I will get as f_{TL} times 1 plus Q_{tank} by L minus Q_{tank} by L , right. Q_s by L will be f and then Q_{tank} by L will be here and L by L is 1 plus Q_{tank} by L . So, that goes onto f_{TL} minus Q_{tank} by L . So, this is nothing but you respect the tank losses. You fully need that and then remaining amount of energy goes to meet the load.

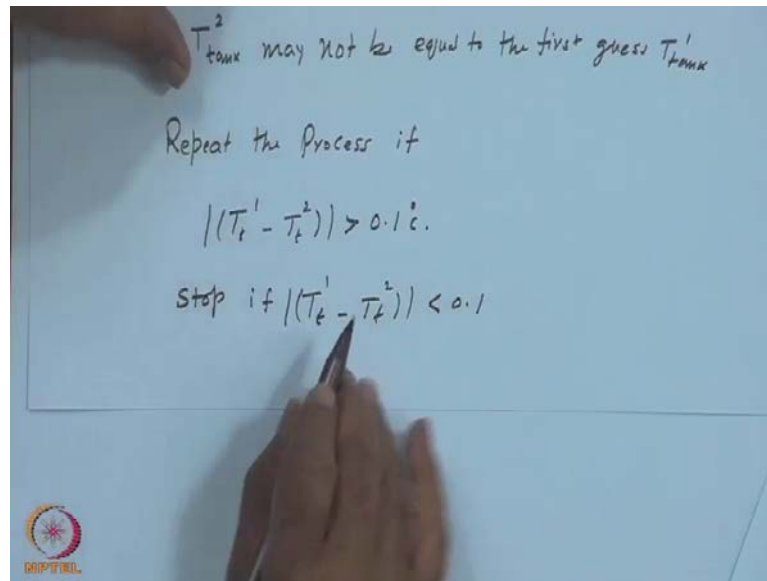
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Now, there are two limiting cases because I must have a way of correcting the T_{t1} limiting cases to estimate tank losses. One is a lowest estimate. Assume that the tank is at T_{\min} because the system is supposed to deliver energy at about the temperature T_{\min} . The tank temperature cannot be lower than T_{\min} and an upper limit is assumed that the tank is at the average collector inlet temperature. This is essentially equal to the outlet temperature because when once it stops increasing, the pump is cut off. So, this I will call it T_i bar.

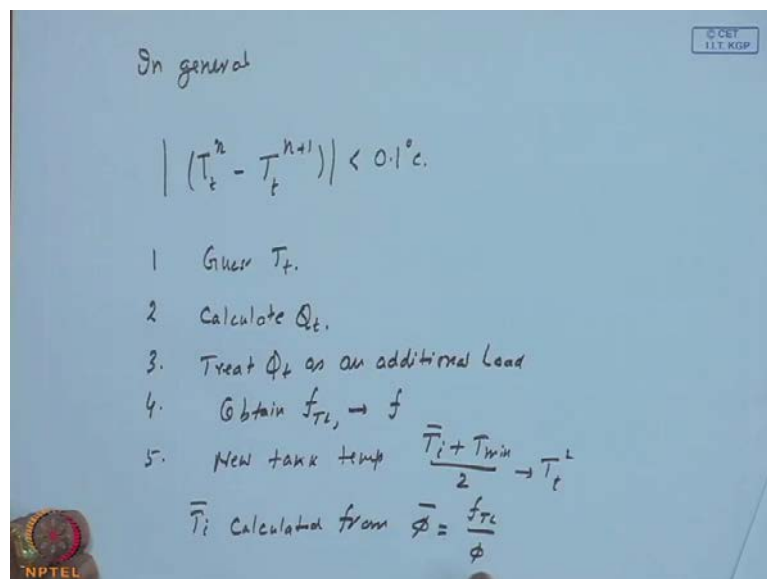
Now, this T_{\min} is no problem. This is known and is specified. This T_i bar, I will estimate from $\bar{\phi}$ relation. Since, this is corresponding to be an inlet temperature of the collector. Based upon this inlet temperature, there is a term utilization $\bar{\phi}$ and if there is a certain utilization, what the T_i bar temperature is. Now, this is where this $\bar{\phi}$ is. I am deliberately not writing $\bar{\phi}_{\max}$, just $\bar{\phi}$. It is essentially f_{TL} upon y . In other words, if y is the non-dimensional (()) energy and f_{TL} is the load fraction and include all the tank losses, then that corresponds to some sort of a $\bar{\phi}$ y equal to f_{TL} . As I was telling $\bar{\phi}_{\max}$ into y is the maximum solar load fraction. Some average utilization multiplied by y should be my solar load fraction with the tank losses.

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So, now the new tank temperature, at least I have whatever. Basis should be somewhere in between these two numbers. $T_{\text{tank } 2}$, I will call it T_i bar plus $T_{\text{minimum by } 2}$. Now, this $T_{\text{tank } 2}$ may not be equal to first guess $T_{\text{tank } 1}$. So, repeat the process. If $T_{\text{tank } 1}$ minus $T_{\text{tank } 2}$, we make sure modulus is, let us say greater than 0.1 degree c or stop, if $T_{\text{tank } 1}$ minus $T_{\text{tank } 2}$ mod is less than 0.1.

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In general, if we continue iterate, I can write it as modulus of $T_{\text{tank } n}$ minus $T_{\text{tank } n+1}$ mod is less than 0.1 degree celsius. So, you continue this calculation n

number of times until the difference in that estimated T_i bar and T_{\min} by 2 is pretty close to the previous value you guessed as the tank temperature. So, let me quickly go through this again step by step.

1. Guess T_t
2. Calculate Q_{tank}
3. Treat Q_{tank} as an additional load
4. Obtain f_{TL} inclusive of the tank losses from which you can calculate f_{solar} in meeting the load on the system, and
5. New tank temperature is T_i bar plus T_{\min} by 2 which may be your T_t . T_i bar is calculated from ϕ bar equal to f_{TL} by y because this is the function of explanation $a + b \times X_c$ bar plus $c \times X_c$ bar square. I will find out a X_c bar. From that X_c bar, I will find out a T_i bar, ok, f_{TL} by y . I am sorry.

So, this process is continued iterating n number of times until the temperature difference that you assume, the tank temperature and this final average tank temperature which is an average between upper estimate and the lower estimate of the losses is pretty close to each other. So, now the procedure is clear which shall become even more clear when we solve one example after sometime time.

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Finite Heat Exchanger

The effect of the heat exchanger is to increase the T_{\min}

The collector is now required to supply energy at or above,

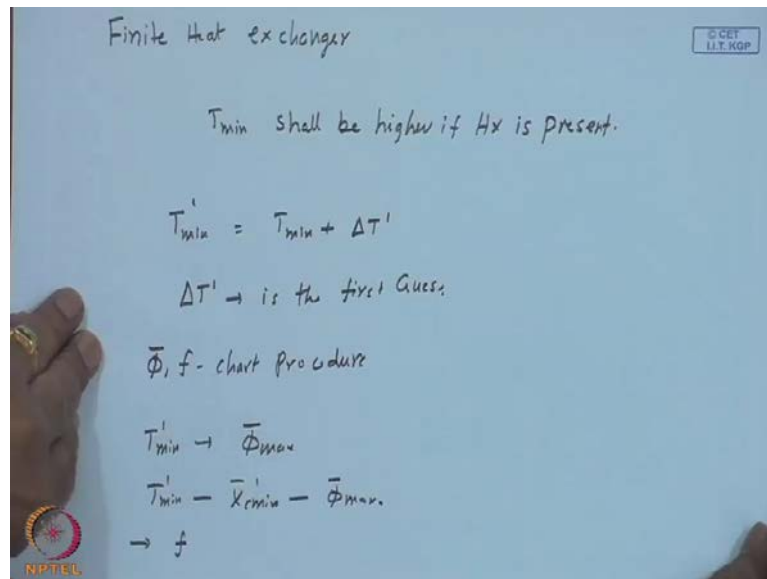
$$T_{\min}^1 = T_{\min} + \Delta T^1$$

Where ΔT^1 is the first guess.



Now, Finite Heat Exchanger.

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If you want to briefly say how the tank losses are accounted for, but to iterate the process and guessing, we might say that the effect of the tank loss is to enhance the load on the system by the amount that the energy is lost from the tank. Now, if you have a heat exchanger from the tank, it goes to the heat exchanger to meet the load. So, if you are calculating about the back flow, the T_{min} , the temperature at which the energy is to be (()) be the collector shall become higher if H x is heat exchanger is present, right. That will be some delta T is required. That delta reflex in the tank temperature, that tank temperature reflex in the T_{min} are the outlet temperature from the collector which needs to be delivered in order that it finally suppress energy originally desire T_{min} .

So, once again I will assume that T_{min} guess 1 equal to T_{min} plus some delta T 1 which may be 4 degrees, 3 degrees like that. So, delta T 1 is the first guess. So, we follow f phi bar f chart procedure. What do we mean by this? T_{min} 1 correspond to some phi bar max or in between if you want T_{min} 1 corresponds to $X_{c,min}$ 1 corresponds phi bar max, then I will get my solar load fraction. Y index are straight forward to calculate because they do not depend upon T_{min} .

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$$\bar{X}_{c, \min} = \frac{T_{c, \min}}{T_{r, n}} = \frac{[F_R U_o (T_{\min}' - T_a)]}{F_R (\bar{T}_a)}$$

$$\Delta T^2 = \frac{Q_s / \Delta \tau_L}{\epsilon_l C_{\min}} = \frac{fL / \Delta \tau_L}{\epsilon_l C_{\min}}$$

$$\epsilon_l C_{\min} \Delta T^1 = fL / \Delta \tau_L$$

$\Delta \tau_L$ may differ from $24 \times 3600 \times 30$.

When once if you want, you can now $\bar{X}_{c, \min}$ rewritten will be as $\bar{I} C_{\min}$ by $\bar{I} T$ noon equal everything is a same $F R U L$. Write T_{\min} guess minus T_a bar by $F R \tau \alpha$ bar over $r t, n R n K T$ bar H_0 bar. This is nothing but the same previous definition except I am replacing the minimum temperature at which energy delivery is desired by T_{\min} plus some ΔT because of the presence of the heat exchanger.

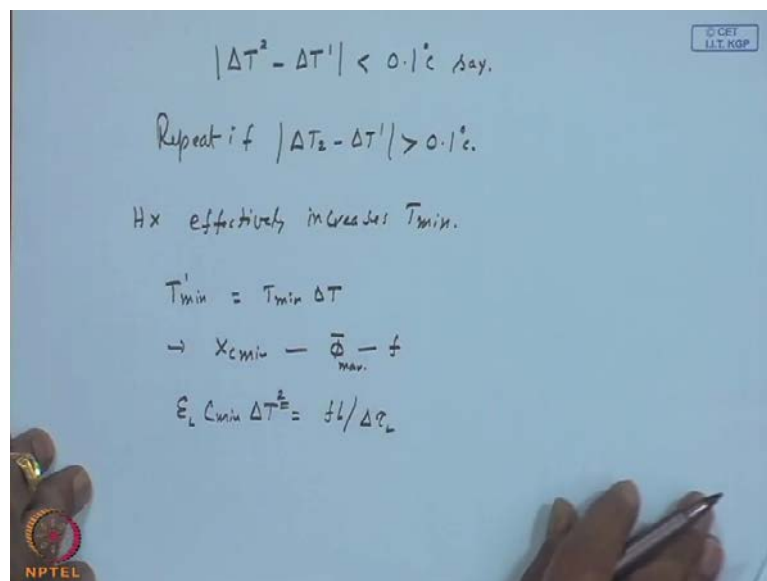
Now, if f is the solar load fraction that means $\bar{\phi}$, I have already calculated $\bar{X}_{c, \min}$ corresponding $\bar{\phi}_{\max}$ $\bar{X}_{c, \min}$ and we know y index dash and we know f . So, if f is the solar load fraction that multiplied by y is the useful energy that is being sent. So, that will be equal to Q_s by, I will explain again, $\Delta \tau_L$ by $\epsilon_l C_{\min}$. The energy that is going to meet the load on the system is f product by $\Delta \tau_L$. We will make this is number of seconds that the load operates that the load exist; it may not be 24 hours. It may be let us say, 12 kilo watts or 12 hours a day by $\epsilon_l C_{\min}$.

So, this is a rate at which the energy delivery is made upon $\epsilon_l C_{\min}$ and you will be very comfortable if you look at the equation as $\epsilon_l C_{\min}$. ΔT^2 is nothing but the energy transferred by the heat exchanger, right because this is the minimum of the heat capacitance rates of the heat exchanger fluids on the two sides of the heat exchanger multiplied by the effectiveness of the heat exchanger

multiplied by the delta T available across the heat exchanger, ok. This is nothing but f L upon delta tau L. So, f L is the total amount of energy supply over a period of delta tau L.

So, this is the rate at which the energy is supplied and that should be matched by the heat exchanger of effectiveness of epsilon L and the minimum capacitance of T minimum with a temperature difference of delta T 2. So, why I emphasis in delta tau L may differ from number of seconds in the month, right. Now, you might ask me the question why did we do not do this upper Q tank estimate. The loss from the tank always takes place, whereas if there is no load heat exchanger does not operate. That is the reason why energy this is a rate epsilon 1 C minimum is the rate at which the heat exchanger is going to supply energy. This is rate at which they load on the system.

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So, again we will set up the same condition. First, guess the difference between these two should be let say 0.1 degree c or else repeat if is greater than 0.1 degree c. So, you always come out with delta T 2, and then go and re estimate your X c bar minimum and then the solar load fraction, ok. So, in a nutshell that once again the heat exchanger effectively increases T minimum. So, the procedure is T min equal to T min plus delta T which may change it to X c minimum from where you calculate phi bar from where (()) f. So, from this epsilon 1 C minimum times delta T 2 should


be equal to f into L by ΔT . So, this is the governing equation. How? Please do not follow in the context. These are not ΔT squares ΔT , but I suppose those of you who have held by repeatedly saying that you would not make that mistake.

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2) Treat the effect of heat exchanger is to enhance the minimum temperature above which energy delivery is desired

An Example:

1. At a location of latitude 40°N , a process heating system employing flat plate collectors, facing south with



So, this is what I have again summarized. Heat tank loss at the additional load of the system, T the effect of heat exchanger is to enhance the minimum temperature above which energy delivery is desired. So, if you remember these two things in physical terms, it is more easy to automatically follow the procedure. Only in the case of tank temperature NU estimate of the tank temperature is T_i bar plus T_{minimum} by 2. So, the T_i bar being coming from the average utilizability concept. Now, we shall to consultate these ideas, we should take up an example and we shall see how much we can do.

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An Example

$\phi = 40^\circ N \rightarrow$ Process heating system
FPC \rightarrow facing south, $\beta = 40^\circ$.

Area, $A_c = 50 \text{ m}^2$

$FRUL = 2.63 \text{ W/m}^2 \text{ }^\circ\text{C}$
 $FR(\tau\alpha)_n = 0.72$

$T_{\text{min}} = 60^\circ\text{C}$

12 kW for 12 hrs a day.

Ground reflectance $\rightarrow 0.2$

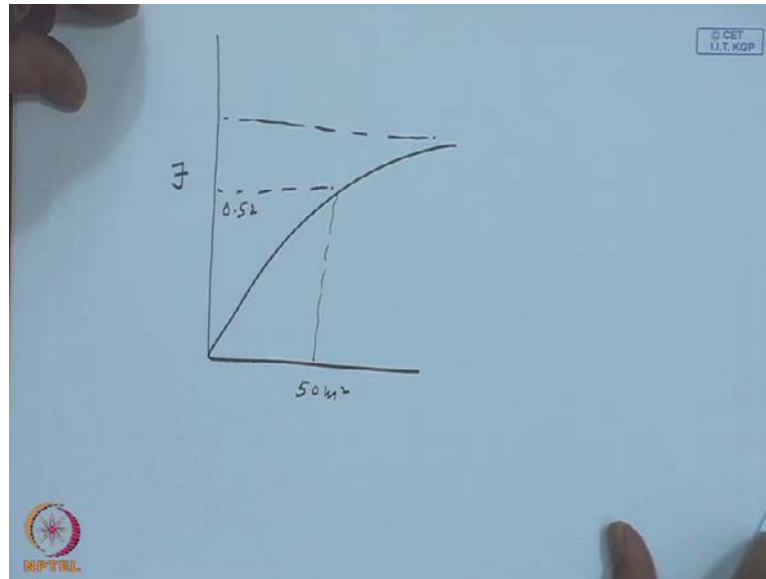
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I am deliberately giving many relations which we have gone through earlier. In fact, if you solve this problem, more or less, almost all the method that we have covered would be covered. So, the location is the latitude of 40 degrees and in fact, this example is from the text book by Duffy and Beckman except few things, I have changed here and there and this is for a process heating system, this is to bring out certain details, and flat plate collectors facing south and the slope beta is equal to 40 degrees, area is equal to 50 meter square and The collector parameters are FRUL is 2.63 watts per meter square degree centigrade, and FR tau alpha are normal. In fact, these values we have used in other publications also, this is sub T minimum is 60 degrees of 12 kilo watts for 12 hours a day.

Now, for our calculations, you may assume ground reflectance to be 0.2. So, the characteristics you should know of course the location slope flat plate collectors, the collector parameters, and then the minimum temperature and the total load and of course, this is the area that we have provided. What is the solar load fraction that comes out which may not know.

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We do not know appropriately. The idea is to calculate and if you plot the annual solar load fraction, it may be something like this, and if you have this 50 meter square, this may be 0.52 whereas, if you make any one monthly calculation, so for January, it may be less. For June, it may be more because the solar radiation is more and the load on the system is constant is 10 kilo watts for 12 hours a day.

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January $\bar{H} = 8.6 \text{ MJ}/(\text{m}^2\text{-day})$
 $\bar{T}_a = -5^\circ\text{C}$

- What is the (critical radiation level)
- What is the non-dimensional critical radiation level
- Monthly Average daily Utilizability
- What is the solar load fraction met by the system?

Standard storage.

The NPTEL logo is visible in the bottom left corner.

So, I take the data of January which you can calculate for the T_m by 2 also, \bar{H} bar is 8.6 mega joules per meter square day. In other words, there is no ambiguity. The

monthly average daily radiation on a horizontal z phase is 8.6 mega joules per meter square day and the ambient temperature is T_a is equal to minus 5 degrees c.

Now, only difference is if you take a location from India, typicality T_a may be 10, 15, 20 whatever even in the month of January. So, first since we know we are going to use ϕ of chart method, my question will be what the critical radiation level is, so that I try to put it in the form that you could see it, so that the methodology is remembered. What is the non-dimensional critical radiation level and then you have, c , monthly average utilizability, and what is the solar load fraction met by the system. Have I forgotten anything? Yeah, I think let us assume standard storage. It is 350 kilo joules degrees c.

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1st Part

Negligible tank losses
infinitely large H_x

$$\delta_m = 23.45^\circ \sin\left(\frac{284+n}{365} \cdot 360^\circ\right)$$

$$= 23.45^\circ \sin\left(\frac{284+17}{365} \cdot 360^\circ\right)$$

$$= -20.9^\circ$$

$n = 17$, from the Table
or you can use $n = 15$, for Jan 15.

These are one by one we try to calculate. Also, first part is negligible tank losses and infinitely large H_x . I suppose, those few are familiar with heat transfer. You would not find it surprising when I say infinitely large heat exchanger. It is nothing but saying that ΔT is 0 across the heat exchanger, ok. So, first we need the mean declination. You can see this almost one of the very first relations we have written, 23.45, the whole idea of 284 percent by 365 into 360. That will be equal to 23.45 times $\sin(284 + 17 \text{ by } 365 \text{ into } 360)$. Of course, n is equal to 17 from the table or you can use n is equal to 15 for January 15. Without much, it does not make, second

would be that should be minus 20.9 degrees. It may be 0.88 or 0.92. I do not know ((
)).

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$$\begin{aligned} \omega_s &= \cos^{-1}(-\tan \phi \tan \delta_m) \\ &= \cos^{-1}(-\tan \phi \tan(-20.9)) \\ &= 71.3^\circ \\ \omega_s' &= \text{Min} \{ 71.3, \cos^{-1}[-\tan(\phi - \beta) \tan \delta] \} \\ &= 71.3^\circ \\ \bar{H}_0 &= \frac{24 \times 3600 \times G_{sc}}{\pi} \left[1 + 0.033 \cos \frac{360n}{365} \right] \\ &\quad \times \left[\cos \phi \cos \delta \sin \omega_s + \frac{2\pi \omega_s}{360} \sin \phi \sin \delta \right] \end{aligned}$$

So, this is the main declination from which we calculate the sunset or angel omega s which is once again cos inverse minus tan phi tan delta, which is do not worry about this delta and delta m. We are making only mean day calculation. So, you need to forget that subscript, it means the single number in this problem. This comes out to be 71.3 degrees. So, this is lower than 90 as can be expected fairly high latitude and the January, it will be less than pi by 2 lower value and omega s dash for the south facing surface we know trigonometry. Obviously, this will be lower than that (()). However, you can calculate by minimum of 71.3 and cos inverse minus tan phi minus beta tan delta which is 71.3. What next because all my correlation will be in terms of the a clearness index. I need H 0 bar which will be 24 into 3600 times Gsc, the solar constant by pi times 1 plus 0.033 cos 360 n by 365 times cos phi cos delta sin omega s plus 2 pi omega s by 360 times sin phi sin delta.

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$$= \frac{24 \times 3600 \times 1357}{\pi} \left[1 + 0.033 \cos \frac{360 \times 17}{365} \right] \times$$

$$\left[\cos 40 \cos(-20.9) \sin(71.3) + \frac{2\pi \times 71.3}{360} \sin 40 \sin(-20.9) \right]$$

$$= 14.4 \text{ MJ/m}^2\text{-day.}$$

$$\bar{K}_T = \frac{\bar{H}}{H_0} = \frac{8.6}{14.4} = 0.6$$

Please see this factor 2 pi by 360 same as pi by 180 because omega s I am going to put in degrees. That is once again 24 times 3600 times 1357 north 63, as the guess may be by pi times 1 plus 0.033 cos 360 into 17 by 365 times cos 40 cos minus 20.9 sin 71.3, fine. It is all right. Not necessary, plus 2 pi into 71.3 by 360 sin 40 sin minus 20.9. You can check up with your relation which you have given n number of times that comes to 14.4 mega joules per meter square d. So, the monthly average clearness index is H bar by H 0 bar which is 8.6 by 14.4 equal to 0.6.

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Colloves - Yeviro and Rabl

$$\frac{\bar{H}_d}{\bar{H}} = 0.775 + 0.006553(\omega_s - 90)$$

$$- [0.505 + 0.004553(\omega_s - 90)] \times$$

$$\cos [115^\circ \bar{K}_T - 103]$$

$$= 0.3$$

$$\bar{R} = \frac{\bar{H}_T}{\bar{H}} = \left(1 - \frac{\bar{H}_d}{\bar{H}}\right) \bar{R}_b + \frac{\bar{H}_d}{\bar{H}} \left(\frac{1 + \cos 40}{2}\right)$$

$$+ \rho \left(\frac{1 - \cos \beta}{2}\right)$$

So, from this, we can calculate again monthly average daily diffuse fraction. This is Collares-Pereira and Rabl co-relation again which is 0.775 plus 0.006553 times omega s minus 90 minus 0.505 plus 0.00455 times omega s minus 90 times cos 115 K T bar minus 100. Now, please check the equations and the brackets. I think I am copying all right, but sometimes there may be mix up. 0.3 is reasonably expected in the bar per value. So, R bar will be H T bar by H bar which can be written as 1 minus H T bar by H bar. Now, you understand why we calculated this times R b bar plus H T bar by H bar times 1 plus cos beta by 2 plus rho which is 0.2 minus cos beta by 2.

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$$\bar{R}_b = \frac{\cos(\phi - \beta) \cos \delta \sin \omega_s' + \sin(\phi - \beta) \sin \delta \omega_s' \frac{\pi}{180}}{\cos \phi \cos \delta \sin \omega_s + \sin \phi \sin \delta \omega_s \frac{\pi}{180}}$$

$$\bar{R}_b = 2.32$$

$$\bar{R} = (1 - 0.3) 2.32 + 0.3 \left(\frac{1 + \cos \beta}{2} \right) + 0.2 \left(\frac{1 - \cos \beta}{2} \right)$$

$$= 1.91$$

This detailed factor for the entire radiation, this is the beam component, direct component, this is diffused, and this is the ground refraction and you have your R b bar cos phi minus beta cos delta sin omega s dash plus sin phi minus beta sin delta omega s dash into phi by 180 by cos phi cos delta sin omega s plus sin phi sin delta omega s into pi by 180. Please note that the numerator is with the appearance, sunrise, sunset or angles. So, I am deliberately writing omega s dash and omega is dash and the denominator is with respect to the actual sunrise or sunset or angle omega s. So, simply because these are numerical equal, do not interchange. That will become a bad habit.

So, this R_b if you calculate it, it is 2.32 and R_b will be $1 - 0.3$ that is H_d bar by H_d bar into R_b bar 2.32 plus 0.3 times $1 + \cos \beta$ by $2 + 0.2$ into $1 - \cos \beta$ by 2 , which is 40 and this comes out to be 1.91. Let us just halt for a while. So, in the course of these problems so far we used how to calculate the declination, how to calculate H_0 bar and the clearness index, and then the diffuse fraction and the parameter R_b . So, tilted radiation, diffuse fraction, extra (()) variation and declination, these relations has come so far.

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Handwritten mathematical derivations on a blue background:

$$R_{b,n} = \frac{\pi}{24} (a + b \cos \omega) \frac{\cos \omega - \cos \omega_s}{\sin \omega_s - \frac{2\pi \omega_s}{360} \cos \omega_s}$$

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$\omega = 0$

$$a = 0.409 + 0.5016 \sin(\omega_s - 60)$$

$$b = 0.6609 - 0.4769 \sin(\omega_s - 60)$$

$$R_{b,n} = 0.178$$

$$R_{d,n} = \frac{\pi}{24} \frac{\cos \omega - \cos \omega_s}{\sin \omega_s - \frac{2\pi \omega_s}{360} \cos \omega_s}$$

$R_{d,n} = 0.164$

NPTEL

Now, we go for estimating r_t at noon time because what was the first question. The first question was what the critical radiation level is. If I am about to, I can calculate FRUL into $T_i - T_a$ be something from which we have to calculate the non-dimensional critical level. If I want to calculate the non-dimensional critical level, we need the noon time radiation on the average day, $r_{t,n}$ is $\frac{\pi}{24} (a + b \cos \omega)$. Again Collares-Pereira and Rabl, these relations which you had it earlier, $\cos \omega$ minus $\cos \omega_s$ by $\sin \omega_s$ minus $2\pi \omega_s$ by $360 \cos \omega_s$. This is to be calculated noon means ω is equal to 0. Typically, it may not be at times. However, we will do it at ω is equal to 0. Assume solar time is the same as the local time that is one part and there are certain, where ω may not be equal to 0. $K = 0.409 + 0.5016 \sin \omega_s - 60$ and b is $0.6609 - 0.4769 \sin \omega_s - 60$. So, if you calculate $r_{t,n}$ will be 0.178, ok.

Now, you start developing a feel for the numbers, your r factors, r_b factors, higher r factors will be less because that will be multiplied by $1 - \frac{H_d}{H}$. So, this is $1.91 r_{bar}$ and r_b bar is 2.32 . Later on when we make a calculation and similarly for summer, then my r_b bar and r bar factors will be smaller than what you have got for the winter months. So, r_{tn} is 0.178 . This is about the noon time. That means 11.30 to 12.30 and that is almost 0.178 . That means 20 percent of the total radiation. It can be expected, though it is only 1 hour and the total hours are less than about 11 hours 90 degree, 90 hours corresponds to 610 hour that is 10 and 10 and half hours. So, 20 percent can be expected in one single hour because there is around the noon time.

So, this one way you can check whether the numbers that you get are reasonable or not. For example, r_{tn} should be higher than the proportionate amount around a noon time, and it will be lower than that means, there are 10 hours or each one will be 0.1 , but around noon time, it is almost 0.2 and r_{dn} is simpler that a plus $b \cos \omega$ is also not there. $\cos \omega - \cos \omega_s$ by $\sin \omega_s - \sin \omega_s$ minus $2 \pi \omega_s$ by $360 \cos \omega_s$, this is 0.164 . Let me repeat 0.164 .

Now, you may wonder the diffuse radiation r_{dn} , the fraction also is 0.164 . Pretty close to 0.178 , but you should remember this is 20 percent of the diffuse radiation whereas, this is about 20 percent of the global or the total radiation. So, these numbers looks similar out of magnitude. They do not differ much because one is normal as with respect to the global radiation and the other is normal with respect to the diffuse radiation. So, I am just trying, so that you can develop patience of field for the numbers that you get. It cannot be 10 to the power 6 and 10 to the power minus 3 or so.

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$$R_n = \frac{H_d}{H} \text{ at } K_T = \bar{K}_T$$

$$\frac{H_d}{H} = 1.188 - 2.272 K_T + 9.473 K_T^2 - 21.865 K_T^3 + 14.648 K_T^4$$

$$= 0.41$$

$$R_{b,n} = \frac{\cos(\phi - \beta) \cos \delta \cos \omega + \sin(\phi - \beta) \sin \delta}{\cos \phi \cos \delta \cos \omega + \sin \phi \sin \delta}$$

$$R_{b,n} = 1.97$$

Now, we are emphasizing for some reason or the other, R factor at the noon time in calculating the non-dimensional critical level. I need H_d by H , right at clearness index daily is numerical equal to average clearness index, ok whereas, when we calculate R_b and $R_{b,n}$, we used H_d by H whereas, when we are going to calculate R_n , I will use H_d by H . Again, the Collares-Pereira and Rabl correlation, but the individual daily diffuse fraction correlation. There may be different theories, there are explanations for these. One simplest could be this may be working better as a correlation.

The other reason could be since the critical level is going to be found out on the moon time equal to this solar radiation on the mean day, no matter what, it is mean day, a single day, so we shall use the single days diffuse fraction. In other words, for equating it, it will be individual diffuse fraction that matters. Incidentally, K_T bar is a number. That is all. See this bar I should have not drawn it plus $14.648 K_T$ to the power 4. So, if you calculate, this is 0.41.

So, at the given numerical values, at least in this way, it looks like the daily diffuse fraction is higher than the monthly average diffuse fraction. That is sometime back also I had explained, H_d by H is not summation of H_d by H divided by n , but it is summation of H_d by summation of H . R_b at the noon time $\cos \phi$ minus β \cos

delta plus omega plus sin phi minus beta sin delta by cos phi cos delta cos omega plus sin by sin delta. So, R b noon will be 1.97.

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$$R_n = \left(1 - \frac{0.164 \times 0.41}{0.178}\right) 1.97 + \frac{0.164 \times 0.41}{0.178} \left(\frac{1 + \cos 40}{2}\right) + 0.2 \left(\frac{1 - \cos 40}{2}\right) = 1.59$$

$$\frac{\bar{R}}{R_n} = \frac{1.91}{1.59} = 1.20$$

$$\rightarrow \bar{\Phi}_s = f\left(\underline{a}, \underline{b}, \underline{c}, \frac{R_n}{\bar{R}}, \bar{x}_c\right)$$

So, R n as I have explained 0.164 by 178, that is R d by R t into the diffuse fraction 0.41 times 1.97 that is R b noon plus again R d by R t times the diffuse fraction 41 times 1 plus cos beta which is 40 by 2 plus the ground effectance 0.2 into 1 minus cos beta by 2. This comes to be 1.59. So, R bar upon R n equal to 1.91 by 1.59 equal to 1.20, right. So, this is one parameter I need in calculating phi bar. In addition to this is a function of, of course that a b c and then R n by R bar R by R n does not matter and X c bar. So, this we have obtained. This can be calculated straight forward from the clearness indices.

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Critical radiation

$$\begin{aligned} \bar{I}_c &= \frac{2.63 [60 - (-5)] \times 3600}{0.72 \times 0.94} \\ &= 909308 \text{ J/m}^2 \cdot \text{hr} \\ &= 0.909 \text{ MJ/m}^2 \cdot \text{hr} \end{aligned}$$

Now, the critical radiation. I_c FRUL to T_{\min} minus T_a times; time is important that is what I was telling. We have to put that factor by 0.72 into 0.94. This now you will have in 909308 joules per meter square in that hour. That will be 0.909 mega joules per meter square root hour, ok. So, this is, otherwise you will get a very small number. It should be watts per meter square, but then I had to adjust the other thing also accordingly.

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$$\begin{aligned} \bar{X}_c &= \frac{\bar{I}_c}{\bar{I}_{T,n}} = \frac{0.909}{\frac{V_{t,n} \times 1.59 \times 8.6}{R_n \cdot H}} \\ &= \frac{0.909}{0.178 \times 1.59 \times 8.6} \\ &= \underline{0.37} \end{aligned}$$

So, non-dimensional critical level will be $X C \bar{}$ will be $I C$ by $I T$ noon time, that is equal to, now we got 0.909 which is $r t n$ into $R n$ into $H \bar{}$. This is $H \bar{}$ $R n$ and $r t n$. So, this will be 0.909 by 0.178 into 1.59 into 8.6. This is mega joules, this is also mega joules. I have no problem, 0.37. So, we calculate up to the non-dimensional critical level, then we will go to the calculation of the utilizability and the solar load fraction met.

Now, I think many of the relations which we have developed or studied empirically or analytically, they are already under use. So, if you solve one problem on $\phi \bar{}$ f chart, almost all the relations you will recall. We shall continue next time.

Thank you.