

Advanced Composites
Prof. Nachiketa Tiwari
Department of Mechanical Engineering
Indian Institute of Technology, Kanpur

Lecture – 41
Thermal Effects in Composite Laminates (Part-I)

Hello welcome to Advanced Composites. In the current seventh week of this course today is the fifth day. And what we start plan to start today is a new theme of discussion and that relates to Thermal Effects in Composite Laminates. The reason we are going to discuss these effects because these effects are extremely important; in context of composite laminates because most of times when composite laminates are fabricated, they are fabricated at elevated temperatures which could be more than 100 degrees or 100 and 50 degrees centigrade.

So, it is at that temperatures that several phases of the laminate fibers and matrix they are processed and they are heated till that temperature. And then at that temperature the reaction happens and everything solidifies and then the composite is brought down to room temperatures. So, as the material is brought down to room temperatures in built stresses may develop because of temperature effects even though there may not be any external forces applied on the system.

So, how do we compute these stresses and strains in composites? It is an is extremely important question. And what we planned to do today tomorrow and may be 1 day in the next week is to discuss how these effects can be accounted for and calculated. So, that is what is the agenda for next few days.

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THERMAL EFFECTS IN COMPOSITE LAMINATES

If there is no thermal effect:

$$\left. \begin{aligned} \sigma_1 &= Q_{11} \epsilon_1 + Q_{12} \epsilon_2 \\ \sigma_2 &= Q_{12} \epsilon_1 + Q_{22} \epsilon_2 \\ \tau_{12} &= Q_{66} \gamma_{12} \end{aligned} \right\} \text{Hooke's Law}$$

So, what we are going to discuss is thermal effects, thermal effects in composites. Now, if there is no thermal effect then we already know that stress strain relations between for a single layer and that is sigma we are talking about an axis system which is aligned to the material axis of the layer. So, we are talking about the stresses that are at an individual layer. So, this is equal to Q_{11} times epsilon 1 plus Q_{12} times epsilon 2. And there is no Q_{16} because this is the material axis system is aligned to the loading axis.

Similarly, sigma 2 is equal to Q_{11} excuse me Q_{12} times epsilon 1 plus Q_{22} times epsilon 2 and tau x y no sorry tau 1 2 equals Q_{66} gamma 1 2. So, these expressions essentially come out from the original Hooke's law. But when temperature effects are there then this Hooke's law has to be modified.

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if there is no thermal effect:

$$\begin{aligned}\sigma_1 &= Q_{11} \epsilon_1 + Q_{12} \epsilon_2 \\ \sigma_2 &= Q_{12} \epsilon_1 + Q_{22} \epsilon_2 \\ \tau_{12} &= Q_{66} \gamma_{12}\end{aligned}$$

Hooke's Law

$\alpha_1 \rightarrow$ CTE in 1-dir.
 $\alpha_2 \rightarrow$ 2-dir.
 $\Delta T \rightarrow$ change in temp.

MODIFIED Hooke's Law

$$\begin{aligned}\sigma_1 &= Q_{11} (\epsilon_1 - \alpha_1 \Delta T) + Q_{12} (\epsilon_2 - \alpha_2 \Delta T) \\ \sigma_2 &= Q_{12} (\epsilon_1 - \alpha_1 \Delta T) + Q_{22} (\epsilon_2 - \alpha_2 \Delta T) \\ \tau_{12} &= Q_{66} \gamma_{12}\end{aligned}$$

$\therefore Q_{12} = 0$

And once it is modified these are the relations for modified Hooke's law sigma 1 equals Q 1 1 and instead of epsilon 1, I have epsilon 1 minus alpha 1 times delta T plus Q 1 2 times epsilon 2 minus alpha 2 times delta T. So, this is a modified Hooke's law and what are alpha? Alpha 1 is coefficient of thermal expansion of the material in 1 direction. Alpha 2 is coefficient of thermal expansion in 2 direction ok. And delta T is change in temperature. It could be positive or negative.

So, similarly sigma 2 using modified Hooke's law is Q 1 2 epsilon 1 minus alpha 1 delta T plus Q 2 2 times epsilon 2 minus alpha 2 delta T. And tau 1 2 is equal to Q 6 6 times gamma 1 2. Because alpha 1 2 bar is (Refer Time: 06:00) is alpha 1 2 bar ok.

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$$\begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix} = \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & \bar{Q}_{16} \\ \bar{Q}_{21} & \bar{Q}_{22} & \bar{Q}_{26} \\ \bar{Q}_{61} & \bar{Q}_{62} & \bar{Q}_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix} - \begin{Bmatrix} \alpha_x \Delta T \\ \alpha_y \Delta T \\ \alpha_{xy} \Delta T \end{Bmatrix}$$

ACTUAL STRAINS THERMAL EFFECTS

Case 1: $\epsilon_x \neq 0$, $\sigma_x = 0$
 $\epsilon_x = \alpha_x \Delta T$

Case 2: $\epsilon_x = 0$, $\sigma_x \neq 0$
 $\sigma_x = -E \alpha_x \Delta T$

So, this is the modified Hooke's law. Now if I want to move from 1 2 axis system to an x y axis system. Then what do I do? So, $\sigma_x \sigma_y \tau_{xy}$ is equal to so, this is in the 1 2 axis system. So, we will write down the relations for x y coordinate system also. So, for x y coordinate systems we have a \bar{Q} matrix. So, $\bar{Q}_{11} \bar{Q}_{12} \bar{Q}_{16} \bar{Q}_{22} \bar{Q}_{26} \bar{Q}_{66}$ and all these symmetric matrix. And we know how to compute it is values, but then it is not just simplified to $\epsilon_x \epsilon_y$ and γ_{xy} vector, but it also accounts for thermal strains.

So, this is $\epsilon_x \epsilon_y \gamma_{xy} - \alpha_x \Delta T - \alpha_y \Delta T$ and $\alpha_{xy} \Delta T$. So, it should be remembered that in x y coordinate system there is an α_{xy} which is non zero, but in the 1 2 coordinate system α_{12} is 0. Now what is the difference between ϵ_x and this thing. So, these strains are the actual strains in the system. These are actual strains.

So, if you what do you mean by actual strain? If you put a strain gauge on it we will measure it. And whatever is the measured value these are those strains actual strain physically observable strains. These strains they are thermal effects. These are thermal effects. So, we will illustrate by an example. Suppose we have an aluminum bar and I heat it and the bar is free to expand, the bar is free to expand. Then what will happen when it expands ϵ_x will be not 0 right. It is expanding so, if I put a strain gauge

here. If I put a strain gauge here it will actually measure the strain. So, ϵ_x what do I say I said these are the actual strains.

So, ϵ_x will not be 0. And α_x and ΔT will also be not 0. And it will be just that it will expand by the same amount which will be equal to α_x times ΔT right. So, in this case ϵ_x will be equal to α_x times ΔT . It will expand by that amount and then the unit per unit expansion is called the strain. So, this ϵ_x minus α_x times ΔT when you add these up it will come to be 0. The sum of this and this will come out to be 0.

So, what will be the stress when a aluminum bar is free to expand? It is not experiencing any stresses. So, our model should be such that it should predict 0 stress. So, all these three vectors in that case components of the vectors they will be 0. And when they are 0 it will mean that σ_x will be Q_{11} times 0 plus Q_{12} times 0 plus Q_{16} times 0. So, external stress will come out to be 0 understood. And that is what is the reality so, it works.

Now, consider the case other case when we have an aluminum bar are and again we are heating it. And we are again we are putting some strain gauges to measure the expansions and ϵ_{xy} and all these things. But one thing what we are doing is we are constraining. So, it cannot expand and we are heating it. So, what will happen the actual strain and the bar will be 0, actual strain in the bar will be 0 and α_x times ΔT will not be 0.

Similarly, ϵ_y will be 0 and α_y times ΔT will not be 0 and so on and so forth. So, one thing happens the sum of this ϵ_x and $\alpha_x \Delta T$ will not be 0, but it will be negative quantity and when it is a negative quantity and it gets multiplied by Q_{11} Q_{12} . And all these you will find that σ_x σ_y and τ_{xy} they may not be necessarily 0 then may be compressive entities ok.

So, this is how we compute thermal stresses. So, in the strain vector we have not just ϵ_x ϵ_y and γ_{xy} which is the actual strain, but we subtract from that thermal effects. And then we multiply that modified strain vector with the \bar{Q} matrix to get stresses at each ply level. This is very important to understand. So, the next question is that α_1 and α_2 these are material properties. And they are so, we can measure them we can measure them. But, if we move from 1 2 axis system to x y axis

system, how do we calculate α_x , α_y and γ_{xy} because, unless we know these I cannot compute all these numbers in the green block.

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The image shows a digital whiteboard with handwritten notes. On the left, a box contains the text "AL" and "HEAT". To the right, there are two sets of equations:

$$\begin{aligned} \epsilon_x &= 0 & \alpha_x \Delta T \\ \epsilon_y &= 0 & \alpha_y \Delta T \end{aligned}$$

Below these, there are two vertical ellipses. To the right of these equations, there is a vertical stack of terms: σ_x , σ_y , and τ_{xy} , enclosed in a large right curly brace.

Below the horizontal line, there is a matrix equation for strain components:

$$\begin{Bmatrix} \alpha_x \\ \alpha_y \\ \gamma_{xy} \end{Bmatrix} = \begin{bmatrix} \cos^2 \theta & \sin^2 \theta & 0 \\ \sin^2 \theta & \cos^2 \theta & 0 \\ \sin 2\theta & -\sin 2\theta & 0 \end{bmatrix} \begin{Bmatrix} \alpha_1 \\ \alpha_2 \\ 0 \end{Bmatrix} \rightarrow \text{Calculate } \alpha_x, \alpha_y, \alpha_{xy}$$

The bottom right corner of the whiteboard shows the text "16 / 47".

So, that is the next thing I will write down. So, these are basically strain transformation equations. So, α_x , α_y and γ_{xy} is equal to $\cos^2 \theta$, $\sin^2 \theta$, $\sin 2\theta$, $\sin^2 \theta$, $\cos^2 \theta$, $-\sin 2\theta$. And actually the third column it really does not matter and the reason it does not matter is because α_1 is not 0, α_2 is not 0, $\alpha_1^2 + \alpha_2^2$ is always 0 ok. So, using these relations you can calculate α_x , α_y , α_{xy} . So, this is the modified Hooke's law. Now if we have this modified Hooke's law then how do.

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Handwritten notes on a digital whiteboard:

At the top, a transformation matrix is shown:

$$\begin{bmatrix} \gamma_{xy} \\ \sin 2\theta & -\sin 2\theta & 0 \end{bmatrix} \begin{bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{bmatrix}$$

Below this, the title "A-B-D RELATIONS" is underlined. The main equation is:

$$\begin{Bmatrix} N \\ M \end{Bmatrix} = \begin{bmatrix} A & B \\ B & D \end{bmatrix} \begin{Bmatrix} \epsilon_x^0 \\ \kappa_{xy}^0 \end{Bmatrix} \quad \text{when no thermal effect}$$

Below this, the expression for the resultant force N_x is given as an integral:

$$N_x = \int_{-h/2}^{h/2} \sigma_x dz = \int_{-h/2}^{h/2} (\bar{Q}_{11} \epsilon_x + \bar{Q}_{12} \epsilon_y + \bar{Q}_{16} \gamma_{xy}) dx$$

The whiteboard interface includes a toolbar at the top and a status bar at the bottom showing "16 / 47".

So, next thing is A B D relations. How do they change, how do they change ok. So, our original relations are N M is equal to A B B and D. epsilon x naught and k xy naught right these are the original relations. So, these relations are valid when no thermal effects, no thermal effects. Now let us look at what happens when thermal effects are present.

So, we will just do an example what is a N x N x we have defined was integral from minus h by 2 to h by 2 sigma x dz. This is how we had defined x right it is the fourth resultant that is force per unit length. And this was this is sigma x is what Q 1 1 bar epsilon x ha plus Q 1 2 bar epsilon y plus Q 1 6 bar gamma xy dx ok.

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When no temperature effect

$$\begin{Bmatrix} N \\ M \end{Bmatrix} = \begin{bmatrix} A & B \\ B & D \end{bmatrix} \begin{Bmatrix} \epsilon_x \\ \kappa_{xy} \end{Bmatrix}$$

$$N_x = \int_{-h/2}^{h/2} \sigma_x dz = \int_{-h/2}^{h/2} (\bar{Q}_{11} \epsilon_x + \bar{Q}_{12} \epsilon_y + \bar{Q}_{16} \gamma_{xy}) dz \leftarrow \text{No Temp. Effect}$$

In presence of temp.

$$N_x = \int_{-h/2}^{h/2} [\bar{Q}_{11} (\epsilon_x - \alpha_x \Delta T) + \bar{Q}_{12} (\epsilon_y - \alpha_y \Delta T) + \bar{Q}_{16} (\gamma_{xy} - \alpha_{xy} \Delta T)] dz$$

So, this is the, this was the relation when no temperature effect. But now we have a modified Hooke's law right. So, in presence of temperature in presence of temperature what is N_x if I have to compute N_x then it will be minus h by 2 to h by 2 Q_{11} bar times epsilon x minus alpha x delta T right plus Q_{12} bar times epsilon y minus alpha y times delta T plus Q_{16} bar times gamma x y minus alpha x y times delta T . And this entire thing has to be integrated dz . So, there should be here also it should be dz .

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$\epsilon_x = \epsilon_x^0 + \epsilon \kappa_y^0$

$$N_x = \int_{-h/2}^{h/2} [\bar{Q}_{11} \epsilon_x^0 + \bar{Q}_{12} \epsilon_y^0 + \bar{Q}_{16} \gamma_{xy}^0] dz + \int_{-h/2}^{h/2} [\bar{Q}_{11} \kappa_x^0 + \bar{Q}_{12} \kappa_y^0 + \bar{Q}_{16} \kappa_{xy}^0] dz - \int_{-h/2}^{h/2} [\bar{Q}_{11} \alpha_x + \bar{Q}_{12} \alpha_y + \bar{Q}_{16} \alpha_{xy}] \Delta T \cdot dz$$

$$N_x = A_{11} \epsilon_x^0 + A_{12} \epsilon_y^0 + A_{16} \gamma_{xy}^0 + B_{11} \kappa_x^0 + B_{12} \kappa_y^0 + B_{16} \kappa_{xy}^0$$

And minus h by 2 to h by 2; if I integrate it now I can break it up in to a temperature related component and a non temperature related components. So, Q_{11} bar and we also know that what is ϵ_x , this is equal to ϵ_x naught plus $z k_x$ naught this is right. So, this is and so on and so forth. So, I do that substitution. So, I get ϵ_x naught plus Q_{12} ϵ_y naught plus Q_{12} bar γ_{xy} naught.

So, these are the mid plane strain components $d z$ plus minus h by 2 h by 2 I get Q_{11} bar. And then there is a z here k_x naught plus Q_{12} k_y naught plus Q_{16} bar. I am sorry this should be Q_{16} $1/6$ and this is also Q_{16} Q_{16} bar k_{xy} bar $d z$ plus the temperature component, that is the temperature component. So, when the temperature component I have negative. So, I will put a minus sign here minus sign h by 2 to h by 2 minus. And I have Q_{11} bar $\alpha_x \Delta T$. Actually I will take ΔT outside plus Q_{12} bar times α_y plus Q_{16} bar times $\alpha_x y$ times ΔT times $d z$.

Student: (Refer Time: 20:55).

Ha.

Student: $\alpha_x y$ term.

$\alpha_x y$ term is here.

Student: (Refer Time: 21:03).

[FL] Because when we do the strain transformations you are seeing this see this. So, $\alpha_x y$ is there and $\alpha_x y$ and α_x and α_y you can compute from these relations. $\alpha_x y$ is not necessarily 0. So, now when I do all this essentially what I end up getting is N_x is equal to A_{11} ϵ_x naught plus A_{12} ϵ_y naught plus A_{16} γ_{xy} naught.

So, these terms are coming from the first integration. Then from the second integral I get the B terms B_{11} $k_x y$ naught. I am sorry k_x naught plus B_{12} k_y naught plus B_{16} k_{xy} naught. So, this I am getting from the second integral from here. And the first component set of components are coming from the integral. And then from the third integral I can add up all these things, all these things ok.

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The image shows a digital whiteboard with handwritten mathematical equations. At the top, there is an integral expression:
$$N_x = \int_{-h/2}^{h/2} [\bar{a}_{11} \alpha_x + \bar{a}_{12} \alpha_y + \bar{a}_{16} \alpha_{xy}] dz$$
 Below this, the equation is simplified to:
$$N_x = A_{11} \epsilon_x^0 + A_{12} \epsilon_y^0 + A_{16} \gamma_{xy}^0 + B_{11} \kappa_x^0 + B_{12} \kappa_y^0 + B_{16} \kappa_{xy}^0$$
 Then, the equation is written in terms of stress resultants:
$$\begin{matrix} N_x^T \\ N_y^T \\ N_{xy}^T \end{matrix} = \begin{matrix} A \\ B \end{matrix}^T \begin{matrix} \epsilon \\ \kappa \end{matrix} \Delta T$$
 The final equation is:
$$\begin{matrix} N_x^T \\ N_y^T \\ N_{xy}^T \end{matrix} = \sum_{k=1}^N [\bar{a}_{11} \alpha_x + \bar{a}_{12} \alpha_y + \bar{a}_{16} \alpha_{xy}]_k (z_k - z_{k-1})$$

So, I get a third thing from the third one so, I get minus and I get N_x times ΔT . And this I am getting from the third integral N_x hat ok. So, what is N_x hat? It is equal to so, if I am integrating it is and because this is our laminated composite I have to replace this integral by a summation sign. So, this is equal to k is equal to 1 to n $Q_{11} \alpha_x$ bar plus $Q_{12} \alpha_y$ bar plus $Q_{16} \alpha_{xy}$.

And this is we compute for the k th layer and then we add up all these things and I am sorry so, this has to be multiplied by z_k minus z_{k-1} , basically the thickness of the layer. So, this is N_x hat ok. So, likewise I can compute other things and y hat and N_{xy} hat. So, I get so, these are the three relations I get for N . And then I get similarly three other relations for moment resultants.

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$$\begin{Bmatrix} N \\ M \end{Bmatrix} = \begin{bmatrix} A & B \\ B & D \end{bmatrix} \begin{Bmatrix} \epsilon^0 \\ \kappa^0 \end{Bmatrix} - \begin{Bmatrix} N^T \\ M^T \end{Bmatrix} \quad \text{where}$$

$$\begin{Bmatrix} N^T \\ M^T \end{Bmatrix} = \begin{Bmatrix} \hat{N}^T \\ \hat{M}^T \end{Bmatrix} \Delta T.$$

So, overall my set of equations become $N M$ equals $A B B D$ times ϵ^0 and κ^0 minus $N^T M^T$, where N^T and M^T equals $\hat{N}^T \hat{M}^T$ times ΔT ok. Because, there is a ΔT term here if you remember So, the integral of everything in the bracket over the thickness is $N_x \hat{N}_y$ and so on and so forth.

But, then we have to multiply it by ΔT term. So, this is what I wanted to cover today. Tomorrow we will extend this discussion and we will also see what happens to the governing differential equations and the boundary conditions and then we will start solving an actual problem and see where it leads us. So, that is all for today and I look forward to seeing you tomorrow.

Thank you.