

Indian Institute of Technology Kanpur

National Programme on Technology Enhanced Learning (NPTEL)

Course Title

Manufacturing Process Technology – Part- 2

Module- 46

Rolling Process part-2

by

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So hello and welcome to this manufacturing process technology part 2, Module 46.

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We were talking and discussing about the rolling example where we had arrived at this formulation here.

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$$2d\sigma_x + \sigma_x dy - 2d\sigma_y + \sigma_y dx = 0$$

$$d(\sigma_x) - \sigma_y (1-\nu) d\theta = 0$$

$$\frac{d(\sigma_x)}{d\theta} - (1-\nu) \sigma_y = 0 \quad \text{--- finally } d\theta = \nu$$

Because the friction force is assumed to be small (cold rolling), the principle stress in the element can be taken as $(\sigma_x = \sigma_y) = \nu$

$$\sigma_3 = -p$$

Since it is a case of plane strain, the third principle stress $(\sigma_3) = \frac{1}{2}(\sigma_1 + \sigma_2)$

From Hooke's law for plane strain $\epsilon_3 = \frac{1}{E}[\sigma_3 - \nu(\sigma_1 + \sigma_2)] = 0$

Hooke's law for plane strain $\epsilon_3 = 0 \Rightarrow \sigma_3 = \nu(\sigma_1 + \sigma_2)$

$$\sigma_3 = E \epsilon_3 = \frac{E}{2}(\sigma_1 + \sigma_2) = \frac{1}{2}(\sigma_1 + \sigma_2)$$

Von Mises yield criteria $(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 = 6k^2$

$$\left[\sigma_x - \frac{1}{2}(\sigma_x + p)\right]^2 + \left[\frac{1}{2}(\sigma_x - p) + p\right]^2 + [p - p]^2 = 6k^2$$

$$\frac{1}{4}(\sigma_x + p)^2 + \frac{1}{4}(\sigma_x + p)^2 + (p + p)^2 = 6k^2$$

$$p + \sigma_x = 2k$$

Where we actually to look up you know into what is going to be the plane strain condition for this particular rolling so that we could substitute for the value of σ_x as we had obtained earlier this is σ_x I am sorry for in the earlier governing equation case so let us now apply the one misses yield criteria here so let us say if we apply the one misses yield criteria to look at what is going to be the condition for which plastic flow can be initiated we have here σ_x – you know.

So basically the criteria says $\sigma_1 - \sigma_3$ sorry $(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2$ should be greater than or equal to in this particular case it should be equal to $6k^2$ I think I had evaluated this condition earlier looking at the you know strain energy module of the deformation module and here in this particular example if we just substitute the values of the various σ , so σ_1 is basically σ_x in this case σ_2 as just being found out to be $\frac{1}{2}\sigma_x - \text{pressure } p$.

So square of this + you can say again $\frac{1}{2}$ of $\sigma_x - p + p^2 + - p - \sigma_x^2$ should be = to $6k^2$ or in other words we have a you know actually if we just looked at or if you just biblically solve this little basically lead us to $\frac{1}{4}\sigma_x + p^2 + \frac{1}{4}(6x + p)$ again whole square + $e\sigma_x^2$ should be equal to $6k^2$ or in other words $p + \sigma_x$ can be recorder as twice k okay so that is going to be the condition for 1 misses yield criteria to hold were there can be a relationship between the ultimate yield stress k and the you know the roll pressure as well as the principle stress of .

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$$\begin{aligned} (\sigma + \pi) &= 2k - p \Rightarrow \sigma_x = \left(\frac{1}{2} - p\right) \\ \frac{d[(\sigma - \pi)^2]}{d\theta} &= L \end{aligned}$$

So basically now if I put $p \sigma_x = 2k$ from the last relationship I have value for σ_x principle stress and terms of the maximum shear yield stress of the material twice $k - p$ so you now indicating a variable σ_x of the rolling process in terms of a material property k and also the roll pressure p and that is going to be our goal here to ultimately solve the governing equation so the governing equation can now be changed to $d/d\theta$ of you know just substituting the value of a σ_x here makes it twice $k - p$ times of y okay.

Minus of and please understand also one more issue is that you know the change in the direction of the friction force should happen before and after the neutral point.

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$$y d\sigma_x + x dy - 2d\sigma_y + \rho dy dz = 0$$

$$d(y\sigma_x) - \rho y (1-\nu)\rho dy = 0$$

$$\frac{d(\sigma_x)}{dy} - (1-\nu)\rho = 0 \quad \text{--- finally } d\sigma_x = \rho y$$

Because the frictional force is assumed to be small (elastic), the principle stress in the element can be taken as $(\sigma_1 = \sigma_2) = \sigma$

$$\sigma_3 = -p$$

Since it is a case of plane strain, the third principle stress $(\sigma_3) = \frac{1}{2}(\sigma_1 + \sigma_2)$

From Hooke's law for plane strain $\frac{d\sigma_2}{E} = \frac{1}{E}[\sigma_1 - \nu(\sigma_1 + \sigma_2)] = 0$

which should be $\frac{d\sigma_2}{E} = \frac{1}{E}[\sigma_1 - \nu(\sigma_1 + \sigma_2)] = 0$ in plasticity

$$\sigma_2 = E \epsilon_2 = \frac{1}{2}(\sigma_1 + \sigma_2)$$

Von Mises yield criteria $\frac{1}{2}(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 = 6k^2$

$$\left[\sigma_x - \frac{1}{2}(\sigma_x + p)\right]^2 + \left[\frac{1}{2}(\sigma_x + p) + p\right]^2 + [p - \sigma_x]^2 = 6k^2$$

$$\frac{1}{4}(\sigma_x + p)^2 + \frac{1}{4}(\sigma_x + p)^2 + (p + \sigma_x)^2 = 6k^2$$

$$p + \sigma_x = 2k$$

So in one case the whole frictional term as been suggested here in this equation you know this is the frictional term related to this equation, so this is a sort of a enabler and in the other case the after the neutral point as been crossed over this should be disabler so we should consider the + sign here you remember the way we had in some of our pervious side.

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$\int d\sigma_x + \tau dy - R d\theta + P dx = 0$
 $d(\gamma \sigma_x) - (R - \theta) P dx > 0$ — frictionly dir. σ_x
 $\frac{d(\sigma_x)}{dx} - (R - \theta) P > 0$
 Because the friction force is assumed to be small (cold rolling), the principle stress in the element can be taken as $(\sigma_x = \sigma_y) = \sigma$
 $\sigma_x = -P$
 Since it is a case of plane strain, the second principle stress $(\sigma_y)_{2nd} = (\sigma_x - P)$
 From Hooke law for plane strain $\epsilon_2 = \frac{1}{E} [\sigma_1 - \nu(\sigma_1 + \sigma_2)] > 0$
 Material should be loaded in the elastic range $(\nu > \frac{1}{2})$ in plasticity
 $\sigma_2 = E \epsilon_2 = \left[\frac{\sigma_1 + \sigma_2}{2} \right] = \frac{1}{2} (\sigma_x - P)$
 Von Mises yield criteria $(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 \geq 3k^2$
 $\left[\sigma_x - \frac{1}{2} (\sigma_x - P) \right]^2 + \left[\frac{1}{2} (\sigma_x - P) + P \right]^2 + \left[-P - \sigma_x \right]^2 \geq 3k^2$
 $\frac{1}{4} (\sigma_x + P)^2 + \frac{1}{4} (\sigma_x + P)^2 + (\sigma_x + P)^2 \geq 3k^2$
 $P + \sigma_x \geq 2k$

Try to enable or disable this frictional force so bit elemental the value this element reaches the neutral point where the velocity of the material is equal to the roll velocity we assume the friction to be assisting in the rolling process and beyond that obviously the roll velocity falls down and the informational deformational velocity + the infinity velocity that is V_f is more than the roll velocity so this going to be a disabler so it is going to pull the material back you know because of the relative lower relative velocity or there because of the positive relative velocity between the direction of the motion of the element and the wheel element okay.

So because of all this the you have to go up to the neutral point for the friction to be enabler and beyond the neutral point for a friction to be disabler for the rolling to happen so in this event also here we are just change this expression to + - to you know really get to.

So in one case the I am sorry this as to be here so in one case you know the rolling can be positive rolling before the neutral point and in another case the extent of friction should be a desirable so we have a + - μ term incorporated here and so therefore we can say this as - + - μ - θ or be equal to 0 okay, so if it is aiding the process before the neutral point the Sin of the friction coefficient is positive and we have assuming that the friction coefficient is descending the rolling process or it is going to be in the direction which will prevent the rolling from happening that is what - μ okay, we understand this very carefully so therefore we can write down this whole you know expression as.

If I make an assumption here that as a material is being rolled and it undergoes strain hardening so let say as the material, is rolled in that strain harden and also increased k values on the other hand you know you can say that because of the rolling process the y decreases and in other hand because only rolling process the y decreases so I can you know just assume in the problem here conditioning where the although it is only in the approximation that this is strain hard is not very huge.

In that event it kind of remains valid numerically when the k_y that is though increase in this strain hardening in the reduction the sectional area or you can say the sectional thickness of the material may remain a constant or may be a constant and independent of θ so this is only a assumption however in actual rolling process may not be typically true but you know in cold rolling process where the friction does not get varied very much and also we can assume the strain hardening mean.

You know may not be so drastic okay this assumption is thinking valid okay, so if we have this simplification without resulting in much error, we can pull out the k_y term from here because it is now independent of θ and we can write twice $k_y d/d\theta$ of $1 - P / 2k$ so the expression here plus obviously then we would like to write this term in a different way it is now domain \sin here writing $\theta - + \mu$ just opposite so just \sin from this $+ -$ okay times of R_p R times of the pressure P R times of the pressure P R is the roll radius P is the pressure equal to 0 and have you said that I further try to find out for small θ how y can be expressed so as θ is small okay, we can easily say y to be.

Represented as $y = Tf / 2 + R\theta^2/2$ so you know we can save the assume the $R\theta$ to be the element length and obviously because θ is a small and it is $R\theta$ is at angle θ with respect to the horizontal direction or at angle $90 - \theta$ the vertical direction, and if I look at such a.

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$(P+2k) = 2k \Rightarrow \sigma_y = (1 - \frac{P}{2k})$
 $\frac{d}{d\theta} \left[\left(\frac{1 - \frac{P}{2k}}{2} \right) \right] - (2k - \frac{P}{2}) R P = 0$
 $2k \frac{d}{d\theta} \left[\left(1 - \frac{P}{2k} \right) \right] + (2k - \frac{P}{2}) R P = 0$
 As θ is small, y can be expressed in the form $y = \frac{Tf}{2} + \frac{R\theta^2}{2}$

As the method is valid in jobs (like handball and also around the water)
 on the other hand because of the velocity process they decrease
 $(P) = \text{constant and independent of } \theta$

$\frac{d}{d\theta} \left[\left(1 - \frac{P}{2k} \right) \right] + (2k - \frac{P}{2}) R P = 0$
 $-(2k - \frac{P}{2}) \frac{d(P/2k)}{d\theta} + (2k - \frac{P}{2}) R \frac{P}{2k} = 0$

$\frac{d(P/2k)}{(P/2k)} = \frac{2R(2k - \frac{P}{2}) d\theta}{(2k - \frac{P}{2})}$

Integrate the equation

Illustration here write herein this method where we are talking about this to be $R\theta$ and this component of $R\theta$ to be $R\theta \sin \theta$ okay and $\sin \theta$ can be approximated as θ so therefore $R\theta$ times of $\sin \theta$ is $R\theta$ times of θ and that is going to be the additional which is going to here and therefore we can say that you know y can be estimated as $Tf + R\theta^2$ whole / 2 so having estimated the value of y here if I substituted this y from the relationship we would like to try to obtain.

Again you know for the y we can actually/ this whole expression by $2k$ therefore the expression comes out the y times of $d/d\theta$ of $1 - P/2k$ okay $+ \theta - +\mu$ times of R times of P /again $2k$ imam dividing the whole expression by $2k = 0$ or if I put the value of y so here as you know $Ts + R\theta^2$ and obviously $d/d\theta$ of one is zero so I can have this as $-(d P/2k)/d\theta$ and this divided by $2 + \theta \pm \mu R P/2k = 0$, okay. So we can actually arrive at some logarithmic form as a solution for these equation based on all this criteria's and further I can multiply this whole expression by 2.

So both sides so I have a twice coefficient $x du^2 = 0$ or then I can actually rearrange this expression and write this $d P/2k/P/2k$ okay, $P/2K$ basically is equal to now twice R times of $\theta \pm \mu$ times of $d\theta/Tf + R\theta^2$ so if we where to integrate this final form for the solution so we integrate this equation in the next step we are left with.

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$$\int \frac{d(P/2k)}{P/2k} = \int \frac{2R\theta dy}{y+R\theta^2} + \int \frac{2R\theta d\theta}{y+R\theta^2} + C_1 \quad (C_1 \text{ is the constant of integration})$$

$$\ln\left(\frac{P}{2k}\right) = \ln(y+R\theta^2) + \frac{2R\theta}{\sqrt{y}} \tan^{-1}\left(\frac{\sqrt{y}}{R}\theta\right) + \ln\left(\frac{C}{2R}\right) \quad \text{where } C \text{ is a modified constant.}$$

$$\lambda = 2 \frac{\sqrt{y}}{R} \tan^{-1}\left(\frac{\sqrt{y}}{R}\theta\right) \quad \gamma = y + R\theta^2$$

$$\frac{P}{2k} = C \frac{y}{R} e^{\lambda} \quad (P_1 + P_2 = 2k) \quad \frac{P_1}{2k} = \left(1 - \frac{P_2}{2k}\right)$$

Integral of $d(P/2k)/P/2k$ okay = integral of $2R\theta dy/(y+R\theta^2)$ one component \pm integral of twice $R\theta d\theta/(y+R\theta^2)$ we are just algebraically simplifying this expression enabling it to contribute and plus C_1 where C_1 is the constant of integration, okay. So we have C_1 is the constant of integration so having said that now if we looked at this solution this obviously would be okay, so this becomes equal to $\ln(P/2k)$.

And this particular integral here because obviously this is y you have dy so $dy/(y+R\theta^2)$ becomes $\ln(y+R\theta^2)$ and then this particular integral would have solution so basically you can this is actually a \tan^{-1} integral so this is going to be $2\mu\sqrt{R}/1$ times of $1/\sqrt{y}$ times of $\tan^{-1}(R/\sqrt{y}\theta)$ + the $\ln(C/2R)$ where C is a modified constant, okay. So it would come up from just dividing this R/μ , okay.

So you have T_f and then also dividing by T_f further so that you can have a $1+x^2$ so $dx/(1+x^2)$ formulation of which \tan^{-1} is the integral, so the argument there would come out to be \sqrt{R}/T_f times of θ okay and that is how you arrive at this formulation so therefore if this is going to be the final solution for the definite integral the goal that we must have is to be able to estimate what is going to be this C value.

For you to be enabling or for you to be able to tell about exactly what is the roll pressure which is needed to set forth yield criteria of rolling which is as per one vises as we have developed earlier as $p + \sigma_x = 2k$ where k is the sort of you can say the ultimate here the yield stress of the

material, so you know the if we write this sort of clear manner let us say we make λ which is equal to $2\sqrt{R/Ef}$ times of $\tan^{-1}\sqrt{R/Tf}$ θ .

So we assume this as the λ so we can leave with here you know a case where $P/2k$ can be estimated as some values C times of Y/R okay. Because you already know y to be equal to $Tf + R\theta^2$ okay, so y/R times of $e^{\pm\mu\lambda}$ okay. So which is this particular term here, so basically we are talking about you know $CY/2R$ and also this would necessitate so we already know that our initial pressure P_i of the roll + the σ_{xi} which is the initial.

The initial reaction force the inlet side of the roll should be equal to $2k$ by the warmness's criterion so therefore the $p_i/2k$ could be easily written as $1 - \sigma_{xi} / 2k$, and initial you know point or at initial point obviously the λ I which is going to be there okay it can be defined in terms of the θ I which is going to be the initial contact length okay of the roll so $\theta = \theta_i$ I think I had illustrated this earlier how θ_i is that at snap shot at t equal to exactly 0 supposing the rolling process was already continuing and you have at $t = 0$ you are taking a snap shot that is basically the contact at the length that the roll would have at the beginning of the process with respect to the work piece which is actually the angle θ_i .

So λ_i here can be recorded as $2\sqrt{r/tf}$ $\tan^{-1}\sqrt{r/tf}$ times of the variable θ at initial condition is θ_i , so having set that now if we assume the initial pressure roll pressure to be p_i and the initial principle stress to be σ_{xi} , here of course the material property which is not getting change too much in this particular case so we are left with $p/2k = c$ times of, so let us just substitute this corresponding to the initial condition we have $p_i/2k$ let us assume the constants value c to be $c -$ okay it is constant of integration mind you which is also quite dependent in general with θ here. So θ will change really this particular you know final formulation of c . So let us assume this value to be $c -$ constant of course will be something before the neutral point and after the neutral point the constant would be $c+$ okay. You already know that in one case friction design μ four in other case the friction is disabler.

So obviously this c value will be $c -$ before the neutral point has reached and will be $c +$ after neutral point, so from calculation say call so you just changing the coefficients to $\pm 2\mu\sqrt{r}$ so assume $c -$ here and why That the initial point is equal to the initial thickness t_i and r e because the friction in this particular case is playing a Saar because it is playing is a enabler we have $-\mu\Delta_i$

okay. And so from this expression we can easily get the value of so you know I am sorry there is a $\frac{1}{2}$ term here you have to remember y is actually the half the thickness.

So actually y is $t/2$ whether it is a initial or final that is how you represented y before. So we have the c^- value here calculated as from this expression write here as if I substitute at again the value of p_i which is $1 - \sigma_{xi} / 2k$ from this expression c^- comes to be equal to $2r / t$ times of $1 - \sigma_{xi} / 2k$ times $e^{+\mu\lambda i}$ so that is how you calculate one of the constants before the neutral point has been arrived at obviously because of the change of the value here this constant would also significant will change value wise.

And therefore as I told you before the neutral point and after the neutral point because switch in components are completely separate the constant of integrations would be c^- c^+ in both the cases. So let us now apply this equation at the end the rolling process to find out what is going to be the you know the region beyond the neutral point.

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$c \rightarrow c^+$ beyond the neutral point
 $\frac{P}{2k} = c^+ \frac{1}{R} (e^{+\mu\lambda x})^1$ $\frac{P_i}{2k} = 1 - \frac{\sigma_{xi}}{2k}$
 $\lambda = 2\sqrt{\frac{E}{t}} \tan^{-1}\left(\sqrt{\frac{E}{3}}R\right)$ $\lambda > 0$
 $\lambda > 0$
 $\left(1 - \frac{\sigma_{xi}}{2k}\right) = c^+ \frac{5t}{2R}$ $y = t/2$
 $c^+ = \frac{2R}{5} \left(1 - \frac{\sigma_{xi}}{2k}\right)$
 Before neutral point -
 $\left(\frac{P}{2k}\right) = c^- \frac{1}{R} e^{-\mu\lambda x}$
 $\left(\frac{P}{2k}\right) = \frac{2R}{5} \left(1 - \frac{\sigma_{xi}}{2k}\right) e^{-\mu\lambda x}$
 $= \frac{2R}{5} \left(1 - \frac{\sigma_{xi}}{2k}\right) e^{-\mu\lambda x}$
 After neutral point -
 $\left(\frac{P}{2k}\right) = c^+ \frac{1}{R} e^{+\mu\lambda x} = \frac{2R}{5} \left(1 - \frac{\sigma_{xi}}{2k}\right) e^{+\mu\lambda x}$

So let us look at so you know the beyond the neutral point the constant c is going to be c^+ and if I want to write the expression for the pressure of $p / 2k$ becomes equal to $c + y / r e^{+\mu\lambda}$, that is how because this now an enabler so the friction is sorry a disabler so therefore the value has to be $+\mu\lambda$ and obviously the constant of integration is c^+ I had earlier mentioned.

So in this particular case again at the final separation or at the end of the rolling process if thickness is t_f you know which has been achieved the roll pressure $p_f / 2k = 1 - \sigma_x f / 2k$ where this is the final principle stress which is there or the from the sheet and at the exit point therefore I can easily have the θ_i the equal to you know again if I look at how we have formulated λ here so λ is basically equal to twice root of R/t_f than inverse of root of t_f times of θ .

And I am saying that at the exit point of the roll the condition is corresponding to θ equal to 0 okay because the 5 we look at here again let us go back you know to the in the earliest module where you are talking about such a θ so the expression of θ equal to θ_i is really for the initial condition of the roll pressure but at the final extend you can see the value of θ is really from starting between 0 and 0 and 0.

So the value here is 0 actually okay so in this particular value there is no angle when the formulation you know its leaves so corresponding to this θ equal to 0 with this you know the exit condition for the roll were equal to t_f we can say that the θ is 0 turning with 0 so the λ becomes equal to 0 here and we are left through this 1.

And $1 - \sigma_x f / 2k$ becomes equal to $c + y$ is t_f in this particular case times of R because the t_f you know as I told you why is actually $t_f / 2$ initial thickness that you get y as a design it is from the origin which is located at centrally across the neutral axis alternate the at end of the element so $t_f / 2R$ and in other words we have the C^+ here estimated after the element passes the neutral point estimated as twice R/t_f times of $1 - \sigma_x f / 2k$.

So these two are very important for us because we would like to apply these to what is going to be the pressure before the neutral point what is going to be the pressure after the neutral point so as I said of values which are because of all this calculations is that at you know I adjust but I took down as before neutral point and after neutral point.

I have the $P/2k$ value just write this before here as if you made a numbers it was $C-y/R e^{-\lambda}$ and that corresponds to if I just substitute this value C -here from the expression then earlier that C - is equal to twice r/t_i $1-\sigma_{xi}/2k$ times of $e^{-\lambda i}$ okay so this is actually – and sorry this is – okay because as I told you before the neutral point so if I substitute this value for C - in this particular expression the final form which emerges here can be represented as twice y you know t_i , where t_i is initial thickness of the roll I am sorry it is take it off here okay

And after this beyond equal point or after the neutral point the $P/2k$ becomes equal to twice y/t_i times of $1-\sigma_{xi}/2k$ times of $e^{-\lambda}$ so that is how you can estimate the roll p [pressure before and roll pressure after and in fact you know if you really wanted to determine what is going to be the pressure at the neutral point. So you can nearly hither use the equation before or the equation after to actually look at what is going to be the pressure after and before the neutral point obviously physically what it would mean is that there is the continuity there is pressure continuity in the way limit is going towards the neutral point beyond the neutral point. So at that particular point where $\theta=\theta_n$, the θ is the neutral point the pressure before and after should be exactly same. So there is one continuity of the pressure, otherwise there is a step and that is not what really happens what there is in a actually situation. So therefore I can say that I can probably neutral point analysis calculate what is going to be the value of that.

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The image shows handwritten mathematical derivations in red ink. At the top, it states $\theta \rightarrow \theta_n$ and shows the limit of the pressure equation as θ approaches the neutral point angle θ_n . The equation is:

$$\left(\frac{P}{2k}\right)_{\theta \rightarrow \theta_n} = \frac{2y}{H} \left(1 - \frac{\sigma_{xi}}{2k}\right) e^{-\lambda(\theta - \theta_n)}$$

$$= \left(\frac{P}{2k}\right)_{\theta = \theta_n} = \frac{2y}{H} \left(1 - \frac{\sigma_{xi}}{2k}\right) e^{-\lambda \cdot 0}$$

Below this, there are more complex equations involving λ and θ_n . One equation shows λ as a function of θ_n :

$$\lambda = \frac{1}{2} \left[\frac{1}{H} \ln \left\{ \frac{2y}{H} \left(1 - \frac{\sigma_{xi}}{2k}\right) \right\} + \theta_n \right]$$

Another equation shows λ as a function of θ_n and θ :

$$\lambda = 2 \sqrt{\frac{P}{H}} \ln \left(\sqrt{\frac{P}{H}} \theta_n \right)$$

The final part of the derivation shows θ_n as a function of λ :

$$\theta_n = \frac{1}{2} \left[\frac{1}{H} \ln \left\{ \frac{2y}{H} \left(1 - \frac{\sigma_{xi}}{2k}\right) \right\} + \lambda \right]$$

Γ corresponding to the neutral points, so from the equations, so I actually can predict the $\lambda = \lambda_n$ by equating equation before and after that is $p/2k$ before which is actually $1y = 2 y/ti - 1\sigma / 2k e^{\mu y - y}$ neutral let say, we are λ_n and that is $= p/2k$ which is actually twice after by $tf - 1\sigma, f / p/2k \mu \lambda$. So if I equate these two expressions together, we can find out by taking natural sides γn value which comes out to $1/21/\mu$ natural log of tf/ti times of $1-\sigma x_i \times k / 1-\sigma x_f / k^2$ okay + the λ which was incorporated earlier from the $2/k$ before equation.

So that is how you find out the λ neutral corresponds to position which is corresponded to $2/r$ or tf times θ and so I will be able to sort of λn value what is going to neutral angle of a particular element in a rolling condition. So with this I would like to sort of end this particular module. I am also going to look at probably how to determine the rolls separating force, or what kind of talks that can be used or what kind of power loss in the bearing would happen but here you could actually have a very good estimate of rolling pressure.

To calculate you know all these different values related to power in equation while rolling, so with this i would like to sort of close this module in the next module we would like to look at again the roll separating force or the power transmitted bearing and then may be time permits to little bit some forming process and just schematically mention what the process like, in fact this kind of module need to transferred everywhere, forging extra I have shown for the rolling.

And then we would also like to have small introduction to manufacturing and beyond which I will show the video module 48 where we can actually use VMD title to which is very small. So with t is I would like to conclude thank you so much for being with me see you in next module bye.

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