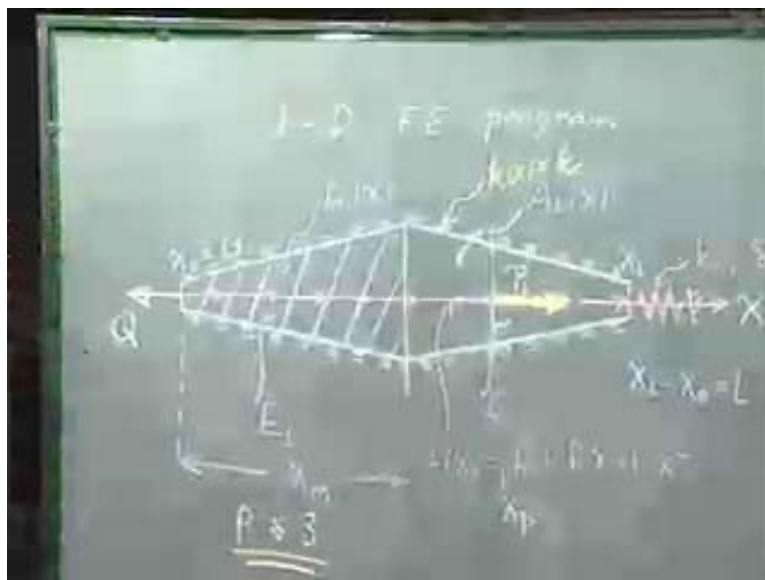


**Finite Element Method**  
**Prof. C.S. Upadhyay**  
**Department of Mechanical Engineering**  
**Indian Institute of Technology, Kanpur**

**Module - 5 Lecture - 1**

Today in this lecture, we are going to talk about the one dimensional finite element program. Before we write the one dimensional finite element program, we have to fix what should be the features of the program; that is, what the code should be able to do, what kind of material data, loading data, and boundary condition it should be able to handle.

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Let us take an example for the following case. This is our member (Refer Slide Time: 01:24) with this line being the line of symmetry or we call it as x-axis, in this member I can have a material in this region and some other material in this region. Here we would like to have material constant  $E_1$  and the area, if we see in both the members, is tapered but with different taperation. Here (Refer Slide Time: 02:14) I say area is  $A_1(x)$ . Here the material constant is or the Young modulus can be  $E_2$ . Taper ratio or the taper is given by  $A_2(x)$ . This is what we would like the code to handle. Where does this transition happen? This transition we are going to say is

at the point  $x_m$ , this can be point  $x_0$ . This could be point  $x_1$ . We know that  $x_1$  minus  $x_0$  is equal to  $L$ . We are going to set our  $x_0$  to 0.

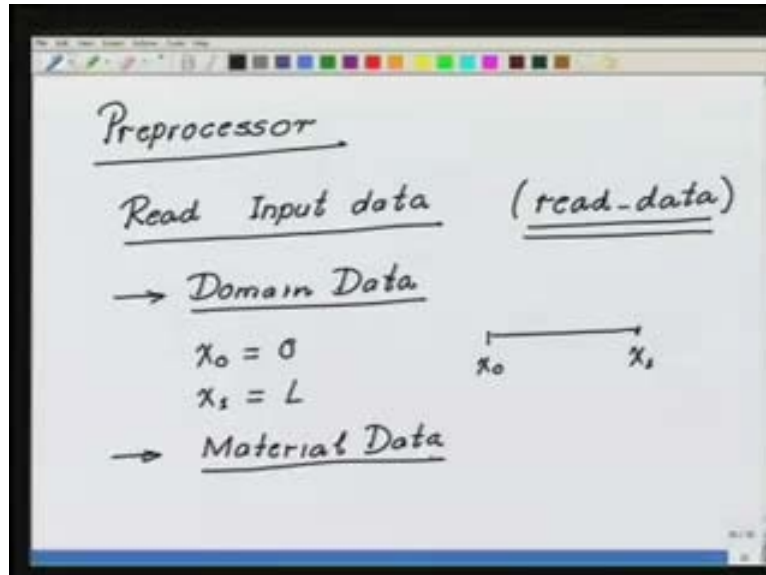
Further, the other features are the following. I would like a distributed load along the centre line (Refer Slide Time: 03:51) this intensity  $f(x)$  given by  $f_0$  plus  $f_{1x}$  plus  $f_{2x}(x \text{ square})$  that is we would like up to a quadratic distributed load. We would like to add some more features to our program. We like this program, if we desire on elastic support but what we would like is the elastic support should be distributed through the whole domain; that is, it should be applied to the full domain. We would like this elastic support to have an intensity  $k(x)$  is equal to  $k_0$  so our elastic support has uniform spring constant we do not want the spring constant to be very different; it should not vary.

Another feature we would like to have is, let us say there is a point here (Refer Slide Time: 05:35) the location of which will be given by  $x_p$ . At this point I apply a concentrated load of intensity  $P_1$  or  $\phi P_1$ . This is the concentrated load, I am applying here  $\phi P_1$  and we would like to handle only one concentrated load, not more than that. Further now, we would like our code to handle various sets of boundary conditions. What could be the boundary condition? I can have here an end load  $Q$  which is tensile. While, at the other end I could have the end force given in terms of the force applied by end spring.  $F$  is a force applied by end spring, for the spring I have to specify the stiffness coefficient of the spring and whether it is free compression or not  $\Delta L$  is the initial compression of the spring. This spring could be applied here this force could be applied here we could have any mix and match of various kinds of boundary conditions.

Further, we are also looking at boundary conditions which are symmetric in nature, we could fix this in. We would like to handle all these kinds of boundary conditions at each of these ends. I could choose anyone of the three results there; this is the problem we would like to solve. The other approximation that we want is  $P$ ; let say is less than equal or to 3. If we look at this problem, as far as making the mesh is concerned, we have to honor certain boundaries. Which boundaries we have to honor? This boundary (Refer Slide Time: 07:45) where there is a transition of a cross sectional area or the material we want both the transition to be the same, we do not want separate. Another boundary is here (Refer Slide Time: 08:00); we have to put a node here because at this point a concentrated load is applied. If we see that the mesh is broken, the

domain is broken into three parts; from this part from  $x_0$  to  $x_m$  from  $x_m$  to  $x_p$  and from  $x_p$  to  $x_0$ . These are the three parts that we would like to handle. Let us now go and essentially look at, how to design the pre-processor to handle all these cases?

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For the pre-processor design, we would like to first read the input data. This could be through a program. We can call it Read Input data; one can name it whatever he wants to but the name should reflect what that program is doing; so read data is our program. What would we like to read in? First, we would like to read in the Domain Data. From our figure that we have drawn there (Refer Slide Time: 09:45) domain data means I am interested in the extremities of the domain  $x_0$  and  $x_1$ . So  $x_0$  which for us is going to be 0 and  $x_1$  which is equal to the length of the member or we could have said  $x_1$  minus  $x_0$  is equal to  $L$  but here we are setting  $x_0$  is equal to 0 and  $x_1$  is equal to  $L$ ; this is what we have input first. We would like to input the material data, so how would we write the material data. In the material data we have to input the values of  $E$ ,  $E_1$ .

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$$\underline{E_1}, A_1(x) = \underline{a_0} + \underline{a_1}(x - \underline{x_m}), \quad \textcircled{x_m}$$

$$\underline{E_2}, A_2(x) = \underline{b_0} + \underline{b_1}(x - x_m)$$

$$E_1 = E_2 = E$$

$$A_1 = A_2 = A$$

$$x_m = x_1$$

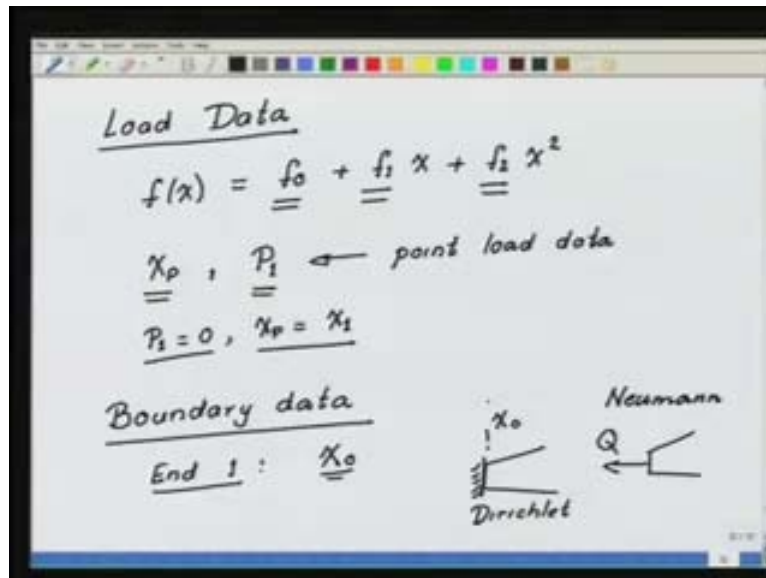
$$k(x) = \underline{k_0} \rightarrow 8 \text{ quantities}$$

We want to have the values of  $E_1$  as the function of  $x$  which we are going to given in terms of two parameter  $a_0$  plus  $a_1(x \text{ minus } x_m)$  so this is given in terms of  $a_0$  plus  $a_1(x \text{ minus } x_m)$ . If I have to give this then I have to also input  $x_m$ . This  $E_1$   $a_0$ ,  $a_1$   $x_m$  have to be input first, we have to input for the second part of the domain  $E_2$ ,  $A_2$  as the function of  $X$  is equal to  $b_0$  plus  $b_1(x \text{ minus } x_m)$ . So the first part of the domain we input these two things, for the second part of the domain we input this quantity (Refer Slide Time: 12:00). It will be better to first input the transition point  $x_m$  as a separate entity and then for the two parts of the domain we give  $E_1A_1$  and  $E_2A_2$ . In case I want only a single domain, material domain; for example, I take this bar for which  $E_1$  is equal to  $E_2$  is equal to  $E$  and  $A_1$  is equal to  $A_2$  is equal to  $A$ . It could be tapered but the taper is not changing. In that case, we will make  $x_m$  is equal to  $x_1$ . If we do that then what will happen is your program is going to take only  $E_1$  and  $A_1$  so as the domain because the second part is essentially at the end of the domain so this is will not come into play.

These data can be input all the time irrespective of whether I have a transition or not and according to the plot of  $x_m$  the appropriate data will be taken. If I see this is given, now if I have  $k(x)$  we know we wanted to be a constant  $k_0$ . This  $k_0$  also has to be input as an input data. As far as the material data is concerned, we need to input the point of transition  $x_m$ , we need to input  $a_0$ ,  $a_1$ ,  $b_0$ ,  $b_1$ ,  $E_1$  and  $E_2$  and  $K_0$ . Totally we need 1, 4, 7, 8 data so 8 quantities has to be input. This quantity is a user supplied quantity because, this is something that the user is selling this

corresponds to the domain the material data of interest to me. We have the material data we would like to give the load data.

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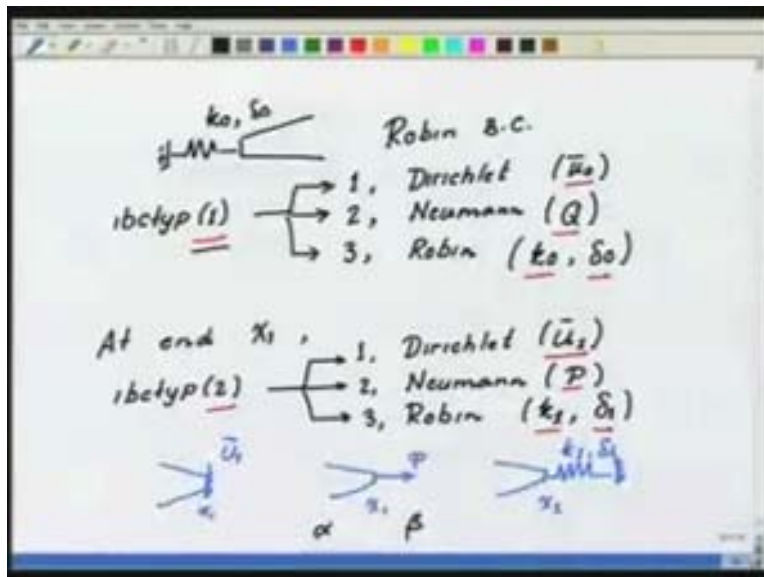


What is that we have to input as far the load data is concerned? We have said that our distributed load is defined over the whole domain if not defined piecewise. It could define in piecewise that it will only add little more to the complexity. Here we want to define it globally that is in the whole domain as  $f(x)$  as I have drawn in the figure is  $f_0$  plus  $f_1 x$  plus  $f_2 x$  square so these quantities  $f_0, f_1, f_2$  has to be input. If I want a constant load I make  $f_1$  and  $f_2$  is equal to 0. If I want a linear load, I will make  $f_2$  equal to 0. If I want no load then I make  $f_0, f_1, f_2$  equal to 0. This way we can upto a quadratic loading as a distributed load that we want.

What else do we want? We want to give whether point load or not, so if there is a point load we would like to know what this point is? Where the point load is applied and the value of the load  $P_1$ ? Both these data has to be given. If I do not want the point load, then in case I do not want it then I will set  $P_1$  equal to 0 and  $x_p$  equal to  $x_p$  plus 1. I am going to make this value 0 and I am moving this point where this 0 load is applied which is meaningless to the end of the load, just for our convenience. We have to remember that when  $P_1$  equal to 0, I will have to set  $x_p$  is equal to  $x_1$  we do not give  $x_p$  equal to some point interior of the domain; this data is available to us.

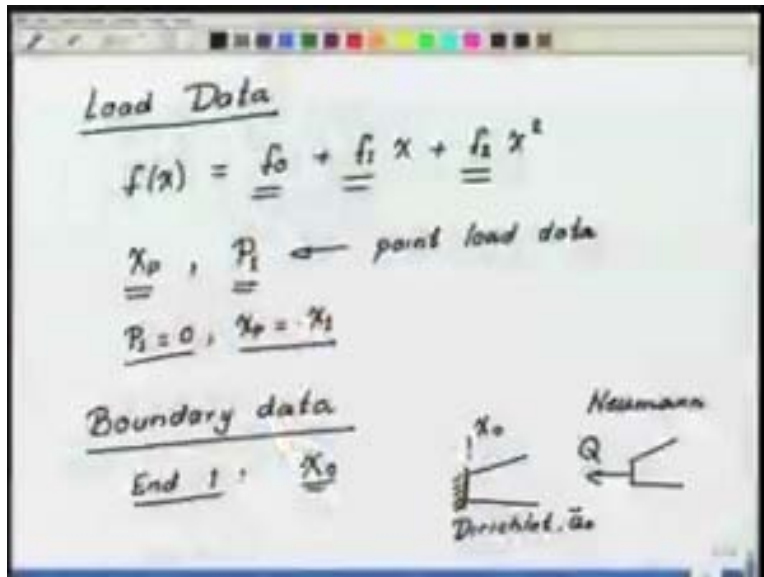
We have the domain data, the material data, and the load data. What else do we need? We need the boundary data because we need to specify the various boundary conditions. Let us now say End 1 (Refer Slide Time: 17:21) this corresponds to the point  $x_0$ . If I take member, what boundary conditions can I have at this point  $x_0$ ? I could have (1) the displacement is given it need not be 0, it could be some value.

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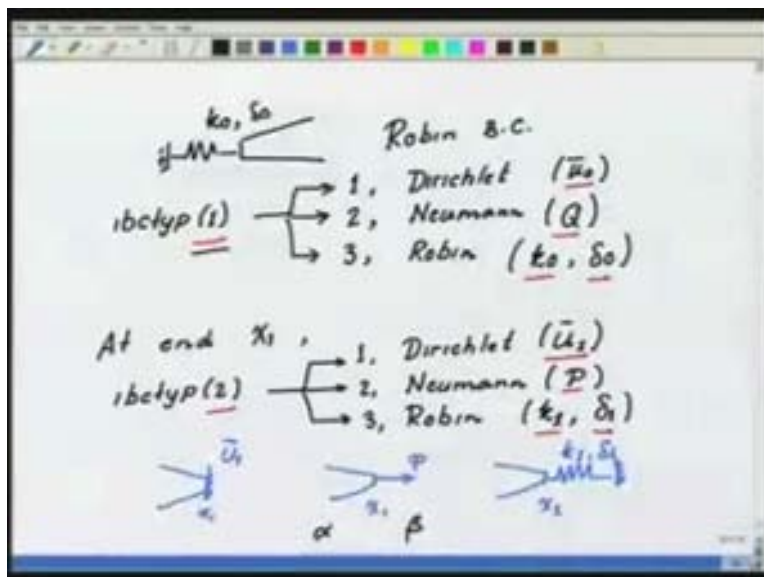
I could have a second candidate here;  $Q$  is given at the end, and the third candidate that the end has a spring with a constant  $k_0$  and an initial compression  $\delta_0$  given. We are going to call them this (Refer Slide Time: 18:10) if we remember that we had introduced this as the Robin boundary condition and the previous one if I go up this is Dirichlet and this one is Neumann.

(Refer Slide Time: 18:13)



We have to have the feature that at this end I should be able to give any one of the boundary conditions and along the boundary condition type I should be able to input the data that goes with it. Here if I say Dirichlet means,  $u_0$  is given as  $\bar{u}_0$ .

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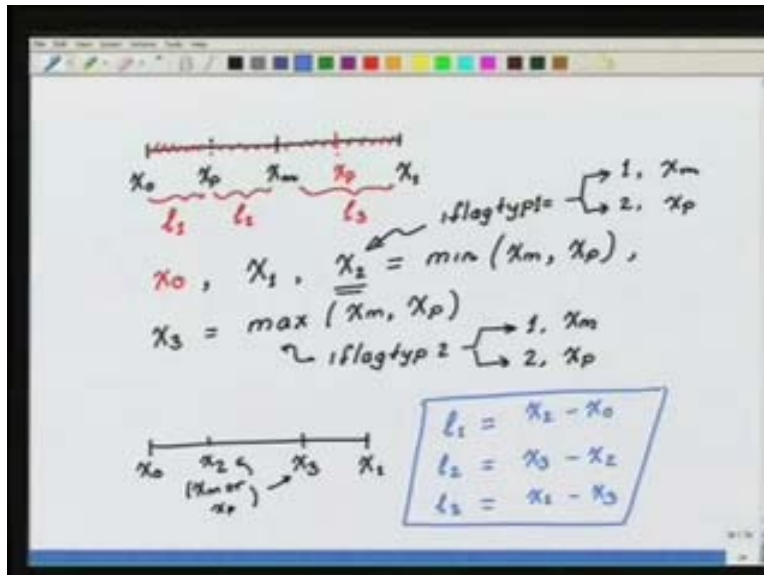
Let us say that we will specify the boundary condition type at End 1 by the following number: it is 1 when it is Dirichlet, it is 2 when it is Neumann, and it is 3 when it is Robin or mixed. When

it is Dirichlet we have to input the value at the end - the displacement  $u_0$  bar; if it is Neumann I have to specify what this end load is. It is  $Q$ , it is positive if it is tensile. If it is Robin then I have to specify the spring constant  $k_0$  and initial compression  $\delta_0$ . These are the information that we need to have at the boundary. Similarly, at the other boundary at end  $x_1$  we say is given by the boundary node 2, I will say of type 2. This would be again 1, 2, or 3 depending on whether it is Dirichlet Neumann or whatever or Robin. If it is Dirichlet then at this end (Refer Slide Time: 20:38) it is  $u_1$  bar, if it is Neumann then it is the end tensile load  $P$  and if it is Robin then we have to give the end spring constant  $k_1$  and the end compression  $\delta_1$ . So all these data has to be input, so this has to be given, this has to be given, this has to be given, and this has to be given the type the boundary condition type has to be specified at each of the end. What is the type and accordingly these data has to be input?

For completeness sake specify what this means at the other end. Here (Refer Slide Time: 21:44) I mean that when it is 1 then I am specifying the displacement here. When it is 2, I am specifying the end load  $P$ , when it is 3, then I am specifying this spring constant and the initial compression  $\delta$  at the end  $x_1$ . As far as the boundary condition is concerned this is the information we would like to have. The boundary condition site and the corresponding data, how do we input this data? We could have one input as the boundary condition type the other input as the two parameters  $\alpha$  and  $\beta$ . For the Dirichlet site, it will give  $\alpha$  equal to  $u_0$  so at each of the end  $\alpha$  equal to  $u_0$  bar. For the Neumann part, at  $n_1$  it will give  $\alpha$  equal to  $Q$  and for the Robin, it will be  $\alpha$  equal to  $k_0$   $\beta$  equal to  $\delta_0$ . Similarly, I will do at the end  $x$  is equal to  $x_1$ ; at the two ends we have to give these two parameters. The second parameter for the first two types is irrelevant parameters but anyway we can specify anything and it will not be used. After I input all these data, I have to create some information which is needed. In the read data program itself we construct that information; what is that information? The information is as follows.



(Refer Slide Time: 23:11)



Let us say this is the point  $x_0$ , this is the point  $x_1$ , this is the point  $x_m$ , the  $x_p$  could be here (Refer Slide Time: 23; 39). The  $x_p$  could be here but I have to honor this transition as we have said. Transition means if the  $x_p$  is here then this is now a piece by itself, this is a piece by itself and this is a piece by itself. We have the 3 pieces which are given by in this case  $x_0$  to  $x_p$ ,  $x_p$  to  $x_m$  and  $x_m$  to  $x_1$  and the length will be given by  $l_1, l_2, l_3$ ; this information has to be available to us. The computer does not know whether the  $x_p$  is happening before  $x_m$  or it is coming after  $x_m$ . This information has to be created for the computer to understand that this partition has these points  $x_p, x_m$  and so on as the end points. Let us now find the three points;  $x_0$  is fixed, we will say  $x_1$  is fixed,  $x_2$  is equal to minimum of  $x_m$  and  $x_p$  and  $x_3$  is equal to maximum of  $x_m$  and  $x_p$ . When we say  $x_2$  is the minimum of  $x_m$  and  $x_p$  along with it we will carry the flag, whether it is material point or it is the point of point load. Let us make the flag, let say that this will have an indicator  $x_2$  will have an indicator  $iflag$  type it is equal to 1 if this is  $x_m$  or it is equal to 2 if it is  $x_p$  (Refer Slide Time: 26:15).

Similarly, here we will call it  $iflag$  type 1. Here for the max again, we will have the corresponding flag associated with it  $iflag$  type 2 this is equal to 1. If this is the point  $x_m$  and it is equal to two if it is  $x_p$ . This is very useful information that we are creating along with this point  $x_2$  and  $x_3$ . It is going to be very useful when we are further constructing the mesh and deciding the element where they lie in, which material domain they lie in. Let us have this flag and along

this point; what do we have now? I look at the domain I have the point  $x_0$ , the point  $x_1$  then I have  $x_2$  and I have the point  $x_3$  and the flag tells me whether it is  $x_m$  or  $x_p$  here and here (Refer Slide Time: 27:38). This is the flag telling me this thing, but as far as the partitioning the domain is concerned we have this distinct partition. I will measure the length  $l_1$  is equal to  $x_2$  minus  $x_0$ ,  $l_2$  is equal to  $x_3$  minus  $x_2$  and  $l_3$  is equal to  $x_1 - x_3$ . These lengths we are going to measure; note something that this  $l_1, l_2$  and  $l_3$  can each be either 0 or  $l$ .

For example, if I say that my material point and both the material point and the point load are at  $x_1$  that is I do not have two material domain and I do not have point load applied in that case this  $x_2$  minus  $x_0$  is going to be  $l$ ,  $x_3$  minus  $x_2$  is going to be 0,  $x_1$  minus  $x_3$  is going to 0 only  $l_1$  going to be coming out to be  $l$  in that case and other two things are coming out to be 0. If I say that point load does not exist, I only have a material transition. If I have material transition then I will only get, in that case,  $x_2$  is equal to  $x_3$ ,  $x_3$  is equal to  $x_1$ . So I get  $l_1$  is equal to  $x_2$  minus  $x_0$ ,  $l_2$  is equal to  $x_3$  minus  $x_2$  which is perfectly fine,  $l_3$  is equal to 0. I will get only the two partitions. With this is data, now do we have all the information that we need? We now need to get one more data which is the mesh data.

(Refer Slide Time: 29:53)

The image shows a whiteboard with handwritten mathematical equations. The main equation is:

$$\int_{x_0}^{x_m} E_1 A_1(x) \frac{du}{dx} \frac{dv}{dx} dx + \int_{x_m}^{x_1} E_2 A_2(x) \frac{du}{dx} \frac{dv}{dx} dx + \int_{x_0}^{x_1} k_0 u v dx = \int_{x_0}^{x_1} f v dx - Q v \Big|_{x_0}^{x_1} - k_1 (u|_1 + \delta_1) + P_1 v|_{x_p}$$

Below this, there are two diagrams illustrating boundary conditions:

- A diagram on the left shows a material boundary at  $x_1$  with a spring constant  $k_1$  and displacement  $\delta_1$ . The displacement on the right side is  $u|_1$ .
- A diagram on the right shows a point load  $P_1$  applied at  $x_p$ . The displacement at that point is  $u|_1 + \delta_1$ .

We have defined the three partitions of the domain remember that these are not nodes, but these are the partitions of the domain honoring the material boundaries the point of application of the

point load and the extremities of the domain so it is  $x_2, x_3, x_1$  with length  $l_1, l_2, l_3$ . We can ask the following question; in each of the partition, how many elements do we need to put? This is an input data which has to come from the user. User decide to put different number of elements in each of the partition based on his needs, based on certain apriory information that he has about the smoothness of the solution and so on.

Let us say that in each of this region, we are specifying the number of elements we need  $n_1, n_2, n_3$  and we are assuming piecewise uniform mesh. What we are saying is, scalar as an input data, what number of sub division we want in the first piece which is length  $l_1$  in the second piece which is  $l_2$  and the third piece which has length  $l_3$ ; so these are given by  $n_1, n_2, n_3$ . Take in each piece a uniform sub division so that I get  $n_1$  number of elements in the first piece,  $n_2$  number of elements in the second,  $n_3$  number of elements in the third. The total number of elements if we see will be equal to  $n_1$  plus  $n_2$  plus  $n_3$ , if all these partitions are distinct.

We see that in case, one of these lengths or two of these lengths become 0, then this partition even though we specified it is going to be trivial information; we really not use that information. What more we need to specify? We have specified all the information that we needed to have with respect to the domain, with respect to the material, with respect to the loading data, with respect to the boundary condition, and here with respect to the number of elements we want in each of the pieces. The last thing that we need to specify is the order of approximation, (small)  $p$  this we are assuming is uniform in the whole domain (Refer Slide Time: 33:14). This is very important that we are going to certify the order of approximation to be uniform. We need not do that, we could specify different orders of approximations which can also be easily handled but, let us stick to that we are going to fix this order of approximation to be uniform.

For the problem that we have written, that is I have drawn on the black board for that problem what is the weak formulation? The weak formulation will come out to be integral from  $x_0$  to  $x_m$  (Refer Slide Time: 34:00) according to the figure on the board. I will have  $E_1, A_1(x) du dx dv dx$  plus integral  $x_m$  to  $x_1$   $E_2 A_2(x) du dx$  plus I will have integral of  $x_0$  to  $x_1$   $k_0 u v dx$ . This is the part due to the spring, this is stiffness part due to the spring; this will be equal to integral  $x_0$  to  $x_1$   $f v dx$  . From the figure on the black board, we have an end load at the point  $x_0$  that will be the tensile load so it will be,  $Q$  into this will give work,  $Q$  into  $v$  at  $x$  equal to  $x_0$ . At the other end at

$x$  equal to  $x_1$ , if I look at the end there let me draw here end  $x_1$ , (Refer Slide Time: 36:05) here I have  $k_1$  and initial compression  $\delta_{11}$ . If there is an initial compression  $\delta_{11}$  then further displacement of this end due to the applied force as the reaction to the applied forces of an amount  $u$  at  $L$ . The spring actually compresses by an amount  $\delta_{11}$  plus  $u$  at  $L$  so the total compression is  $\delta_{11}$  plus  $u$  at  $L$ . The force applied by the spring is back on the member  $P$  equal to minus  $k_1 u$  at  $L$  plus  $\delta_{11}$ . The  $u$  at  $L$  is an unknown, so this part minus  $k_1$ , let me first write it here, minus  $k_1 u$  at  $L$  plus  $\delta_{11}$  multiplied by  $v$  at  $L$ . This part (Refer Slide Time: 37:05) minus  $k_1$  into  $u$  of  $L$  because this is in terms of the unknown, I am going to carry it over to this and I am going to leave to this into  $v$  of  $L$  obviously, I am going to leave minus  $k_1 \delta_{11}$  into  $v$  of  $L$  on the right hand side. This part goes to the left hand side because we are collecting all the unknowns on the left hand side.

Remember this part because this is going to be important as far as applying the boundary conditions is concerned. Similarly, at the end  $0$  we can do the same thing; where this end load  $Q$  which can be now given in terms of the compression of the end  $x_0$  if I had a spring load there. Let us now see if this is the weak formulation. In the weak formulation which we could also derive using the variational principle something else is missing here (Refer Slide Time: 38:28). We have not added the part due to the point load at point  $x_0$ ; so  $v$  at  $x_p$  so this part has to be added to the load vector site.

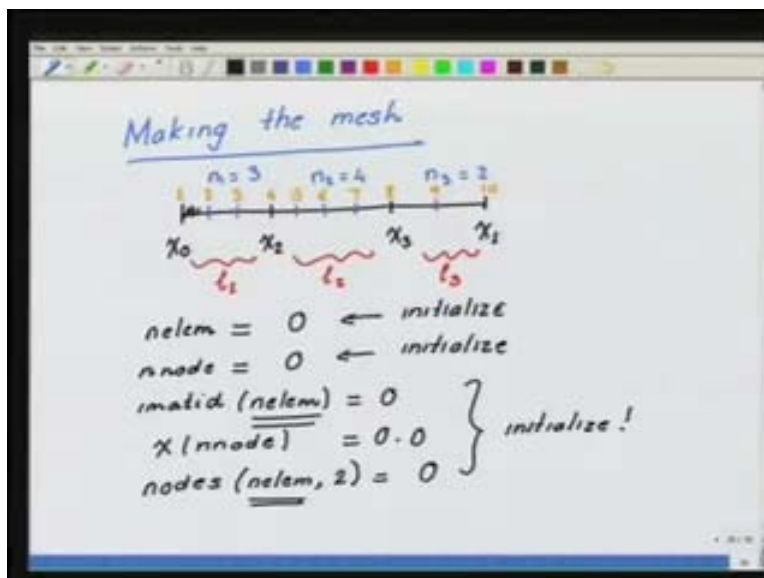
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$$\begin{aligned}
 & \int_{x_0}^{x_m} E_1 A_1(x) \frac{du}{dx} \frac{dv}{dx} dx + \int_{x_m}^{x_1} E_2 A_2(x) \frac{du}{dx} \frac{dv}{dx} dx \\
 & + k_1 u|_L v|_L + \int_{x_0}^{x_1} k_0 u v dx \\
 & = \int_{x_0}^{x_2} f v dx - Q v|_{x_0} - k_1 \delta_{11} v|_{x_1} + P_1 v|_{x_p}
 \end{aligned}$$

If I look at this weak form in its complete entirety, it will be  $\int_{x_0}^{x_1} E_1 A_1(x) du dx$ .  $\int_{x_0}^{x_m} E_2 A_2(x) du dx$ ,  $\int_{x_m}^{x_1} E_2 A_2(x) du dx$ ,  $\int_{x_0}^{x_1} f v dx$  minus the right hand side will be  $\int_{x_0}^{x_1} f v dx$  minus  $Q$  into  $v$  at  $x_0$  minus  $k_1 \delta u$  at  $x_1$  plus  $P_1 v$  at  $x_p$ . This is going to be our complete weak formulation or variational formulation whichever way we decide to obtain.

We see that this thing should reflect in what we are doing in the program. In the program we are going to do the integration piecewise for this material constant because the material constant is changing so I have to go from  $x_0$  to  $x_m$  from  $x_m$  to  $x_1$  and this anyway will be added from the boundary conditions, this is not a problem. This part, (Refer Slide Time: 41:20) since, we are doing this piecewise we could do this from  $x_0$  to  $x_m$   $x_m$  to  $x_1$ ; again it can be piecewise and this is the same thing. Here also in the load I am going from  $x_0$  to  $x_1$  which means I can do it  $x_0$  to  $x_m$  and  $x_m$  to  $x_1$  and this part is from the boundary condition, this part is from the boundary condition. This (Refer Slide Time: 41:44) I will have to add as an extra part due to the point load. With this in mind, what we have developed till now as far as leading the data is concerned; the data is going to be used. Once we have read the data then what is the first thing we have to do? First thing is given that the data that we have read we have to make the mesh.

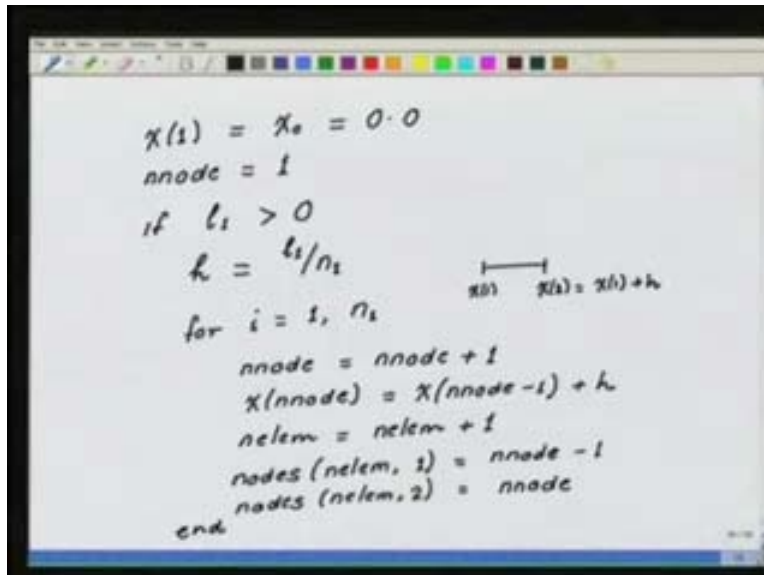
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In order to make the mesh we have said that as far as the computer is concerned it has these points  $x_0, x_1, x_2, x_3$ ; with length  $l_1, l_2, l_3$ . How do we go about constructing the mesh? We want a certain amount of partition in each piece. Let us say in the first one I am putting three elements so here I am going to put three equal elements so here  $n_1$  is equal to 3. Let us say  $n_2$  is equal to 4. I want to put elements here so let us say  $n_3$  is equal to 2 so I put one element here. We should be able to construct these partitions using our program and along with this partition we should be able to construct the coordinates of these nodes we have created. How do we number the nodes? We are going to say this is going to be node 1, this is going to be node 2, 3, 4. We are going to number them sequentially 6, 7, 8, 9, 10 so this thing should be possible while we are making mesh as we are numbering the node we should be able to give the coordinate of the node. Let total amount of element in the mesh will be given by the parameter  $n_{elem}$  that is the total number of elements. First we are going to initialize it.

Further, we are going to say that let the total number of nodes in the element will be given by  $n_{node}$ ; we are going to initialize this also. The initialization of the quantity is very important. We have written the program, we have to know what material properties to use in a particular element? If I am interested in this element, how does the computer know what  $E$  to put there? What  $A$  to put there? So somehow I have to inherit the information for each of the element from somewhere. We are going to have an ID or a pointer or an indicator of which material domain a particular element lies in through this vector  $imatid$ , this is the integral. To this we are going to set this is of size, so we fix some total number of elements. Let us say we say the maximum number of elements in the mesh cannot be more than hundred. We will initialize, we make this vector of size hundred and initialize all entries to 0. Further, we will have the nodes of each of the points also initialized to 0. Also, we will have another information which we need; we will say it will create these array nodes, this is also initialized to 0 (Refer Slide Time: 46:50); so all entries of this array which is if we see this element is based so this total number element is 100. This is 100, 2 array; all entries of this is set to 0. These are all the initializations that we will have to do.

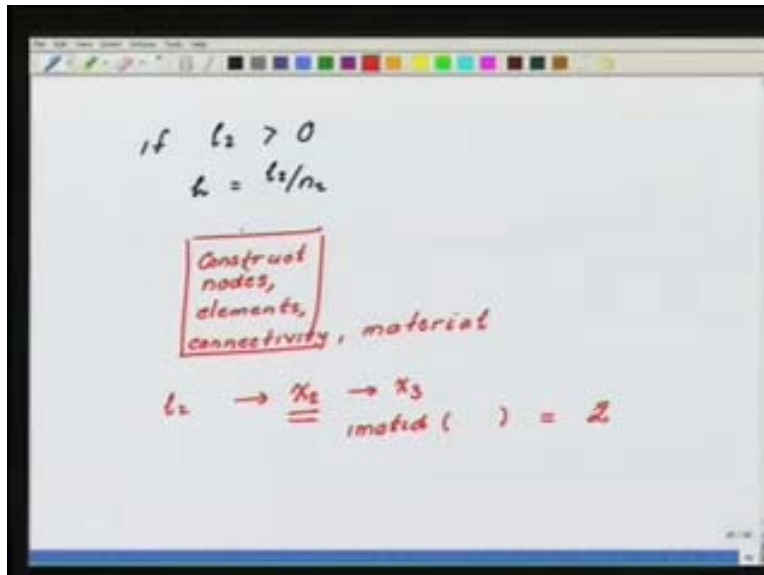
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After this initialization, we are going to construct the nodes. The first node is given the value  $x_0$  which is equal to 0.0. I am going to set, because I already created this node,  $n$  node is equal to 1 (Refer Slide Time: 47:47). I am incrementing the total number of nodes in the mesh as I am making the nodes.

If  $l_1$  is greater than 0, if  $l_1$  is equal to 0, I do not do anything. If  $l_1$  is greater than 0, then I find the mesh type because this is uniform meshing in this case  $l_1$  divided by  $n_1$ . For  $i$  equal to 1 to  $n_1$  that is I take all these partitions, I am going to increment the number of nodes and I am going to say  $x$   $n$  node is equal to  $x$   $n$  node minus 1 plus  $h$ . Given the first node which is  $x_1$ , I created the coordinate of  $x_2$  is equal to  $x_1$  plus  $h$  and so on. I construct all the  $n$  plus 1 nodes in this piece. As I am constructing these consecutive nodes I am also incrementing the number of elements. So I am going to say  $nelem$  is equal to  $nelem$  plus 1 that is two consecutive nodes have formed an element in between. I am going to say  $nodes$   $nelem$  comma 1 is equal to  $n$  node minus 1  $nodes$   $nelem$  comma 2 is equal to  $n$  node. I am telling what are the two  $n$  nodes for this two particular element are. Then I am going to end this loop. I have found the mesh size in the piece, looped over to load in the piece, found what the loads are, and constructed it and so on.

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I repeat the same process for  $l_2$ ; we will say if  $l_2$  is greater than 0 then I will find  $h$  is equal to  $l_2$  by  $n_2$  and continue. Similarly, I will do for  $l_3$  for each of the piece. While I am constructing the element, I am also going to check whether the first point that is the  $x_2$  if we remember  $x_2$  and  $x_3$  that is the first point it is due to a material interface or whether it is due to a point load. If it is due to a material interface then we know that  $l_1$  is from  $x_0$  to  $x_2$  so here I am going to give  $imtid$  of element is equal to 1 or 2 depending on where it interfaces, at least here depending on what this flag was (Refer Slide Time: 51:38). This I am going to appropriately assign at least in the first phase it is always going to be free element, the material one.

This way I continue here also (Refer Slide Time: 51:50) I assign while I construct the node, so construct nodes, elements and connectivity and then material information which material domain it lies in. In this case for  $l_2$  we know that  $l_2$  actually goes from  $x_2$  to  $x_3$ . If  $x_2$  was a material point then here my  $imtid$  of all the elements will be equal to 2. If  $x_2$  was  $x_3$  that is, it was a point load point, not a material point then it remains as 1 because the material is not changing because only the point load has been applied and so on we continue doing. This gives us all the information about the mesh, the elements, nodes, the coordinate, connectivity, number of elements, all these information comes out at the end of the program. This is a very important component of the pre-processor that is we have made the mesh and all the relevant information that has to go with the mesh as far as our computations are concerned. We will continue with this in the next lecture and



we are going to now talk of how to construct the degrees of freedom information for the given mesh for the given order of approximation. From there we will continue further, talk about assembly solver and so on.

Thank you.