

Computational Fluid Dynamics for Incompressible Flows
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Lecture 37
Boundary Conditions

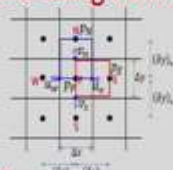
Hello everyone, so in last class, we discussed about the simple algorithm and also we have derived the pressure correction equation from the continuity equation. Simple algorithm has been widely used in the literature, but there are number of attempts to accelerate the convergence. So, today we will discuss two such algorithms, one is simpler and another is simplec. So, there is a drawback a simpler algorithm.

So, dropping the summation of u' and v' slows down the convergence. So today we will discuss how to accelerate this simple algorithm. So, first we will discuss about the simpler algorithm. So, what is simpler? It is simple devised. So, semi-implicit method for pressure limit equation revised.

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Solution of Navier-Stokes Equations using FVM

SIMPLER
 Revised



$$a_e u_e = \sum a_{nb} u_{nb} + b_e - \Delta y (P_E - P_P)$$

$$a_w u_w = \sum a_{nb} u_{nb} + b_w - \Delta x (P_W - P_P)$$

$$u_e = \frac{\sum a_{nb} u_{nb} + b_e}{a_e} - d_e (P_E - P_P)$$

$$u_w = \frac{\sum a_{nb} u_{nb} + b_w}{a_w} - d_w (P_W - P_P)$$

$$d_e = \frac{\Delta y}{a_e} \quad d_w = \frac{\Delta y}{a_w}$$

$$d_n = \frac{\Delta x}{a_n} \quad d_s = \frac{\Delta x}{a_s}$$

$$u_e = \hat{u}_e - d_e (P_E - P_P) \quad u_w = \hat{u}_w - d_w (P_W - P_P)$$

$$v_n = \hat{v}_n - d_n (P_N - P_P) \quad v_s = \hat{v}_s - d_s (P_P - P_S)$$

Discretized continuity eqn

$$F_e - F_w + F_n - F_s = 0$$

$$F_e = \rho_e u_e \Delta y = \rho_e \hat{u}_e \Delta y - \rho_e d_e \Delta y (P_E - P_P) = \hat{F}_e - \rho_e d_e \Delta y (P_E - P_P)$$

$$F_w = \rho_w u_w \Delta y = \rho_w \hat{u}_w \Delta y - \rho_w d_w \Delta y (P_P - P_W) = \hat{F}_w - \rho_w d_w \Delta y (P_P - P_W)$$

$$F_n = \rho_n v_n \Delta x = \rho_n \hat{v}_n \Delta x - \rho_n d_n \Delta x (P_N - P_P) = \hat{F}_n - \rho_n d_n \Delta x (P_N - P_P)$$

$$F_s = \rho_s v_s \Delta x = \rho_s \hat{v}_s \Delta x - \rho_s d_s \Delta x (P_P - P_S) = \hat{F}_s - \rho_s d_s \Delta x (P_P - P_S)$$

Solution of Navier-Stokes Equations using FVM

$$F_e - F_w + F_n - F_s = 0$$

$$\hat{F}_e - \hat{F}_w + \hat{F}_n - \hat{F}_s - \rho_e d_e \Delta y (P_e - P_p) + \rho_w d_w \Delta y (P_p - P_w) - \rho_n d_n \Delta x (P_n - P_p) + \rho_s d_s \Delta x (P_p - P_s) = 0$$

$$(\rho_e d_e \Delta y + \rho_w d_w \Delta y + \rho_n d_n \Delta x + \rho_s d_s \Delta x) P_p = \rho_e d_e \Delta y P_e + \rho_w d_w \Delta y P_w + \rho_n d_n \Delta x P_n + \rho_s d_s \Delta x P_s - (\hat{F}_e - \hat{F}_w + \hat{F}_n - \hat{F}_s)$$

$$a_p P_p = a_e P_e + a_w P_w + a_n P_n + a_s P_s + b$$

$$a_p P_p = \sum a_{nb} P_{nb} + b$$

where $a_p = a_e + a_w + a_n + a_s = \sum a_{nb}$
 $b = -(\hat{F}_e - \hat{F}_w + \hat{F}_n - \hat{F}_s)$
↳ It does not represent the mass balance.

SIMPLE R, okay. So, this is your simple semi-implicit method for pressure linked equation and R for revised. So, this is your simpler algorithm. So, you have seen that in simple algorithm, first we need to guess the pressure field and if you have a good pressure field obviously the simple algorithm will converge faster.

In simpler algorithm, we calculate the pressure from the known velocity field. Now, let us write the discretized momentum equation for u_e and v_n , okay. So for u_e we can see at this main control volume for velocity u_e we are writing the discretized equation $a_e u_e$ is equal to summation of $a_{nb} u_{nb}$, okay, for all the neighbors plus the source term for u at point u_e and the pressure gradient term.

What is the pressure gradient term? So, you can see so, in the, for u velocity. So, the pressure gradient is $P_e - P_p$ into Δy . So, minus Δy into $P_e - P_p$, okay, and an v_n is equal to so for the V momentum equation you can see here we are solving for v_n . So, discretized equation you can write for this control column as summation of $a_{nb} v_{nb}$ plus b source term for b at point n minus, so what is the area Δx and the gradient is $P_n - P_p$, okay.

So, now, let us write the velocities dividing the diagonal term, so, you can write u_e as summation of $a_{nb} u_{nb}$ plus b_{ue} divided by a_e , okay, and minus we will write this as $a_e (P_e - P_p)$, okay where a_e is your Δy by Δx , Δy by Δx . Similarly for v_n you can write v_n is equal to summation of $a_{nb} v_{nb}$ plus b_{vn} divided by a_n minus $d_n (P_n - P_p)$, where d_n is your Δx divided by the diagonal coefficient a_n , okay.

So, this term, okay, will represent as u_e hat, okay and these term will represent as v_n hat. So, if you represent like this, then you will then we can write u_e is equal to u_e hat minus $d_e P_e$ minus P_p and v_n is equal to v_n hat minus $d_n P_n$ minus P_p , okay. So, similarly you can write for the u_w and u_s . So, u_w you can write u_w hat minus d_w into P .

So, for the west space you can see the pressure gradient is P_p minus P_w , so, it is P_p minus P_w and for v_s you can write and the pressure gradient for this south space u_s you can write P_p minus P_s and d_w is your Δy by a_w and d_s , is your Δx divided by a_s , okay. So, now let us write the discretized continuity equation. So, integrating the continuity equation about this main control volume P , okay.

In last class we have written the equation as F_e minus F_w plus F_n minus F_s is equal to 0. So, this is your discretized continuity equation, okay. So, now if you put this mass flow rate F_e is equal to $\rho_e u_e a_e$ then what you can write, so F_e you can write as, so, F_e will be, okay, $\rho_e u_e$ and what is a_e ? a_e is Δy , okay, a_e is Δy .

So, you can write this as, so now, u_e you just substitute u_e as u_e hat minus $d_e P_e$ minus P_p . So, if you write that then you can write $\rho_e u_e$ hat Δy , okay, minus ρ_e . So, now, this is the term so, it is d_e you have Δy into P_e minus P_p , okay. So, now you can see this $\rho_e u_e$ hat Δy we can write it as F_e hat minus $\rho_e d_e \Delta y P_e$ minus P_p . Now, similarly, you just write F_w , F_n and F_s .

So, F_w will be $v_w \rho_w u_w \Delta y$ and u_w is this one. So, it will be $\rho_w u_w$ hat Δy minus $\rho_w d_w \Delta y P_p$ minus P_w and this if you represent with if F_e hat w then it will be $\rho_w d_w \Delta y P_p$ minus P_w . Similarly, we write a F_n is equal to $\rho_n v_n \Delta x$, ok. So, v_n so, v_n is this one. So, you can write, $\rho_n v_n$ hat Δx minus $\rho_n d_n \Delta x$ into P_n minus P_p and this will represent as F_n hat.

So, it will be $\rho_n d_n \Delta x P_n$ minus P_p and F_s is equal to $\rho_s v_s \Delta x$ is equal to ρ_s , so, this one sorry this is your v_e , so, it will be v_s hat Δx minus $\rho_s d_s \Delta x P_p$ minus P_s is equal to F_s hat minus $\rho_s d_s \Delta x$ into P_p minus P_s . So now we will substitute in the discretized continuity equation, okay. So, if you substitute what you will get?

So, we have F_e minus F_w plus F_n minus F_s is equal to 0, okay, and now you substitute all these four and write F_e plus F_w plus F_n plus F_s . So, it will be. So, now, first you write F_e minus F_w

plus a F_n minus F_s , okay. So, if you write it, so we will get F_e hat minus F_w hat plus F_n hat minus F_s hat, okay, and rest we have minus $\rho_e \frac{d}{dy} P_e$ minus P_p , okay. For F_w , so it is minus F_w so minus-minus plus so $\rho_w \frac{d}{dy} P_p$ minus P_w , okay.

Then for F_n you will get minus $\rho_n \frac{d}{dx} P_n$ minus P_p and plus $\rho_s \frac{d}{dy} P_p$ minus P_s , okay, is equal to 0. So, this is your $\frac{d}{dx} P_n$ equal to 0, okay. So, now, you see we are deriving the pressure equation from the known velocity field, so, now you can write in terms of F_e Φ P is equal to summation of $n_b P_p$ n_b . So, you can write a P_p , P_p is equal to summation of $a_n b P_n$ plus b in this form. So, all the pressure P_p you take in the left hand side and all other terms you take in the right hand side.

So, if you do that you can see, we have these P_p , P_p , and P_p . So, these coefficients you take in the left hand side. So, if you write it, it is minus-minus plus it will be $\rho_e \frac{d}{dy} P_e$ then this plus $\rho_w \frac{d}{dy} P_p$ minus-minus plus. So, it will be $\rho_n \frac{d}{dx} P_n$ and this is your plus $\rho_s \frac{d}{dx} P_s$, okay. So, this is the coefficient of P_p and now you take these terms in the right hand side, okay.

So, what you will write, so it will be if you take in the right hand side it will be positive $\rho_e \frac{d}{dy} P_e$, okay, plus $\rho_w \frac{d}{dy} P_w$ plus $\rho_n \frac{d}{dx} P_n$ and we have $\rho_s \frac{d}{dx} P_s$ and these terms you take in the right hand side. So, you can write minus F_e hat minus F_w hat plus F_n hat minus F_s hat, okay. So, now, these you can write as $a_p P_p$ is equal to $a_e P_e$ plus $a_w P_w$ plus $a_n P_n$ plus $a_s P_s$ plus b , okay.

Or $a_p P_p$ is equal to summation of $a_n b P_n$ plus b where a_p is equal to a_e plus a_w plus a_n plus a_s you can see the coefficient a_e , this is your a_e a_w , a_w , this is you are a_n , this is your a_n and this is your a_s and this is your a_s . So, a_p is equal to summation of all the neighbor and b is equal to minus if F_e hat minus F_w hat plus F_n hat minus F_s hat, okay, and we have a_e as $\rho_e \frac{d}{dy} P_e$ a_w as $\rho_w \frac{d}{dy} P_p$ a_n is equal to $\rho_n \frac{d}{dx} P_n$ and a_s is equal to $\rho_s \frac{d}{dx} P_s$, okay.

So, in the simpler algorithm from the known velocity field first we derive this pressure equation. So, first you solve this pressure equation in simpler algorithm, which we are actually first guessing the pressure field, instead of that, you solve this equation and get a better pressure field so that it converges faster. So, you can see whatever this, whatever we have written, it actually does not represent the mass balance, okay.

Because \hat{p} is your $\rho \hat{v}$, so, it will not actually satisfy the continuity equation. So, it does not represent the mass balance and you have seen that up to these we have not made any approximation while deriving the pressure equation. So, this pressure with the appropriate boundary condition, this pressure equation with appropriate boundary condition you need to solve to get a pressure field and using this pressure field now you follow the simple algorithm, okay.

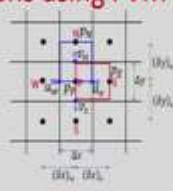
So, then all the rest algorithm will be similar to that simple algorithm except you do not correct the pressure, okay. When you solve the pressure correction equation in the simple algorithm. So, here when you are using simpler algorithm, you do not correct the pressure, because already you are solving the pressure equation, ok. But you need to correct the velocity field. So, now we will write the simpler algorithm.

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Solution of Navier-Stokes Equations using FVM

SIMPLER algorithm

1. Guess the velocity field
2. Calculate \hat{F}_s
3. Solve pressure equation, \hat{p}
4. Solve u^*, v^*
5. Calculate \hat{F}_s^*
6. Solve pressure correction equation, \hat{p}'
7. Correct velocity only.
 - $u = u^* + u'$
 - $v = v^* + v'$
 - Do not correct pressure.
8. Check for convergence.
 - If converged, stop.
 - Else go to step 2.



So, simpler algorithm like these, okay. So, first is guess the velocity field, okay. Once you guess the velocity field you can calculate the \hat{F}_s . So, if \hat{F}_e , \hat{F}_w , \hat{F}_n , \hat{F}_s you can calculate. Then you solve for the pressure equation, okay. So, if the velocity field is exact the correct pressure field will be recovered, okay. So, towards the convergence when you have the velocity field, correct velocity field anyway the, you will get a correct pressure field.

Then, now, you follow the simple algorithm, okay. So, now we have solve u^* , v^* , okay. So, this is now guess pressure field, okay. So, what do you have solved the pressure equation that

you will use in this equation to solve u^* , v^* , okay. Then calculate f^* and then solve for the pressure correction equation, pressure correction equation, okay. So, whatever we derived in the simple algorithm that you solve, okay. Then correct velocity only.

So, now you know u^* , and you have solved the P' , so from P' you can calculate the u' , okay. So, there is a relation and v is equal to v^* plus v' . Do not correct pressure, okay. Because in this step, at third step you have already solved for the correct pressure field with the known velocity field, okay. So, here you do not correct pressure so this is the difference with the simple algorithm.

Then check for convergence, okay so, if converged, stop, okay, else go to step 2. So, you go to step 2 with the new velocity whatever you have got it, okay with this you calculate this f^* and solve for the pressure equation. So whatever velocity now you are getting so that will be used to solve this pressure equation. So, you can see that in simpler algorithm, although there is you need to solve one additional equation for pressure, but overall it will take less time to converge as you are giving a correct result field, okay.

In a simpler algorithm if you use a correct guessed pressure field then obviously we will get faster result but it is very difficult to get a good guessed pressure field. But here you are solving for the pressure with the known velocity field. So obviously, it will converge faster than the simple algorithm, but you need to solve one additional equation for the pressure. Next, we will discuss another algorithm that is simpler, okay. Simple, corrected.

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Solution of Navier-Stokes Equations using FVM

SIMPLEC
L Corrected

$$a_e u'_e = \sum a_{nb} u'_{nb} - \Delta y (P'_e - P'_p)$$

$$a_n v'_n = \sum a_{nb} v'_{nb} - \Delta x (P'_n - P'_p)$$

Instead of ignoring these completely
SIMPLEC algorithm attempts to approximate
the neighbor corrections using the cell correction as

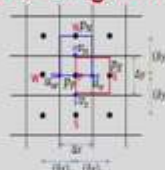
$$\sum a_{nb} u'_{nb} = u'_e \sum a_{nb}$$

$$\sum a_{nb} v'_{nb} = v'_n \sum a_{nb}$$

$$a_e u'_e = u'_e \sum a_{nb} - \Delta y (P'_e - P'_p)$$

$$a_n v'_n = v'_n \sum a_{nb} - \Delta x (P'_n - P'_p)$$

$$(a_e - \sum a_{nb}) u'_e = -\Delta y (P'_e - P'_p) \Rightarrow u'_e = \frac{-\Delta y}{a_e - \sum a_{nb}} (P'_e - P'_p) = -d_e (P'_e - P'_p)$$

$$(a_n - \sum a_{nb}) v'_n = -\Delta x (P'_n - P'_p) \Rightarrow v'_n = \frac{-\Delta x}{a_n - \sum a_{nb}} (P'_n - P'_p) = -d_n (P'_n - P'_p)$$


Simple C, so this is your corrected. So, in this algorithm, it is similar to simple algorithm except without dropping the term, we will just use summation of $a_{nb} u'_{nb}$ is equal to u'_{nb} , $b u'_{nb}$ is equal to u'_{nb} , $b u'_{nb}$ is equal to u'_{nb} . So, with the main cell value of u'_{nb} , we will use, the main cell value of u'_{nb} at cell center small e we will use instead of dropping the full terms.

So, that is the difference with simple algorithm. So if you see, so we have the corrected momentum equation, we have derived in the simple algorithm like this, okay. If you see that we have written first the momentum equation, then we have guessed the pressure field then we have written $a_e u'_{nb}$ and subtracting the second one from the first we got this corrected momentum equation.

So, that is $a_e u'_{nb}$ is equal to summation of $a_{nb} u'_{nb}$ minus $\Delta y (P'_e - P'_p)$ and $a_n v'_{nb}$ is equal to summation of $a_{nb} v'_{nb}$, okay, minus $\Delta x (P'_n - P'_p)$. So, if you see, in a simple algorithm what we did, we drop these 2 terms, okay. So, we drop these terms, but now we will use some simplification here. So, instead of ignoring this completely simplec algorithm attempts to approximate the neighbor corrections using cell correction as, so we will use summation of $a_{nb} u'_{nb}$ we will write as u'_{nb} summation of a_{nb} , okay.

So, now you can see. So, these actually it has to be summation of all these $a_{nb} u'_{nb}$ so neighbor (coefficient) neighbor values but neighbor corrections. So that we are approximating as

the cell correction e' similarly summation of anb , v' nb we will write it as v' summation of anb , okay. So v' obviously, you can see, so, this is your correction at the cell of this v control volume.

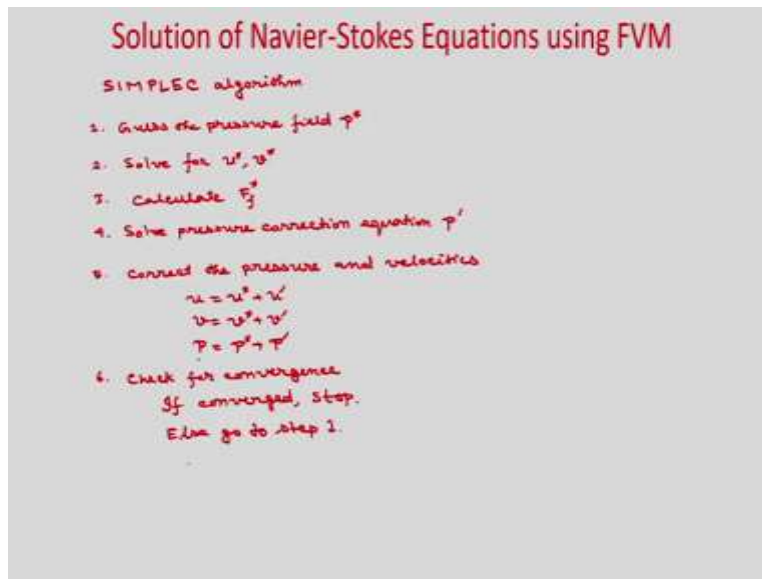
So, now, from this equation now, if you write it then you can write, $ae u'$. So, this now we are approximating as this. So, it will be u' summation of anb minus $\Delta y P_e'$ minus P_p' . Similarly, an v' you can write to this we are now approximating as this. So v' summation of anb minus $\Delta x P_n'$ minus P_p' , okay. So, now if you take in left hand side, so, what you can write ae minus summation of anb , okay.

So, now you can take common this u' , so, u' is equal to minus $\Delta y P_e'$ minus P_p' and now, this you can write as you u' is equal to minus Δy divided by ae minus summation of $anb P_e'$ minus P_p' and this we will denote as d_e , okay. So it will be minus d_e , minus d_e into P_e' minus P_p' . Similarly, for this equation you can write an minus summation of $anb v'$ n is equal to minus $\Delta x P_n'$ minus P_p' .

So, that will give you v' is equal to minus Δx divided by an minus summation of $anb P_n'$ minus P_p' and this we will denote as minus d_w , sorry minus d_n into P_n' minus P_p' , okay. So, these we have represented as d_e and these we have represented as d_n , okay. So, now, you can see that we have approximated these Navier correction value, okay, which we dropped in the simpler algorithm.

So, that is the difference with the simple algorithm. So, in the simplec algorithm now we follow the same procedure as simple except calculating this term d_e and d_n .

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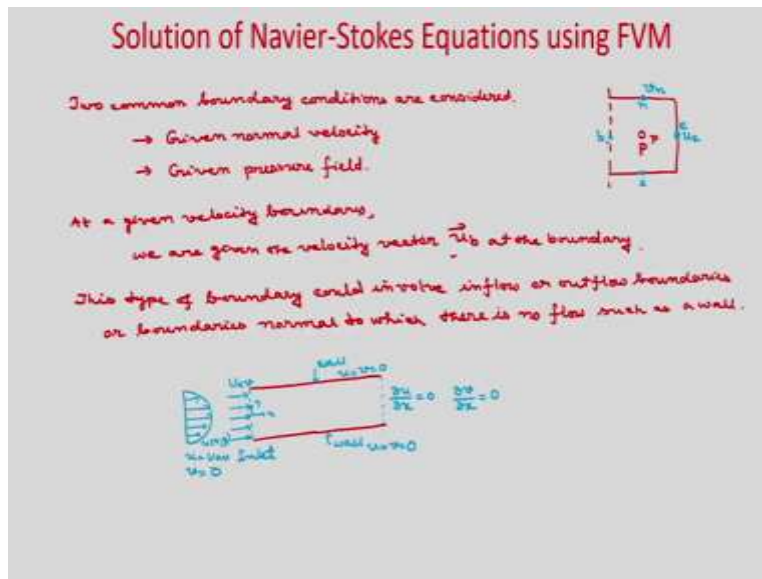


Now, let us write the simplec algorithm, simplec algorithm. So first, now similar to simple algorithm, guess the pressure field, guess the pressure field, p^* , okay. Once you guess the pressure field solve for u^* and v^* , okay. So, once you solve for u^* , v^* you can calculate F_f^* , once you know the F_f^* you can solve for the pressure correction equation, pressure correction equation, p' , okay.

Now you correct the pressure and the velocities but do not under relax it, okay the pressure, correct the pressure and velocities. So, u correct u as $u^* + u'$ v is equal to $v^* + v'$ and P is equal to $P^* + P'$, okay. Now check for convergence. So, if converged stop, if converged stop, else go to step 1, okay. So with the, now this pressure you can take as the guessed pressure field, okay.

So, the simplec algorithm is actually shown as faster than the simple algorithm because you are not fully dropping this summation of u' and v' . But also here you do not need to solve additional equation as you do in the simpler algorithm. So, now, we will discuss about the boundary conditions while solving these discretized equations. Mostly you know the velocity boundary conditions again we will discuss, but we have derived the pressure correction equation. So, we need to know the boundary conditions for this pressure correction p' .

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So, if you consider boundary in the left side let us say this is your cell, okay. So, this is your main control volume P where you are solving small P and here you are solving u_e here are you solving v_n and this is your boundary, okay. So this is your small e, this your small n, this your small s and the west space is now boundary, okay, for this particular case let us write the boundary conditions.

So, generally two boundary conditions we will discuss, one is normal velocity is given or the pressure is given, okay.

So two common boundary conditions are considered, one is your given normal velocity, another is given pressure field, ok. So, you can see that the velocity is known at the boundary then you can represent for the fluid flow problem in flow, okay, where velocity will be specified at the inlet, outflow where velocity may be specified at the outlet and the no slip walls where you have velocities are considered to be 0.

So, at a given velocity boundary we are given the velocity vector u_b . So, at this boundary, okay, u_b is specified okay. So, in that now if it is inflow then velocity will be provided if it is outflow, if you know the outflow boundary condition then also we can use the velocities specified or we will can use some outflow boundary condition and if it is a wall then there will be no penetration through the boundary and velocities will be 0.

So, this type of boundary could involve inflow or outflow boundaries or boundaries normal to which there is no flow such as a wall, okay. So, if you see, let us consider a channel. So, let us consider flow between 2 parallel plates. So, third direction is infinite and we are considering flow between 2 parallel plates. So, we can see that if you want to give the velocity boundary condition then on the walls there will be no slip.

So, velocities will be 0, at the inlet you can specify the velocity boundary condition, you can specify as a uniform velocity boundary condition or some time varying or specially varying boundary condition. So, at this boundary okay you can specify their uniform velocity boundary condition. So, you can see let us say u average you are given, or you can specify a parabolic boundary condition also where u is function of y , okay.

So, if it is x and this is your y then u is function of x and y . So, this is your inlet similarly at the outlet you can generally specify the normal direction velocity gradient is 0, $\frac{\partial u}{\partial x}$ is equal to 0 and $\frac{\partial v}{\partial x}$ is equal to 0, okay. So, it is a kind of fully developed condition. So, no axial change of velocities in this direction and these are the no slip condition. So, wall, so, no slip condition.

So, you can put u is equal to v is equal to 0 and u is equal to v is equal to 0. If you are specifying normal velocity then here u is equal to u average and v is equal to 0. So, this in detail we have discussed already in earlier modules. So, now, let us discuss about the pressure correction boundary condition.

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Solution of Navier-Stokes Equations using FVM

$\checkmark a_p P_p' = \sum a_{nb} P_{nb}' + b$
 $b = -(F_e^* - F_w^* + F_n^* - F_s^*)$

For pressure correction equation,

$F_e - F_w + F_n - F_s = 0$ ✓
 $F_e' = -\rho_e d_e \delta y (P_e' - P_p')$
 $F_w' = \rho_w d_w \delta y (P_w' - P_p')$
 $F_n = F_n^* + E_n'$
 $F_s = F_s^* + E_s'$

For known velocity at boundary, the known flow rate is incorporated directly into the mass balance equation for cell P.

At a given pressure boundary, the pressure correction P_b' is set equal to zero.

$P_b' = 0$

$F_e' = -\rho_e d_e \delta y (P_e' - P_p')$
 $F_w' = \rho_w d_w \delta y (P_w' - P_p')$
 $F_b = F_b^* + F_b'$

Velocity is known at the boundary.

$F_b' = 0$
 $T_e' - P_b' = 0$
 $P_b' = P_p'$
 $\frac{\partial P'}{\partial x} = 0$

So, if you see the pressure correction equation is $a_p P_p'$ is equal to summation of $a_{nb} P_{nb}'$ plus b okay where b is equal to minus, $F_e^* - F_w^* + F_n^* - F_s^*$, okay.

So, for pressure correction equation, if you write the continuity equation in general so it is $F_e - F_w + F_n - F_s = 0$ and that substituting the star quantities and the prime quantities we have derived these pressure correction equations, okay. So, for this boundary let us say again let us consider these left boundary. So, this is your capital P, this is your small e, small n, small s and this is your west space which is your boundary, okay.

So, for east space if you write the F_e' then it will be $\rho_e d_e \delta y (P_e' - P_p')$, and F_e^* is your $\rho_e u_e^* \delta y$. So, F_e you have written as $F_e^* + F_e'$. Now, you see if velocity is specified at the boundary okay then mass flow rate is known. So, you do not need any correction in the mass flow rate, okay. So, you can see this is your F_e is equal to $F_e^* + F_e'$ and if velocity is known, then this is your correct velocity, okay.

So, that means, you do not need any correction F_e' . So, F_e' will be 0, so you can see that your mass flow rate we have written in terms of the star quantities plus the correction quantity. Now, for this boundary if you write then F_b' is equal to you can write $\rho_b d_b \delta y (P_b' - P_p')$ so this is not being δy , yes $\delta y (P_b' - P_p')$, okay. So, now you can see that if velocity is known then F_e is equal to F_b is equal to $F_b^* + F_b'$.

So, at the boundary velocity is known then the mass flux is known. If mass flow rate is known, so if velocity is known the mass flow rate is known, if mass flow rate is known, you do not need any correction at the boundary. So, that means, in this case your F_b prime will be 0, okay. Because you do not need any correction because you have the velocity field is specified. okay. Velocity is known at the boundary, okay.

So, F_b prime. So, you do not need actually any boundary condition for the pressure correction, because this term you will not add while adding in this equation, okay. So, wherever this F_w which is actually coming as F_b , so, F_b will be just in terms of velocities not in terms of the pressure correction. So, you do not need any boundary condition for pressure correction for the known velocity.

And in other words also you can say that if it is 0, if F_b prime is 0 then this will be 0 that means, it will be P_p prime minus P_b prime will be 0 and hence, you can write these as the P_b prime is equal to P_p prime that means, at this point you are applying the kind of normal boundary condition P_p prime by Δx is equal to 0, okay, so this is x direction this is your y direction so obviously, Δp prime by Δx_0 however these pressure correction values are not required for the boundary.

Because you will not add it in the pressure correction equation while solving it, okay, and if pressure is specified at the boundary what will happen? So, in general when you have a flow inside a channel at the outlet we specify the pressure which is maybe the atmospheric pressure open to the atmosphere, so the, you can set as the atmospheric pressure at the outlet.

So, if it is so, then you do not need any correction at the boundary, right, so, that means P prime will be 0 at the boundary. So, at a given pressure boundary the pressure correction, the pressure correction, P_b prime is set equal to 0 that means P_b prime will be 0, okay. So, you can see that if pressure is specified at the boundary, then you do not need any actually correction in the pressure.

So, P_b prime will be 0 and if velocity specified, then you will know the mass flow rate and if you know the mass flow rate you directly put it in the pressure correction equation so that you do not need any pressure correction boundary condition that means. So, for the known velocity at

boundary the known flow rate is incorporated directly into the mass balance equation for cell P, okay.

So, which you have written here. So, there we directly put the value of F_b is equal to $\rho b u \Delta y$. So, this we directly put so because F_b prime is 0, okay. So, this we directly put for this F_w which is for this particular case it is F_b and rest we can represent in terms of pressure correction, okay. So, those are interior points so, you and if the pressure is specified then P is P^* plus P' , right.

So, P_b is P_b^* plus P_b' . So, if pressure is specified then P_b is known so you do not need any pressure correction. In today's class we will discuss 2 variants of simple algorithm which are known as simpler, simple revised and simplec, simple corrected. So, these algorithm we have discussed, which are actually to accelerate the convergence rate.

So, in simpler algorithm, we have seen that instead of guessing the pressure field with the known velocity field we solve for the pressure equation and that is obviously a good guess pressure field and it converges faster than the simple algorithm. Rest of the algorithm is similar to simple algorithm except you do not need to correct the pressure because you are solving a pressure equation.

So, that is your correct pressure field from the known velocity field so you do not correct the pressure in the simpler algorithm. In simplec algorithm, it is also faster than the simple algorithm. Here, without dropping the Navier correction term, you write it in terms of the correction term of the cell center other algorithm is similar to simple algorithm, but as you are not dropping these summation of $\sum u' n_b$ you get faster convergence than the simple algorithm.

Finally, we discussed the velocity boundary condition and the pressure correction boundary condition. For pressure correction boundary condition if velocity is known, so, you know the mass flux. So, you know the mass flow rate so, you do not need any pressure correction boundary condition, and if pressure is specified at the boundary, then you know do not need any pressure correction. So, that means P' at the boundary is 0. Thank you.