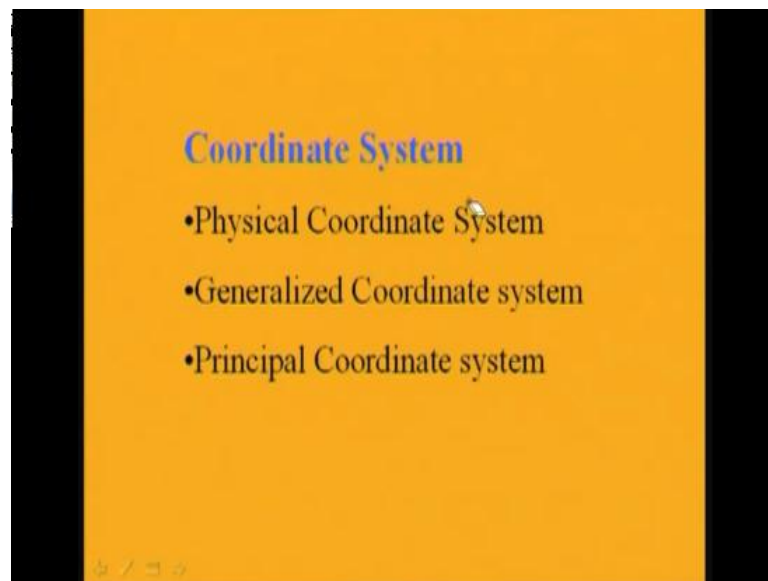


**Mechanical Vibrations**  
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**Module - 5**  
**Two DOF Free Vibrations**  
**Lecture - 2**  
**Lagrange's equation**

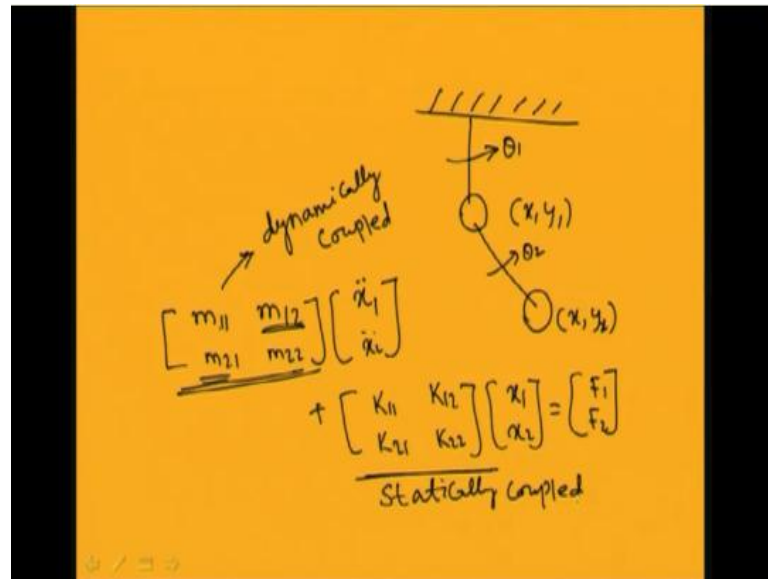
In the last class, we have studied about the two degree of freedom system. There we have found the equation motion by using the Newton's principle or d'Alembert principle; also you have used Lagrange's principle and extended Hamilton principle to derive the equation motion. So, in case of two degrees of freedom system, we required minimum two coordinates to define the motion of the system. And in this case, I have already told you about three different coordinate systems.

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One is the physical coordinate system, second one is the generalized coordinate system, and the third one is the principle coordinate system. In case of a physical coordinate system, you can fix a physical coordinate or you can fix a reference frame. And from this reference frame, you just take the coordinates of this system or coordinates of the point where you are finding or using which you are finding the equation motion. In case of generalized coordinates, so these are the coordinates or minimum number of coordinates required to express the motion of the system completely.

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So, already you have seen in the case of a double pendulum. So, this is the double pendulum I have already told. So, in this case of a double pendulum, you have used  $x_1$   $y_1$   $x_2$   $y_2$  as the physical coordinates and this  $\theta_1$  rotation of this  $\theta_1$  and rotation of this  $\theta_2$  as the generalized coordinates. And already I told you by using these coordinates, you can write the equation motion in case of a two degree of a freedom system and you can write it in the matrix form.

So, in the matrix form you can write it like this  $m_{11}$   $m_{12}$   $m_{21}$  and  $m_{22}$   $x_1$  or in this case you can write it equal to  $\theta_1$  double dot or you can write in terms of  $x_1$   $x_2$ . So, you can write it is equal to  $x_1$  double dot  $x_2$  double dot plus  $k_{11}$   $k_{12}$   $k_{21}$   $k_{22}$   $x_1$   $x_2$  equal to  $f_1$   $f_2$ . So, in case of free vibration this force term the right hand side terms are zero and I told you about the dynamic and static coupling. So, if this mass matrix is coupled that is all the terms are present are half diagonal; some of the half diagonal terms are present, then the system is said to be dynamically coupled, so dynamically coupled.

So, in this case some of the terms of diagonal terms that is this is the diagonal terms; this  $m_{11}$  and  $m_{22}$  are the diagonal terms. So, this one and these two are the half diagonal terms. If these half diagonal terms are not zero, then the system is still to be dynamically coupled. Similarly, in case of the stiffness matrix if the half diagonal terms are not zero or if their present some half diagonal terms, then the system is said to be statically

coupled. So, already you have seen the case of statically coupled and dynamically coupled system.

So, in this case to make the system dynamically and statically uncoupled you can take another set of coordinate system. So, those coordinate systems using which you can write the equation motion in the uncoupled form, or the system will reduce to a dynamically uncoupled and statically uncoupled form are known as the principle coordinate system. So, the principle coordinate systems are the generalized coordinate system by using which you can write the equation motion in an uncoupled way or in that case the dynamic matrix and static matrix will be uncoupled. So, in that case it will reduce to two single degree of free system equation and using these two equations you can find the solution of the system.

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**DERIVATION OF EQUATION OF MOTION**  
Lagrange Principle

$$\frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}_k} \right) - \frac{\partial T}{\partial q_k} = Q_k \quad Q_k = \sum F_i \frac{\partial r_i}{\partial q_k}, k=1,2,\dots,n$$

$$\frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}_k} \right) - \frac{\partial T}{\partial q_k} + \frac{\partial V}{\partial q_k} = Q_{knc}$$

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_k} \right) - \frac{\partial L}{\partial q_k} = Q_{knc} \quad L = T - V$$

So, already I told you about the extended Hamilton principle and Lagrange's Principle. In case of Lagrange principle I told you, you can use this equation  $\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_k} \right) - \frac{\partial L}{\partial q_k} = Q_{knc}$ . So, here  $L$  is the Lagrangian of the system that is equal to  $T$  minus  $V$ . So, this Lagrangian  $L$  equal to  $T$  minus  $V$ ,  $T$  is the kinetic energy,  $V$  is the potential energy of the system, and this  $Q_k$  is the generalized force which you can find from this. So, in this equation  $F_i$  is the force acting at  $i$ th station, and  $r_i$  is the position vector of the  $i$ th point on this system.

So, small  $q_k$  is the generalized coordinates you are taking. So, in these two degrees of freedom system  $k$  will be equal to 2. So, it will be either 1 or 2 and using this formula you can find the generalized coordinates.

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Lagrange equation including damping

$$\frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}_k} \right) - \frac{\partial T}{\partial q_k} + \frac{\partial D}{\partial \dot{q}_k} + \frac{\partial V}{\partial q_k} = Q_k$$

And for a system with damping, you can use this dissipation energy; this  $d$  is the dissipation energy. And using this dissipation energy you can find the equation motion.

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Extended Hamilton's Principle

$$\int_{t_1}^{t_2} (\delta T + \delta \bar{W}) dt = 0,$$

$$\delta r_1(t_1) = \delta r_2(t_2) = 0, i = 1, 2, \dots, N$$

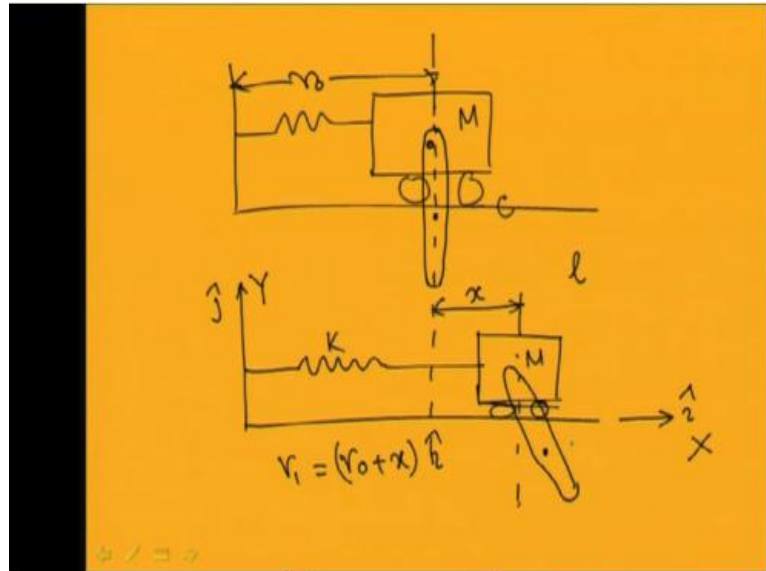
$$\int_{t_1}^{t_2} (\delta L + \delta \bar{W}_{nc}) dt = 0, \quad L = T - V$$

$$\underline{\delta q_k}(t_1) = \underline{\delta q_k}(t_2) = 0$$

So, already also I told you about the extended Hamilton Principle. In extended Hamilton principle, you can write the equation motion by using this equation. So, it is equal to del

$L$  plus  $\delta W$  n  $C$  d  $t$  equal to zero, where this virtual displacement  $\delta q_k$  at  $t_1$  equal to virtual displacement and  $\delta q_k$  at  $t_2$  equal to 0.

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So, previously we have derived the equation motion of a system. So, these systems we have taken; to derive the equation motion we have take a spring and a mass. So, these spring mass system we have taken. So, in this spring mass system one rod is hanging from this cart. So, this is the mass. So, from this mass, a rod is hanging, and I have retained this point C. So, let C is the position of the mass centre of this rod. So, for simplicity I have assumed this mass centre is situated at  $l/2$  distances from this  $l/2$  distance. Let  $l$  is the length of the rod and at  $l/2$  distance this mass centre is situated.

So, when this body is not moving or when it is not undergoing any vibration. So, this is the undeformed position and the deformed position if I will plot, so in the deformed position. So, this is the undeformed position, and you can draw the deformed position. So, in the deformed position the mass is at this position. So, the rod position is like this. So, this is the rod. So, this is the mass centre. So, you can have. So, this is the spring and the spring you can draw it. So, spring is traced, and this is the spring.

So, if I will take this undeformed position as  $r_0$  and I can take this displacement as  $x$ . So, I can take a coordinate system this direction this is  $x$ , and position of this unit vector is  $i$ , and this is  $y$  direction, and unit vector I can take this. So, position vector of this mass

m can be written as  $r_0$  plus. So,  $r_1$  I can write equal to  $r_0$  plus  $x$   $i$  and the position vector of point C already I told you that position vector of...

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Position vector of point C =

$$\left( r_0 + x + \frac{l}{2} \sin \theta \right) \hat{i} - \frac{l}{2} \cos \theta \hat{j}$$

$$\vec{V}_c = \left( \dot{x} + \frac{l}{2} \cos \theta \dot{\theta} \right) \hat{i} + \frac{l}{2} \dot{\theta} \sin \theta \hat{j}$$

Kinetic Energy  $T = \frac{1}{2} M \dot{x}^2 + \frac{1}{2} m \vec{V}_c \cdot \vec{V}_c + \frac{1}{2} I_c \dot{\theta}^2$


So, point C you can write. So, position vector of point C equal to. So, position vector of point C is given by  $r_0$  plus  $x$  plus  $\frac{1}{2} l \sin \theta$   $i$  minus  $\frac{1}{2} l \cos \theta$   $j$ . So, from this by differentiating this position vector, you can find the velocity of that mass centre. So,  $V_c$  will be equal to. So, if you differentiate this thing, this part is constant. So, it will become  $\dot{x}$  plus  $\frac{1}{2} l \cos \theta \dot{\theta}$   $i$  plus  $\frac{1}{2} l \dot{\theta} \sin \theta$   $j$ . And kinetic energy already I told you this is equal to  $\frac{1}{2} M \dot{x}^2$  plus  $\frac{1}{2} m \vec{V}_c \cdot \vec{V}_c$  plus  $\frac{1}{2} I_c \dot{\theta}^2$ .

So, if you are taking in these. So, this mass centre either you can take the body or the kinetic energy of this body by taking the purely rotation about this point or you can take the translation and rotation about the mass centre. So, both will yield the same thing. So, if you are taking the mass centre. So, it will have a translational kinetic energy that is  $\frac{1}{2} m \vec{V}_c \cdot \vec{V}_c$  and plus rotational kinetic energy, or if we are taking the rotation about these then it will be if I will take this point  $i$ . So, it will be  $\frac{1}{2} I_0 \dot{\theta}^2$ . So, this  $I_0$  is nothing but it will be equal to  $m l^2$  by 3. So, if you add these two terms, it will give the same expression.

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$$T = \frac{1}{2} M \dot{x}^2 + \frac{1}{2} m \left[ \left( \dot{x} + \frac{l}{2} \dot{\theta} \cos \theta \right)^2 + \left( \frac{l}{2} \dot{\theta} \sin \theta \right)^2 \right] + \frac{1}{2} \frac{m l^2}{12} \dot{\theta}^2$$

$$= \frac{1}{2} \left[ (M+m) \dot{x}^2 + m l \dot{x} \dot{\theta} \cos \theta + \frac{1}{3} m l^2 \dot{\theta}^2 \right]$$

$$\checkmark \text{ P.E.} = V = \frac{1}{2} K x^2 + m g \frac{l}{2} (1 - \cos \theta)$$


So, in this case you already know that this kinetic energy is given by this. So, this is the expression. So, one-third  $m l^2 \theta \dot{\theta}^2$  already you are getting, and this is due to the translation of this, and this is the couple term. So, now you can find the potential energy of the system. So, potential energy due to the spring; so that is equal to  $K x^2$  and due to change in position of the rod. So, initially the rod was straight; now it has come to this position.

So, this change in position; so this is the mass centre at a distance  $l$  by  $2$ . So, this angle you have taken  $\theta$ . So, change in position will be this to this. So, this length initially it is  $l$  by  $2$  at a distance  $l$  by  $2$ . Initially the mass centre is here; now it has come to this position. So, this is the change in the position of the mass. So, potential energy equal to  $m g$  into change in position of the mass. So, this is equal to  $m g$  into  $l$  by  $2$   $1 - \cos \theta$ . So, this is the potential energy  $V$  and already you know the kinetic energy.

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$$L = T - V = \frac{1}{2} \left[ (M + m) \dot{x}^2 + mL \dot{x} \dot{\theta} \cos \theta + \frac{1}{3} mL^2 \dot{\theta}^2 \right]$$

$$- \left[ \frac{1}{2} kx^2 + mg \frac{L}{2} (1 - \cos \theta) \right]$$

$$\frac{\delta L}{\delta x} = \frac{1}{2} \left[ (M + m) 2x \delta x + mL \delta x \dot{\theta} \cos \theta + \right]$$

$$- \left[ \frac{1}{2} k \cdot 2x \delta x + mg \frac{L}{2} (+ \sin \theta \delta \theta) \right]$$

So, the Lagrangian of the system equal to T minus V equal to this. So, now you just take this del operator. So, this del operator will give you. So, x dot square if you take the del of this. So, it becomes 2 x dot in to delta x dot. Similarly, for this you have product of three terms here. So, you can write this as m L delta x dot into theta dot cos theta plus now you just take m L x dot into delta theta dot into cos theta, and the third term will be differentiation of cos. So, it becomes minus.

So, minus m L x dot theta dot m L x dot theta dot sin theta delta theta, and for this it becomes one-third m L square. So, this becomes 2 theta dot delta theta dot and for these terms. So, for these terms this becomes half half k into 2 into x into delta x plus m g into half. So, this term you can write. So, this is equal to 1 by 2. So, this is 1 by 2, and this one differentiation one is zero and minus cos theta equal to plus sin theta and theta differentiation with del. So, this becomes del theta. So, this term becomes del theta. So, now you can take these terms.



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$$\delta L = [(M+m)2\dot{x} + ml\dot{\theta}\cos\theta]\delta\dot{x} + \frac{1}{6}ml(3\dot{x}\cos\theta + 2l\dot{\theta})\delta\dot{\theta} - kx\delta x - \frac{1}{2}ml(\dot{x}\dot{\theta} + g)\sin\theta\delta\theta$$

So, this delta L you now know. So, you can write this del L equal to m plus m 2 x dot plus ml theta dot cos theta in to delta x dot. Now you have to integrate this whole thing from t 1 to t 2.


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$$\overline{\delta W}_{nc} = F\hat{i}\cdot\delta\{(r_0 + x + l\sin\theta)\hat{i} - l\cos\theta\hat{j}\} = F\delta x + Fl\cos\theta\delta\theta$$

From extended Hamilton's Principle

$$\int_{t_1}^{t_2} (\delta L + \overline{\delta W}_{nc}) dt = 0,$$

$$\delta x = 0, \delta\theta = 0, \text{ at } t = t_1, t_2$$

$$r = (r_0 + x + l\sin\theta)\hat{i} - l\cos\theta\hat{j}$$


So, while integrating this thing from t 1 to t 2, you have to take these following or you have to do in the following way. And before that you should find the work done by this non-conservative force and already you know the position vector of the mass. So, this is the rod where a horizontal force F is acting. So, this horizontal force as it is acting in i

direction. So, it is written  $F \cdot i$ . So, work done will be  $F \cdot i \cdot \text{virtual displacement}$ . So, this is the position vector; you just find the position vector of this point. So, this point position vector will be equal to.

So, this is equal to position vector of this point equal to  $r_0 \cdot x + l \sin \theta \cdot i - l \cos \theta \cdot j$ . So, this is the position vector of this point. So, you have to find  $\delta r$ . So, you have find  $\delta r$ . So,  $F \cdot i \cdot \delta r$ . So, this is  $\delta r$ , and you can find this is equal to  $F \cdot \delta x + F \cos \theta \cdot \delta \theta$ . So, using this extended Hamilton principle  $\delta \int_{t_1}^{t_2} (L + W_{nc}) dt = 0$ . So, here  $\delta x$  would be equal to 0, and  $\delta \theta$  will be equal to 0 at  $t = t_1$  and  $t = t_2$ .

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The image shows a handwritten derivation on a yellow background. At the top, it shows the integral  $\int_{t_1}^{t_2} (\delta L + \delta W_{nc}) dt$ . To the right, there is a note:  $\delta \dot{x} = \delta \left( \frac{\partial x}{\partial t} \right) = \frac{\partial}{\partial t} (\delta x)$ . Below this, the main expression is expanded into three terms within large square brackets, all multiplied by  $dt$  and integrated from  $t_1$  to  $t_2$ :

$$= \int_{t_1}^{t_2} \left[ \left\{ (M+m)\dot{x} + \frac{1}{2}ml\dot{\theta} \cos \theta \right\} \delta \dot{x} + \frac{1}{6}ml(3\dot{x} \cos \theta + 2l\dot{\theta}) \delta \dot{\theta} + (-kx + F) \delta x + \left\{ F \cos \theta - \frac{1}{2}ml(\dot{x}\dot{\theta} + g) \sin \theta \right\} \delta \theta \right] dt$$

So, using this expression now you have to expand these terms. So, while expanding this term you just note that this  $\delta \dot{x}$  and this is  $d \dot{x}$ . So, while integrating this  $\delta \dot{x}$  you can change. So, operators see this  $\dot{x}$  equal to  $\frac{dx}{dt}$ . So,  $\delta \dot{x}$  you can write it equal to  $\frac{\delta x}{dt}$ . So, you can exchange this and you can write it is equal to  $\frac{\delta x}{dt}$  by  $\frac{dx}{dt}$ . So, this way you can write this  $\delta \dot{x}$  term and  $\delta \theta$  term. So,  $\delta \theta$  also you can write  $\frac{\delta \theta}{dt}$  by  $\frac{d\theta}{dt}$  and you are integrating it with respect to  $dt$ . So, it will be the integration.

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1<sup>st</sup> term

$$\int_t^{t_2} \left[ (M+m)\dot{x} + \frac{1}{2}ml\dot{\theta}\cos\theta \right] \frac{d}{dt}(\delta x) dt$$

$$= \left[ (M+m)\dot{x} + \frac{1}{2}ml\dot{\theta}\cos\theta \right] \delta x \Big|_t^{t_2} -$$

$$\int_t^{t_2} \left[ (M+m)\ddot{x} + \frac{1}{2}ml\ddot{\theta}\cos\theta - \frac{1}{2}ml\dot{\theta}^2 \sin\theta \right] \delta x dt$$

So, the integration will be like this. So, this term I have expressed in this form. So, the first term I have exchanged it d by d t of del x d t and in the second terms also I have written in the same way.

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$$\int_t^{t_2} (\delta L + \delta W_{nc}) dt$$

$$= \int_t^{t_2} \left[ \underbrace{\left( (M+m)\dot{x} + \frac{1}{2}ml\dot{\theta}\cos\theta \right) \delta \dot{x}}_{\delta \dot{x} = \delta \left( \frac{\partial x}{\partial t} \right) = \frac{\partial}{\partial t}(\delta x)} + \frac{1}{6}ml(3\dot{x}\cos\theta + 2l\dot{\theta})\delta\dot{\theta} + (-kx + F)\delta x + \left( Fl\cos\theta - \frac{1}{2}ml(\dot{x}\dot{\theta} + g)\sin\theta\delta\theta \right) \right] dt$$

So, if I integrate this first term. So, this is the first term I am taking. So, this first term if you integrate it.

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1<sup>st</sup> term

$$\int_{t_1}^{t_2} \left[ (M+m)\dot{x} + \frac{1}{2}ml\dot{\theta}\cos\theta \right] \frac{d(\delta x)}{dt} dt$$

$$= \left[ (M+m)\dot{x} + \frac{1}{2}ml\dot{\theta}\cos\theta \right] \delta x \Big|_{t_1}^{t_2} - \int_{t_1}^{t_2} \left[ (M+m)\ddot{x} + \frac{1}{2}ml\ddot{\theta}\cos\theta - \frac{1}{2}ml\dot{\theta}^2 \sin\theta \right] \delta x dt$$

So, you can take the first. So,  $M$  plus  $m \dot{x}$  plus half  $m l \dot{\theta} \cos \theta$  into  $d$  by  $d$  of  $\delta x$   $d t$ . So, this will be equal to. So, you just take this as the first function, one as the first function and other as the second function. So, this is the first function you take, and this is the second function you take. So, you just integrate it by parts. So, integrating it by parts; so first function remain as it is and integration of the second function. So,  $d$  by  $d t$  of  $\delta x$ . So, when you are integrating this thing. So, it becomes  $\delta x$  only from  $t_1$  to  $t_2$ .

Already you know that this  $\delta x$  at  $t_1$  and  $t_2$  are 0. So, this term will tend to 0, and the remaining term will be minus integration  $t_1$  to  $t_2$ . So, this is the first function remain as it is integration of the second. So, now minus integration of the second; so this  $\delta x$  and differentiation of the first. So, differentiation of first will give you. So,  $\dot{x}$  differentiation is giving  $\ddot{x}$ , and these differentiation is giving half  $m l \dot{\theta} \dot{\theta}$  differentiation  $\theta$  double dot  $\cos \theta$  minus. So, this contain two terms. So, you will have two terms here. So, minus  $m l \dot{\theta} \cos \theta$  differentiation is  $\sin \theta$ .

So, this becomes minus  $\sin \theta$  into  $\dot{\theta}$ . So,  $\dot{\theta}$  into  $\dot{\theta}$  giving rise to  $\dot{\theta}^2$ . So, this is equal to half  $m l \dot{\theta}^2 \sin \theta$   $\delta x$   $d t$ . So, in this way you should integrate it by parts. So, using integration by parts, this term you have seen that it is equal to 0, and the remaining term is this.

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2<sup>nd</sup> term

$$\int_{t_1}^{t_2} \frac{1}{6} ml (3\dot{x} \cos \theta + 2l\dot{\theta}) \frac{d}{dt} (\delta\theta) dt \quad \int \frac{d(F\theta)}{\delta\theta}$$

$$= \frac{1}{6} ml (3\dot{x} \cos \theta + 2l\dot{\theta}) \delta\theta \Big|_{t_1}^{t_2} -$$

$$\int_{t_1}^{t_2} \frac{1}{6} ml (3\ddot{x} \cos \theta - 3\dot{x}\dot{\theta} \sin \theta + 2l\ddot{\theta}) \delta\theta dt$$

Similarly, you can carry out the second terms. So, it is equal to  $\frac{1}{6} ml (3\dot{x} \cos \theta + 2l\dot{\theta}) \delta\theta$  at  $t_2$  minus  $\frac{1}{6} ml (3\dot{x} \cos \theta + 2l\dot{\theta}) \delta\theta$  at  $t_1$ . So, this is the first function you can also take. So, up to this you can take the first function. So, this is the first function, and this is the second function. So, integration of the second function  $\frac{d}{dt} \delta\theta$  dt. So, this is equal to integration  $\delta\theta$ . So, this becomes  $\delta\theta$ .

So, integration of this equal to  $\delta\theta$ , and so first function into  $\delta\theta$  at  $t_2$  and then  $\delta\theta$  into differentiation of this. So, differentiation of this is this. So, this becomes this. So, this is  $t_1$  to  $t_2$ . So, this is. So, this part already you know that this virtual displacement  $\delta\theta$  at  $t_1$  equal to  $t_2$  equal to 0. So, this part becomes 0.

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$$\begin{aligned} & \text{3rd term} \\ & \int_{t_1}^{t_2} (F - kx) \delta x dt \\ & \text{4th term} \\ & \int_{t_1}^{t_2} \left[ F \cos \theta - \frac{1}{2} m (\dot{x}\theta + g) \sin \theta \right] \delta \theta dt \end{aligned}$$

Similarly, you can go for the third term. Third term is F minus k x delta x d x. So, you just note that you need not have to go further or integrate it further; you just keep it, keep the term in this form. So, something into del x to d t. Similarly, the fourth term becomes this. So, now you combine the terms with del x d t or del theta d t.

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Hence combining the 4 terms,

$$\begin{aligned} \Rightarrow & \int_{t_1}^{t_2} \left[ - \left\{ (M+m)\ddot{x} + \frac{1}{2} m \ddot{\theta} \cos \theta - \frac{1}{2} m \dot{\theta}^2 \sin \theta \right\} \delta x dt \right. \\ & \left. + (F - kx) \right] \\ & + \int_{t_1}^{t_2} \left[ - \frac{1}{6} m (3\ddot{x} \cos \theta - 3\dot{x}\dot{\theta} \sin \theta + 2\ddot{\theta}) + \right. \\ & \left. F \cos \theta - \frac{1}{2} m (\dot{x}\theta + g) \sin \theta \right] \delta \theta dt = 0 \end{aligned}$$

And combining that thing you just write this. So, this becomes this into del x d t plus integration this into del theta d t. Already you know that this del x and del theta are arbitrary as these are the virtual displacement. So, it can take any value. So, this

integration to be equal to zero; in that case the coefficients of this  $\frac{dx}{dt}$  or  $\frac{d\theta}{dt}$  should be equal to zero. So, as the coefficients of these two are zero, you can find the equation motion from this. So, this part will be equal to 0, and this part will be equal to zero. So, these are the equation motion of this system.

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Equations of motions are

$$(M + m)\ddot{x} + \frac{1}{2}ml\ddot{\theta}\cos\theta - \frac{1}{2}ml\dot{\theta}^2\sin\theta - kx = F$$

$$\frac{1}{6}ml(3\ddot{x}\cos\theta - 3\dot{x}\dot{\theta}\sin\theta + 2l\ddot{\theta}) +$$

$$\frac{1}{2}ml(\dot{x}\dot{\theta} + g)\sin\theta = Fl\cos\theta$$

So, in case of this Hamilton using the extended Hamilton principle, so you got the equation motion. So, this is the first equation motion this equal to zero, and this is the second equation motion that is equal to zero. So, you are equating the coefficients of  $\frac{dx}{dt}$  equal to zero to get the first equation. And equating  $\frac{d\theta}{dt}$  equal to zero to get the second equation. So, in this way you can get the two equations. So, in this case you just see that for this in this case you are getting the equation motion to be in this form.

So, this is  $M + m \ddot{x} + \frac{1}{2}ml\ddot{\theta}\cos\theta - \frac{1}{2}ml\dot{\theta}^2\sin\theta - kx = F$  and  $\frac{1}{6}ml(3\ddot{x}\cos\theta - 3\dot{x}\dot{\theta}\sin\theta + 2l\ddot{\theta}) + \frac{1}{2}ml(\dot{x}\dot{\theta} + g)\sin\theta = Fl\cos\theta$ . So, in this way you can find these two equations. So, these equations you just observe that these are non-linear equations as it contain the product of different terms.

So, this non-linear equation you may make linearization to find the linearized equation. So, you can substitute this for  $\theta$  to be small. So, you can substitute  $\cos\theta$  equal to

$1 \sin \theta$  equal to  $\theta$ . And you may neglect the product of the higher order terms, and you can write the linearized equation motion. So, similarly you can find the same equation motion by using the Lagrange principle also.

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The image shows handwritten mathematical expressions on a yellow background. The first three lines are:
 
$$T =$$

$$V$$

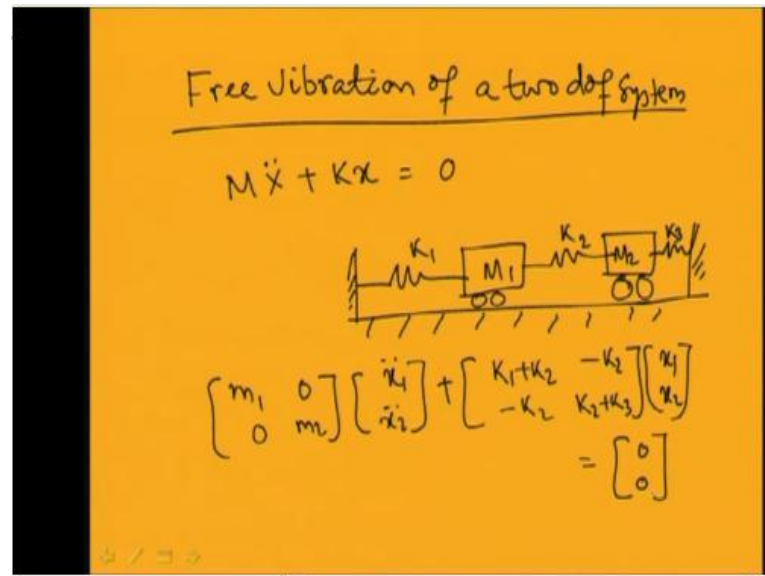
$$L$$
 The fourth line is the Lagrange equation:
 
$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_k} \right) - \frac{\partial L}{\partial q_k} = Q_k$$
 There is a small number '4' in the bottom left corner of the yellow area.

So, by using the Lagrange principle already you have found the kinetic energy of the system, already you have found the potential energy of the system. So, you know the Lagrangian of the system. So, after knowing the Lagrangian of the system, then you just use this equation. So,  $\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_k} \right) - \frac{\partial L}{\partial q_k} = Q_k$ . So, by using this equation you can find the equation in this case. So, you will get the same equation whatever you have got here by using the extended Hamilton principle.

So, till now you know the four different methods to find the equation motion of the system. First you know the Newton's second law, second by using this d'Alembert principle, third the Lagrange principle, and fourth the extended Hamilton principle. Generally, this Lagrange principle is used for multi degree of freedom systems, and Hamilton principle is used for continuous or distributed mass systems. So, most of the cases for this multi degree of freedom system or two degree of freedom system, you should use Lagrange principle or the Newton's or d'Alembert principle instead of going for this extended Hamilton principle.



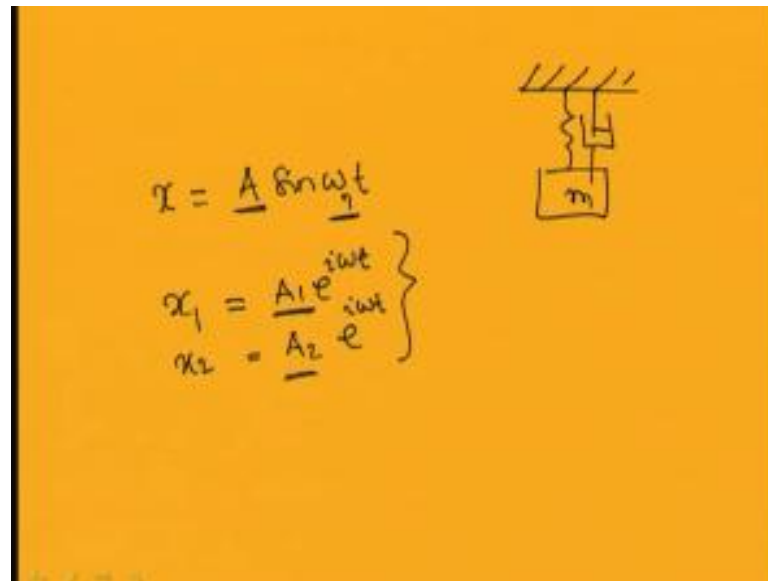
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So, now we will see the free vibration of a two degree of freedom system. So, free vibration of a two degree of freedom system. So, already you have derived the equation motion, and you have written the equation motion in this form. So, mass matrix plus. So, mass matrix or in matrix form we have written the equation motion in this form that is  $M \times$  double dot plus  $k \times M \times$  double dot plus  $k \times$  equal to 0. So, let me take this simple system. So, this is the spring mass damper system.

So, with mass  $M_1$  and already I have derived the equation motion for the system. So, this is the system I have taken. This is  $k$ ; I have taken this is  $k_1$ ; this is  $k_2$ ; this is  $k_3$ , and I have derived the equation motion for the system. And already you know the equation motion for this system equal to  $m_1 \ 0 \ 0 \ m_2 \ x_1 \ \text{double dot} \ x_2 \ \text{double dot} \ \text{plus} \ k_1 \ \text{plus} \ k_2 \ \text{minus} \ k_2 \ \text{minus} \ k_2$ . This is  $k_2 \ \text{plus} \ k_3 \ x_1 \ x_2 \ \text{equal to} \ 0$ . Now I want to study the free vibration response of the system. So, in case of a single degree of freedom system you have seen the free vibration response can be written in this form.

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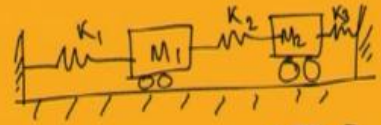


So, in case of a single degree of spring mass damper system, you have seen this is the mass. So, you have seen the solution to be written in this form. So, your  $x$  you have written in this form  $A \sin \omega_n t$ . So, you have written where  $\omega_n$  is the natural frequency of the system, and  $A$  is the amplitude of the response. So, you have written the free vibration response of the system in this way.

Similarly, in this case also I can assume the solution to be in this form  $x_1$  equal to  $A e^{i\omega t}$  or I can assume it  $e^{i\omega t}$  to the power  $i\omega t$ . And similarly,  $x_2$  I can assume it equal to  $A_2 e^{i\omega t}$ . So, I can assume this type of solution in which both the frequency I am assumed to be same. So, in this case you can find this response amplitude  $A_1$  and  $A_2$ .

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Free Vibration of a two dof system

$$M\ddot{x} + Kx = 0$$


$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{bmatrix} + \begin{bmatrix} K_1+K_2 & -K_2 \\ -K_2 & K_2+K_3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

So, now substituting this expression in the previous equation, so you can write the equation motion in this form.

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$$x = A \sin \omega t$$

$$\left. \begin{aligned} x_1 &= A_1 e^{i\omega t} \\ x_2 &= A_2 e^{-i\omega t} \end{aligned} \right\}$$

$$\left[ K_1 + K_2 - m_1 \omega^2 \right]$$


$$\begin{aligned} &A_1 \sin \omega t \\ &A_2 \sin \omega t \end{aligned}$$

So, you can write. So, instead of assuming  $e$  to the power of  $i \omega t$  also you can take this equal to  $A \sin \omega t$  and  $A_2 \sin \omega t$  also. So, either you take  $A_1 \sin \omega t$   $A_2 \sin \omega t$  equal to  $x_1$  and  $x_2$ , or you just take  $x_1$  equal to  $A_1 e$  to the power of  $i \omega t$ ,  $x_2$  equal to  $A_2 e$  to the power  $-i \omega t$ . Then you substitute this expression in this equation motion. So, if you substitute in equation motion, then you can obtain the

equation in this form. So, you can write it in this form. So, it will become  $k_1 + k_2$  minus  $m_1 \omega^2$ .

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Free Vibration of a two dof system

$$M \ddot{x} + Kx = 0$$


$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{bmatrix} + \begin{bmatrix} k_1+k_2 & -k_2 \\ -k_2 & k_2+k_3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

So, this  $m_1 \times 1$  double dot will become  $A_1 \omega^2 e^{i\omega t}$ .

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$$x = A \sin \omega t$$

$$\left. \begin{aligned} x_1 &= A_1 e^{i\omega t} \\ x_2 &= A_2 e^{-i\omega t} \end{aligned} \right\} \begin{aligned} &A_1 \sin \omega t \\ &A_2 \sin \omega t \end{aligned}$$

$$\begin{bmatrix} k_1+k_2-m_1\omega^2 & -k_2 \\ -k_2 & k_2+k_3-m_2\omega^2 \end{bmatrix} \begin{bmatrix} A_1 \\ A_2 \end{bmatrix} e^{i\omega t} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

So, I can write and this is minus  $k_2$ . So, minus  $k_2$  and then this is minus  $k_2$  and this becomes  $k_2 + k_3 - m_2 \omega^2$  and  $x_1$  or  $A_1 A_2$ , I have written it  $A_1 A_2$ . So, you can put it  $A_1 A_2 e^{i\omega t}$ ;  $e^{i\omega t}$  will be equal to 0. So, from this equation you can observe that as  $e^{i\omega t}$

omega t is not equal to zero, and you are interested for a non-trivial solution where A 1 and A 2 are not 0. So, the determinant of this matrix should be equal to 0. So, you have to find the determinant of this matrix and equate to 0.

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$$K_1 = K_2 = K_3 = K$$

$$m_1 = m, \quad m_2 = 2m$$

So, let us take a simple case in which this K 1 equal to k 2 equal to k 3 equal to k, and this m 1 equal m and m 2 equal to 2 m. So, in this case let us assume that this is the case we are considering.

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$$x = A \sin \omega t$$

$$x_1 = A_1 e^{i\omega t}$$

$$x_2 = A_2 e^{-i\omega t}$$

$$\begin{bmatrix} K_1 + K_2 - m_1 \omega^2 & -K_2 \\ -K_2 & K_2 + K_3 - m_2 \omega^2 \end{bmatrix} \begin{bmatrix} A_1 \\ A_2 \end{bmatrix} e^{i\omega t} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$


So, in that case this equation will reduce to. So, this becomes 2 k minus m omega square.

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$$\begin{aligned} k_1 = k_2 = k_3 = k \\ m_1 = m, \quad m_2 = 2m \end{aligned} \quad \Bigg\|$$
$$\begin{bmatrix} 2k - m\omega^2 & -k_2 \\ -k_2 & \end{bmatrix}$$

So, this becomes 2 k minus m omega square, and this becomes minus k 2; this becomes minus k 2 and this is equal to...

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$$x = \underline{A} \sin \omega t$$
$$\left. \begin{aligned} x_1 &= \underline{A}_1 e^{i\omega t} \\ x_2 &= \underline{A}_2 e^{i\omega t} \end{aligned} \right\} \begin{aligned} &A_1 \sin \omega t \\ &A_2 \sin \omega t \end{aligned}$$
$$\begin{bmatrix} k_1 + k_2 - m_1 \omega^2 & -k_2 \\ -k_2 & k_2 + k_3 - m_2 \omega^2 \end{bmatrix} \begin{bmatrix} A_1 \\ A_2 \end{bmatrix} e^{i\omega t} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

So, this is also 2 k minus 2 m.

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$$\begin{aligned}
 &K_1 = K_2 = K_3 = K \\
 &m_1 = m, \quad m_2 = 2m \quad || \\
 &\begin{bmatrix} 2K - m\omega^2 & -K \\ -K & 2K - 2m\omega^2 \end{bmatrix} \begin{bmatrix} A_1 \\ A_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \\
 &\underline{\underline{(2K - m\omega^2)(2K - 2m\omega^2) - K^2 = 0}} \\
 &\omega^2 = \lambda \\
 &(2K - m\lambda)(2K - 2m\lambda) - K^2 = 0 \\
 &4K^2 - 6mK\lambda - 2mK\lambda + 2m^2\lambda^2 - K^2 = 0
 \end{aligned}$$

So, this becomes  $2k$  minus  $2m$  omega square into  $A_1 A_2$ . So,  $A_1 A_2$  will be equal to  $00$ . So, in this case I have to find the determinant of these to find the non-trivial solution. So, that is where  $A_1$  and  $A_2$  either  $A_1$  or  $A_2$  not equal to  $0$ . So, in case of trivial solutions  $A_1$  and  $A_2$  are zero, but as we are interested to find the non-trivial solution; so this determinant part will be equal to zero. So, the determinant of this equation will become. So, this is equal to  $2k$  minus  $m$  omega square into  $2k$  minus  $2m$  square.

So, minus  $k^2$  square, so this becomes minus  $k^2$ . So, already I have taken  $k^2$  equal to  $k$ . So, this becomes  $k$  square. So, this becomes  $k$  square will be equal to  $0$ . So, this is the determinant. So, let me substitute this omega square equal to lambda. So, this equation will become. So, this is  $2k$  minus  $m$  lambda into  $2k$  minus  $2m$  lambda minus  $k$  square equal to  $0$ , or I can write this equation. So, this equation I can write equal to  $4k$  square. So, this is  $4k$  square minus  $6mk$  lambda. So, this becomes minus  $2mk$  lambda, then minus minus plus. So, this becomes  $2m^2$  lambda square minus  $k$  square equal to  $0$ . So, from this I can write the equation becomes.

(Refer Slide Time: 31:15)

$$\lambda^2 - \left(3\frac{k}{m}\right)\lambda + \frac{3}{2}\left(\frac{k}{m}\right) = 0$$
$$\lambda_1 = \frac{-3\left(\frac{k}{m}\right) \pm \sqrt{9\frac{k^2}{m^2} - 4 \cdot \frac{3}{2} \frac{k}{m} \cdot 1}}{2}$$
$$=$$

So, from this I can write lambda square. The equation reduces to lambda square minus 3 k by m lambda plus 3 by 2 k by m equal to 0. So, from this I can find this. This is the quadratic equation, and solution of this quadratic equation will give me the frequency equation. So, I can have two value of lambda. So, the lambda 1, I will have this lambda 1 equal to. So, this becomes. So, this is lambda 1 2 you can find minus b, so minus minus plus. So, 3 k by m minus b plus minus root over b square minus 4 a c by 2 a b square. This becomes 9 k square by m square and minus b square minus 4 a c. So, this is equal to 3 by 2 k by m into 1 by 2. So, this becomes. So, this is 3 k by m. So, this is 2, so this becomes. So, this is 2, so 2 into 3 is 6. So, 9 k square by m square and this becomes.



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$$\begin{aligned} &K_1 = K_2 = K_3 = K \\ &m_1 = m, \quad m_2 = 2m \end{aligned} \parallel$$
$$\begin{bmatrix} 2K - m\omega^2 & -K \\ -K & 2K - 2m\omega^2 \end{bmatrix} \begin{bmatrix} A_1 \\ A_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$
$$(2K - m\omega^2)(2K - 2m\omega^2) - K^2 = 0$$
$$(2K - m\lambda)(2K - 2m\lambda) - K^2 = 0 \quad \omega^2 = \lambda$$
$$4K^2 - 6mK\lambda - 2mK\lambda + 2m^2\lambda^2 - K^2 = 0$$

So, from this expression you can see that this is equal to.

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$$\lambda^2 - \left(3\frac{K}{m}\right)\lambda + \frac{3}{2}\left(\frac{K}{m}\right)^2 = 0$$
$$\lambda_{1,2} = \frac{+3\left(\frac{K}{m}\right) \pm \sqrt{9\frac{K^2}{m^2} - 4\frac{3}{2}\left(\frac{K}{m}\right)^2}}{2}$$
$$\lambda_1 = 0.634 \frac{K}{m}$$
$$\lambda_2 = 2.366 \frac{K}{m}$$
$$\omega_1 = \sqrt{0.634 \frac{K}{m}}, \quad \omega_2 = \sqrt{2.366 \frac{K}{m}}$$

The square term here 3 by 2 k by m square. So, this is k by m whole square into 1. So, this becomes 0.634 k by m, and this is lambda 1, and lambda 2 equal to 2.366 k by m. So, I got two value of the natural frequency; one is lambda 1 that is 0.634 k by m, and lambda 2 equal to 2.366 k by m. So, from this I can find omega 1 that is the first natural frequency that omega 1 will be root over. So, root over 0.634 k by m and omega 2 will become two root over 2.366 k by m.

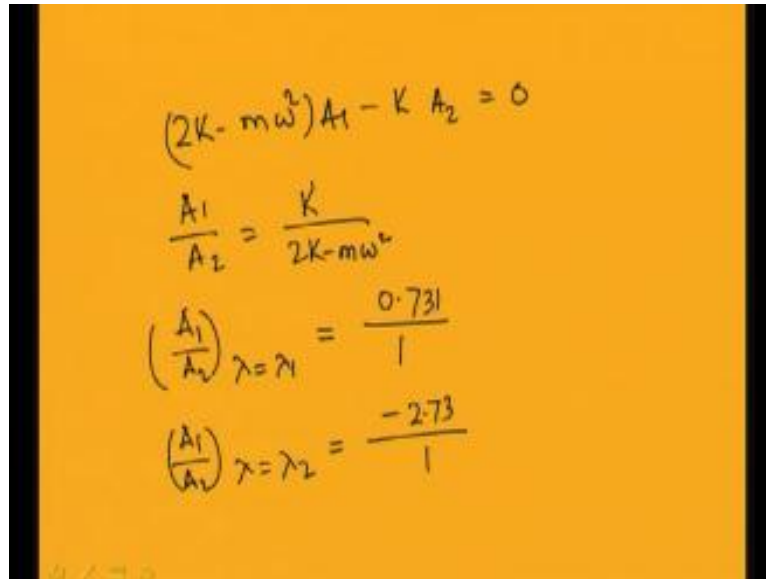
So, in this case I am getting two natural frequencies that is omega 1 and omega 2. In case of single degree of freedom system, you can recall that you got only single natural frequency that is equal to root over k by m. So, in this case you are getting two natural frequency.

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$$\begin{aligned}
 & k_1 = k_2 = k_3 = k \\
 & m_1 = m, \quad m_2 = 2m \quad || \\
 & \begin{bmatrix} 2k - m\omega^2 & -k \\ -k & 2k - 2m\omega^2 \end{bmatrix} \begin{bmatrix} A_1 \\ A_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \\
 & \underline{\underline{(2k - m\omega^2)(2k - 2m\omega^2) - k^2 = 0}} \\
 & \omega^2 = \lambda \\
 & (2k - m\lambda)(2k - 2m\lambda) - k^2 = 0 \\
 & 4k^2 - 6mk\lambda - 2mk\lambda + 2m^2\lambda^2 - k^2 = 0
 \end{aligned}$$

And substituting these two natural frequency in this equation  $2k - m\omega^2 - k$   $2k - 2m\omega^2$   $A_1$   $A_2$  equal to  $0$   $0$ . So, you can find the expression for  $A_1$  and  $A_2$  as you can see this right hand side equal to  $0$ . So, you cannot get a unique solution for  $A_1$  and  $A_2$ . So, taking any of this equation you can find the relation between  $A_1$  and  $A_2$ . So, taking this first expression you can write this  $2k - m\omega^2 A_1 - k A_2$  equal to  $0$  or you can write this  $2k$ .

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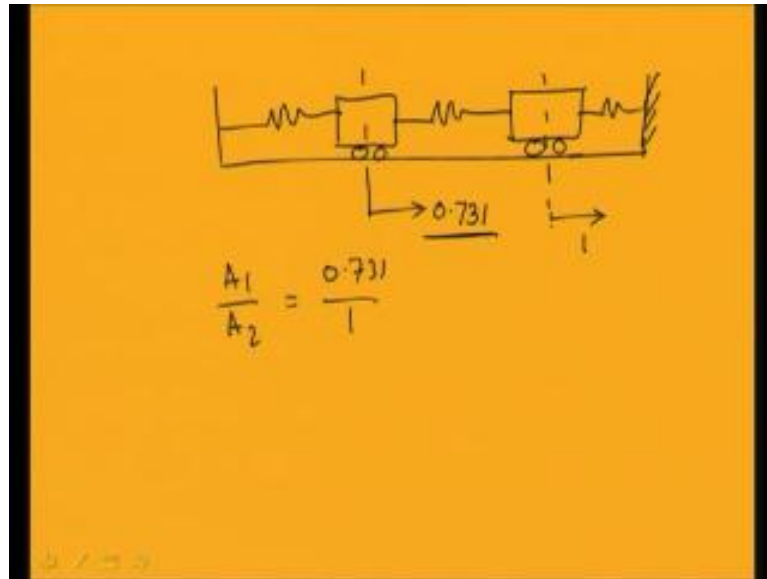
The image shows a handwritten derivation on a yellow background. The equations are as follows:

$$(2k - m\omega^2)A_1 - kA_2 = 0$$
$$\frac{A_1}{A_2} = \frac{k}{2k - m\omega^2}$$
$$\left(\frac{A_1}{A_2}\right)_{\lambda = \lambda_1} = \frac{0.731}{1}$$
$$\left(\frac{A_1}{A_2}\right)_{\lambda = \lambda_2} = \frac{-2.73}{1}$$

Two  $k$  minus  $m$  omega square  $A_1$  minus  $k A_2$  equal to 0 or  $A_1$  by  $A_2$  you can write in this form. So,  $A_1$  by  $A_2$  equal to. So, this  $k$  already I have taken it equal to  $k$ . So, this becomes  $k$  by  $2k$  minus  $m$  omega square and already I found the two values of omega or two values of omega square or two values of lambda. So, by substituting those two values, you can get. So, by substituting the first value, so  $A_1$  by  $A_2$  you can get. So, when lambda equal to lambda 1. So, you are getting this is equal to 0.731 is to 1.

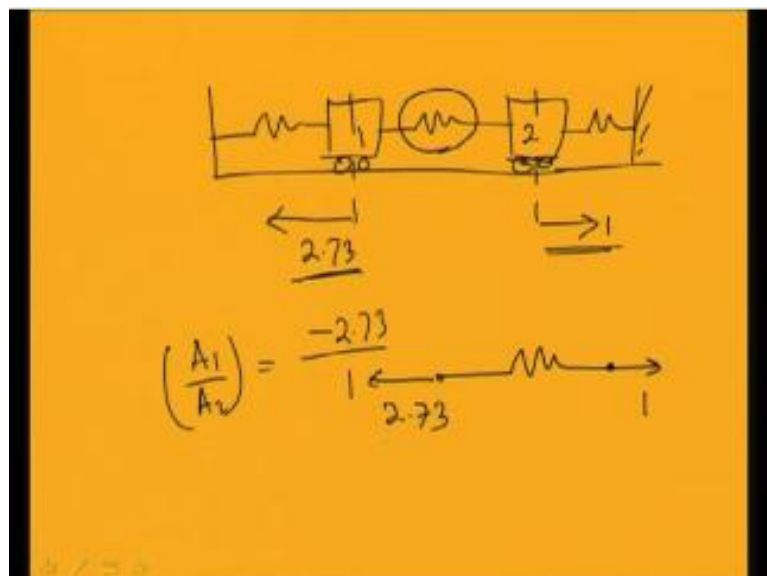
Similarly, when you are substituting lambda equal to lambda 2, you are getting two value of lambda. So, lambda 2; so this becomes minus 2.731 1. So, in this case you can observe that in the first case when the natural frequency or lambda 1 equal to 0.634 root over 0.634  $k$  by  $m$ .

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So, in that case you can observe or your system will be like this. So, this is the system. So, this is the original system. Now when it will have a frequency lambda equal to lambda 1, you have observed that when this is moving. So, this A 1 by A 2 you have seen it equal to 0.731 by 1; that means when this x 2 moves a distance one unit in distance. So, this will move a distance of 0.731. So, when x 2 is moving a distance or the second mass is moving a distance of one the first mass is moving a distance of 0.731.

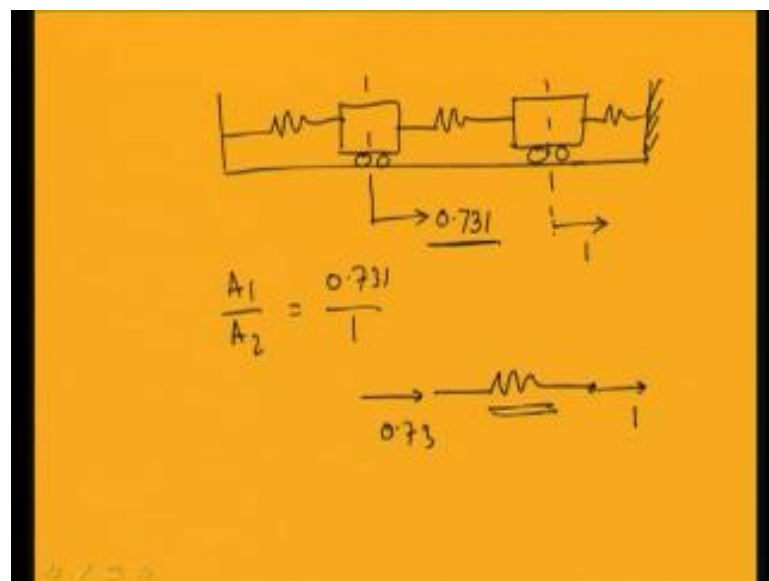
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Similarly, in the second case you can show that or you can draw it like this. So, when the first mass is moving. So, you have these are the two masses. So, in the second case you have seen that this is  $x_1$ , this is  $x_2$ . So, you have seen  $A_1$  by  $A_2$  equal to minus 2.73 by 1. So, in this case, when this is moving a distance; so this negative sign indicates that it is moving towards left. So, this is 2.73; that means when it is moving a distance of one in this direction. So, when  $A_1$  or second mass is moving a distance. So, this is the second mass; this is the first mass.

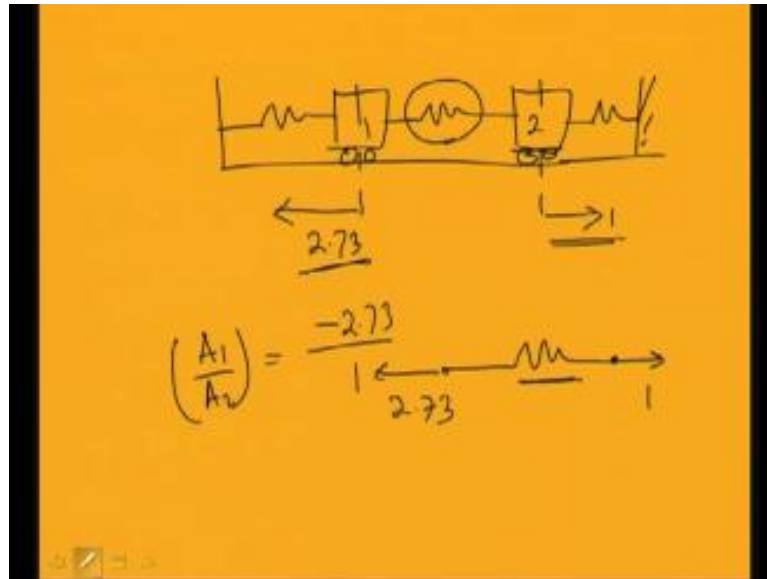
So, when the second mass is moving a distance of one towards right, the first mass is moving a distance of 2.73 towards left. So, in this case this spring the second spring will be under tension. So, this spring in this way it is moving a distance one and in this way it is moving a distance 2.73.

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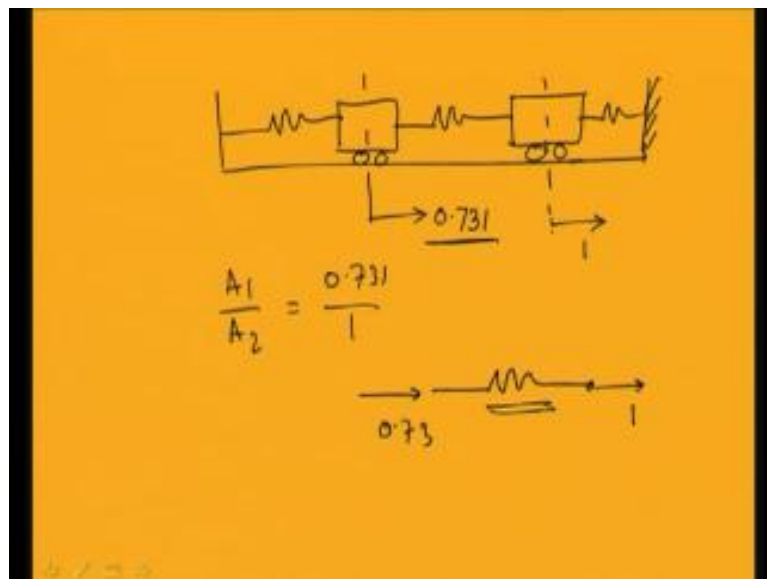
But in the previous case you can observe that this spring both of them are moving in the same direction when it is moving one. So, this is moving 0.73. So, the spring is compressed. So, in this case the spring is compressed.

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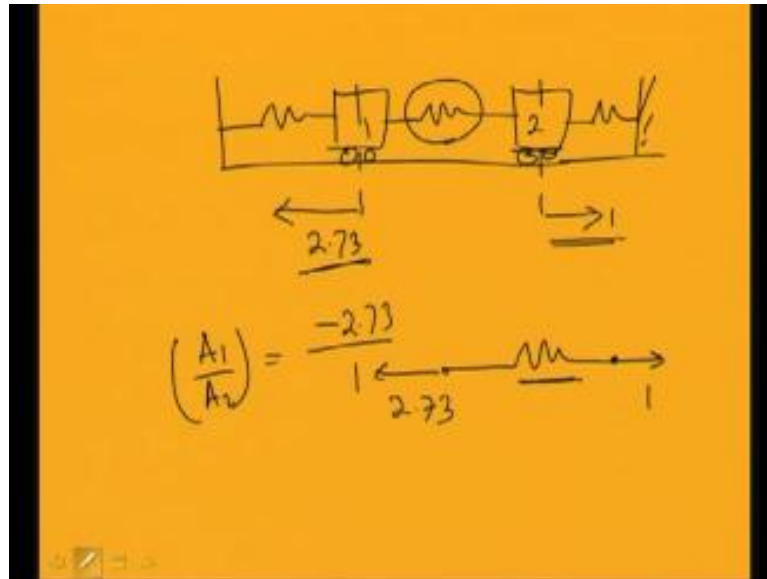
And in this case the spring is in tension.

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So, in the first case you can tell that both the masses are in phase or they are moving in the same direction, but in the second case you can tell that both the masses are out of phase.

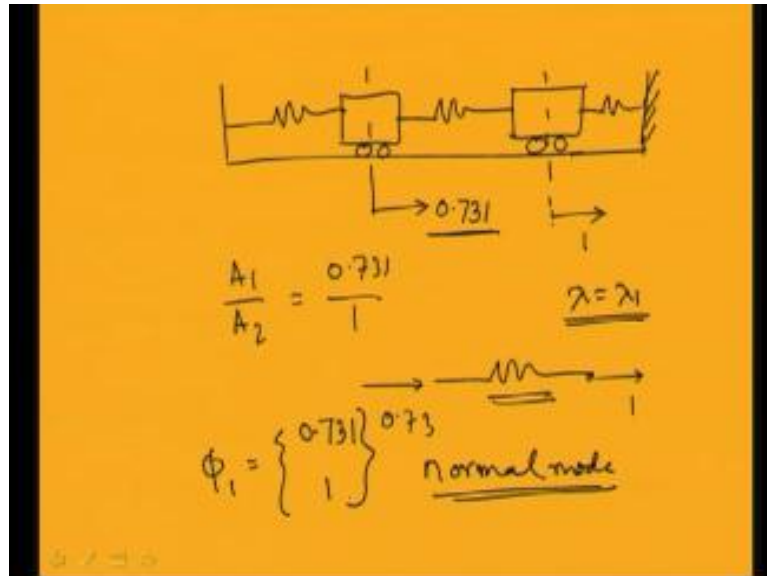
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That is when one is moving towards right, the other one is moving towards left. So, this is the normal mode. So, when you are normalizing this thing. So, in this case in both the cases we have made this  $A_2$  equal to 1, and we have found the value of  $A_1$ . So, in both the cases we are normalizing the displacement of the second mass, and in both the cases we have assumed that both the masses are moving with the same frequency and crossing the equilibrium position at the same time.

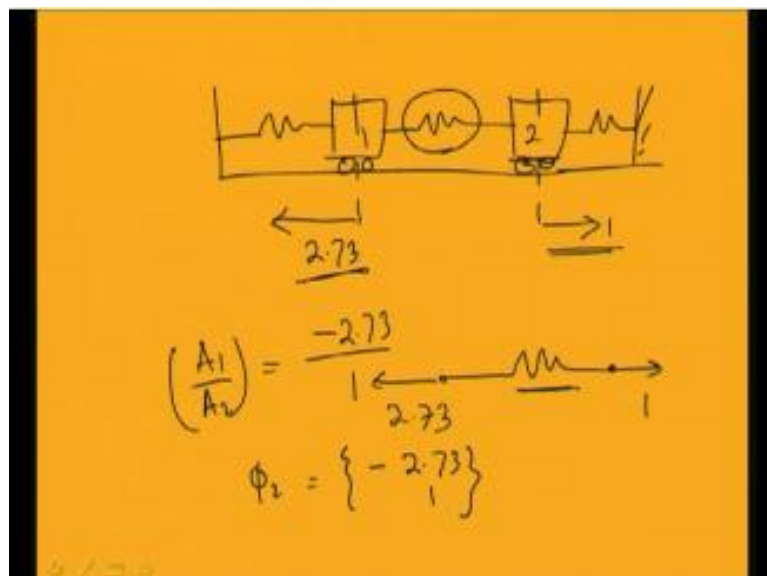
That means we are taking that they are moving with a particular frequency. So, these types of motion in which we are considering the motion is taking place at the same frequency are known as normal modes. So, we have two normal modes in this case. So, the first normal mode equal to...

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So, this is the first normal mode, I can write  $\phi_n$ . So, this  $\phi_1$  or I can write this first normal mode  $\phi_1$  equal to. So, this indicates the motion of these masses when it is moving with the first frequency  $\lambda$  equal to  $\lambda_1$ . So, when it is moving with  $\lambda$  equal to  $\lambda_1$ . So, these two masses will have displacement 0.731 and 1. So, this  $\phi_1$  this is the normal mode. So, the normal mode I can write for the first mass. So, the first normal mode equal to 0.731 1.

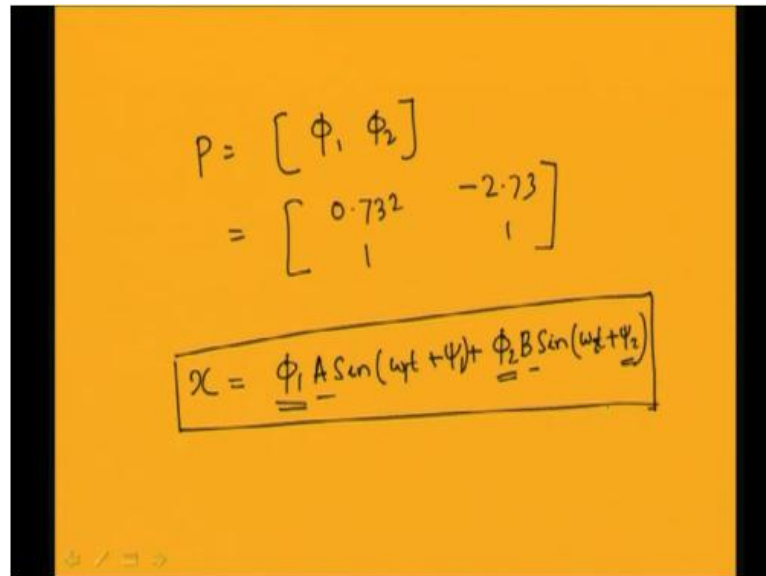
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Similarly the second normal mode  $\phi_2$  is equal to minus 2.731 1.



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$$P = \begin{bmatrix} \phi_1 & \phi_2 \\ 0.732 & -2.73 \\ 1 & 1 \end{bmatrix}$$
$$x = \phi_1 A \sin(\omega_1 t + \psi_1) + \phi_2 B \sin(\omega_2 t + \psi_2)$$

So, I can write the modal matrix  $P$  equal to  $\phi_1 \phi_2$ , and this thing can be written in this form. So, it will be equal to  $0.732 \ 1$  and  $-2.73 \ 1$ . So, in this way you can determine the normal mode of a system. So, to determine the free vibration response of the system, you can use these normal modes to find free vibration and at a particular time the free vibration can be considered to be the summation of the normal modes. So, the free vibration you can assume to be the summation of the normal modes.

So, at a particular time the free vibration  $x$  you can write will be equal to. So,  $x$  will be equal to  $\phi_1 A \sin \omega_1 t + \psi_1 + \phi_2 B \sin \omega_2 t + \psi_2$ . So, in this case already you know this  $\phi_1$  and  $\phi_2$ . So, these are the normal modes and  $A \ B$  and this  $\psi_1 \ \psi_2$  can be determined from the initial conditions. So, given the initial conditions of this mass one and two, so initial conditions represent the displacement and velocity. So, for each mass you have the displacement and velocity.

So, you have four known quantities, so two displacement quantities and two velocity quantities; so four known quantities. So, you have here four unknowns that is  $A \ B$  and  $\psi_1 \ \psi_2$ . So, you have four unknowns and four equations for displacement and velocity. So, by solving those equations you can find this  $A \ \psi_1$  and  $B \ \psi_2$ , and you can find the response of the system. So, the free vibration response of the system you can find by using this normal modes. So, let us take the same problem.

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$$\left. \begin{aligned} x_1(0) &= 5 \\ x_2(0) &= 1 \\ \dot{x}_1(0) &= 0 \\ \dot{x}_2(0) &= 0 \end{aligned} \right\} \text{Initial Conditions}$$

$$\begin{cases} x_1 \\ x_2 \end{cases} = \begin{cases} 0.731 \\ 1 \end{cases} \sin \omega_1 t$$

$$\begin{cases} x_1 \\ x_2 \end{cases} = \begin{cases} -2.73 \\ 1 \end{cases} \sin \omega_2 t$$

So, in this problem let me assume that this  $x_1$ , so this  $x_1$  equal to. So, let  $x_1(0)$  equal to 5 and  $x_2(0)$  equal to 1 and this velocity terms are. So,  $\dot{x}_1(0)$  equal to 0 and  $\dot{x}_2(0)$  equal to 0. So, these are the initial conditions for the system. So, taking these initial conditions I have to determine the response of the system. So, already I know the response of the system  $x_1$  and  $x_2$  can be written in this form. So,  $x_1$   $x_2$  you can write. So, already I know that  $x_1$   $x_2$ . So, for the first mode I can write. So, the first mode this is equal to  $x_1$   $x_2$  equal to 0.731 1 sin  $\omega_1 t$  and for the second mode  $x_1$  and  $x_2$  already you know this is equal to minus 2.731 1 sin  $\omega_2 t$ .

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$$\begin{cases} x_1 \\ x_2 \end{cases} = A \begin{cases} 0.731 \\ 1 \end{cases} \sin(\omega_1 t + \psi_1) + B \begin{cases} -2.73 \\ 1 \end{cases} \sin(\omega_2 t + \psi_2)$$

$$\begin{cases} 5 \\ 1 \end{cases} = A \begin{cases} 0.731 \\ 1 \end{cases} \sin \psi_1 + B \begin{cases} -2.73 \\ 1 \end{cases} \sin \psi_2$$

So, the general solution  $x_1$   $x_2$  at any particular time  $t$  can be written as  $A \phi_1$ . So,  $A \phi_1$  equal to  $0.731 \sin \omega_1 t$  plus  $\psi_1$  plus  $B$  into minus  $2.731 \sin \omega_2 t$  plus  $\psi_2$ . So, here we have to find these  $A$   $B$  and  $\psi_1$   $\psi_2$ . So, from this initial conditions so this  $x_1(0)$ . So, when  $t$  equal to zero, I can substitute this in that expression. So, I can find  $x_1(0)$  and  $x_2(0)$ . So, this  $5$   $1$  I can write. So, taking  $t$  equal to  $0$ , I can write this equal to. So,  $5$   $1$  will be equal to  $A$  into  $0.731$   $1$   $\sin$ . So,  $t$  equal to  $0$ . So,  $\sin 0$  plus  $\psi_1$ . So, this is equal to  $\sin \psi_1$  plus  $B$  into minus  $2.731$   $1$   $\sin \psi_2$ . Now I can differentiate this equation to get the velocity terms, and from this I can substitute this velocity at initial velocity  $0$   $0$ .

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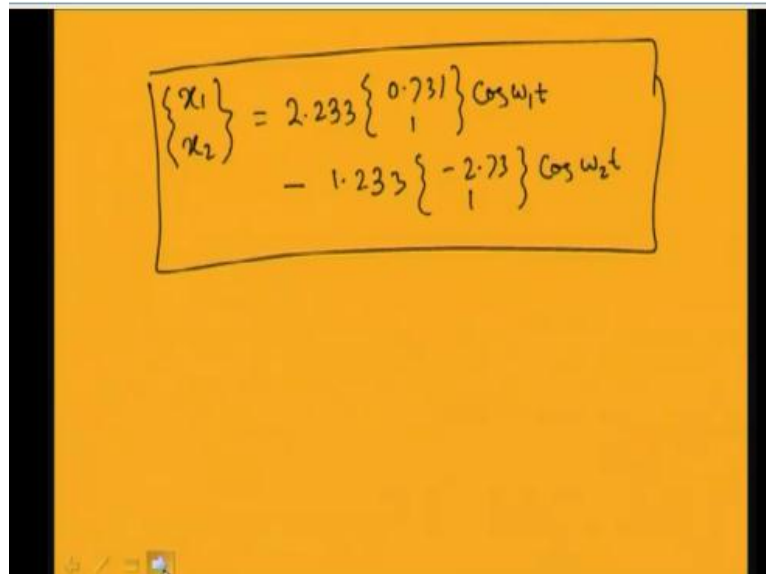
$$\begin{aligned} \begin{Bmatrix} \dot{x}_1(0) \\ \dot{x}_2(0) \end{Bmatrix} &= \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} = \omega_1 A \begin{Bmatrix} 0.731 \\ 1 \end{Bmatrix} \cos \psi_1 \\ &\quad + \omega_2 B \begin{Bmatrix} -2.731 \\ 1 \end{Bmatrix} \cos \psi_2 \\ \cos \psi_1 &= \cos \psi_2 = 0 \\ \psi_1 &= \psi_2 = 90^\circ \\ \begin{Bmatrix} 5 \\ 1 \end{Bmatrix} &= A \begin{Bmatrix} 0.731 \\ 1 \end{Bmatrix} + B \begin{Bmatrix} -2.731 \\ 1 \end{Bmatrix} \\ A &= 2.33 \\ B &= -1.233 \end{aligned}$$

So, I will get this expression. So,  $x_1 \dot{0}$   $x_2 \dot{0}$ . So, these are already known to be  $0$   $0$ . So, this becomes this will be equal to  $\omega_1 A$  into  $0.731$   $1$ , differentiation  $\sin$  is  $\cos$ . So, this becomes  $\cos \psi_1$  plus  $\omega_2 B$  into  $2.731$   $1$   $\cos \psi_2$ . So, from these expression you can see that  $\omega_1 A$  into this into  $\cos \psi_1$  equal to  $0$ . So,  $\omega_1 A$  into  $0.731 \cos \psi_1$  plus  $\omega_2 B$  into minus  $2.731$   $1$   $\cos \psi_2$  equal to  $0$ , so from these expression you can tell or you can find that this  $\cos \psi_1$  equal to  $\cos \psi_2$  equal to  $0$ . So, this expression is valid.

So, when  $\cos \psi_1$  equal to  $\cos \psi_2$  equal to  $0$  or  $\psi_1$  equal to  $\psi_2$  equal to  $90$  degree; so you can substitute this expression for  $\psi_1$   $\psi_2$  in the previous equations and you can get this  $A$   $1$ . So, this equation you will get. So, this is equal to  $5$   $1$  will be equal to  $A$  into

0.731 1 plus B into minus 2.731 1. So, now you have two expressions. So, 0.731 1 A plus minus 2.73 1 B equal to 5 and A plus B equal to 1. So, you have two equations. So, while solving these two equations you can find A equal to 2.233 and B equal to minus 1.233.

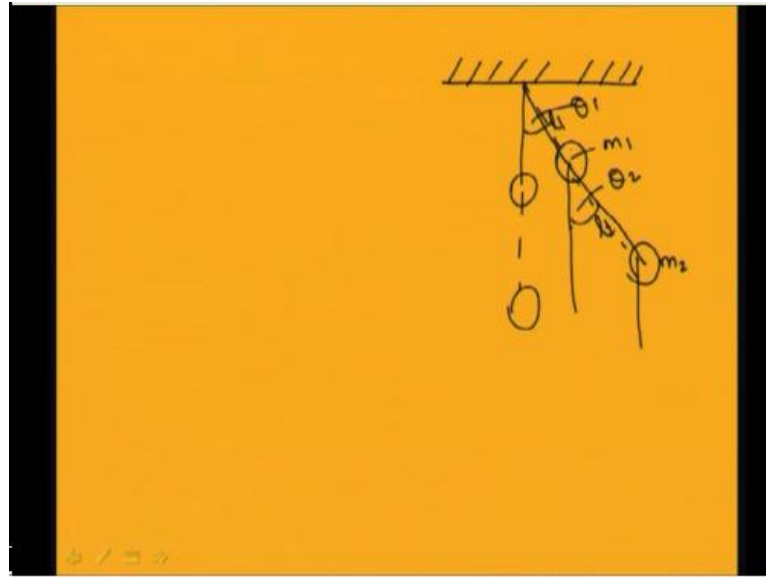
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$$\begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = 2.233 \begin{Bmatrix} 0.731 \\ 1 \end{Bmatrix} \cos \omega_1 t - 1.233 \begin{Bmatrix} -2.73 \\ 1 \end{Bmatrix} \cos \omega_2 t$$

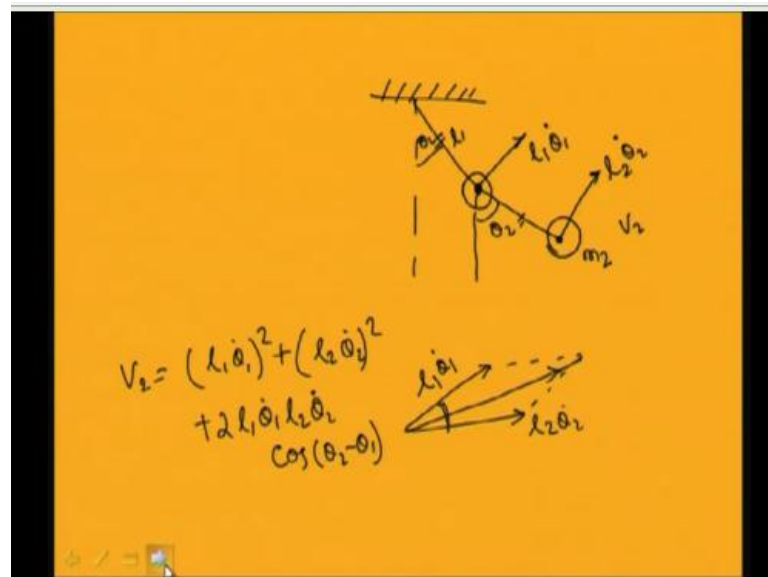
So, your equation is reduced to this form. So, the resulting free vibration response of the system can be written as  $x_1$   $x_2$  equal to 2.233 0.731 1 cos. So, sin omega t plus 90 degree is cos omega 1 t. So, this becomes cos omega 1 t minus 1.233 into 2.73 1 cos omega 2 t. So, this is the required expression for this free vibration of the system. So, you know how to determine the normal mode of a system, and using that normal mode you can find the free vibration response of the system. So, let us take some other systems to find the normal mode of that system.

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Let us take a double pendulum and for that double pendulum let us determine the normal mode of that system. So, for a double pendulum, this is the initial position of a double pendulum, now due to vibration. So, let it has come to this position; this mass has come to this position, and this mass the second mass has come to this position. So, you can express the motion of these two masses with this theta 1 and theta 2. So, this is theta 1 and theta 2. So, this is mass  $m_2$  and this is mass  $m_1$ . So, let us first determine the equation motion using Lagrange principle and later we will find the normal mode of the system. So, in this case to find the kinetic energy of the system we should find the velocity of this mass  $m_1$  and velocity of this mass  $m_2$ . So, let this length is  $l_1$ , this length is  $l_2$ .

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So, I can draw the velocity. So, for this case the velocity is perpendicular to this line. So, it will be perpendicular to this line, and this length is  $l_1$ , and this is  $\theta_1$ . So, this is equal to  $l_1 \dot{\theta}_1$ . And for the second mass, so this is for the second mass. So, the second mass makes an angle  $\theta_2$  with the vertical. So, it will have a velocity relative to this position which is perpendicular to this line. So, it will be perpendicular to this line and will have a magnitude  $l_2 \dot{\theta}_2$ .

So, the velocity of this position or velocity of mass  $m_2$  will be the velocity of mass  $m_1$  plus the relative velocity of mass  $m_2$  with respect to mass  $m_1$ . So, it will be the summation of. So, this is  $l_1 \dot{\theta}_1$ , and this is  $l_2 \dot{\theta}_2$ . So, the  $V_2$  or the velocity of this mass  $m_2$  will be equal to the vector sum of this  $l_1 \dot{\theta}_1$  and  $l_2 \dot{\theta}_2$ . So, you may use this parallelogram laws to find the velocity and already you know this velocity  $V_2$  can be written as  $l_1 \dot{\theta}_1$  square plus  $l_2 \dot{\theta}_2$  square plus  $2 l_1 \dot{\theta}_1 l_2 \dot{\theta}_2 \cos$ .

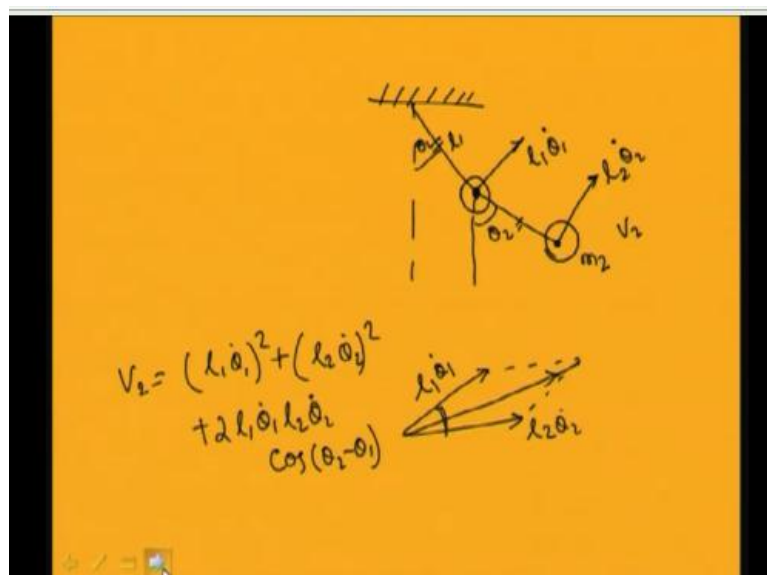
So, this angle between these two becomes  $\theta_2$  minus  $\theta_1$ . So, this becomes  $\theta_2$  minus  $\theta_1$ . So, this is the expression for velocity of this mass two. So, the kinetic energy of the system will be equal to half  $m_1$  in to  $V_1$  square that is equal to  $l_1 \dot{\theta}_1$  dot square and plus half  $m_2 V_2$  square. So, the total kinetic energy system I can write.

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$$T = \frac{1}{2} m_1 l_1^2 \dot{\theta}_1^2 + \frac{1}{2} m_2 (l_1^2 \dot{\theta}_1^2 + l_2^2 \dot{\theta}_2^2 + 2 l_1 l_2 \dot{\theta}_1 \dot{\theta}_2 \cos(\theta_2 - \theta_1))$$

So, T will be equal to half  $m_1 l_1^2 \dot{\theta}_1^2$  plus half  $m_2$ ; already I have written the expression for  $V_2$ . So,  $V_2 \cdot V_2$  that will be the velocity square of this mass  $m_2$ , and this will be written as  $l_1^2 \dot{\theta}_1^2$  plus  $l_2^2 \dot{\theta}_2^2$  plus  $2 l_1 l_2 \dot{\theta}_1 \dot{\theta}_2 \cos(\theta_2 - \theta_1)$ . So, this is  $\dot{\theta}_1$ , this is  $\dot{\theta}_2$ .

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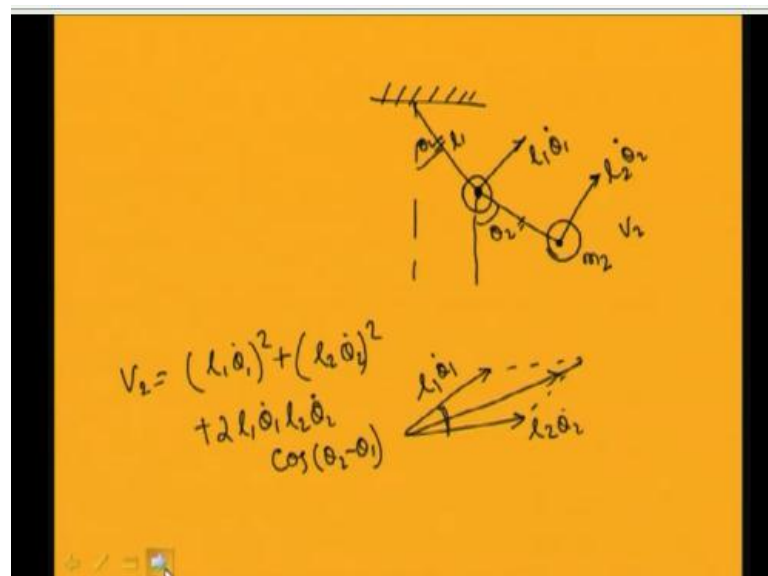
So, this is the expression  $2 l_1 l_2 \dot{\theta}_1 \dot{\theta}_2 \cos(\theta_2 - \theta_1)$ .

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$$T = \frac{1}{2} m_1 l_1^2 \dot{\theta}_1^2 + \frac{1}{2} m_2 \left\{ l_1^2 \dot{\theta}_1^2 + l_2^2 \dot{\theta}_2^2 + 2 l_1 l_2 \dot{\theta}_1 \dot{\theta}_2 \cos(\theta_2 - \theta_1) \right\}$$
$$U =$$

So, this is the expression for the kinetic energy of the system. Similarly, you can have the expression for the potential energy of the term. So, the potential energy of the system can be written as.

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So, potential energy due to change in position of this first mass; so the change in position of the first mass, the first mass initially has a length of  $l$ . So, now it has come to this position. So, change in height. So, this is the change in height of this first mass. So, this



becomes  $l_1$  into  $l_1 \cos \theta_1$ . So, this length equal to  $l_1 \cos \theta_1$ . So, this becomes  $l_1 - l_1 \cos \theta_1$ .

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$$T = \frac{1}{2} m_1 l_1^2 \dot{\theta}_1^2 + \frac{1}{2} m_2 \left\{ l_1^2 \dot{\theta}_1^2 + l_2^2 \dot{\theta}_2^2 + 2 l_1 l_2 \dot{\theta}_1 \dot{\theta}_2 \cos(\theta_2 - \theta_1) \right\}$$

$$U = m_1 g l_1 (1 - \cos \theta_1) + m_2 g \left\{ l_1 + l_2 - l_1 \cos \theta_1 - l_2 \cos \theta_2 \right\}$$

$$L = T - U$$

So, change in height. So, this is  $m_1 g l_1 (1 - \cos \theta_1)$ . Similarly, the potential energy for mass two you can find. So, the change in height will be equal to  $l_1 + l_2 - l_1 \cos \theta_1 - l_2 \cos \theta_2$ . So, this will be equal to  $m_2 g$  into change in height of that. So, this becomes  $l_1 + l_2 - l_1 \cos \theta_1 - l_2 \cos \theta_2$ . So, in this way you can find this T and U. So, using this T and U, now you can write this L equal to T minus U. So, L equal to T minus U you can write.

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$$\begin{bmatrix} (m_1+m_2)l_1^2 & m_2 l_1 l_2 \\ m_2 l_1 l_2 & m_2 l_2^2 \end{bmatrix} \begin{bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{bmatrix} + \begin{bmatrix} (m_1+m_2)l_1 g & 0 \\ 0 & m_2 l_2 g \end{bmatrix} \begin{bmatrix} \theta_1 \\ \theta_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

And now applying the Lagrange principle you can write the equation motion which can be given by this expression. So, you find that equation motion. So, this becomes  $m_1 + m_2$  into  $l_1^2$  double dot theta 1 plus  $m_2 l_1 l_2$  double dot theta 2 plus  $(m_1 + m_2) l_1 g$  theta 1 equals 0. So, this is 0, and this is  $m_2 l_2 g$  into theta 2. So, this becomes 0 0.

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$$T = \frac{1}{2} m_1 l_1^2 \dot{\theta}_1^2 + \frac{1}{2} m_2 \left\{ l_1^2 \dot{\theta}_1^2 + l_2^2 \dot{\theta}_2^2 + 2 l_1 l_2 \dot{\theta}_1 \dot{\theta}_2 \cos(\theta_1 - \theta_2) \right\}$$

$$U = m_1 g l_1 (1 - \cos \theta_1) + m_2 g \left\{ l_1 + l_2 - l_1 \cos \theta_1 - l_2 \cos \theta_2 \right\}$$

$$L = T - U \quad \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = 0$$

$$\underline{q_1 = \theta_1} \quad \underline{q_2 = \theta_2}$$

So, by using the Lagrange principle that is  $d$  by  $d t$  of  $\Delta l$  by  $\Delta q_k$  dot. So, minus  $\Delta l$  by  $\Delta q_k$  equal to 0. So, you can find the equation. So, here  $q_1$  I have taken equal to  $\theta_1$ , and  $q_2$  equal to  $\theta_2$ . So, by taking this is  $\theta_1$ , this is  $\theta_2$ .

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$$\begin{bmatrix} (m_1+m_2)l_1^2 & m_2 l_1 l_2 \\ m_2 l_1 l_2 & m_2 l_2^2 \end{bmatrix} \begin{bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{bmatrix} + \begin{bmatrix} (m_1+m_2)l_1 g & 0 \\ 0 & m_2 l_2 g \end{bmatrix} \begin{bmatrix} \theta_1 \\ \theta_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

So, you can write the equation motion in this way.

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$$m_1 = m_2 = m, \quad l_1 = l_2 = l$$

$$m l^2 \begin{bmatrix} 2 & 1 \\ 1 & 1 \end{bmatrix} \begin{bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{bmatrix} + m g l \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} \theta_1 \\ \theta_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} \theta_1 \\ \theta_2 \end{bmatrix} = \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} e^{i\omega t}$$

$$\underline{|A - \lambda I|} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

So, now let us take a simple case in which this mass  $M_1$  equal to  $m$ ,  $M_2$  equal to  $m$  and this  $l_1$  equal to  $l_2$  equal to  $l$ . So, this equation will reduce to this form. So, this becomes  $m l^2 \begin{bmatrix} 2 & 1 \\ 1 & 1 \end{bmatrix} \ddot{\theta}_1 \ddot{\theta}_2 + m g l \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix} \theta_1 \theta_2 = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$

equal to 0. So, proceeding in the previous way I can assume that this  $\theta_1$   $\theta_2$  equal to  $A_1 A_2 e^{i\omega t}$ . I can substitute it in this equation, and I can write the equation in this form.

So, I can write this equation or I can assume this equal to  $X_1 X_2$ . I can write let me write  $X_1 X_2$  instead of  $A_1 A_2$ . So, I can write this equal to  $X_1 X_2 e^{i\omega t}$ . So, you can substitute this equation in this form and get an equation  $A - \lambda$  into  $X_1 X_2$  will be equal to 0. So, now finding the determinant of this  $A - \lambda$ . So,  $A - \lambda$  into  $X_1 X_2$  equal to 0. So, this reduces to an Eigen value problem. So, either you just find the Eigen value of this matrix  $A$  or you just find the determinant of  $A$  to find  $\lambda$ . So, either you can find the Eigen value or the determinant.

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$$\lambda = (2 \pm \sqrt{2}) g/L$$

$$\omega_1 = 0.7653 \sqrt{g/L}$$

$$\omega_2 = 1.8478 \sqrt{g/L}$$

$$\begin{pmatrix} \theta_1 \\ \theta_2 \end{pmatrix} = \frac{1}{\sqrt{2}}$$

$$\begin{pmatrix} \theta_1 \\ \theta_2 \end{pmatrix} = -\frac{1}{\sqrt{2}}$$

So, if you find. So, this  $\lambda$  you will get will be equal to  $2 \pm \sqrt{2} g/L$ . So, your  $\omega_1$  will be equal to  $0.7653 \sqrt{g/L}$ . And  $\omega_2$  you can find equal to  $1.8478 \sqrt{g/L}$ . So,  $\sqrt{g/L}$  you will get. By substituting the first one, you can find  $\theta_1$   $\theta_2$ . So,  $\theta_1$   $\theta_2$  for the first one you can find will be equal to  $1/\sqrt{2}$  and the second one if you substitute then this  $\theta_1$   $\theta_2$ , you will find equal to  $-1/\sqrt{2}$ . So, in this first case both the masses are moving in same direction if you plot.

So, both the masses are moving in the same direction. So, when this  $\theta_1$  it is rotating at angle 1. So, this will rotate root 2, and in the second case when this will rotate at an angle  $\theta_1$ , this will rotate in opposite direction that is minus root 2. Next class we will study about the coupling or coordinate coupling of the system. Already I told about the coordinate coupling where static and dynamic of coupling system I told. I will solve some problems on this dynamic and static coupling. Next class also I will tell you about the force vibration of two degree of freedom system.