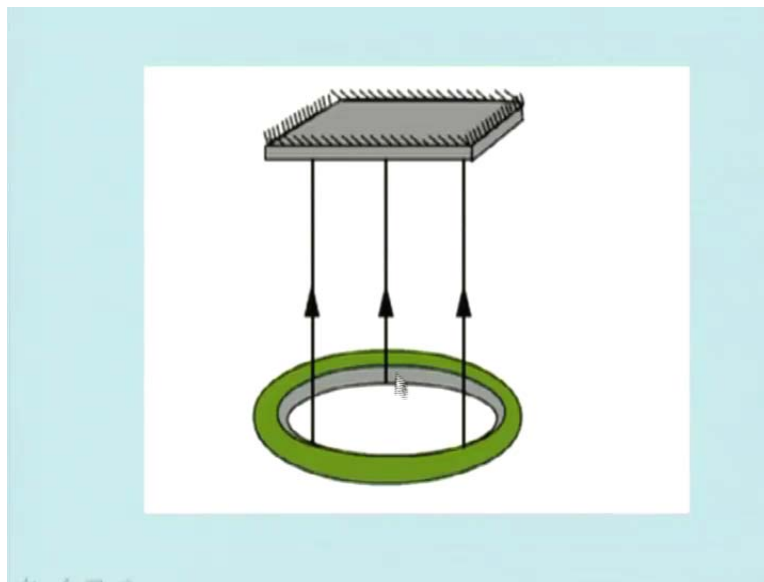


**Engineering Mechanics**  
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**Introduction to vibration**

**Module 15 Lecture No. 39**  
**Vibration of rigid bodies Part-2**

We will continue our discussion on vibration of rigid bodies. Last lecture, I discussed about bifilar suspension. Now, we discuss a case of trifilar suspension. Sometimes, it is convenient to find the moment of inertia of a disc by observing the frequency of oscillation. When the disc is suspended by three wires symmetrically about the axis of the disc, then that is called trifilar suspension. This arrangement is shown in this figure.

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We can find out the moment of inertia of the disc by seeing its oscillations.

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The force balance gives,

$$T = \frac{1}{3}W$$

where  $W$  is the weight of the disc.



This force balance gives,  $T$  is equal to  $\frac{1}{3}W$ . Disc is acted by a gravity force  $W$ , and then there is a tension  $T$  in the strings which is  $\frac{1}{3}W$ .  $W$  is the weight of the disc. We can provide a small twisting in this plane of the disc. That means, disc remains in its own plane, but this radial point, radius moves to another point that is  $\theta$ . Therefore, these strings get inclined. So, this is  $\phi$ .

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$$r\theta = l\phi$$

where  $r$  is the radius of the disc and  $l$  is the length of the string.

For very small rotation  $\theta$  (and hence  $\phi$ ), the tensions will still be  $T$ .

Horizontal component of the tension

$$= T\phi = \frac{1}{3}W \frac{r\theta}{l}$$

Therefore,  $r\theta$  is equal to  $l\phi$ . In this case,  $r\theta$  will be equal to  $l\phi$ .  $l$  is the length of this thing. So,  $r\theta$  is equal to  $l\phi$ , where  $r$  is the radius of the disc and  $l$  is the length of the string. For very small rotation  $\theta$ , and hence if  $\theta$  is small then  $\phi$  is also small. The tension will still be  $T$ . No change in the tension then the horizontal component of the tension is equal to  $T\phi$ . Actually, it is  $T\sin\phi$  but  $\sin\phi$  becomes equal to  $\phi$ . So, that is equal to  $1$  by  $3Wr\theta$  by  $l$ .

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There are three wires and horizontal components produce restoring torque.

Restoring torque  $= -3T\phi r = -\frac{3W}{l}r^2\theta$

By D'Alembert's principle,

Inertia torque  $= -I\ddot{\theta}$        $\ddot{\theta} = \alpha$

The body is in equilibrium under the action of these torques. Hence,

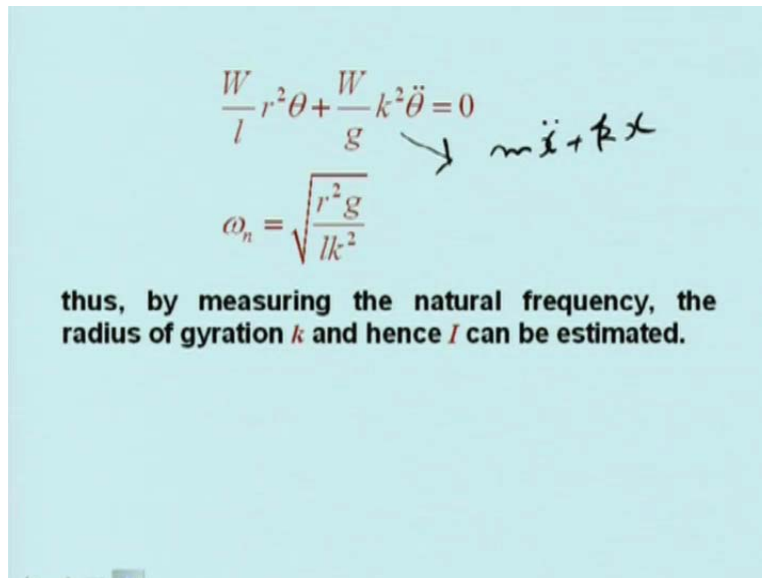
$$I\alpha + \frac{W}{g}k^2\ddot{\theta} + \frac{3W}{l}r^2\theta = 0$$

There are three wires and horizontal components produce restoring torque. Therefore, restoring torque is equal to minus  $3T\phi$  times  $r$ . So, this is  $3T\phi$  into  $r$ . So, that is minus  $W$  by  $l$   $r$  square  $\theta$ . Now, by D'Alembert's principle, inertia torque is equal to minus  $I\alpha$ . That must be that and restoring torque together should keep the body in equilibrium. The body is in equilibrium under the action of these torques. That means,  $I\alpha$  plus  $W$  by  $l$   $r$  square  $\theta$  is equal to this one. Here, this is actually, instead of  $I\alpha$  this is basically  $I\ddot{\theta}$ . This is the thing that it is  $I\ddot{\theta}$ .  $I\ddot{\theta}$ , if I say that disc rotates by  $\theta$  and this is  $W$  by  $l$ , this is  $r$  square.

This is  $W$  by  $I\ddot{\theta}$ . Inertia torque is  $I\ddot{\theta}$ . There, you know that  $\ddot{\theta}$  is basically same as angular acceleration  $\alpha$ . Therefore, this is the difference in the notation. At  $\alpha$  is equal to  $\ddot{\theta}$  and this is basically  $I\alpha$  can be written as  $W$  by  $g$   $k$

square alpha, because I is equal to W by g k square and plus you add W by l here. This is 3T here. Also, it must be 3W by l r square and this is theta, this is theta double dot.

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$$\frac{W}{l} r^2 \theta + \frac{W}{g} k^2 \ddot{\theta} = 0$$

$\rightarrow m \ddot{x} + kx$

$$\omega_n = \sqrt{\frac{r^2 g}{l k^2}}$$

**thus, by measuring the natural frequency, the radius of gyration  $k$  and hence  $I$  can be estimated.**

This is basically, you get equation like this: This is W by l r square theta plus W by g k square theta double dot is equal to 0. This is compared with mx double dot plus kx is equal to 0. So, it is basically, the same way that W W gets cancelled; it becomes r square g l k square.

This is r square by l and g by k square. Thus, by measuring the natural frequency, the radius of gyration k and hence, I can be estimated. We can find out, if you measure the natural frequency omega<sub>n</sub>, you know the gravitational constant, you can find out its r. l is known. So, one can find out this thing.

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**Forced Vibration:**  
We present an example of the forced vibration of a rigid rod.

Handwritten calculations:

$$\frac{ml^2}{12}$$

$$\frac{ml^2}{12} + \frac{ml^2}{4}$$

$$= \frac{ml^2}{3}$$

Now, we have talked about the free vibration problems. Let us go to the forced vibration problem. An example of a forced vibration of a rigid rod is like this; this rod is hinged at this point and an excitation force  $F_0 \sin \omega t$  is provided at point A. Dashpot is at a distance of  $b$ . This distance is  $l$ . This is the spring. Here, the spring stiffness is  $k$ . This is  $a$ , this is  $b$ . Now, this is  $d$ . This is a dashpot, linear dashpot.

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OA is a rigid rod hinged at O. There is a spring attached at a distance of  $a$  from the hinge and the dashpot at a distance  $b$  from the hinge point.

For small  $\theta$ , the equilibrium equation is

$$\frac{ml^2}{3} \ddot{\theta} + ka^2 \theta + cb^2 \dot{\theta} = (F_0 \sin \omega t) l$$

Handwritten calculations:

$$\ddot{x} + \frac{c}{m} \dot{x} + \frac{k}{m} x = \frac{F_0 \sin \omega t}{m}$$

$$\omega_n = \sqrt{\frac{3ka^2}{ml^2}} \quad \gamma = \frac{3cb^2}{2ml^2}$$

~~Handwritten crossed-out equation:~~

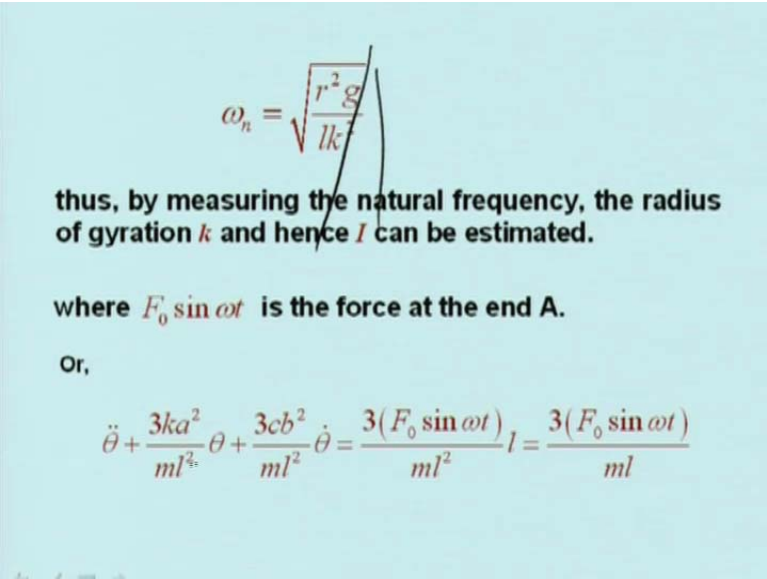
$$\frac{W}{l} r^2 \ddot{\theta} + \frac{W}{g} k^2 \ddot{\theta} = 0$$

OA is the rigid rod hinged at O. There is a spring attached at a distance of  $a$  from the hinge, and the dashpot at a distance  $b$  from the hinge point. So, dashpot is that. For a small  $\theta$ , the equilibrium equation is  $ml^2 \ddot{\theta} + ka^2 \theta$ .  $ml^2$  is the mass moment of inertia of the cylinder rod about point O.

Mass moment of inertia of a uniform rod about the center of mass is  $ml^2/12$  about its one end. It is actually  $ml^2/12$  by this thing, because apply the parallel axis theorem,  $ml^2/12$  plus  $m$  into  $l/2$  squared; distance is  $l/2$  squared by 4. This is  $3l^2$  by 4 therefore, this is  $ml^2/3$ . For a small  $\theta$ , the equilibrium equation is  $ml^2 \ddot{\theta} + ka^2 \theta$ ; that is the restoring force due to spring. When you displace it by  $\theta$ , this spring gets stressed by  $a\theta$ . So,  $k$  times  $a\theta$  is the force and it acts at a distance  $a$  from the hinge. So,  $k a^2$ . Similarly, the dashpot is moving. So,  $b\dot{\theta}$  is the displacement of the dashpot,  $b\dot{\theta}$  is the velocity and therefore, the force is equal to  $c$  times  $b\dot{\theta}$  but it is acting at a distance of  $b$ .

Therefore, it becomes  $cb^2 \dot{\theta}$  and this is equal to  $F_0 \sin \omega t$  multiplied by  $l$ ,  $l$  is the distance from here. Therefore, you get  $W$  by  $g$  by  $l$ . This will be  $ml^2 \ddot{\theta} + 3(F_0 \sin \omega t)l$  and this becomes this one. There is no need to write this equation,  $F_0 \sin \omega t$  into  $l$ . Actually, compare this equation.

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$$\omega_n = \sqrt{\frac{r^2 g}{lk}}$$

thus, by measuring the natural frequency, the radius of gyration  $k$  and hence  $I$  can be estimated.

where  $F_0 \sin \omega t$  is the force at the end A.

Or,

$$\ddot{\theta} + \frac{3ka^2}{ml^2} \theta + \frac{3cb^2}{ml^2} \dot{\theta} = \frac{3(F_0 \sin \omega t)}{ml^2} l = \frac{3(F_0 \sin \omega t)}{ml}$$

Now, compare this equation with equation  $x$  double dot plus  $k$  by  $m$   $x$  plus  $c$  by  $m$   $\dot{x}$  is equal to  $F_0 \sin \omega t$  by  $m$ , which was spring mass and dashpot system. So, by this you can find out,  $\omega_n$  is equal to natural and damped frequency,  $\omega_n$  is equal to under root  $3ka$  square divided by  $ml$  square and damping factor  $\zeta$  is equal to  $3cb$  square divided by  $2ml$  square  $\omega_n$  and  $\omega_d$  is equal to  $\omega_n$  under root  $1 - \zeta^2$ . Therefore,  $F_0 \theta \dot{}$ .

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**Compare it with,**

$$\ddot{x} + \frac{k}{m}x + \frac{c}{m}\dot{x} = \frac{F_0 \sin \omega t}{m}$$

**Here,**

$$\omega_n = \sqrt{\frac{3ka^2}{ml^2}}$$

$$\zeta = \frac{3cb^2}{2ml^2\omega_n}$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2}$$

Now,  $\omega_n$  is equal to square root  $3ka$  square by  $ml$  square. This is  $\zeta$  equal to  $3cb$  square divided by  $2ml$  square  $\omega_n$  and  $\omega_d$  is equal to  $\omega_n$  under root  $1 - \zeta^2$ , this is  $\tau$  square.

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The particular solution is

$$\theta = A \cos(\omega t - \phi)$$

Where,

$$\phi = \tan^{-1} \frac{2\zeta \omega / \omega_n}{1 - (\omega / \omega_n)^2}$$

The particular solution of this is, theta is equal to A cos omega t minus phi, where phi is equal to tan inverse 2 zeta omega by omega<sub>n</sub> divided by 1 minus omega by omega<sub>n</sub> whole square.

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and

$$A = \frac{\frac{3F_0 l}{ka^2}}{\left[ \left( 1 - \left( \frac{\omega}{\omega_n} \right)^2 \right)^2 + \left[ 2\zeta \frac{\omega}{\omega_n} \right]^2 \right]^{1/2}}$$

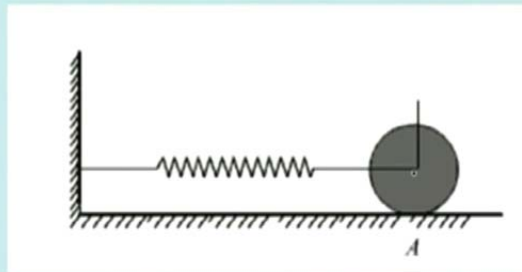
A is equal to 3F<sub>0</sub>l by 1 k square. This is 1 minus omega by omega<sub>n</sub> whole square. This is whole square plus 2 zeta omega by omega<sub>n</sub> whole square this is 1 by 2.



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### Vibration of rolling cylinder:

Consider a cylinder which rolls without slipping. It is held by spring.



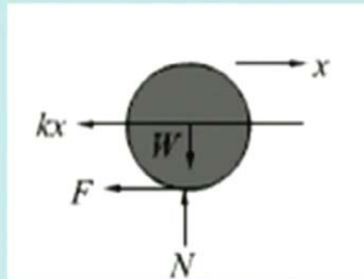
Let us discuss another problem; vibration of a rolling cylinder. A cylinder can, when it rolls on a straight line and if it is having a pure rolling motion then there is only 1 degree of freedom, because its angular displacement and the displacement of the center are related.  $r$  into  $\theta$  is equal to the displacement of the center. However, if it is not a pure rolling motion, if there is some slipping, then you know this type of thing is not correct; that means, there is no relation between the velocity of the center or and the angular velocity.

Suppose, you give the angular displacement of the cylinder, we cannot know how it much has moved, because there is some slippage. Therefore, one degree of freedom is not enough. We have to have 2 degree of freedom.

Therefore, we discuss the case of that motion in which the cylinder rolls without slipping. This cylinder is held by spring. This is the spring and it is attached at the centre and it is displaced slightly.

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Free body diagram of the cylinder is shown below. It is subjected to the following forces:



i) Spring force  $kx$ , (ii) friction force  $F$ , (iii) Normal reaction  $N$ , (iv) weight  $W$ .

Now, make the free body diagram of the cylinder which is shown here. I have shown the cylinder. Make the free body diagram. What are the forces? Let us discuss the forces coming on the centre.

One is the spring force, restoring force of the spring when the cylinder center has moved by distance  $x$  then the spring gets stretched by amount  $kx$ . So,  $kx$  is the force of the restoring force of the spring.  $W$  is the weight of the cylinder acts downward.  $N$  is the vertical reaction of the floor and  $F$  is the force due to friction. We have got these four forces: spring force  $kx$ , friction forces  $F$ , normal reaction  $N$  and weight  $W$ . Now,  $N$  if there is no vertical motion then  $N$  must be equal to  $W$ , but then there is a horizontal direction there is motion.

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Equation of motion in  $x$ -direction is,

$$-F - kx = M \ddot{x}$$

Now employing the angular momentum equation about the geometric axis of the cylinder at O using  $\theta$  to measure the rotation of the cylinder,

$$FR = \frac{1}{2}MR^2\ddot{\theta}$$

Note that  $I$  for the cylinder is  $\frac{1}{2}MR^2$ , where  $M$  is the mass and  $R$  is the radius)

So, equation of motion in  $x$  direction is minus  $F$  minus  $kx$  is equal to  $Mx$  double dot, because  $F$  is the friction force which we do not know as yet, but we have to write equation minus  $F$  minus  $k$ . If you employ the angular momentum equation about the geometric axis of the cylinder at O using  $\theta$  to measure the rotation of the cylinder, using  $\theta$  about this, therefore, it is  $F$  into  $R$  is equal to  $\frac{1}{2}MR^2\ddot{\theta}$ .

If you take about the geometric axis, when you take the moment about this point,  $kx$  will not produce any moment because it is passing through O;  $W$  will not produce,  $N$  will not, only  $F$  will produce a moment. That magnitude is given as  $F$  into  $R$  and that is equal to half  $MR^2\ddot{\theta}$ . Because, if it is a solid cylinder, its moment of inertia is half  $MR^2$ ; otherwise, if it is a hollow cylinder then it can be  $MR^2$ .  $I$  for the cylinder is half  $MR^2$ , where  $M$  is the mass and  $R$  is the radius.

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As there is no slip,  $\ddot{x} = R\ddot{\theta}$

$$FR = \frac{1}{2}MR^2 \left( \frac{\ddot{x}}{R} \right)$$

Therefore,

$$F = \frac{1}{2}M\ddot{x}$$

Substituting for  $F$ , we have

$$M\ddot{x} = -\frac{1}{2}M\ddot{x} - kx$$

$\frac{3}{2}M\ddot{x} + kx = 0$

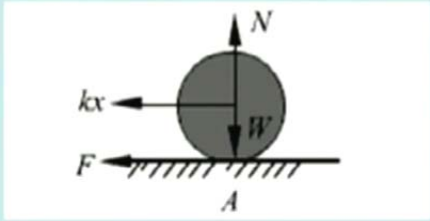
As there is no slip, we can write that equation,  $x$  double dot is equal to  $R$  theta double dot; that is, the linear acceleration is equal to  $R$  times the angular acceleration. Therefore,  $F$  into  $R$  is equal to half  $MR$  square into  $x$  double dot by  $R$  and therefore,  $F$  is equal to half  $M$   $x$  double dot. Now, substituting for  $F$ , we have  $M$   $x$  double dot is equal to minus half  $M$   $x$  double dot minus  $kx$ .

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$$\ddot{x} + \frac{2K}{3M}x = 0$$

This is the differential equation of motion.

Alternate way to obtain this differential equations:



Therefore,  $x \ddot{\phantom{x}} + \frac{2}{3} M \cdot x$  is equal to 0, because here, this will be  $\frac{3}{2} Mx \ddot{\phantom{x}} + kx$  equal to 0. Compare this equation with the spring-mass system, where we had  $Mx \ddot{\phantom{x}} + kx$  equal to 0. We are having  $\frac{3}{2} Mx \ddot{\phantom{x}} + kx$ . Therefore, effective mass is actually more. Therefore,  $x \ddot{\phantom{x}} + \frac{2}{3} kx$  plus  $x$  equal to 0. This is the differential equation of the motion of a cylinder held by the spring.

There is one alternate way to obtain this differential equation.

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**A is the instantaneous center of rotation. Taking moment about the point of contact**

$$\left( \frac{1}{2}MR^2 + MR^2 \right) \ddot{\theta} = -kxR$$

Now,

$$\ddot{\theta} = -\frac{\ddot{x}}{R}$$

$$-\frac{3}{2}MR^2 \frac{\ddot{x}}{R} = KxR$$

In this, we draw the free body diagram like this, where A is the instantaneous center of rotation. We take the moment about this fixed point of contact. About this point, mass moment of inertia is equal to half  $MR^2$  plus  $MR^2$  using parallel axis theorem; because, the mass moment of inertia about the center of the cylinder is  $MR^2$ . In that, you add half  $MR^2$  and  $\ddot{\theta}$  is equal to minus  $Kx$  times  $R$ . Now,  $\ddot{\theta}$  is equal to minus  $x \ddot{\phantom{x}}$  by  $R$ , as it is a pure rolling motion. In pure original motion, angular acceleration and linear acceleration of the center are related. This is, minus  $\frac{3}{2} MR^2 x \ddot{\phantom{x}}$  by  $R$  is equal to  $Kx$  by  $R$ .

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Therefore,

$$\ddot{x} + \frac{2K}{3M}x = 0$$

Solving the differential equation and putting the boundary conditions

$$x = x_0 \cos \sqrt{\frac{2K}{3M}}t + \frac{\dot{x}_0}{\sqrt{\frac{2K}{3M}}} \sin \sqrt{\frac{2K}{3M}}t$$

Therefore,  $\ddot{x} + \frac{2K}{3M}x = 0$ . We get the same type of differential equation. This differential equation can be solved. Solving this differential equation,  $\omega$  will be naturally under root  $\frac{2K}{3M}$   $x_0 \cos \sqrt{\frac{2K}{3M}}t + \frac{\dot{x}_0}{\sqrt{\frac{2K}{3M}}} \sin \sqrt{\frac{2K}{3M}}t$ ,  $t$  is outside the square root sign.

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where  $x_0$  and  $\dot{x}_0$  are the initial position and speed of the center of mass.

$$\theta = \frac{x}{R}$$

$$\theta = \frac{x_0}{R} \cos \sqrt{\frac{2K}{3M}}t + \frac{\dot{x}_0}{R \sqrt{\frac{2K}{3M}}} \sin \sqrt{\frac{2K}{3M}}t$$

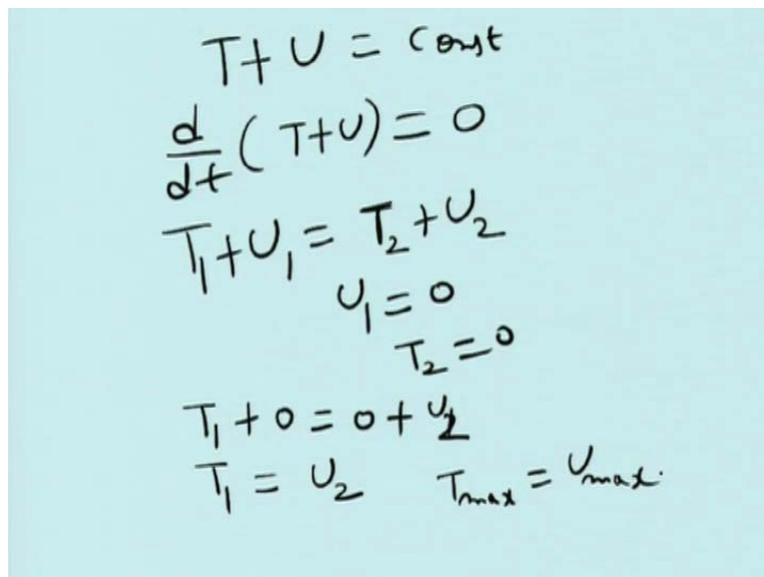
The natural frequency of the cylinder is  $\sqrt{\frac{2KR}{3M}}$

$x_0$  and  $\dot{x}_0$  are the initial position and speed of the center of the mass,  $\theta$  is equal to  $x$  by  $R$ . Now,  $\theta$  is equal to  $x$  by  $R$ . We put, in terms of that  $\theta$  is equal to  $x_0$  by  $R \cos$  under root this one. Similarly, here you will put  $\dot{x}_0$  by  $R$  under root  $2$  by  $3 K$  by  $M \sin$  under root  $2$  by  $3 R$  by  $M t$ . Natural frequency of the cylinder is  $2$  by  $3$ , this is  $K$  by  $M$ . This is the motion of that cylinder. So, we have explained about this.

Now, we have discussed these problems. Let us go to another problem. We can solve the vibration problems by another method, by energy method. That portion also, we will discuss here. We know that in the conservative system, the total energy is constant and the differential equation of motion can be established by the principle of conservation of energy. For the free vibration of an undamped system, the energy is partly kinetic and partly potential.

The kinetic energy is stored in the mass by virtue of its velocity, whereas the potential energy is stored in the form of strain energy in elastic deformation or work done in a force field such as gravity. You can have spring mass system, where the strain energy is stored in the spring and we can have simple pendulum, where the energy is stored in the force field like gravity. Now, total energy is always constant in the free vibration. Therefore, the rate of change is 0; therefore, what happens? We can write this equation.

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$$\begin{aligned}
 T + U &= \text{const} \\
 \frac{d}{dt}(T + U) &= 0 \\
 T_1 + U_1 &= T_2 + U_2 \\
 U_1 &= 0 \\
 T_2 &= 0 \\
 T_1 + 0 &= 0 + U_2 \\
 T_1 &= U_2 \quad T_{\max} = U_{\max}
 \end{aligned}$$

T plus U equal to constant in a free vibration. T is the kinetic energy, U is this one. This means that basically,  $\frac{d}{dt} T + U$  is equal to 0; that means, if you take the derivative with respect to time, you should get 0. Although in the kinetic energy you get the velocity term which may be function of time, and in the potential energy you get displacement which is also function of time. If our interest is only in the natural frequency of the system, it can be determined like this. From the principle of conservation of energy, we can write for two instants  $T_1 + U_1$  is equal to  $T_2 + U_2$ , where one and two's subscripts represent two instances of time.

Let one be the time when the mass is passing through its static equilibrium position. Choose  $U_1$  is equal to 0 as the reference for the potential energy. You know kinetic energy is 0 when the velocity is 0, but for potential energy, there is no such displacement is 0 then potential energy is 0. You have to take a datum. With respect to that datum, you measure the change in potential energy. Therefore, when  $U_1$  is equal to 0, as the reference for the potential energy  $U_1$  is equal to 0 at equilibrium position and  $T_2$  be the time, corresponding to the maximum displacement of the mass. At this position, the velocity of the mass is 0, because extreme position that is the maximum displacement has occurred. It cannot go beyond that. So, velocity must be 0; otherwise, it will go in that direction. So,  $T_2$  is equal to 0; therefore,  $T_2$  is also 0. You have a relation; then,  $T_1 + 0$  is equal to 0 plus  $U_1$ , or we have got like this  $T_1$  is equal to  $U_2$ , this is  $U_2$ . So, this is  $T_1$  is equal to  $U_2$ .

If this system is undergoing harmonic motion then  $T_1$  and  $U_1$  are of maximum value; that means, at the center,  $T_1$  will be maximum when the potential energy is 0. So, obviously that  $T_1$  is maximum and then, here  $U_2$  is of maximum. Therefore  $T_{\max}$  is equal to  $U_{\max}$ .



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$$\begin{aligned}T &= \frac{1}{2} m \dot{x}^2 \\U &= \frac{1}{2} k x^2 \\ \frac{d}{dt} \left( \frac{1}{2} m \dot{x}^2 + \frac{1}{2} k x^2 \right) &= m \dot{x} \ddot{x} + k x \dot{x} = 0 \\ m \dot{x} \ddot{x} + k x \dot{x} &= 0 \\ m \ddot{x} + k x &= 0 \\ x &= A \sin \omega t \\ T &= \frac{1}{2} (A \omega \cos \omega t)^2 m \\ T_{\max} &= \frac{1}{2} (A \omega)^2 m = \frac{1}{2} m A^2 \omega^2\end{aligned}$$

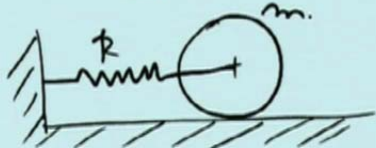
Let us see how you can apply this condition for a spring mass system.  $T$  is equal to half  $m \dot{x}$  square. At any instant, its kinetic energy is given by half  $m \dot{x}$  square and  $U$  is equal to half  $k x$  square. Now,  $T$  plus  $U$  is constant; therefore, half  $m \dot{x}$  square plus half  $k x$  square  $d$  by  $dt$  is equal to  $m \dot{x} \ddot{x}$  plus  $k x \dot{x}$  equal to 0, or you get the equation  $m \dot{x} \ddot{x} + k x \dot{x}$  equal to 0. Therefore, we get a differential equation  $m \ddot{x} + k x$  equal to 0.

This is because  $\dot{x}$  can be eliminated except at the extreme position, where  $\dot{x}$  is equal to 0. Therefore, you get the differential equation that is  $m \ddot{x} + k x$  is equal to 0. This differential equation is equal to like this. Now, for finding out the natural frequency, let us assume that  $x$  is equal to  $A \sin \omega t$ . If you assume that  $x$  is equal to  $A \sin \omega t$ , then we get  $T$  is equal to half  $A \omega \cos \omega t$  whole square multiplied by  $m$ . As the maximum value of  $\cos \omega t$  is 1, therefore,  $T_{\max}$  is equal to half  $A \omega$  square into  $m$  is equal to half  $m A$  square  $\omega$  square.

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$$U = \frac{1}{2} k (A \sin \omega t)^2$$
$$U_{\max} = \frac{1}{2} k A^2$$

Now,  $T_{\max} = U_{\max}$

$$\frac{1}{2} m A^2 \omega^2 = \frac{1}{2} k A^2$$
$$\omega = \sqrt{\frac{k}{m}}$$


Similarly, the potential energy is given by  $U$  is equal to half  $k A \sin \omega t$  whole square. As the maximum value of  $\sin \omega t$  is 1, the maximum potential energy  $U_{\max}$  will be half  $k A$  square; maximum value is half  $k A$  square. Now  $T_{\max}$  is equal to  $U_{\max}$ . Therefore, half  $m A$  square  $\omega$  square is equal to half  $k A$  square.

From this equation, we get  $\omega$  is equal to under root  $k$  by  $m$  which is the expression for the natural frequency. Therefore, you get  $\omega$  is equal to under root  $k$  by  $m$ . Like that, we can find out. In the last lecture, we have solved one problem, vibration problem. The cylinder rolls without slipping on this thing  $k$  and this is the mass  $m$ . Now, it is a cylinder which rolls without slipping and is held by spring. The total energy of the system is given by half  $mV$  square. So, let me write the expression.

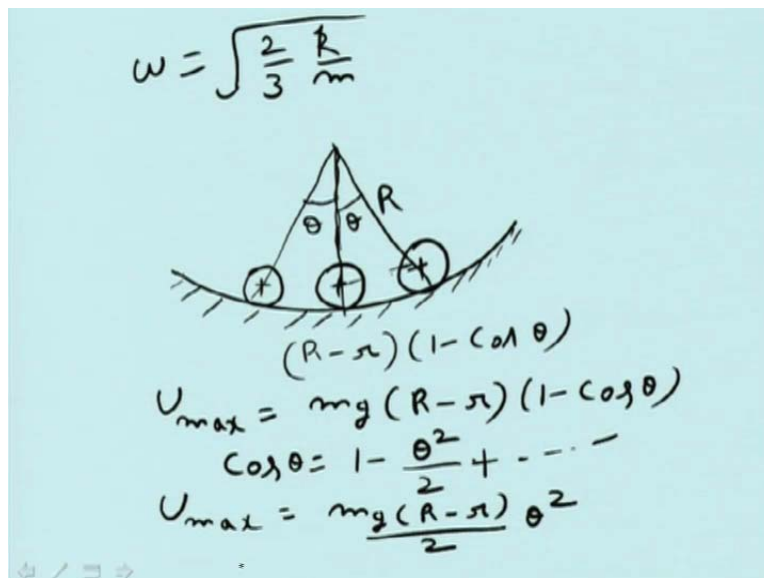
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$$\begin{aligned}\frac{1}{2} m v^2 + \frac{1}{2} I \omega^2 + \frac{1}{2} k x^2 &= C \\ &= \frac{1}{2} m (\dot{x})^2 + \frac{1}{2} I \left(\frac{\dot{x}}{r}\right)^2 + \frac{1}{2} k x^2 \\ m \ddot{x} + \frac{I}{r^2} \ddot{x} + k x &= 0 \\ \omega &= \sqrt{\frac{k}{\left(m + \frac{I}{r^2}\right)}} \\ &\text{For a solid cylinder} \\ I &= \frac{1}{2} m r^2\end{aligned}$$

Here, half  $m v^2$  square plus half  $I \omega^2$  square plus half  $k x^2$  square is equal to constant. Total energy, you know is constant. In this case, this can be written as half  $m \dot{x}^2$  square plus half  $I \dot{x}^2 / r^2$  square plus half  $k x^2$  square. If you differentiate this equation for total energy with respect to time, you get  $m \ddot{x} + I \ddot{x} / r^2 + k x = 0$ . Note that, we could write  $\omega$  is equal to  $\dot{x} / r$ , because of the rolling motion, this is pure rolling motion.

So,  $\omega$  is equal to under root  $k$  divided by  $m + I / r^2$ . You get this term. For a solid cylinder,  $I$  is equal to half  $m r^2$ .

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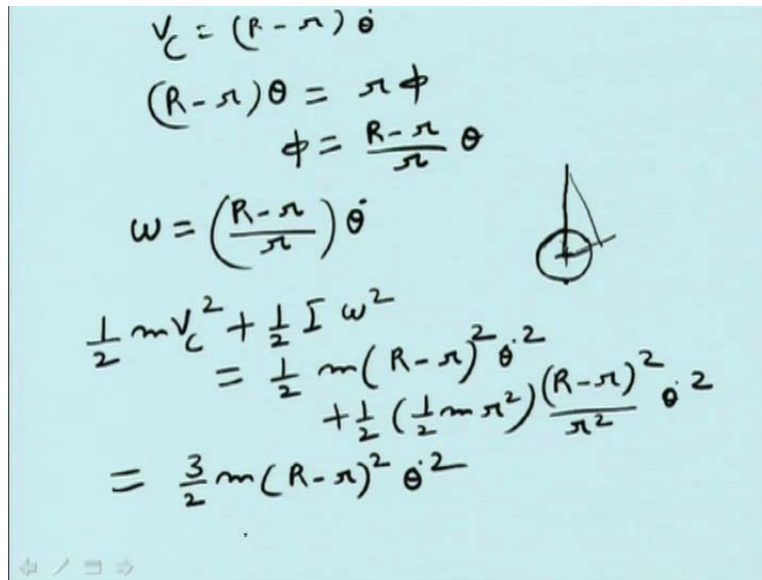
Therefore, omega is equal to under root 2 by 3 k by m; that is, the same expression. Let us take another example, on the use of the energy method. Here, if we have got a concave surface like this and you have kept a ball just like that. Now, displace it slightly. So, it goes like this here, Then its potential energy increases. At this point, it tries to come back but by that time it has attained velocity, it will go on the other side. Neglect the friction. You see that this goes this side then again it tries to come and like that it keeps oscillating.

Here, it attains the maximum velocity. After that, comes back and after that goes in this direction. Again, its velocity becomes zero; then, this is theta and this is R. If this is the maximum angle of the theta and increase in the height of the centre of the cylinder is basically R minus r 1 minus cos theta, then that is the increase in the height of the center of the cylinder. Cylinder is of radius r, this distance is R minus r. Therefore, this is R minus r into 1 minus cos theta.

Assume that the potential energy at the equilibrium position in the middle is 0. So, maximum potential energy,  $U_{\max}$  is equal to mg R minus r into 1 minus cos theta, theta is the angular amplitude. Here, if we assume that theta is small, then cos theta can be written as 1 minus, expression for cos theta is 1 minus theta square by 2, plus other terms, where theta is meridian.

Therefore,  $1 - \cos \theta$  is  $\theta^2$  by 2. Therefore,  $U_{\max}$  is equal to  $mgR$  minus  $r$  divided by 2  $\theta^2$  or small  $\theta$ .

(Refer Slide Time: 40:32)



The image shows a handwritten derivation on a light blue background. It starts with the velocity of the center of mass  $V_c = (R - r)\dot{\theta}$ . Then, it relates the rotation angle  $\phi$  to the cylinder's rotation  $\theta$  using  $(R - r)\theta = r\phi$ , leading to  $\phi = \frac{R - r}{r}\theta$ . The angular velocity is then  $\omega = \left(\frac{R - r}{r}\right)\dot{\theta}$ . A small diagram of a cylinder with radius  $r$  and center of mass at distance  $R - r$  from the contact point is shown. Finally, the kinetic energy is calculated as  $\frac{1}{2}mV_c^2 + \frac{1}{2}I\omega^2$ , which simplifies to  $\frac{3}{2}m(R - r)^2\dot{\theta}^2$  after substituting  $I = \frac{1}{2}mr^2$ .

$$\begin{aligned}
 V_c &= (R - r)\dot{\theta} \\
 (R - r)\theta &= r\phi \\
 \phi &= \frac{R - r}{r}\theta \\
 \omega &= \left(\frac{R - r}{r}\right)\dot{\theta} \\
 \frac{1}{2}mV_c^2 + \frac{1}{2}I\omega^2 &= \frac{1}{2}m(R - r)^2\dot{\theta}^2 + \frac{1}{2}\left(\frac{1}{2}mr^2\right)\frac{(R - r)^2}{r^2}\dot{\theta}^2 \\
 &= \frac{3}{2}m(R - r)^2\dot{\theta}^2
 \end{aligned}$$

For a small rotation, the velocity of the mass center,  $V_c$  is equal to  $R$  minus  $r$   $\theta$  dot, a small rotation. Let the angle of rotation of this cylinder is  $\phi$ , then  $R$  minus  $r$  into  $\theta$ ; that is, the moment of the center is equal to  $r\phi$ , where  $\phi$  is the angle of rotation of the cylinder about its own center. Therefore,  $\phi$  is equal to  $r\phi$ .  $\phi$  is equal to  $R$  minus  $r$  divided by  $r$   $\theta$  dot. Hence, the angular velocity of the cylinder is equal to  $\omega$   $R$  minus  $r$  divided by  $r$  into  $\theta$  dot, or else, see it like this that there is a cylinder, this is  $R$  and this is moved by some distance, center moves  $R$  minus  $r$  into  $\theta$ .

Therefore, about instantaneous center, the distance is  $r$ . Therefore,  $\omega$  is equal to  $R$  minus  $r$  by  $r$   $\theta$  dot. The total kinetic energy of the cylinder is equal to half  $mV_c$  square plus half  $I\omega$  square is equal to half  $m$   $R$  minus  $r$  square  $\theta$  dot square plus half half  $m$   $r$  square  $R$  minus  $r$  square by  $r$  square  $\theta$  dot square, which is equal to  $\frac{3}{2}m$   $R$  minus  $r$  square  $\theta$  dot square.

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The image shows a handwritten derivation in a Notepad window. The equations are as follows:

$$\frac{1}{2} m v_c^2 + \frac{1}{2} I \omega^2 = \frac{3}{2} m (R-r)^2 \dot{\theta}^2$$

$$\frac{3}{2} m (R-r)^2 \dot{\theta}^2 + \frac{mg(R-r)\theta^2}{2} = C$$

$$\frac{3}{2} m (R-r)^2 2 \ddot{\theta} \dot{\theta} + \frac{mg(R-r)}{2} 2 \theta \dot{\theta} = 0$$

$$\text{or } 3 \ddot{\theta} + \frac{2g}{(R-r)} \theta = 0$$

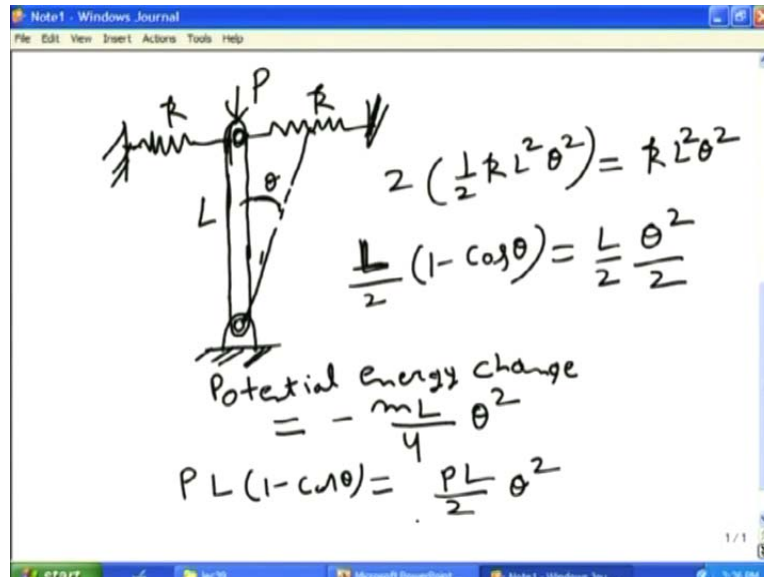
$$m \ddot{x} + kx = 0$$

$$\omega_n = \sqrt{\frac{g}{\frac{3}{2}(R-r)}}$$

Total kinetic energy of the cylinder is half  $m v_c^2$  square plus half  $I \omega^2$  square which we got 3 by 2  $m R$  minus  $r$  square  $\theta$  dot square. Total energy comes out to be 3 by 2  $m R$  minus  $r$  square  $\theta$  dot square plus  $mg R$  minus  $r$   $\theta$  square by 2 is equal to constant. Differentiate this expression with time; this must be 0. That means, 3 by 2  $m R$  minus  $r$  square 2  $\theta$  double dot  $\theta$  plus  $mg R$  minus  $r$  by 2  $\theta$   $\theta$  dot equal to 0 or  $\theta$  dot dot plus  $2g$  divided by  $R$  minus  $r$  into  $\theta$  equal to 0.

Compare it with  $m \ddot{x} + kx = 0$ . We see that  $\omega_n$  is equal to under root  $g$  divided by 3 by 2  $R$  minus  $r$ . We get this type of expression. If  $R$  is equal to infinite, that means that it is plane surface. Then, there is no vibration and if  $R$  is equal to, instead of that if this is just a ball, if a point mass particle is there then it will be  $g$  by 3 by 2  $R$  minus  $r$ . This is the velocity of that.

(Refer Slide Time: 46:11)



If you take a rod and there is a spring attached here,  $k$  and this spring is attached on this side as well this is  $k$ , this is  $P$  and this is  $L$ . Let a rod of mass  $m$  and length  $L$  is restrained in the vertical position by two identical springs having spring constant  $k$ , a vertical load  $P$  acts on the top of the rod. This is acting if given a small displacement  $\theta$  to the rod, one spring compresses by amount  $L \theta$ .

If you just give a displacement  $\theta$ , so I am showing very exaggerated,  $\theta$ , this is  $L \theta$  and the other spring stretches by the same amount that is this one. So, strain energy in the spring is equal to,  $2 \times \frac{1}{2} k L^2 \theta^2$ , because this is the thing that is  $k L^2 \theta^2$ . When the rod moves like this, then the center of the gravity of the mass, moves by  $L$  by  $2$  into  $1 - \cos \theta$ . That means, again that is  $L$  by  $2$  into  $\theta^2$  by  $2$ , provided  $\theta$  is very small.

The potential energy of the rod changes, and potential energy change due to this is given by potential energy change is equal to  $-\frac{mL}{4} \theta^2$ . Now, this is due to this one but the force  $P$  which is acting, also gets displaced. That also comes down. So, potential energy change due to rod  $P$  is  $PL$ , because this is  $PL$ ,  $P$  into  $L$   $1 - \cos \theta$  or this is  $PL$  by  $2$   $\theta^2$ .

(Refer Slide Time: 50:16)

The image shows a handwritten derivation in a Notepad window. The equations are as follows:

$$\frac{1}{2} m \left( \frac{l}{2} \dot{\theta} \right)^2 + \frac{1}{2} m l^2 \dot{\theta}^2$$

$$= \frac{1}{2} \frac{4 m l^2}{12} \dot{\theta}^2 = \frac{1}{2} \frac{m l^2}{3} \dot{\theta}^2$$

$$= \frac{1}{2} m \frac{l^3}{3} \dot{\theta}^2$$

$$\text{Total energy} = \frac{m l^2}{6} \dot{\theta}^2 + \left( k L^2 - \frac{P L}{2} - \frac{m L}{4} \right)$$

So, kinetic energy of the rod is equal to half  $m l$  by 2 theta square plus 1 by 2  $m l$  square theta dot square, or basically it will come  $l$  square by 4 and this is 1 by 12. It is  $I$  theta square, because this will become equal to half  $m l$  square by 12 and this is 4. So, you can take 4 common and this becomes  $l$  square  $m l$  square by 12. This is 3 plus 1, 4  $m l$  square by 12 theta dot square or  $m l$  square by 3 theta dot square into half 4  $m l$  square.

This is equal to half  $m l$  cube by 3 theta dot square. Therefore, total energy is equal to  $m L$  square by 6 theta dot square plus  $k L$  square minus  $P L$  by 2 minus  $m L$  by 4 theta square. This is total energy, previous 2 by 3.



(Refer Slide Time: 53:18)

The image shows a handwritten derivation in a Notepad window. The steps are as follows:

$$= \frac{1}{2} \frac{4 m l^2}{12} \dot{\theta}^2 = \frac{1}{2} \frac{m l^2}{3} \dot{\theta}^2$$

$$= \frac{1}{2} m \frac{l^3}{3} \dot{\theta}^2$$

$$\text{Total energy} = \frac{m l^2}{6} \dot{\theta}^2 + \left( k L^2 - \frac{P L}{2} - \frac{m L}{4} \right) \theta^2$$

$$\frac{m l^2}{6} \dot{\theta}^2 + \left( k L^2 - \frac{P L}{2} - \frac{m L}{4} \right) \theta^2 = C$$

$$0 = \frac{m l^2}{6} 2 \dot{\theta} \ddot{\theta} + \left( k L^2 - \frac{P L}{2} - \frac{m L}{4} \right) 2 \dot{\theta} \theta$$

$$\frac{m l^2}{6} \ddot{\theta} + \left( k L^2 - \frac{P L}{2} - \frac{m L}{4} \right) \theta = 0$$

Total energy is equal to, this is the expression theta is this is theta square that means let me write it again. Total energy is  $m L$  square by 6 theta dot square plus  $k L$  square minus  $P L$  by 2 minus  $m L$  by 4 theta square. Now, differentiate. This is equal to constant. So, differentiate it with respect to time, this expression can be differentiated with respect to time and then you get, 0 is equal to  $m L$  square by 6 into 2 theta dot theta double dot plus  $k L$  square minus  $P L$  by 2 minus  $m L$  by 4 into 2 theta dot theta.

Therefore, theta dot can be cancelled throughout. Therefore, you get  $m L$  square by 6 theta double dot plus  $k L$  square minus  $P L$  by 2 minus  $m L$  by 4 theta is equal to 0. Now, observe this expression that you have written  $m L$  square by 6 theta double dot plus  $k L$  square minus  $P L$  by 2 minus  $m L$  square.

(Refer Slide Time: 55:27)

The image shows a handwritten derivation in a Notepad window. The equations are as follows:

$$0 = \frac{mL}{6} 2\ddot{\theta} + \left( kL^2 - \frac{PL}{2} - \frac{mL}{4} \right) \ddot{\theta}$$

$$\frac{mL}{6} \ddot{\theta} + \left( kL^2 - \frac{PL}{2} - \frac{mL}{4} \right) \ddot{\theta} = 0$$


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$$m\ddot{x} + kx = 0$$

$$\left( kL^2 - \frac{PL}{2} - \frac{mL}{4} \right) \ddot{\theta} = 0$$

$$kL^2 - \frac{PL}{2} - \frac{mL}{4} = 0$$

$$\text{or } P = 2kL^2 - \frac{mL}{4}$$

Compare this expression with  $m\ddot{x} + kx = 0$ . When you compare this expression, you notice that this was positive, then only we got  $k$  is positive. Therefore, this expression in the parenthesis must also be positive, then only the system will vibrate. If the expression is not positive then the system will not be vibrating. Therefore, the condition for this is that, this expression must be this one then it will be vibrating, and restoring force will be able to bring to original position.

However, if you go on increasing  $P$  then you can say this expression  $P L$  by  $2 k L$  square minus because that expression pay attention to this expression  $k L$  square minus  $P L$  by  $2$  minus  $m L$  by  $4$ . What happens if this expression is less than  $0$ ? If we increase  $P$ , what will happen? In that case, there will be a value of  $P$ , at which this expression will become negative. Beyond that, if you put that also, at that point it will be unstable. The rod will not be able to come back when you displace it. It will go in the other direction exactly when  $k L$  square minus  $P L$  by  $2$  minus  $m L$  by  $4$  equal to  $0$ . At that point, you can give any displacement and the rod will remain at that position. It will not try to come back.

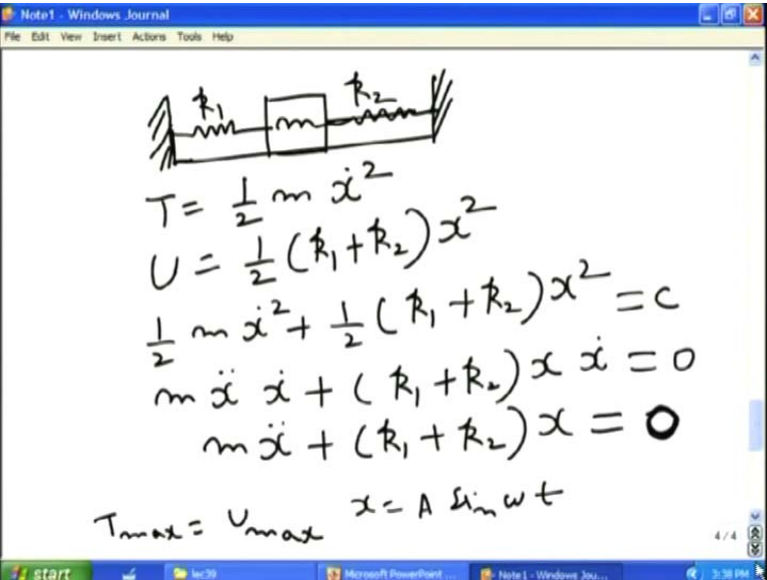
So,  $P$  can be called critical load, critical load at which this will become unstable. There will be not be any vibrations but this is the thing and if  $P$  is more than that, naturally it will have tendency to go to the other direction. Therefore, the critical load is  $k L$  square minus  $P L$  by  $2$

minus  $mL$  by 4 equal to 0, or  $P$  is equal to  $kL^2/2$  because that is minus  $mL$  by 4.

At that point, this rod will try to come back to that original position. That means, if  $P$  is less than  $kL^2/2$  minus  $mL$  by 4 the rod will try to come back to the previous position. This is how we have obtained the expression for that vibration. Here, to summarize what we have done; we have shown that there is another approach to obtain the differential equation of a motion that is not by solving the system forced by this, but by the energy consideration.

In the free vibration case, total energy of the system remains constant. Therefore, by that consideration, we have obtained these expressions and we can find out the angular speed. That is another way of solving the vibration equation. In many cases, this system becomes very simple.

(Refer Slide Time: 59:44)



The image shows a handwritten diagram of a mass-spring system. A mass  $m$  is connected to two fixed walls on either side by two springs with spring constants  $k_1$  and  $k_2$ . Below the diagram, the following equations are written:

$$T = \frac{1}{2} m \dot{x}^2$$

$$U = \frac{1}{2} (k_1 + k_2) x^2$$

$$\frac{1}{2} m \dot{x}^2 + \frac{1}{2} (k_1 + k_2) x^2 = C$$

$$m \dot{x} \ddot{x} + (k_1 + k_2) x \dot{x} = 0$$

$$m \ddot{x} + (k_1 + k_2) x = 0$$

$$T_{\max} = U_{\max} \quad x = A \sin \omega t$$

Let us quickly do one more problem and then conclude this thing. Supposing you have the vibration of this rod  $m$  and this is  $kx$   $k_1$ , this is  $k_2$ . Then if you put kinetic energy  $T$  is equal to half  $m \dot{x}^2$  then  $U$  is equal to half  $k_1$  plus  $k_2 x^2$ , because half  $k_1 x^2$ . Energy is stored in both and therefore, half  $m \dot{x}^2$  plus half  $k_1$  plus  $k_2 x^2$  is equal to  $T$  plus  $U$ , that is constant. Differentiate it with respect to time. You get,  $m \dot{x} \ddot{x}$  plus  $k_1$  plus  $k_2 x \dot{x}$  is equal to 0. You get the differential equation,  $m \ddot{x}$  plus  $k_1$  plus  $k_2$  into  $x$  is equal to 0. In

this case, the equation of this one, if you quickly want to find the frequency, we can always put,  $x$  is equal to  $A \sin \omega t$  and then we can find out  $\dot{x}$ . Then by equating  $T_{\max}$  is equal to  $U_{\max}$ , we can find out the expression for angular  $\omega$  that means circular frequency. Therefore, I stop at this point.