


Non-Linear Vibration
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Module - 6
Applications
Lecture – 3
Free Nonlinear Vibration of
Multi Degree of Freedom System

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Points to be learned from this lecture

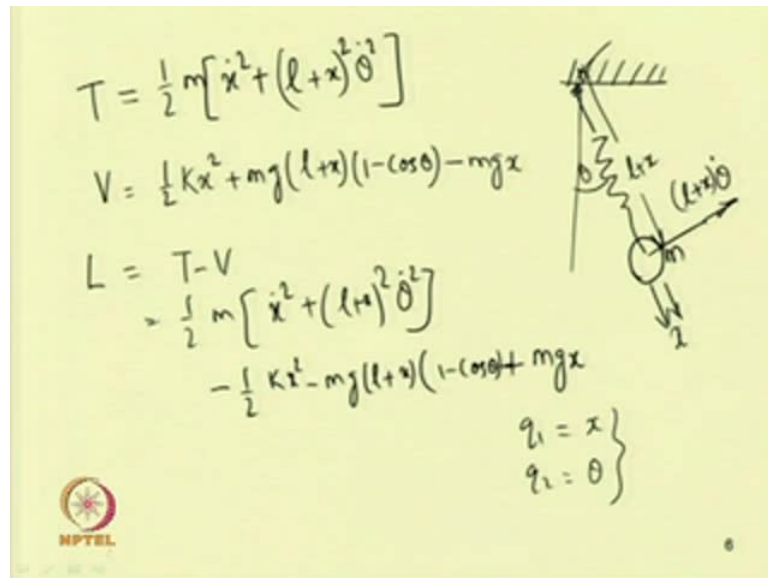
- Governing equations of multi degree of freedom systems
 - Quadratic nonlinearities, cubic nonlinearities
 - Systems with and without internal resonances
- Solution methods
- Determination of steady state response
- Comparison of Linear and nonlinear system response


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So, welcome to today class of non-linear vibration, so in this module, we are discussing about the applications of non-linear systems, where we are discussing about the free vibration force vibration and then parametrically excited systems, so last two classes we are discussing about the free vibration of single degree of freedom system and today class we are going to discuss about non-linear free vibration of multi degree of freedom system, so in last two classes, so we have seen, so how this single degree of freedom systems non-linear behaviour can be can be obtained from this free vibration analysis, so difference between free vibration of linear and non-linear systems we have discussed, so we have seen in case of linear system, so there is the frequency is not dependent on the amplitude but, in case of non-linear system the frequency of oscillation depends on the amplitude of the non-linear system, and in case of multi degree of freedom system today we are going to study how to determine the governing equation of motion in case of the multi degree of freedom system.

So, here we will discuss about this quadratic and cubic nonlinearities systems with and without internal resonance, and then the solution methods and then we will study or we will discuss about the determination of steady state response stability, and then we will compare the response of a linear and non-linear system.

(Refer Slide Time: 02:05)



$$T = \frac{1}{2} m [\dot{x}^2 + (l+x)^2 \dot{\theta}^2]$$

$$V = \frac{1}{2} k x^2 + mg(l+x)(1-\cos\theta) - mgx$$

$$L = T - V$$

$$= \frac{1}{2} m [\dot{x}^2 + (l+x)^2 \dot{\theta}^2] - \frac{1}{2} k x^2 - mg(l+x)(1-\cos\theta) + mgx$$

$q_1 = x$
 $q_2 = \theta$

So, let us take a simple example of a spring mass system, so unlike the spring mass systems we have studied before let it is spring mass, where so the spring can extend with this mass m , so let the original length of the spring is l now with extension, so it has extended in this direction by an amount x , so this extension extended length becomes l plus x original length was l , now the length change to l plus x , so to derive the equation motion, so one can use different methods but, particularly for multi degree of freedom systems it is convenient to use Lagrange equation, so here to use this Lagrange equation one should know, the velocity of the system, so the velocity in this direction that is in this direction it is \dot{x} and the velocity in this direction, so as it is moving by an amount θ or rotating by an amount θ .

So, the velocity here will be l plus x that is the distance into $\dot{\theta}$, so the kinetic energy one can write the kinetic energy and potential energy of the system the kinetic energy of the system k or T kinetic energy T can be written as half m into \dot{x} square plus velocity in this direction, so plus velocity in this direction also, it will be equal to l

plus x square and theta dot square, so the potential energy u can or v can be written as half, so for the spring.

So, let us assume this spring as a constant spring constant k, so then this potential becomes half k into x square plus m g into l plus x into 1 minus cos theta, so this angle is theta 1 minus cos theta minus m g x, so the potential energy equal to half k x square that is this extension of the spring, if you are assuming the extensions of the spring is x then it is equal to half k x square then due to change in position of the mass, so the change in position of the mass becomes m g into l plus x into 1 minus cos x cos theta 1 minus cos theta minus m g x, so now one can write this L equal to T minus V.

(Refer Slide Time: 05:49)

Handwritten mathematical derivation for a spring-mass system with a pivot. The derivation shows the Lagrangian equation, equations of motion for x and theta, and their solutions. A diagram of a mass on a spring is also shown.

$$\frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_k} \right) - \frac{\partial L}{\partial q_k} = Q_k^o$$

$$\begin{cases} \ddot{x} + \omega_2^2 x - (l+x)\dot{\theta}^2 - g \cos \theta = 0 \\ \ddot{\theta} + \frac{g \sin \theta + 2\dot{x}\dot{\theta}}{l+x} = 0 \end{cases}$$

$$\begin{cases} \ddot{x} + \omega_2^2 x = 0 \\ \ddot{\theta} + \frac{g}{l} \theta = 0 \end{cases}$$

$$\omega_2 = \frac{k}{m}$$

$$\omega_1 = \sqrt{\frac{g}{l}}$$

$$\begin{cases} x = a \sin(\omega_2 t + \phi) \\ \theta = b \sin(\omega_1 t + \beta) \end{cases}$$

Diagram: A mass m is attached to a spring with constant k and a pivot. The spring is extended by x. The mass is at an angle theta from the vertical.

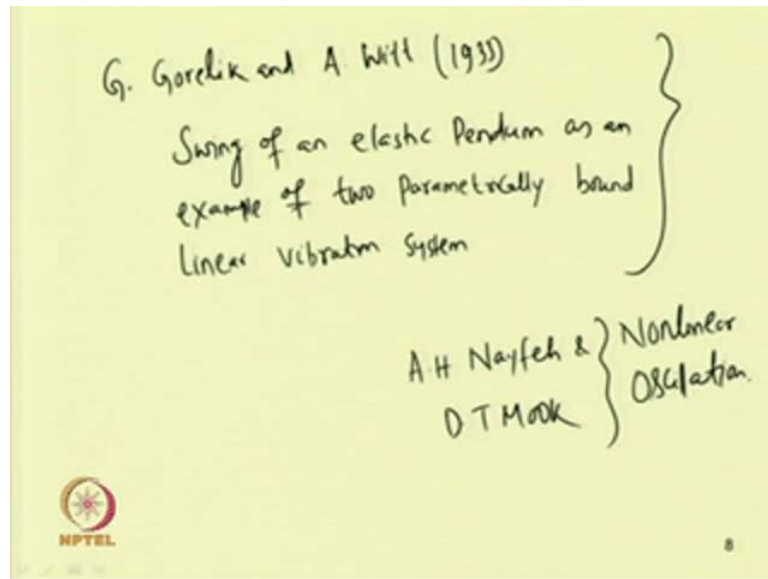
So, this will be equal to half m x dot square plus l plus x whole square theta dot square minus half k x square minus m g into l plus x into 1 minus cos theta, and minus minus plus, so this becomes m g into x, so this is the Lagrangian of the systems, so after getting the Lagrangian of system, so as we have two generalized coordinate, the generalized coordinate q 1 equal to we can take x and q 2 equal to theta, so if you take these two generalized coordinate then the equation motion can be written in this form that is d by d t of del l by del q k dot minus del l by del q k, so this will be equal to capital Q k as there is no force acting on the systems as we are considering only the free vibration in this case, so this Q k equal to 0, so now taking this q or q 1 equal to x and q 2 equal to theta.

So, we can write the equation motion in this form, that is $x \ddot{x} + \omega^2 x - l \dot{\theta}^2 - g \cos \theta = 0$, and second equation becomes $\ddot{\theta} + g \sin \theta + 2 \dot{x} \dot{\theta} / l + x \ddot{\theta} = 0$, so here $\omega^2 = k/m$ and this ω , we can write $\omega = \sqrt{g/l}$, so which is the we have not used this ω term till now, so ω you can write the oscillation of the frequency of the pendulum, so if when we are considering the spring.

So, if we have a simple pendulum of length l , we know the frequency equal to $\sqrt{g/l}$, so we got these two equations, so in this two equation, so these are the non-linear terms that is $\dot{\theta}^2$ terms containing $\dot{\theta}^2$ and this $\cos \theta$ term also, we can expand this $\cos \theta$, if θ is not small, so then we can expand $\cos \theta$, so that is also a non-linear term. Similarly, in the second equation, so we have this $\sin \theta$ term which is non-linear and $2 \dot{x} \dot{\theta} / l$ term also non-linear terms, so if we neglect these non-linear terms assuming small amplitude of oscillation then this equation reduces to $x \ddot{x} + \omega^2 x = 0$, and second equation reduce to $\ddot{\theta} + g \sin \theta = g/l \sin \theta$, so this is equal to $g/l \sin \theta$, and this $\sin \theta$ taking, $\sin \theta = \theta$ for θ to be small, so this equation becomes this, so these two are two uncoupled equations and one can obtain the solution of this uncoupled equation like $x = a \sin \omega t + \phi$.

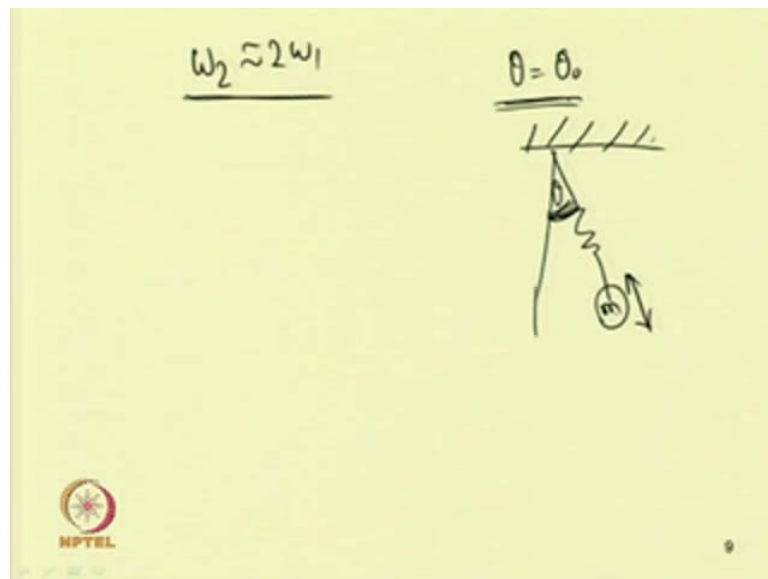
So, x can be $a \sin \omega t + \phi$, and this θ can be $b \sin \omega_1 t + \beta$, so here this a and ϕ are the initial, so a and ϕ can be obtained from the initial condition similarly, b and β can be obtained from the initial condition, so in this case we have two uncoupled equation, so if θ is small, so we have two uncoupled equations and the response are uncoupled; that means, the extension of this spring that is by x does not depends on θ that is the rotation of the pendulum, so these two are two different modes of oscillation but, it has been shown.

(Refer Slide Time: 09:51)



So, it has been shown by Gorelik and A. Witt, G. Gorelik g o Gorelik and A Witt, so it has been shown by A Gorelik and Witt in 1933, so this is published in swing in published in or the title of the paper is swing of an elastic pendulum, swing of an elastic pendulum as an example of two parametrically, two parametrically bound linear vibration system also one may refer the book by A.H Nayfeh and, and D T Mook for detailed discussion on this non-linear oscillations.

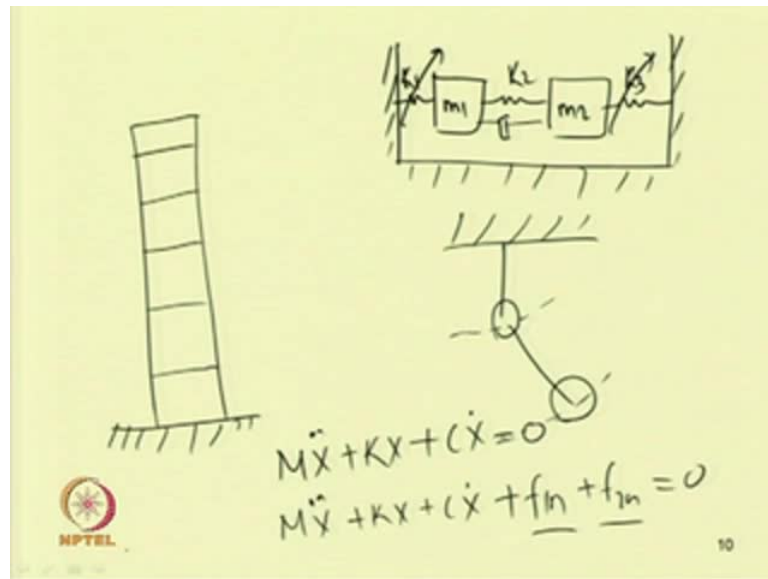
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So, here it has been shown that if there is relation between the second mode or there is relation between this ω_2 and ω_1 , for example, if one take this ω_2 nearly equal to $2\omega_1$, then this though the amplitude of oscillation be small, so in that case, so by extending or by taking this motion, initial motion θ equal to θ_0 , where θ_0 is not equal to 0 then there is, so, this is the spring, so by giving the small oscillation θ , θ equal to θ_0 and released, so if it is released then it is observed that by pulling this mass m slightly down, so there is energy transfer between these two modes; that means, so there is alternate, there is alternate rotation of this pendulum or there is there is oscillation displacement in this direction and also rotation about this direction.

So, if one starts the motion when θ equal to θ_0 , where θ_0 equal to 0 but, a very small by pulling the mass m down, one finds that the mass oscillate up and down first, so the mass oscillates up and down first and then a pendulum type of component of motion develops and grows at the expense of spring type motion after a while the pendulum type motion starts to decrease and the spring type motion starts to grow, so thus energy is transferred from one mode to other mode, so this shows the internal resonance, so though there is no external force acting on the systems as ω_2 is taken equal to twice ω_1 , so there is transfer of energy between these two modes of vibration, so we have seen or it is shown that though the amplitude is small, in case of or considering the system to be non-linear, so there is energy transfer between two modes of vibration.

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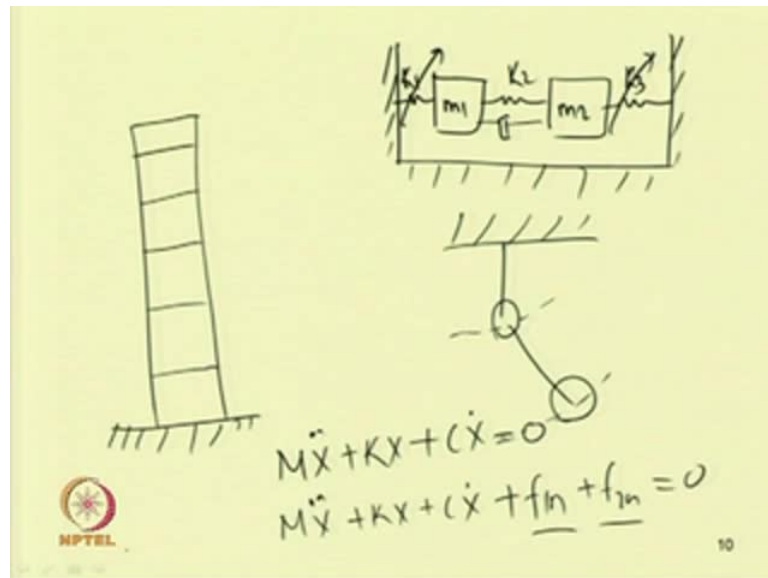


So, like these systems one can take several other type of systems also to derive the equation motion non-linear equation motion of the system for example, one can take this simple spring mass system, so where let us take only two one can take more also more number of spring mass system, and let us add this damper and this spring, so only damper let us put, we can put the damper or we may have a damper in the middle, so this is k_1 this is k_2 and this is k_3 , so in these systems by considering this spring to be non-linear, so let this k_1 and k_3 are non-linear spring then one can find the governing equation motion, governing non-linear equation motion similarly, one can take another example of the double pendulum, so in case of the double pendulum, so as the oscillation, so taking the oscillation not to be very small then one can find a coupled equation motion in this case, so in case of the double pendulum similarly many other systems.

For example, let us take a multi stored building, multi stored building subjected to earthquake excitation, so in that case after the excitation earthquake excitation, the systems will be reduced to that of a non-linear multi degree of freedom system, similarly, many other systems can be modelled, so many other systems can be modelled as multi degree of freedom systems and one can study the equation motion in those cases, so for the linear multi degree of freedom systems, the equation becomes $M \ddot{x} + Kx + C \dot{x} = 0$. But, for non-linear, so we can add the non-linear terms, so we can write $M \ddot{x} + Kx + C \dot{x}$, and here the non-linear terms can be

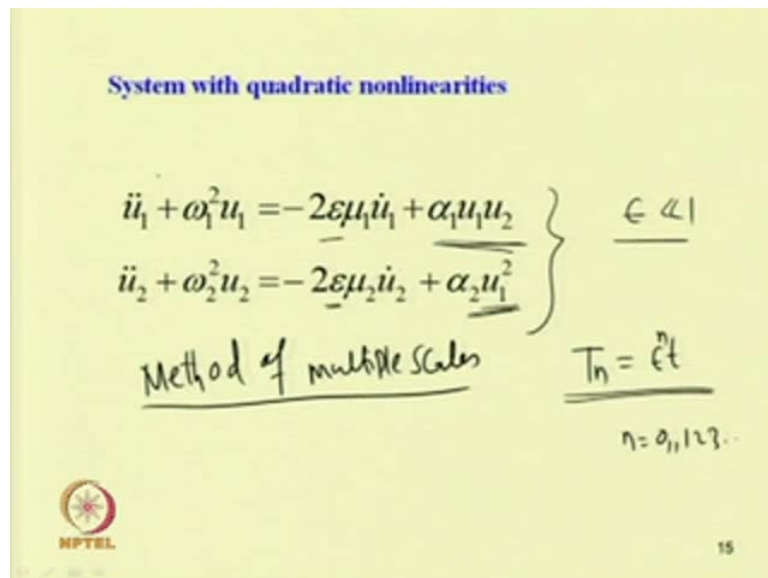
added, so non-linear, so this is this is f_1 f_2 , so one can go on adding these non-linear terms which will be equal to 0, so now, you may study or we can study a simple case that is the non-linear systems with quadratic nonlinearity in detail and then and briefly will study about a system with cubic nonlinearity.

(Refer Slide Time: 17:17)



So it has been shown by in this book of Nayfeh and Mook and by other researcher that the non-linear systems with quadratic damping with quadratic nonlinearity, so will behave as a linear systems, so let us find the equation motion in that case.

(Refer Slide Time: 17:20)



So let us take this non-linear systems with quadratic nonlinearity, so here we can write the equation two, we have taken a two degree of freedom system, where the equation is written $u_1 \ddot{u}_1 + \omega_1^2 u_1 - 2\epsilon \mu_1 \dot{u}_1 + \alpha_1 u_1^2 = u_2$ then $u_2 \ddot{u}_2 = 0$ or $u_2 \ddot{u}_2 + \omega_2^2 u_2 = 0$ minus $2\epsilon \mu_2 \dot{u}_2 + \alpha_2 u_1^2$.

So here this epsilon term is used as a book keeping parameter to make this damping term of the same order as that of the non-linear terms, so this book keeping parameter is generally taken very very less than 1 and by taking this non-linear term, so here we have a coupled non-linear term that is u_1 into u_2 , and here also in this second equation u_2 equation we have taken this u_1 square, so as we are taking this coupled equation though there is no external force acting on the systems, so one may think that there will be energy transfer between this u_1 and u_2 , so we can consider two different type of case, so in one case one is the resonant case, so this resonant case will come due to the internal resonance in the systems, and the second case you may consider non resonant case.


So one can find the solution of these type systems by using different numerical technique such as Runge kutta method, central difference method or one can use this Newmark beta method or Wilson theta method to find the non-linear response of the system but, to get the frequency amplitude relation, one can use this perturbation methods, so already we have studied the different types of perturbation method and today class we will take the method of multiple scale to solve this equation otherwise one may use this Lindstedt poincare technique, modified Lindstedt poincare technique or this harmonic balance method, and one can take also other different types of harmonic balance method for example, one can take intrinsic harmonic balance method or combination of harmonic balance method and method of multiple scales method of averaging, and different other types of averaging methods.

Also one can take to solve these type of equations and in this equation, so to use method of multiple scale, so let us use method of multiple scale to solve this equation, so to use method of multiple scale, so we have to take different time scales that is T_n , so let T_n , n equal to 0 1 2 3, so T_n equal to epsilon to the power n t so; that means, T_0 equal to T , T_1 equal to epsilon T and T_2 equal to epsilon square T .

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$$\begin{aligned}
 u_1 &= \varepsilon u_{11}(T_0, T_1) + \varepsilon^2 u_{12}(T_0, T_1) + \dots \\
 u_2 &= \varepsilon u_{21}(T_0, T_1) + \varepsilon^2 u_{22}(T_0, T_1) + \dots
 \end{aligned}$$

$$\left\{ \begin{aligned}
 \frac{d}{dt} &= D_0 + \varepsilon D_1 + \varepsilon^2 D_2 \\
 \frac{d^2}{dt^2} &= D_0^2 + 2\varepsilon D_0 D_1 \\
 &\quad + \varepsilon^2 (D_1^2 + 2D_0 D_2) + \dots
 \end{aligned} \right.$$

$$\underline{D_n = \frac{\partial}{\partial t_n}}$$


16

Also we can write this u_1 and u_2 using different time scales for example, we have written this u_1 equal to $\varepsilon u_{11}(T_0, T_1) + \varepsilon^2 u_{12}(T_0, T_1) + \dots$ and u_2 equal to $\varepsilon u_{21}(T_0, T_1) + \varepsilon^2 u_{22}(T_0, T_1) + \dots$, which is a function of T_0, T_1 plus $\varepsilon^2 u_{22}(T_0, T_1) + \dots$, so one can take also higher order terms in this case but, in this case we have taken only two terms or upto ε^2 now taking this d by dt , so as in the first equation, so we have $u_{\ddot{\quad}}$; that means, $d^2 u_1 / dt^2$ so, so you can write this d by dt equal to taking only single term, so this is $D_0 + \varepsilon D_1$ similarly, one can write this d^2 by dt^2 equal to $D_0^2 + 2\varepsilon D_0 D_1 + \varepsilon^2 (D_1^2 + 2D_0 D_2) + \dots$ plus higher order terms, one can take here also one can take $\varepsilon^2 D_2$, so taking this d by dt equal to D_0 , so where D_0 equal to or D_n is nothing but, $\partial / \partial t_n$.


(Refer Slide Time: 22:48)

System with quadratic nonlinearities

$$\left. \begin{aligned} \ddot{u}_1 + \omega_1^2 u_1 &= -2\varepsilon\mu_1 \dot{u}_1 + \alpha_1 u_1 u_2 \\ \ddot{u}_2 + \omega_2^2 u_2 &= -2\varepsilon\mu_2 \dot{u}_2 + \alpha_2 u_1^2 \end{aligned} \right\} \varepsilon \ll 1$$

Method of multiple scales $T_n = \varepsilon^t$


$n = 0, 1, 2, 3, \dots$



15

(Refer Slide Time: 22:57)

$$\begin{aligned} & D_0^2 + 2\varepsilon D_0 D_1 + \varepsilon^2 (D_1^2 + 2D_0 D_2) \{ \varepsilon u_{11} + \varepsilon^2 u_{12} \} \\ & + \omega_1^2 \{ \varepsilon u_{11} + \varepsilon^2 u_{12} \} = -2\varepsilon\mu_1 (D_0 + \varepsilon D_1) \\ & \quad (\varepsilon u_{11} + \varepsilon^2 u_{12}) + \\ & \quad \alpha_1 (\varepsilon u_{11} + \varepsilon^2 u_{12}) (\varepsilon u_{21} + \varepsilon^2 u_{22}) \end{aligned}$$

$$\begin{aligned} & \{ D_0^2 + 2\varepsilon D_0 D_1 + \varepsilon^2 (D_1^2 + 2D_0 D_2) \} \{ \varepsilon u_{21} + \varepsilon^2 u_{22} \} \\ & + \omega_2^2 (\varepsilon u_{21} + \varepsilon^2 u_{22}) = -2\varepsilon\mu_2 (D_0 + \varepsilon D_1) (\varepsilon u_{21} + \varepsilon^2 u_{22}) \\ & + \alpha_2 (\varepsilon u_{11} + \varepsilon^2 u_{12}) (\varepsilon u_{21} + \varepsilon^2 u_{22}) \end{aligned}$$


17

So del by del T n, so taking this equation this and this equation in equation original equation, so one can write D 0 square now expanding this thing, one can write this D 0 square plus 2 epsilon D 0 D 1 plus epsilon square D 1 square plus 2 D 0 D 2 into, so for u 1 we can substitute this epsilon u 1 1 plus epsilon square u 1 2 plus omega 1 square epsilon u 1 1 plus epsilon square u 1 2 equal to minus 2 epsilon mu 1 D 0 plus epsilon D 1 into epsilon u 1 1 plus epsilon square u 1 2 plus alpha 1 into epsilon u 1 1 plus epsilon square u 1 2 and into epsilon u 2 1 plus epsilon square u 2 2. So now, the second equation one can write D 0 square plus 2 epsilon D 0 D 1 plus epsilon square into D 1

square plus $2 D_0 D_2$, so this operated on u_{21} plus epsilon square u_{22} plus ω_2 square into epsilon u_{21} plus epsilon square u_{22} , so this will be equal to minus $2 \epsilon \mu_2$ into D_0 plus epsilon D_1 into epsilon u_{21} plus epsilon square u_{22} plus α_2 into epsilon u_{11} plus epsilon square u_{12} into epsilon u_{11} plus epsilon square u_{12} .


(Refer Slide Time: 25:24)

$$D_0^2 u_{12} + \omega_1^2 u_{12} = -2\omega_1 (A_1 + \mu_1 A_1) \exp(i\omega_1 T_0) + \alpha_1 \left\{ \begin{aligned} &A_1 A_2 \exp[i(\omega_1 + \omega_2) T_0] + \\ &A_2 \bar{A}_1 \exp[i(\omega_2 - \omega_1) T_0] \end{aligned} \right\} + cc$$

$$D_0^2 u_{22} + \omega_2^2 u_{22} = -2\omega_2 (A_2 + \mu_2 A_2) \exp(i\omega_2 T_0) + \alpha_2 [A_1^2 \exp(2i\omega_1 T_0) + A_1 \bar{A}_1] + cc$$

Non-resonant case

Resonant Case



18

Now we can collect the term of the order of epsilon and epsilon square and we can write these equations, so by taking.


(Refer Slide Time: 25:40)

Order ϵ

$$\left. \begin{aligned} D_0^2 u_{10} + \omega_1^2 u_{10} &= 0 \\ D_0^2 u_{20} + \omega_2^2 u_{20} &= 0 \end{aligned} \right\}$$

Order ϵ^2

$$\left. \begin{aligned} D_0^2 u_{12} + \omega_1^2 u_{12} &= -2D_0(D_1 u_{10} + \mu_1 u_{10}) + \alpha_1 u_{10} u_{20} \\ D_0^2 u_{22} + \omega_2^2 u_{22} &= -2D_0(D_1 u_{20} + \mu_2 u_{20}) + \alpha_2 u_{10}^2 \end{aligned} \right\}$$

$$\left. \begin{aligned} u_{11} &= A_1(T_1) \exp(i\omega_1 T_0) + \bar{A}_1 \exp(-i\omega_1 T_0) \\ u_{21} &= A_2(T_1) \exp(i\omega_2 T_0) + \bar{A}_2 \exp(-i\omega_2 T_0) \end{aligned} \right\}$$


22

Let us take so order or epsilon, so you can write this order of epsilon, so the equation of the order epsilon equal to $D_0^2 u_{11} + \omega_1^2 u_{11} = 0$ and $D_0^2 u_{21} + \omega_2^2 u_{21} = 0$. Similarly, order of epsilon square, so this becomes $D_0^2 u_{12} + \omega_1^2 u_{12} = -2D_0 D_1 u_{11} + \mu_1 u_{11} + \alpha_1 u_{11} u_{21}$. Similarly, $D_0^2 u_{22} + \omega_2^2 u_{22} = -2D_0 D_1 u_{21} + \mu_2 u_{21} + \alpha_2 u_{11}^2$. So, it will be equal to minus 2 D_0 into $D_1 u_{21}$ plus $\mu_2 u_{21}$ plus $\alpha_2 u_{11}^2$, so in this order of epsilon, now one can write the solution of u_{11} from this equation equal to A_1 , so A_1 should not be a function of T_0 , so A_1 should be a function $T_1 T_2$ or higher order terms, so u_{11} equal to $A_1 e^{i\omega_1 T_0}$ plus its complex conjugate that is $\bar{A}_1 e^{-i\omega_1 T_0}$ and u_{21} equal to $A_2 e^{i\omega_2 T_0}$ plus $\bar{A}_2 e^{-i\omega_2 T_0}$. A_2 also be a function of $T_1 T_2$, and should not be a function of T_0 e to the power $i\omega_2 T_0$ plus $\bar{A}_2 e^{-i\omega_2 T_0}$. Now substituting these two equation that is u_{11} and u_{21} these two equations, so one can get the equation in this form.

(Refer Slide Time: 27:53)

The slide contains the following equations and text:

$$D_0^2 u_{12} + \omega_1^2 u_{12} = -2\omega_1(A_1 + \mu_1 A_1) \exp(i\omega_1 T_0) + \alpha_1 \left\{ \begin{array}{l} A_1 A_2 \exp[i(\omega_1 + \omega_2)T_0] + \\ A_2 \bar{A}_1 \exp[i(\omega_2 - \omega_1)T_0] \end{array} \right\} + cc$$

$$D_0^2 u_{22} + \omega_2^2 u_{22} = -2\omega_2(A_2 + \mu_2 A_2) \exp(i\omega_2 T_0) + \alpha_2 [A_1^2 \exp(2i\omega_1 T_0) + A_1 \bar{A}_1] + cc$$

Non-resonant case ✓

Resonant Case ✓

$\omega_2 \approx 2\omega_1$

NPTTEL 18

So $D_0^2 u_{12} + \omega_1^2 u_{12} = -2\omega_1 A_1 + \mu_1 A_1 e^{i\omega_1 T_0} + \alpha_1 A_1 A_2 e^{i(\omega_1 + \omega_2)T_0} + A_2 \bar{A}_1 e^{i(\omega_2 - \omega_1)T_0}$ plus its complex conjugate, and the second equation reduces to $D_0^2 u_{22} + \omega_2^2 u_{22} = -2\omega_2 A_2 + \mu_2 A_2 e^{i\omega_2 T_0} + \alpha_2 A_1^2 e^{2i\omega_1 T_0} + A_1 \bar{A}_1$ plus its

complex conjugate, so here one may see in the right hand side, so we can consider two case one is non resonant case.

So as there is no external force acting on the systems, so if we take the frequency, so we can consider this non resonant case, so if we are not considering this omega 2 omega 2, so if omega 2 is nearly equal to 2 omega 1 then this term that is omega 2 minus omega 1, so this becomes nearly equal to omega 1, and this will be a secular term in addition to this term, so already this term is the secular term that is minus 2 omega 1 A 1 dash plus mu 1 A 1, and this term will also be a secular term, so if you are considering omega 2 nearly equal to twice omega 1, so if you are considering this is away from 2 omega 1 then the case will be a non resonant case; that means, there will be no resonance in the system, and if we consider this omega 2 nearly equal to 2 omega 1, so there will be internal energy flow from second system to the first system, and there will be resonant in the system.

So we can consider two cases in one case we will consider omega 2 not nearly equal to 2 omega 1 and in the second case will consider the resonant case in which we will take this omega 2 nearly equal to twice omega 1, so if we consider this non resonant case first, so then we can write the secular term, so this is the only secular term, so we can write the secular term minus 2 omega 1 A 1 dash plus mu 1 A 1 and as the solution is bounded, so we can put this term equal to 0 so; that means, this minus 2 omega 1 A 1 dash plus mu 1 A 1 equal to 0.

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Non-resonant case


$$\begin{aligned} \dot{A}_1 + \mu_1 A_1 &= 0 \\ \dot{A}_2 + \mu_2 A_2 &= 0 \end{aligned}$$

$\mu_1 = \epsilon \omega_1 + \epsilon^2 \omega_{11}$
 $\mu_2 = \epsilon \omega_2 + \epsilon^2 \omega_{21}$

$$\underline{A_1 = a_1 \exp(-\mu_1 t)} \quad \underline{A_2 = a_2 \exp(-\mu_2 t)}$$

$u_1 = \epsilon \exp(-\epsilon \mu_1 t) [a_1 \exp(i\omega_1 t) + cc] + O(\epsilon^2)$
 $u_2 = \epsilon \exp(-\epsilon \mu_2 t) [a_2 \exp(i\omega_2 t) + cc] + O(\epsilon^2)$

Steady state response
 $\left. \begin{aligned} u_1 &= 0 \\ u_2 &= 0 \end{aligned} \right\} t \rightarrow \infty$



19

So we can substitute that thing and in the second equation also, this is the secular term in the second equation this term is the secular term and as this is the coefficient of $i\omega^2$, so as here we have this ω^2 square, so this is the secular term, this will lead to infinity as t tends to infinity but, as the solution is bounded, so we should kill this term, so we have two terms to kill or two terms to eliminate, so first term is $A_1 \dot{a} + \mu_1 A_1 = 0$, and second term is $A_2 \ddot{a} + \mu_2 A_2 = 0$. So now, the solution of this $A_1 \dot{a}$ that means, $d a_1$ by $d t + \mu_1 A_1 = 0$. Similarly, $d A_2$ by $d t^2 + \mu_2 A_2 = 0$, so the solution $A_1 = A_1 e^{-\mu_1 t}$, and similarly, $A_2 = A_2 e^{-\mu_2 t}$ and the solution u_1 now, so for this non resonant case one can write this u_1 as we have written this $u_1 = \epsilon u_{11} + \epsilon^2 u_{12}$.

Similarly, $u_2 = \epsilon u_{21} + \epsilon^2 u_{22}$, so by substituting this only, we are taking the first order then we have the, we are not taking the second order that is order of ϵ^2 taking only, the first order you can write this $u_1 = \epsilon e^{-\mu_1 t}$, so as $\epsilon t = t$, so this $-\mu_1 t$ can be written as $-\epsilon \mu_1 t$ into $a_1 e^{i\omega_1 t}$ plus its complex conjugate. Similarly, this $u_2 = \epsilon e^{-\mu_2 t}$ into $a_2 e^{i\omega_2 t}$ plus its complex conjugate, so one can see that as t tends to infinity, this $u_1 = 0$ and $u_2 = 0$ as t tends to infinity that is steady state condition, so in case of non resonant case, so the steady state response becomes this u_1 and u_2 becomes 0 as t tends to 0 and let us see the non resonant case.

(Refer Slide Time: 33:51)

Resonant Case

$$\omega_2 = 2\omega_1 + \varepsilon\alpha$$

$$2\omega_1 T_0 = \omega_2 T_0 - \varepsilon\sigma T_0 = \omega_2 T_0 - \sigma T_1$$


$$(\omega_2 - \omega_1) T_0 = \omega_1 T_0 + \varepsilon\alpha T_0 = \omega_1 T_0 + \alpha T_1$$

$$-2i\omega_1 (A_1 + \mu_1 A_1) + \alpha_1 A_2 \bar{A}_1 \exp(i\sigma T_1) = 0$$

$$-2i\omega_2 (A_2 + \mu_2 A_2) + \alpha_2 A_1^2 \exp(-i\sigma T_1) = 0$$

$$A_1 = \frac{1}{2} \alpha \exp(i\beta_1), A_2 = \frac{1}{2} \alpha \exp(i\beta_2)$$

$\omega_2 \simeq 2\omega_1$
 $\alpha \rightarrow$ detuning
 Parameters
 $\omega_1 = 1$
 $\omega_2 \simeq 2$
 $1.9 \leq \omega_2 \leq 2.1$
 $\varepsilon\sigma = 0.1$



20

So, in resonant case, we are taking this omega 2 nearly equal to 2 omega 1, so if we are taking this omega 2 nearly equal to 2 omega 1, so we can use one detuning parameter to express the nearness of omega 2 to this twice omega 1, so two express that thing we are taking a detuning parameter alpha, so alpha is the detuning parameter, detuning parameter which shows the nearness of this twice omega 1 to omega 2, so it can take plus minus value for example, if we are taking this omega 1 equal to 1, so this omega 2 should be nearly equal to 2; that means, we can take this value this omega 2.

So, for example, we can vary this omega 2 from 1.9 to, so taking 10 percents taking, so you can vary this omega 2 to 2.1, so you can vary this omega 2, so from 1.9 less than equal to omega 2 less than equal to 2.1, so; that means, it can take this value, so if you are taking 1.9 then this epsilon sigma equal to 0.1 taking different value of epsilon, so you can find sigma so, this sigma is the detuning parameter. So, in this way you can find taking different value of omega 2, we can find sigma or for different value of sigma we can find the omega 2. Now, taking this detuning parameter, omega 2 equal to 2 omega 1 plus epsilon sigma.


(Refer Slide Time: 35:50)

$$D_0^2 u_{12} + \omega_1^2 u_{12} = -2\omega_1 (A_1 + \mu_1 A_1) \exp(i\omega_1 T_0) + \alpha_1 \left\{ \begin{array}{l} A_1 A_2 \exp[i(\omega_1 + \omega_2)T_0] + \\ A_2 \bar{A}_1 \exp[i(\omega_2 - \omega_1)T_0] \end{array} \right\} + cc$$

$$D_0^2 u_{22} + \omega_2^2 u_{22} = -2\omega_2 (A_2 + \mu_2 A_2) \exp(i\omega_2 T_0) + \alpha_2 [A_1^2 \exp(2i\omega_1 T_0) + A_1 \bar{A}_1] + cc$$

Non-resonant case ✓
} $\omega_2 \approx 2\omega_1$

Resonant Case ✓



18

So we can write this $2\omega_1 T_0$, so as this is present in our equation this equation ω_2 minus $\omega_1 T_0$, so you can express this in terms of this $i\omega_1$.

(Refer Slide Time: 36:01)

Resonant Case

$$\omega_2 = 2\omega_1 + \varepsilon\alpha$$

$$2\omega_1 T_0 = \omega_2 T_0 - \varepsilon\sigma T_0 = \omega_2 T_0 - \sigma T_1$$


$$(\omega_2 - \omega_1) T_0 = \omega_1 T_0 + \varepsilon\alpha T_0 = \omega_1 T_0 + \alpha T_1$$

$$-2i\omega_1 (A_1 + \mu_1 A_1) + \alpha_1 A_2 \bar{A}_1 \exp(i\sigma T_1) = 0$$

$$-2i\omega_2 (A_2 + \mu_2 A_2) + \alpha_2 A_1^2 \exp(-i\sigma T_1) = 0$$

$$A_1 = \frac{1}{2} a \exp(i\beta), A_2 = \frac{1}{2} a \exp(i\beta)$$

$\omega_2 \approx 2\omega_1$
 $\alpha \rightarrow \text{detuning Parameters}$
 $\omega_1 = 1$
 $\omega_2 \approx 2$
 $1.9 \leq \omega_2 \leq 2.1$
 $\varepsilon = 0.1$



20

So that we can have a mixed secular or near secular term which we have to eliminate along with the secular terms to find the equation required reduced equation, so we can write this $2\omega_1 T_0$ equal to $\omega_2 T_0$, so this is equal to $\omega_2 T_0$ by taking this thing to left hand side that is ω_2 minus $\varepsilon\alpha$ equal to $2\omega_1$, so $2\omega_1 T_0$ will be equal to $\omega_2 T_0$ minus $\varepsilon\sigma T_0$, so we can write this

$\omega_2 T_0$ minus this ϵT_0 equal to T_1 , so you can write this is equal to ϵ σT_1 , so this $2\omega_1 T_0$ equal to $\omega_2 T_0$ minus σT_1 .

So we can write this ω_2 minus $\omega_1 T_0$ equal to $\omega_1 T_0$ plus ϵ αT_0 , so we can write this, so $\omega_2 T_0$ minus σT_1 , so minus ω_1 again we are putting, so we can write this is equal to $\omega_1 T_0$ plus ϵ σT_0 or this is equal to $\omega_1 T_0$ plus αT_1 , so in this expression if you are substituting that thing that is $2\omega_2$ minus $\omega_1 T_0$ equal to $\omega_1 T_0$ into αT_1 , so e to the power of ω_2 minus $\omega_1 T_0$ will be equal to e to the power $i\omega_1 T_0$ plus e to the power $i\alpha T_1$. So this term this is the additional term which shows the nearness of this $2\omega_1$ to ω_2 , so gives rise to this mixed secular term or nearly secular term now we have to eliminate this secular and mixed secular terms as these terms leads to infinite response while the actual system response is finite or bounded, so we have to eliminate these terms.

(Refer Slide Time: 38:14)

$$D_0^2 u_{12} + \omega_1^2 u_{12} = -2\omega_1(A_1' + \mu_1 A_1) \exp(i\omega_1 T_0) + \alpha_1 \left\{ A_1 A_2 \exp[i(\omega_1 + \omega_2) T_0] + A_2 \bar{A}_1 \exp[i(\omega_2 - \omega_1) T_0] \right\} + cc$$

$$D_0^2 u_{22} + \omega_2^2 u_{22} = -2\omega_2(A_2' + \mu_2 A_2) \exp(i\omega_2 T_0) + \alpha_2 [A_1^2 \exp(2i\omega_1 T_0) + A_1 \bar{A}_1] + cc$$

Non-resonant case ✓

Resonant Case ✓

$\omega_2 \approx 2\omega_1$

So in the first equation, so from this first equation, so we can write this minus $2\omega_1$ A_1 dash plus $\mu_1 A_1$, this should be equal to 0 and plus, so from this equation also we have this $A_2 A_1$ bar e to the power.

(Refer Slide Time: 38:32)

Resonant Case


$$\omega_2 = 2\omega_1 + \epsilon\alpha$$

$$2\omega_1 T_0 = \omega_2 T_0 - \epsilon\sigma T_0 = \omega_2 T_0 - \sigma T_1$$

$$(\omega_2 - \omega_1) T_0 = \omega_1 T_0 + \epsilon\sigma T_0 = \omega_1 T_0 + \sigma T_1$$

$$\left. \begin{aligned} -2i\omega_1 (A_1 + \mu_1 A_1) + \alpha_1 A_2 \bar{A}_1 \exp(i\sigma T_1) &= 0 \\ -2i\omega_2 (A_2 + \mu_2 A_2) + \alpha_2 A_1^2 \exp(-i\sigma T_1) &= 0 \end{aligned} \right\}$$


$\omega_2 \approx 2\omega_1$
 $\alpha \rightarrow$ detuning
 Parameters
 $\omega_1 = 1$
 $\omega_2 \approx 2$
 $1.9 < \omega_2 < 2.1$
 $\epsilon\sigma = 0.1$

$$A_1 = \frac{1}{2} a_1 \exp(i\beta_1), \quad A_2 = \frac{1}{2} a_2 \exp(i\beta_2)$$


20

So we have this this additional term, $\alpha_1 A_2 \bar{A}_1$ this α so, for the first equation we can substitute, so this is α_1 , so this is not α this is σT_1 this is σT_1 , so this becomes $-2i\omega_1 A_1 + \mu_1 A_1 + \alpha_1 A_2 \bar{A}_1 e^{i\sigma T_1}$. Similarly, the second equation becomes $-2i\omega_2 A_2 + \mu_2 A_2 + \alpha_2 A_1^2 e^{-i\sigma T_1} = 0$, so from these two equation now we can write, so from these two equation now writing A_1 and A_2 in polar form, so $A_1 = \frac{1}{2} a_1 e^{i\beta_1}$. Similarly, $A_2 = \frac{1}{2} a_2 e^{i\beta_2}$.

(Refer Slide Time: 39:42)

$$\left. \begin{aligned} a_1' &= -\mu_1 a_1 + \frac{\alpha_1}{4\omega_1} a_1 a_2 \sin \gamma \\ a_2' &= -\mu_2 a_2 + \frac{\alpha_2}{4\omega_2} a_1^2 \sin \gamma \\ a_1 \beta_1' &= -\frac{\alpha_1}{4\omega_1} a_1 a_2 \cos \gamma \\ a_2 \beta_2' &= -\frac{\alpha_2}{4\omega_2} a_1^2 \cos \gamma \end{aligned} \right\} \underline{\underline{\gamma = \beta_2 - 2\beta_1 + \sigma T_1}}$$


21

So substituting these two equations, so substituting this polar form in these two equations and then separating the real and imaginary parts, we are getting a set of reduced equations. So, in case of the single degree of freedom systems, we have got two equations and in this two degree of freedom systems, we are getting four equations, so two for the amplitude and two for the phase. So, we can write this as $a_1^2 = \frac{\mu_1}{4\omega_1^2} \sin^2 \gamma$, so where γ is $\beta_2 - \beta_1 + \sigma t$ and similarly, a_2^2 can be written equal to $\frac{\mu_2}{4\omega_2^2} \sin^2 \gamma$.

So, now we to eliminate the secular term, so one can eliminate the secular term to get this expression, so by eliminating the secular term, so one can find the solution of the equation, so in this case one can observe that by eliminating this term.

(Refer Slide Time: 41:05)

$$a_1^2 + \frac{\mu_2 \omega_2 \alpha_1}{\mu_1 \omega_1 \alpha_2} a_2^2 = 0$$

$$\cot \gamma = -\frac{\sigma}{\mu_1 + 2\mu_2} \quad \alpha_2 \sin \gamma = \frac{4\mu_1 \mu_2}{\alpha_1}$$

$$a_1^2 = -\left(\frac{\mu_2 \omega_2 \alpha_1}{\mu_1 \omega_1 \alpha_2}\right) a_2^2$$

$$a_1 = c a_2$$

$$\omega_2 = 2\omega_1$$

So, the equation will reduce to this form, that is one can write as $a_1^2 + \mu_2 \omega_2 \alpha_1 \mu_1 \omega_1 \alpha_2 a_2^2 = 0$ and one can write this $\cot \gamma = -\frac{\sigma}{\mu_2 + 2\mu_1}$, and this $\alpha_2 \sin \gamma = \frac{4\mu_1 \mu_2}{\alpha_1}$. So, in this case one can see that this equation does not depend on the phase that is γ , and here a_1 and a_2 are related, so $a_1^2 + \mu_2 \omega_2 \alpha_1 \mu_1 \omega_1 \alpha_2 a_2^2 = 0$ as μ_2 and ω_2 , so they are positive numbers, so either this α_1 or α_2 to be negative, so that or of opposite sign this α_1 and α_2 should be of opposite

sign, so that one can have this real value of a_1 and a_2 otherwise, so one can if they are of the same sign, so one can write this a_1^2 equal to minus, so minus of $\mu_2 \omega_2 \alpha_1$ by $\mu_1 \omega_1 \alpha_2$ into a_2^2 square, so this term becomes negative, so one can get this imaginary value, so it will not yield any solution, so if α_1 and α_2 are of the same sign.

So, when this α_1 and α_2 are of opposite sign, so this term becomes positive and one can get this a_1 in terms of a_2 . So, let, let us take this a_1 equal to $c_1 a_2$; that means, there is some energy transverse, so though there is no external force, so here the energy from a_1 or amplitude from a_1 is transferred to a_2 and vice versa. So, in this case one can first obtain this γ term from this equation by solving the σ by so for different value of σ or different value of frequency, so one can get this value of γ then by substituting this value of γ in a_1 and a_2 equation, so one can get this a_1 and a_2 , and this a_1 and a_2 are related by this expression, so in this case we have seen the though there is no external force due to this internal resonance that is when ω_2 nearly equal to ω_1 , so this a_1 and a_2 are related; that means; so there is some energy transfer from the first body to the second body.


The equation what we have represented can be or can be shown by a physical system of this to mass system, so you have a two mass systems; this is mass 1 this is mass 2. So, let u_1 is the displacement of this mass and u_2 is the displacement of the second mass, then if we are taking, so this spring, so if this spring's are taken in such way that this spring and dampers are taken in such way that, this $k_1 k_2$ this is k_3 taken such way that, this ω_2 nearly equal to $2 \omega_1$, so then in that case we can obtain this, so if we start vibrating this first mass, so it will transfer energy to the second mass and in turn second mass will transfer energy to the first mass and there will be coupled motion. So, unlike the uncoupled motion we have seen in case of or we have predicted in case of the simple pendulum, so in this case the motion will be coupled as the first mass will try to move the second mass and in turn this second mass tries to move the first mass, so this way when one can study the system with quadratic nonlinearity in the presence of re internal resonance, so one can consider the secular and mixed secular terms and one can obtain the response of the system.

(Refer Slide Time: 46:03)

System with Cubic nonlinearities

$$\begin{cases} \ddot{u}_1 + \omega_1^2 u_1 = -2\hat{\mu}_1 \dot{u}_1 + \alpha_1 u_1^3 + \alpha_2 u_1^2 u_2 + \alpha_3 u_1 u_2^2 + \alpha_4 u_2^3 \\ \ddot{u}_2 + \omega_2^2 u_2 = -2\hat{\mu}_2 \dot{u}_2 + \alpha_1 u_1^3 + \alpha_2 u_1^2 u_2 + \alpha_3 u_1 u_2^2 + \alpha_4 u_2^3 \end{cases}$$

$$\begin{cases} \hat{\mu}_1 = \epsilon \mu_1 \\ \hat{\mu}_2 = \epsilon \mu_2 \end{cases}$$

$$\begin{cases} \mu_1 = \epsilon u_{11}(T_0, T_1) + \epsilon^2 u_{13}(T_0, T_2) + \dots \\ \mu_2 = \epsilon u_{21}(T_0, T_1) + \epsilon^2 u_{23}(T_0, T_2) + \dots \end{cases}$$


Similarly, one can study the systems with cubic nonlinearity, so here one can take, so let us take the system in this way that is u_1 double dot plus ω_1 square u_1 equal to minus $2\mu_1 u_1$ dot plus $\alpha_1 u_1$ cube plus $\alpha_2 u_1$ square u_2 plus $\alpha_3 u_1 u_2$ square plus $\alpha_4 u_2$ cube, so here we have taken, so this is a cubic non-linear term this is also cubic, where we have taken u_1 square u_2 and this is $u_1 u_1 u_2$ square and this is u_2 cube, so due to this presence of this coupled and non-linear term, so these terms give rise to coupled terms and these are the non-linear terms, this u_1 cube is the non-linear term present here, so this is coupled non-linear to the coupled terms coupled and non-linear because this u_2 also affect this u_1 behaviour of this u_1 .

Similarly, in the second equation, we have this coupled equation that is $\alpha_1 u_1$ cube plus $\alpha_2 u_1$ square u_2 plus $\alpha_3 u_1 u_2$ square plus $\alpha_4 u_2$ cube, so we have all these coupled terms, so to make this non-linear and this damping term of the same order, so we can substitute this μ_1 equal to $\epsilon \mu_1$, and this μ_2 equal to $\epsilon \mu_2$, and proceeding in the same way as we did in case of the quadratic nonlinearity, here also we can find the equations or we can let us take this equation in this form that is u_1 equal to we can take this u_1 equal to ϵu_1 , which is a function of $T_0, T_1, T_2, T_0, T_1, T_2$ plus ϵ^2 , so in this case we can take as we are taking this cubic nonlinearity. We may not take the ϵ^2 term, directly let us take this ϵu_1 plus $\epsilon^2 u_1$ cube plus $\epsilon^2 u_1$ square u_2 plus $\epsilon^2 u_1 u_2$ square plus $\epsilon^2 u_2$ cube, which is a function of T_0, T_2 , so this, so you can eliminate this T_1 . Similarly, we can write this ϵu_2 equal to ϵu_2 plus $\epsilon^2 u_2$ plus $\epsilon^2 u_2$ cube.

2 3 T 0, T 2, so by substituting this equation in this original equation, so we can eliminate or we can find the terms with epsilon to the power 0 epsilon to the power 1 epsilon to the power 3, so in the equation epsilon to the power 1.

(Refer Slide Time: 49:11)

$$\left. \begin{aligned} D_0^2 u_{11} + \omega_1^2 u_{11} &= 0 \\ D_0^2 u_{21} + \omega_2^2 u_{21} &= 0 \end{aligned} \right\}$$

$$D_0^2 u_{23} + \omega_2^2 u_{23} = -2D_0(D_2 u_{21} + \mu_2 u_{21}) + \alpha$$

So, we can write this equation to be $D_0^2 u_{11} + \omega_1^2 u_{11} = 0$ and similarly, we can write other equation $D_0^2 u_{21} + \omega_2^2 u_{21} = 0$, so it will be $2\omega_2^2 u_{21} = 0$. Now of the order of epsilon to the power 3, we can write $D_0 D_0^2 u_{23} + \omega_2^2 u_{23} = -2D_0(D_2 u_{21} + \mu_2 u_{21}) + \alpha$.

(Refer Slide Time: 50:16)

System with Cubic nonlinearities

$$\left\{ \begin{aligned} \ddot{u}_1 + \omega_1^2 u_1 &= -2\hat{\mu}_1 \dot{u}_1 + \alpha_1 u_1^3 + \alpha_2 u_1^2 u_2 + \alpha_3 u_1 u_2^2 + \alpha_4 u_2^3 \\ \ddot{u}_2 + \omega_2^2 u_2 &= -2\hat{\mu}_2 \dot{u}_2 + \alpha_5 u_2^3 + \alpha_6 u_1^2 u_2 + \alpha_7 u_1 u_2^2 + \alpha_8 u_1^3 \end{aligned} \right.$$

$$\left. \begin{aligned} \hat{\mu}_1 &= \epsilon \mu_1 \\ \hat{\mu}_2 &= \epsilon \mu_2 \end{aligned} \right\}$$

$$\left\{ \begin{aligned} \mu_1 &= \epsilon u_{11}(T_0, T_1) + \epsilon^2 u_{13}(T_0, T_2) + \dots \\ \mu_2 &= \epsilon u_{21}(T_0, T_1) + \epsilon^2 u_{23}(T_0, T_2) + \dots \end{aligned} \right.$$

So, one can go on writing this term that is we have, so by substituting these terms we can write, so let us modify this equation slightly to, so $u_1 \ddot{u}_1 + \omega_1^2 u_1 = \epsilon u_1 \dot{u}_1 + \alpha_1 u_1^3 + \alpha_2 u_1^2 u_2 + \alpha_3 u_1^2 u_2^2 + \alpha_4 u_2^3$ and the second equation let us write, so this is equal to this coefficient $\alpha_5 u_1^3$, $\alpha_5 u_1^3$, this is $\alpha_6 u_1^2 u_2$, α_7 , so this is $\alpha_7 u_1 u_2^2$ and $\alpha_8 u_2^3$. Now taking this u_1 equal to $\epsilon u_1 + \epsilon^3 u_1^3$ and u_2 equal to $\epsilon u_2 + \epsilon^3 u_2^3$.

(Refer Slide Time: 51:22)


So, we can write this equation, so order of epsilon cube, we can write the equation in this form, so $D_0^2 u_2 + \omega_2^2 u_2 = \epsilon^3 [-2D_0(D_2 u_1 + \mu_1 u_1) + \alpha_1 u_1^3 + \alpha_2 u_1^2 u_2 + \alpha_3 u_1 u_2^2 + \alpha_4 u_2^3]$, so this equation we can now order of epsilon cube, so you can write this is equal to $D_0^2 u_1 + \omega_1^2 u_1 = \epsilon^3 [-2D_0(D_2 u_1 + \mu_1 u_1) + \alpha_1 u_1^3 + \alpha_2 u_1^2 u_2 + \alpha_3 u_1 u_2^2 + \alpha_4 u_2^3]$. Similarly, the second equation can be written in this form $D_0^2 u_2 + \omega_2^2 u_2 = \epsilon^3 [-2D_0(D_2 u_2 + \mu_2 u_2) + \alpha_5 u_1^3 + \alpha_6 u_1^2 u_2 + \alpha_7 u_1 u_2^2 + \alpha_8 u_2^3]$.

(Refer Slide Time: 53:31)

$$D_0^2 u_{13} + \omega_1^2 u_{13} = \left[-2i\omega_1 (\bar{A}_1 + \mu_1 A_1) + 3\alpha_1 A_1^2 \bar{A}_1 + 2\alpha_1 A_2 \bar{A}_2 A_1 \right] \exp(i\omega_1 T_0) + (2\alpha_2 A_1 \bar{A}_1 + 3\alpha_2 A_2 \bar{A}_2) A_2 \exp(i\omega_2 T_0) + \alpha_1 A_1^3 \exp(3i\omega_1 T_0) + \alpha_1 A_2^3 \exp(3i\omega_2 T_0) + \alpha_2 A_1^2 A_2 \exp[i(2\omega_1 + \omega_2) T_0] + \alpha_2 \bar{A}_1^2 A_2 \exp[i(\omega_2 - 2\omega_1) T_0] + \alpha_1 A_1 A_2^2 \exp[i(\omega_1 + 2\omega_2) T_0] + \alpha_1 A_1 \bar{A}_2^2 \exp[i(\omega_1 - 2\omega_2) T_0] + cc$$

$$D_0^2 u_{23} + \omega_2^2 u_{23} = \left[-2i\omega_2 (\bar{A}_2 + \mu_2 A_2) + 3\alpha_2 A_2^2 \bar{A}_2 + 2\alpha_2 A_1 \bar{A}_1 A_2 \right] \exp(i\omega_2 T_0) + (2\alpha_1 A_2 \bar{A}_2 + 3\alpha_1 A_1 \bar{A}_1) A_1 \exp(i\omega_1 T_0) + \alpha_2 A_1^3 \exp(3i\omega_1 T_0) + \alpha_2 A_2^3 \exp(3i\omega_2 T_0) + \alpha_1 A_1^2 A_2 \exp[i(2\omega_1 + \omega_2) T_0] + \alpha_1 \bar{A}_1^2 A_2 \exp[i(\omega_2 - 2\omega_1) T_0] + \alpha_2 A_1 A_2^2 \exp[i(\omega_1 + 2\omega_2) T_0] + \alpha_2 A_1 \bar{A}_2^2 \exp[i(\omega_1 - 2\omega_2) T_0] + cc$$


$$2\omega_2 - \omega_1 = \omega_1$$


27

Now the solution of this equation can be written in this form, a $1 e$ to the power $i \omega_1 t$ and u_{21} equal to $a_2 e$ to the power $i \omega_2 t$ and substituting this equation in this equation one can get the equation in this form, so now to get the reduced equation one can take the resonant and non resonant conditions, so here one can see, one can have resonant condition, so when these ω_2 , so this term may will leads to resonant condition that is if we have this $\omega_2 - 2\omega_1$ equal to ω_1 , so we will have resonance; that means, if ω_2 is nearly equal to $3\omega_1$, so we will have this resonance similarly, here $\omega_1 - 2\omega_2$ same as this one $\omega_1 - 2\omega_2$ equal to 0, so here we can put this $\omega_1 - 2\omega_2$ or its complex conjugate term will gives us $2\omega_2 - \omega_1$ equal to ω_1 , so $2\omega_2 - \omega_1$ equal to ω_1 or $2\omega_2$ equal to $2\omega_1$.

(Refer Slide Time: 54:49)

$$\begin{aligned}
 D_0^2 u_{13} + \omega_1^2 u_{13} = & \left[-2i\omega_1 (\dot{A}_1 + \mu_1 A_1) + 3\alpha_1 A_1^2 \bar{A}_1 + 2\alpha_1 A_2 \bar{A}_2 A_1 \right] \\
 & \exp(i\omega_1 T_0) + (2\alpha_2 A_1 \bar{A}_1 + 3\alpha_2 A_2 \bar{A}_2) A_2 \exp(i\omega_2 T_0) + \\
 & \alpha_1 A_1^3 \exp(3i\omega_1 T_0) + \alpha_1 A_2^3 \exp(3i\omega_2 T_0) \\
 & \alpha_2 A_1^2 A_2 \exp[i(2\omega_1 + \omega_2) T_0] + \alpha_2 \bar{A}_1^2 A_2 \exp[i(\omega_2 - 2\omega_1) T_0] + \\
 & \alpha_1 A_1 A_2^2 \exp[i(\omega_1 + 2\omega_2) T_0] + \alpha_1 A_1 \bar{A}_2^2 \exp[i(\omega_1 - 2\omega_2) T_0] + cc
 \end{aligned}$$

$$\begin{aligned}
 D_0^2 u_{23} + \omega_2^2 u_{23} = & \left[-2i\omega_2 (\dot{A}_2 + \mu_2 A_2) + 3\alpha_2 A_2^2 \bar{A}_2 + 2\alpha_2 A_1 \bar{A}_1 A_2 \right] \\
 & \exp(i\omega_2 T_0) + (2\alpha_1 A_1 \bar{A}_1 + 3\alpha_1 A_2 \bar{A}_2) A_1 \exp(i\omega_1 T_0) \\
 & + \alpha_2 A_1^3 \exp(3i\omega_1 T_0) + \alpha_2 A_2^3 \exp(3i\omega_2 T_0) \\
 & + \alpha_1 A_1^2 A_2 \exp[i(2\omega_1 + \omega_2) T_0] + \alpha_1 \bar{A}_1^2 A_2 \\
 & \exp[i(\omega_2 - 2\omega_1) T_0] + \alpha_2 A_1 A_2^2 \exp[i(\omega_1 + 2\omega_2) T_0] + \\
 & \alpha_2 A_1 \bar{A}_2^2 \exp[i(\omega_1 - 2\omega_2) T_0] + cc
 \end{aligned}$$


27

So, that means, when omega 2 equal to omega 1 also, we have resonant condition and when this omega 2 equal to 3 omega 1 also, we have resonance condition, so this is left as an exercise problem to find this response of the system for the resonant and non resonant condition, and one can see that in case of the non resonant condition the response rise down in steady state condition and in case of this resonant condition there will be energy transfer between different modes, so in the next class we will study about the force vibration of single degree of freedom system.

Thank you.