


Non-Linear Vibration
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Module - 4
Stability and Bifurcation Analysis
of Nonlinear Responses
Lecture - 2
Stability and Bifurcation Analysis of
Nonlinear Fixed Point Responses

Welcome to today's class of linear vibration. So, in today's class we are going to discuss about the stability and bifurcation analysis of non-linear fixed point responses.

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4 Stability and Bifurcation Analysis	21	Stability and Bifurcation of fixed point response, static bifurcation:
	22	pitch fork, saddle-node and trans-critical bifurcation, dynamic
	23	bifurcation: Hopf bifurcation
	24	Stability and Bifurcation of periodic response, monodromy
	25	matrix, Lyapunov exponents
	26	Different routes to chaotic response (period doubling, intermittency, torus break down, attractor merging etc.), crisis



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Particularly, we will discuss about the equilibrium points and their stability and about the static bifurcation points. So, in static bifurcation points we will discuss about these saddle-node bifurcation, pitch fork and trans-critical bifurcation. Next class will study about these Hopf bifurcation points.

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Different Types of Nonlinear Equation

Duffing Equation
 $\ddot{x} + \omega_n^2 x + 2\zeta\omega_n \dot{x} + \alpha x^3 = \varepsilon f \cos \Omega t$

Van der Pol's Equation $\ddot{x} + x = \mu(1 - x^2)\dot{x}$

Hill's Equation $\ddot{x} + p(t)x = 0$

Mathieu's Equation $\ddot{x} + (\delta + 2\varepsilon \cos 2t)x = 0$

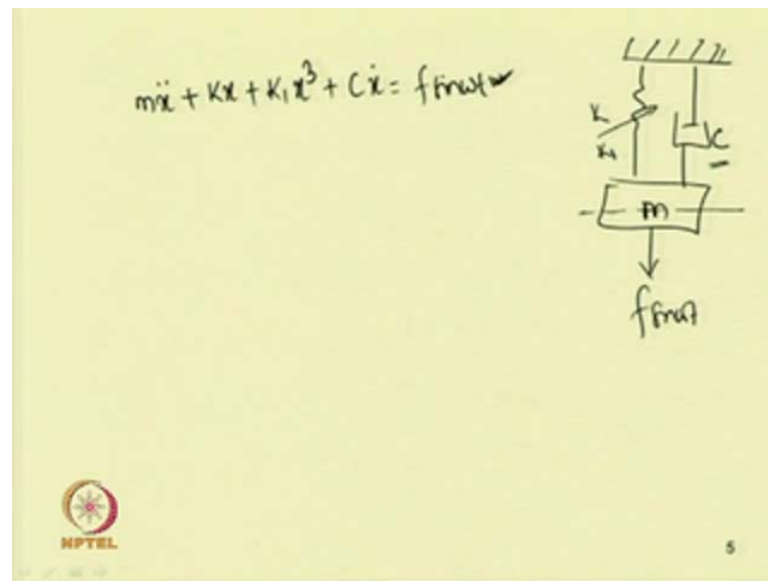


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So, in the previous class we have discussed about several non-linear equations and in those equations how to find the equilibrium points. So, equilibrium points are points for which there is no or these will be these steady state solutions of the systems which will not vary with time. So, to obtain these equilibrium points we will not consider the terms containing the time so, in different equations we can find the equilibrium points and we have to study their stability. So, in case of to find the equilibrium points in different non-linear equations so, previously we have studied different types of methods, different perturbation methods or directly we can use this harmonic balance method or other different types of method solution methods to find the equilibrium points.

So, after finding the equilibrium points we are interested to find so, what should be its stability whether, the obtained solutions is stable unstable or what will happen to the system as time progresses.

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So, in case of a stable system for example, if you consider the simple spring mass system which can be represented by a Duffing equation also so, if we are considering the non-linear spring so, it can be represented by this Duffing equation so, this spring we have considered is non linear. So, this equation can be written in this form that is $m \ddot{x} + kx + k_1x^3 + c\dot{x} = f \sin \omega t$. So, this is k , this is c damping and this is linear stiffness k and non-linear stiffness k_1 . So, in this equation so, first we have to solve the equation to obtain the equilibrium position.

So, for example, the static equilibrium position so, we for which x equal to 0 so, will be the equilibrium position. So, by using several methods we can find what will be the response of the system one may also, solve this equation using numerical methods to find the solutions. So, after getting different types of solution so, we will find the stability of these solutions.

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Solution of Equilibrium points

Fixed point solutions of continuous time systems:

$$\dot{x} = F(x; M) \quad \checkmark$$

Here, fixed point solutions can be obtained by vanishing vector field that is

$$F(x, M) = 0 \quad \checkmark$$

Singular points: Location in the state space where the vector field is vanished is called singular point where integral curve of vector field corresponding to point itself.

Linearization near an Equilibrium solution

Let, for $M = M_0$, solution of $F(x, M) = 0$ is $\underline{x_0}$

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So, to obtain the stability for example, let us take a one dimensional equation so, in this one dimensional equation can be written in this form that is \dot{x} equal to $F(x, M)$ where, M is the control parameter and F is the so, to find the equilibrium position the time derivative term that is \dot{x} we have to put this \dot{x} equal to 0 so, it will reduce to $F(x, M)$ equal to 0. So, it will solve this equation for different value of x so, that will give us the equilibrium position. So, singular points or equilibrium positions are the location in the state space where, the vector field is vanished and we can find that thing by equating this $F(x, M)$ equal to 0. So, we may have to solve a set of algebraic equation or transcendental equation or different types of equations to find this equilibrium position.

So, to study the stability near the equilibrium position one may due to this linearization. So, let for the control parameter M equal to M_0 the solution becomes x equal to x_0 .

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To determine the stability of this singular point, it is required to superimpose on it a small disturbance y and obtain as

$$x(t) = x_0 + y(t) \longrightarrow \dot{y} = F(x_0 + y, M_0)$$


$$\dot{y} = F(x_0 + y, M_0) + D_x F(x_0; M_0) y + O(\|y\|^2)$$

$$\dot{y} = D_x F(x_0; M_0) y = Ay$$

Where

$$A = \begin{bmatrix} \frac{dF_1}{dx_1} & \frac{dF_1}{dx_2} & \dots & \frac{dF_1}{dx_n} \\ \frac{dF_2}{dx_1} & \frac{dF_2}{dx_2} & \dots & \frac{dF_2}{dx_n} \\ \vdots & \vdots & \ddots & \vdots \\ \frac{dF_n}{dx_1} & \frac{dF_n}{dx_2} & \dots & \frac{dF_n}{dx_n} \end{bmatrix}$$

Eigenvalues of the constant matrix A provide the information about the local stability of the fixed point x_0 .



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So, if x equal to x_0 is the solution for this then, one can write by whatever being the solution that means so, let this is the equilibrium position 0. So, now what will happen next to x_0 so, if we perturb this thing then, we can write this at x_t that means a time which is slightly aware from this x_0 . So, we can write this x_t equal to x_0 plus y_t . So, y_t is the small perturbation you could take.

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Solution of Equilibrium points

Fixed point solutions of continuous time systems:

$$\dot{x} = F(x; M) \quad \checkmark$$


Here, fixed point solutions can be obtained by vanishing vector field that is

$$F(x, M) = 0 \quad \checkmark$$

Singular points: Location in the state space where the vector field is vanished is called singular point where integral curve of vector field corresponding to point itself.

Linearization near an Equilibrium solution

Let, for $M = M_0$, solution of $F(x, M) = 0$ is x_0



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Then, we can write this equation previous equation this \dot{x} equal to $F(x, M)$ by this equation so, \dot{y} equal to $F(x_0) + y M$.

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To determine the stability of this singular point, it is required to superimpose on it a small disturbance y and obtain as

$$x(t) = x_0 + y(t) \longrightarrow \dot{y} = F(x_0 + y, M_0)$$


$$\dot{y} = F(x_0 + y, M_0) + D_x F(x_0; M_0) y + O(\|y\|^2)$$

$$\dot{y} = D_x F(x_0; M_0) y = Ay$$

Where

$$A = \begin{bmatrix} \frac{dF_1}{dx_1} & \frac{dF_1}{dx_2} & \dots & \frac{dF_1}{dx_n} \\ \frac{dF_2}{dx_1} & \frac{dF_2}{dx_2} & \dots & \frac{dF_2}{dx_n} \\ \vdots & \vdots & \ddots & \vdots \\ \frac{dF_n}{dx_1} & \frac{dF_n}{dx_2} & \dots & \frac{dF_n}{dx_n} \end{bmatrix}$$

Eigenvalues of the constant matrix A provide the information about the local stability of the fixed point x_0 .




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Now, expanding this thing using Taylor series one can write this y dot will be equal to $F(x_0 + y, M_0)$ plus y dot M_0 plus $d_x F$. So, that is first derivative of this F at x_0, M_0 and one can neglect these higher order terms so, higher order neglecting the higher order terms one can write this equation \dot{y} equal to as already this $F(x_0, M_0)$ equal to 0. So, according to our assumptions here, $F(x_0, M_0)$ equal to 0 which where we are getting x equal to x_0 so, this part is 0 so, one can obtain this y dot equal to y dot equal to $d_x F, x_0, M_0$. So, this matrix that is $d_x F$ or this A is known as the Jacobian matrix. So, by finding the Eigen value of this Jacobian matrix so, we can study the stability of the system. So, we after finding these Eigen values so, if all the real parts of this Eigen value are negative then, the system is stable otherwise the system is unstable.

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Hartman–Grobman theorem or **linearization theorem** is a theorem about the local behaviour of dynamical systems in the neighbourhood of a hyperbolic equilibrium point.

Basically the theorem states that the behaviour of a dynamical system near a hyperbolic equilibrium point is qualitatively the same as the behaviour of its linearization near this equilibrium point provided that no eigenvalue of the linearization has its real part equal to 0.

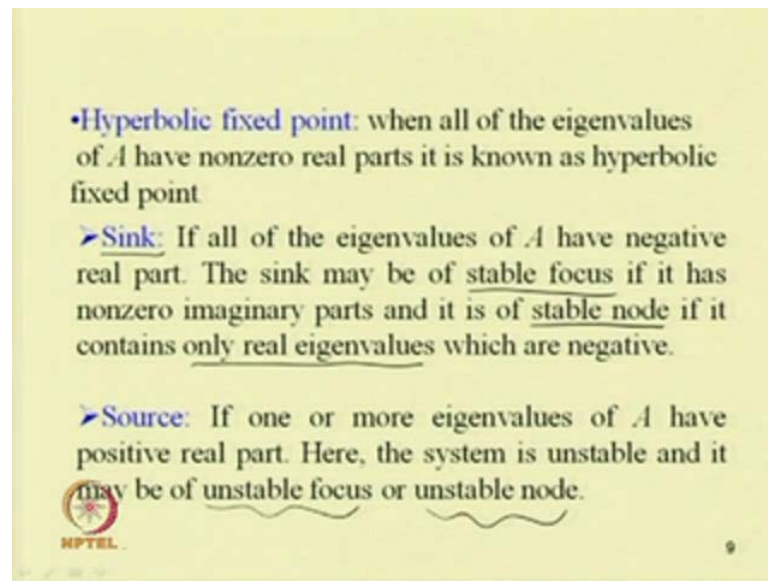


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So, let us take one example so, in case of the so, before that let us see whether this linearization technique so, what we are finding so, whether this is valid for all the cases. And so, there is one theorem that is by a Hartman and Grobman theorem or linearization theorem so, this tells about this theorem is about the local behavior of the dynamical system in the neighborhood of a hyperbolic equilibrium point.

Basically, the theorem states that the behavior of a dynamical system near a hyperbolic equilibrium point is qualitatively the same as the behavior of its linearization near its equilibrium point provided, that no Eigen value of the linearization has its real part equal to 0. So, if we have a hyperbolic fixed point then, according to this theorem so, this linearization linearized system behavior and the original system behavior will be similar. So, if we can linearize near the equilibrium position and study its stability so, this will give the local stability of the system.


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• **Hyperbolic fixed point:** when all of the eigenvalues of A have nonzero real parts it is known as hyperbolic fixed point

➤ **Sink:** If all of the eigenvalues of A have negative real part. The sink may be of stable focus if it has nonzero imaginary parts and it is of stable node if it contains only real eigenvalues which are negative.

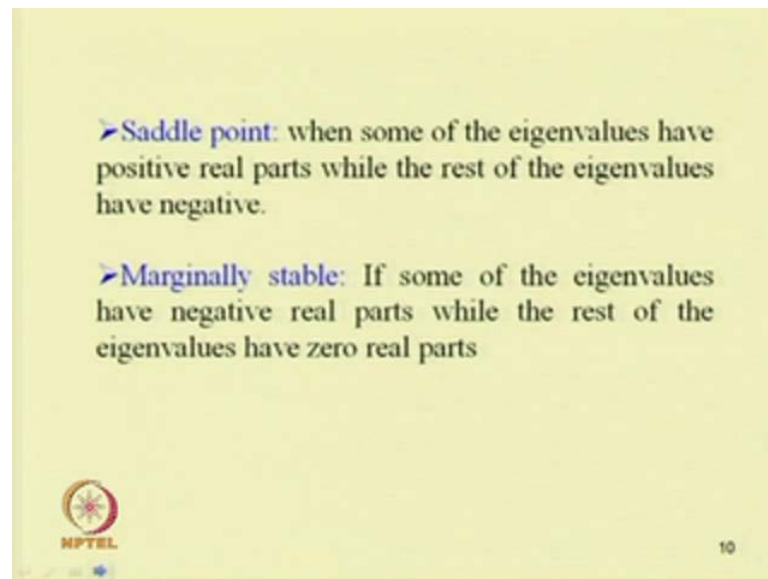
➤ **Source:** If one or more eigenvalues of A have positive real part. Here, the system is unstable and it may be of unstable focus or unstable node.

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So, for finding the global stability of a system one may use this Lyapunov method. So, here some definitions also we one can see. What is hyperbolic fixed point? When all the Eigen values of the Jacobian matrix are non 0 real parts it is known as hyperbolic fixed point. So, if all the Eigen values of a have negative real part then it is known as sink. The sink may be stable focus if it has non 0 imaginary parts and it is a stable node so, either it can be stable focus or it can be stable node.


So, in case of stable focus so, it has non 0 imaginary parts and in case stable node it contains only real Eigen it contains only real Eigen values that means the imaginary parts are 0. So, in the Eigen value so, in case of a sink all the Eigen values have negative real parts and if it contains non 0 imaginary part then, it is stable focus and if it contains only real Eigen values that means the imaginary part is 0 then, it is known as stable node. So, the sink is stable so and source if one or more Eigen values of a have positive real part then, it is known as source. So, here the system is unstable and it may be unstable focus or unstable node. So, in case of unstable focus so, it will have non 0 imaginary parts and in case of unstable node it will it will have 0 imaginary parts. That means, all the Eigen values will be real and some of the Eigen values will contain positive real parts. So, in that case it will be a source or the system will be unstable.

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➤ **Saddle point:** when some of the eigenvalues have positive real parts while the rest of the eigenvalues have negative.

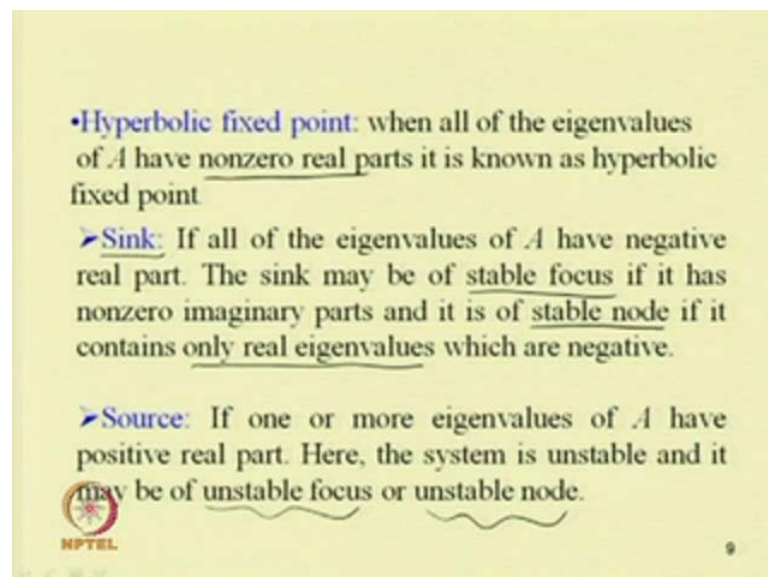
➤ **Marginally stable:** If some of the eigenvalues have negative real parts while the rest of the eigenvalues have zero real parts

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Similarly, saddle point when some of the Eigen values have positive real parts while the rest of the Eigen values have negative parts then, it will be a saddle point.


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• **Hyperbolic fixed point:** when all of the eigenvalues of A have nonzero real parts it is known as hyperbolic fixed point

➤ **Sink:** If all of the eigenvalues of A have negative real part. The sink may be of stable focus if it has nonzero imaginary parts and it is of stable node if it contains only real eigenvalues which are negative.

➤ **Source:** If one or more eigenvalues of A have positive real part. Here, the system is unstable and it may be of unstable focus or unstable node.

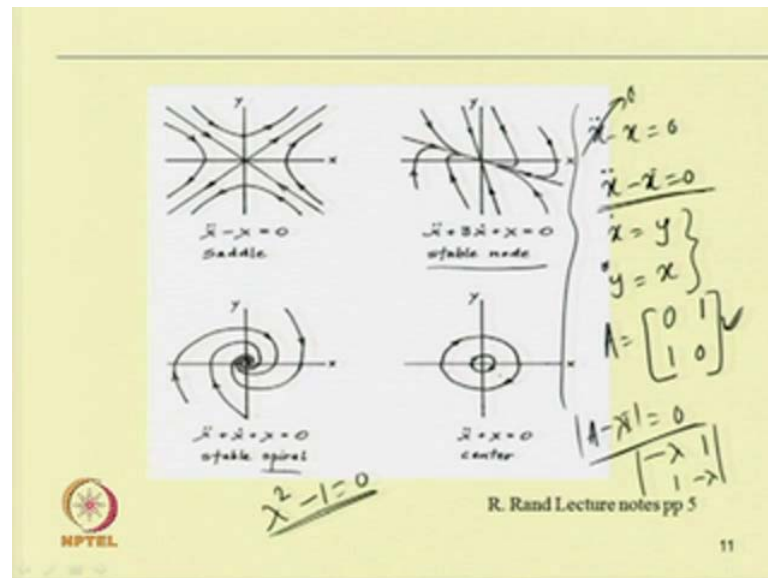
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So, in case of the source if one or more Eigen values have positive real part this is source and in case of saddle point that is some of the Eigen values have positive real parts while the rest of the Eigen values have negative real parts. So, the systems can be called marginally stable if some of the Eigen values have negative real parts while the rest of the Eigen values have 0 real parts. So, these are different terminology for the equilibrium

points so, we have this hyperbolic fixed point when all the Eigen values of a has non 0 real parts then, it is hyperbolic so, if it has 0 real part if some of the Eigen values have 0 real part then, it is called non hyperbolic fixed point. So, in case of the hyperbolic fixed point we have seen source, saddle point and based on these things we can study the equilibrium system.

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For example, already we have seen in this example $\ddot{x} - x = 0$ so, if we have $\ddot{x} - x = 0$. So, by putting this part equal to 0 which varies with time so, the equilibrium position is $x = 0$ that is, the origin so $x = 0$ is the equilibrium position.

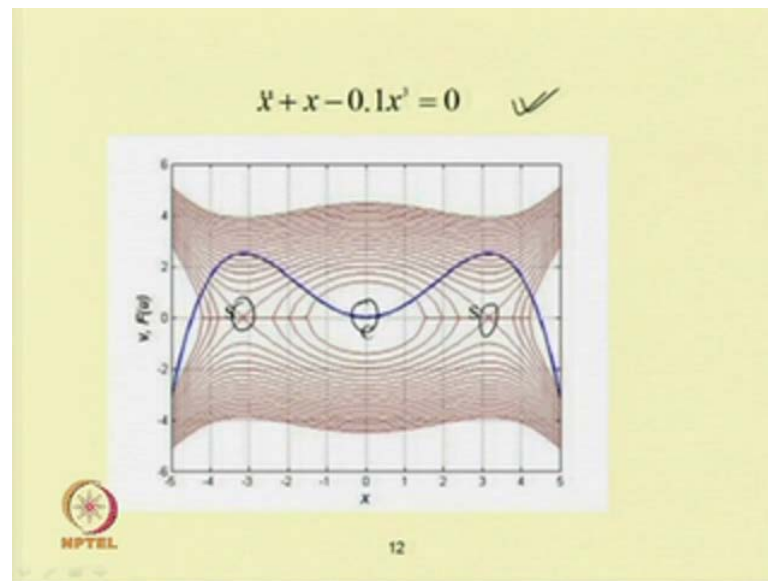
So, in this equilibrium position one can see that this point is a saddle point. So, later we will see how we have checked its Eigen value to find whether it is a saddle point or sink or source. For example, this equation so, this $\ddot{x} - x = 0$ can be written by using 2 equations so, this is second order differential equation that thing can be written as a first order differential equation. So, let me put the first equation $\dot{x} = y$ so, the second equation will be this second order differential equation can be written by using these 2 so, $\dot{x} = y$ and I can write this \ddot{x} that is equal to \dot{y} so, $\dot{y} = x$ so, \dot{y} will be equal to x so, $\dot{y} = x$ so, $\ddot{x} = x$ so, $\dot{y} = x$ so, $\dot{y} = x$. So, these are the 2 equations these are the 2 first order equation one can solve to check whether the point is a saddle node or whether the point is a saddle node

point or not. So, in this case $\dot{x} = y$ and $\dot{y} = x$ so, one can write this A matrix as so, A matrix can be written so, for the first equation $\dot{x} = y$ so, this is $x = 0$ and x there is no x term that is 0 so, this is 1 and in case of the second equation one can write this is equal to 1 0.

So, this is the equation. As this is the linear equation, one can find its Δx dot Δx dot will be equal to Δy and Δy dot equal to Δx . So, A can be written in this way or a can be written by Δf by Δx Δf by Δx equal to 0 and Δf by Δy equal to 1. Similarly, in the second equation Δf by Δx equal to 1 and as there is no y term so, this is equal to this. So, from this $A - \lambda I$ so, $A - \lambda I$ determine of $A - \lambda I$ equal to 0 where λ is the Eigen value so, from this one can obtain this λ so, this is $\lambda A - \lambda$ so, λ so, this equation so, from this equation one can write so, this is $\lambda - 1$ so, then this is $1 - \lambda$. So, $A - \lambda$ is 0 determinant of this thing so, from this one can get this λ^2 so, $\lambda^2 - 1$ equal to 0 so, if $\lambda^2 - 1$ equal to 0 so, 1 so, from this one can find the roots so, from these roots so $\lambda^2 - 1$ equal to 0.

So, by finding the roots one can study whether the point is stable, marginally stable or it is a saddle point or node point. So, in this case also one can similarly, proceeding in this way write writing the set of first order equation so, one can check whether the system is a stable node whether, the system is a so, this spiral means this so, this spiral out or in so, this is a stable spiral as all the points are spiral in inside this is a stable point similarly, this is a center point.

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So, in this case also in this example $\ddot{x} + x - 0.1x^3 = 0$, which is similar to that of a duffing equation equal to 0. So, one can see this point is a saddle point this point correspond to the center point and this point correspond to the saddle point corresponding to this maximum the saddle point corresponding to the maximum potential energy and center correspond to the minimum potential energy.

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Example $F(x, y) = \lambda y$

$\dot{y} = \lambda y$ ✓

Equilibrium point $y(t) = 0$ ✓

Stable Equilibrium point if $\lambda < 0$

Unstable Equilibrium point if $\lambda > 0$

$y(t) = \exp(\lambda t)$ $k =$

Handwritten notes on the right side of the slide:

$$\frac{dy}{dt} = \lambda y$$

$$\frac{dy}{y} = \lambda dt$$

$$\ln y = \lambda t$$

$$y = \exp(\lambda t)$$

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So, as another further one example one can take this y dot simple example y dot equal to λy so, the equilibrium position y t will be equal to so, in this case y dot we will put

this \dot{y} equal to 0. So, y can be written equal to 0 the stable equilibrium position. So, for stable equilibrium position we can write so, our equation reduces to so, as \dot{y} equal to λy so, we can find this A minus λ i so, we can write A so, if λ less than minus λ less than 0 so, the point will be stable and if λ greater than 0 so, it will show the system unstable. For example, in this case the solution is y equal to as $d y$ by $d t$ equal to λy so, \dot{y} equal to this or $d y$ by $d t$ equal to λy or one can find this $d y$ by y equal to $\lambda d t$. So, by integrating one can find this $\ln y$ equal to λt so, from this one can write y equal to some constant some constant let C 1 or C so, e to the power λt . So, this is the solution. So, in this case if λ is positive then, exponentially it will increase the response will exponentially increase and it will become unstable and if λ is negative exponentially it will decrease to make the system stable. So, depending on the value of λ one can find this thing.

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Example $F(y, \dot{y}) = \lambda y$

$\dot{y} = \lambda y$ ✓

Equilibrium point $y(t) = 0$ ✓

Stable Equilibrium point if $\lambda < 0$

Unstable Equilibrium point if $\lambda > 0$

$y(t) = \exp(\lambda t)$ $\lambda = \lambda$

Handwritten derivations on the right:
 $\frac{dy}{dt} = \lambda y$
 $\frac{dy}{y} = \lambda dt$
 $\ln y = \lambda t$
 $y = \exp(\lambda t)$

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So, in this case if we use our original method that is \dot{y} equal to in this case so, our A matrix to obtain this A matrix one can find this $\frac{\partial f}{\partial y}$ so, this is so our $F \times M$. So, in this case our $F \times M$ equal to λy so, $\frac{\partial f}{\partial y}$ by $\frac{\partial x}{\partial y}$ or you can write so, for x instead of writing x as our variable is y so, we can write this is y m and so, in this case we can write $\frac{\partial f}{\partial y}$ by so, $\frac{\partial f}{\partial y}$ by $\frac{\partial y}{\partial y}$ so, A equal to λ A so, A equal to λ or i can so, from this one can study or one can find the Eigen value and from this Eigen value we have seen that if λ less than 0 it is stable and if λ greater than 0 it is unstable.

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Example: Duffing Equation

$$\ddot{u} + \omega_0^2 u + 2\epsilon \mu \dot{u} + \epsilon \alpha u^3 = \epsilon K \cos \Omega t$$

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$$\begin{cases} a' = -\mu a + \frac{1}{2} \frac{K}{\omega_0} \sin \gamma \\ a \gamma' = 6a - \frac{3}{8} a^3 + \frac{1}{2} \frac{K}{\omega_0} \cos \gamma \end{cases}$$

$$\begin{cases} \mu a = \frac{1}{2} \frac{K}{\omega_0} \sin \gamma \\ a \left(6 - \frac{3}{8} \frac{a^2}{\omega_0^2} \right) = -\frac{1}{2} \frac{K}{\omega_0} \cos \gamma \end{cases}$$

$$\left[\mu^2 + \left(6 - \frac{3}{8} \frac{a^2}{\omega_0^2} \right)^2 \right] a^2 = \frac{K^2}{4\omega_0^2}$$

$$u = \hat{a} \cos(\omega_0 t + \beta)$$

$$\gamma = \sigma T_1 - \beta$$

$$\Omega = \omega_0 + \epsilon \sigma$$

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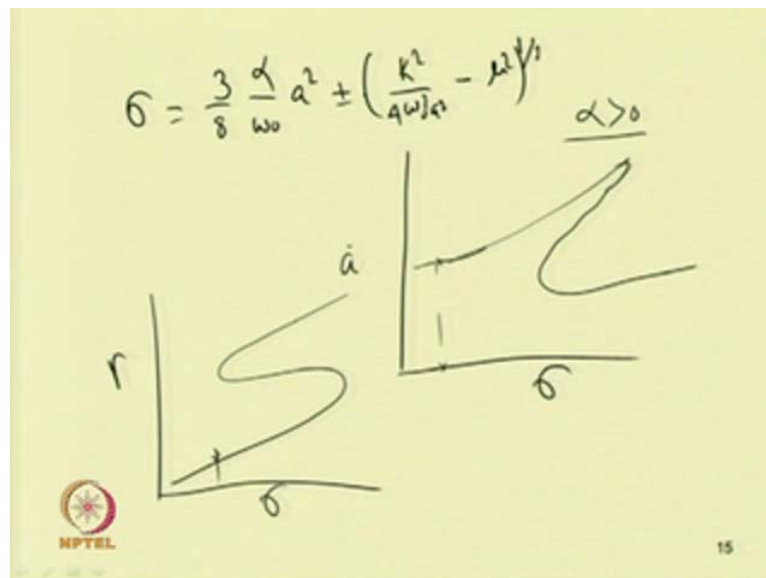
So, let us take this another example that is duffing equation. In case of the duffing equation either we can directly solve this equation to find its stability or we can use the perturb equation using method of multiple scale. For example, if one use method of multiple scales then, one can obtain a set of reduce equation which can be written in this form. So, a dash equal to minus mu a. So, let us take the duffing equation in this form that is u double dot plus omega 0 square u plus 2 epsilon mu u dot plus epsilon alfa u cube equal to epsilon k cos omega t.

So, here this is the epsilon mu u dot that is the damping and this is the non-linear term and this is the forcing term this part is the forcing term so, here we have assumed the force to be very small that is why have used this book keeping parameter epsilon omega square e t square of the frequency and mu is the damping factor. So, in this equation the solution can be written in this form u will be equal to a cos omega 0 t plus beta where, this beta or gamma can be written as sigma t 1 minus beta where, sigma is the detuning parameter sigma is the detuning parameter to express the nearness of this parameter with respect to the excitation frequency omega. So, in this case this equation the reduce equation can be written a dash equal to minus mu a plus half k by omega 0 sin gamma and this a gamma dash equal to sigma a minus 3 by 8 a cube plus half k by omega 0 cos gamma. So, these 2 are the reduced equation obtained by using method of multiple scale so, using method of multiple scale one can find these 2 reduced equation. Now, for steady state motion so, as they are not function of time so, this a dash and gamma dash

will be equal to 0. So, one can write the equation in this form μa will be equal to half k by $\omega_0 \sin \gamma$ and σ minus $\frac{3}{8} \alpha$ by $\omega_0 a^3$ so, this will be equal to minus half k by $\omega_0 \cos \gamma$.

So, from these 2 equation right hand side, squaring the right hand side and add adding these 2 so, one can obtain the frequency equation so, the frequency equation will be in this form so, $\mu^2 a^2$ plus σ minus $\frac{3}{8} \alpha$ by $\omega_0 a^3$ square whole square into a square equal to k^2 by $4 \omega_0^2$ square. So, this is the frequency this is the frequency response equation. So, from this one can find the amplitude so, we have seen the solution of this equation equal to $a \cos \omega_0 t$ plus β so, this amplitude a and frequency they are related by using this equation. So, in this equation we have this σ which is detuning parameter so, that can be that shows the nearness of this ω so, ω will be equal to ω_0 plus $\epsilon \sigma$.

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So, from this equation from this frequency equation so, one can plot the response so, this equation can further be written using this σ . So, σ will be equal to $\frac{3}{8} \alpha$ by $\omega_0 a^3$ plus minus k^2 by $4 \omega_0^2$ square minus μ^2 square to the power half. So, μ^2 square to the power so, using this σ and a equation this equation is a quadratic equation in terms of a , in terms of σ so, that is why solving this quadratic equation one can obtain the σ . But, one can numerically also solve this equation this will be a sixth order equation in terms of a square as we have a square here

square term is there another a square is multiplied outside so, this is a sixth order equation in terms of a or a quadratic equation in terms of sigma as it is easier to solve this quadratic equation so, one can write or one can solve this quadratic equation to write sigma otherwise, one can use the numeric method some numerical method to solve this equation for a. Now, solving this equation one can obtain the response so, the response a and sigma one can obtain for example, for alpha greater than 0 the curve will be curve will looks like this. So, in this case each point on this curve whether these points or this solution what we obtained so, this is for a and one can plot the gamma equation also gamma also, this is a versus sigma is the detuning parameter a is the amplitude.

Similarly, gamma is the phase and with respect to sigma one can plot and here one can see that the plot will look like this. So, in this case for corresponding to a particular value of a for example, sigma equal to for this sigma value so, this is a value and this is the sigma value so, correspond this is the gamma value. So, corresponding to this a and gamma value so, one can check whether this point is stable or not.

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$$\left. \begin{aligned} a &= a_0 + a_1 \\ r &= r_0 + r_1 \end{aligned} \right\}$$

$$J = \begin{bmatrix} -\mu & -a\left(5 - \frac{3\alpha a_0^2}{8\omega_0}\right) \\ \frac{1}{a_0}\left(5 - \frac{9\alpha a_0^2}{8\omega_0}\right) & -\mu \end{bmatrix}$$

So, to check the stability of this point so, one can perturb the solution that means taking this a equal to a 0 plus let me take this is equal to a 1, a 0 plus a 1 and gamma equal to gamma 0 plus gamma 1.

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Example: Duffing Equation

$$\ddot{u} + \omega_0^2 u + \frac{1}{2} \epsilon \ddot{u} + \epsilon \alpha u^3 = \epsilon K \cos \Omega t$$

$$u = \bar{a} \cos(\omega_0 t + \beta)$$

$$\gamma = \frac{\sigma}{T_1} - \beta$$

$$\Omega = \omega_0 + \epsilon \sigma$$

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$$\begin{cases} a' = -\mu a + \frac{1}{2} \frac{K}{\omega_0} \sin \gamma \\ a r' = 6a - \frac{3}{8} a^3 + \frac{1}{2} \frac{K}{\omega_0} \cos \gamma \end{cases}$$

$$\begin{cases} \mu a = \frac{1}{2} \frac{K}{\omega_0} \sin \gamma \\ a \left(6 - \frac{3}{8} \frac{a^2}{\omega_0} \right) = -\frac{1}{2} \frac{K}{\omega_0} \cos \gamma \end{cases}$$

$$\left[\mu^2 + \left(6 - \frac{3}{8} \frac{a^2}{\omega_0} \right)^2 \right] a^2 = \frac{K^2}{4 \omega_0^2}$$

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So, one can perturb this equation that means this equation a dash gamma dash equation can be perturbed.

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$$\begin{cases} a = a_0 + \epsilon a_1 \\ \gamma = \gamma_0 + \epsilon \gamma_1 \end{cases}$$

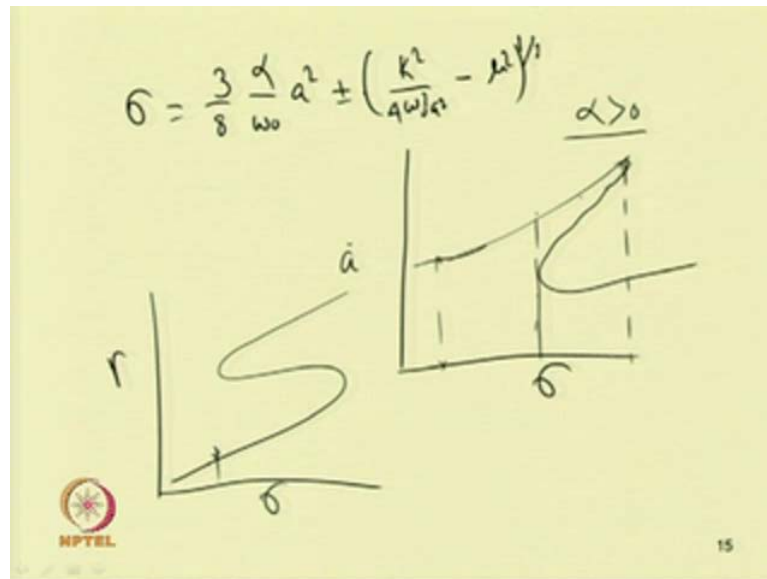
$$J = \begin{bmatrix} -\mu & -a \left(6 - \frac{3\alpha a_0^2}{8\omega_0} \right) \\ \frac{1}{a_0} \left(6 - \frac{9\alpha a_0^2}{8\omega_0} \right) & -\mu \end{bmatrix}$$

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And by substituting this a and gamma in that equation one can write this one can obtain the Jacobian matrix. So, Jacobian matrix can be written in this form that is minus mu minus a 0 into sigma minus 3 alpha a 0 square by 8 omega 0. So, this is 1 by a 0 into sigma minus 9 alpha a 0 square by 8 omega 0 so, this is minus mu.

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Now, one can find the Eigen value of this Jacobian matrix corresponding to this value of a and γ and study the stability whether, this branch is stable or unstable and after studying the stability of each branch one can know whether the corresponding response is stability or unstable. So, one can note in this case so, up to the σ_0 , the system has a single branch but, after this value of σ_0 , one can have 3 solutions so, out of these 3 solutions some solutions maybe stable some maybe unstable and one can study this stability by finding the Eigen value of this Jacobian matrix.

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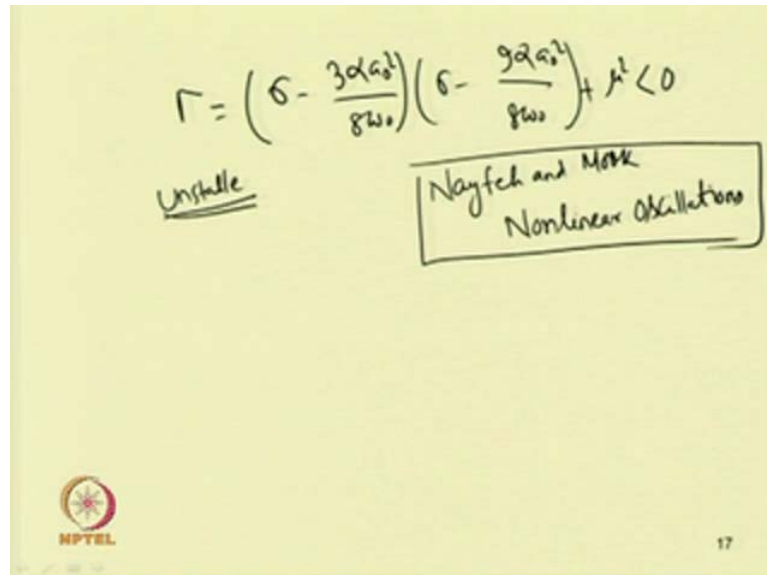
$$\begin{aligned} a &= a_0 + a_1 \\ r &= r_0 + r_1 \end{aligned}$$

$$J = \begin{bmatrix} -\mu & -a_0 \left(\sigma - \frac{3\alpha a_0^2}{8\omega_0} \right) \\ \frac{1}{a_0} \left(\sigma - \frac{9\alpha a_0^2}{8\omega_0} \right) & -\mu \end{bmatrix}$$

$$|A - \lambda I| = 0 \quad \lambda^2 + 2\mu\lambda + \mu^2 + \left(\sigma - \frac{3\alpha a_0^2}{8\omega_0} \right) \left(\sigma - \frac{9\alpha a_0^2}{8\omega_0} \right) = 0$$

So, these Eigen value so, for example, in this case so, by putting this a minus lambda i equal to 0 we will have the equation lambda square plus 2 mu lambda plus mu square plus sigma minus 3 alpha a 0 square by 8 omega 0 into sigma minus 9 alpha a 0 square by 8 omega 0 equal to 0 for steady state motion. So, the steady state motion will be unstable.

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Handwritten slide content:

$$\Gamma = \left(\sigma - \frac{3\alpha a_0^2}{8\omega_0} \right) \left(\sigma - \frac{9\alpha a_0^2}{8\omega_0} \right) + \mu^2 < 0$$

Unstable

Nayfeh and Mook
Nonlinear Oscillations


NPTL 17

So, if we have this term gamma equal to sigma minus 3 alpha a 0 square by 8 omega 0 into sigma minus 9 alpha a 0 square by 8 omega 0 plus mu square less than 0. So, one can see the book by Nayfeh and Mook non-linear oscillation non-linear oscillation to know more about the stability of this type of system non-linear oscillations.

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$$\left. \begin{aligned} a &= a_0 + a_1 \\ r &= r_0 + r_1 \end{aligned} \right\}$$

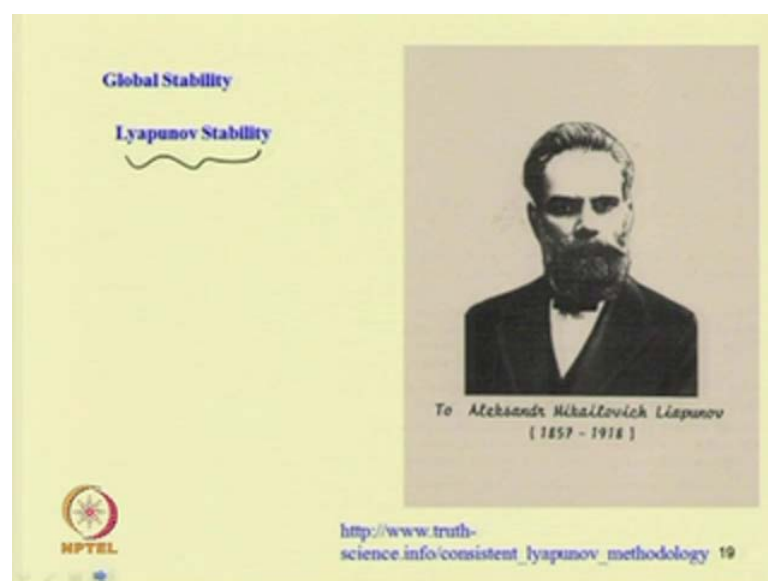
$$J = \begin{bmatrix} -\mu & -a\left(6 - \frac{3\alpha a_0^2}{8\omega_0}\right) \\ \frac{1}{a_0}\left(6 - \frac{9\alpha a_0^2}{8\omega_0}\right) & -\mu \end{bmatrix}$$

$$|A - \lambda I| = 0 \quad \lambda^2 + 2\mu\lambda + \mu^2 + \left(6 - \frac{3\alpha a_0^2}{8\omega_0}\right)\left(6 - \frac{9\alpha a_0^2}{8\omega_0}\right) = 0$$


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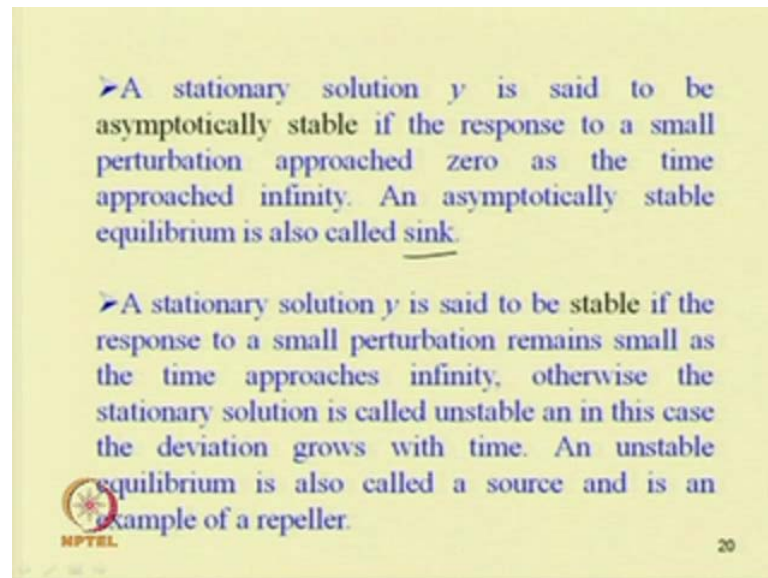
So, one can obtain the stability from this equation when λ is less than 0 the real part of λ is less than 0 then, the system will be stable so, from this one can solve and see this equation or find that if this term less than 0 then, the system will be unstable. So, in this way one can study the stability of the system. Sometimes, this Jacobian matrix may have distinct Eigen values or sometimes they may not be distinct so, depending on these Eigen values one can study the stability and one can obtain the bifurcation diagrams.

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So, let us to study the global stability one can use this Lyapunov stability method so, instead of going for the linearization technique so, one can use Lyapunov stability. So, in case of the Lyapunov stability one has to develop the difficulty of this Lyapunov stability method. One has to develop a function Lyapunov function to find to study the stability of the system.

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So, in case of this Lyapunov stability according to this a stationary solution y is said to be asymptotically stable if the response to a small perturbation approach 0 as the time approach infinity. An asymptotically stable equilibrium is called a sink. Stationary solution y is said to be stable if the response to a small perturbation remains small as the time approaches infinity. Otherwise, the stationary solution is called unstable as in this case the deviation grows with time an unstable equilibrium is also called a source and is an example of repeller. So, in this case one has to find one Lyapunov function and one has to check whether the system is asymptotically stable or stable or unstable.

So, in this case so, let this is the equilibrium solution so, near this equilibrium solution if we or if we perturb this equilibrium solution so, if it remains within a bounded region then it becomes stable and if with time it approaches this equilibrium position then, it is asymptotically stable. So, let us take one example so, let us take the van der pol equation so, according to this Lyapunov method we have to take one Lyapunov function which is

a positive definite function so, if its derivative becomes negative definite then, the system is stable otherwise, the system is unstable.

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$$\ddot{y} + y - \epsilon \left(\frac{y^3}{3} - y \right) = 0$$

Van der Pol's oscillator

$$\begin{cases} \dot{y} = x \\ \dot{x} = -y + \epsilon \left(\frac{x^3}{3} - x \right) \end{cases}$$

Equilibrium pt = $x=0$ and $y=0$

$$V = \frac{1}{2} (x^2 + y^2)$$

$$\dot{V} = \frac{1}{2} 2x\dot{x} + \frac{1}{2} 2y\dot{y} = x\dot{x} + y\dot{y}$$

$$= x(-y + \epsilon (\frac{x^3}{3} - x)) + y(x)$$

So, for example, taking this van der pol equation let us take the van der pol equation that is, $x \ y \ double \ dot \ plus \ y \ minus \ epsilon \ y \ dot \ cube \ by \ 3 \ minus \ y \ dot$ so, this is equal to 0. So, in this case so, this is the well known van der pol equation so, one can write this equation by using a set of first order equation.

So, in this case first order equation will be in let me put this $y \ dot$ equal to x so, the second equation becomes $x \ dot$ that means $y \ double \ dot \ x \ dot$ equal to $minus \ y \ plus \ epsilon \ into \ x \ cube \ by \ 3 \ minus \ x$ so, this is equal to 0. So, in this case the equilibrium points will be $x \ dot$ equal to 0 and by putting this $x \ dot$ equal to 0 and $y \ dot$ equal to 0 so, we will find the equilibrium position by putting this $y \ dot$ equal to 0 so, x equal to 0 is x we obtain x equal to 0. Now, by putting $x \ dot$ equal to 0 and x equal to 0 so, we obtain the equilibrium so, the equilibrium point so the equilibrium point is x equal to 0 and y equal to 0.

So, we have to check whether this equilibrium point is stable or not so, we can take a Lyapunov function let us take a Lyapunov function V equal to half x square plus y square so, this function is positive so, this is a positive definite function so, this is always positive for all the values of x and y . Now, we can form this equation so, we can write

this \dot{V} so, if you differentiate this thing so, this \dot{V} equal to half into 2 into x into \dot{x} plus half into 2 into y into \dot{y} so, this becomes $x \dot{x}$ plus $y \dot{y}$. So, this thing can be written so, from this we know, what is our \dot{x} and \dot{y} value from this equation.

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$$\ddot{y} + y - \epsilon \left(\frac{y^3}{3} - y \right) = 0$$
Van der Pol's oscillator

$$\begin{cases} \dot{y} = x \\ \dot{x} = -y + \epsilon \left(\frac{x^3}{3} - x \right) \end{cases}$$

Equilibrium Pt = $x=0$ and $y=0$ ✓

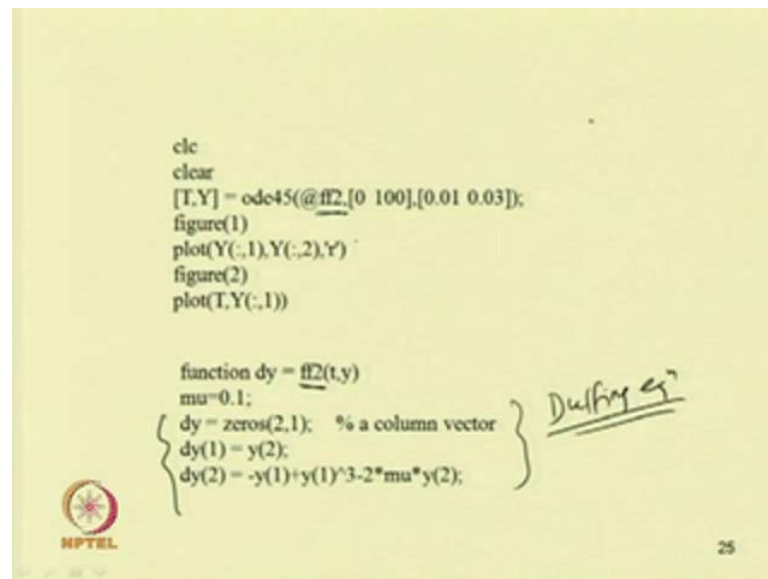
$$V = \frac{1}{2} (x^2 + y^2)$$

$$\begin{aligned} \dot{V} &= \frac{1}{2} 2x\dot{x} + \frac{1}{2} 2y\dot{y} = x\dot{x} + y\dot{y} \\ &= x(-y + \epsilon(\frac{x^3}{3} - x)) + yx \\ &= -xy + \epsilon x(\frac{x^3}{3} - x) + xy \\ &= -\epsilon(x^2 - \frac{x^4}{3}) \quad \checkmark \end{aligned}$$


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Now, substituting that thing so, we can write x into so \dot{V} will be x into so for \dot{x} one can substitute this is equal to minus y plus epsilon into x cube by 3 minus x so, into minus x plus y into \dot{y} so, \dot{y} equal to x so, this becomes so, minus $x y$ plus epsilon x into x cube by 3 minus x plus so, this is $x y$ so, minus $x y$ plus $x y$ cancel. So, this term can be written as minus epsilon x into x becomes x square minus x fourth by 3. So, this becomes minus epsilon x square minus x fourth by 3. So, this term is as x square and x fourth so, they are positive so, this term can be so, this term for so, this term will be always negative. So, from this we have seen that this point so, will have a stable point at x equal to 0 and y equal to 0.

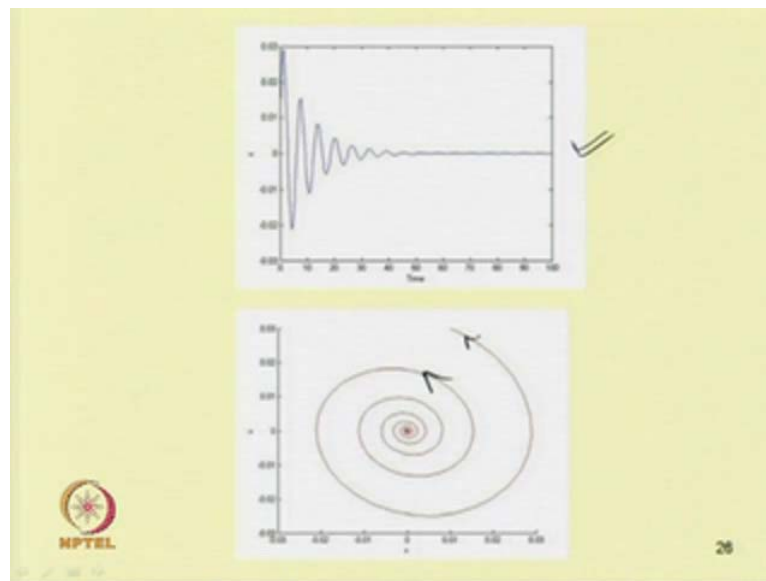
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So, let us see to obtain the solution so one can use this numerical technique using mat lab also one can find the response of the system. For example, in case of this duffing equation so, one can write this program using this ode45 so, for ode45 one can use the simple comments so, t y equal to ode 45 so, this is the function and using ff 2.

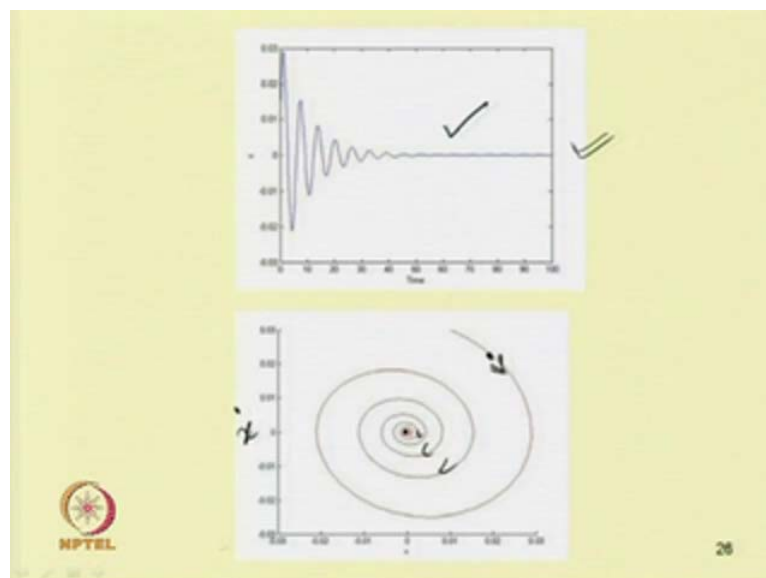
So, this is the function so, which where this first order differential equations 2 first order differential equations are written that is $dy_1/dt = y_2$ and this $dy_2/dt = -y_1 + y_1^3 - 2\mu y_2$. So, this is for the duffing equation. Similarly, one can write so, similarly, one can write for the van der pol equation and by using this command one can find the response of the system. So, here the response are plotted that is y_1 y_2 y_1 and y_2 are these displacement and velocity so, one can obtain the phase portrait so, also one can obtain by plotting this t and y_1 so, one can obtain the time response so, from the time response and frequency response one can study whether the system is stable or unstable.

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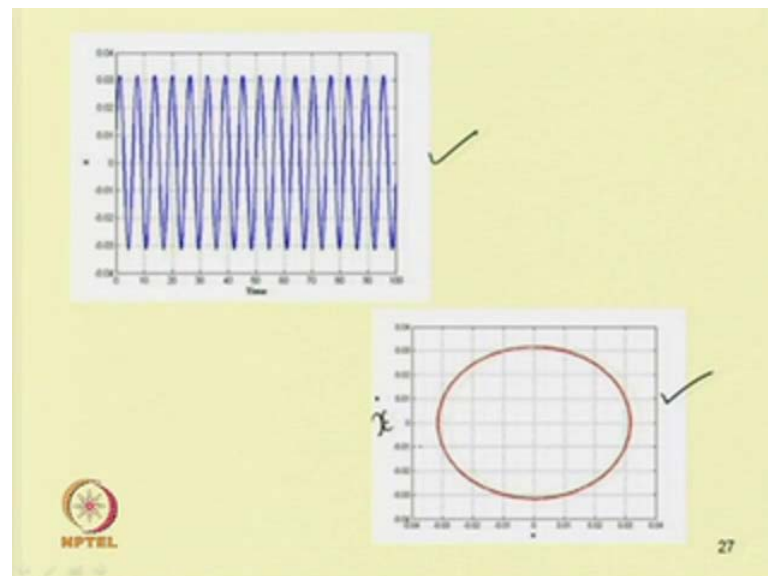
So, one can find the response so, from this response also one can study whether it is decreasing exponentially decreasing or this is decreasing and finally, it is stable. In this case or from the phase portrait so this is the time response so, one can plot the phase portrait also. So, in case of the phase portrait so, starting from this initial position if the response grows that means with time it increases and increases so, than it will be unstable.

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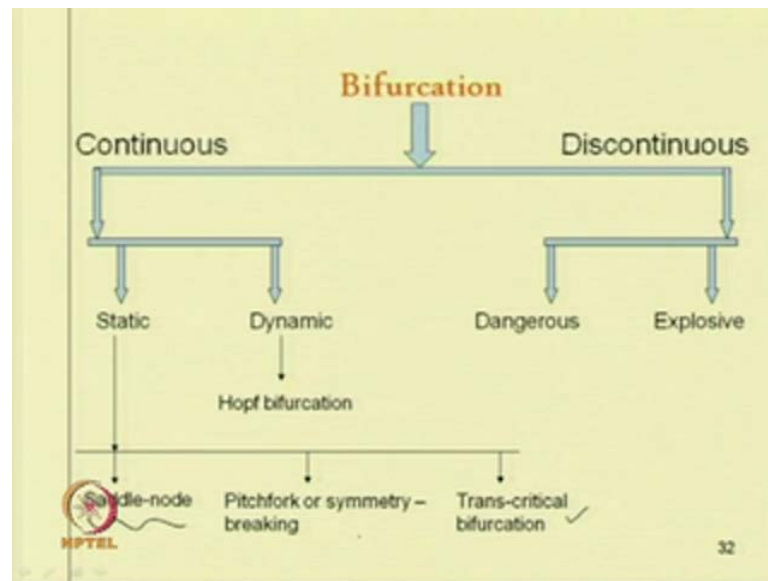
Or if it is starting from this position let us let me start from this position. So, in this case if the response decreases and finally, comes to this position so, this position is a stable position. So, if one plot so, for this cases the phase portrait is this so, this shows a stable point. So, by using these numerical methods also one can find the time response phase Portrait and from this time response and phase portrait one can visualize whether actually this system is stable at the equilibrium position. So, this is the equilibrium position corresponding to x equal to 0 and \dot{x} equal to 0, you are so, this is x equal to 0 and \dot{x} equal to 0 in the equilibrium position. So, it shows that this equilibrium position is stable.

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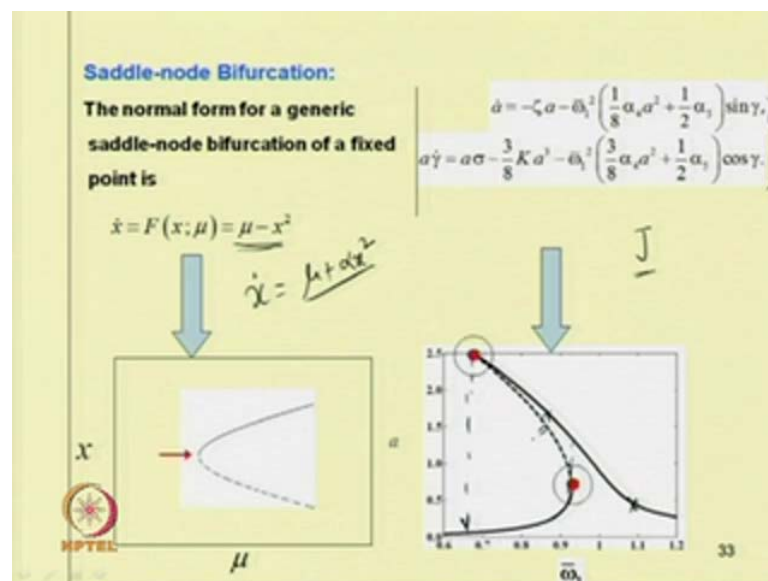
So, from this time response one can understand whether the response is stable or not. So, this also a periodical response in case of a periodical response the phase portrait so, the phase portrait is periodic. So, one can use these methods this time response and phase portrait to visualize actually whether the system is stable or not and by finding the Eigen values from the Jacobian matrix one can theoretically or analytically predict whether the systems will be stable or unstable at that point.

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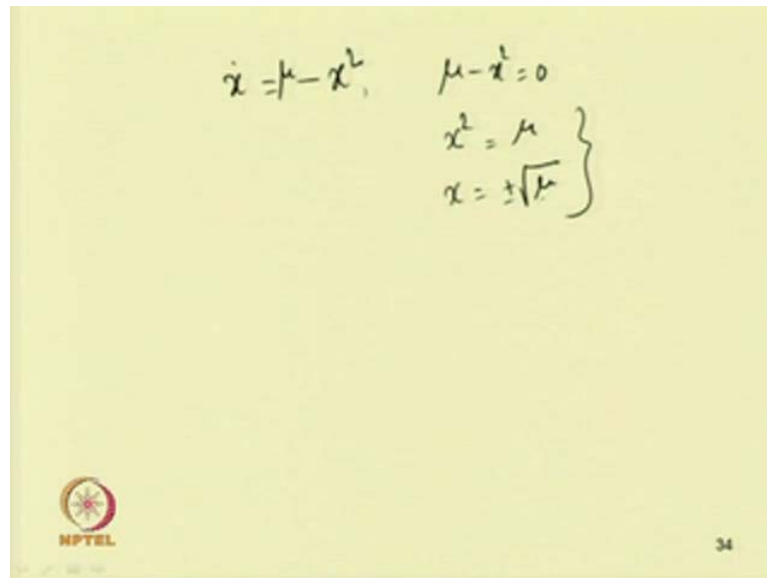
So, already we have seen the bifurcation so, bifurcation points and in case of the bifurcation point we know we have 2 different bifurcations one is continuous bifurcation and other one is discontinuous bifurcation. In case of continuous bifurcation we have static bifurcation and dynamic bifurcation. In case of static bifurcation, we have this saddle node bifurcation pitchfork or symmetry breaking bifurcation and trans-critical bifurcation and in case of the dynamic bifurcation we have hopf bifurcation. So, today class we will basically study about the saddle node and pitchfork symmetry breaking bifurcation.

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And so, in case of the saddle node bifurcation the normal form for a generic saddle node bifurcation fixed point is so, \dot{x} equal to or \dot{x} equal to $F(x)$ mean or this thing can be written so, this is the generic form that is $\mu - x^2$ or one can write also this generic form \dot{x} equal to so, \dot{x} equal to $\mu - \alpha x^2$ or $\mu + \alpha x^2$. So, in this form also one can write the equation so, in this case α equal to minus 1 so, it becomes $\mu - x^2$.

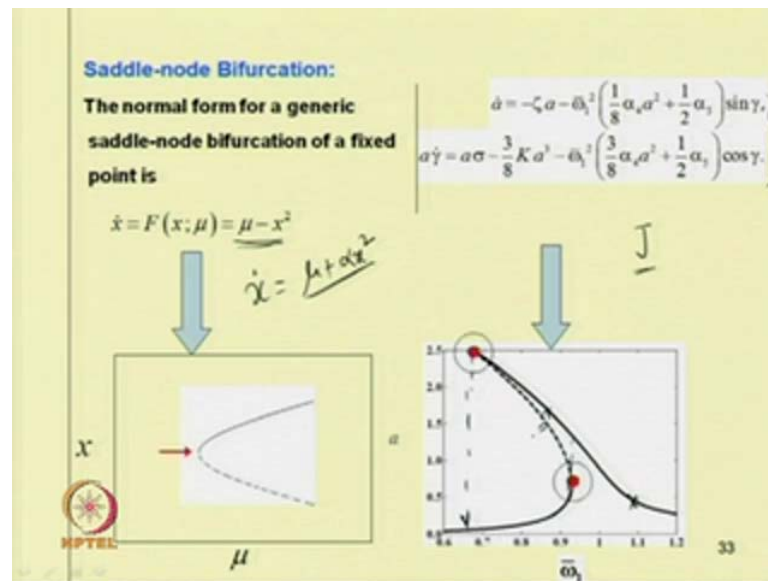
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$$\begin{aligned} \dot{x} &= \mu - x^2, & \mu - x^2 &= 0 \\ & & x^2 &= \mu \\ & & x &= \pm\sqrt{\mu} \end{aligned} \quad \left. \vphantom{\begin{aligned} \dot{x} &= \mu - x^2, \\ \mu - x^2 &= 0 \\ x^2 &= \mu \\ x &= \pm\sqrt{\mu} \end{aligned}} \right\}$$

So, for the saddle node bifurcation point now, one can find the Jacobian matrix as \dot{x} equal to $\mu - x^2$ so, as we have this \dot{x} equal to $\mu - x^2$. So, in this case this equilibrium point becomes putting this \dot{x} equal to 0 so, equilibrium point $\mu - x^2$ equal to 0. So, from this one obtain this x^2 equal to μ or x equal to $\pm\sqrt{\mu}$. So, corresponding to positive and negative value of μ one can plot the responds plot. So, this is the x vs μ plot.

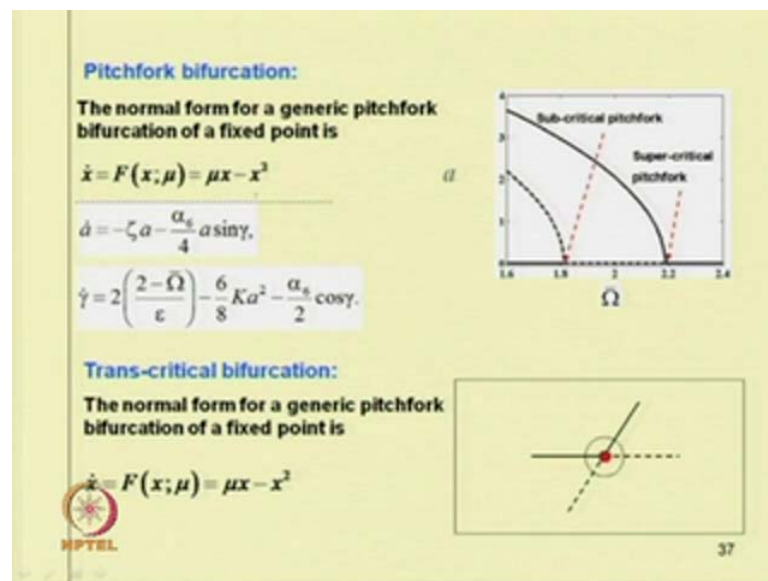
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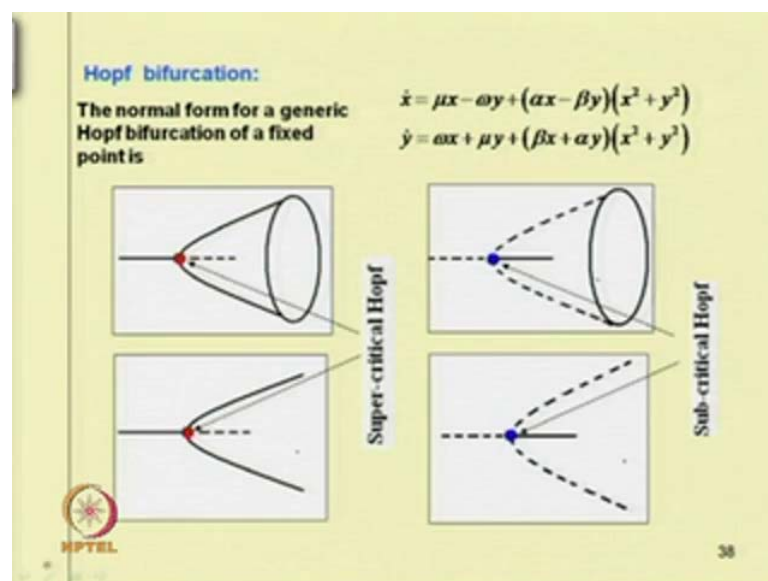
So, one can see that corresponding to different value positive value of μ so, this is positive value of μ and negative for negative value of μ one can see so from this equation so, if μ is negative so, this is imaginary so, the root will be or x will be imaginary so, that is why it is not plotted and for x after x equal to μ equal to 0 only we will have the solution. So, here one may note that before μ equal to 0 so, there no solution exists in this case and after this μ equal to 0 so, we have 2 branches of solution. So, this point μ corresponding to 0 is a bifurcation point as here there is change in the number of the solutions and also we can see that the quality of the solutions that is the stability type of the solutions also changes. So, this is for a one dimensional one dimensional equation.

So, for two dimensional equations also, this is example of a two dimensional equation so, in this case one can find the Jacobian matrix so, find after finding this Jacobian matrix so, one can obtain the Eigen values of this Jacobian matrix and one can observe that this part of the solution is unstable and this is stable so, as it is unstable at this position. So, when one sweep off the frequency so, at this position the system will show a jump of phenomena and further increase in this frequency it will follow this part. Similarly, during sweeping down the frequency that means if you decrease the frequency so, one can follow this path so, one can follow this path during sweeping down and at this point so, it will jump down. So, one can observe this jump up and jump down phenomena at the bifurcation points.

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So, similarly, one can take this pitchfork bifurcation so, in case of pitchfork bifurcation the normal form of the pitchfork bifurcation is $\mu x + \alpha x^2 = 0$.

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$$\dot{x} = \mu x + \alpha x^2$$

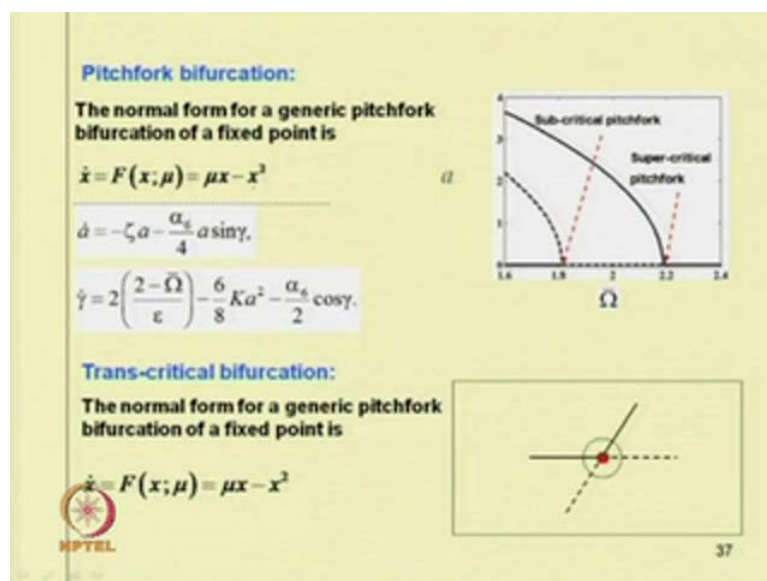
$$\dot{x} = 0 \Rightarrow \mu x + \alpha x^2 = 0$$

$$x(\mu + \alpha x) = 0$$

$$\alpha = -1$$

So, if we write this $\mu x + \alpha x^2 = 0$ so, this is the αx^2 equal to 0. So, in this case our equation one dimensional equation becomes $\dot{x} = \mu x + \alpha x^2$. Now, putting this $\dot{x} = 0$ so, we have this equation so, from this equation we can find the equilibrium position so, in this case to find the equilibrium position this will give $\mu + \alpha x = 0$ so, if I will take this α so, depending on different value of α we can have a set of curves. So, let us take α equal to minus 1 so, this curve is shown in this figure.

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So, corresponding to alpha equal to minus 1 so, we have this equation \dot{x} equal to μx minus x square.

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Handwritten mathematical derivation on a yellow background:

$$\dot{x} = \mu x + \alpha x^2$$

$$\mu x + \alpha x^2 = 0$$

$$x(\mu + \alpha x) = 0$$

$$\alpha = -1$$

$$A = \mu + 2\alpha x$$

$$|A - \lambda I| = 0$$

$$\mu + 2\alpha x - \lambda = 0$$

$$\lambda = \mu + 2\alpha x$$

$$= \mu - 2x$$

$$\lambda = 0$$

$$\mu = x$$

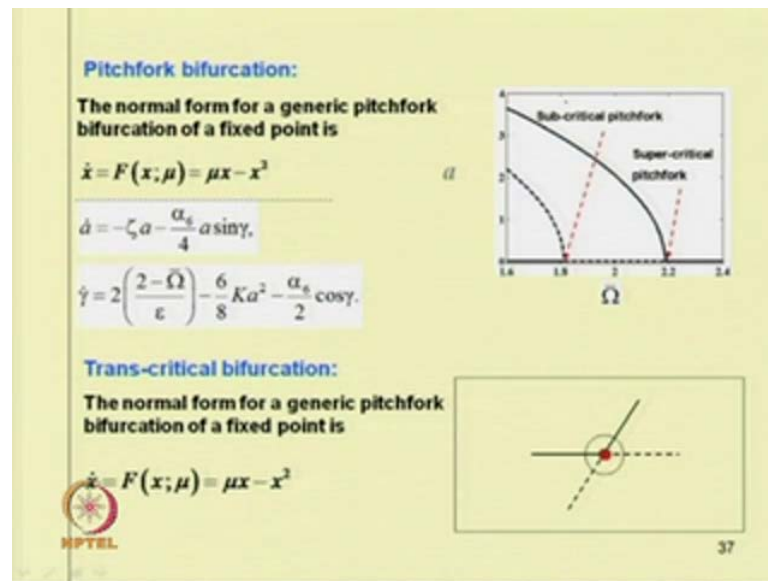
$$\mu - x = 0$$

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So, in this case the so, our Jacobian matrix so, to obtain this Jacobian matrix we will find this $\frac{df}{dx}$ the matrix with $\frac{df}{dx}$ at is as it is one dimensional so, we will have only 1 so, this becomes μ plus so, this becomes μ plus $2\alpha x$. So, as our A minus λI so, we have to make A minus λI equal to A minus λI determinant of A minus λI equal to 0. So, in this case as only we have one term so, we can write this μ plus $2\alpha x$ minus λ equal to 0 so, our λ becomes the Eigen value becomes μ plus $2\alpha x$. So, as α equal to minus 1 so, we can write this λ equal to μ minus $2x$ so, as λ equal to μ minus $2x$ so, depending on the value of μ we can obtain different value of x and we can study whether the branch is stable or unstable.

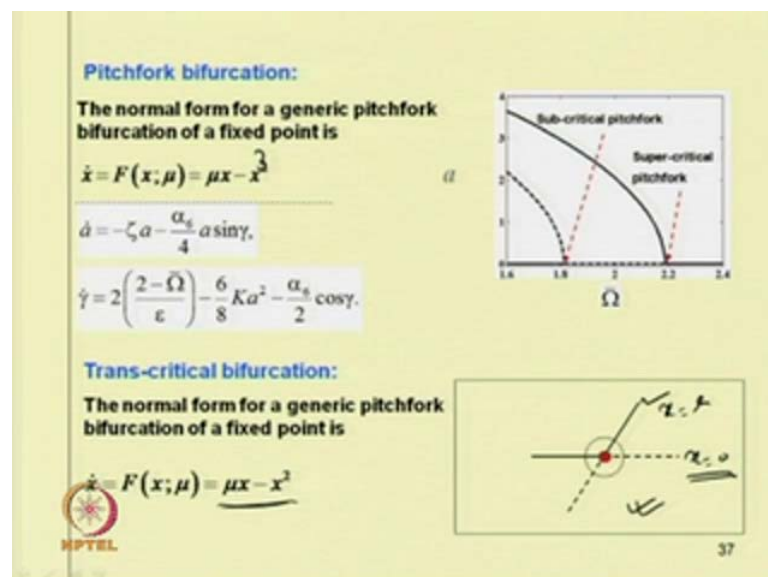
So, in this case we have seen this trivial solution so, one solution is so, x into μ plus αx equal to 0. So, the equilibrium position becomes x equal to equilibrium position becomes x equal to 0 so, that is the trivial solution and also we have a non trivial solution that is μ plus αx by putting this μ plus αx equal to 0 so, we have or this putting this α equal to minus 1 so, μ minus x equal to 0 or μ equal to x so, we have 2 points one is x equal to 0 and another one is μ equal to x equal to μ .

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So, we have these 2 x equal to mu. Now, we can plot so, from this plot we can see so, we have this trivial branch. So, if this trivial branch becomes stable so, if the Jacobian matrix has negative real parts similarly, so here one can observe that at this point at this so, from this branch so, if one solve this equation so, one can find the solution. So, in this case for pitchfork bifurcation it will be x dot will be equal to mu x minus x cube.

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$$\dot{x} = \mu + \alpha x^2$$

$$\mu + \alpha x^2 = 0$$

$$x(\mu + \alpha x) = 0$$

$$\alpha = -1$$

$$A = \mu + 2\alpha x$$

$$|A - \lambda I| = 0$$

$$\mu + 2\alpha x - \lambda = 0$$

$$\lambda = \mu + 2\alpha x$$

$$\lambda = \mu - 2\alpha x$$

$$x = 0$$

$$\mu = \lambda$$

$$\mu - x = 0$$

$$\mu = x$$

So, the previous things what we have taken this example μx plus αx^2 equal to 0 so, we have 2 solution that is x equal to 0 and x equal to μ so, this is the example of this trans-critical bifurcation so, in this case this is μx minus x square. So, this is the x equal to 0 line x equal to 0 line and x equal to the second one is x equal to μ so, this is x equal to μ line so, x equal to μ this is x equal to 0 so, this is pitch fork bifurcation and in case of this is trans-critical bifurcation.

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$$\dot{x} = \mu - x^2$$

$$x(\mu - x^2) = 0$$

$$x = 0 \rightarrow \text{Trivial state}$$

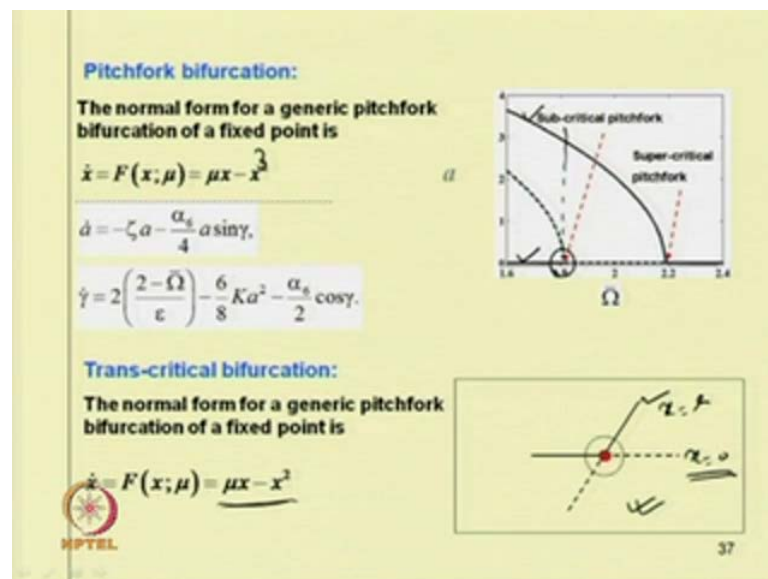
$$\mu - x^2 = 0 \rightarrow x^2 = \mu$$

$$x = \pm\sqrt{\mu} \rightarrow \text{Nontrivial}$$

$$A = \mu - 3x^2$$

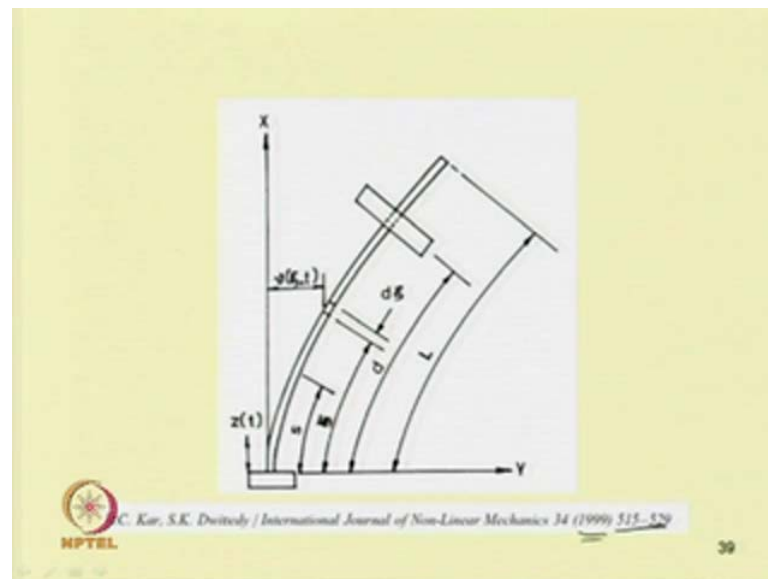
So, in case of the pitch fork bifurcation so, $\mu x - x^3$ will be equal to 0 so, \dot{x} equal to $\mu x - x^3$ so, taking this x common we have $\mu - x^2$ equal to 0 or x equal to 0 is the trivial state response trivial state response and $\mu - x^2$ equal to 0 or x^2 equal to $\pm \sqrt{\mu}$ so, x equal to $\pm \sqrt{\mu}$ so, from this x^2 equal to μ or x equal to $\pm \sqrt{\mu}$ so, this is non-trivial response. So, in this case one can find the Jacobian matrix from this. So, the Jacobian matrix A will be equal to so, this becomes $\mu - 3x^2$.

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Now, by substituting this value for example, taking this x equal to 0 that is, for the trivial state so, one can observe this branch to be stable this branch to be stable and this is unstable and this is stable. So, at this point so at this at this point one can observe that the response before that one has 3 solutions so, out of the 3 solution the upper branch and lower branch were stable and the middle branch is unstable and at this point so, this becomes the trivial state become unstable and one has a one has 2 solutions. So, out of in before this thing out of these 3 solutions, these 2 are stable and this is unstable and after this one has a unstable solution. So, these points where it tends to jump from this position to this upper one is known as sub critical pitch fork bifurcation point. Similarly, while sweeping down this thing so, we have this super critical pitch fork bifurcation point.

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So, next class we will study about the hopf bifurcation point. One more example one can see so, were this one can apply the stability analysis. For example, one can refer this paper by Kar and Dwivedy international journal of non-linear mechanics 1999 page number 315 to 515 to 529. So, this is where the non-linear dynamics of a slender beam is studied.

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$$\begin{aligned} \ddot{u}_n + 2\varepsilon\zeta_n\dot{u}_n + \omega_n^2 u_n - \varepsilon \sum_{m=1}^{\infty} f_{nm} u_m \cos \phi\tau \\ + \varepsilon \sum_{k=1}^{\infty} \sum_{l=1}^{\infty} \sum_{m=1}^{\infty} \{ \alpha_{klm}^n u_k u_l u_m + \beta_{klm}^n u_k \dot{u}_l \dot{u}_m \\ + \gamma_{klm}^n u_k u_l \ddot{u}_m \} = 0, \quad n = 1, 2, \dots, \infty \end{aligned}$$

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So, in this case equation can be written in this form and here 2 mode interactions has been taken.

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Principal parametric resonance ($\phi \approx 2\omega_1$)

$$\left. \begin{aligned} \phi &= 2\omega_1 + \varepsilon\sigma_1, \\ \omega_2 &= 3\omega_1 + \varepsilon\sigma_2. \end{aligned} \right\}$$


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So, to study the principal parametric resonance case by taking this phi external frequency nearly equal to twice the first mode frequency and using this detuning parameter sigma 1 and sigma 2 as the internal resonance is taken so, this second mode frequency was taken thrice near trice the first mode frequency so, this is for internal resonance and this is for external resonance. So, in this case one can get by using this method of multiple scales.

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$$\begin{aligned} &2\omega_1(\zeta_1 a_1 + a_1') - \frac{1}{2}\{f_{11}a_1 \sin 2\gamma_1 \\ &\quad + f_{12}a_2 \sin(\gamma_1 - \gamma_2)\} \\ &\quad + 0.25Q_{12}a_2a_1^2 \sin(3\gamma_1 - \gamma_2) = 0, \\ &2\omega_1a_1(\gamma_1' - \frac{1}{2}\sigma_1) - \frac{1}{2}\{f_{11}a_1 \cos 2\gamma_1 \\ &\quad + f_{12}a_2 \cos(\gamma_1 - \gamma_2)\} + \frac{1}{4}\sum_{j=1}^2 \alpha_{e1j}a_j^2a_1 \\ &\quad + \frac{1}{4}Q_{12}a_2a_1^2 \cos(3\gamma_1 - \gamma_2) = 0, \\ &2\omega_2(\zeta_2 a_2 + a_2') - \frac{1}{2}f_{21}a_1 \sin(\gamma_2 - \gamma_1) \\ &\quad + \frac{1}{4}Q_{21}a_1^3 \sin(\gamma_2 - 3\gamma_1) = 0, \\ &2\omega_2a_2(\gamma_2' + \sigma_2 - 1.5\sigma_1) - \frac{1}{2}f_{21}a_1 \cos(\gamma_2 - \gamma_1) \\ &\quad + \frac{1}{4}\sum_{j=1}^2 \alpha_{e2j}a_j^2a_2 + \frac{1}{4}Q_{21}a_1^3 \cos(\gamma_2 - 3\gamma_1) = 0 \end{aligned}$$


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So, one can get a set of 4 first order equation. So, previously we have seen 2 equations now, one can take a set of 4 equations.

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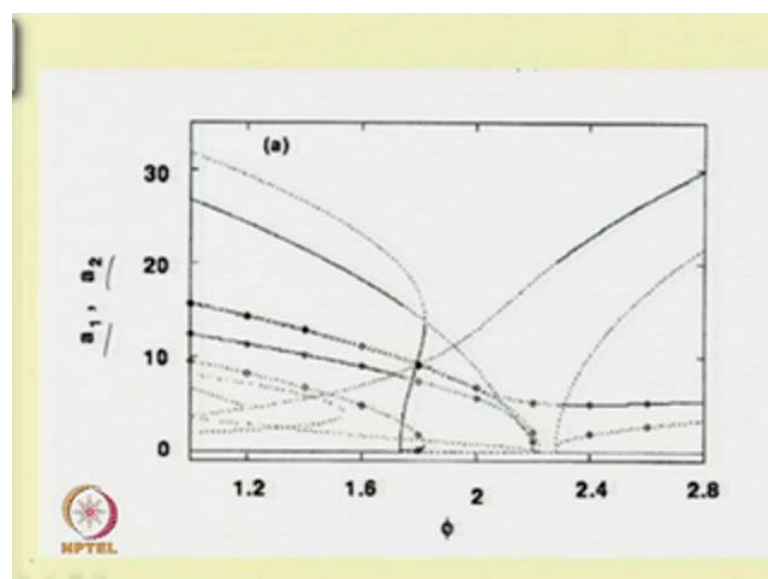
$$p_i = a_i \cos \gamma_i, \quad q_i = a_i \sin \gamma_i, \quad i = 1, 2$$

$$\left. \begin{aligned} & + \frac{1}{4} Q_{12} \{q_2(q_1^2 - p_1^2) + 2p_1 p_2 q_1\} \\ & - \frac{1}{4} \sum_{j=1}^2 \alpha_{e1j} q_1 (p_j^2 + q_j^2) = 0, \\ & 2\omega_1 (q_1' + \zeta_1 q_1) - \left(\omega_1 \sigma_1 + \frac{1}{2} f_{11} \right) p_1 - \frac{1}{2} f_{12} p_2 \\ & + \frac{1}{4} Q_{12} \{p_2(p_1^2 - q_1^2) + 2p_1 q_1 q_2\} \\ & + \frac{1}{4} \sum_{j=1}^2 \alpha_{e1j} p_1 (p_j^2 + q_j^2) = 0, \end{aligned} \right\}$$


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By perturbing this thing so, one can obtain the Jacobian matrix and one can study the stability. But, in this case one can see that while perturbing some of these perturbation will not will for the trivial state whose, overcome that thing so, one can use this transformation. So, here p_i equal to $a_i \cos \gamma_i$ and q_i equal to $a_i \sin \gamma_i$ has been taken. So, then one can write this equation by writing this transform form in terms of p and q so, one obtain these 4 equations so, using these 4 equations.

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Now, perturbing these 4 equations one can find the Jacobian matrix and slowly the stability. So, in this case one can plot this a 1 and a 2 so, this is your a 1 and a 2 have been plotted so, a 1 a 2 have been plotted.

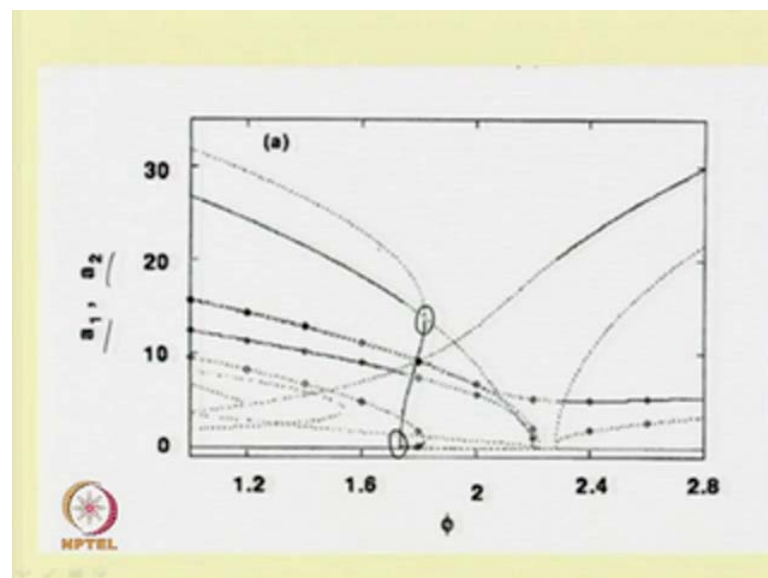
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$$\left. \begin{aligned} &2\omega_2(p'_2 + \zeta_2 p_2) + \frac{1}{2}f_{21}q_1 + \omega_2(3\sigma_1 - 2\sigma_2)q_2 \\ &\quad - \frac{1}{4}Q_{21}q_1(3p_1^2 - q_1^2) - \frac{1}{4}\sum_{j=1}^2 \alpha_{e2j}q_2(p_j^2 + q_j^2) = 0, \\ &2\omega_2(q'_2 + \zeta_2 q_2) - \frac{1}{2}f_{21}p_1 \\ &\quad - \omega_2(3\sigma_1 - 2\sigma_2)p_2 + \frac{1}{4}Q_{21}p_1(p_1^2 - 3q_1^2) \\ &\quad + \frac{1}{4}\sum_{j=1}^2 \alpha_{e2j}p_2(p_j^2 + q_j^2) = 0. \end{aligned} \right\}$$


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So, using by solving the set of equations by substituting first by substituting this p dash q dash p 1 dash q 1 dash p 2 dash q 2 dash equal to 0.

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So, one can plot this response. So, after plotting this response, one can study whether the branch is stable or unstable so by using the Jacobian matrix. So, in this way one can study the stability of a system. So, here one can observe so, this point of the pitch fork bifurcation point so, this point is a saddle node bifurcation point. So, later we will see that in this curve, there are also some points which bifurcation points those things we will study in the next class.

Thank you.