

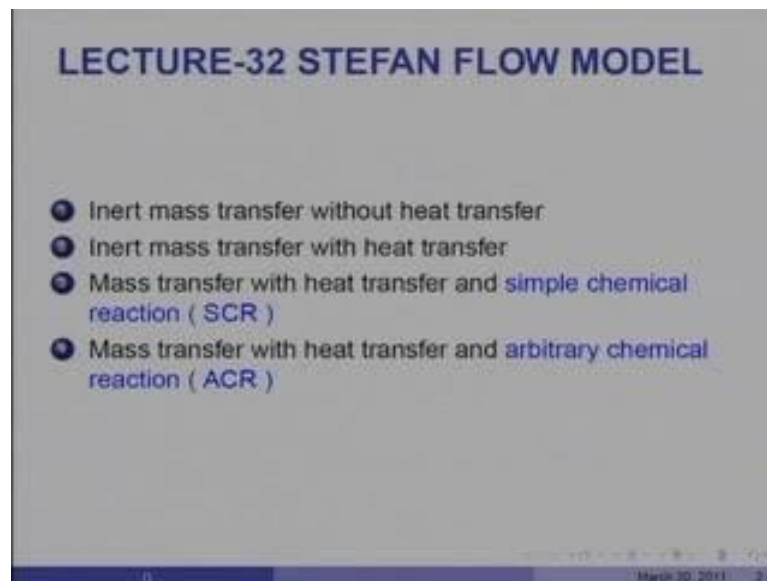
Convective Heat and Mass Transfer
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Module No. # 01
Lecture No. # 32
Stefan Flow Model

We are now familiar with the 3 simplifications of the boundary layer flow model. The first one was the Stefan flow model; second one was the Couette flow model and the third is the Reynolds flow model.

In today's lecture, I shall develop the Stefan flow model further for variety of mass transfer problems that we encounter in engineering practice.

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The types of problems that we encounter are as follows: first is the inert mass transfer without heat transfer or chemical reaction. This means, something like simple evaporation of water where the water and the environment are all at the same temperatures. So, there is no temperature gradient and as a result, there is no heat transfer. Nor is there any chemical reaction because, water simply evaporates without any chemical reaction and the mass transfer however takes place because of the concentration gradients.

The next would be **where** let us say, the water droplet or water itself in a lake is at a different temperature from that at the infinity state or the environment. In which case, there would be heat transfer either through the water or from the water to the environment depending on which temperature is larger. Then, we will move to the situation in which there would be chemical reaction or combustion in which, let us say, liquid droplet **or a** would be burning along with heat transfer from the environment to the droplet and finally, but there we would use what is called as simple chemical reaction and I will explain what that means.

But many a times, particularly when dealing with burning of solids, we need the reaction. Mechanism is so complex that we have to deal with that situation somewhat differently. So, I will develop the forms of mass transfer relationships that evolve for the 4 types of mass transfer problems.

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Reminder of Gov Eqns - L32(¹/₂₀)

In the Stefan flow model, under steady state

$$\frac{1}{A} \frac{d}{dy} [N_{\psi} y A] = \frac{1}{A} \frac{d}{dy} \left[\rho_m v A \psi - \Gamma_{\psi} A \frac{d\psi}{dy} \right] = S_{\psi}$$

ψ	Γ_{ψ}	S_{ψ}
ω_k	$\rho_m D$	R_k
T_{k1}	$\rho_m D$	0
h_m	$k_m / c p_m$	$- A^{-1} d(\sum_k A m_{y,k}'' h_k) / dy$

where \dot{Q}_{rad} is neglected and $m_{y,k}'' = -\rho_m D (d\omega_k / dy)$


As you recall, under steady state, the mass transfer equation would be something like this: $N_{\psi} y A$ equal to all that and ψ can stand for mass fraction or element mass fraction or enthalpy; the radiation term is of course a neglected in this energy equation and $m_{y,k}''$ as you will recall, is the diffusion mass flux as per the Fick's law of diffusion.

So, these are the source terms in each of these equations; R_k would be finite when there is a chemical reaction; if not, it will be 0.

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Inert MT without HT - 1 - L32($\frac{2}{20}$)

- 1. Consider evaporation of pure water through a stagnant column of air
- 2. Both water and air are at same temperature. Hence, no HT
- 3. Air does not dissolve in water
- 4. Steady state prevails. That is, water is supplied at the bottom at the evaporation rate. So, L is constant.
- 5. There are two species only. Air (a) - water vapour (v)



Two Governing equations are

$$\frac{d}{dy} [N_{a,y} A] = \frac{d}{dy} [N_{v,y} A] = 0$$

or,

$$\dot{m}_w = A_w N_w = A (N_{v,y} + N_{a,y}) = \text{const.}$$

But, in stagnant air,

$$\dot{m}_{a,w} = A N_{a,y} = A_w N_{a,w} = 0.$$

Also, $\omega_a + \omega_v = 1$.

So, let us consider the first type: inert mass transfer without heat transfer. Let us say I have a tank here with water in it and the water evaporates because in the infinity state the air is either dry or has relative humidity less than 100 percent. Therefore, there would be mass transfer from water to the infinity state. The column height that I have considered above the water is L and the air in the column is stagnant. Stefan flow model as you know, is primarily applied to diffusion mass transfer; both water and air are at the same temperature and therefore there is no heat transfer.

Air also does not dissolve in water; as such there will be no transfer of air from the considered phase into the water phase steady state prevails. That is, water is supplied at the bottom at the evaporation rate. What this means is that, somehow we have constructed an apparatus in which water is supplied at the same rate at which it is evaporating and therefore the column height L would remain constant or the water level will remain constant.

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Reminder of Gov Eqns - L32($\frac{1}{20}$)

In the Stefan flow model, under steady state

$$\frac{1}{A} \frac{d}{dy} [N_{v,y} A] = \frac{1}{A} \frac{d}{dy} \left[\rho_m v A \Psi - \Gamma_\psi A \frac{d\Psi}{dy} \right] = S_\psi$$

Ψ	Γ_ψ	S_ψ
ω_k	$\rho_m D$	R_k
$\eta_{v,s}$	$\rho_m D$	0
h_m	$k_m / c p_m$	$-A^{-1} d(\sum A m_{y,k} h_k) / dy$

where \dot{Q}_{rad} is neglected and $m''_{y,k} = -\rho_m D (d\omega_k / dy)$

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Inert MT without HT - 1 - L32($\frac{2}{20}$)

- 1. Consider evaporation of pure water through a stagnant column of air
- 2. Both water and air are at same temperature. Hence, no HT
- 3. Air does not dissolve in water
- 4. Steady state prevails. That is, water is supplied at the bottom at the evaporation rate. So, L is constant.
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Two Governing equations are

$$\frac{d}{dy} [N_{a,y} A] = \frac{d}{dy} [N_{v,y} A] = 0$$

or,

$$\dot{m}_w = A_w N_w = A (N_{v,y} + N_{a,y}) = \text{const.}$$

But, in stagnant air, $\dot{m}_{a,w} = A N_{a,y} = A_w N_{a,w} = 0$.

Also, $\omega_a + \omega_v = 1$.

Now, in this problem we have two species: one is the air a and the water vapour v and therefore the governing equation for this, because there is no mass there is no chemical reaction as you will see in the previous equation, $N_{v,y} A$ would be equal to 0 for both air and water. So, that is what I have written here; this means that the mass transfer rate \dot{m}_w in kg per second would be area A into N_w equal to A times $N_{v,y}$ plus $N_{a,y}$ equal to the constant because this is gradient, is 0.

(Refer Slide Time: 06:04)

Inert MT without HT - 2 - L32($\frac{3}{20}$)

● Therefore,

$$\dot{m}_w = \dot{m}_v = A (\rho_m V \omega_v - \rho_m D \frac{d\omega_v}{dy}) = A \rho_m V$$

or $\dot{m}_w \omega_v - \rho_m D A \frac{d\omega_v}{dy} = \dot{m}_w$

or $-\frac{d\omega_v}{1 - \omega_v} = \left(\frac{\dot{m}_w}{\rho_m D}\right) \frac{dy}{A}$

● If $A = A_w = \text{const}$, then $N_w = \dot{m}_w / A_w$ and integration from $y = 0$ to $y = L$ gives

$$N_w = \left(\frac{\rho_m D}{L}\right) \ln\left(\frac{1 - \omega_{v,w}}{1 - \omega_{v,\infty}}\right) - g_m^* \ln(1 + B_m) \text{ where}$$

$$B_m = \frac{\omega_{v,\infty} - \omega_{v,w}}{\omega_{v,w} - 1} \text{ and } g_m^* = \frac{\rho_m D}{L} \left(\frac{\text{kg}}{\text{m}^2 \cdot \text{s}}\right)$$


But, in stagnant air, \dot{m}_w is equal to 0 because there is no mass transfer of air in this. It is stagnant and therefore, $\omega_a + \omega_v$ is equal to 1. We note this and therefore in this equation you will see that N_w will be set to 0 and \dot{m}_w would be simply A times N_w and that is what I have written here (Refer Slide Time: 06:15).

So, \dot{m}_w will be \dot{m}_w of vapour only equal to A times N_w and N_w as you know is the convective mass flux plus diffusion mass flux and that would be equal to $A \rho_m V$. Or if I rearrange this, notice that $A \rho_m V$ is simply \dot{m}_w . Therefore, \dot{m}_w into ω_v minus $\rho_m D A$ into $d\omega_v$ by dy would be equal to \dot{m}_w or if I transfer this on the right hand side then, you will see it will be $1 - \omega_v$ equal to \dot{m}_w by $\rho_m D dy$ by A (Refer Slide Time: 06:50).

(Refer Slide Time: 06:53)

Inert MT without HT - 1 - L32($\frac{2}{20}$)

- 1 Consider evaporation of pure water through a stagnant column of air
- 2 Both water and air are at same temperature. Hence, no HT
- 3 Air does not dissolve in water
- 4 Steady state prevails. That is, water is supplied at the bottom at the evaporation rate. So, L is constant.
- 5 There are two species only. Air (a) - water vapour (v)



Two Governing equations are

$$\frac{d}{dy} [N_{a,y} A] = \frac{d}{dy} [N_{v,y} A] = 0$$

or,

$$\dot{m}_w = A_w N_w = A (N_{v,y} + N_{a,y}) = \text{const.}$$

But, in stagnant air,

$$\dot{m}_{a,w} = A N_{a,y} - A_w N_{a,w} = 0.$$

Also, $\omega_a + \omega_v = 1$.

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Inert MT without HT - 3 - L32($\frac{4}{20}$)

- 1 For a spherical droplet evaporation in stagnant surroundings, $A = 4\pi r^2$. Then,

$$-\frac{d\omega_v}{1-\omega_v} = \left(\frac{\dot{m}_w}{\rho_m D}\right) \frac{dr}{4\pi r^2}$$

- 2 Integration from $r = r_w$ (droplet radius) to $r = \infty$ gives

$$\ln\left(\frac{1-\omega_{v,\infty}}{1-\omega_{v,w}}\right) = \frac{\dot{m}_w}{4\pi\rho_m D r_w}$$

- 3 Hence

$$N_w = \frac{\dot{m}_w}{4\pi r_w^2} = \left(\frac{\rho_m D}{r_w}\right) \ln\left(\frac{1-\omega_{v,\infty}}{1-\omega_{v,w}}\right) = g_m^* \ln(1+B_m) \text{ where}$$

$$B_m = \frac{\omega_{v,\infty} - \omega_{v,w}}{\omega_{v,w} - 1} \text{ and } g_m^* = \frac{\rho_m D}{r_w} \left(\frac{\text{kg}}{\text{m}^2 \cdot \text{s}}\right)$$

If A is equal to A w is equal to constant, that is, if I take the constant area model then simply, N w which is equal to m dot w by A w and integration would give me rho m D by L ln 1 minus omega v infinity over omega 1 minus omega v w equal to g star m into ln 1 plus B m. This bracket 1 minus omega v infinity into 1 minus omega v w, can also be written as omega v infinity minus omega v w over omega v w minus 1 and g m star would be rho m D by L. It has the same units as the mass flux N w and this is a constant

and written in this form. B_m is called the driving force from mass transfer to occur and it is given in this fashion.

So, you get a very simple logarithmic formula for mass flux or the evaporation rate, instantaneous evaporation rate of water in the spherical system. Supposing, I have a droplet then, the surface area A would go on changing with the radius and A would be equal to $4\pi r^2$ as we go along and A would go on changing. So, A would be function of r if you like. In that case, you will see $d\omega_v / (1 - \omega_v)$ would equal $m \dot{w} \rho_m D / (4\pi r^2 dr)$. Integration from $r = r_w$ which is the droplet radius to $r = \infty$, gives $\ln(1 - \omega_v) / (1 - \omega_v)$ equal to this quantity with r_w in the denominator. Remember, $1/r$ at infinity would be 0 and therefore only r_w survives.

If I rearrange this equation it would be written in this fashion N_w will be equal to $m \dot{w}$ divided by area of the spherical droplet $4\pi r_w^2$, would be equal to $\rho_m D / r_w$ into this quantity (Refer Slide Time: 09:30) which again can be written as $1 / \ln(1 + B_m)$, where B_m is again as before $\omega_{B\infty} - \omega_{v,w}$ into $\omega_{v,w} - 1$ and g_m^* which is the coefficient. If you like a is $\rho_m D / r_w$ instead of l in the previous problem when the area is constant.

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Comments - L32($\frac{5}{20}$)

- 1 Both results show that in diffusion mass transfer $N_w = g_m^* \ln(1 + B_m)$
- 2 But, as $B_m \rightarrow 0$, $\ln(1 + B_m) \rightarrow B_m$ for both $\pm B_m$. Thus, the linear relation $N_w = g \times B_m$ holds only for very small B_m or N_w .
- 3 Negative B_m implies Condensation
- 4 Therefore, in general, we may write

$$N_w = g \times B_m \quad \text{with} \quad \frac{g}{g_m^*} = \frac{\ln(1 + B_m)}{B_m}$$

where g_m^* is value of g when $|B_m| \rightarrow 0$

- 5 This result has significance even in Conv MT

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So, we get mass flux in this. Again, logarithmic form, even area changes; so both the results show that in diffusion mass transfer, you get N_w equal to $g \star m \ln(1 + B_m)$.

But, as B_m tends to 0 you can check out on your pocket calculator that $\ln(1 + B_m)$ tends to B_m for both positive or negative B_m . Thus, the linear variation N_w into B_m holds only for very small mass transfer rate B_m or N_w whichever way you want to look at it. Typically, when B_m is of the order of 0.02 or less, this relationship holds very well and when B_m is negative then of course, it will imply condensation. When B_m is positive it would imply evaporation therefore, in general we may write N_w equal to $g B_m$ where g over $g \star m$ is equal to $\ln(1 + B_m)$ by B_m and where $g \star$ is the value of g when B_m tends to 0.

Now, although this result has been found **from** for diffusion mass transfer, we shall later on show that the result has significance even in convective mass transfer. Let us now consider the case of inert mass transfer with heat transfer in which case let us say, the water and the environment are at different temperatures.

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Inert MT with HT - 1 - L32($\frac{6}{20}$)

Let the air in the previous example be at $T_\infty > T_w$ where T_w is the temperature of the water surface. Then, under steady state, besides species conservation, energy eqn is

$$\frac{d(N_{v,y} h_m A)}{dy} = \frac{d}{dy} \left[A(k_m \frac{dT}{dy} + \rho_m D \left\{ \frac{d\omega_v}{dy} h_v + \frac{d\omega_a}{dy} h_a \right\} \right]$$

where $h_m = \omega_v h_v + (1 - \omega_v) h_a$, $h_v = c_{p,v} (T - T_{ref}) + \lambda_{ref}$, $h_a = c_{p,a} (T - T_{ref})$ and $c_{p,m} = \omega_v c_{p,v} + (1 - \omega_v) c_{p,a}$. λ_{ref} is latent heat at T_{ref} .

Further,

$$k_m \frac{dT}{dy} = \rho_m \alpha_m c_{p,m} \frac{dT}{dy} = \rho_m \alpha_m \left[\omega_v \frac{dh_v}{dy} + \omega_a \frac{dh_a}{dy} \right]$$

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Reminder of Gov Eqns - L32($\frac{1}{20}$)

In the Stefan flow model, under steady state

$$\frac{1}{A} \frac{d}{dy} [N_{v,y} A] = \frac{1}{A} \frac{d}{dy} \left[\rho_m v A \Psi - \Gamma_\Psi A \frac{d\Psi}{dy} \right] = S_\Psi$$

Ψ	Γ_Ψ	S_Ψ
ω_k	$\rho_m D$	R_k
$T_{h,v}$	$\rho_m D$	0
h_m	$k_m/c\rho_m$	$-A^{-1} d(\sum_k A m_{y,k}'' h_k)/dy$

where \dot{Q}_{rad} is neglected and $m_{y,k}'' = -\rho_m D (d\omega_k/dy)$

So, let us say in temperature the environment is greater than that of the surface where T_w is the temperature of the water surface. Then, under steady state besides species conservation equation, we must now invoke the energy equation and that **for** as you will see from our first line here we are invoking this equation.

(Refer Slide Time: 11:36)

Inert MT with HT - 1 - L32($\frac{6}{20}$)

Let the air in the previous example be at $T_\infty > T_w$ where T_w is the temperature of the water surface. Then, under steady state, besides species conservation, energy eqn is

$$\frac{d(N_{v,y} h_m A)}{dy} = \frac{d}{dy} \left[A(k_m \frac{dT}{dy} + \rho_m D \left\{ \frac{d\omega_v}{dy} h_v + \frac{d\omega_a}{dy} h_a \right\}) \right]$$

where $h_m = \omega_v h_v + (1 - \omega_v) h_a$, $h_v = c_{p,v} (T - T_{ref}) + \lambda_{ref}$, $h_a = c_{p,a} (T - T_{ref})$ and $c_{p,m} = \omega_v c_{p,v} + (1 - \omega_v) c_{p,a}$. λ_{ref} is latent heat at T_{ref} .

Further,

$$k_m \frac{dT}{dy} = \rho_m \alpha_m c_{p,m} \frac{dT}{dy} = \rho_m \alpha_m \left[\omega_v \frac{dh_v}{dy} + \omega_a \frac{dh_a}{dy} \right]$$

So $d N v y h m A$ divided by dy would be d by dy of $A k m dT dy$ which is the conduction heat transfer plus this is the Fick's law of diffusion, multiplied by h_k which is in this case would be ω_v by dy h of vapour plus $d \omega_a$ by dy h of air and the

mixture enthalpy in this case would be the mass fraction of water vapour into enthalpy water vapor plus 1 minus omega v which is the mass fraction of air into enthalpy of air. h v would be given by specific heat of vapor into T minus T ref plus lambda ref which is the latent heat at temperature T ref and h a would have only the sensible part C p a into T minus T ref. The mixture specific heat could be simply omega v into C p v plus omega air into C p a; so that is what this formula **end**.

Now, let us look at this term k m dT dy; now k m can be written as rho m alpha m C p m dT by dy and that is equal to rho m alpha m. If I absorb C p m dT dy and put C p m equal to all this, you will see that you get omega v into d h v by dy plus omega a into d h a by dy.

(Refer Slide Time: 13:13)

Inert MT with HT - 2 - L32($\frac{7}{20}$)
 Substitution gives

$$\frac{d(N_{v,y} h_m A)}{dy} = \frac{d}{dy} \left[\rho_m A \alpha_m \left(\omega_v \frac{d h_v}{dy} + \omega_a \frac{d h_a}{dy} \right) \right] + \frac{d}{dy} \left[\rho_m A D \left(\frac{d \omega_v}{dy} h_v + \frac{d \omega_a}{dy} h_a \right) \right]$$

We now define Schmidt No (Sc) = ν/D and Lewis No (Le) = $Pr/Sc = D/\alpha$. For **gaseous mixtures**, Le \approx 1 assumption (or, D = α) is routinely made. Then

$$\frac{d(N_{v,y} h_m A)}{dy} = \frac{d}{dy} \left[\Gamma_{m,h} A \left\{ \frac{d}{dy} (\omega_v h_v + \omega_a h_a) \right\} \right] = \frac{d}{dy} \left[\Gamma_{m,h} A \frac{d h_m}{dy} \right]$$

where $\Gamma_{m,h} = \rho_m D = \rho_m \alpha_m$

This result... this is an important one which we are now going to substitute here. So, you will get d N v y h m A by dy equal to d by dy of rho m A alpha m omega v d h v by dy plus omega d h a by dy plus the second term which is the Fick's law of diffusion term which carries with it enthalpy of the species.

Now, in mass transfer problem it is common to define Schmidt number as nu by diffusivity and we define Lewis number as Prandtl divided by Schmidt number which is equal to diffusivity of mass divided by diffusivity of heat.

Now, for gaseous mixtures, Lewis number is very close to 1 for example, as you know Prandtl number is about 0.7, Schmidt number would be about 0.672.68 in this kind of system. So, in effect Lewis number Le can be taken as very nearly 1 which implies that D is equal to α . If I make that assumption, then you will see that and replace these ρ_m into D into ρ_m into α as γ . As I have defined here (Refer Slide Time:15:00), $\gamma_m h$ equal to $\rho_m D$ equal to $\rho_m \alpha$. Then you will see this becomes $\gamma_m h$ into A plus d by dy of simply product of $\omega_a v$ plus $\omega_a h_a$, which is nothing but the mixture enthalpy $d h_m$ by dy so we essentially get d by dy $N v$ by $h_m A$ equal to d by dy of this.

(Refer Slide Time: 15:14)

Inert MT with HT - 3 - L32($\frac{8}{20}$)

- Now, from species conservation, $N_{v,y} = N_w$. Hence, the last result can also be written as

$$\frac{d}{dy} \left[A \left\{ N_w (h_m - h_{m,T}) - \Gamma_{m,h} \frac{d}{dy} (h_m - h_{m,T}) \right\} \right] = 0$$

where $h_{m,T} = c_{p,l} (T_T - T_{ref})$ is the specific enthalpy of the make-up water deep inside the neighbouring phase. $c_{p,l}$ is liquid specific heat. This is again an eqn in conserved property ($h_m - h_{m,T}$)

- Integration as before gives

$$N_w = g_{mh}^* \ln \left[\frac{h_{m,\infty} - h_{m,T}}{h_{m,w} - h_{m,T}} \right] = g_{mh}^* \ln (1 + B_h) \text{ where}$$

$$B_h = \frac{h_{m,\infty} - h_{m,w}}{h_{m,w} - h_{m,T}} \text{ and } g_{mh}^* = \frac{\Gamma_{mh}}{r_w} \text{ or } \frac{\Gamma_{mh}}{L}$$

Now from species conservation we have learnt that $N v y$ remains constant which is equal to $N w$. This the mass transfer as the surface itself remains constant throughout in a constant area problem. Then hence, the last result can be written as d by dy of $N N w$ h_m minus $h_m T$ minus $\gamma_m h$ into d by dy h_m minus $h_m T$ where $h_m T$ is simply the enthalpy of the transferred substance and it is a constant.

(Refer Slide Time: 15:32)

Inert MT with HT - 2 - L32($\frac{7}{20}$)

Substitution gives

$$\frac{d(N_{v,y} h_m A)}{dy} = \frac{d}{dy} \left[\rho_m A \alpha_m \left(\omega_v \frac{dh_v}{dy} + \omega_a \frac{dh_a}{dy} \right) \right] + \frac{d}{dy} \left[\rho_m A D \left(\frac{d\omega_v}{dy} h_v + \frac{d\omega_a}{dy} h_a \right) \right]$$

We now define Schmidt No (Sc) = ν/D and Lewis No (Le) = $Pr/Sc = D/\alpha$. For **gaseous mixtures** , $Le \approx 1$ assumption (or, $D = \alpha$) is routinely made. Then

$$\frac{d(N_{v,y} h_m A)}{dy} = \frac{d}{dy} \left[\Gamma_{m,h} A \left\{ \frac{d}{dy} (\omega_v h_v + \omega_a h_a) \right\} \right] = \frac{d}{dy} \left[\Gamma_{m,h} A \frac{dh_m}{dy} \right]$$

where $\Gamma_{m,h} = \rho_m D = \rho_m \alpha_m$

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Inert MT with HT - 3 - L32($\frac{8}{20}$)

Now, from species conservation, $N_{v,y} = N_w$. Hence, the last result can also be written as

$$\frac{d}{dy} \left[A \left\{ N_w (h_m - h_{m,T}) - \Gamma_{m,h} \frac{d}{dy} (h_m - h_{m,T}) \right\} \right] = 0$$

where $h_{m,T} = c_{p,l} (T_T - T_{ref})$ is the specific enthalpy of the make-up water deep inside the neighbouring phase. $c_{p,l}$ is liquid specific heat. This is again an eqn in **conserved property** ($h_m - h_{m,T}$)

Integration as before gives

$$N_w = g_{mh}^* \ln \left[\frac{h_{m,\infty} - h_{m,T}}{h_{m,w} - h_{m,T}} \right] = g_{mh}^* \ln (1 + B_h) \text{ where}$$

$$B_h = \frac{h_{m,\infty} - h_{m,w}}{h_{m,w} - h_{m,T}} \text{ and } g_{mh}^* = \frac{\Gamma_{mh}}{r_w} \text{ or } \frac{\Gamma_{mh}}{L}$$

So, all I have done is really added or subtracted h_m minus h_T and here h_m minus h_T and therefore, I have made really no change. So, h_m T would be the C_p of the liquid into temperature of the transfer substance which is water in this particular example minus T_{ref} is the specific enthalpy of the makeup water deep inside the neighboring phase. $C_{p,l}$ is the specific heat of the liquid. Now, this is again a conserved property equation in h_m minus h_m T . So, it is exactly same as the equation we had for ω_v and therefore its solution 2 would be identical m_w equal to $g_{mh}^* \ln (h_m \text{ infinity} - h_m T$

over $h_{m,w} - h_{m,T}$ which I would write as g_{mh} into $\ln(1 + B_h)$. In this case, where B_h is now formed from $h_{m,\infty} - h_{m,w}$ over $h_{m,w} - h_{m,T}$ and g_{mh} would be γ_{mh} over r_w in case of a spherical system and this would be the case in case of a linear system (Refer Slide Time: 16:50).

So, we find that the logarithmic form is again retrieved from the solution of the energy equation because just as we had recovered the logarithmic form in case of mass transfer without heat transfer

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Comments - 1 - L32($\frac{9}{20}$)

- Because $Le = 1$, $\Gamma_{mh} = \Gamma_m = \Gamma_h$. Hence,

$$B_h = \frac{h_{m,\infty} - h_{m,w}}{h_{m,w} - h_{m,T}} = B_m = \frac{\omega_{v,\infty} - \omega_{v,w}}{\omega_{v,w} - 1}$$
- This relation provides the important link between $\omega_{v,w}$ and T_w because $h_{m,w} = h_{v,w} \omega_{v,w} + h_{a,w} (1 - \omega_{v,w})$. Hence, taking $T_{ref} = 0$,

$$h_{m,w} = c_{p,a} T_w + [(c_{p,v} - c_{p,a}) T_w + \lambda_{ref}] \omega_{v,w}$$
- Hence, for given T_∞ and T_T , the $B_m = B_h$ relation will iteratively give $\omega_{v,w} \sim T_w$. If $Le = 1$ assumption is correct, this relation must be same as the saturation equilibrium relationship corresponding to $RH = 100\%$ of the psychrometric chart.

Now, since Lewis number is equal to 1, γ_{mh} would be equal to γ_m and γ_h because thermal diffusivity is equal to mass diffusivity and hence you will see B_h which is $h_{m,\infty} - h_{m,w}$ or $h_{m,w} - h_{m,T}$ would also be equal to $\omega_{v,\infty} - \omega_{v,w}$ over $\omega_{v,w} - 1$. Now, this relation provides the important link **with** between the mass fraction of vapor at the wall and the temperature of the wall. Remember, when the temperature of the surroundings is at T_∞ and the liquid which is T ; T is at some other temperature we still do not know what the surface temperature T_w will be and we need to determine that in such problems.

So, how do we determine that? We use this relationship that is, B_m is equal to B_h and $h_{m,w}$ would be then $h_{v,w} \omega_{v,w} + h_{a,w} (1 - \omega_{v,w})$ and if I take T_{ref}

equal to 0 then $h_m w$ will be simply $C_p a$ into T_w into $C_p v$ minus $C_p a$ T_w plus λ at 0 degree λ at T_{ref} equal to 0 into $\omega v w$.

Hence, for a given T_{infty} and T_w the B_m equal to B_h relationship will iteratively give $\omega v n$ ω and T_w . So, what one does is, one simply assumes a value of T_w h_m infinity of course, can be obtained because you know already ωv infinity. You know T_{infty} and therefore that can be obtained; these two can be obtained as we saw on the last slide. This can be obtained knowing T_w . Only thing is, you do not know $\omega v w$. So, how do we get that $\omega v w$? We can get that by saying that at the surface of the water saturation conditions would prevail and corresponding to RH equal to 100 percent. Therefore, that value of $\omega v w$ can be noted either from the psychrometric chart or you can also use steam tables in which case, you will get a partial pressure **so that** from which you can recover the mass fraction of the wall.

If, after assuming T_w you have determined $\omega v w$ in this way, you substitute in the B_m expression; if you find that B_h is equal to B_m then obviously, your choice of T_w was correct and therefore you already have T_w $\omega v w$ relationship. If however, the 2 equations do not balance then, you must change the value of T_w till you get balance between B_m and B_h . So, iterations are involved in discovering temperature T_w .

(Refer Slide Time: 20:05)

Comments - 2 - L32($\frac{10}{20}$)
 For air-water vapour mixture, saturation condition is correlated as

$$\omega_{v,w} \approx 3.416 \times 10^{-3} + (2.7308 \times 10^{-4}) T_w + (1.372 \times 10^{-5}) T_w^2 + (8.2516 \times 10^{-6}) T_w^3 - (6.9092 \times 10^{-9}) T_w^4 + (3.5313 \times 10^{-10}) T_w^5 - (3.7037 \times 10^{-12}) T_w^6 + (6.1923 \times 10^{-15}) T_w^7 + (9.9349 \times 10^{-17}) T_w^8$$

where $-20 < T_w (C) < 100$.
 For the evaporating fuel, $\omega_{v,w} \sim T_w$ relation must be determined from Clausius-Clapeyron equation. Thus

$$\omega_{v,w} = \left(\frac{p_{sat}(T_w)}{p_{tot}} \right) \times \left(\frac{M_v}{M_{mix}} \right) = x_{v,w} \left(\frac{M_v}{M_{mix}} \right)$$

$$x_{v,w} = \exp \left\{ - \frac{h_{fg}}{R_g} \left(\frac{1}{T_w} - \frac{1}{T_{bp}} \right) \right\} \rightarrow T_{bp} \equiv (\text{Boiling Point})$$

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Now, in order to help, if you are doing something on the computer then for air water vapor mixture saturation condition, that is the saturation curve on the psychrometric chart is given like this; where this is the T drizzle and this is the specific humidity and you have the 100 percent RH line (Refer Slide Time: 20:15). The values corresponding to this have been correlated here in ω as a function of T for the range of T minus 20 to 100. For computer applications, in order to help iterations you one can use this relationship or simply try by hand; this is just for your information.

So, now of course, when we consider a fuel we do not know the ω T relationship by way of a psychrometric chart or anything like that and therefore, such a relationship must be determined from what is called that the Clausius-Clapeyron equation. In that case, you will see that ω is related to X into molecular weight of vapor divided by molecular weight of the mixture and X of course, is equal to P saturation at T divided by P tot. X would be given by exponential of minus h fg divided by R g into 1 over T minus T bp for all liquid fuels. Typically, you will have a boiling point known and h fg known and therefore X can be recovered for a given T.

So, one try is in such cases assume at T recover or calculate X; from that, you calculate ω and check whether B m is equal to B h as on the previous slide or otherwise go on changing T till you get convergence.

(Refer Slide Time: 22:15)

MT with HT and SCR - 1 - L32($\frac{11}{20}$)

- Consider highly volatile liquid fuel that burns in the considered phase according to Simple Chemical Reaction (SCR)

$$1 \text{ kg of fuel} + r_{st} \text{ kg of } O_2 = (1 + r_{st}) \text{ kg of products}$$

where r_{st} is stoichiometric ratio for the fuel.

- We have 3 species, fuel, oxygen and products. Hence,

$$\frac{d}{dy} \left[A (N_w \omega_{fu} - \rho_m D \frac{d \omega_{fu}}{dy}) \right] = - |R_{fu}| A$$

$$\frac{d}{dy} \left[A (N_w \omega_{O_2} - \rho_m D \frac{d \omega_{O_2}}{dy}) \right] = - |R_{O_2}| A$$

$$\frac{d}{dy} \left[A (N_w \omega_{pr} - \rho_m D \frac{d \omega_{pr}}{dy}) \right] = R_{pr} A$$

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Now, I turn to the problem of mass transfer with heat transfer and simple chemical reaction. Now, what is simple chemical reaction? Let us consider highly volatile liquid fuel that burns in the considered phase according to the Simple Chemical Reaction SCR. Liquids usually burn by first evaporating in the gas phase without any change in composition. That is, the chemical formula of the fuel does not change when it comes out in the form of a vapor; but then, it burns in the vapor phase, in the gaseous phase as the homogeneous combustion and we say that simple chemical from reaction simply implies that 1 kilogram of fuel combines with r_{st} kilograms of oxygen to give you 1 plus r_{st} kilograms of products. Now, where r_{st} is the stoichiometric ratio of the fuel and you must have **run** from your stoichiometry, how to evaluate say for example, hydrocarbon fuel? How to evaluate the value of r_{st} ?

In this problem therefore, we have 3 species. The fuel specie, the oxygen specie and the product which is itself a mixture but it is a product specie which we take it as a single specie. Then we will have 3 equations and because the chemical reaction is present, you will have convective flux and a diffusion flux d by dy of **wood**, equal rate of depletion of fuel. We therefore, I have said magnitude of R_{fu} with a negative sine likewise, oxygen 2 would deplete minus R_o . Whereas, the product would increase and therefore it has a plus sign in front of here so we have 3 equations because we have 3 species - 3 mass transfer equations.

(Refer Slide Time: 24:14)

MT with HT and SCR - 2 - L32(¹²/₂₀)

- Adding the 3 eqns, we retrieve bulk mass conservation. Hence, $\sum_k R_k = 0$ and $\sum_k \dot{m}_{y,k} = 0$.
- Thus, SCR implies $R_{O_2} = r_{st} R_{fu}$, $R_{pr} = -(1 + r_{st}) R_{fu}$ and $\dot{m}_{O_2} = r_{st} \dot{m}_{fu}$, $\dot{m}_{pr} = -(1 + r_{st}) \dot{m}_{fu}$.
- Further dividing 2nd eqn by r_{st} and third eqn by $(1 + r_{st})$, and subtracting from 1st eqn, it follows that

$$\frac{d}{dy} \left[A \left(N_w \phi - \Gamma_m \frac{d\phi}{dy} \right) \right] = 0$$

$$\phi = \omega_{fu} - \frac{\omega_{O_2}}{r_{st}} = \omega_{fu} + \frac{\omega_{pr}}{1 + r_{st}} \quad \text{Conserved Property}$$

$$\text{soln: } N_w = g_m^* \ln \left[\frac{\phi_\infty - \phi_T}{\phi_w - \phi_T} \right] = g_m^* \ln(1 + B_m) \quad \text{where}$$

$$B_m = \frac{\phi_\infty - \phi_w}{\phi_w - \phi_T} \quad \text{and} \quad g_m^* = \frac{\Gamma_m}{r_w} \quad \text{or} \quad \frac{\Gamma_m}{L}$$

(Refer Slide Time: 24:20)

MT with HT and SCR - 1 - L32(11/20)

- Consider highly volatile liquid fuel that burns in the considered phase according to Simple Chemical Reaction (SCR)

$$1 \text{ kg of fuel} + r_{st} \text{ kg of } O_2 = (1 + r_{st}) \text{ kg of products}$$

where r_{st} is stoichiometric ratio for the fuel.

- We have 3 species, fuel, oxygen and products. Hence,

$$\frac{d}{dy} \left[A (N_w \omega_{fu} - \rho_m D \frac{d \omega_{fu}}{dy}) \right] = - |R_{fu}| A$$

$$\frac{d}{dy} \left[A (N_w \omega_{O_2} - \rho_m D \frac{d \omega_{O_2}}{dy}) \right] = - |R_{O_2}| A$$

$$\frac{d}{dy} \left[A (N_w \omega_{pr} - \rho_m D \frac{d \omega_{pr}}{dy}) \right] = R_{pr} A$$

If we add the 3, we must retrieve the bulk mass conservation equation because the addition of the fuels, fuel oxygen and product; some of these will be equal to 1 and when all these are sum that will be equal to 1, so $D \cdot 1$ by dy would be 0.

(Refer Slide Time: 24:49)

MT with HT and SCR - 2 - L32(12/20)

- Adding the 3 eqns, we retrieve bulk mass conservation. Hence, $\sum_k R_k = 0$ and $\sum_k \dot{m}_{y,k} = 0$
- Thus, SCR implies $R_{O_2} = r_{st} R_{fu}$, $R_{pr} = -(1 + r_{st}) R_{fu}$ and $\dot{m}_{O_2} = r_{st} \dot{m}_{fu}$, $\dot{m}_{pr} = -(1 + r_{st}) \dot{m}_{fu}$
- Further dividing 2nd eqn by r_{st} and third eqn by $(1 + r_{st})$, and subtracting from 1st eqn, it follows that

$$\frac{d}{dy} \left[A (N_w \Phi - \Gamma_m \frac{d \Phi}{dy}) \right] = 0$$

$$\Phi = \omega_{fu} - \frac{\omega_{O_2}}{r_{st}} = \omega_{fu} + \frac{\omega_{pr}}{1 + r_{st}} \quad \text{Conserved Property}$$

soln: $N_w = g_m^* \ln \left[\frac{\Phi_\infty - \Phi_T}{\Phi_w - \Phi_T} \right] = g_m^* \ln (1 + B_m)$ where

$$B_m = \frac{\Phi_\infty - \Phi_w}{\Phi_w - \Phi_T} \quad \text{and} \quad g_m^* = \frac{\Gamma_m}{r_w} \quad \text{or} \quad \frac{\Gamma_m}{L}$$

So, some of the diffusions, fluxes would add to 0 and therefore you will be having $A \cdot N_w \cdot \frac{d}{dy}$ of N_w into sum of all these quantities and therefore, the sum of the R_k must also be equal to 0 because the bulk mass conservation.

So, therefore SCR simply implies that R_{O_2} the oxygen depletion rate is equal to r_{st} times fuel depletion rate. On the other hand, the product generation rate would be minus $1 + r_{st}$ times fuel depletion rate. Likewise, the diffusion of oxygen would be r_{st} times $m \dot{f}$ double preview and product will be minus $1 + r_{st}$ m double dot fu.

(Refer Slide Time: 25:26)

MT with HT and SCR - 1 - L32($\frac{11}{20}$)

- Consider highly volatile liquid fuel that burns in the considered phase according to Simple Chemical Reaction (SCR)

$$1 \text{ kg of fuel} + r_{st} \text{ kg of } O_2 = (1 + r_{st}) \text{ kg of products}$$

where r_{st} is stoichiometric ratio for the fuel.

- We have 3 species, fuel, oxygen and products. Hence,

$$\frac{d}{dy} \left[A (N_w \omega_{fu} - \rho_m D \frac{d \omega_{fu}}{dy}) \right] = - |R_{fu}| A$$

$$\frac{d}{dy} \left[A (N_w \omega_{O_2} - \rho_m D \frac{d \omega_{O_2}}{dy}) \right] = - |R_{O_2}| A$$

$$\frac{d}{dy} \left[A (N_w \omega_{pr} - \rho_m D \frac{d \omega_{pr}}{dy}) \right] = R_{pr} A$$

If I now divide this equation by r_{st} and then subtract the resulting equation, I divide by r_{st} throughout and which is a constant and subtract that equation from this equation, then you will see that R_{fu} minus R_{O_2} divide by r_{st} would be 0.

(Refer Slide Time: 25:48)

MT with HT and SCR - 2 - L32($\frac{12}{20}$)

- Adding the 3 eqns, we retrieve bulk mass conservation. Hence, $\sum_k R_k = 0$ and $\sum_k \dot{m}_{y,k} = 0$
- Thus, SCR implies $R_{O_2} = r_{st} R_{fu}$, $R_{pr} = -(1 + r_{st}) R_{fu}$ and $\dot{m}_{O_2} = r_{st} \dot{m}_{fu}$, $\dot{m}_{pr} = -(1 + r_{st}) \dot{m}_{fu}$.
- Further dividing 2nd eqn by r_{st} and third eqn by $(1 + r_{st})$, and subtracting from 1st eqn, it follows that

$$\frac{d}{dy} \left[A (N_w \Phi - \Gamma_m \frac{d \Phi}{dy}) \right] = 0$$

$$\Phi = \omega_{fu} - \frac{\omega_{O_2}}{r_{st}} = \omega_{fu} + \frac{\omega_{pr}}{1 + r_{st}} \quad \text{Conserved Property}$$

soln: $N_w = g_m^* \ln \left[\frac{\Phi_{\infty} - \Phi_T}{\Phi_w - \Phi_T} \right] = g_m^* \ln (1 + B_m)$ where

$$B_m = \frac{\Phi_{\infty} - \Phi_w}{\Phi_w - \Phi_T} \quad \text{and} \quad g_m^* = \frac{\Gamma_m}{r_w} \quad \text{or} \quad \frac{\Gamma_m}{L}$$

(Refer Slide Time: 26:06)

MT with HT and SCR - 1 - L32(¹¹/₂₀)

- Consider highly volatile liquid fuel that burns in the considered phase according to Simple Chemical Reaction (SCR)

$$1 \text{ kg of fuel} + r_{st} \text{ kg of } O_2 = (1 + r_{st}) \text{ kg of products}$$

where r_{st} is stoichiometric ratio for the fuel.

- We have 3 species, fuel, oxygen and products. Hence,

$$\frac{d}{dy} \left[A (N_w \omega_{fu} - \rho_m D \frac{d \omega_{fu}}{dy}) \right] = - |R_{fu}| A$$

$$\frac{d}{dy} \left[A (N_w \omega_{O_2} - \rho_m D \frac{d \omega_{O_2}}{dy}) \right] = - |R_{O_2}| A$$

$$\frac{d}{dy} \left[A (N_w \omega_{pr} - \rho_m D \frac{d \omega_{pr}}{dy}) \right] = R_{pr} A$$

(Refer Slide Time: 26:14)

MT with HT and SCR - 2 - L32(¹²/₂₀)

- Adding the 3 eqns, we retrieve bulk mass conservation. Hence, $\sum_k R_k = 0$ and $\sum_k \dot{m}_{y,k} = 0$
- Thus, SCR implies $R_{O_2} = r_{st} R_{fu}$, $R_{pr} = -(1 + r_{st}) R_{fu}$ and $\dot{m}_{O_2} = r_{st} \dot{m}_{fu}$, $\dot{m}_{pr} = -(1 + r_{st}) \dot{m}_{fu}$
- Further dividing 2nd eqn by r_{st} and third eqn by $(1 + r_{st})$, and subtracting from 1st eqn, it follows that

$$\frac{d}{dy} \left[A (N_w \phi - \Gamma_m \frac{d \phi}{dy}) \right] = 0$$

$$\phi = \omega_{fu} - \frac{\omega_{O_2}}{r_{st}} = \omega_{fu} + \frac{\omega_{pr}}{1 + r_{st}} \quad \text{Conserved Property}$$

soln : $N_w = g_m^* \ln \left[\frac{\phi_w - \phi_T}{\phi_w - \phi_T} \right] = g_m^* \ln (1 + B_m)$ where

$$B_m = \frac{\phi_w - \phi_w}{\phi_w - \phi_T} \quad \text{and} \quad g_m^* = \frac{\Gamma_m}{r_w} \text{ or } \frac{\Gamma_m}{L}$$

As a result, you will get an equation which is like this $\frac{d}{dy} [A N_w \phi - \Gamma_m \frac{d \phi}{dy}] = 0$ and ϕ will be $\omega_{fu} - \frac{\omega_{O_2}}{r_{st}}$. Likewise, if I divide this third equation by $1 + r_{st}$ throughout **n** add then again the right hand side would be 0 and I will have a ϕ which **you** will stand for $\omega_{fu} + \frac{\omega_{pr}}{1 + r_{st}}$. We said that any equation of this form with the 0 source term implies that ϕ is a conserved property.

So, we have again got an equation of conserved property like the inert mass transfer problem and therefore we would have N . Solution of that would be $N w$ equal to $g \text{ star } m \ln \phi_{\infty} - \phi_T$ over $\phi_w - \phi_T$ equal to $g \text{ star } m \ln 1 + B m$, where $B m$ would be given. Now, by this quantity $\phi_{\infty} - \phi_w$ over $\phi_w - \phi_T$ and ϕ , **can stand for this group** or it can stand for this group, do not one use the appropriate group depending on the convenience of the problems at hand. $g \text{ star } m$ would be $\gamma m d$ or by $r w$ in spherical system or γm divided by L in the plane system. The important thing to note is that, even in a problem involving combustion we are able to reduce the problem mathematically to a form just like that of evaporation of water in the absence of heat transfer.

(Refer Slide Time: 27:36)

MT with HT and SCR - 3 - L32(13/20)

- The energy eqn will read as

$$\frac{d}{dy} \left[A (N_w h_m - k_m \frac{dT}{dy}) \right] = - \frac{dA}{dy} \sum_k (m''_{y,k} h_k) = \text{(RHS)}$$
 where $h_k = h_{f,k}^0 + c_{p,k} (T - T_{ref}) = h_{f,k}^0 + c_{p,k} \Delta T$.
- Hence, making use of definitions of Φ

$$\begin{aligned} h_m &= \sum \omega_k h_k = \sum \omega_k h_{f,k}^0 + \Delta T \sum c_{p,k} \omega_k \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{O_2} h_{f,O_2}^0 + \omega_{pr} h_{f,pr}^0 + c_{pm} \Delta T \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{O_2} \left\{ h_{f,O_2}^0 - \left(\frac{1+r_{st}}{r_{st}} \right) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{fu} r_{st} \left\{ h_{f,O_2}^0 - \left(\frac{1+r_{st}}{r_{st}} \right) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \left\{ h_{f,fu}^0 + r_{st} h_{f,O_2}^0 - (1+r_{st}) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \Delta h_c + c_{pm} (T - T_{ref}) \end{aligned}$$

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(Refer Slide Time: 27:55)

Reminder of Gov Eqns - L32($\frac{1}{20}$)

In the Stefan flow model, under steady state

$$\frac{1}{A} \frac{d}{dy} [N_{\psi,y} A] = \frac{1}{A} \frac{d}{dy} \left[\rho_m v A \Psi - \Gamma_{\psi} A \frac{d\Psi}{dy} \right] = S_{\psi}$$

Ψ	Γ_{ψ}	S_{ψ}
ω_k	$\rho_m D$	R_k
T_{br}	$\rho_m D$	0
h_m	k_m / cp_m	$-A^{-1} d(\sum_k A m''_{y,k} h_k) / dy$

where \dot{Q}_{rad} is neglected and $m''_{y,k} = -\rho_m D (d\omega_k / dy)$

So let me now turn to energy part of the... because, whenever combustion takes place there would be heat transfer. Now, we would have d by dy of A N w h m minus k m d T dy would equal d of the diffusion mass transfer of dy. You remember from slide 1, this is the right hand side and this is the equation that I am writing now.

(Refer Slide Time: 28:02)

MT with HT and SCR - 3 - L32($\frac{13}{20}$)

● The energy eqn will read as

$$\frac{d}{dy} \left[A (N_w h_m - k_m \frac{dT}{dy}) \right] = - \frac{d A \sum_k (m''_{y,k} h_k)}{dy} = (\text{RHS})$$

where $h_k = h_{f,k}^0 + cp_k (T - T_{ref}) = h_{f,k}^0 + cp_k \Delta T$.

● Hence, making use of definitions of Φ

$$\begin{aligned} h_m &= \sum \omega_k h_k = \sum \omega_k h_{f,k}^0 + \Delta T \sum cp_k \omega_k \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{O_2} h_{f,O_2}^0 + \omega_{pr} h_{f,pr}^0 + c_{pm} \Delta T \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{O_2} \left\{ h_{f,O_2}^0 - \left(\frac{1+r_{st}}{r_{st}} \right) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{fu} r_{st} \left\{ h_{f,O_2}^0 - \left(\frac{1+r_{st}}{r_{st}} \right) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \left\{ h_{f,fu}^0 + r_{st} h_{f,O_2}^0 - (1+r_{st}) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \Delta h_c + c_{pm} (T - T_{ref}) \end{aligned}$$

This is the energy equation, then equal to the right hand side; this is the right hand side where h sub k is the species enthalpy and as you will know that whenever you have reacting fuel we write it as h naught f k as the enthalpy of formation plus C p k into T

minus T_{ref} or for short I will write as $h_{f,k} c_{p,k} \Delta T$. Hence, making use of definitions of ϕ , let us say h_m would be simply $\omega_k h_k$ and therefore, it will be $\omega_k h_{f,k} + \Delta T \sum c_{p,k} \omega_k$. Then the first part of the term would simply result in $\omega_{fu} h_{fu} + \omega_{O_2} h_{O_2} + \omega_{pr} h_{pr} + \Delta T \sum c_{p,m} \omega_k$ the mixture specific heat into ΔT .

(Refer Slide Time: 29:08)

MT with HT and SCR - 2 - L32(12/20)

- Adding the 3 eqns, we retrieve bulk mass conservation. Hence, $\sum_k R_k = 0$ and $\sum_k \dot{m}_{y,k} = 0$.
- Thus, SCR implies $R_{O_2} = r_{st} R_{fu}$, $R_{pr} = -(1 + r_{st}) R_{fu}$ and $\dot{m}_{O_2} = r_{st} \dot{m}_{fu}$, $\dot{m}_{pr} = -(1 + r_{st}) \dot{m}_{fu}$.
- Further dividing 2nd eqn by r_{st} and third eqn by $(1 + r_{st})$, and subtracting from 1st eqn, it follows that

$$\frac{d}{dy} \left[A \left(N_w \phi - \Gamma_m \frac{d\phi}{dy} \right) \right] = 0$$

$$\phi = \omega_{fu} - \frac{\omega_{O_2}}{r_{st}} = \omega_{fu} + \frac{\omega_{pr}}{1 + r_{st}} \quad \text{Conserved Property}$$

soln: $N_w = g_m^* \ln \left[\frac{\phi_{\infty} - \phi_T}{\phi_w - \phi_T} \right] = g_m^* \ln(1 + B_m)$ where

$$B_m = \frac{\phi_{\infty} - \phi_w}{\phi_w - \phi_T} \quad \text{and} \quad g_m^* = \frac{\Gamma_m}{r_w} \quad \text{or} \quad \frac{\Gamma_m}{L}$$

(Refer Slide Time: 29:16)

MT with HT and SCR - 3 - L32(13/20)

- The energy eqn will read as

$$\frac{d}{dy} \left[A \left(N_w h_m - k_m \frac{dT}{dy} \right) \right] = - \frac{dA}{dy} \sum_k (\dot{m}_{y,k} h_k) = \text{(RHS)}$$

where $h_k = h_{f,k}^0 + c_{p,k} (T - T_{ref}) = h_{f,k}^0 + c_{p,k} \Delta T$.

- Hence, making use of definitions of ϕ

$$\begin{aligned} h_m &= \sum \omega_k h_k = \sum \omega_k h_{f,k}^0 + \Delta T \sum c_{p,k} \omega_k \\ &= \omega_{fu} h_{fu}^0 + \omega_{O_2} h_{O_2}^0 + \omega_{pr} h_{pr}^0 + c_{pm} \Delta T \\ &= \omega_{fu} h_{fu}^0 + \omega_{O_2} \left\{ h_{O_2}^0 - \left(\frac{1 + r_{st}}{r_{st}} \right) h_{pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} h_{fu}^0 + \omega_{fu} r_{st} \left\{ h_{O_2}^0 - \left(\frac{1 + r_{st}}{r_{st}} \right) h_{pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \left\{ h_{fu}^0 + r_{st} h_{O_2}^0 - (1 + r_{st}) h_{pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \Delta h_c^0 + c_{pm} (T - T_{ref}) \end{aligned}$$

Now, from the previous slide we have defined phi. So, I can say that omega o 2 by r st will be minus omega product over r st. So, that is what I am going to use here. I am replacing omega product by 1 minus r st over r st omega o 2 equal plus C p m delta T.

Omega o 2 itself would be equal to omega fu r st into all these and as a result, you will see I get omega fu into h naught fu plus r st h naught o 2 h naught f o 2 minus 1 plus r st h naught f product plus C p m delta T.

This essentially means that this is the enthalpy, total enthalpy of the reactants. 1 mole of fuel combines with r st moles of oxygen therefore, this is the enthalpy of the reactants and this is the enthalpy of the products. So, the reactant enthalpy minus product enthalpy as you know, is the heat of the combustion and therefore, you will get omega fu del h c plus C p m into T minus T ref, where delta T is written as T minus T ref; again, as I have said there.

So, we get a, enthalpy equal to heat of combustion into omega fuel now, I have here replaced omega o 2 and omega product in terms of omega fu but I could do it the other way. I can replace omega fu and omega o 2 in terms of omega product or I can replace omega fu and omega product in terms of omega 2 and I will get different expressions involving del h c which I will show you shortly.

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MT with HT and SCR - 3 - L32(13/20)

- The energy eqn will read as

$$\frac{d}{dy} \left[A (N_w h_m - k_m \frac{dT}{dy}) \right] = - \frac{dA \sum_k (\dot{m}_{y,k} h_k)}{dy} = \text{(RHS)}$$

where $h_k = h_{f,k}^0 + c_{p,k} (T - T_{ref}) = h_{f,k}^0 + c_{p,k} \Delta T$.

- Hence, making use of definitions of Φ

$$\begin{aligned} h_m &= \sum \omega_k h_k = \sum \omega_k h_{f,k}^0 + \Delta T \sum c_{p,k} \omega_k \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{O_2} h_{f,O_2}^0 + \omega_{pr} h_{f,pr}^0 + c_{pm} \Delta T \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{O_2} \left\{ h_{f,O_2}^0 - \left(\frac{1+r_{st}}{r_{st}} \right) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{fu} r_{st} \left\{ h_{f,O_2}^0 - \left(\frac{1+r_{st}}{r_{st}} \right) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \left\{ h_{f,fu}^0 + r_{st} h_{f,O_2}^0 - (1+r_{st}) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \Delta h_c + c_{pm} (T - T_{ref}) \end{aligned}$$

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MT with HT and SCR - 4 - L32 ^(14/20)

We now consider the RHS. Then

$$-\sum_k (\dot{m}_{y,k} h_k) = (h_{fu}^0 + c_{p,fu} \Delta T) \rho_m D \frac{d\omega_{fu}}{dy} + (h_{O_2}^0 + c_{p,O_2} \Delta T) \rho_m D \frac{d\omega_{O_2}}{dy} + (h_{pr}^0 + c_{p,pr} \Delta T) \rho_m D \frac{d\omega_{pr}}{dy}$$

If we now assume that $c_{p,k} = c_{pm}$ (equal specific heats) and use stoichiometric relations $\omega_{O_2} = r_{st} \omega_{fu}$ and $\omega_{pr} = -(1 + r_{st}) \omega_{fu}$

$$-\sum_k (\dot{m}_{y,k} h_k) = \Delta h_c \rho_m D \frac{d\omega_{fu}}{dy}$$

because $c_{pm} \Delta T \sum_k D \frac{d\omega_k}{dy} = 0$. Also, from the previous slide

$$c_{pm} \frac{dT}{dy} = \frac{dh_m}{dy} - \Delta h_c \frac{d\omega_{fu}}{dy}$$

But, now let us consider the right hand side which is minus d by dy of A sigma k m y k h k. So, here summation of k m double prime y k h k would be simply h naught fu C p fu delta T into this, is the expression for m y fu plus the same quantity for oxygen and the same quantity for product.

Now, we make an important assumption which is always made for gaseous mixtures. Specific heats are functions of temperature and for different species they are somewhat different; but they are not so different that we cannot make an assumption of equal specific heats. If you assume that the each specie has the same specific heat then it would simply equal in the mixture specific heats.

So, that is the assumption I am going to make and use the stoichiometric relation omega o 2 equal to r st fu omega fu and omega product equal to minus 1 minus r st omega fu. If I do that then this entire relationship can be written as minus del h c rho m D omega fu by dy, because C p m delta T into sigma k D d omega k by dy would **because** sigma omega k would simply add up to 1 and that is equal to 0. Also, from the previous slide we have shown that C p m dT by dy is equal to d h m by dy minus, remember?

(Refer Slide Time: 32:31)

MT with HT and SCR - 3 - L32(¹³/₂₀)

● The energy eqn will read as

$$\frac{d}{dy} \left[A (N_w h_m - k_m \frac{dT}{dy}) \right] = - \frac{dA \sum_k (\dot{m}_{y,k}'' h_k)}{dy} = (\text{RHS})$$

where $h_k = h_{f,k}^0 + c_{p,k} (T - T_{ref}) = h_{f,k}^0 + c_{p,k} \Delta T$.

● Hence, making use of definitions of Φ

$$\begin{aligned} h_m &= \sum \omega_k h_k = \sum \omega_k h_{f,k}^0 + \Delta T \sum c_{p,k} \omega_k \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{O_2} h_{f,O_2}^0 + \omega_{pr} h_{f,pr}^0 + c_{pm} \Delta T \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{O_2} \left\{ h_{f,O_2}^0 - \left(\frac{1+r_{st}}{r_{st}} \right) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{fu} r_{st} \left\{ h_{f,O_2}^0 - \left(\frac{1+r_{st}}{r_{st}} \right) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \left\{ h_{f,fu}^0 + r_{st} h_{f,O_2}^0 - (1+r_{st}) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \Delta h_c + c_{pm} (T - T_{ref}) \end{aligned}$$

(Refer Slide Time: 32:44)

MT with HT and SCR - 4 - L32(¹⁴/₂₀)

We now consider the RHS. Then

$$\begin{aligned} - \sum_k (\dot{m}_{y,k}'' h_k) &= (h_{fu}^0 + c_{p,fu} \Delta T) \rho_m D \frac{d \omega_{fu}}{dy} + \\ &(h_{O_2}^0 + c_{p,O_2} \Delta T) \rho_m D \frac{d \omega_{O_2}}{dy} + (h_{pr}^0 + c_{p,pr} \Delta T) \rho_m D \frac{d \omega_{pr}}{dy} \end{aligned}$$

If we now assume that $c_{p,k} = c_{pm}$ (equal specific heats) and use stoichiometric relations $\omega_{O_2} = r_{st} \omega_{fu}$ and $\omega_{pr} = -(1+r_{st}) \omega_{fu}$

$$- \sum_k (\dot{m}_{y,k}'' h_k) = \Delta h_c \rho_m D \frac{d \omega_{fu}}{dy}$$

because $c_{pm} \Delta T \sum_k D d \omega_k / dy = 0$. Also, from the previous slide

$$c_{pm} \frac{dT}{dy} = \frac{dh_m}{dy} - \Delta h_c \frac{d \omega_{fu}}{dy}$$

So, if I take differential of this equation with respect to y then C p m into dT by dy would be d h m by dy into minus del h c into d omega fu by dy and that is what I have written here. Now, I have this expression and I have this expression which I am going to make use of in deriving the right hand side.

(Refer Slide Time: 32:56)

MT with HT and SCR - 4 - L32(¹⁴/₂₀)

We now consider the RHS. Then

$$-\sum_k (\dot{m}_{y,k} h_k) = (h_{fu}^0 + c_{p,fu} \Delta T) \rho_m D \frac{d \omega_{fu}}{dy} + (h_{O_2}^0 + c_{p,O_2} \Delta T) \rho_m D \frac{d \omega_{O_2}}{dy} + (h_{pr}^0 + c_{p,pr} \Delta T) \rho_m D \frac{d \omega_{pr}}{dy}$$

If we now assume that $c_{p,k} = c_{pm}$ (equal specific heats) and use stoichiometric relations $\omega_{O_2} = r_{st} \omega_{fu}$ and $\omega_{pr} = -(1 + r_{st}) \omega_{fu}$

$$-\sum_k (\dot{m}_{y,k} h_k) = \Delta h_c \rho_m D \frac{d \omega_{fu}}{dy}$$

because $c_{pm} \Delta T \sum_k D d \omega_k / dy = 0$. Also, from the previous slide

$$c_{pm} \frac{dT}{dy} = \frac{dh_m}{dy} - \Delta h_c \frac{d \omega_{fu}}{dy}$$

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MT with HT and SCR - 3 - L32(¹³/₂₀)

① The energy eqn will read as

$$\frac{d}{dy} \left[A (N_w h_m - k_m \frac{dT}{dy}) \right] = - \frac{d A \sum_k (\dot{m}_{y,k} h_k)}{dy} = (\text{RHS})$$

where $h_k = h_{f,k}^0 + c_{p,k} (T - T_{ref}) = h_{f,k}^0 + c_{p,k} \Delta T$.

② Hence, making use of definitions of Φ

$$\begin{aligned} h_m &= \sum \omega_k h_k = \sum \omega_k h_{f,k}^0 + \Delta T \sum c_{p,k} \omega_k \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{O_2} h_{f,O_2}^0 + \omega_{pr} h_{f,pr}^0 + c_{pm} \Delta T \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{O_2} \left\{ h_{f,O_2}^0 - \left(\frac{1+r_{st}}{r_{st}} \right) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} h_{f,fu}^0 + \omega_{fu} r_{st} \left\{ h_{f,O_2}^0 - \left(\frac{1+r_{st}}{r_{st}} \right) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \left\{ h_{f,fu}^0 + r_{st} h_{f,O_2}^0 - (1+r_{st}) h_{f,pr}^0 \right\} + c_{pm} \Delta T \\ &= \omega_{fu} \Delta h_c + c_{pm} (T - T_{ref}) \end{aligned}$$

(Refer Slide Time: 33:07)

MT with HT and SCR - 5 - L32(15/20)
Hence, the energy Eqn will read as

$$\frac{d}{dy} \left[A \left\{ N_w h_m - \frac{k_m}{c_{pm}} \left(\frac{dh_m}{dy} - \Delta h_c \frac{d\omega_{fu}}{dy} \right) - \Delta h_c \rho_m D \frac{d\omega_{fu}}{dy} \right\} \right] = 0$$

Noting that $k_m/c_{pm} = \rho_m \alpha_m = \Gamma_h$ and assuming $\alpha_m = D$ (or $Le = 1$), the above eqn can be rearranged so that the Burning flux can be calculated from

$$\frac{d}{dy} \left[A \left(N_w h_m - \Gamma_h \frac{dh_m}{dy} \right) \right] = 0$$

$$N_w = g_{mh}^* \ln \left[\frac{h_{m,\infty} - h_{m,T}}{h_{m,w} - h_{m,T}} \right] = g_{mh}^* \ln (1 + B_h) \text{ where}$$

$$B_h = \frac{h_{m,\infty} - h_{m,w}}{h_{m,w} - h_{m,T}} \text{ and } g_{mh}^* = \frac{\Gamma_{mh}}{r_w} \text{ or } \frac{\Gamma_{mh}}{L}$$

So, the right hand side can now be formed which is as you know is d by dy. I can now take everything on this side and you can see that I can transform this equation into A times N w h m into k m by C p m d h m by dy minus del h c d omega fu by dy. This is essentially the k d T dy term. This is the del h c rho m D d omega fu by dy which is the right hand side term. Then, if I again make the Lewis number equal to 1 assumption that is alpha m equal to D, then you will see that gamma m this k m by C p m can also be written as rho m into D and equal to gamma h. **let us say** So, then you will see that this term would get cancelled with that term and I would get A times N w h m gamma h into d h m by dy equal to 0 and again a conserved property relationship has been obtained from the energy equation.

The solution would be again same as that for inert mass transfer with B h defined as h m infinity minus h m w h m w minus h m T and this is the g star m h. Again, you will see that in a simple chemical reaction we have got the same formula; both from mass transfer equation as well as from the energy equation.

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Comments - L32 ⁽¹⁶⁾/₍₂₀₎

- In general, for an SCR,
 - $h_{fu} = c_{pm} \Delta T + \omega_{fu} \Delta h_c$, $h_{O_2} = c_{pm} \Delta T$ and $h_{pr} = c_{pm} \Delta T$
 - $h_{fu} = c_{pm} \Delta T$, $h_{O_2} = c_{pm} \Delta T + \frac{\omega_{O_2}}{r_{st}} \Delta h_c$ and $h_{pr} = c_{pm} \Delta T$
 - $h_{fu} = c_{pm} \Delta T$, $h_{O_2} = c_{pm} \Delta T$ and $h_{pr} = c_{pm} \Delta T - \frac{\omega_{O_2}}{r_{st}} \Delta h_c$
- For a liquid fuel burning in air, we choose 2nd type

$$h_m = \sum_k \omega_k h_k = c_{p,m} (T - T_{ref}) + \frac{\omega_{O_2}}{r_{st}} \Delta h_c$$

where $T_{ref} = T_w$. Then, B_h can be calculated from

$$B_h = \frac{c_{pm} (T_\infty - T_w) + \Delta h_c (\omega_{O_2,\infty} - \omega_{O_2,w}) / r_{st}}{\Delta h_c \omega_{O_2,w} / r_{st} - c_{p,l} (T_T - T_w)}$$

where $\omega_{O_2,T} = 0$ and T_T is known or knowable

 - If $T_w = T_{bp}$ (boiling point) then $\omega_{O_2,w} = 0$
 - If $T_w \neq T_{bp}$ then $\omega_{O_2,w} \sim T_w$ is iteratively determined from $B_m = B_h$.

So, as I said earlier that we can define our mixture in variety of ways for a simple chemical reaction, one is to say h_{fu} is equal to $C_p m \Delta T + \omega_{fu} \Delta h_c$ which is what I did earlier. h_{O_2} will then be equal to $C_p m \Delta T$ and h_{pr} will be $C_p m \Delta T$, because of the equal specific heat. But, I can also associate Δh_c with oxygen in which case h_{fu} will be $C_p m \Delta T$, h_{O_2} will be $C_p m \Delta T + \frac{\omega_{O_2}}{r_{st}} \Delta h_c$ and h_{pr} . Then, again you have h_{fu} , h_{O_2} and I can associate now with ω_{O_2} with the product mass fraction.

So, there are three ways in which you can do it for a liquid fuel burning in air. We often choose second type because we often know ω_{O_2} concentrations much better. Then h_m will be $\omega_k h_k$ would be simply $C_p m (T - T_{ref}) + \frac{\omega_{O_2}}{r_{st}} \Delta h_c$ which is from these three relationships. If I now take for a moment that T_{ref} is equal to T_w which I do not know usually, then I will get for convenience, rather I am not knowing where T_{ref} . If, I take T_{ref} equal to T_w then B_h which was $h_m(T_\infty) - h_m(T_w)$ would transform to $C_p m (T_\infty - T_w) + \Delta h_c (\omega_{O_2,\infty} - \omega_{O_2,w}) / r_{st}$ divided by $\Delta h_c \omega_{O_2,w} / r_{st} - c_{p,l} (T_T - T_w)$, where of course $\omega_{O_2,T}$ is 0 because, oxygen does not exist in the fuel and T_T is known or knowable somehow.

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MT with HT and SCR - 5 - L32(15/20)

Hence, the energy Eqn will read as

$$\frac{d}{dy} \left[A \left\{ N_w h_m - \frac{k_m}{c_{pm}} \left(\frac{d h_m}{dy} - \Delta h_c \frac{d \omega_{fu}}{dy} \right) - \Delta h_c \rho_m D \frac{d \omega_{fu}}{dy} \right\} \right] = 0$$

Noting that $k_m/c_{pm} = \rho_m \alpha_m = \Gamma_h$ and assuming $\alpha_m = D$ (or $Le = 1$), the above eqn can be rearranged so that the Burning flux can be calculated from

$$\frac{d}{dy} \left[A \left(N_w h_m - \Gamma_h \frac{d h_m}{dy} \right) \right] = 0$$

$$N_w = g_{mh}^* \ln \left[\frac{h_{m,\infty} - h_{m,T}}{h_{m,w} - h_{m,T}} \right] = g_{mh}^* \ln (1 + B_h) \text{ where}$$

$$B_h = \frac{h_{m,\infty} - h_{m,w}}{h_{m,w} - h_{m,T}} \text{ and } g_{mh}^* = \frac{\Gamma_{mh}}{r_w} \text{ or } \frac{\Gamma_{mh}}{L}$$

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Comments - L32(16/20)

- ① In general, for an SCR,
 - ③ $h_{fu} = c_{pm} \Delta T + \omega_{fu} \Delta h_c$, $h_{O_2} = c_{pm} \Delta T$ and $h_{pr} = c_{pm} \Delta T$
 - ③ $h_{fu} = c_{pm} \Delta T$, $h_{O_2} = c_{pm} \Delta T + \frac{\omega_{O_2}}{r_{st}} \Delta h_c$ and $h_{pr} = c_{pm} \Delta T$
 - ③ $h_{fu} = c_{pm} \Delta T$, $h_{O_2} = c_{pm} \Delta T$ and $h_{pr} = c_{pm} \Delta T - \frac{\omega_{pr}}{1+r_{st}} \Delta h_c$
- ② For a liquid fuel burning in air, we choose 2nd type

$$h_m = \sum_k \omega_k h_k = c_{p,m} (T - T_{ref}) + \frac{\omega_{O_2}}{r_{st}} \Delta h_c$$

where $T_{ref} = T_w$. Then, B_h can be calculated from

$$B_h = \frac{c_{pm} (T_\infty - T_w) + \Delta h_c (\omega_{O_2,\infty} - \omega_{O_2,w}) / r_{st}}{\Delta h_c \omega_{O_2,w} / r_{st} - c_{p,f} (T_T - T_w)}$$

where $\omega_{O_2,T} = 0$ and T_T is known or knowable

 - ③ If $T_w = T_{bp}$ (boiling point) then $\omega_{O_2,w} = 0$
 - ③ If $T_w \neq T_{bp}$ then $\omega_{O_2,w} \sim T_w$ is iteratively determined from $B_m = B_h$.

If T_w was already, the fuel surface was already at the boiling point T_w by T_{bp} , then of course, no oxygen would survive at the surface and therefore that would be 0 because the fuel concentration there would be 1.

If T_w is not equal to T_{bp} then of course, ω_{O_2} and T_w relations must be established iteratively by balancing B_m and B_h , where B_h is given by this (Refer Slide Time:37:05). So, you assume at T_w evaluate the B_h value then evaluate the B_m then

evaluate the omega v w rather omega o 2 w and then again get the balance done in favor of till convergence is obtained.

(Refer Slide Time: 37:18)

MT with HT and SCR - 2 - L32($\frac{12}{20}$)

- Adding the 3 eqns, we retrieve bulk mass conservation. Hence, $\sum_k R_k = 0$ and $\sum_k \dot{m}_{y,k} = 0$
- Thus, SCR implies $R_{O_2} = r_{st} R_{fu}$, $R_{pr} = -(1 + r_{st}) R_{fu}$ and $\dot{m}_{O_2} = r_{st} \dot{m}_{fu}$, $\dot{m}_{pr} = -(1 + r_{st}) \dot{m}_{fu}$
- Further dividing 2nd eqn by r_{st} and third eqn by $(1 + r_{st})$, and subtracting from 1st eqn, it follows that

$$\frac{d}{dy} \left[A \left(N_w \phi - \Gamma_m \frac{d\phi}{dy} \right) \right] = 0$$

$$\phi = \omega_{fu} - \frac{\omega_{O_2}}{r_{st}} = \omega_{fu} + \frac{\omega_{pr}}{1 + r_{st}} \quad \text{Conserved Property}$$

soln : $N_w = g_m^* \ln \left[\frac{\phi_\infty - \phi_T}{\phi_w - \phi_T} \right] = g_m^* \ln (1 + B_m)$ where

$$B_m = \frac{\phi_\infty - \phi_w}{\phi_w - \phi_T} \quad \text{and} \quad g_m^* = \frac{\Gamma_m}{r_w} \quad \text{or} \quad \frac{\Gamma_m}{L}$$

Where B m is given by this relationship and you will see phi value now will be known in all states for example omega fu in the infinity state is 0. Whereas, omega o 2 is known so phi infinity is known in the T state omega fu will be 1 and omega o 2 will be 0 so phi T is known. omega w will depend on the temperature which will determine omega fu and the remaining will be omega o 2.

(Refer Slide Time: 37:53)

Comments - L32($\frac{16}{20}$)

- In general, for an SCR,
 - $h_{fu} = c_{pm} \Delta T + \omega_{fu} \Delta h_c$, $h_{O_2} = c_{pm} \Delta T$ and $h_{pr} = c_{pm} \Delta T$
 - $h_{fu} = c_{pm} \Delta T$, $h_{O_2} = c_{pm} \Delta T + \frac{\omega_{O_2}}{r_{st}} \Delta h_c$ and $h_{pr} = c_{pm} \Delta T$
 - $h_{fu} = c_{pm} \Delta T$, $h_{O_2} = c_{pm} \Delta T$ and $h_{pr} = c_{pm} \Delta T - \frac{\omega_{pr}}{1+r_{st}} \Delta h_c$
- For a liquid fuel burning in air, we choose 2nd type

$$h_m = \sum_k \omega_k h_k = c_{p,m} (T - T_{ref}) + \frac{\omega_{O_2}}{r_{st}} \Delta h_c$$

where $T_{ref} = T_w$. Then, B_m can be calculated from

$$B_m = \frac{c_{pm} (T_\infty - T_w) + \Delta h_c (\omega_{O_2,\infty} - \omega_{O_2,w}) / r_{st}}{\Delta h_c \omega_{O_2,w} / r_{st} - c_{p,l} (T_T - T_w)}$$

where $\omega_{O_2,T} = 0$ and T_T is known or knowable

- If $T_w = T_{bp}$ (boiling point) then $\omega_{O_2,w} = 0$
- If $T_w \neq T_{bp}$ then $\omega_{O_2,w} \sim T_w$ is iteratively determined from $B_m = B_h$.

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MT with HT and ACR - 1 - L32(17/20)

● Here, the governing eqn is

$$\frac{d}{dy} \left[A \left(N_w \eta_{\alpha} - \Gamma_{\alpha} \frac{d \eta_{\alpha}}{dy} \right) \right] = 0$$

where η_{α} is a conserved property with $\alpha = \text{C, H, N or O}$.

● This eqn is often used for **solids combustion or ablation**.

● Thus, consider burning of Graphite (C^*) at high temperature ($T_w \sim 1950 \text{ K}$). Then the reaction mechanism is

R1 : $\text{C}^* + 0.5 \text{O}_2 = \text{CO}$ ($K_p = \infty$),
R2 : $\text{C}^* + \text{CO}_2 = 2 \text{CO}$ ($K_p = 4000$),
R3 : $\text{C}^* + \text{H}_2\text{O} = \text{CO} + \text{H}_2$ ($K_p = 1230$),
R4 : $\text{C}^* + 2 \text{H}_2 = \text{CH}_4$ ($K_p = 1/790$),
R5 : $\text{CO}_2 = \text{CO} + 0.5 \text{O}_2$ ($K_p = 1/1800$),
R6 : $\text{H}_2\text{O} = \text{H}_2 + 0.5 \text{O}_2$ ($K_p = 1/40,000$)

We can determine the relationships in that manner and carry out iterations. Now, I come to mass transfer with heat transfer and arbitrary chemical reaction. What do I mean by arbitrary chemical reaction? By arbitrary chemical reaction, is typically a curve in solids combustion? For example, let us say consider burning of a graphite C star is given by C star at very high temperature T_w of the order of 1950 or much higher then, there are several reactions taking place. The first 4 reactions take place at the surface of the graphite C star half O_2 equal to CO whose equilibrium constant infinity C star CO_2 would be 2 CO whose again K_p is 4000 C star H_2O equal to CO plus H_2 1230 and C star equal to 2 H_2 equal to CH_4 1 by 790.

CO does generate a hydrogen and C H_4 would then burn in the gas phase. Of course, here the K_p so low at 1950, there hardly any C H_4 would be formed and therefore, we can say that very small amounts of C H_4 will be present. CO would then react with oxygen to produce CO_2 which would then dissociate in this fashion. The CO_2 will be CO plus half O_2 , giving you CO_2 and H_2O would dissociate to give H_2 plus O_2 . Again, the K_p is very low and therefore the reverse reactions would be dominant you producing CO_2 and H_2O .

So, for such a complex mechanism it is best to conserve the elements we have. Elements, C H and O in this case; you can see elements are C H and O of course, nitrogen is not

present in the fuel and therefore we write this and as you recall, the element mass fraction equation is always a conserved property equation.

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MT with HT and ACR - 2 - L32($\frac{18}{20}$)

- ① Let the ∞ state comprise complete products CO_2 , H_2O and N_2 only
- ② Noting the equilibrium constants K_p for each reaction, it can be shown that in the considered phase, CH_4 cannot survive in appreciable magnitudes. Hence, it will comprise CO_2 , H_2 , CO and H_2O only.
- ③ Similarly, in the w-state, only CO and H_2 will survive
- ④ Since, species change in different states, it is best to define

$$\eta_C = \omega_C + \frac{12}{44} \omega_{\text{CO}_2} + \frac{12}{28} \omega_{\text{CO}}$$

$$\eta_H = \omega_{\text{H}_2} + \frac{2}{18} \omega_{\text{H}_2\text{O}}$$

$$\eta_O = \frac{32}{44} \omega_{\text{CO}_2} + \frac{16}{28} \omega_{\text{CO}} + \frac{16}{18} \omega_{\text{H}_2\text{O}}$$

- ⑤ Thus, we have 3 eqns in η_C , η_O and η_H .

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So let the infinity state comprise of CO_2 , H_2O and N_2 **only that is the full products that so, the noting** the equilibrium constant K_p for each reaction it can be shown that in the considered phase, CH_4 cannot survive in appreciable magnitudes. Hence, it will comprise, the considered phase will essentially comprise of CO_2 , H_2 , CO and H_2O only.

Similarly, in the w-state only CO and H_2 will survive and therefore since species change in different states it is best to define a η_C equal to $\frac{\omega_C}{12} + \frac{\omega_{\text{CO}_2}}{44} + \frac{\omega_{\text{CO}}}{28}$ then η_H would be given by that and η_O it will be given by that composition.

(Refer Slide Time: 41:08)

MT with HT and ACR - 3 - L32(¹⁹/₂₀)

- ① We need to solve for any one element because the conserved property solution are same for each $\eta_{i,c}$. Difficulty arises in applying BCs
- ② Since mass-fractions of CO and H_2 are not known in the w-state, we define a composite variable

$$\Phi = \eta_C - \frac{3}{4} \eta_O = \omega_C - \frac{3}{11} \omega_{CO_2} - \frac{2}{3} \omega_{H_2O}$$

- ③ Then,

$$\Phi_w = (\eta_C - \frac{3}{4} \eta_O)_w = 0, \quad \Phi_T = 1 - 0 - 0$$

$$\Phi_\infty = 0 - \frac{3}{11} \omega_{CO_2,\infty} - \frac{2}{3} \omega_{H_2O,\infty}$$

$$N_w = g_m^* \ln(1 + B_m) \rightarrow B_m = \frac{3}{11} \omega_{CO_2,\infty} + \frac{2}{3} \omega_{H_2O,\infty}$$

(Refer Slide Time: 41:17)

MT with HT and ACR - 2 - L32(¹⁸/₂₀)

- ① Let the ∞ state comprise complete products CO_2 , H_2O and N_2 only
- ② Noting the equilibrium constants K_p for each reaction, it can be shown that in the considered phase, CH_4 cannot survive in appreciable magnitudes. Hence, it will comprise CO_2 , H_2 , CO and H_2O only.
- ③ Similarly, in the w-state, only CO and H_2 will survive
- ④ Since, species change in different states, it is best to define

$$\eta_C = \omega_C + \frac{12}{44} \omega_{CO_2} + \frac{12}{28} \omega_{CO}$$

$$\eta_H = \omega_{H_2} + \frac{2}{18} \omega_{H_2O}$$

$$\eta_O = \frac{32}{44} \omega_{CO_2} + \frac{16}{28} \omega_{CO} + \frac{16}{18} \omega_{H_2O}$$

- ⑤ Thus, we have 3 eqns in η_C , η_O and η_H .

Thus we have 3 equations for eta C eta H and eta O. So, instead of solving 3 equations any 1 can be solved; but you will find that not all these quantities on the right hand side are very well known in the three states.

(Refer Slide Time: 41:22)

MT with HT and ACR - 3 - L32(¹⁹/₂₀)

- ① We need to solve for any one element because the conserved property solution are same for each $\eta_{i,w}$. Difficulty arises in applying BCs
- ② Since mass-fractions of CO and H_2 are not known in the w-state, we define a composite variable

$$\Phi = \eta_C - \frac{3}{4} \eta_O = \omega_C - \frac{3}{11} \omega_{CO_2} - \frac{2}{3} \omega_{H_2O}$$

- ③ Then,

$$\Phi_w = (\eta_C - \frac{3}{4} \eta_O)_w = 0, \quad \Phi_T = 1 - 0 - 0$$

$$\Phi_\infty = 0 - \frac{3}{11} \omega_{CO_2,\infty} - \frac{2}{3} \omega_{H_2O,\infty}$$

$$N_w = g_m^* \ln(1 + B_m) \rightarrow B_m = \frac{3}{11} \omega_{CO_2,\infty} + \frac{2}{3} \omega_{H_2O,\infty}$$

(Refer Slide Time: 41:32)

MT with HT and ACR - 2 - L32(¹⁸/₂₀)

- ① Let the ∞ state comprise complete products CO_2 , H_2O and N_2 only
- ② Noting the equilibrium constants K_p for each reaction, it can be shown that in the considered phase, CH_4 cannot survive in appreciable magnitudes. Hence, it will comprise CO_2 , H_2 , CO and H_2O only.
- ③ Similarly, in the w-state, only CO and H_2 will survive
- ④ Since, species change in different states, it is best to define

$$\eta_C = \omega_C + \frac{12}{44} \omega_{CO_2} + \frac{12}{28} \omega_{CO}$$

$$\eta_H = \omega_{H_2} + \frac{2}{18} \omega_{H_2O}$$

$$\eta_O = \frac{32}{44} \omega_{CO_2} + \frac{16}{28} \omega_{CO} + \frac{16}{18} \omega_{H_2O}$$

- ⑤ Thus, we have 3 eqns in η_C , η_O and η_H .

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MT with HT and ACR - 3 - L32(19/20)

- We need to solve for any one element because the conserved property solution are same for each $\eta_{i,w}$. Difficulty arises in applying BCs
- Since mass-fractions of CO and H_2 are not known in the w-state, we define a composite variable

$$\Phi = \eta_C - \frac{3}{4} \eta_O = \omega_C - \frac{3}{11} \omega_{CO_2} - \frac{2}{3} \omega_{H_2O}$$

- Then,

$$\Phi_w = (\eta_C - \frac{3}{4} \eta_O)_w = 0, \quad \Phi_T = 1 - 0 - 0$$

$$\Phi_\infty = 0 - \frac{3}{11} \omega_{CO_2,\infty} - \frac{2}{3} \omega_{H_2O,\infty}$$

$$N_w = g_m^* \ln(1 + B_m) \rightarrow B_m = \frac{3}{11} \omega_{CO_2,\infty} + \frac{2}{3} \omega_{H_2O,\infty}$$

So, the best thing is to derive a composite quantity which we shall take as eta C minus 3 by 4 n o and from this, these relationship you will see that I can form a nu variable omega C minus 3 by 11 omega CO 2 minus 2 by 3 omega H 2 O. Now, I can definitely form phi w equal to n c minus 3 by n o w equal to 0 in the w state in the T state only eta C will be 1 or omega C will be 1. All these are zeros, so they have put to 0. And in the infinity state, I do not have any carbon but there is omega CO 2 and omega H 2 O so they are retained and as a result I will get N w equal to ln 1 plus B m, where B m would be 3 by 11 CO 2 infinity plus 2 by 3 omega H 2 O infinity.

(Refer Slide Time: 42:32)

MT with HT and ACR - 2 - L32(¹⁸/₂₀)

- Let the ∞ state comprise complete products CO_2 , H_2O and N_2 only
- Noting the equilibrium constants K_p for each reaction, it can be shown that in the considered phase, CH_4 cannot survive in appreciable magnitudes. Hence, it will comprise CO_2 , H_2 , CO and H_2O only.
- Similarly, in the w-state, only CO and H_2 will survive
- Since, species change in different states, it is best to define

$$\eta_C = \omega_C + \frac{12}{44} \omega_{\text{CO}_2} + \frac{12}{28} \omega_{\text{CO}}$$

$$\eta_H = \omega_{\text{H}_2} + \frac{2}{18} \omega_{\text{H}_2\text{O}}$$

$$\eta_O = \frac{32}{44} \omega_{\text{CO}_2} + \frac{16}{28} \omega_{\text{CO}} + \frac{16}{18} \omega_{\text{H}_2\text{O}}$$

- Thus, we have 3 eqns in η_C , η_O and η_H .

(Refer Slide Time: 42:37)

MT with HT and ACR - 3 - L32(¹⁹/₂₀)

- We need to solve for any one element because the conserved property solution are same for each η_i . Difficulty arises in applying BCs
- Since mass-fractions of CO and H_2 are not known in the w-state, we define a composite variable

$$\Phi = \eta_C - \frac{3}{4} \eta_O = \omega_C - \frac{3}{11} \omega_{\text{CO}_2} - \frac{2}{3} \omega_{\text{H}_2\text{O}}$$

- Then,

$$\Phi_w = (\eta_C - \frac{3}{4} \eta_O)_w = 0, \quad \Phi_T = 1 - 0 - 0$$

$$\Phi_\infty = 0 - \frac{3}{11} \omega_{\text{CO}_2,\infty} - \frac{2}{3} \omega_{\text{H}_2\text{O},\infty}$$

$$N_w = g_m^* \ln(1 + B_m) \rightarrow B_m = \frac{3}{11} \omega_{\text{CO}_2,\infty} + \frac{2}{3} \omega_{\text{H}_2\text{O},\infty}$$

Both these are known in the infinity state, because that is what we said that we know. only the product in the infinity state. So, and therefore, I can calculate the mass transfer rate of graphite burning simply by knowing CO_2 at in the infinity state and $\omega_{\text{H}_2\text{O}}$ in infinity state.

(Refer Slide Time: 43:03)

MT with HT and ACR - 2 - L32(¹⁸/₂₀)

- Let the ∞ state comprise complete products CO_2 , H_2O and N_2 only
- Noting the equilibrium constants K_p for each reaction, it can be shown that in the considered phase, CH_4 cannot survive in appreciable magnitudes. Hence, it will comprise CO_2 , H_2 , CO and H_2O only.
- Similarly, in the w-state, only CO and H_2 will survive
- Since, species change in different states, it is best to define

$$\eta_C = \omega_C + \frac{12}{44} \omega_{\text{CO}_2} + \frac{12}{28} \omega_{\text{CO}}$$

$$\eta_{\text{H}} = \omega_{\text{H}_2} + \frac{2}{18} \omega_{\text{H}_2\text{O}}$$

$$\eta_{\text{O}} = \frac{32}{44} \omega_{\text{CO}_2} + \frac{16}{28} \omega_{\text{CO}} + \frac{16}{18} \omega_{\text{H}_2\text{O}}$$

- Thus, we have 3 eqns in η_C , η_{O} and η_{H} .

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MT with HT and ACR - 3 - L32(¹⁹/₂₀)

- We need to solve for any one element because the conserved property solution are same for each η_i . Difficulty arises in applying BCs
- Since mass-fractions of CO and H_2 are not known in the w-state, we define a composite variable

$$\Phi = \eta_C - \frac{3}{4} \eta_{\text{O}} = \omega_C - \frac{3}{11} \omega_{\text{CO}_2} - \frac{2}{3} \omega_{\text{H}_2\text{O}}$$

- Then,

$$\Phi_w = (\eta_C - \frac{3}{4} \eta_{\text{O}})_w = 0, \quad \Phi_T = 1 - 0 - 0$$

$$\Phi_\infty = 0 - \frac{3}{11} \omega_{\text{CO}_2,\infty} - \frac{2}{3} \omega_{\text{H}_2\text{O},\infty}$$

$$N_w = g_m^* \ln(1 + B_m) \rightarrow B_m = \frac{3}{11} \omega_{\text{CO}_2,\infty} + \frac{2}{3} \omega_{\text{H}_2\text{O},\infty}$$

So in this case, because I know the relationship connecting element mass fractions with the species mass fractions, I am able to create a composite phi as eta C minus 3 by 4; simply by observation, this and this manipulation is simply by observation.

Such that I do not want any of the things, any specie on the right hand side whose concentration I would not know in the infinity w and T states and this is what I have been able to achieve; so that the calculation of mass transfer becomes easy.

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Summary - L32(20/20)

- Thus, in all types of mass transfer, we have shown that MT is governed by conserved property Eqn

$$\frac{d}{dy} \left[A (N_w \Psi - \Gamma \frac{d\Psi}{dy}) \right] = 0 \quad \text{where } A = \text{const or } = 4\pi r^2$$

$$N_w = g \times B \quad \text{with} \quad \frac{g}{g^*} = \frac{\ln(1+B)}{B} \quad \rightarrow \quad B = \frac{\Psi_\infty - \Psi_w}{\Psi_w - \Psi_T}$$

where $g^* = (\Gamma/r_w \text{ or } \Gamma/L)$ is value of g when $|B| \rightarrow 0$

- In **Inert MT without HT**, $\Psi = \omega_v$ and $\Gamma = \rho_m D$
- In **Inert MT with HT**, $\Psi = \omega_v$ and h_m and $\Gamma_{mh} = \rho_m D = \rho_m \alpha_m$ with $Le = 1$
- In **MT with SCR**, $\Psi = \text{appropriate } \Phi$ and h_m and $\Gamma_{mh} = \rho_m D = \rho_m \alpha_m$ with $Le = 1$ and equal $c_{p,k} = c_{p,m}$
- In **MT with ACR**, $\Psi = \text{appropriate } \Phi$ and $\Gamma_m = \rho_m D$

So, in some way I would say that we have analyzed all types of mass transfer problems **from the** by converting every problem to a conserved property equation and psi has to be defined appropriately, so that N_w equal to $g B$ with g by g^* equal to $\ln(1+B)/B$ and B is equal to $(\Psi_\infty - \Psi_w) / (\Psi_w - \Psi_T)$ and A is $4\pi r^2$ or a constant.

For inert mass transfer without heat transfer, Ψ was equal to ω_v , as you know, and Γ was simply equal to $\rho_m D$. For inert mass transfer with heat transfer, we had Ψ equal to ω_v and h_m and we made the assumption of Lewis number equal to 1. From mass transfer with heat transfer and chemical reaction we choose Ψ equal to appropriate Φ and mixture enthalpy. We make the Lewis number equal to 1 - assumption and also we say that the specific heats of participating species would be equal to specific heat of mixture and in the mass transfer with arbitrary chemical reaction, we showed that Ψ can be simply appropriate Φ and Γ_m as $\rho_m D$ diffusivity.

So you can see that a variety of problems have been reduced to conserve property relationship through appropriate and justifiable assumption which makes calculation of the mass transfer rate simple and we are able to derive analytically derived relationship connecting N_w , the mass transfer to the driving force B and the relationship we have

found is a logarithmic one. In the next lecture, we will see how, what the Couette flow model has to say.