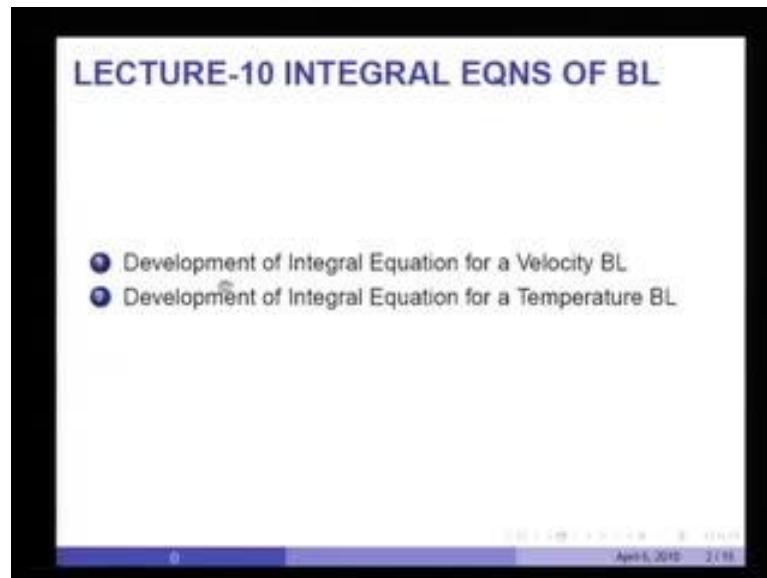


Convective Heat and Mass Transfer
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Module No. # 01
Lecture No. # 10
Integral Equations of BL

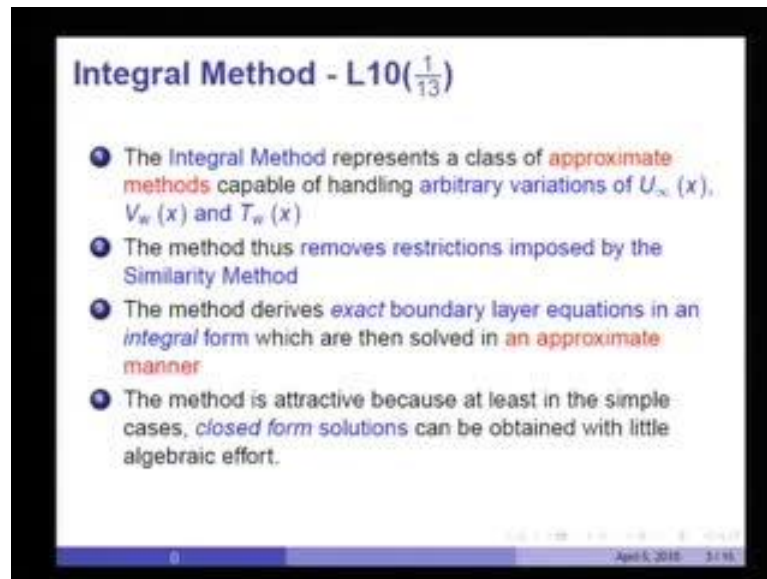
How to derive and solve similarity equations to obtain friction factor and Nusselt number for two-dimensional laminar boundary layers? Now, we will move to the next method; that is, the integral method for solving boundary layer equations. Therefore, the task for this lecture is to derive the integral equations of a boundary layer.

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First is the development of the integral equation for a velocity boundary layer. Next is the development of integral equation for a temperature boundary layer. Solve these equations to obtain friction factor and Nusselt number for two-dimensional laminar boundary layers.

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We now move to the next method, it is the integral method for solving boundary layer equations. Therefore, the task for this lecture is to derive the integral equations of a boundary layer. So, we shall first begin with the development of the equation for a temperature boundary layer.

The integral method represents a class of approximate methods capable of handling arbitrary variations of U_∞ , suction or blowing velocity - V_w and wall temperature variation - T_w . In this respect, you will recall that similarity method permitted only certain type of variations of the free stream velocity, the wall velocity and the wall temperature variations. The method is called approximate method, not because its equations are approximate. The method actually derives exact boundary layer equations, but in integral form and not in differential. It gives them solution methodology, which is approximate and not the equations.

A very important point to remember in all further development, unlike the similarity method, this method is attractive because at least in some simple cases, closed form solutions can be obtained with little algebraic effort. At one time, people used to call the integral because you could do all the calculations by simple algebra without requiring a computer. The second advantage of the method is of course that it can now deal with any arbitrary variations of U_∞ , V_w and T_w .

(Refer Slide Time: 03:45)

Velocity B L - L10($\frac{2}{13}$)

Consider Continuity and Momentum Equations

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad (1)$$

$$\frac{\partial(u u)}{\partial x} + \frac{\partial(v u)}{\partial y} = U_{\infty} \frac{d U_{\infty}}{d x} + \nu \frac{\partial^2 u}{\partial y^2} \quad (2)$$

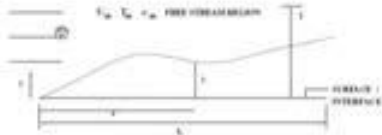
The equations are integrated with respect to y
 from $y = 0$ ($u = 0, v = V_w$)
 to $y = l$ ($u = U_{\infty}, v = V_l$)

where $l > \delta_{max}$ in the region $0 < x < L$ (see next slide)

First of all, let us consider the boundary layer- $\frac{d u}{d x}$ plus $\frac{d v}{d y}$ equal to 0. The convection terms or the momentum transfer terms, the pressure gradient term and the viscous term. These equations are integrated term by term with respect to y of the boundary layer. From y equal to 0, where u is equal to 0 and v is equal to V_w and to y equal to l, where u equals U_{∞} and v equals some fictitious velocity V_l . l is greater than the δ_{max} in the region $0 < x < L$; you will see on the next slide.

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Integral Continuity Eqn - L10($\frac{3}{13}$)



$$\int_0^l \frac{\partial u}{\partial x} dy + \int_0^l \frac{\partial v}{\partial y} dy = 0$$

$$V_l - V_w = - \int_0^l \frac{\partial u}{\partial x} dy = - \frac{\partial}{\partial x} \int_0^l u dy \quad (3)$$

V_l is a Fictitious velocity at $y = l$
 $V_w(x)$ is the Suction/Blowing Velocity

This figure shows a surface on which a boundary layer grows. Of course, a boundary layer development can be uneven. The boundary layer can grow, shrink and then grow depending on the pressure gradient that is applied. When U_∞ varies arbitrarily with x , the boundary layer thickness will vary arbitrarily with x . We always choose to analyze a boundary layer over a given length L . In this length L , we make sure that we choose a dimension ΔL , which is greater than Δx_{max} in this region as shown here.

Integrating continuity equation term by term, we will get $\int_0^l \frac{\partial}{\partial x} (u v) dy$. This term will simply reduce to $V_1 - V_w$ and this term is taken to the right hand side as $-\int_0^l u \frac{\partial u}{\partial x} dy$. I can take the differential out and write it as $-\frac{d}{dx} \int_0^l u^2 dy$. You will see then that $V_1 - V_w$ simply represents the change in the flow rate between two successive stations. V_1 is a fictitious velocity at l and $V_w(x)$ is the familiar suction or blowing velocity.

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Integral Momentum Eqn - 1 - L10($\frac{4}{13}$)

$$\int_0^l \frac{\partial(uv)}{\partial x} dy + \int_0^l \frac{\partial(v^2)}{\partial y} dy = \int_0^l U_\infty \frac{dU_\infty}{dx} dy + \nu \int_0^l \frac{\partial^2 u}{\partial y^2} dy \quad (4)$$

$$\frac{d}{dx} \int_0^l u v dy + U_\infty V_1 - u_w V_w = U_\infty \frac{dU_\infty}{dx} l + \nu \left\{ \left(\frac{\partial u}{\partial y} \right)_{y=l} - \left(\frac{\partial u}{\partial y} \right)_{y=0} \right\}$$

Using *no-slip* condition $u_w = 0$ and noting that $\partial u / \partial y|_{y=l} = 0$

$$\frac{d}{dx} \int_0^l u v dy + U_\infty \left[V_w - \frac{d}{dx} \int_0^l u^2 dy \right] = U_\infty \frac{dU_\infty}{dx} l - \frac{\tau_{wx}}{\rho} \quad (5)$$

If I have to integrate momentum equation term by term, then you will see $\int_0^l \frac{\partial}{\partial x} (u v) dy$ of $u v dy$ plus $\int_0^l \frac{\partial}{\partial y} (v^2) dy$ equal to $\int_0^l U_\infty \frac{dU_\infty}{dx} dy$ plus $\nu \int_0^l \frac{\partial^2 u}{\partial y^2} dy$. On taking the differential out, this term will simply be $\frac{d}{dx} \int_0^l u v dy$ plus $U_\infty V_1 - u_w V_w$ equal to $U_\infty \frac{dU_\infty}{dx} l$ minus $\frac{\tau_{wx}}{\rho}$. Therefore, it can be taken out. $\int_0^l \frac{\partial}{\partial y} (v^2) dy$ would simply yield 1 , whereas this in turn would integrate to $\nu \int_0^l \frac{\partial^2 u}{\partial y^2} dy$ equal to 1 minus $\frac{\tau_{wx}}{\rho}$.

Notice, this term (Refer Slide Time: 07:37) will vanish because u_w is equal to 0. So that term goes to 0 and likewise, du by dy , y at 1 is also equal to 0. The term that survives is mu by ρ du by dy at y equal to 0. mu into du by dy at y equal to 0, it is nothing but the shear stress divided by ρ and that is what you see on the right hand side. This term is represented here and I have replaced V_1 into U_∞ . From the previous result as shown here, V_1 is V_w minus d by dx of u dy and of course, this term survives.

(Refer Slide Time: 08:38)

Integral Momentum Eqn - 2 - L10($\frac{5}{13}$)
 Identity: $U_\infty \frac{dU_\infty}{dx} = U_\infty \frac{dU_\infty}{dx} \int_0^1 dy = \frac{dU_\infty}{dx} \int_0^1 U_\infty dy$
 Hence,

$$\frac{d}{dx} \int_0^1 u u dy + U_\infty \left[V_w - \frac{d}{dx} \int_0^1 u dy \right] = \frac{dU_\infty}{dx} \int_0^1 U_\infty dy - \frac{\tau_{wx}}{\rho}$$

 Identity:

$$\frac{d}{dx} \int_0^1 u U_\infty dy = \frac{dU_\infty}{dx} \int_0^1 u dy + U_\infty \frac{d}{dx} \int_0^1 u dy$$

 Hence,

$$\frac{d}{dx} \int_0^1 u(u - U_\infty) dy + \frac{dU_\infty}{dx} \int_0^1 (u - U_\infty) dy = -\frac{\tau_{wx}}{\rho} - U_\infty V_w$$

We will move further. $U_\infty \frac{dU_\infty}{dx}$ is nothing but $U_\infty \frac{dU_\infty}{dx}$ from 0 to 1 dy . Since U_∞ is not a function of y , I can absorb it inside the integral and write it as $\frac{dU_\infty}{dx} \int_0^1 U_\infty dy$. Then, I would read the first term plus U_∞ into V_w minus d by dx into u dy . Now, this term is replaced by **$\frac{dU_\infty}{dx} \int_0^1 U_\infty dy$**

Now, consider the identity d by dx of a product. $\int_0^1 u dy$ plus $U_\infty \frac{d}{dx} \int_0^1 u dy$. This is precisely the term you see here; so, I am going to replace here. If I do it, then you will notice that I can write this equation in this manner, d by dx $\int_0^1 u(u - U_\infty) dy$ plus $\frac{dU_\infty}{dx} \int_0^1 (u - U_\infty) dy$. This equals to $-\frac{\tau_{wx}}{\rho}$ and $-U_\infty V_w$.

(Refer Slide Time: 10:20)

Integral Momentum Eqn- 3 - L10 ($\frac{6}{13}$)
 Divide and Multiply by the same quantity

$$\frac{d}{dx} \left[U_\infty^2 \int_0^{\delta} \frac{u}{U_\infty} \left(\frac{u}{U_\infty} - 1 \right) dy \right] + U_\infty \frac{dU_\infty}{dx} \int_0^{\delta} \left(\frac{u}{U_\infty} - 1 \right) dy = - \left(\frac{\tau_{wx}}{\rho} + V_w U_\infty \right)$$

Recall

- $\delta_1 = \int_0^{\infty} \left(1 - \frac{u}{U_\infty} \right) dy = \int_0^{\delta} \left(1 - \frac{u}{U_\infty} \right) dy$
- $\delta_2 = \int_0^{\infty} \frac{u}{U_\infty} \left(1 - \frac{u}{U_\infty} \right) dy = \int_0^{\delta} \frac{u}{U_\infty} \left(1 - \frac{u}{U_\infty} \right) dy$

Hence,

$$\frac{d}{dx} \left[U_\infty^2 \delta_2 \right] + U_\infty \frac{dU_\infty}{dx} \delta_1 = \left(\frac{\tau_{wx}}{\rho} + V_w U_\infty \right)$$

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If I divide and multiply each term by the same quantity that is If I multiply this equation by U infinity square and divide by U infinity square and if I multiply this equation by U infinity and divide by U infinity, then you will notice that I can write the equation in this manner. It is written as d by dx of U infinity square u over U infinity into u over U infinity minus 1 dy plus U infinity d U infinity by dx equal to 0 to 1 u over U infinity minus 1 dy and all this.

Recall that this integral is nothing but our momentum thickness. This term is nothing but minus delta 1 or the displacement thickness. Therefore, this term will become minus d by dx into U infinity square delta 2. This term will become minus U infinity d U infinity by dx into delta 1. So, canceling the minus sign, which appears in each term, we would have the equation which looks like this (Refer Slide Time: 11:30) d by dx U infinity square delta 2 plus U infinity d u by d U infinity by dx delta 1 equal to the shear stress term and the suction or blowing term. I will further manipulate this equation as shown on the next slide.

(Refer Slide Time: 12:12)

Integral Momentum Eqn- 4 - L10 (7 / 13)

Dividing throughout by U_∞^2

$$\frac{d \delta_2}{dx} + \frac{1}{U_\infty} \frac{d U_\infty}{dx} (2 \delta_2 + \delta_1) = \frac{C_{f,x}}{2} + \frac{V_w}{U_\infty} \quad (6)$$

- 1 This is known as Integral Momentum Eqn
- 2 It is an Exact Equation - No assumptions are introduced
- 3 It is an ODE. Thus, PDEs of the BL are converted to an ODE for integral parameter δ_2
- 4 $C_{f,x} = \tau_{w,x} / (\rho U_\infty^2 / 2)$

If I divide by U_∞^2 and open up this differential of a product as $U_\infty^2 \frac{d \delta_2}{dx} + \delta_2 \frac{d U_\infty^2}{dx}$, then you will notice that the equation will be read as $d \delta_2 + \frac{1}{U_\infty} d U_\infty (2 \delta_2 + \delta_1) = \frac{C_{f,x}}{2} + \frac{V_w}{U_\infty}$.

So, each term now has dimension as $x U_\infty$. U_∞ has the velocity dimension. δ_2 and δ_1 have length dimension and so does x . The terms on the right hand side completely dimensionless. It is this equation, which is known as the integral momentum equation. We have an exact equation because we have not introduced any assumptions in its derivation. We have simply integrated the partial differential equation from 0 to 1 because all quantities vary only with x .

The partial differential equation of the boundary layer is converted to an ODE for an integral parameter δ_2 . This is very similar to what we did in similarity method, where the PDEs were converted to third order ordinary differential equation. Here, the equations are converted to a first order ordinary differential equation for an integral parameter δ_2 . $C_{f,x}$ is the coefficient of friction and that is simply defined as $\tau_w x$ over ρU_∞^2 divided by 2. So, it is well known to you and this is the integral momentum equation.

(Refer Slide Time: 14:05)

Integral Kinetic Energy Eqn-1 - L10($\frac{8}{13}$)

$$\rho \left[u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right] = \rho U_{\infty} \frac{d U_{\infty}}{d x} + \mu \frac{\partial^2 u}{\partial y^2} \quad (7)$$

Multiply by u / ρ throughout

$$\frac{\partial u}{\partial x} E + \frac{\partial v}{\partial y} E = u U_{\infty} \frac{d U_{\infty}}{d x} + \nu u \frac{\partial^2 u}{\partial y^2} \quad E = \frac{u^2}{2} \quad (8)$$

Integrate from $y = 0$ to $y = \delta$ and note that $(vE)_{y=0} = 0$

$$\begin{aligned} \frac{d}{dx} \left[\int_0^{\delta} \left(\frac{u^3}{2} \right) dy \right] + \left[V_w - \frac{\partial}{\partial x} \int_0^{\delta} u dy \right] \frac{U_{\infty}^2}{2} \\ = \frac{d U_{\infty}}{dx} \int_0^{\delta} u U_{\infty} dy \\ + \nu \int_0^{\delta} u \left(\frac{\partial^2 u}{\partial y^2} \right) dy \end{aligned} \quad (9)$$

There is another variant of the equation, which is called the integral kinetic energy equation. It can be derived from the earlier momentum equation, but I will derive it from first principles. Our momentum equation reads like this, where these are the terms. If I multiply each term by u divided by ρ throughout, then you will see and define E as u square by 2. You will see this term can be simply written as $du E$ by dx , this term can be written as $dv E$ by dy equal to U infinity $d U$ infinity by dx plus μ into u d square u by dy square. I repeat E is the energy of the actual velocity, u square by 2.

If we integrate this equation from y equal to 0 to y equal to δ , where it does not really matter what we do. This term will simply become u into u square by 2, so that becomes u cube by 2 d by dx integral 0 to δ dy . This term will be $v E$ at δ minus $v E$ at y equal to 0, but at y equal to 0, u is equal to 0. Therefore, E is equal to 0 and that term will go up. V into E at 1 or infinity is simply V w minus d by dx of 0 to δ u dy into U infinity square by 2. It would be equal to $d U$ infinity by dx into $u U$ infinity dy plus μ integral 0 to δ u into d square u by dy square dy . It is this last term, which I shall... because this is just in result on the next slide.

(Refer Slide Time: 16:25)

Integral Kinetic Energy Eqn-2 - L10($\frac{9}{13}$)

Integration by parts gives

$$\nu \int_0^{\delta} u \left(\frac{\partial^2 u}{\partial y^2} \right) dy = -\nu \int_0^{\delta} \left(\frac{\partial u}{\partial y} \right)^2 dy$$

Define *Kinetic Energy Thickness* δ_3 as

$$\delta_3 \equiv \int_0^{\infty} \frac{u}{U_{\infty}} \left[1 - \left(\frac{u}{U_{\infty}} \right)^2 \right] dy \quad (10)$$

Manipulation gives Integral Kinetic Energy Eqn

$$\frac{d}{dx} (U_{\infty}^3 \delta_3) = V_w U_{\infty}^2 + 2\nu \int_0^{\delta} \left(\frac{\partial u}{\partial y} \right)^2 dy \quad (11)$$

Integration by parts will give $\nu \int_0^{\delta} u \frac{\partial^2 u}{\partial y^2} dy = -\nu \int_0^{\delta} \left(\frac{\partial u}{\partial y} \right)^2 dy$. Now, in the integral kinetic energy equation, we introduce another thickness called the kinetic energy thickness. It is defined very much like the momentum thickness, except that we now have $1 - \frac{u}{U_{\infty}}$ over U_{∞}^2 , which represents the kinetic energy deficit caused by $\left(\frac{u}{U_{\infty}} \right)^2$ of the previous equation (Refer Slide Time: 17:11), which is shown here. You will see that this can be manipulated and read as $\frac{d}{dx} (U_{\infty}^3 \delta_3) = V_w U_{\infty}^2 + 2\nu \int_0^{\delta} \left(\frac{\partial u}{\partial y} \right)^2 dy$.

(Refer Slide Time: 17:51)

Integral Energy Eqn - 1 - L10(10/13)

Diagram showing velocity profile \$u\$ vs \$y\$ with boundary layer thickness \$\delta\$ and thermal boundary layer thickness \$l\$. The velocity profile is shown as a solid line, and the thermal boundary layer is shown as a dashed line. The wall temperature is \$T_w\$ and the free stream temperature is \$T_\infty\$.

$$\frac{\partial(uT)}{\partial x} + \frac{\partial(vT)}{\partial y} = \alpha \frac{\partial^2 T}{\partial y^2} + \frac{\nu}{C_p} \left(\frac{\partial u}{\partial y}\right)^2 \quad (12)$$

Define $\theta = (T - T_\infty)/(T_w - T_\infty)$ $T_\infty = \text{constant}$, $T_w = F(x)$

$$\frac{\partial(u\theta)}{\partial x} + \frac{\partial(v\theta)}{\partial y} + \frac{u\theta}{(T_w - T_\infty)} \frac{d}{dx}(T_w - T_\infty) = \alpha \frac{\partial^2 \theta}{\partial y^2} + \frac{\nu}{C_p(T_w - T_\infty)} \left(\frac{\partial u}{\partial y}\right)^2 \quad (13)$$

The equation is integrated with respect to y from $y = 0$ to $y = l$ where $l > \delta_{\max}$ for $Pr > 1$ and $l > \Delta_{\max}$ for $Pr < 1$.

April 8, 2010 12/18

Integral kinetic energy equation is sometimes used in boundary layers analysis of boundary layers with suction and blowing. We now turn to integral energy equation. Again, consider the surface on which a boundary layer is developing. This solid line represents the velocity boundary layer. Now, the thermal boundary layer will either have a thickness greater than the velocity boundary layer thickness or smaller than velocity boundary layer thickness. You already know from your similarity solutions that when Prandtl number is greater than 1, the thermal boundary layer thickness is smaller than the velocity boundary layer thickness; whereas, if Prandtl number is less, boundary layer thickness is greater than the momentum or the velocity boundary layer thickness δ .

So, while integrating this energy equation, which includes this viscous dissipation term. We will choose l big enough, so that it is either greater than δ or it is greater than thermal boundary layer thickness, capital Δ for Prandtl number less than 1. We define for convenience that θ equal to T , where T_∞ is constant with x , but T_w is some function of x and I will introduce that function shortly.

This equation (Refer Slide Time: 19:21) will simply become $du \theta$ by dx . Additional term is arising out of the fact that T_w is a function of x . Therefore, you will have a term called by T_w minus T_∞ into d by dx of T_w minus T_∞ . This term will become $\alpha \delta^2 \theta$ by dy square. This term will become μ into $C_p T_w$ minus T_∞ du by dy whole square. It is this equation that we shall integrate from 0 to l ,

where l is greater than δ_{max} for Prandtl equal to 1, Prandtl greater than 1 and l is greater than δ_{max} for Prandtl less than 1.

(Refer Slide Time: 20:39)

Integral Energy Eqn - 2 - L10(11/13)

Integration gives

$$\frac{d}{dx} \left[U_{\infty} \int_0^l \frac{u}{U_{\infty}} \theta dy \right] + V_l \theta_l - V_w \theta_w$$

$$+ \frac{\rho_1}{(T_w - T_{\infty})} \frac{d}{dx} (T_w - T_{\infty}) \left\{ U_{\infty} \int_0^l \frac{u}{U_{\infty}} \theta dy \right\}$$

$$= \alpha \left\{ \left(\frac{\partial \theta}{\partial y} \right)_{y=l} - \left(\frac{\partial \theta}{\partial y} \right)_{y=0} \right\} + \frac{\nu}{C_p} \int_0^l \left(\frac{\partial u}{\partial y} \right)^2 dy \quad (14)$$


- Recall $\Delta_2 = \int_0^{\infty} \frac{u(T - T_{\infty})}{U_{\infty}(T_w - T_{\infty})} dy = \int_0^l \frac{u}{U_{\infty}} \theta dy$
- $\theta_l = \theta_{\infty} = 0$ and $\theta_w = 1$
- $\alpha \left(\frac{\partial \theta}{\partial y} \right)_{y=l} = 0$ and $\alpha \left(\frac{\partial \theta}{\partial y} \right)_{y=0} = -\frac{q_w}{\rho C_p (T_w - T_{\infty})} = -\frac{h_0}{\rho C_p}$

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Let us do the integration on the next slide. You will notice that integration of this will simply become d by dx of integral $u \theta dy$. If I divide and multiply by U infinity, it will be read as d by dx of U infinity from 0 to l U infinity u by U infinity θdy .

(Refer Slide Time: 20:49)

Integral Energy Eqn - 1 - L10(10/13)



$$\frac{\partial(uT)}{\partial x} + \frac{\partial(vT)}{\partial y} = \alpha \frac{\partial^2 T}{\partial y^2} + \frac{\nu}{C_p} \left(\frac{\partial u}{\partial y} \right)^2 \quad (12)$$

Define $\theta = (T - T_{\infty}) / (T_w - T_{\infty})$ $T_{\infty} = \text{constant}$, $T_w = F(x)$

$$\frac{\partial(u\theta)}{\partial x} + \frac{\partial(v\theta)}{\partial y} + \frac{u\theta}{(T_w - T_{\infty})} \frac{d}{dx} (T_w - T_{\infty})$$

$$= \alpha \frac{\partial^2 \theta}{\partial y^2} + \frac{\nu}{C_p(T_w - T_{\infty})} \left(\frac{\partial u}{\partial y} \right)^2 \quad (13)$$

The equation is integrated with respect to y from $y = 0$ to $y = l$ where $l > \delta_{max}$ for $Pr > 1$ and $l > \Delta_{max}$ for $Pr < 1$.

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The integration of this term will simply give $V_l \theta_l - V_w \theta_w$ and θ_w as is 1. you will see $V_l \theta_l - V_w \theta_w$ plus this term will be simply $\frac{1}{T_w - T_\infty} \frac{d}{dx} (T_w - T_\infty) \int_0^l \frac{u}{U_\infty} \theta dy$ and that is what I have written and $\alpha \frac{d\theta}{dy}|_{y=l}$ I have divided by U_∞ and multiplied by U_∞ .

(Refer Slide Time: 21:23)

Integral Energy Eqn - 2 - L10(11/13)
Integration gives

$$\begin{aligned} & \frac{d}{dx} \left[U_\infty \int_0^l \frac{u}{U_\infty} \theta dy \right] + V_l \theta_l - V_w \theta_w \\ & + \frac{1}{(T_w - T_\infty)} \frac{d}{dx} (T_w - T_\infty) \left\{ U_\infty \int_0^l \frac{u}{U_\infty} \theta dy \right\} \\ & = \alpha \left\{ \left(\frac{\partial \theta}{\partial y} \right)_{y=l} - \left(\frac{\partial \theta}{\partial y} \right)_{y=0} \right\} + \frac{\nu}{C_p} \int_0^l \left(\frac{\partial u}{\partial y} \right)^2 dy \quad (14) \end{aligned}$$

- Recall $\Delta_2 = \int_0^\infty \frac{u(T - T_\infty)}{U_\infty(T_w - T_\infty)} dy = \int_0^l \frac{u}{U_\infty} \theta dy$
- $\theta_l = \theta_\infty = 0$ and $\theta_w = 1$
- $\alpha \left(\frac{\partial \theta}{\partial y} \right)_{y=l} = 0$ and $\alpha \left(\frac{\partial \theta}{\partial y} \right)_{y=0} = -\frac{q_w}{\rho C_p (T_w - T_\infty)} = -\frac{q_w}{\rho C_p}$

You will see $V_l \theta_l - V_w \theta_w$ plus this term (Refer Slide Time: 21:12) will be simply $\frac{1}{T_w - T_\infty} \frac{d}{dx} (T_w - T_\infty) \int_0^l \frac{u}{U_\infty} \theta dy$ and that is what I have written. I have divided by U_∞ and multiplied by U_∞ . The diffusion term will simply give $\alpha \frac{d\theta}{dy}|_{y=l} - \alpha \frac{d\theta}{dy}|_{y=0}$ plus $\frac{\nu}{C_p} \int_0^l \left(\frac{\partial u}{\partial y} \right)^2 dy$.

Now, if you recall that our enthalpy thickness Δ_2 was defined as $\int_0^\infty \frac{u(T - T_\infty)}{U_\infty(T_w - T_\infty)} dy = \int_0^l \frac{u}{U_\infty} \theta dy$. In other words, this quantity is nothing but Δ_2 because of its definition as Δ_2 .

Therefore that term vanishes, whereas θ_w is equal to 1 because the temperature gradient at the infinities and at the edge of the boundary layer is 0. What this term means is that $\alpha \frac{d\theta}{dy}|_{y=0}$ multiplied by $\alpha \frac{d\theta}{dy}|_{y=l}$. It will be simply equal to $\frac{q_w}{\rho C_p (T_w - T_\infty)}$. Therefore, q_w into ρC_p with a negative sign and q_w divided by $T_w - T_\infty$ is nothing but local heat transfer

coefficient h_x divided by ρ into C_p . So, this term (Refer Slide Time: 22:58) will be replaced by $h_x / \rho C_p$.

(Refer Slide Time: 23:03)

Integral Energy Eqn - 3 - L10(12/13)

Substitution gives

$$\frac{d}{dx} [U_\infty \Delta_2] + \frac{U_\infty \Delta_2}{(T_w - T_\infty)} \frac{d}{dx} (T_w - T_\infty) = \frac{h_x}{\rho C_p} + V_w + \frac{\nu}{C_p} \int_0^1 \left(\frac{\partial u}{\partial y}\right)^2 dy \quad (15)$$

Division by U_∞ gives Integral Energy Eqn

$$\frac{d \Delta_2}{d x} + \Delta_2 \left[\frac{1}{(T_w - T_\infty)} \frac{d}{d x} (T_w - T_\infty) + \frac{1}{U_\infty} \frac{d U_\infty}{d x} \right] = St_x + \frac{V_w}{U_\infty} + 2 Ec_x \frac{\nu}{U_\infty^2} \int_0^1 \left(\frac{\partial u}{\partial y}\right)^2 dy \quad (16)$$

This ODE is Exact, $St_x = h_x / (\rho C_p U_\infty) = Nu_x / (Re_x Pr)$

You will see the equation is now read as d by dx of $U_\infty \Delta_2$ plus $U_\infty \Delta_2$ into $T_w - T_\infty$ into d by dx into $T_w - T_\infty$ is equal to $h_x / \rho C_p$ plus V_w plus ν / C_p into integral 0 to 1 **du by dy** whole square into dy .

If I divide by U_∞ after opening this differential, then I would get the integral energy equation as $d \Delta_2$ by dx plus Δ_2 into the wall variation term into the free stream variation or the pressure gradient term - Stanton x . It is $h_x / \rho C_p U_\infty$. I have shown here, which is nothing but Nusselt number divided by Reynolds number into Prandtl number and we have seen this before.

The wall velocity variation term, V_w / U_∞ and the viscous dissipation term is this. Each term here is dimensionless (Refer Slide Time: 24:24) and as you can see, this is dimensionless and this is dimensionless, Δ_2 is dimensionless, this is dimensionless. Is this in dimension? It can be shown quite easily. Now, each term is dimensionless and therefore, we have an ordinary differential equation for capital Δ_2 analogous to an ODE for small Δ_2 in the velocity boundary layer equation.

(Refer Slide Time: 25:02)

Summary of Integral Eqns - L10(13/13)

Continuity eqn

$$V_1 - V_w = -\frac{d}{dx} \int_0^l u dy \quad (17)$$

Momentum Eqn

$$\frac{d \delta_2}{dx} + \frac{1}{U_\infty} \frac{d U_\infty}{dx} (2 \delta_2 + \delta_1) = \frac{C_{f,x}}{2} + \frac{V_w}{U_\infty} \quad (18)$$

Energy Equation

$$\begin{aligned} \frac{d \Delta_2}{dx} + \Delta_2 \left[\frac{1}{(T_w - T_\infty)} \frac{d}{dx} (T_w - T_\infty) + \frac{1}{U_\infty} \frac{d U_\infty}{dx} \right] \\ = St_x + \frac{V_w}{U_\infty} + 2 Ec_x \frac{\nu}{U_\infty^2} \int_0^l \left(\frac{\partial u}{\partial y} \right)^2 dy \quad (19) \end{aligned}$$

Let me summarize all the equations that we have derived. Integral continuity equation simply gives $V_1 - V_w$ equal to minus d by dx of 0 to l u dy . The integral momentum equation gives variation of $d \delta_2$ by dx and accounts for the wall velocity. The integral energy equation gives the rate of change of enthalpy thickness as Δ_2 and accounts for the wall temperature variation, the free stream variation, wall velocity variation and the viscous dissipation. Our interest is always to determine Stanton x and C_x . In the next lecture, I will take the solution of the velocity boundary layer equations in their integral form. Thank you.