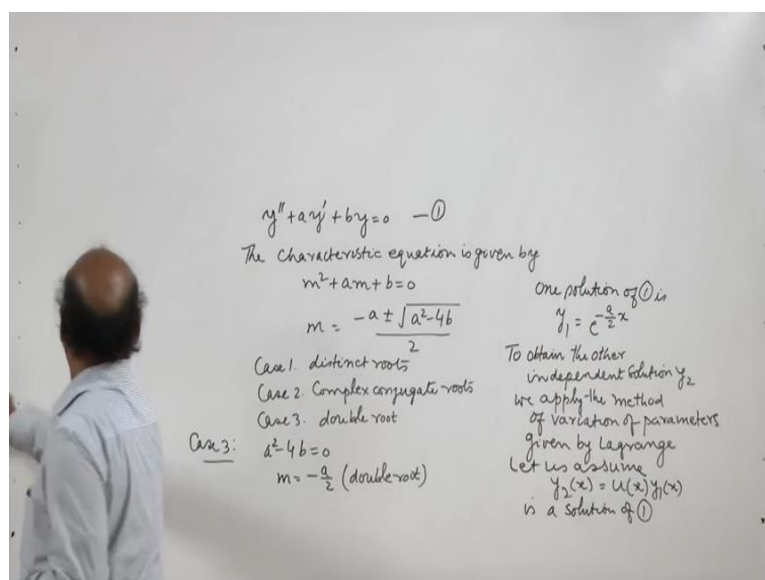


Mathematical methods and its applications
Dr. P. N. Agrawal
Department of Mathematics
Indian Institute of Technology, Roorkee

Lecture – 04
Solution of second order homogenous
linear differential equation with constant coefficients II

Hello friends. Welcome to my second lecture on Solution of Second Order Homogenous Linear Differential Equation with Constant Coefficients.

(Refer Slide Time: 00:30)



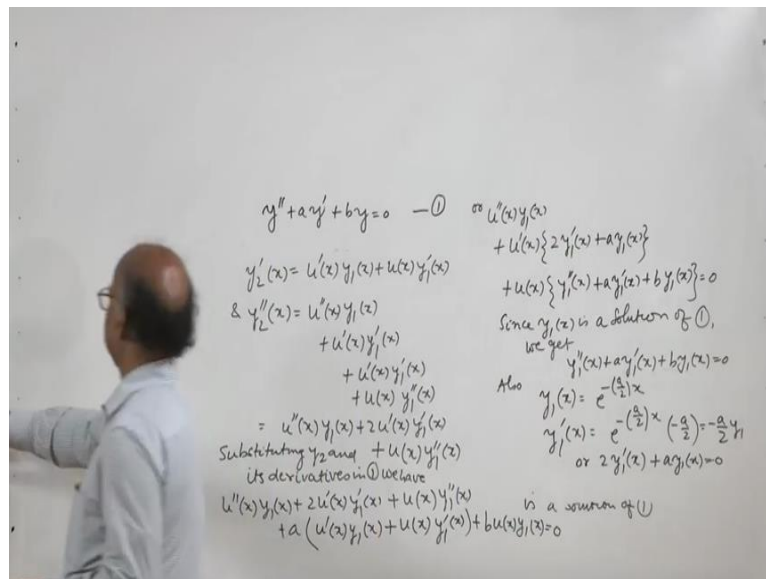
We are discussing the solution of second order homogenous linear differential equation with constant coefficients that is given by $y'' + ay' + by = 0$. The characteristic equation here is given by $m^2 + am + b = 0$. Let us find the roots of this quadratic equation $m = \frac{-a \pm \sqrt{a^2 - 4b}}{2}$. So, there are two values of m , we have three cases - case 1 distinct roots, case 2 complex conjugate roots, and then case 3 double root. So, the case 1, when the two roots of the characteristic equation are distinct, we have already covered in previous lecture; and the case 2, where the two roots are complex conjugate that also we have covered in the previous lecture.

So, in this lecture, we begin with the case 3. Let us say that the root of the quadratic the characteristic equation has double root that is in the case 3 the discriminant a square

minus 4 b equal to 0, in this case the discriminant a square minus 4 b equal to 0. So, we have m equal to minus a by 2 which is a double root. Now this double root at first gives us one solution of the second order linear differential equation one. We can say the one solution is one solution of the given solution one, we can write as y 1 equal to e to the power minus a by 2 into x.

Now to obtain the other independent solution by 2, we apply the method of variation of parameters this method of variation of parameters was given by Lagrange for first order linear differential equation with first order linear differential equation. So, in this method, what we do is let us assume the other solution by 2 x to be equal to u x into y 1 x, we shall find out u x, so that y 2 is the other independent solution of the given differential equation. So, let us assume that y 2 x is equal to u x into y 1 x is the solution of equation 1

(Refer Slide Time: 05:11)

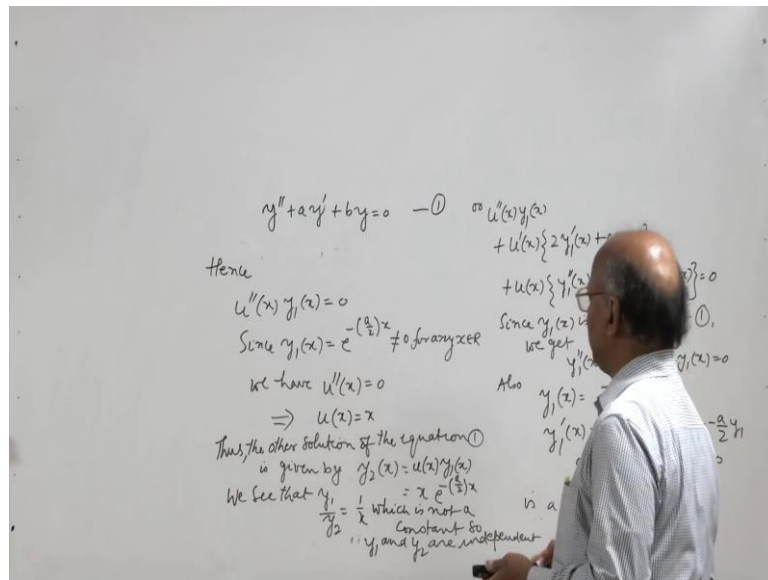


Then we shall have let us substitute y 2 x. So, let us y 2 x gives y 2 dash x equal to u dash x into y 1 x plus u x into y 1 dash x; and y 2 double dash x equal to u double dash x into y 1 x plus u dash x into y 1 dash x plus u dash x into y 1 dash x plus u x into y 1 double dash x. Or we can say this u double dash x into y 1 x plus 2 times u dash x into y 1 dash x plus u x into y 1 double dash x. Now, let us substitute the values of y 2, y 2 dash, y 2 double dash in equation 1. So, substituting y 2 and its derivatives in 1, we have u double dash x into y 1 x plus 2 u dash x into y 1 dash x plus u x y 1 double dash x plus

a times y dash is y 2 dash. So, y 2 dash is equal to u dash x into y 1 x plus u x into y 1 dash x plus b times, y is replace by y 2, and y 2 is u into y 1.

Now, let us collect the coefficients of the derivatives of u. So, we may write this equation as u double dash, the coefficients of u dash x are 2 y 1 dash x plus a times y 1 x, the coefficient of u x is y 1 double dash x plus a y 1 dash x plus b y 1 x equal to 0. Now, since y 1 is a solution of equation 1, we get y 1 double dash x plus a y 1 dash x plus b y 1 x equal to 0; also y 1 x is equal to e to the power minus a by 2 into x. So, let us differentiate it with respect to x, it will give me which is equal to minus a by 2 y 1 or we can say 2 y 1 dash x plus a y 1 x equal to 0.

(Refer Slide Time: 10:15)



So, substituting these values here, it reduces we get u double dash x into y 1 x equal to 0; the coefficient of u dash x is 0 from here, the coefficient of u x is 0 from here. So, u double dash x into y 1 x is equal to 0. Since, y 1 x is e to the power minus a y 2 into x which is not equal to 0 for any x belonging to R, we have u double dash x equal to 0. So, when we integrate this twice, we get which implies u x equal to x, thus the other solution of the equation 1 is given by x into e to the power minus a by 2 into x. So, this is for one solution and then we to find the other solution we applied the method of variation parameters where we assumed y 2 x equal to u x into y 1 x.

(Refer Slide Time: 12:00)

To find another solution y_2 , we apply the method of variation of parameters.

Method of variation of parameters:
In this method, using the known function $y_1(x)$ we try to determine a function $u(x)$ such that

$$y_2(x) = u(x)y_1(x).$$

IT ROORKEE NPTEL ONLINE CERTIFICATION COURSE 3

So, the now let us see that y_1 and y_2 we see that y_1 over y_2 is equal to 1 over x which is not a constant. So, the y_1 and y_2 are independent of two each other.

(Refer Slide Time: 12:51)

$y'' + ay' + by = 0 \quad \text{--- (1)}$

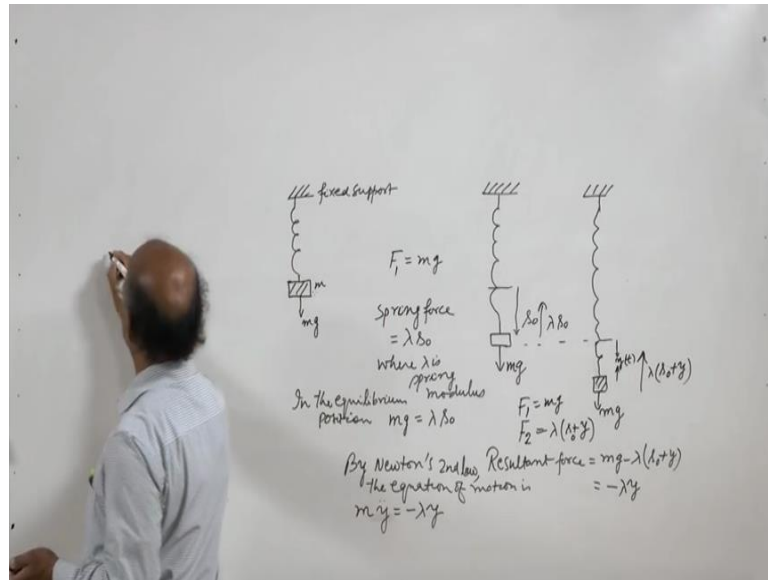
$y_1(x) = 0$
 $y_2(x) = e^{-\left(\frac{a}{2}\right)x} \neq 0$ for any $x \in \mathbb{R}$

have $u'(x) = 0$
 $u(x) = x$
solution of the equation (1)
by $y_2(x) = u(x)y_1(x)$
 $y_2 = x e^{-\left(\frac{a}{2}\right)x}$
 $\frac{y_2}{y_1} = \frac{1}{x}$ which is not a constant so y_1 and y_2 are independent.

Hence, the general solution of (1) is
 $y(x) = (c_1 + c_2 x) e^{-\left(\frac{a}{2}\right)x}$
where c_1 and c_2 are arbitrary constants.

So, we got two independent solutions of the equation 1, and therefore, we can write the general solution as $y(x)$ equal to $c_1 + c_2 x$ into $e^{-\frac{a}{2}x}$, where c_1 and c_2 are arbitrary constants.

(Refer Slide Time: 15:37)



Now, let us take an example on this. First due to gravity which will act downward will be mg . So, let us take F_1 equal to mg . Now, due to the mass m that is due to the weight mg there will be an extension in this spring, before it comes to the equilibrium position. So, let us say in the equilibrium position, the force due to gravity that is F_1 equal to mg will act downward, while the spring force the spring force will be λs_0 where this spring force is, when λ is spring modulus, and s_0 is the extension or stretch in the spring.

So, let us say this is s_0 . This spring force will act in the direction of opposite to mg . So, in the equilibrium position mg and λs_0 will balance each other, and we shall have mg equal to λs_0 . Here, we are using Hooke's law, which says that the restoring force of the spring or we can also call it as the spring force is proportional to the stretch or extension of the spring.

(Refer Slide Time: 17:31)

Applications

1. Motion of a vertical spring

Consider an ordinary spring which is suspended vertically from a fixed support. Suppose a mass is suspended from the lower end of the spring.



Let $F_1 = mg$, the force due to gravity.

In the equilibrium position

$$mg = \lambda s_0,$$

where s_0 is the extension in the spring due to weight of the mass and λ is the spring modulus.

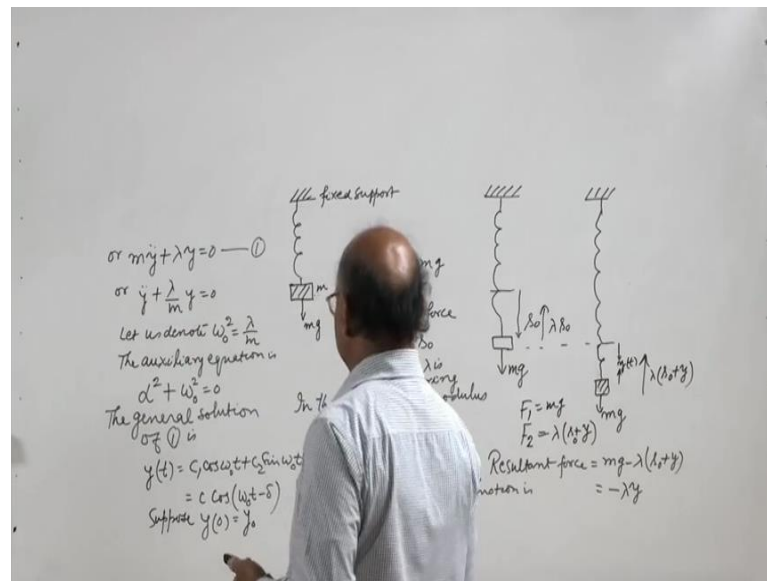
By **Hook's law**, the restoring force of the spring is proportional to the stretch or extension of the spring.

5

So, here there is a stretch or extension s_0 in this spring. So, the spring force or restoring force is proportional to s_0 and therefore, we write it as equal to λs_0 by λ is constant or proportionality we call it as a spring modulus. Now, let us pull the mass m downward and then release it. So, the mass m is pulled down and then released, then there will be a vertical motion in the spring the forces that will be obtained let say at some time t during the motion of the spring y may be displacement from the equilibrium position at time t . So, then the forces that will be acting on the mass m will be the force due to gravity mg downwards f_1 equal to mg , f_2 equal to this spring force which will be $\lambda (s_0 + y)$ because $s_0 + y$ is the total extension in the a spring.

So, this $\lambda (s_0 + y)$ will act in the direction opposite to mg . The resultant force therefore is since mg is equal to λs_0 , we have resultant force as minus λy . Now, let us apply the Newton's second law, by Newton's second law, resultant force is equal to $m \frac{d^2 y}{dt^2}$.

(Refer Slide Time: 20:18)



So, we have the equation of motion as $m \ddot{y} = -\lambda y$ or $\ddot{y} + \frac{\lambda}{m} y = 0$. Now, we can also write it as $\ddot{y} + \omega_0^2 y = 0$. So, if we denote ω_0 , let us denote by $\omega_0^2 = \frac{\lambda}{m}$ then we have the auxiliary equation is $d^2 + \omega_0^2 = 0$. So, alpha square plus omega naught square equal to 0.

And now since this equation has two complex roots. So, we have the general solution of equation 1 as $y(t) = c_1 \cos(\omega_0 t) + c_2 \sin(\omega_0 t)$. Now, it can also be expressed as $c \cos(\omega_0 t - \delta)$, where c is equal to $\sqrt{c_1^2 + c_2^2}$ and $\tan \delta = \frac{c_2}{c_1}$ it can be easily checked. Now, if $y(0) = y_0$ that means, when we release the mass m at that time $t = 0$, the displacement of the mass m from the equilibrium position is y_0 and the velocity at that instant is given by $\dot{y}(0) = y_1$, then we can see from here, then put $t = 0$ in this we get $y_0 = c \cos \delta$, $y_1 = -c \omega_0 \sin \delta$.

And from $y(t) = c \cos(\omega_0 t - \delta)$, if you use $\dot{y}(0) = y_1$, you get $y_1 = -c \omega_0 \sin \delta$.

(Refer Slide Time: 23:18)


$$y(t) = c_1 \cos \omega_0 t + c_2 \sin \omega_0 t, \quad \omega_0 = \sqrt{\frac{\lambda}{m}}$$

or $y(t) = c \cos(\omega_0 t - \delta)$

where $c = \sqrt{c_1^2 + c_2^2}$ and $\delta = \tan^{-1} \frac{c_2}{c_1}$.

if $y(0) = y_0$ and $y'(0) = y_1$,

then $y_0 = c \cos \delta$ and $y_1 = c \omega_0 \sin \delta$


$$c^2 = y_0^2 + \frac{y_1^2}{\omega_0^2} \quad \text{and} \quad \delta = \tan^{-1} \frac{y_1}{y_0 \omega_0}.$$


So, from these two equations, y naught equal to $c \cos \delta$ and y_1 equal to $c \omega_0 \sin \delta$, we can find the value of c , c^2 is equal to $y_0^2 + y_1^2 / \omega_0^2$ and $\delta = \tan^{-1} y_1 / y_0 \omega_0$. So, the two constants c and δ can be found from the initial conditions $y(0) = y_0$ and $y'(0) = y_1$. Or we can also put c as $1 / \omega_0 \sqrt{y_1^2 + \omega_0^2 y_0^2}$ until $\delta = \tan^{-1} y_1 / y_0 \omega_0$ to this.

(Refer Slide Time: 23:53)

Or then $c = \frac{1}{\omega_0} \sqrt{y_1^2 + \omega_0^2 y_0^2}$ and $\delta = \tan^{-1} \frac{y_1}{y_0 \omega_0}$.

The corresponding motion is a harmonic oscillation.



Now, we can see here that since $y(t)$ is equal to $c \cos(\omega t - \delta)$, the motion is a harmonic oscillation.

(Refer Slide Time: 24:18)

2. Damped oscillation

Suppose the mass is connected to a dash pot then we have to consider the corresponding viscous damping into account due to the viscosity of the liquid in the dash pot.

The damping force acts in the direction opposite to the instantaneous motion and is proportional to the viscosity when it is small.

Thus, we have

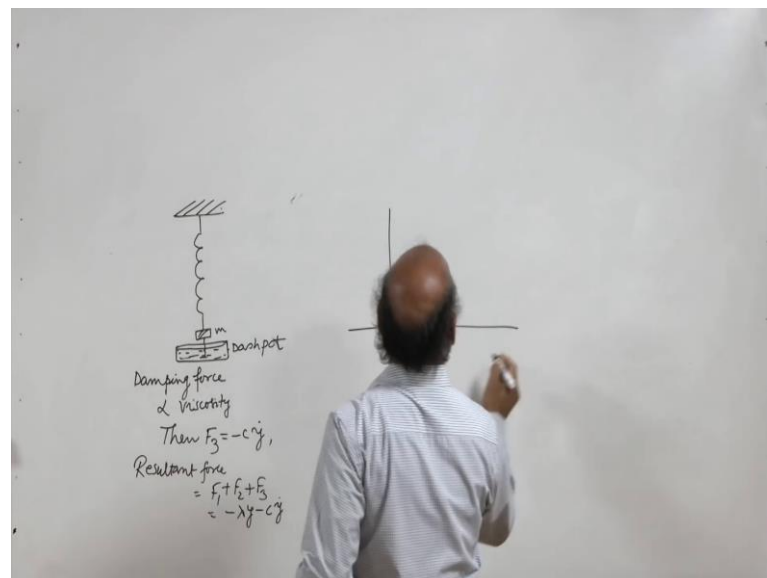
$$F_3 = -cy'$$

where c is a damping constant.

IT ROORKEE NPTEL ONLINE CERTIFICATION COURSE 9

Now, let us go to damped oscillation.

(Refer Slide Time: 24:28)



So, in the case of damped oscillation suppose the mass is connected to a dashpot, this is dashpot. So, suppose the mass is connected to a dashpot, then we have to consider the viscous force due to the viscosity of the liquid in the dashpot. Now, the damping forces

due to the viscous liquid x in the direction opposite to the instantaneous motion and is proportional to the viscosity. So, the damping force is proportional to the viscosity, when it is small. So, then we have the third force which is F we have taking as F_3 , F_3 equal to minus $c y \dot{}$, $y \dot{}$ is the velocity at instant t .

So, the force due to the viscosity of the liquid, which is there in the dashpot, there will be a force which is called damping force it x in the direction opposite to the instantaneous motion. And this c is a dumping constant; this c is a damping constant.

(Refer Slide Time: 26:42)

The equation of motion is

$$m\ddot{y} = F_1 + F_2 + F_3 = -\lambda y - c\dot{y}$$

or

$$m\ddot{y} + c\dot{y} + \lambda y = 0 \quad (\text{damped oscillation}).$$

The auxiliary equation is

$$\alpha^2 + (c/m)\alpha + (\lambda/m) = 0.$$

The roots are $\alpha_1 = -\frac{c}{2m} + \frac{1}{2m}\sqrt{c^2 - 4m\lambda}$ and $\alpha_2 = -\frac{c}{2m} - \frac{1}{2m}\sqrt{c^2 - 4m\lambda}$.

IT ROORKEE | NPTEL ONLINE CERTIFICATION COURSE | 10

Now, let us see a ; so in the now the resultant force will be F_1 plus F_2 plus F_3 which is equal to f_1 plus f_2 we have seen is minus λy , so minus λy plus minus $c y \dot{}$. Thus we get the equation motion as $m y \ddot{}$ plus $c y \dot{}$ plus λy equal to 0. The auxiliary equation will be given by α^2 plus c by m into α plus λ by m equal to 0. The two roots of this auxiliary equation are given by α_1 and α_2 . The α_1 is minus c by $2m$ plus 1 by $2m$ under root c square minus $4m\lambda$; and α_2 is minus c by $2m$ minus 1 by $2m$ under root c square minus $4m\lambda$.

(Refer Slide Time: 27:33)

Case 1. When $c^2 > 4m\lambda$ (over damping)
then $y(t) = c_1 e^{-(p-q)t} + c_2 e^{-(p+q)t}$, ... (1)
where $p = \frac{c}{2m}$ and $q = \frac{1}{2m} \sqrt{c^2 - 4m\lambda}$.

Case 2. When $c^2 < 4m\lambda$ (under damping)
then $y(t) = e^{-pt}(A \cos \bar{\omega} t + B \sin \bar{\omega} t)$
 $= c e^{-pt} \cos(\bar{\omega} t - \delta)$, ... (2)
where $q = i \bar{\omega}$, $\bar{\omega} = \frac{1}{2m} \sqrt{4m\lambda - c^2}$.

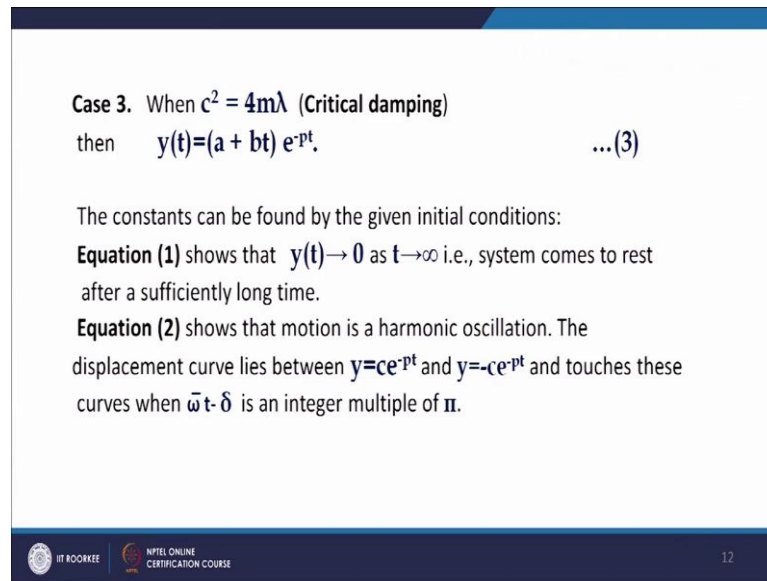
Now, due to the discriminant $c^2 - 4m\lambda$, there arise three cases. The case 1 is $c^2 > 4m\lambda$; in that case, the two roots α_1 and α_2 of the auxiliary equation will be distinct. So, now we have three cases, in the case 1 when $c^2 > 4m\lambda$, we have the case of over damping because the damping constant c is high here.

So, in this case, the general solution is given by $y(t) = c_1 e^{-(p-q)t} + c_2 e^{-(p+q)t}$. Now, where p is equal to $\frac{c}{2m}$ and q equal to $\frac{1}{2m} \sqrt{c^2 - 4m\lambda}$. Now from here we can see that p is strictly greater than q . So, when t goes to infinity, $e^{-(p-q)t}$ goes to 0 as well as $e^{-(p+q)t}$ goes to 0. So, in the equation 1, when $y(t)$ goes to 0, t goes to infinity that is the system comes to rest after a sufficiently long time.

(Refer Slide Time: 28:12)

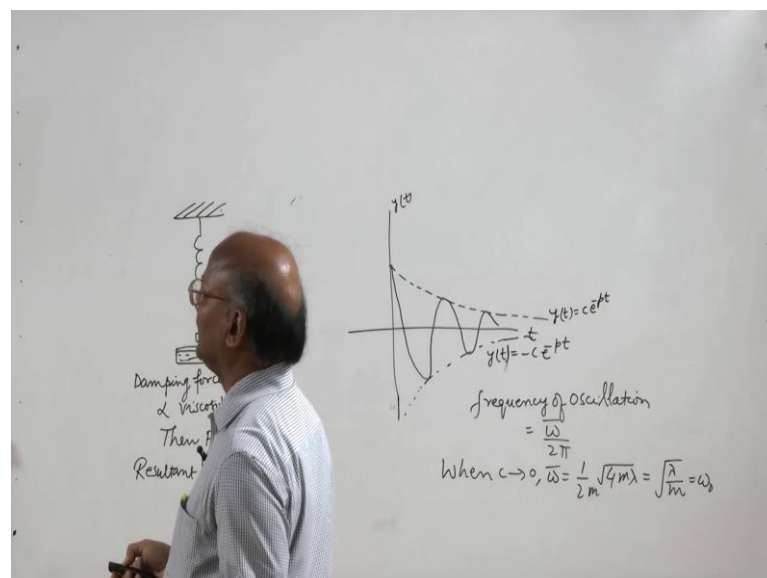
Case 3. When $c^2 = 4m\lambda$ (Critical damping)
then $y(t) = (a + bt) e^{-pt}$ (3)

The constants can be found by the given initial conditions:
Equation (1) shows that $y(t) \rightarrow 0$ as $t \rightarrow \infty$ i.e., system comes to rest after a sufficiently long time.
Equation (2) shows that motion is a harmonic oscillation. The displacement curve lies between $y = ce^{-pt}$ and $y = -ce^{-pt}$ and touches these curves when $\bar{\omega} t - \delta$ is an integer multiple of π .



In equation 2, we see that $y(t)$ is given by $ce^{-pt} \cos \bar{\omega} t$, and therefore the motion is a harmonic oscillation. The displacement curve $y(t)$ lies between y equal to c times e^{-pt} and y equal to $-c$ times e^{-pt} . So, this is let us say t axis, this is $y(t)$.

(Refer Slide Time: 28:38)

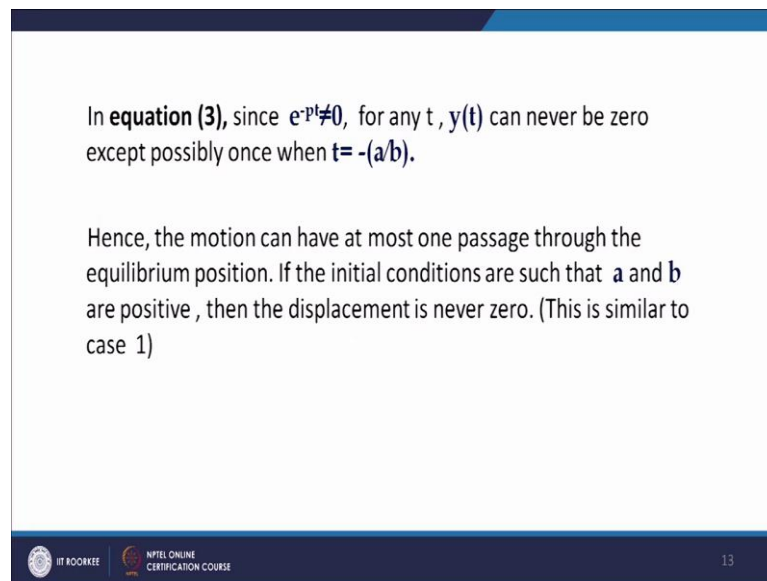


These are the curves $y(t) = ce^{-pt}$ and $y(t) = -ce^{-pt}$, and the displacement curve lies between these two curves, and touches these curves when $\bar{\omega} t - \delta$ is an integer multiple of π .

$\bar{\omega} t - \delta$ is an interior multiple of π . Moreover, the frequency here is frequency of oscillation is equal to $\bar{\omega} / 2\pi$.

So, when $\bar{\omega}$ is more frequency is more and when let us see when $\bar{\omega}$ will be more, $\bar{\omega}$ will be more when c is smaller. When c is smaller here and it will be greatest when c tends to 0. So, when c tends to 0, $\bar{\omega}$ goes to $1 / 2m\sqrt{4m\lambda}$, which is equal to $\sqrt{\lambda} / m$ which is equal to ω_0 . The ω_0 which be found in the case of the simple harmonic oscillation this ω_0 $y(t)$ equal to $c \cos \omega_0 t$, here ω_0 you can see ω_0 is $\sqrt{\lambda} / m$. Now, let us discuss the last case, in the equation 3, you can see, we have $y(t)$ equal to $a + b e^{-pt}$. Since, e^{-pt} is not equal to 0 $y(t)$ can never be 0 except possibly once when $a + b e^{-pt}$ is equal to 0 that is t is equal to $-\ln(a/b) / p$.

(Refer Slide Time: 31:22)



In **equation (3)**, since $e^{-pt} \neq 0$, for any t , $y(t)$ can never be zero except possibly once when $t = -\ln(a/b) / p$.

Hence, the motion can have at most one passage through the equilibrium position. If the initial conditions are such that a and b are positive, then the displacement is never zero. (This is similar to case 1)

IT ROORKEE NPTEL ONLINE CERTIFICATION COURSE 13

Hence the motion can have at most one passage through the equilibrium position now if the initial conditions are such that a and b are positive then you can see $y(t)$ can never be 0, because if a and b are positive then $y(t)$ can never be 0. So, the displacement will never be 0. And this case is similar to the case 1; in the case 1, you can see $y(t)$ tends to 0 as t goes to infinity. So, the system comes to rest after sufficiently long time.

Thanks.