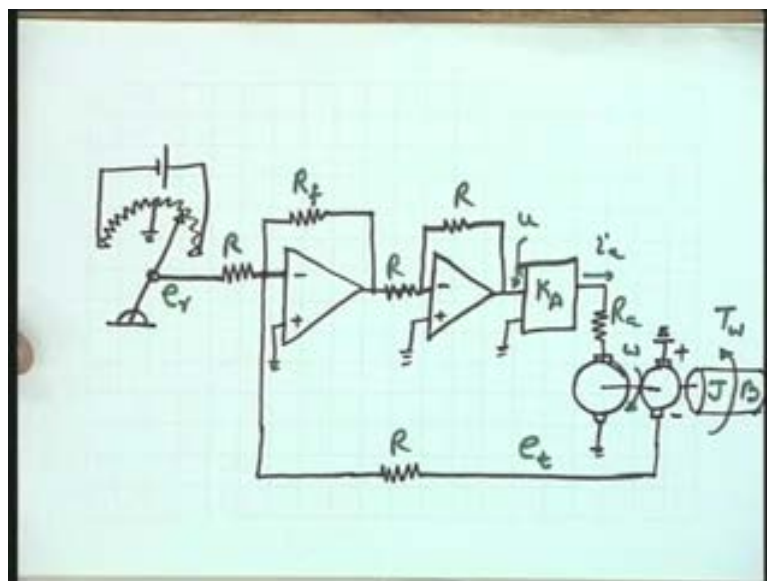


Control Engineering
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Lecture - 12

Models of Industrial Control Devices and Systems (Contd....)

We can now start with the hardware of the control systems. I introduce some case studies wherein the hardware used in the industrial sector will be introduced to you. Well, naturally we cannot go to the reasonable depth of discussion in terms of the hardware details but the purpose will be their basic characteristics and how those characteristics affect a particular control system that will be the basic objective of our discussion. I take up the first control system as a speed controlled system.

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This is the speed control system and now you see how the system works and what are the basic ingredients in this particular system. In this particular case you want to control the speed of the load. so here is a load the parameters are J and B . J and B are the parameters, the movement of inertia and the viscous friction B . T_w is the load torque on the system; that is it is a disturbance torque acting on this particular system. now ω being the speed so it means there is a specified ω and you want the control system to act in such a way that this particular load runs at the specified value ω in spite of disturbances T_w acting on the system. This is the objective.

So in this case now, first of all, I needed an actuator, the power device in this particular case the actuator as you see is a motor with a power amplifier. So this power amplifier and the motor they constitute an actuator for this particular load because they provide the necessary torque for the load to run at the speed ω . Now this particular actuator (Refer Slide Time: 2:52) is getting the control signal u from a suitable controller, here is the controller circuit which in this particular case is a simple amplifier. At a later stage we will see that this controller could consist of complex functions like derivative of an error signal, integral of an

error signal or in general any function you could take at this particular point. But here in this configuration in the basic structure I assume that the controller is a simple amplifier because the objective at this particular point is not to go to the design of the controller but the objective is to give you the basic hardware required for a control system. So you find that this amplifier now this is getting two signals: One is the signal reference signal, e_r is the reference signal please see and this e_r you are able to vary by varying the position of this wiper arm. So it means you will change the position of the wiper arm (Refer Slide Time: 3:56) with respect to your requirement on the commanded speed.

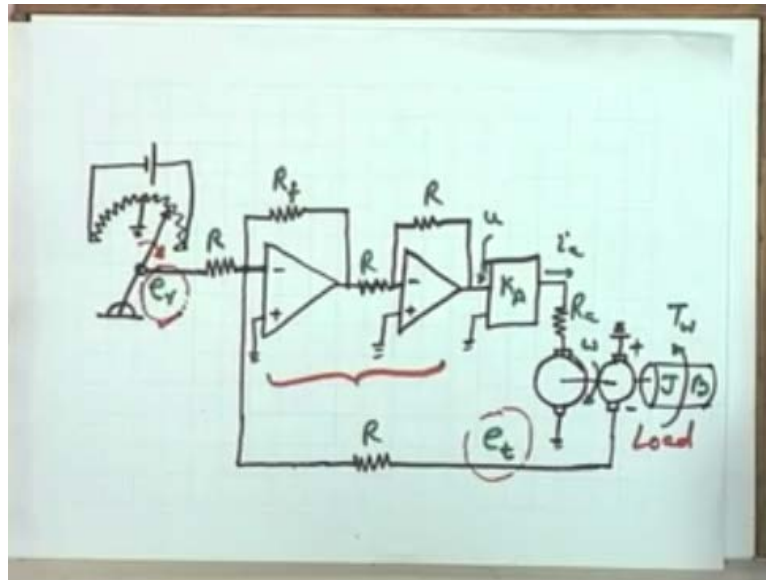
Ω_r in this particular case, not explicitly shown. Ω_r is the commanded speed and you want ω to follow the commanded speed Ω_r . So what I am doing is proportional to Ω_r I set a signal e_r a larger Ω_r means a larger e_r and I can set it accordingly by setting the wiper on the potentiometer. So this e_r becomes the reference signal and where is the feedback signal; the feedback signal in this particular case is coming from a tachogenerator.

Tachogenerator as you know will generate a signal e_t which is proportional to its speed the speed being ω so it means e_t is proportional to ω . So it means e_t has got the necessary information about the controlled variable, e_r has got the necessary information about the commanded value and the difference of the two is going to give you an actuating error signal. So this actuating error signal is being amplified and this amplified signal which now becomes the control signal u is given to the actuator and the actuator is running the load. And the purpose of the controller design will be make suitable changes at this point because all others are fixed components of the system; you will not like to change the motor the tachogenerator you will rather manipulate this particular function so that ω is following the commanded value Ω_r . But at this point we will not go the details of the design rather I will like to express this diagram using a block diagram and from there the visualization of the feedback structure and basic elements required in a system will become very clear.

I may refer to this diagram again I need your help now to set up a block diagram. What I do is I take this as the reference signal e_r this is the reference signal which is proportional to Ω_r which is a function of the commanded signal Ω_r . I think if you do not mind though the physical diagram does not have this component let me have this component and call this component as reference input elements. Please see, physically potentiometer is taking care of this and the reference input elements may have the signal Ω_r . So Ω_r is the commanded position and e_r is the corresponding voltage.

Now let me take an error detector in the standard block diagram. This is plus and here it is minus (Refer Slide Time: 6:45). You will note, I think I can go back to the diagram, u will note that e_t is a minus signal the e_t plus minus over here, the e_t the suitable algebra sign has been given to e_t so as far as the block diagram is concerned as far as this amplifier is concerned you can see e_r minus e_t is being multiplied by the gain of this amplifier.

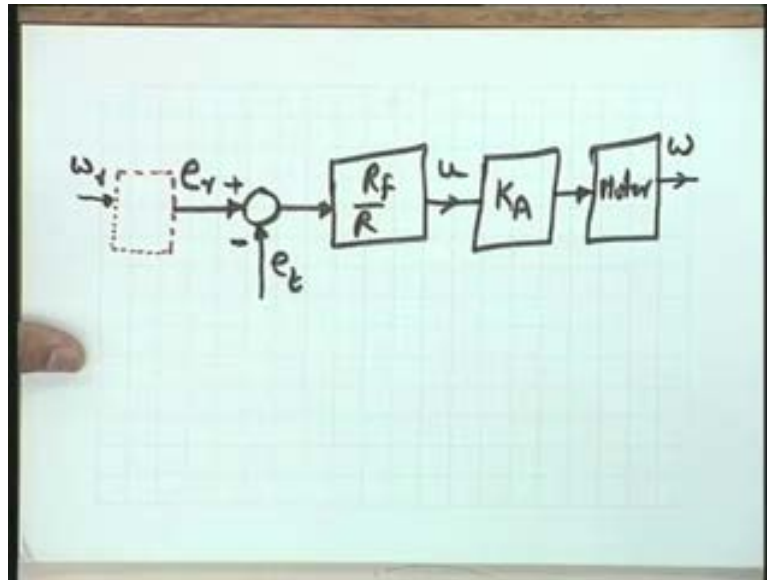
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What is the gain? It is R_f by R is it okay please; R_f by R but with the negative sign. to take care of this negative sign I have used an inverting amplifier with a gain of unity so that the gain of this total amplification unit is R_f by R with a positive sign and inverting amplifier has been used over here just to take care of the sign of the gain of the amplifier and therefore I can say that **the output e** this error detector output e_r minus e_t , so look at the unit now. What is this circle? In terms of physical system, this dotted block in terms of the physical system is your potentiometer. The circle the so-called error detector or the summing junction in terms of the physical system is an Op-Amp because the Op-Amp is getting the two signals e_r and e_t is generating a signal which is proportional to the difference of the two and the proportionality constant is R_f by R .

So here if am putting in this particular block an amplifier gain R_f by R and let me say that u is this control variable. So you will please note, **the physical relevance** the physical equipment also is which is being used to realize the various objectives. Now u is the control signal, this is a signal which has the knowledge about the action to be taken but it does not have the necessary power to drive the actuator. So you will like to add power to the signal keeping the basic intelligence at the same level. So what you do is you simply add here a power amplifier K_A . K_A is the gain of the amplifier. Now this power amplifier is able to drive the motor. So what is the transfer function of the motor I will see but **at the time** for the time being let me say that this block is a motor block; suitable transfer function of the motor I am going to use over here. The output here is the controlled variable ω and this ω is being sensed by the tachogenerator which is a sensor.

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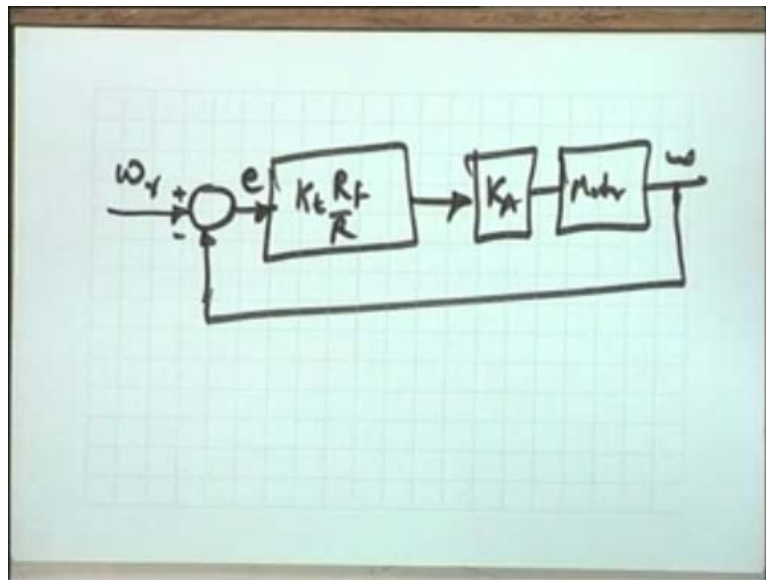
Let me say K_t is the constant of the sensor so that e_t is equal to K_t into ω and this is being fed back here. This is the basic block diagram but for the motor block to be **opened to be th** detailed properly. So I am going to give you the transfer function of the motor block but before we go to the motor block let me look at this block diagram with respect to the general feedback structure which was given to you. Tell me please, this diagram this is a dotted block (Refer Slide Time: 9:59) this does not contain any physical component. If you look at the diagram from e_r to ω and a feedback loop it is a non-unity feedback structure non-unity feedback structure. It is naturally in this particular case there is a sensor transfer function and this you are going to have as e_{cap} , it is an actuating error signal and this is a non unity feedback structure.

Now suppose you see, as I told you, that for the purpose of analysis and design the unity-feedback structure may be more convenient. In that particular case I will like to transform this to a unity-feedback structure without sacrificing the information contained in it. So naturally I like that the basic equation remain the same. If the basic equations remain the same in that particular case you please note that this K_t this particular block should contain K_t and it has to contain K_t . You please see that this K_t is not a physical component there but I will set the potentiometric constant in such a way that ω_r into K_t becomes the desired voltage e_r which you want to feed as the reference voltage.

You will see that it is obvious that the comparison of the two voltages will give you an error signal which is proportional to the errors in speed if and only if the dotted block has got K_t . I hope this point is being taken very well. This e is a voltage signal it is not a speed signal the unit is not radian per second the unit is volts but this volts will be proportional to the error in speed if and only if this block has got K_t and this is in your hand you see. You can set the potentiometer constant. You know that the potentiometer constant is..... what is the potentiometer constant the potentiometer? (Refer Slide Time: 11:58) this voltage equal to a constant K_p into setting of the wiper arm so that is a potentiometer constant. This, you will design in such a way that the e_r which you get, the K_p or the potentiometer constant which you get equals K_t which is the tachometric constant so that in that particular case the error in volts is directly proportional to the error in radian per second the error in speed. So, therefore

the reference input elements, this is what I want to convey, may or may not be physically present in the system in the form, there is no unit like a tachogenerator. K_t is the constant of the tachogenerator there is no unit like a tachogenerator on the input side but still I have a block with transfer function as $\frac{K_t R_f}{R}$ because it makes sense, it explains the working of the system and this realization of the block is suitably done by the potentiometric unit in the system. So this becomes the reference input elements for the system. And now you see that, what I am doing is the conversion of this diagram into a unity-feedback system: ω_r I take, you will now tell me whether it is okay plus minus..... I club K_t here, here it is K_A motor I am yet to explain ω and it goes you can just see.

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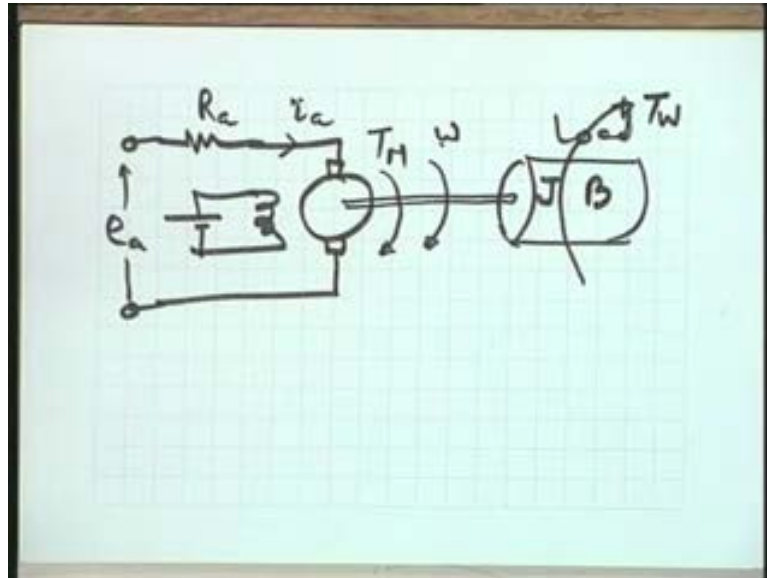
The two equations will remain the same but now it should not be confused; ω_r is the reference input does not mean that physically radian per second is directly going to some unit, it does not mean that. It simply means that it is an equivalent block diagram which has got the same mathematical equation which you get from the original block diagram between ω and ω_r .

Now in this particular case this is e and not e_c because e is the system error, e_c is the error between the commanded speed and the actual speed and the cap can now be removed and a unity-feedback system you can get e variable over here. You can see that block diagram manipulation is required to help analysis and design. I claim that this type of block diagram when we go for analysis and design is going to be more convenient. So a non-unity feedback system has been appropriately converted to a unity-feedback system **for the purpose of** for the usage of this particular block diagram. I hope this is okay.

Now let me come to the motor because the block diagram is not yet complete. If I come to the motor this motor used in this particular control system has got this configuration. In this case the armature voltage is e_a and R_a is the armature resistance, i_a is the current through the armature and here is the motor shaft and I have a load here. The motor produces a torque T_M and the load disturbance torque is T_W , the parameters of the load are J and B and let us say the speed at which the motor runs is ω . I want you to give me the mathematical model of the system so that the motor block in the diagrams we have used could be replaced by that

block diagram model or the transfer function model of the motor. These things are known to you.

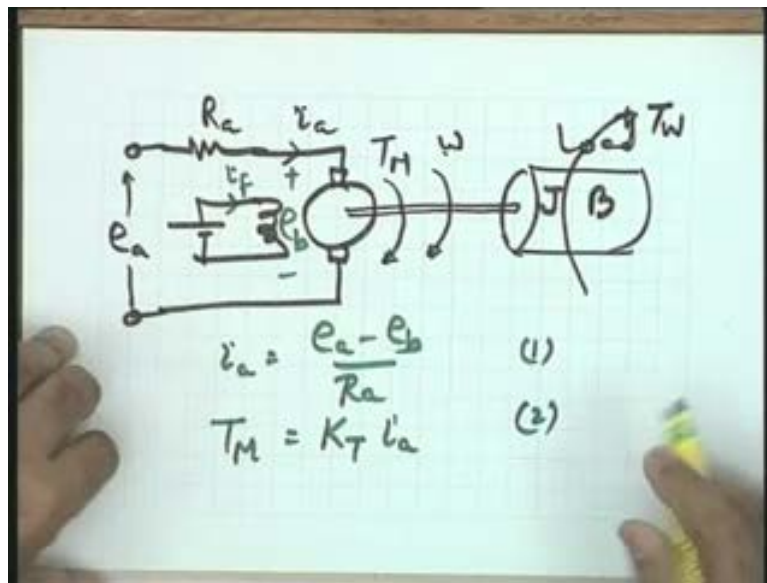
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Well, I will simply revise the basic relationships for you. Let me look at the armature circuit; e_a is the applied armature voltage and e_b is going to be the back EMF. You will see that I have neglected the armature inductance in a simplified model. This is usually okay I mean as far as **the characteristics of the motor is** this armature controlled motor is concerned that the inductance can be neglected. So in that particular case the basic equation becomes i_a equal to e_a minus e_b divided R_a this is equation 1 I get for the motor model.

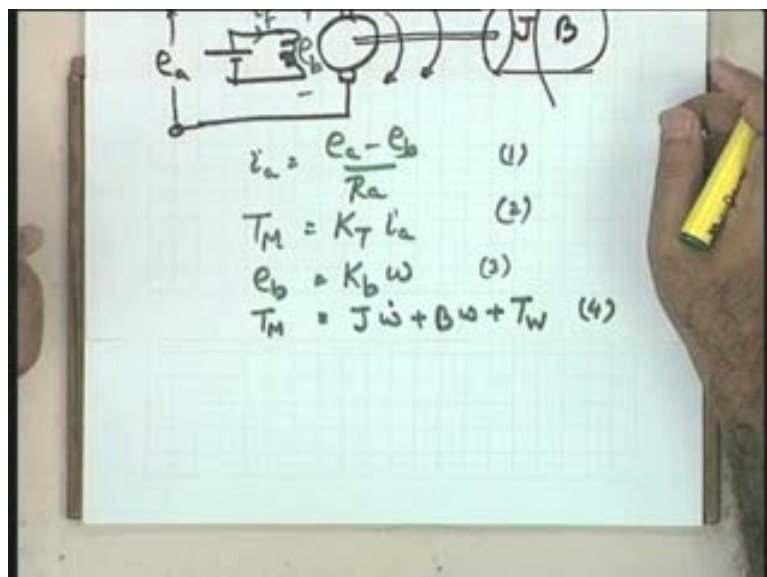
Now this i_a you will note that if the field current is constant and i_a depends upon the applied voltage. However, the torque produced by the motor as you know is proportional to the product of the two fluxes and hence the two currents. Since i_f is constant I can say that the torque produced by the motor is proportional to the armature current i_a i_f being constant. So I now write T_m is equal to K_T into i_a where K_T is the torque constant of the motor, the point may be please be noted; K_T is torque constant of the motor and this has come this way because i_f the field current is constant and it is an armature controlled motor that is why the name is given because you are controlling the armature current through i_a and the field current is held constant. The speed of the motor is being controlled by controlling the current through the armature.

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What other equations will come? What is the value of e_b please? This is equation number 2. e_b , I can take as proportional to speed of course and therefore I take a proportionality constant K_b which is the back EMF constant proportionality constant K_b into ω , ω is the speed in radians per second and this is your equation number 3. And now this torque T_M what is this torque doing; this torque is driving the load. So, write an equation the force balance the torque balance equation over here, in that case I can write T_M is equal to $J \dot{\omega}$ plus $B \omega$ plus T_w this your number 4. T_w is the disturbance torque, $J \dot{\omega}$ is the inertial torque opposing the motion, $B \omega$ is the frictional torque opposing motion so the torque applied by the motor is equal to all the terms opposing the motion so this becomes the basic equation.

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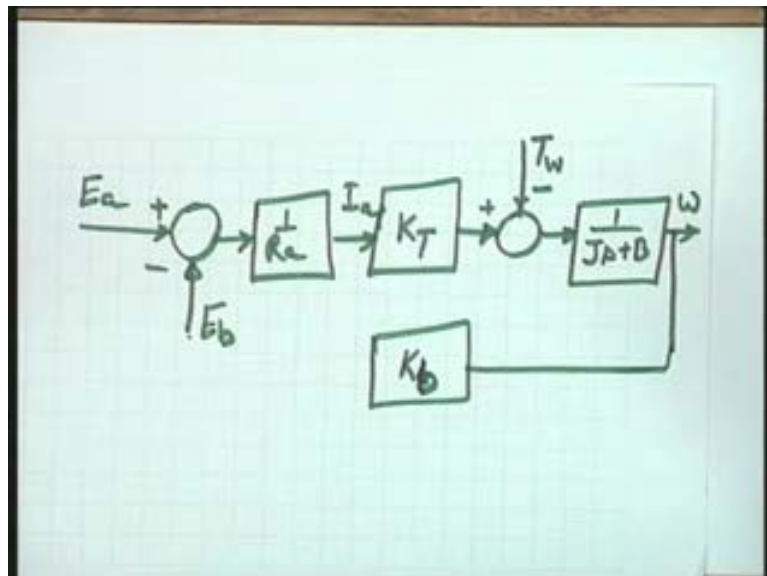
Come on please, help me now. I want you to give me these four equations in terms of a block diagram. Because as I said that manipulation in terms of block diagram will become more

convenient for us in terms of the block diagram and we can avoid writing these equations in Laplace domain. You can directly give me you see. I think, now you should be equipped with the required tools that the manipulation can be done directly. So what do I do? E_a is your input signal, this is the input signal so let me get started with the input signal.

This is E_a ; in Laplace domain I can write this as $E_a(s)$ I am not writing to save space. Now this input signal you find the back EMF signal coming over here plus minus so this is E_b . the difference of the two divided by R_a is your armature current I_a please follow up the equations. And once we have done it you see that for later applications you will be able to write the block diagram directly without explicitly writing the equations. I a you have got, multiply it by the torque constant you have got K_T , **can you help me here what should I do to proceed further?** Now I have got the torque produced by the motor; it is I_a into K_T . Now what is the net torque which is available for driving the inertia, it is not T_M it is T_M minus T_W . T_W may oppose or help I do not know but **you see** the net torque is the algebraic sum of T_M and T_W .

So in that particular case I can use a summing point here. I can say that plus and here it is T_M minus so the net torque available for driving the inertia is T_M minus T_W **I think its I hope it is okay** and now this torque since this is driving the inertia this particular block could be $1 / (J_s + B)$ with output as ω , you can see the basic relationships. This ω (Refer Slide Time: 21:09) now can be picked up by the sensor and here is your sensor transfer function K_b and this is **sorry I am sorry** there is no sensor here this is the transfer function of a motor only in the open-loop mode there is no sensor as far as the motor is concerned, this is actually K_b the back EMF. this feedback link **please see** is not being provided by any sensor it is inherent in the structure of the motor in the motor operation; K_b is the back EMF so K_b into ω as your back EMF and here gets the loop completed.

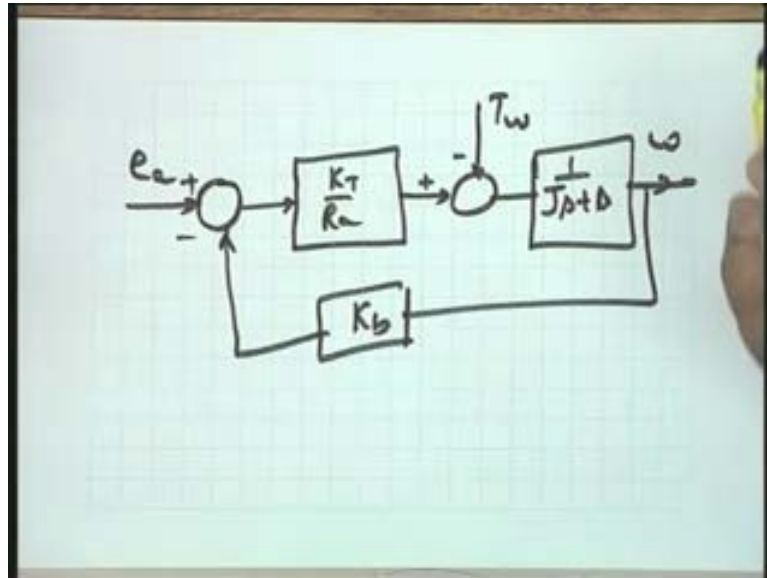
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So you please note there is a feedback loop; though if you look at the motor you have not installed any sensor to give the feedback signal. This is an inherent feedback loop in the motor and this rather is an important characteristic of the armature control motor. you will see that when I take the field control mode the inherent feedback loop is not there and many a

times this particular inherent characteristics of the armature control motor is a useful characteristic because it provides the required damping, this I will show later. So this becomes the block diagram structure. So let me now put it in a more convenient form which I will be using quite often. e_a does not matter I can put the variables in this form as well plus minus K_T by R_a plus minus T_w 1 over J_s plus B omega K_b please see that.

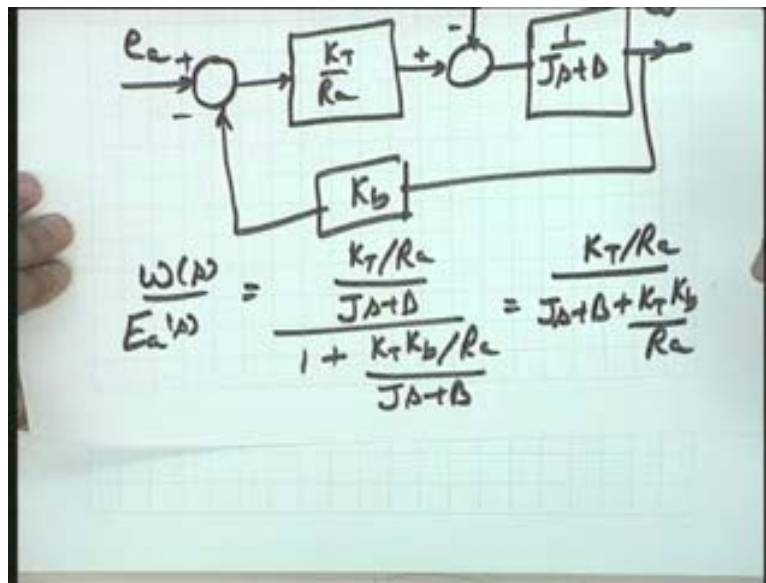
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This is the block diagram model for the motor and I can place this model in the overall system model to get the block diagram model for the speed control system under consideration. Before I go to that let me study the characteristics of this motor. Please see, if I set the disturbance zero I get the transfer function between omega and e_a , I need your help here; $\omega(s)$ over $E_a(s)$ is equal to K_T by R_a divided by J_s plus B over 1 plus $K_T K_b$ by R_a divided by J_s plus B . This becomes the transfer function model between omega and e_a .

Please simplify this. If I simplify this model I get this as equal to K_T divided by K_T by R_a let me take it here J_s plus B plus $K_T K_b$ by R_a this is your transfer function model for the motor.

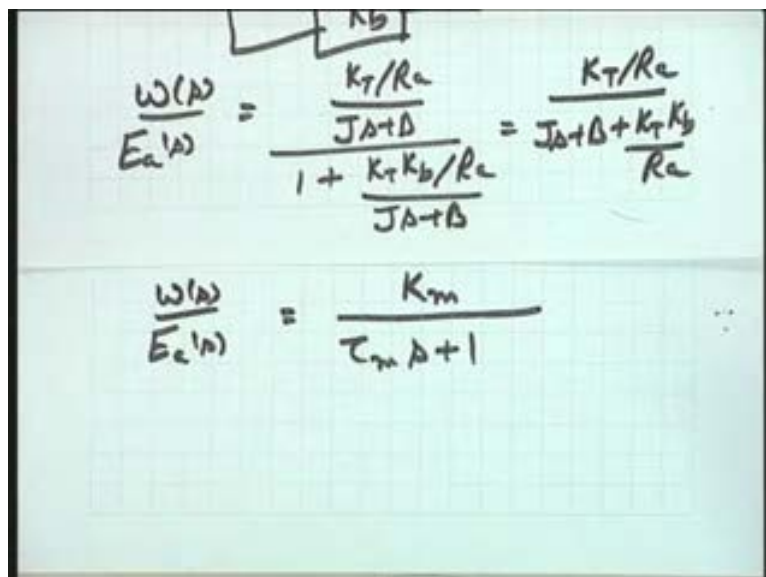
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Now, in terms of the time constant could you give me the transfer function model for the motor in terms of time constant?

I will like to write this as $\omega(s)$ over $E_a(s)$ is equal to K_m the motor gain let me call it divided by $\tau_m s + 1$ it is a first-order transfer function model and τ_m can be suitably written. please see, τ_m will become J divided by this total unit (Refer Slide Time: 24:51) J divided by this total unit is your τ_m and this function divided by this total unit this total term becomes your system gain or the motor gain constant K_m . so the motor transfer function between the output ω and the input e_a is a first-order transfer function K_m over $\tau_m s + 1$. **Please store it in your memory I will be requiring this information quite often.**

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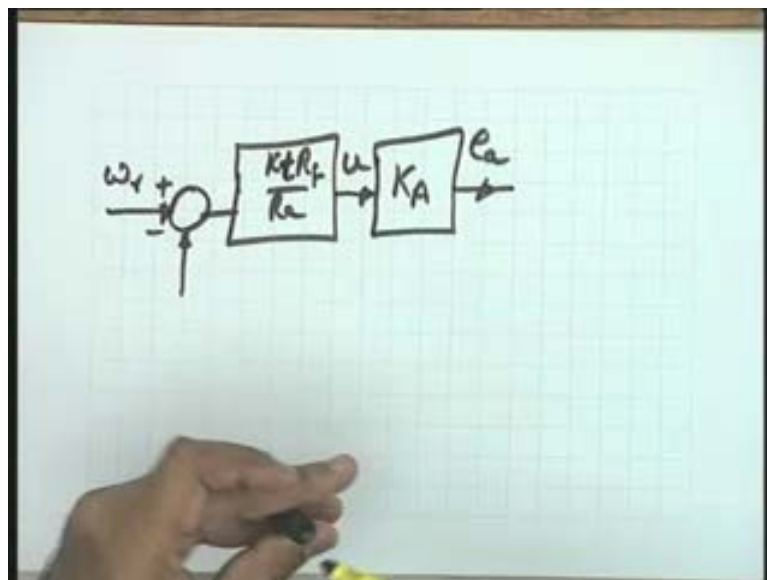


Now I want to comment on the effect of K_b the inherent damping characteristics of the motor. You will please note that K_b increases the value B and hence it increases the effective

damping of the motor. So it means it is because of K_b the oscillations of the system can be quickly damped out. The feature which will not be available as you will see when we come to the field control mode. Equivalently in the time constant form, what happens to the time constant because of K_b , does it increase or decrease the time constant because of K_b ? It decreases and as you know smaller the time constant faster is the response. So quick damping means the oscillations are damped out quickly and the motor comes to the steady value quickly and this is the characteristics of this inherent loop which is there in an armature controlled motor. I hope this is okay.

So now let us go back to the original system wherein I have to place the motor and give you the overall transfer function of this overall block diagram of the system. I can take up the unity feedback system if you understand this very well now. ω_r is the input I am taking. I know that physically it is E_r which is being given as the input but in the block diagram for the purpose of analysis and design I could take ω_r as the input plus minus here $K_t R_f$ by R_a is the gain here K_t small t the tachometric constant, this is your e_a this is your e_a the voltage please see, this is the voltage available from the amplifier this voltage is going to the power this is not e_a this is going to power amplifier (Refer Slide Time: 27:20) this is u signal I am sorry here at this point this is u signal the power amplifier output is e_a the voltage which is going to the armature controlled motor. Fine.

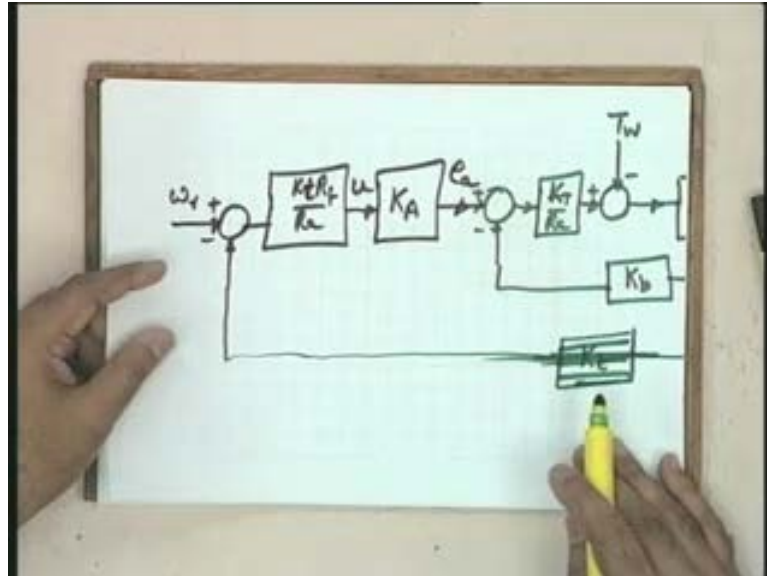
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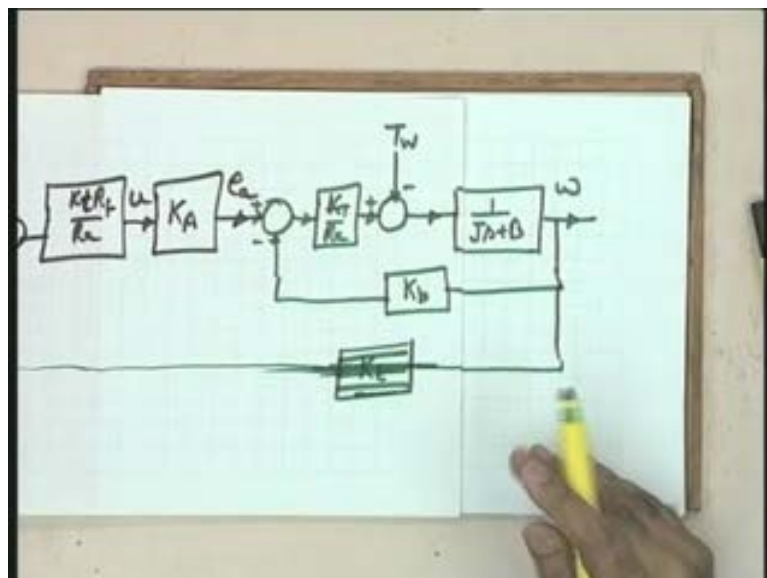
Now I will like to put the summing junction here because at this particular e_a you have got e_b the back EMF of the motor. Now this has a block K_t over R_a here so that at this particular point here we have got the torque, this is torque generated by the motor. Put another summing junction with a negative sign here this is your T_w the disturbance torque on the motor. Yes.....[28:12], this torque is available for driving the inertia so this is 1 over J_s plus B this is ω the speed. From ω I take a loop to generate the back EMF signal, from ω I take another loop it is a multi-loop configuration now through the sensor K_t so that it is going to this particular summing junction and is giving you the primary feedback loop. I will put the diagram this way (Refer Slide Time: 28:52) K_t is going to the summing junction base this summing junction and is giving you the primary feedback loop and this becomes

[Conversation between Student and Professor – Not audible ((00:28:59 min))] oh I am sorry I am sorry K t has been taken over there it is a unity feedback system you are very right. It is a unity feedback block diagram now; K t has been taken in the forward loop.

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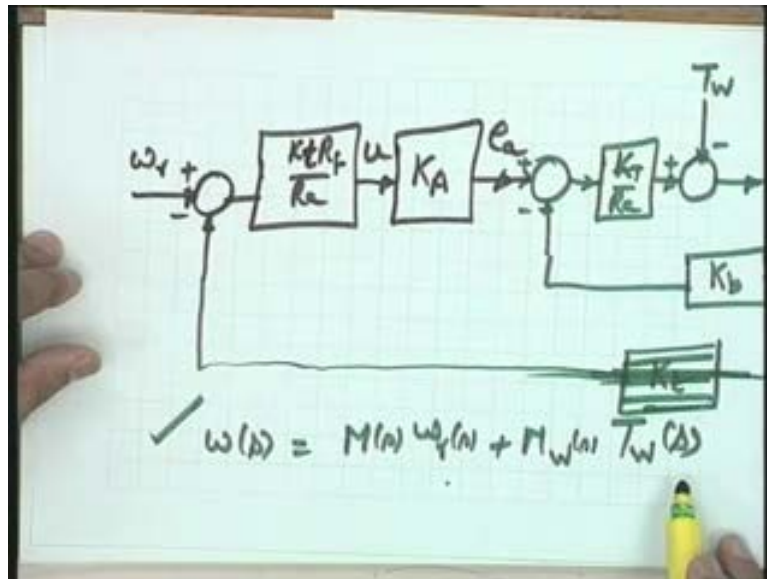
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So, as far as this diagram is concerned it will go through unity so this becomes the total speed controlled system block diagram; the design and analysis aspects will come later. Now you know the block diagram reduction tools. Since the block diagram reduction tools are known to you, using those tools either through signal flow graphs or directly you can get me the transfer function between ω and ω_r ; you can get the transfer function between ω and T_w it depends upon the interest you have. if you want to study, if you want to analyze the system with respect to the commanded position get the transfer function between or in general $\omega(s)$ is equal to $M(s) \omega_r(s) + M W(s) T W(s)$ could be obtained by block diagram reduction. That is why the reduction method Mason's Gain formula and direct

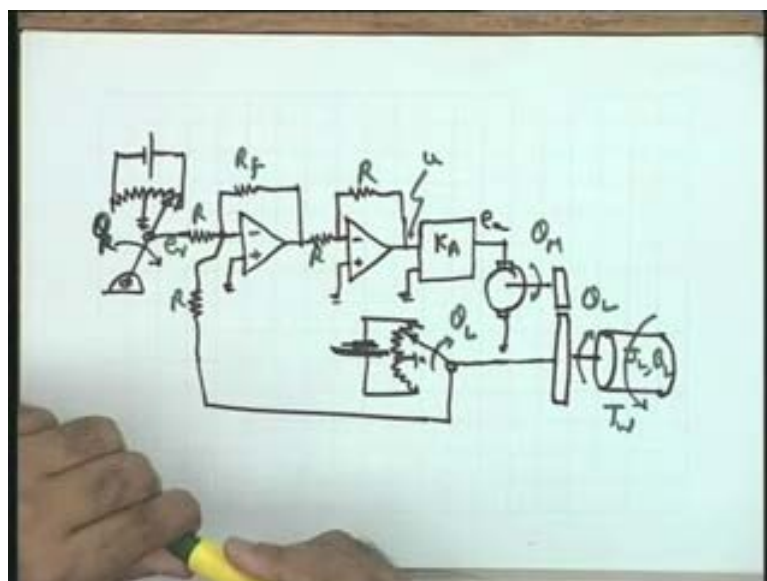
methods were given to you because these will be subsequently required. And I leave this as an exercise to you, for this particular block diagram get me an equation of this form which gives you an output variable in terms of the commanded speed and the disturbance signal. It is a two input one output system.

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One more diagram I wanted to take and that is a position control system. Once you have got the speed control system very clearly there will not be any problem here it will follow very easily. See the objective of **this function** this system how does it work.

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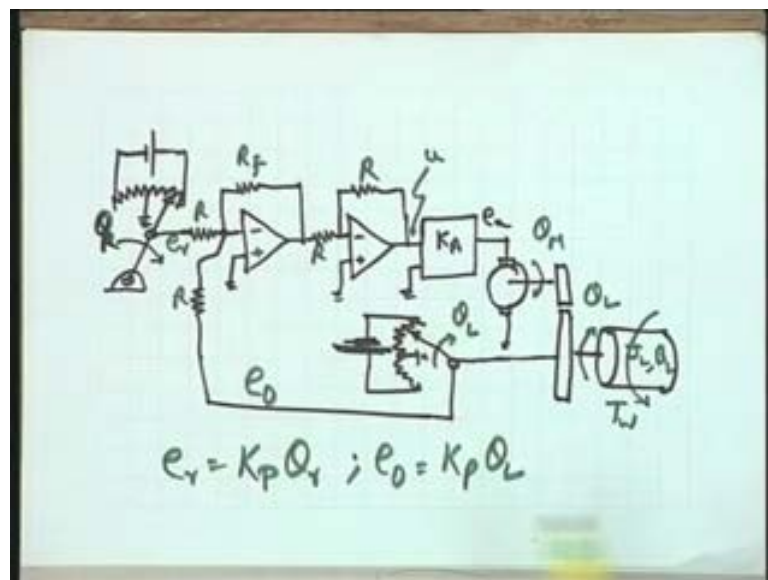
In this particular case again I have a reference input e_r in terms of voltage and this reference input is the commanded position. I want to command the position of the load it is a position control system now in contrast to the earlier system which was a speed control system. So now e_r again summing junction is the two voltages but the meaning of the two voltages

has become different. This e_r is a signal which represents the desired position and this desired position can be set on the potentiometer so that e_r is equal to $K_p \theta_r$ where K_p is the potentiometric constant. I hope you get it.

Now let me see the feedback signal at this stage. Feedback signal is this particular load position θ_L . this load position θ_L could also be converted into a voltage signal by a potentiometer. Now I will like to take the potentiometric constant identical. Please note that this assumption will make a possible to convert the block diagram of the overall system into a unity-feedback block diagram. If that is not there **you may not** you will not be able to get a unity-feedback block diagram and there is no problem in taking two identical potentiometers with the same potentiometric constant.

So e_r here is $K_p \theta_r$ let me call this e_0 so e_0 here is $K_p \theta_L$ where θ_L is the load position it is the controlled position which is required to follow the commanded position θ_r . **So see the variables please:** θ_r and θ_L are the two variables and you will note the sign the sign I have inverted, minus plus and minus plus taken this way you will please note the wiper on the potentiometers have been taken in this way that at this particular point the input of the Op-Amp is e_r minus e_t input to the Op-Amp e_r minus e_0 where e_0 is the output voltage of the potentiometer this you can see with respect to this potentiometer wiper arm movement.

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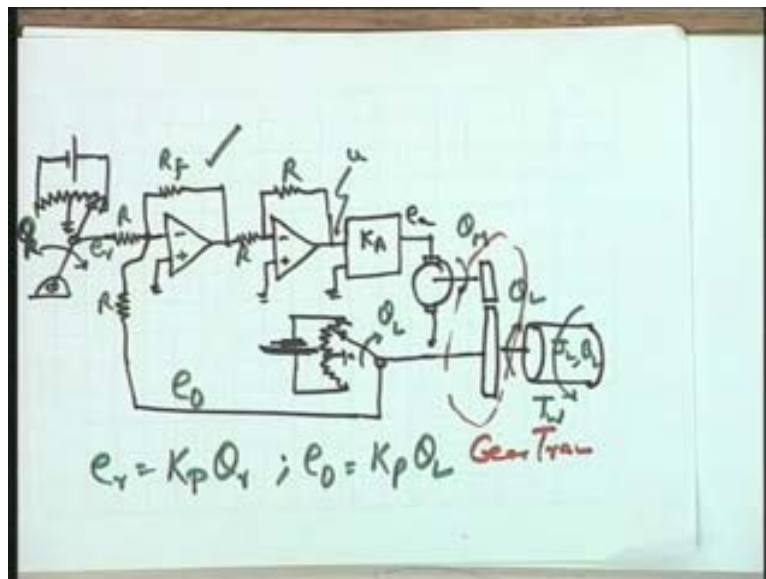


Now this voltage e_r minus e_0 is being amplified. So it means the error detector, this particular amplifier is an error detector which gives you a voltage proportional to the difference of the two. Here again I have an inverting amplifier so that the minus sign is taken care of. I have a power amplifier over here so that the motor this particular signal is to drive the motor. This motor produces the torque and this particular torque is required to drive the load. now a point to be noted that I have not coupled the load directly to the motor shaft. **I need your attention here.** I have not coupled the load directly to the motor shaft, there is a beam that many a times for **these particular for** the movement of the load I need the speed reduction and the torque magnification.

You will find that this particular gear set let me call it, this is set of two gears there may be a set of more gears, so let me call this as a gear train a set of gears suitably connected suitably installed is a gear train this gear train is providing the required amount of torque magnification if the torque **required at this particular torque** generated at this particular point (Refer Slide Time: 34:25) is lower than the torque required at this particular point the load is heavier in that particular case you will like to magnify the torque and this particular gear train is giving you the torque magnification. We are going to derive the model for the gear train. there from you will find that a suitable torque magnification is possible and in the process speed reduction will take place that is θ_M dot and θ_L dot and torque ratio will be in such a way that the gear train reduces the speed and magnifies the torque.

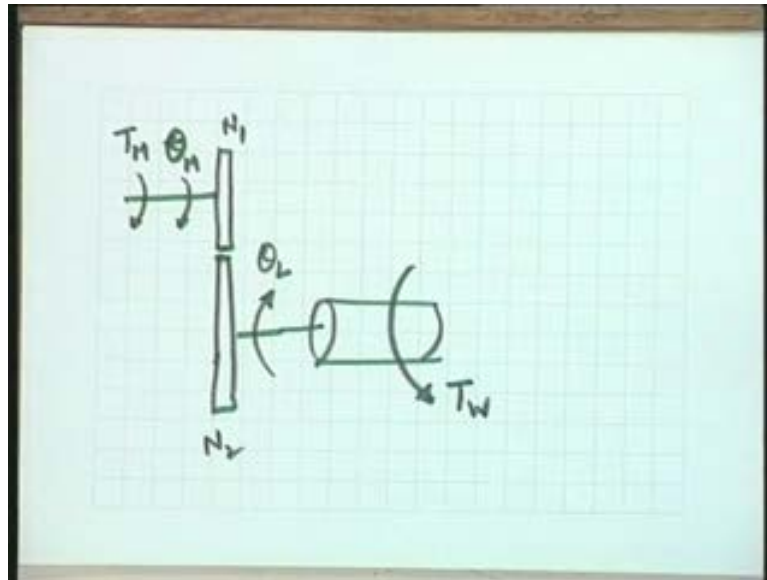
The objective becomes very clear; it as per the load requirements you will install a suitable gear train. So, after that, this is the load position, this is the motor position, the load position is being sensed and is being fed back. I hope the basic configuration of a position control system becomes clear. And the transfer function of each and every model **will become** is clear except for the gear train which I have to explain.

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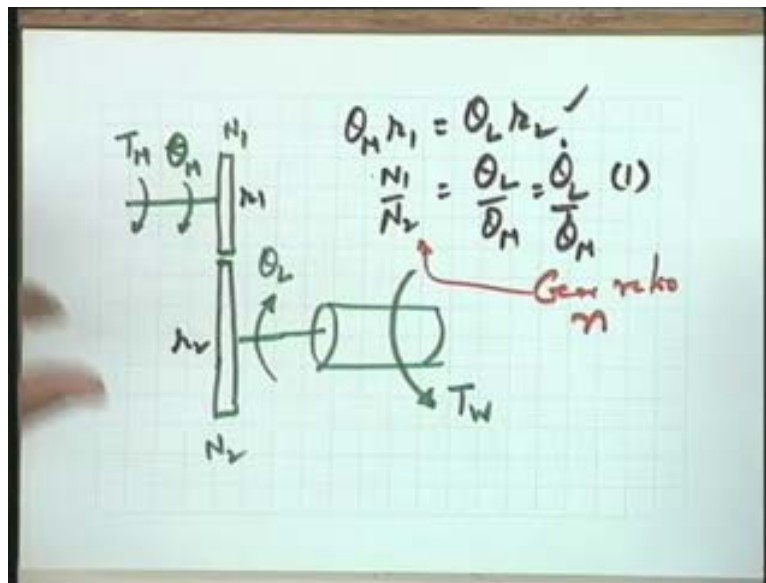
So now let me look at the gear train first so that the complete block diagram could be obtained. A set of two gears I am taking if the gears are more than two accordingly you can generate a suitable transfer function model for the gear train. Let us say that T_M is the torque generated here the motor torque, θ_M is the position, N_1 number of teeth on this gear, N_2 number of teeth on this gear and here is a load; naturally the reversal in direction will take place so naturally I am writing θ_L this way and this drive torque is going to drive the load in this direction, so, if I want to show disturbance I must show a disturbance this way (Refer Slide Time: 36:23) because the drive torque on this particular load is coming this way so this is your disturbance T_w .

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Help me to derive a suitable expression giving you the relationship between T_M and the torque here and the speed here and the speed there, I want a suitable relationship for that. Well, it follows this way. I take r_1 as the radius of this gear, r_2 as the radius of this gear, you will note that the number of revolution of the two gears will be different because the linear distance traveled by the two gears has to be the same. Therefore I can write $\theta_M r_1$ is equal to $\theta_L r_2$. I hope his point is well taken. $\theta_M r_1$ has to be equal to $\theta_L r_2$ because the linear distance traveled has to be the same. Assuming that r_1 and N_1 are proportional naturally they will be; r_2 and N_2 are proportional in that particular case I am writing N_1 by N_2 equal to θ_L by θ_M this I term as my first basic equation N_1 by N_2 is the teeth ratio and θ_L by θ_M is the ratio of the two positions; and you will not mind if I put it this way that this is equal to $\theta_L \dot{\theta}_M$ by $\theta_M \dot{\theta}_L$ as well that is the ratio of the two velocities because you can take the derivative of this equation and you can arrive at this particular expression if you take the derivative of this equation N_1 by N_2 is equal to $\theta_L \dot{\theta}_M$ by $\theta_M \dot{\theta}_L$ is the expression you get which gives you the ratio of the two positions or two velocities as equal to the teeth ratio, the gear ratio that is called N_1 by N_2 let me say is the gear ratio n small n .

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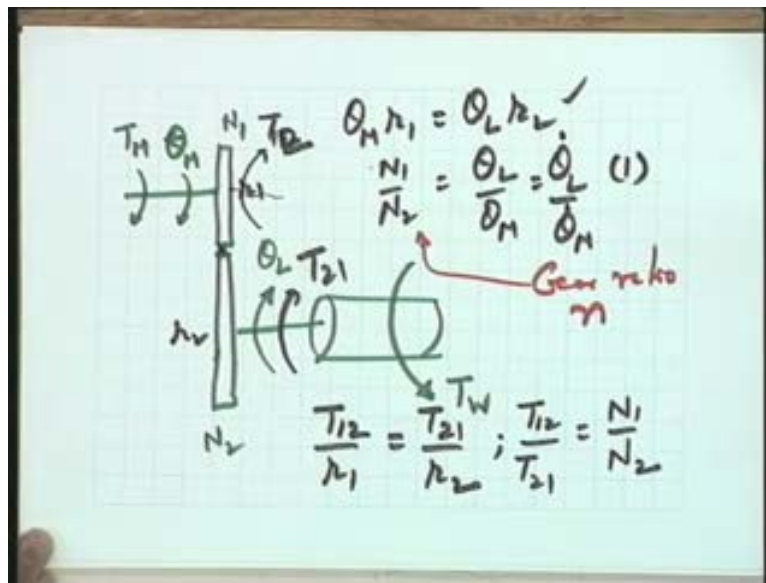


Now look at the torque expression. Now you see T_M is the torque generated by the motor. Now let me assume that the torque transmitted to this particular shaft is T_{21} . T_{21} is the torque transmitted through the gear train on to the load shaft. So T_{21} is responsible for driving the inertia against the disturbance T_W . You see this point: T_{21} is responsible for driving the inertia against the disturbance T_W . How about the gear one? Please see that if I remove gear two the total load torque on gear one will be less compared to the situation when gear two is present. It means, as far as the motor shaft is concerned the gear two and the load is loading the motor shaft.

Just look at the motor shaft only. the motor shaft is being loaded you can see the two situations; without the gear two and with the gear two the situation on the motor shaft becomes different and let me say that the total load provided on the motor shaft by the gear by this particular load and the gear is T_{12} . So I put it again; T_{12} is the loading of the motor shaft due to the load and the gear the secondary gear or gear two and T_{21} is the drive torque transmitted to the load shaft by the motor shaft. So in this case if I want a relationship between the two I know that at this point of contact the forces developed should be the same they cannot be different and therefore I write a relationship $T_{12} \text{ by } r_1$ is equal to $T_{21} \text{ by } r_2$ because the force developed at that particular point is $T_{12} \text{ by } r_1$ and the force developed if you look at T_{21} it is $T_{21} \text{ by } r_2$ and this gives me the relationship T_{12} is divided by T_{21} is equal to $N_1 \text{ by } N_2$. That is the ratio of the two torques is equal to the gear ratio.

This point needs your attention, please see. $N_1 \text{ by } N_2$ is the gear ratio and **this provide** this $T_{12} \text{ by } T_{21}$ the ratio of the two is equal to the gear ratio. And now look at the point I made that the gear train magnifies the torque and it reduces the speed provided your n is less than 1 and n as for the gear train shown over here (Refer Slide Time: 41:25) will be less than 1 because N_1 is smaller than N_2 . typically n could be 1 by 30 let us say, typical value you take n is equal to small n is equal to 1 by 30, if you take 1 by 30 you find $\theta_L \text{ dot}$ is equal to 1 by 30 into $\theta_M \text{ dot}$ it means the load speed has reduced by effector equal to the gear ratio.

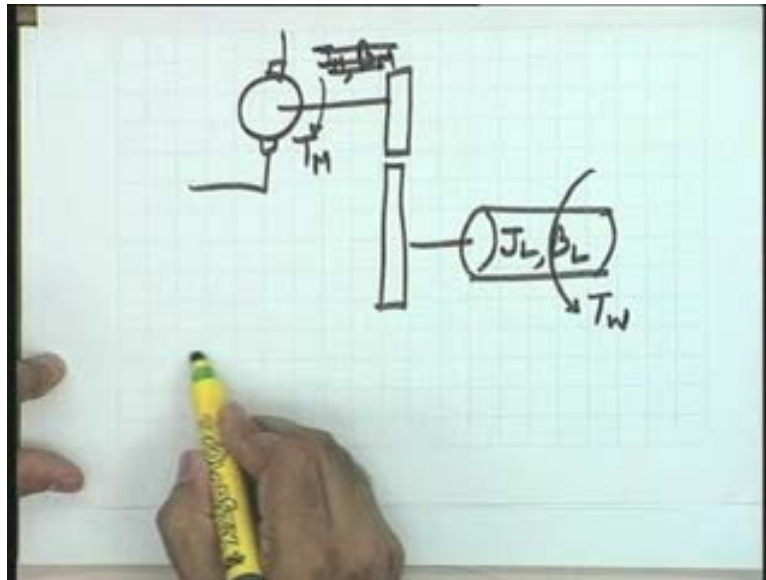
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T 21 equal to.... T 21 now you take in this, T 12 is 1 by 30 so T 21 is equal to 30 into T 12 the torque available at the load shaft has been increased by effecter of 30 which is inverse of the value n the gear ratio, your result. So it means the gear train here provides the required torque magnification and corresponding speed reduction and how do you fix up the gear ratio naturally depends upon the characteristics of the motor and the load. These two units will dictate the value of the gear ratio and accordingly you will design a suitable gear ratio for the system.

Now let me come to equations. Please see here is a motor, this is the load here, **come on give me the equation at the motor shaft**. I assume that this particular motor itself has got certain moment of inertia J M and viscous friction B M. the motor itself has got moment of inertia J M or let us neglect it for the time being (Refer Slide Time: 43:05) T M is the torque developed by the motor I take it this way. Let us say J L is the moment inertia here and B L is the viscous friction and T W is the load torque. I need your attention here please. I am now setting up the equations. Please set up the equations for the secondary term.

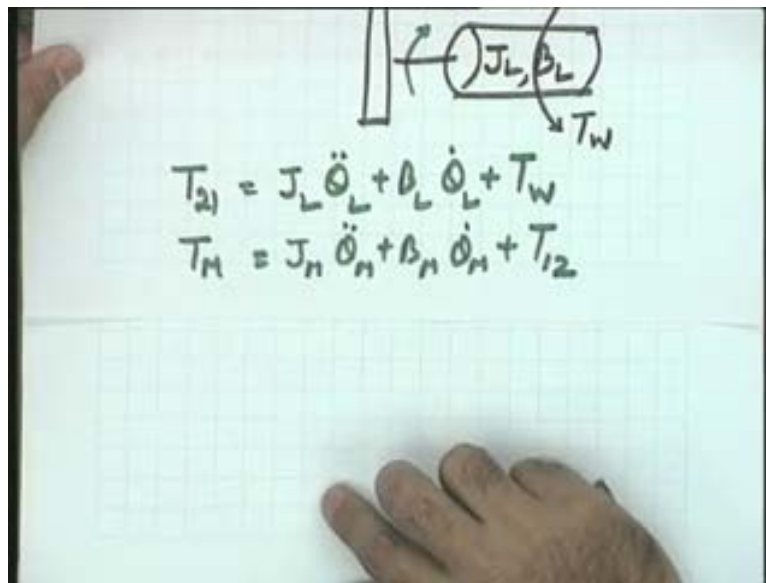
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For the secondary term the equation becomes: T_{12} for the secondary gear and the load, T_{12} is the torque transmitted by the motor this is equal to $J_L \ddot{\theta}_L + B_L \dot{\theta}_L + T_W$ this becomes the equation at the load shaft. Please give me the equation at the motor shaft. What is the applied torque T_M equal to? Yes, we said that the position due to the secondary gear and the load on the motor shaft is represented by T_{12} . Let me at this stage itself make the discussion complete: J_M and B_M could be taken as moment of inertia and viscous friction of the motor shaft there is a friction in bearings of the motor shaft there may be friction in this part of the system which has been captured into a coefficient B_M .

Help me please; give me the equation for the motor shaft T_M is the developed torque, θ_M is the angular displacement of the motor shaft in that particular case T_M is equal to $J_M \ddot{\theta}_M + B_M \dot{\theta}_M + T_{12}$. Now you see that what I want to do is I want to eliminate the variables T_{21} and T_{12} . Help me in that. If I want to do that let me retain these equations here.

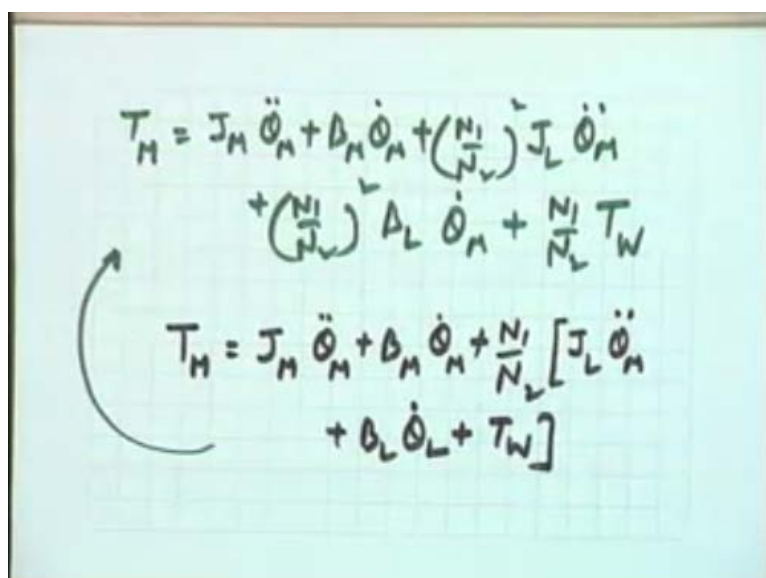
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Eliminating the variables T_{21} and T_{12} I can write the expression as, a very important expression I am going to give you at this stage: T_M is equal to $J_M \theta_M$ double dot plus $B_M \theta_M$ dot plus what is T_{12} T_{12} if you recall is nothing but N_1 by N_2 into T_{21} (Refer Slide Time: 45:43). So, if you want to replace this expression I can write this as: N_1 by N_2 into T_{21} which could be written as $J_L \theta_L$ double dot plus $B_L \theta_L$ dot plus T_W . These variables T_{21} and T_{12} have been eliminated.

Now you see one more point that the variables θ_L and θ_M (Refer Slide Time: 46:17) are not independent; they are related by a constant. So your equation having both the variables θ_M and θ_L is not logical. Your equation should have one of these variables either θ_M or θ_L and mostly the basic equation is written in terms of θ_M . So I want you to eliminate θ_L as well. **Come on please get started.** You eliminate this and I also do it and then compare the result.

(Refer Slide Time: 00:46:45 min)



Let me write it this way. I want to eliminate theta L and write the basic equation. This becomes $T_M = J_M \ddot{\theta}_M + B_M \dot{\theta}_M + \frac{N_1}{N_2} J_L \ddot{\theta}_M + \frac{N_1}{N_2} B_L \dot{\theta}_M + \frac{N_1}{N_2} T_W$. You will see this equation please: Theta L dot and theta L have been replaced as equal in terms of theta M, you know the expression: theta L by theta M is equal to theta L dot by theta M dot is equal to the gear ratio N 1 by N 2. So you see the interesting interpretation of this equation is the following: I can club these terms corresponding to theta M dot J M plus this term **J M plus this term** I am going to get as the total inertia term I can club these two terms B M plus this term I am going to get as the total frictional term and this is the equivalent disturbance on the load and hence the expression T M is equal to J M plus N 1 by N 2 squared J L into theta M double dot plus B M plus N 1 by N 2 whole squared B M B L **sorry yes, this is okay** (Refer Slide Time: 48:36) theta M dot plus N 1 by N 2 T W.

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$$T_M = \left(J_M + \left(\frac{N_1}{N_2} \right)^2 J_L \right) \ddot{\theta}_M + \left(B_M + \left(\frac{N_1}{N_2} \right)^2 B_L \right) \dot{\theta}_M + \frac{N_1}{N_2} T_W$$

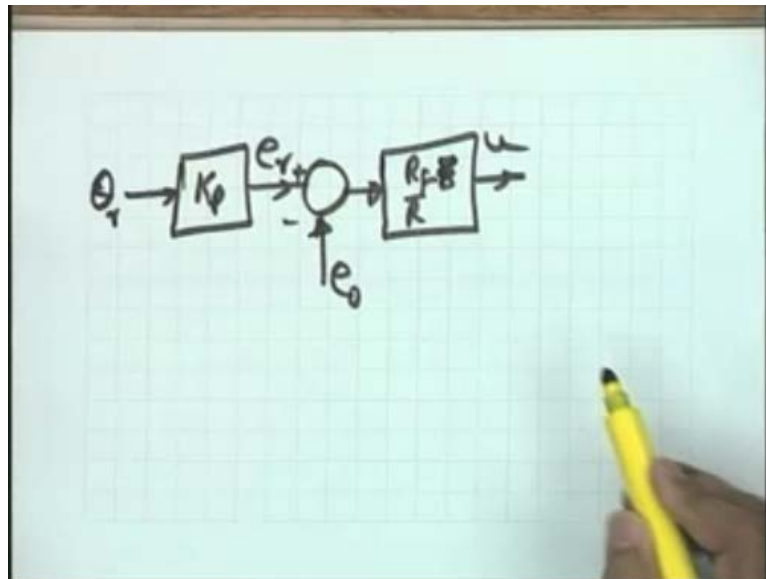
Please see what is the interpretation of this. The interpretation of this particular equation is this that, mathematically you see, you have been simplifying the block diagrams here also. Let us do some simplification. Mathematically your system which we have described as position control system is equivalent to the following situation: a motor, this is load, forget about the gear train altogether. You assume that a particular load is there directly on this particular motor.

What are the parameters of this particular load?

The parameters of this particular load are J equivalent; what is the value of J equivalent? You will find that J equivalent is given by this expression (Refer Slide Time: 49:27): J equivalent and you can take the load on this particular motor to be equal to B equivalent which is given by this particular expression and what is the disturbance directly coming on the motor shaft? Actually, physically speaking, the disturbance is coming on the load shaft but if you take the disturbance as n into T W where n is the gear ratio is equal to N 1 by N 2 these but becomes the representation; and why did we take this representation is because writing the mathematical equation for this representation becomes simple. So, with this representation for the motor shaft now the situation of the overall control system which we have taken becomes identical to the situation we have handled for speed control system and I leave this

as a simple exercise for you to give me the block diagram of the system. Or, well, I know the time limit but just let me quickly give you the block diagram and that is the end.

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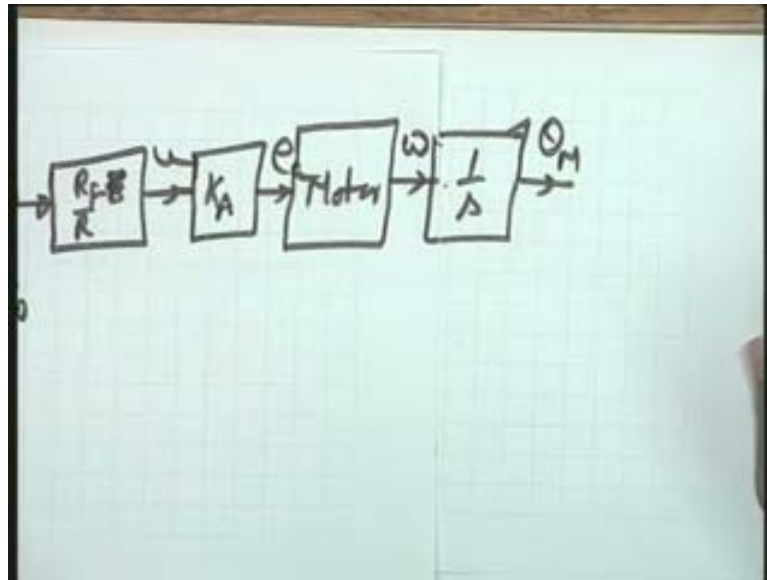


Theta r is the reference input, this is your K P which you can take inside, e r is the this voltage plus minus this is e 0 the feedback this is being compared **R f by R into K** R f by R gives you u here the controlled signal. What is this controlled signal doing? This controlled signal multiplied by K A gives you e a. let me save time over here and I put the motor here; the complete block diagram of the motor could be could be put over here at this particular point.

What is the output of the motor?

Output of the motor in this case is motor speed omega. Motor speed omega into 1 by s an integrator gives you position theta M. This position theta M from this particular point now I think you can connect the block diagram, you can convert it to theta L, take a suitable feedback loop through the output potentiometer and the inherent loop on the back of EMF also will come.

(Refer Slide Time: 51:50)



I leave this as an exercise for you to complete the block diagram for the system. There is a pressure of time, thank you very much.