

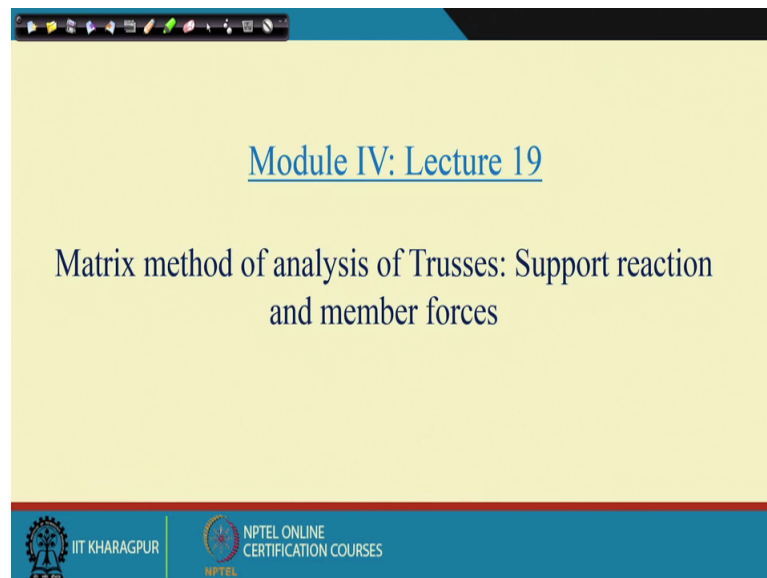
Matrix Method of Structural Analysis
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Lecture – 19
Matrix Method of Analysis Trusses (Contd.)

Hello everyone, welcome you all. Now, in the previous class we discussed how to enforce boundary conditions once you have already assembled the stiffness matrix. And then we discussed how to partition this stiffness matrix based on the known information about degrees of freedom and then you can solve for unknown displacement.

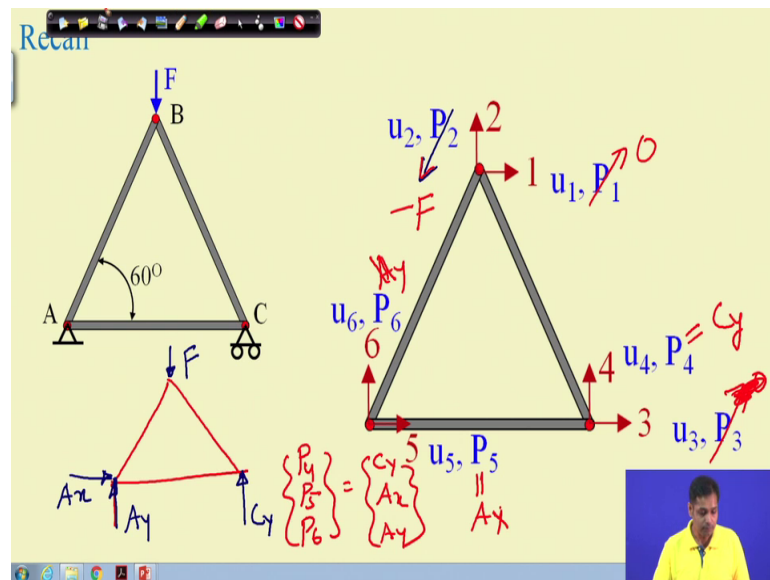
Now, what we do today is once you have obtained the unknown displacement how to determine the next information that we need about the; an about the structure is what are the boundary, what are the support reactions and the member forces.

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So, today that is what we are going to see. So, today is analysis of truss, support reaction and member forces.

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Now, recall this is the problem that we are discussing, and then these are the corresponding degrees of freedom and the forces and then we have the global stiffness matrix like this.

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$$\frac{AE}{L} \begin{bmatrix} \frac{1}{2} & 0 & -\frac{1}{4} & \frac{\sqrt{3}}{4} & -\frac{1}{4} & -\frac{\sqrt{3}}{4} \\ 0 & \frac{3}{2} & \frac{\sqrt{3}}{4} & \frac{3}{4} & -\frac{\sqrt{3}}{4} & -\frac{3}{4} \\ -\frac{1}{4} & \frac{\sqrt{3}}{4} & \frac{5}{4} & -\frac{\sqrt{3}}{4} & -\frac{1}{4} & 0 \\ \frac{\sqrt{3}}{4} & \frac{3}{4} & -\frac{\sqrt{3}}{4} & \frac{3}{4} & 0 & 0 \\ -\frac{1}{4} & -\frac{\sqrt{3}}{4} & -\frac{1}{4} & 0 & \frac{5}{4} & \frac{\sqrt{3}}{4} \\ \frac{\sqrt{3}}{4} & \frac{3}{4} & 0 & 0 & \frac{\sqrt{3}}{4} & \frac{3}{4} \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \\ 0 \\ 0 \\ 0 \end{Bmatrix} = \begin{Bmatrix} 0 \\ -F \\ 0 \\ P_4 \\ P_5 \\ P_6 \end{Bmatrix}$$

And this global stiffness matrix relates the global load vectors and the global displacement field. And then we know some of the information about the displacement and that information is given here and we know some of some information about the

force field and those information's are also those information's also given in the load vector, ok.

Then if you recall what we do is we partition the matrix partition the all the load vector displacement vector and consequently the stiffness matrix as well. And the partitioning is done based on what information you know and what information you do not know. So, in this case we this was our u , we do not know this information so this is u unknown, we know this information so this is u known. And the fault force vector this is force this is P known and this is P unknown this is P unknown, ok, this is how we partition the load vector.

And corresponding we once we have partitioned the load vector and the corresponding the stiffness matrix can also be partition like this, ok. So, this portion, this portion that the this portion gives you, ok. So, this we that is we name this is K known a K_{kk} and this is we used K_{ku} and this is we used K_{uk} , though uk and ku they are same because the matrix the entire matrix is meta matrix is symmetric matrix and this was is K_{uu} . This is how we use the partitioning, but there is no hard, and fast rule that we use the these you use kk K_{ku} something like this in there are many books where instead of K_{kk} it is written as say it is written as say K_{11} , K_{11} , K_{12} and then K_{21} and K_{22} .

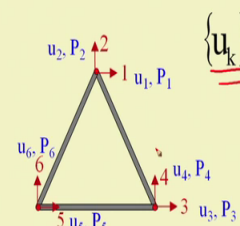
So, essentially how you define, how you represent it is not that important. Important is you understand the partitioning either you can represent this matrix by this or these or any other that you were comfortable with ok.

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Recall

$$\begin{Bmatrix} P_k \\ P_u \end{Bmatrix} = \begin{bmatrix} K_{kk} & K_{ku} \\ K_{uk} & K_{uu} \end{bmatrix} \begin{Bmatrix} u_k \\ u_u \end{Bmatrix}$$

$$\begin{Bmatrix} P_k \end{Bmatrix} = \begin{Bmatrix} P_1 \\ P_2 \\ P_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ -F \\ 0 \end{Bmatrix}$$

$$\begin{Bmatrix} P_u \end{Bmatrix} = \begin{Bmatrix} P_3 \\ P_4 \\ P_5 \end{Bmatrix} = \begin{Bmatrix} C_y \\ A_x \\ A_y \end{Bmatrix}$$


$$\begin{Bmatrix} u_k \end{Bmatrix} = \begin{Bmatrix} u_4 \\ u_5 \\ u_6 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix}$$

$$\begin{Bmatrix} u_u \end{Bmatrix} = \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix} = \frac{PL}{AE} \begin{Bmatrix} 0.1433 \\ -0.7500 \\ 0.2887 \end{Bmatrix}$$

Now, once we have that then if you recall that we can write the entire load displacement relation like this after partitioning we can, this is the entire load displacement relation and where P_K are the $P_1 P_2 P_3$ which are 0 which are which are 0 minus F and 0 and then P_u is equal to this, ok.

Here you see if I if I go back, if I say here you see if I draw the if I draw the stiffness if I draw the free body diagram of this of this structure, free body diagram will be, there will be a support reaction there will be a support reaction this is say A_y and this is say A_x and this is C_y , this is C_y , and the force is this is the force F we have, ok.

Now, from this free body diagram if we compare this free body diagram with this with this diagram, then we know that P_2 is equal to minus F anyway we discuss it already P_2 is equal to minus F and P_1 is equal to 0 there is no force and similarly P_3 is equal to 0. But then other 3 unknown forces they are related to support reaction as this. So, sorry not this one yeah ah other 3 forces are related to like this say P_4 is equal to C_y and then P_5 is equal to A_y and P_6 is equal to this is a A_x a_x and P_6 is equal to a y .

So, if we write P_4 , P_4 , P_5 and P_6 , P_6 they are essentially C_y c_y A_x and a_y . So, if we can determine these unknown force unknown unknown P 's so that is equivalent to determining the reactions, ok. Now, so this is the partitioning. Now, so that that is what he written here, so P_{unknown} is equal to which was $P_3 P_4 P_6$ is equal to support reactions at C_y , A_x and A_y . So, this is unknown that we have to determine.

Now, u unknown that we already determine in the last class that was the solution from the last class and then u known already 0 because that is what the boundary conditions, ok. Now, what we will do today is we will determine what is P_u which is support reactions and then once we have the support reactions how to then forces in each member the internal forces we have to determine, ok.

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$$\begin{Bmatrix} P_k \\ P_u \end{Bmatrix} = \begin{bmatrix} K_{kk} & K_{ku} \\ K_{uk} & K_{uu} \end{bmatrix} \begin{Bmatrix} u_k \\ u_u \end{Bmatrix}$$

$$P_k = K_{kk} u_k + K_{ku} u_u \quad \text{--- (1)}$$

$$P_u = K_{uk} u_k + K_{uu} u_u \quad \text{--- (2)}$$

$$P_u = K_{uk} u_u \quad \text{--- (3)}$$

Now, let us start with this expression. Now, this is the expression and if you recall this expression was written like this we had P_k is equal to K_{kk} into u_k plus K_{ku} into u_u and P_u is equal to P_u is equal to K_{uk} into u_k plus K_{uu} into u_u , ok. Now, ah this is this is u this is K there is a mistake, ok. So, this should be u this should be K and this should be u this should be k right should be k , ok.

Now, this is anyway 0, so we use this expression we use this expression to find out the unknown reaction, ok. And this was x equation number 1 and I told you that how to use this equation number 2 we will be discussing ah in lecture 4. Now, that is what we will be doing now. Now, we can we will be using this expression to find out the unknown forces, ok.

Now, this u_k is anyway 0, so we can straight away we can say that this is 0. So, this expression finally, becomes P_u is equal to K_{uk} , u_u . This expression becomes this, right. Now, what is u_k ? u_k is, if you go back to the expression u_k will be this part will be this part, ok. So, this into this plus this into this is equal to this, ok. So, write it here now.

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The image shows a handwritten derivation of the reduced stiffness matrix and a corresponding truss diagram. At the top left, the global stiffness matrix equation is shown with the rows and columns corresponding to the known displacement u_k crossed out:

$$\begin{Bmatrix} P_k \\ P_u \end{Bmatrix} = \begin{bmatrix} K_{kk} & K_{ku} \\ K_{uk} & K_{uu} \end{bmatrix} \begin{Bmatrix} u_k \\ u_u \end{Bmatrix}$$

Below this, the reduced stiffness matrix is derived by eliminating the known displacement terms:

$$\begin{Bmatrix} P_4 \\ P_5 \\ P_6 \end{Bmatrix} = \begin{bmatrix} \frac{\sqrt{3}}{4} & -\frac{3}{4} & -\frac{\sqrt{3}}{4} \\ -\frac{1}{4} & -\frac{\sqrt{3}}{4} & -1 \\ -\frac{\sqrt{3}}{4} & -\frac{3}{4} & 0 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix}$$

To the right of the matrix is a truss diagram of a triangular structure with a vertical force F applied at the top node. The bottom-left node has a horizontal reaction A_x and a vertical reaction A_y . The bottom-right node has a vertical reaction C_y . A box next to the diagram contains the equilibrium equations:

$$\begin{aligned} C_y &= A_y = \frac{F}{2} \\ A_x &= 0 \end{aligned}$$

At the bottom, the nodal load vector is determined from static equilibrium:

$$\begin{Bmatrix} P_4 \\ P_5 \\ P_6 \end{Bmatrix} = \begin{Bmatrix} 0.5F \\ 0 \\ 0.5F \end{Bmatrix} = \begin{Bmatrix} C_y \\ A_x \\ A_y \end{Bmatrix}$$

So, P_u , so we have expression this we have. Now, P_u which is equal to P_4, P_5, P_6 are unknown is equal to the stiffness matrix and then the displacement u_1, u_2 and u_3 . And this matrix is 3×3 you can check from the global stiffness matrix minus 3×3 and then minus 3×3 . Then minus 1×3 minus 3×1 and then minus 1 and then minus 3×1 and this becomes 3×3 and then become 0 , ok.

Now, now, do not hear the diagonal one of the diagonal term is one of the diagonal term is nonzero ah that is fine because that does not ah the that is not against the property of stiffness matrix that you decide because this K the diagonal terms gives you by these 2 expression this one and this one. So, we are actually this one is K_{uk} term which is off diagonal off diagonal matrix and essentially. So, naturally it can have negative entries, ok.

Now, then we know u_1, u_2, u_3 , just now if we substitute u_1, u_2, u_3 as from the previous from the previous expression u_1, u_2, u_3 , you check u_1, u_2, u_3 , is this. Now, if I substitute that u_1, u_2, u_3 , in that expression what we get is, we get u we get unknown forces and the solution will be P_4, P_5, P_6 they are equal to $0.5, 0, 0.5$ or $0.5F, 0, 0.5F$ these are the force.

What are P_4, P_5, P_6 ? If you recall P_4 was C_y this was A_x and this was A_y and from the static equilibrium condition if it is subjected to force F and these are the your support reactions by applying the this is C_y, A_y and A_x . By applying the equilibrium equation we

can easily say that we can solve this Ay a Cy is equal to a y is equal to F by 2 and a x is equal to 0 that is the solution you can, if you can directly apply equilibrium equation you get it. And then you are getting the same expression here.

You may ask that once you have once we can directly get the solution for by equilibrium equation what is the purpose of all these exercise. The purpose of all these exercise is not for the structure where you can easily solve using manual calculation. These exercise has to be done for basic weight for a structure which is in determine at the very large a very large structure where the manual calculations are really not feasible, ok.

For demonstration we have taken an example where that example can be solved by equilibrium equations and we are solving through the matrix method of structure energy so that the solution can be compared easily. So, this is the support reactions we can obtain from this. So, in this expression now, all these u_k is known u unknown is determined already and then P_k is known and P unknown is this. So, in this expression all these displacement vector, force vector everything we know.

So, what we know about the structure as of now, we know the displacement of all the joints and we know the reactions at all the reactions and the joints wherever we have supports. Now, recall when we analyze truss along with the reactions we also need member forces, ok. Now, next see how to determine the member force, ok.

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The diagram shows a truss structure with nodes 1, 2, 3, and 4. Node 1 is at the bottom left, node 2 is at the top left, node 3 is at the top right, and node 4 is at the bottom right. Members connect nodes 1-2, 1-3, 2-3, and 3-4. Displacements u_1, u_2, u_3, u_4 are shown at each node. Forces F_1, F_2 are shown at nodes 1 and 2. The following equations are written:

$$\begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix} = \frac{AE}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix} \quad (1)$$

$$\begin{Bmatrix} u_1' \\ u_2' \end{Bmatrix} = \begin{bmatrix} \lambda_x & \lambda_y & 0 & 0 \\ 0 & 0 & \lambda_x & \lambda_y \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \end{Bmatrix} \quad (2)$$

$$\begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix} = \frac{AE}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{bmatrix} \lambda_x & \lambda_y & 0 & 0 \\ 0 & 0 & \lambda_x & \lambda_y \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \end{Bmatrix}$$

$$\begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix} = \frac{AE}{L} \begin{bmatrix} \lambda_x & \lambda_y & -\lambda_x & -\lambda_y \\ -\lambda_x & -\lambda_y & \lambda_x & \lambda_y \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \end{Bmatrix}$$

Additional notes in the diagram include $\lambda_x = \cos \theta$ and $\lambda_y = \sin \theta$.

Now, ok; you see take if we recall if we have a if we have a truss member any arbitrary member like this and this is number 1 and this is number 2. And then we have degrees of freedom as u_1 and here degrees of freedom as u_2 and it is F_1 and it is F_2 , then recall the first class we discuss these F_1 and F_2 they are related to u_1 and u_2 as this AE by L and then 1, minus 1, minus 1, 1 and this is u_1 and u_2 , that in that this expression already we know.

Now, we also know that again for the same structure the if we if the degrees of freedom are u_1 , u_2 and then it is u_3 and u_4 , and this angle is say theta, this angle is theta then we also know that that u_1 ; that u_1 and u_2 they are related to u_1 they are related to u_1 , u_2 , u_3 , u_4 as λ_x , λ_y , 0, 0, 0, 0, λ_x and λ_y right and then this is u_1 , u_2 , u_3 , u_4 that is the expression we know, ok.

Now, if I substitute this expression, this expression say expression number 2 and this is expression number 1 if I substitute expression 2 into expression one what we get is this, we get F_1 F_2 that is equal to AE by L you can parallelly try this. If we substitute this is minus this is 1, minus 1 and then minus 1, 1 and this become λ_x , λ_y , 0, 0, 0, 0, λ_x , λ_y and this is u_1 , u_2 , u_3 , and u_4 where λ_x is equal to $\cos \theta$ and λ_y is equal to $\sin \theta$, ok.

Now, if I if I do that what we get is F_1 , F_2 that is equal to AE, this is just simple matrix multiplication AE by L this is λ_x , λ_y this is minus λ_x and minus λ_y and this is minus λ_x , minus λ_y then λ_x , λ_y . And then this is u_1 , u_2 , u_3 , and u_4 this is important, this expression is important.

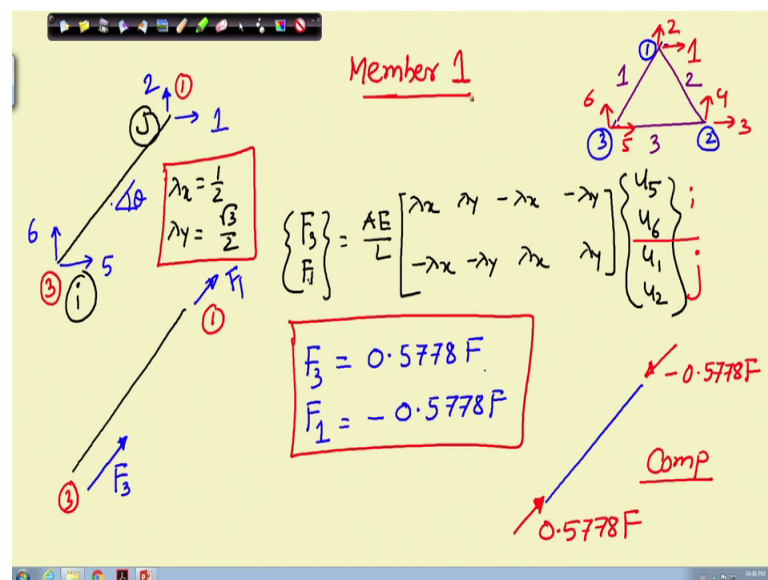
Now, if you observe this expression what you get is F_1 is equal to minus F_2 , right. Either you can use F_1 can you compute F_1 or F_2 , but they are opposite, they are opposite because which is again is consistent with the fact that the all the class members are 2 force members so the force that we have always along the longitudinal axis of the truss, ok. So, which says that this F_1 , this F_1 and F_2 they will be equal and opposite and whether they are whether F_1 is positive or F_2 is positive depending on that we can say the member is either in compression or in tension.

For instance, if your numbering is like this means if your if you are starting from one and if your movement is like this direction. So, F_1 is from 1 to 2 and F_2 is the same direction. If F_1 is positive means F_1 is your that is the direction we considered as

positive if we if your forces are in this direction and F 2 is in this direction suppose your F 1 is this direction and F, F 1 and F 2 is this direction the member is under compression. And if your F 1 is in this direction and F 2 is this direction we can say his member is under tension.

So, if F 1 is positive and F 2 is negative we say it is compression member if F 1 is negative and F 2 is negative we say it is tension member. Now, F 1 and F 2 its not arbitrary it depends on which is your starting know starting joint node and what is your end node because that is how you calculate your define angle theta angle theta is a measure in this direction, ok. So, this is now, this expression will be shortly we will be using to calculate force in the members. Let us do that, ok.

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Now, we have 3 we have this was the class, we have member number 1, member number 2 and this was member number 3 and then your node numbering was in this it was degrees of freedom 1 2 3 4, it was 1, then 2, 3, 4, then 5 and 6 this is how we number the degrees of freedom, ok. Now, and this node number is use different color say and your numbering was this was your node number 1, this was our node number 2 and this was our node number 3, ok. Now, let us find out what is the forces in member 1 first.

The member 1 is the ah I am trying to write all these steps very explicitly, but really when you just for your understanding, but when you actually compute them you do not have to do there are way you can quickly do it ok. So, this is the member number 1,

where your this is node number 3 for member 1, this is node number 3 and this is node number 1 this is how we defined, ok. And these are the degrees of freedom this is your 5, this is 6 and this is 1 and this is 2 right and then this is the, this is this angle is theta, this angle is theta.

And if you refer to our previous classes we for different members what are the lambda x and lambda y are given, in this case the lambda x is equal to lambda x is equal to half and lambda y is equal to root 3 by 2 that we have already, ok. Now, so in this case if I go back if I see the previous one this expression we will be using. Now, this expression we will be using.

Now, in the previous case you see it was node number 1, this was node number 2 that is why it is we wrote F 1 and F 2, but if it is node number 3 node number 4 we can write F 3 and F 4. Similarly for any arbitrary if it is i and if it is j then it is true for F i and F j, where the measure your the numbering from i to j. And this is again I am telling this is very important because that is how we measure the angle ok.

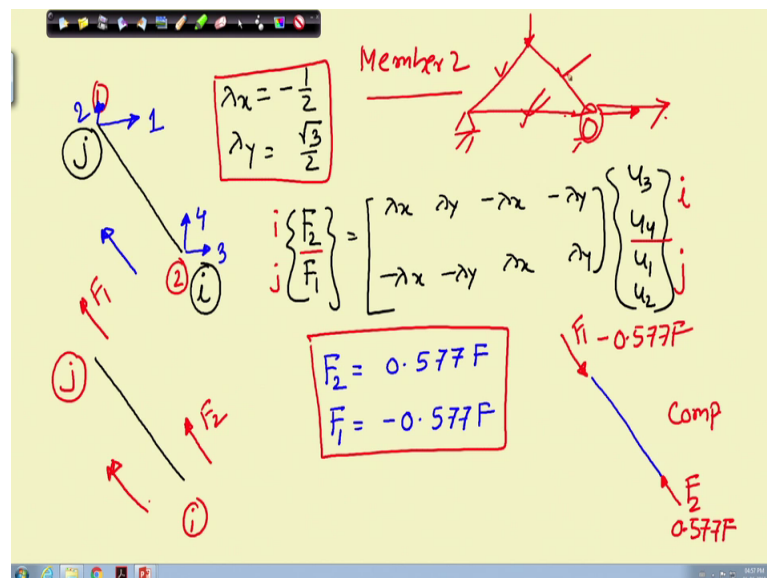
So, now, in this case your i is 3 and j is 1. So, expression will be F 3, F 3 and F 1 they are they will be related as AE by L this expression is lambda x, lambda y, then minus lambda x, minus lambda y and then minus lambda x, minus lambda y, lambda x and lambda y and then this will be your node number 3 your degrees of freedom at 5 6, this is u 5, then u 6, u 1 and u 2, ok. So, in this case actually this is our ith node and this is our jth node, ok. So, this is the force at ith node this is the force at jth node. Similarly these are the displacement these are the degrees of freedom for ith node these are the degrees of freedom for ith node these are the degrees of freedom for jth node, ok.

Now, so if I substitute the values of lambda and lambda x and lambda y in this what we get is we get F 3, F 3 is equal to 0.5778F and F 1 is equal to minus 0.5778F which is obvious because they are equal and opposite. Now, let us understand whether the member is under tension or compression how we, how we do that how we understand that. Draw this member once again.

How we this was this was the num this was 3 and this was 1, this was 3 ah this was 1. Now, how we define the forces? We define the force like this. So, this was F 3 and here it was F 1 right. Now, F 3 is positive and F 1 is negative it means your force in this

You again I said when you actually do it using computer then you do not you do not really draw it and then check whether tension and compression just based on the sign we can say whether the member is under tension and compression. These exercise I am doing so that you can correlate you can understand how to identify a member compression or tension. So, this is for member 1, this was for member 1, member 1.

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Now, if it is then for this what we what we have is i th force, i th force is F_2 say F_2 and j th force is i th force is F_2 and j th force is F_1 , F_1 that is equal to same expression

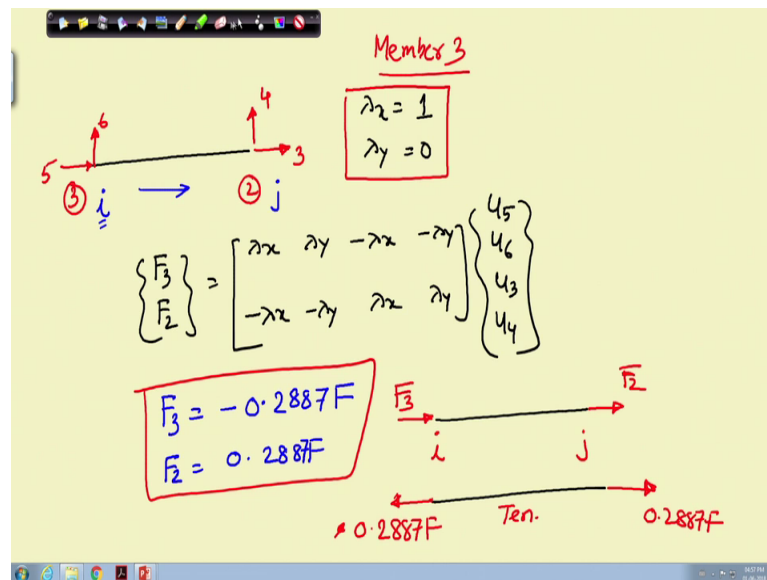
λ_x , λ_y , $-\lambda_x$, $-\lambda_y$, $-\lambda_x$, $-\lambda_y$, λ_x and λ_y . And then here we have displacement at i th node and the displacement at j th node i th node degrees of freedom are u_3 , u_4 and then j th node is u_1 , u_2 right. So, this is for i th node and this is for j th node. This is for i th node and this j th node.

Again if we substitute λ_x and λ_y in these expression we get expression for F_2 , expression for F_2 is equal to F_2 as $0.577F$ and F_1 is equal to $-\text{minus } 0.577F$ that is the expression for F_1 and F_2 .

Let us see where the member is tension and compression. If we if we draw this member once again it is it was a i th node it was a i th node and it was j th node. So, F_2 , F_2 is in the i th node and you are moving in this direction i to j . So, your F_2 is, F_2 and F_1 is this, ok. So, in this case, so essentially the member is then the member is F_2 is this the forces are like this, forces are like this and F_1 is opposite direction F_1 is opposite direction this is F_1 this is F_2 both are same because is a 2 force member. So, this is $0.577F$ and similarly this is $-\text{minus } 0.577F$. So, member is under compression, ok.

So, by intuition we can say that if you have a truss like this and you apply a force like this and actually this member and this member will be under compression. These joints will go in this direction. So, this member will be under tension. Let us see whether we get the same thing or not. So, this was for member number 2, member number 2. Let us do it for the last member, member 6, a member 3.

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So, member 3, if you look at member 3 is this where your i and j is it was the member 3 λ_x and λ_y defined as this. So, let us draw member 3. So, member 3 is horizontal member where this joint is this joint is 3, this joint is 3, this joint is 2 and these are the degrees of freedom, these are the degrees of freedom. This is, this is 3, this is 4 and this is this is 5 and this is 6 right.

And then for this member 3 λ_x and λ_y . λ_x you have λ_x is equal to 1 and λ_y is equal to 0. And you get this λ_x and λ_y while computing λ_x and λ_y it is assumed that this is i th node, this is i th node and this is j th node that is why you get θ is equal to 0. And if you get if you take this as a i th node this as j th node and move from this to this direction you get θ is equal to π . So, for this i is equal to and then you get line λ_x one you get λ_x is equal to minus 1, ok. So, we are moving in this direction i th point this j th point is.

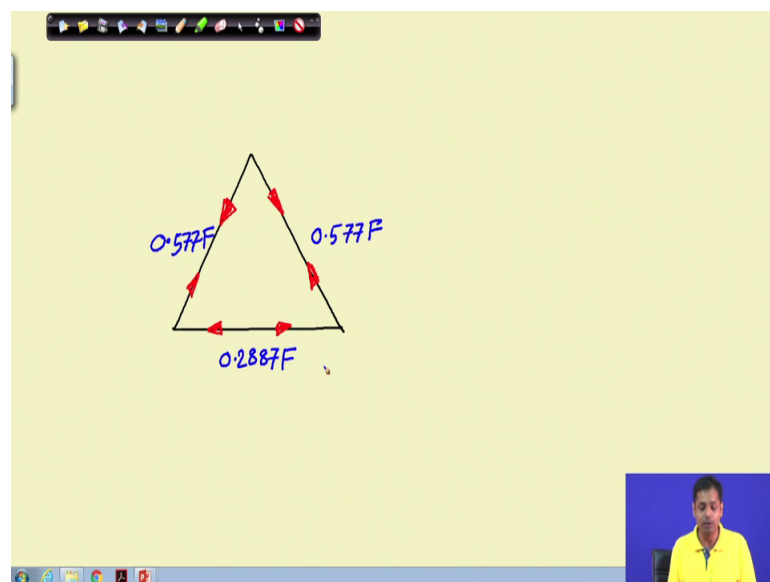
So, now, write the expression for F_1 at the; then this case this F_3 the force at i th point and force at j th point that is equal to the same expression λ_x , λ_x , λ_y , minus λ_x , minus λ_y , minus λ_x , minus λ_y , then λ_x and λ_y . And the degrees of freedom will be since we are moving from the i th point degrees of freedom and j th point degrees of freedom i th point degrees of freedom u_5 and then u_6 , u_3 and u_4 , right.

Then you substitute λ and λ_x and λ_y here if we do that then we get finally, F_3 is equal to F_3 is equal to minus $0.2887F$ and F_2 is equal to $0.2887F$. So, they are opposite direction and this is opposite direction, ok.

Now, let us try to understand what is the what is the weather is the compression or tension, member force is this and then i th point is i th this is our i th point and this is our j th point. So, i th point force is in this direction and this is our this is F_3 , F_3 and j th point force in this direction which is F_2 , F_2 . Now, look at this value F_3 is negative, F_2 is positive, so essentially in this case the forces will be, this is and the force here is this and this value is $0.2887F$ and this value is $0.2887F$. So, this is under 10. ah

This is negative actually when you are showing in opposite direction you do not have to write negative. I wrote it in the previous slides, but when you are showing the direction so this negative is does not does not make any sense, ok. So, so this means it is under, tension it is under tension which is very obvious for this member for this member as this deflects this these joints goes this direction, so this member will be under tension and you get this member under tension.

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So, now, the member forces are we can write we can draw the member forces as this. This is the truss. We just now, saw this member is under compression, this member is under compression and this member is under tension right. And the values are this is this is $0.2887F$, this is $0.577F$ and this is $0.577F$, you may write in 5 decimal 2 decimal any

decimal places you can check whether the equilibrium is being satisfied or not. So, this is the final solution this is the internal force member and then support reactions are the displacement we have already determined, ok.

Now, since you are actually doing some numerical calculation you may not get exactly the equilibrium because of the round off error how many decimal places you stored the value depending on that whether you get 0 or very small number. So, this is how we can determine the member forces. So, what we have done so far we took a truss and then we discuss how to write these stiffness matrices for each member and then we discussed how to write the global stiffness matrix by assembling the all the member stiffness matrices.

Then, we discuss once you have the global stiffness matrix how to apply the boundary condition and then consequently partition the matrix. And then we discuss how to determine unknown displacement, once we have determined unknown displacement we also discussed how to determine the unknown support reactions and then finally, unknown member forces, ok. So, this is the step we follow in matrix method of analysis of trusses, ok.

Now, one thing is very important here is that you will come across as you will probably appreciate once you actually solve some example doing for large problems how you numbering a system, how in what order you are numbering the nodes, how you are numbering the members that is very important. That will not discuss right now, let us first understand how to implement this, how to do this exercise is in computer then that time we will discuss how to give the numbering of different members and nodes, ok.

So, we will stop here today. Next class what you do is we will see some more example of some more example of um, some more examples. Some more members will not solve the here the purpose of all these exercise the first 4 lectures purpose has been to demonstrate entire calculation. In the next class also we will see some of some example couple of examples we will just briefly discuss these steps.

Not doing any calculation, we will see how to what would be the size of the global stiffness matrices, how to what are the known vector displacement vector known force field and then what would be the corresponding partition of the matrixes. Those we will discuss in the next class. Ok then, I stop here today. See you in the next class.

Thank you.