

Course on Design of Steel Structures
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Mod 02 lec08
Design of High Strength Friction Grip Bolts

Hello, in last lecture, I have discussed the design methodology of ordinary black bolt (0:25). Now today I will discuss the design principle of high strength friction grip bolt. Now in case of high strength friction grip bolt, the friction will be coming into picture for calculating the design strength of the bolt. Now as I told earlier also that high strength friction grip bolt is used when the external force is quite high. To accommodate the bolt in a shorter length of the joint, we may have to reduce the number of bolt. So in that case, generally we go for high strength friction grip bolt, with lesser number of friction grip bolt we can design the joint. Now I will come to the design philosophy and first we will see that the shear strength of high strength friction grip bolt how to calculate.

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shear strength of HSFG Bolt

$$V_{nsf} = \mu_s \cdot n_e \cdot K_h \cdot F_0$$

Table 20

$A_{nb} \cdot f_o$

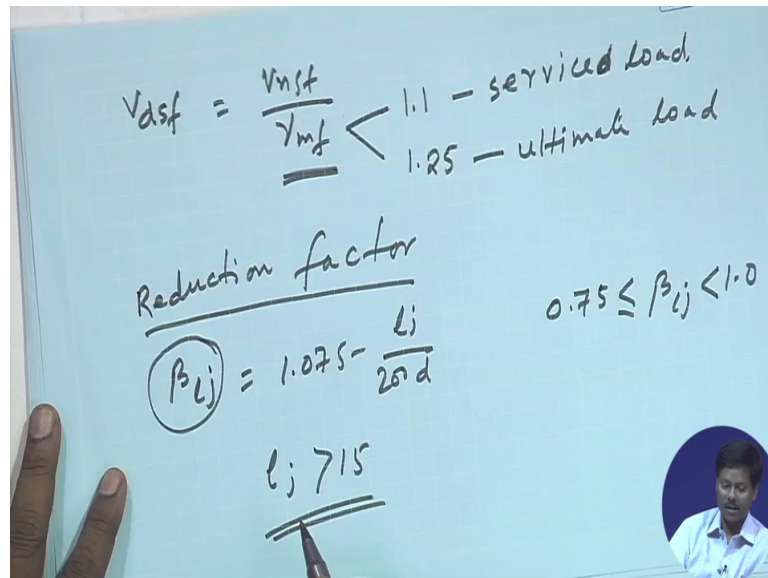
$f_o = 0.7 f_{ub}$

$K_h = 1.0 \rightarrow$
 $= 0.85$
 $= 0.7$

Shear strength of HSFG bolt. Now as per the code provision the formula is given that V_{nsf} is equal to μ_s into n_e into K_h into F_0 . Now this V_{nsf} is the nominal shear capacity of the bolt and this μ_s is the coefficient of friction. This is given in table 20 of IS 800 and in clause 10.4.3, it is described. The value of K_h are also given in the code that is taken as 1.0 when bolts are in standard holes (0:2:32) and it is taken as 0.85 for (0:2:38) in oversized and short slotted holes and long slotted holes loaded perpendicular to the slots and this K_h value can be taken as 0.7 for (0:2:49) in long slotted holes loaded parallel to the slots.

And this n_e value n_e is the number of effective interfaces offering frictional resistance to (()) (3:01) and F_0 , F_0 is the proof load that can be calculated as $A_n b$ into f_0 small f_0 . So $A_n b$ into small f_0 where $A_n b$ is the net area of bolt and f_0 is the proof stress, this is proof load and this is proof stress and this f_0 can be calculated as 0.7 times f_{ub} , f_{ub} is the ultimate tensile stress in bolt. So the proof stress f_0 can be calculated from this formula 0.7 times f_{ub} .

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$$V_{dsf} = \frac{V_{nsf}}{\gamma_{mf}} < \begin{matrix} 1.1 - \text{service load} \\ 1.25 - \text{ultimate load} \end{matrix}$$

Reduction factor

$$\beta_{lj} = 1.075 - \frac{l_j}{200d}$$

$l_j \geq 15d$

$0.75 \leq \beta_{lj} < 1.0$

Now the design shear force V_{dsf} will become V_{nsf} nominal shear force divided by gamma. Gamma γ_{mf} is the partial safety factor. This can be taken as 1.1 if strip resistance is designed at service load, at service load if it is designed then we will take 1.1 and we can take 1.25 if strip resistance is designed at ultimate load, at ultimate load it is designed then we consider 1.25 and like bearing type bolt here, also reduction factor for long joints are consider, for long joints reduction factor reduction factor for long joints. So that β_{lj} as we calculated earlier that will be 1.075 minus l_j by 200d and this value should not be less than 0.75 and more than 1.0 and this length of joint if it is more than 15d then only we will apply this reaction factor β_{lj} otherwise we will not consider.

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1) Shear strength of HSFG Bolts

$$V_{nsf} = \mu_f n_e K_h F_0$$

V_{nsf} = nominal shear capacity of bolt

μ_f = coefficient of friction Ref. Clause 10.4.3, Table 20

$K_h = 1.0$ for fasteners in clearance holes

= 0.85 for fasteners in oversized and short slotted holes and long slotted holes loaded perpendicular to the slots

= 0.7 for fasteners in long slotted holes loaded parallel to the slots

n_e = number of effective interfaces offering frictional resistance to slip

F_0 = proof load = $A_{nb} f_0$

f_0 = proof stress = $0.7 f_{ub}$

A_{nb} = net area of bolts at threads

f_{ub} Ultimate tensile stress in bolt

$$V_{dsf} = V_{nsf} / \gamma_{mf}$$

V_{dsf} = design shear force

γ_{mf} = partial safety factor

= 1.1 if slip resistance is designed at service load

= 1.25 if slip resistance is designed at ultimate load

Reduction Factor for Long Joints

If length of joint $l_j > 15d$, where d = nominal diameter of bolts,

then $\beta_{lj} = 1.075 - l_j / (200d)$, $0.75 \leq \beta_{lj} \leq 1.0$

V_{nsf} is reduced by a factor β_{lj}

2) Bolts in Bearing

$$V_{npb} = 2.5k_b d t f_u$$

V_{npb} = nominal bearing strength of bolt

f_u = ultimate tensile stress

d = nominal diameter of bolt

t = summation of thickness of connected plates experiencing bearing stress in same direction

Now in case of HSFG bolt, we also consider the bearing failure. So this bearing failure we can calculate as we have calculated in case of bearing type of bolt. So if we see the V_{npb} is becoming $2.5 K_b$ into d into t into f_u where V_{npb} is a nominal bearing strength and f_u is the ultimate tensile stress, d is the nominal diameter of bolt and t is the summation of thickness of connected plates experiencing bearing stress in same direction. So this is what we have discussed earlier also, now also the same formula we are using.

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$$k_b = \text{smaller of } \frac{e}{3d_0}, \frac{p}{3d_0} - 0.25, f_{ub}/f_u, 1$$

f_{ub} = ultimate tensile stress of bolts

d_0 = diameter of bolt hole

p = pitch of fastener along bearing direction

e = edge distance

$$V_{dsb} = V_{npb} / \gamma_{mb}$$

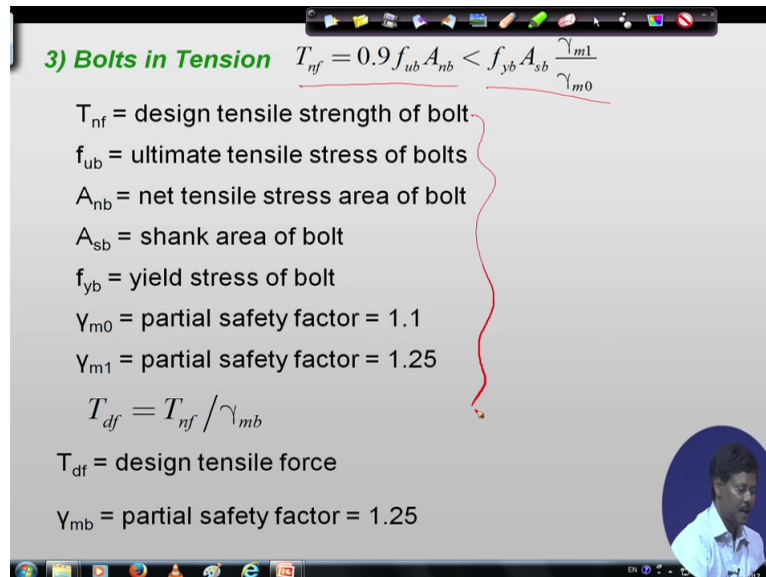
V_{dsb} = design shear force

γ_{mb} = partial safety factor = 1.25

Similarly, we can calculate the strength due to bearing using this value of K_b and this K_b value will be based on this, e by $3d_0$, P by $3d_0$ minus 0.25 , f_{ub} by f_u and 1 . So this we know.

What are the parameters we have consider in case of bearing type of bolt same thing and the design shear force we can calculate V_{dsb} as V_{npb} by γ_{mb} where γ_{mb} will be 1.25.

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3) Bolts in Tension $T_{nf} = 0.9 f_{ub} A_{nb} < f_{yb} A_{sb} \frac{\gamma_{m1}}{\gamma_{m0}}$

T_{nf} = design tensile strength of bolt
 f_{ub} = ultimate tensile stress of bolts
 A_{nb} = net tensile stress area of bolt
 A_{sb} = shank area of bolt
 f_{yb} = yield stress of bolt
 γ_{m0} = partial safety factor = 1.1
 γ_{m1} = partial safety factor = 1.25
 $T_{df} = T_{nf} / \gamma_{mb}$
 T_{df} = design tensile force
 γ_{mb} = partial safety factor = 1.25

Now bolt in tension, in case of bolt in exerted in tension we can calculate the tensile strength of the bolt as T_{nf} as this $0.9 f_{ub}$ into A_{nb} and it has to be less than $f_{yb} A_{sb}$ into γ_{m1} by γ_{m0} , where the parameters defined here has the usual meaning as defined in case of bearing type of bolt that means f_{ub} is the ultimate tensile stress of bolt, A_{nb} is the net tensile stress area of bolt, A_{sb} is the shank area of bolt like this here and the design tensile force will become T_{nf} by γ_{mb} . So similar way whatever we have done here same process we are following in case of HSFG bolt except the shear strength calculation.

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4) Bolts with combined Shear and Tension

$$\left(\frac{V_{sf}}{V_{sdf}}\right)^2 + \left(\frac{T_f}{T_{sdf}}\right)^2 \leq 1.0$$

V_{sf} = applied shear force at service load

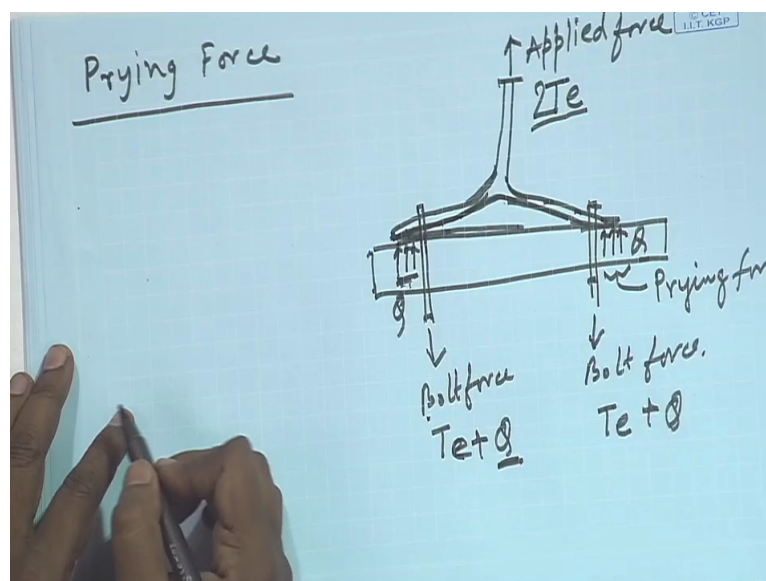
V_{sdf} = design shear capacity

T_f = externally applied tension at service load

T_{sdf} = design tension capacity

And if bolts are in combine shear and tension then also we have to check the interaction formula that means V_{sf} by V_{sdf} whole square plus T_f by T_{sdf} whole square should be less than 1 where V_{sf} is applied shear force at the service load and V_{sdf} is the design shear capacity. Similarly T_f is the externally applied tension at service load and T_{sdf} is the design tension capacity. So this is what we have. So in this way we can calculate the strength of bolt, bolt means HSFG bolt due to shear due to bearing due tension and due to combined (9:25) effect. So what new things we have learned here that the shear capacity shear strength capacity of the bolt due to friction due to bearing and tension are same.

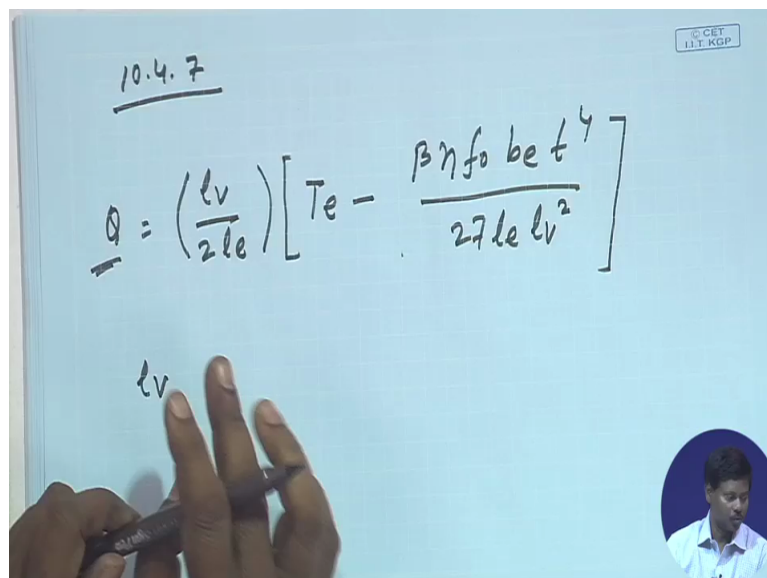
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Now for high strength friction grip bolt, I will high strength friction grip bolt another force comes which is call prying force. This is very important prying force. Prying force is the additional force coming into picture in case of tension and if the deformation is allowed between two plates then prying force develops. Let me show through some figure, say one plate is like this which is attached with a say suppose I section, right.

So I section, right and bolting connection has been done here, right. So what we can see here that it has a applied force and in this direction said bolt force is coming, right. So if this is T_e of $2T_e$ then this is T_e and this is T_e , this equation is true if this is not detached (11:43) that means this deformation if it is not allowed. If deformation is allowed then what will happen that some forces at this portion will be developed, some additional forces will be developed. This additional force is called prying force that means due to application of load of $2T_e$, the bolt is getting force as T_e but if we allow the deformation of the flange then additional prying force will come into picture. So if this is Q then this will be T_e plus Q if this force is called Q . So Q is the prying force we can say and this prying force will be developed here and to withstand that force bolt will face extra force of amount Q and this Q value has been calculated and reported in clause 10.4.7.

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10.4.7

$$Q = \left(\frac{l_v}{2l_e} \right) \left[T_e - \frac{\beta \gamma f_o b_e t^4}{27 l_e l_v^2} \right]$$

l_v

4) Bolts with combined Shear and Tension

$$\left(\frac{V_{sf}}{V_{sdf}}\right)^2 + \left(\frac{T_f}{T_{sdf}}\right)^2 \leq 1.0$$

V_{sf} = applied shear force at service load

V_{sdf} = design shear capacity

T_f = externally applied tension at service load

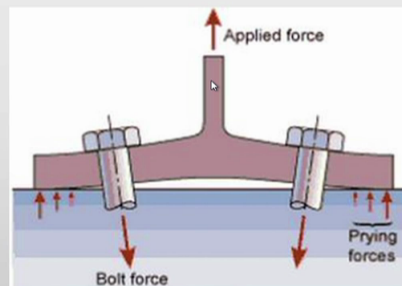
T_{sdf} = design tension capacity



5) Prying forces

(Clause 10.4.7)

$$Q = \left(\frac{l_v}{2l_e}\right) \left[T_e - \frac{\beta \eta f_0 b_e t^4}{27 l_e l_v^2} \right]$$



Q = additional force of fastener due to prying action

l_v = distance from bolt centre line to toe of fillet weld or to half the root radius of a rolled section

l_e = distance between prying force and bolt centre line



In clause 10.4.7 you will get the details of the prying force where it is mention that Q is equal to l_v by $2l_e$ into T_e minus β into η of $b_e t$ to the power 4 by $27 l_e l_v$ square, right. So this Q is the additional force (13:44) due to prying action and l_v is the distance from bolt to center line to toe of fillet weld or to half the root radius of a rolled section that I will show in the figure.

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$$l_e = 1.1t\sqrt{\beta f_0 / f_y}$$

$\beta = 2$ for non pre-tensioned bolts
 $\beta = 1$ for pre-tensioned bolts

$\eta = 1.5$

$f_0 = \text{proof stress}$

$t = \text{thickness of end plate}$

$$t_{\min} = \sqrt{4.4M_p / (f_y b_e)}$$

where $M_p = T_e l_v / 2 = Q l_e$

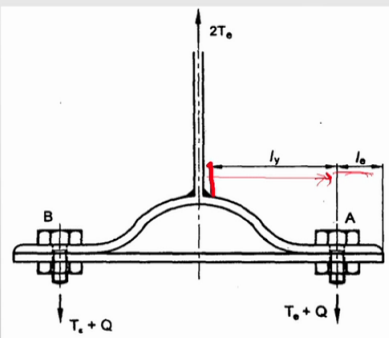


FIG. 16 COMBINED PRYING FORCE AND TENSION

IS 800: 2007

This is l_v , so this is called l_v , the distance from bolt center line to toe of fillet weld or to half the root radius of the rolled section, this is l_v and l_e is the distance between prying force and bolt center line l_e . Distance of the prying force to bolt center line this is l_e , right, this figure I have taken from IS 800-2007. This figure number is 16, you can find out details there. Then l_e we can calculate from this formula where l_e is equal to 1.1 into t into βf_0 by f_y and this β value will become 2 for non-pretension bolt and we can consider 1 for pre-tension bolt. So β value varies according to pre-tension or non-pretension bolt and η is taken as 1.5 f_0 we know already that is proof stress and t can be calculated t is the basically thickness of the end plate and minimum t has to be this where M_p will be $T_e l_v$ by 2 or is equal to Q into l_e , right. So this is how one can find out the prying force Q .

So in case of HSFG bolt apart from shearing bearing and tension prying force may also come into picture. It depends on case to case that means if the plates are detached (())(16:21) means allow deformation then prying action will happen and because of that additional force, which is called prying force has to be calculated. So the bolt we supposed to take the value, it will have some additional value of Q that means if T_e is the external force on bolt then actual force will be T_e plus Q where Q is a prying force. Now we will go through one example of HSFG bolt and we will see how to calculate the shear strength and other strength means bearing strength etcetera for HSFG bolt then it will be clear to us.

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$$l_e = 1.1t\sqrt{\beta f_0 / f_y}$$

$\beta = 2$ for non pre-tensioned bolts
1 for pre-tensioned bolts

$\eta = 1.5$

$f_0 = \text{proof stress}$

$t = \text{thickness of end plate}$

$$t_{\min} = \sqrt{4.4M_p / (f_y b_e)}$$

where $M_p = T_e l_v / 2 = Q l_e$

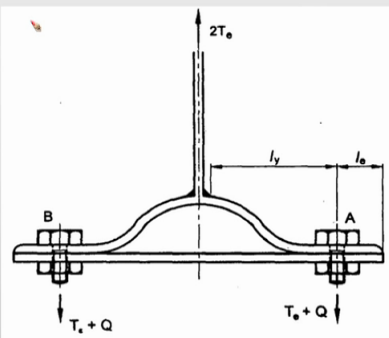


FIG. 16 COMBINED PRYING FORCE AND TENSION

Example:
 An ISA 110 mm × 110 mm × 10 mm carries a factored tensile force of 150 kN. It is to be jointed with a 10 mm thick gusset plate. Design the joint using HSFG bolt when (a) no slip is permitted, (b) when slip is permitted. Assume steel is Fe 410 grade.

Solution:
 Let us provide HSFG bolts of grade 8.8 and of diameter 20 mm.

For 8.8 grade bolts: $f_{ub} = 800 \text{ MPa}$

Net tensile stress area of bolt, $A_{nb} = 0.78 \times \frac{\pi}{4} \times 20^2 = 245 \text{ mm}^2$

For Fe 410 grade of steel: $f_u = 410 \text{ MPa}$

So we will go through one example, this is example that and ISA 110 by 110 by 10 mm means indian standard angle carries a factor tensile force of 150 kilo newton. It is to be jointed with a 10 mm thick gusset plate. Design the joint using HSFG bolt when no slip is permitted and when slip is permitted. No slip is permitted means friction is there and slip is permitted means friction is not there. So it will be like bearing type. So here we can see we can use HSFG bolt of grade, let us provide HSFG bolt of grade 8.8 and let use 20 mm diameter, because nothing has been told, so we can assume suitable data. So as grade is 8.8 bolt means bolt grade is 8.8. So f_{ub} value will become 800 MPa, right.

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Handwritten calculations on a blue background:

$$f_{ub} = 800 \text{ MPa}$$
$$A_{nb} = 0.78 \times \frac{\pi}{4} \times 20^2 = 245 \text{ mm}^2$$

Fe 410 grade steel, $f_u = 410 \text{ MPa}$

slip critical Condition

$$F_o = A_{nb} \times 0.7 f_{ub} = 245 \times 0.7 \times 800 \times 10^{-3}$$
$$= 137.2 \text{ kN}$$
$$V_{dsf} = \frac{\mu_f n_e K_h F_o}{\gamma_{mf}}$$

Assumptions: $\mu_f = 0.5$, $n_e = 1$, $\gamma_{mf} = 1.25$, $K_h = 1$

So f_{ub} value we can find out for 8.8 grade bolt as 800 MPa and now A_{nb} which we have to calculate the net tensile stress area of bolt that will be 0.78 times π by 4 into d square. So this is becoming 245 millimeter square and the gusset plate will be as usual Fe 410 type steel Fe 410 grade steel if we use then the tensile stress of the gusset plate will be 410 MPa, right. So these are the data we have assumed.

Now let us come to the first case that slip critical condition, so that means slip is not permitted slip critical condition means slip is not permitted. So here through put F_0 will become A_{nb} into 0.7 f_{ub} , right so we can calculate the value A_{nb} is 245 and 0.7 into f_{ub} is 800 and to make it kilo newton we multiply 10 to the minus 3. So this is becoming 137.2 kilo newton. Now slip resistance of bolt we can find out that is the v_{dsb} we can make V_{dsf} right, μ_f into n_e into $K_h F_0$ by γ_{mf} , right.

Now we have to assume certain data if it is not given like μ_f is not given it. So μ_f the coefficient of friction we can assume as 0.5 we can assume. Now n_e is a number of effective interface offering frictional resistance to slip that we can considered as 1 in number of effective interface offering friction resistance that we can assume 1. Now γ_{mf} at ultimate load is 1.25, we can consider γ_{mf} as 1.25 we can consider and K_h value we have seen either 1 or 0.7. So here we are considering 1 assuming bolts (21:51) in clearance (21:50), K_h value we are considering 1.

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$$\text{slip resistance of bolt} = 0.5 \times 1 \times 1 \times \frac{137.2}{1.25}$$

$$= 54.88 \text{ kN}$$

$$n = \frac{150}{54.87} = 2.73 \approx \underline{3}$$

So the slip resistance of bolt we can calculate, slip resistance bolt we can calculate as 0.5 into 1 into 137.2 by 1.25. This is come in 54.88 kilo newton. So number of bolt required will be the load divided by force means design strength, which is coming 54.87 is equal to 2.73 or 3. So we can provide 3 numbers of HSFG bolt for this type of problem, if we use HSFG bolt to withstand 150 kilo newton load then we can provide 3 numbers of bolt.

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Handwritten calculation on a blue grid background:

slip is permitted
Bearing type connection

$$V_{dsb} = \frac{A_n b f_{ub}}{\sqrt{3} \gamma_{mb}} = \frac{245 \times 800}{\sqrt{3} \times 1.25} \times 10^{-3} = \boxed{90.53 \text{ kN}}$$

$$V_{dpb} = \frac{2.5 K_b d t f_u}{\gamma_{mb}} = \frac{2.5 \times 0.61 \times 20 \times 10 \times 410}{1.25} \times 10^{-3}$$

$e = 40$
 $p = 60$ } $K_b = 0.61$

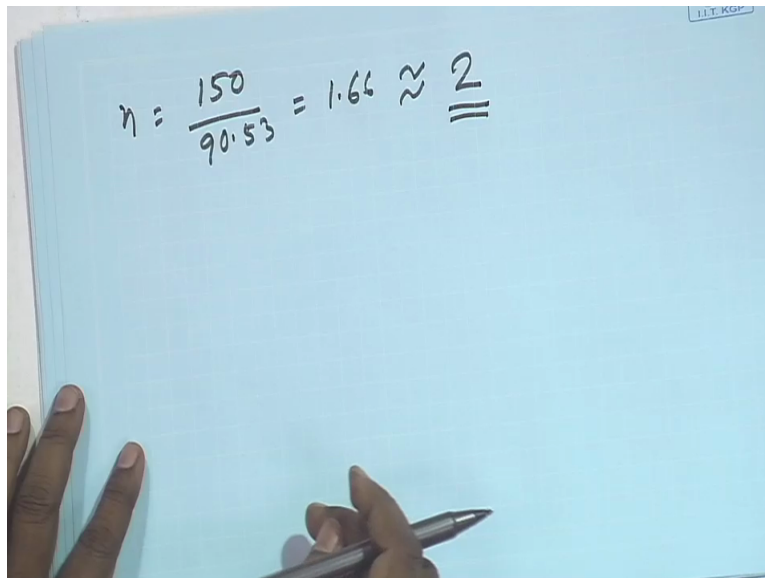
$V_{dpb} = \boxed{100 \text{ kN}}$
 $\boxed{B_v = 90.53}$

Now we will go for next one. This one is the slip critical condition where slip is not permitted. Now if we permit the slip then if slip is permitted then this will become bearing type connection. So this will become bearing type connection if slip is permitted that means frictional resistance is not there. So there we can find out V_{dsb} as $A_n b f_{ub}$ by root 3 gamma

mb, so this is 245 into f_{ub} means 800 by root 3 into the safety factor is 1.25 and we have multiply 10 to the minus 3 to make it in kilo newton. So this is becoming 90.53 kilo newton.

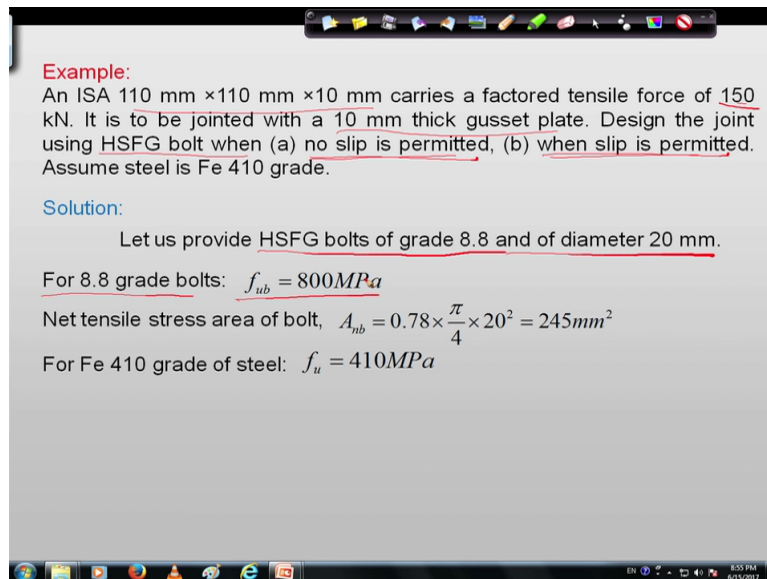
Now strength in bearing also we have to find out, V_{dpb} this will be $2.5 K_b d t f_u$ by gamma mb. Now we have to find out the value of K_b . Now K_b when we are going to find we have to find out means we have to assume some value of edge distance. So let us assume 40 mm and p is let us assumed 60 mm then the K_b value we can find out from this K_b as 0.61. So V_{dpb} we can find out 2.5 into 0.61 into 20 into t is 10 into 410 by 1.25 into 10 to the minus 3. So this is becoming 100 kilo newton that means the strength of the bolt will become 90.53, because due to shear this is coming this value and due to bearing it is coming 100. So the bolt value will become 90.53.

(Refer Slide Time: 26:12)


$$n = \frac{150}{90.53} = 1.66 \approx \underline{\underline{2}}$$

So number of bolt we can find out number of bolt will be 150 divided by 90.53 that means 1.66 that is 2, right. So 2 numbers of bolts require.

(Refer Slide Time: 26:33)



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For Fe 410 grade of steel: $f_u = 410 \text{ MPa}$

So here we have seen that if we consider slip critical and if we do not consider slip, if we allow slip and if we do not allow slip how the values are going to change. So design of high strength friction grip bolt is exactly similar to bearing type of bolt except the friction. Friction component if we had that means slip critical if we consider if we consider the slip resistance then we have to find out, the design shear strength accordingly, other things will be exactly similar to the bearing type, I hope this work out example will be helpful for understanding the design methodology of the ordinary black (())(27:29) bolt as well as the high strength friction grip bolt, thank you very much.