

**Course on Design of Steel Structures**  
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**Mod 10 Lecture 48**  
**Design of Laterally Supported Beams**

Now, I am going to discuss the design procedure of a laterally supported beam. So in last lecture we have discussed how to calculate the design bending strength of a laterally supported beam. How to calculate the design shear of a laterally supported beam. So in today's lecture first, I will go through very quickly the design steps for laterally supported beam and then I will go through one workout example.

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**DESIGN STEPS FOR  
LATERALLY SUPPORTED BEAMS**

- 1) The loads acting on the beam are calculated by multiplying the appropriate partial load factors.
- 2) The distribution of B.M. & S.F. along the length of the beam is determined. The maximum B.M. & S.F. is calculated
- 3) A trial plastic section for the beam is worked out from the following equation:  
$$Z_p = \frac{M_d}{f_y / \gamma_{m0}}$$
- 4) A suitable section is selected which has plastic section modulus greater than the calculated value. ISMB, ISLB, ISWB sections are in general preferred.

So in first step, what we can do we can find out what are the load acting on the beam and then we can calculate the appropriate load by multiplying the partial load factor. So whether it is due to dead load, live load or due to wind (( ))(1:07) load or seismic load depending on that we can find out what is the load coming and what will be the partial load factor. So from that we can find out the bending moment distribution and shear force along the beam length and then from that we can find out maximum bending moment at shear force, because I have to design the beam against the maximum bending moment and maximum shear force.

So on that basis then I can find out a trial plastic section  $Z_p$ ,  $Z_p$  as  $M_d$  by  $F_y$  by  $\gamma_{m0}$ , considering whether it is high shear or low shear irrespective of that, first I will find out a trial plastic section from this formula that is  $Z_p$  is equal to  $M_d$  by  $F_y$  by  $\gamma_{m0}$ , where

$M_d$  is the maximum bending moment, which we have calculated,  $F_y$  is the yield stress of the material and  $\gamma_{m0}$  is a material safety factor, right. So from this we can find out the trial plastic section and then we can select a suitable section, you so from the plastic section modulus we can find out a corresponding section whether we are going for ISMB or ISLB depending on that we can find out section whose plastic section modulus is greater than this slightly greater than this.

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5) The section is classified as plastic, compact or semi compact depending upon the specified limits of  $b/t_f$  and  $d/t_w$  as specified in **Table 2, IS 800: 2007**.

6) Calculate the design shear strength ( $V_d$ ) from the relation:

$$V_d = \frac{f_y}{\sqrt{3}\gamma_{m0}} h t_w$$

7) The beam is checked for high/low shear. If  $V < 0.6 V_d$ , the beam will be low shear and if  $V > 0.6 V_d$ , the beam will be high shear.

8) The trial section is checked for design bending strength

**For low shear:**

$$M_d = \beta_b Z_p f_y / \gamma_{m0}$$

$$\leq 1.2 Z_e f_y / \gamma_{m0} \text{ (for simply supported beams)}$$

$$\leq 1.5 Z_e f_y / \gamma_{m0} \text{ (for cantilever beams)}$$

Then we can classify the section as plastic compact or semi-compact from table two, right compact, semi-compact or plastic. So depending on the  $b$  by  $t_f$  ratio and  $d$  by  $t_w$  ratio, I can find out from table 2 that whether the assumed section is plastic or compact or semi-compact. Once it is done then I can find out the design shear strength  $V_d$ ,  $V_d$  will be calculated as  $F_y$  by root 3 gamma  $m_0$  into  $h$  into  $t_w$ ,  $h$  we can write or  $D$  (3:27) write  $D$  where  $h$  or is a overall depth, right. So design shear strength  $V_d$  can be calculated from this and then I will check whether, it is high shear or low shear. If the  $v$  is less than  $0.6V_d$  then the beam is will be low shear and if it is more than  $0.6V_d$  then beam will be consider as high shear.

So considering high shear or low shear, the formula will be different for calculating design bending strength. So if it is low shear then I can find out  $M_d$  value as this that is  $\beta_b$  into  $Z_p F_y$  by gamma  $m_0$ , right. So now for a particular section we know  $Z_p$  value and depending on the type of section whether it is plastic, semi-plastic sorry, plastic, compact or semi-compact, I can find out the  $\beta_b$  value and gamma  $m_0$  value I know. So from that I can find out the design bending strength  $M_d$  and it has to be less than  $1.2Z_e F_y$  by gamma  $m_0$  and for

simply supported beam and for cantilever, it has to be less than  $1.5Z_e F_y$  by  $\gamma_{m0}$ . So in this step, we could calculate the design bending strength for low shear.

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**For high shear:**

$$M_{dv} = M_d - \beta(M_d - M_{fd}) \leq 1.2 \frac{Z_e f_y}{\gamma_{m0}}$$

(for plastic and compact section)

$$M_{dv} = Z_e \frac{f_y}{\gamma_{m0}}$$

(For semi-compact section)

8) If  $M > M_d$ , increase the section size and repeat from step 5.

9) The design shear strength ( $V_d$ ) should be greater than the maximum factored shear force developed due to external load. If  $V > V_d$ , redesign the section by increasing the section size.

10) The beam is checked for deflection as per **Table 6, IS 800: 2007**.

And for high shear, the design bending strength will be calculate from this formula where  $M_{dv}$  is equal to  $M_d$  minus  $\beta$  into  $M_d$  minus  $M_{fd}$  and it has to be less than or equal to  $1.2 Z_e F_y$  by  $\gamma_{m0}$  and this is for plastic and compact section and for semi-compact section  $M_{dv}$  can be calculated as  $Z_e F_y$  by  $\gamma_{m0}$ . So  $M_{dv}$  value for high shear can be calculated depending on the type of section from this formula and now, if we see that  $M$  is greater than  $M_d$  then we have to increase the section size and repeat from step 5, right. That means if the design bending strength is less than the actual moment then I have to increase the section size and then I have to repeat from step 5 otherwise if it is satisfying then we can go to step 9.

So from step 5 to 8 will be repeated till the design bending strength is more then the bending moment developed bending moment. So if it is more then fine otherwise I have to repeat and once it satisfy then I will calculate the means I will calculate design shear strength  $V_d$  and it should be greater than the maximum factor shear developed due to external load. So maximum factor shear  $V$  has been already calculated. So and that design shear strength should be greater than this, if this is **is** fine then we can go to next step otherwise if  $V$  is greater than then  $V_d$  then redesign the section by increasing the section size.

So if we see that the shear is more than the design shear strength then we have to redesign by increasing the section size and next step we have to check for deflection, which is also

another important aspects that is the beam has to be checked for deflection as per table 6. In table 6, the limiting deflection criteria has been given and the maximum deflection can be calculated from the given boundary conditions and the loading conditions. So from that we can find out the maximum deflection and maximum deflection has to be less than the limiting deflection given in table 6, if it is okay then fine otherwise we have to again increase the section.

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11) The beam is checked for web buckling:  
 If,  $\frac{d}{t_w} \leq 67\epsilon$  (for web without stiffeners) the web is assumed to be safe in web buckling and the shear strength of the web is governed by plastic shear resistance.  
 The web should be checked for buckling in case of high shear even if this limit is satisfied. The web buckling strength of the section,  $f_{wb} = A_b \times f_{cd}$   
 Here,  $A_b$  = area of the web at the neutral axis of the beam =  $Bt_w$  and  $f_{cd}$  = design compressive stress  
 The web buckling strength should be greater than the design shear force.

12) The beam is checked for web crippling,  

$$F_{wc} = \frac{F_{wc} > V}{\gamma_{m0}} = \frac{b_1 t_w f_{yw}}{\gamma_{m0}}$$

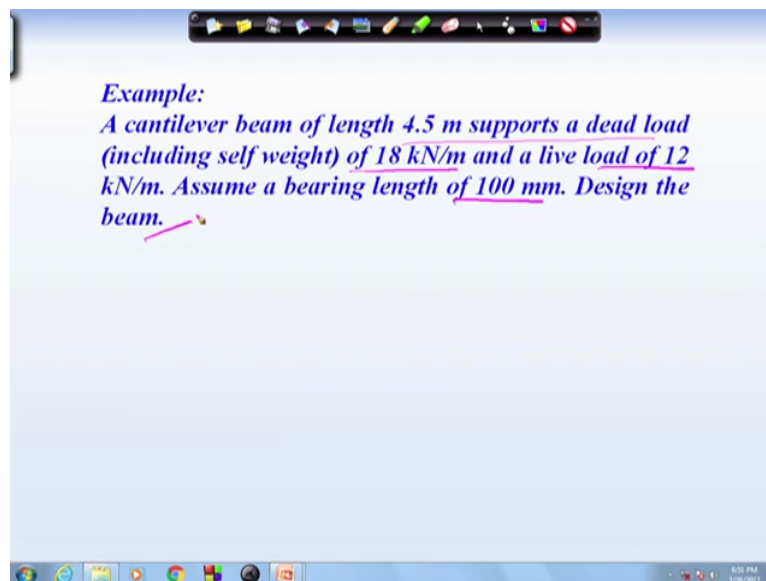
Now we have to check for web buckling. The beam needs to be checked for web buckling that is if we see the  $d$  by  $t_w$ , this is less than 67 epsilon for without for web without stiffness then the web is assumed to be safe in web buckling and we do not need to check. So the shear strength of the web is governed by plastic share resistance and the web should be checked for buckling, in case of high shear even if this limit is satisfied and the web buckling strength can be calculated we know that is  $F_{wd}$  is equal to  $A_v$  into  $F_{cd}$ , where  $A_v$  is the area of the web at the neutral axis that is  $B$  into  $t_w$  and  $F_{cd}$  is the design compressive stress, which can be found from table 9C, because it is buckling class C.

So from this I can find out the web buckling strength and this web buckling strength should be greater than the design shear force at that point, right. So if web buckling strength is greater than the design shear strength then fine otherwise we have to increase the bearing plate (( ))(9:05) length, either we have to increase the bearing plate length or we have to increase the section size to increase the web buckling strength. So once it is okay then we will go for checking web crippling. Web crippling that  $F_{wc}$  web crippling strength, this has to be also less than the shear force at that point and  $F_{wc}$  is a web crippling strength we can

calculate from this formula that is  $b_1 t_w F_{yw}$  by  $\gamma_{m0}$ , where  $b_1$  is the dispersion width and  $t_w$  is the thickness of the web and  $F_{yw}$  is the yield strength of the web, right.

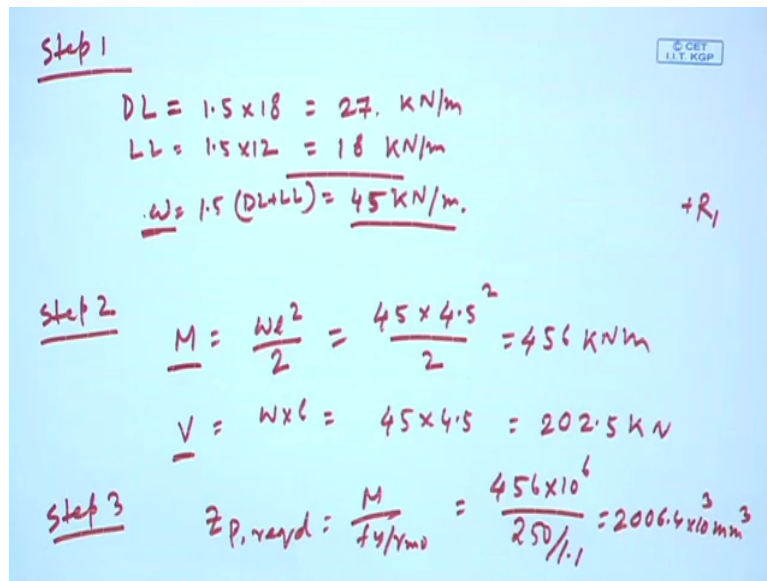
So web crippling strength  $F_{wc}$  can be found from this formula and it has to be more than the shear force coming on that section. So if it is satisfying this criteria then fine otherwise again we have to increase the section size or we have to increase the bearing length to make the section safe from the web crippling. So these are the steps which we need to follow to design a beam with laterally supported.

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So now we can go through this example where a cantilever beam of length 4.5 meter supports a dead load which include self-weight that is 18 kilo newton per meter and a live load of 12 kilo newton per meter. Assuming a bearing length of 100 millimeter. Design the beam. So a cantilever beam you have to design where the length of the member is 4.5 and load is given 18 kilo newton and 12 kilo newton as dead load and live load and we have to assume the bearing length has 100 mm and applying we have to design the beam.

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Step 1

$$DL = 1.5 \times 18 = 27 \text{ kN/m}$$
$$LL = 1.5 \times 12 = 18 \text{ kN/m}$$
$$w = 1.5 (DL + LL) = 45 \text{ kN/m}$$

+R<sub>1</sub>

Step 2

$$M = \frac{wl^2}{2} = \frac{45 \times 4.5^2}{2} = 456 \text{ kNm}$$
$$V = w \times l = 45 \times 4.5 = 202.5 \text{ kN}$$

Step 3

$$Z_{p, reqd} = \frac{M}{f_y / \gamma_{m0}} = \frac{456 \times 10^6}{250 / 1.1} = 2006.4 \times 10^3 \text{ mm}^3$$

So let us come to step one in which we have told we have to find out the dead load with load factor, so dead load 1.5 into 18 and live load also 1.5 into 12 which was given. So this will be 27 kilo newton per meter and this will be 18 kilo newton per meter. So total load dead load plus live load into 1.5 times  $w$  is equal to this is equal to say 45 kilo newton per meter. So in step one, we have calculated the factor load  $w$  which is coming 45 kilo newton per meter.

Now we will go to step two. In step two, we can find out the maximum bending moment as  $wl^2$  square by 2 for cantilever beam. Maximum bending moment will be at the support, so  $w$  is 45  $l$  is 4.5 by 2. So that is becoming 456 kilo newton. Similarly the maximum shear force will be at the support that will be  $w$  into  $l$ , so 45 into 4.5 that will be 202.5 kilo newton. So in step two the maximum bending moment and maximum shear force we have calculated. Now we have to choose a trial section. So in step three, we can choose a trial section for that we can find out  $Z_p$  require that will be  $M$  by  $F_y$  by  $\gamma_{m0}$ ,  $M$  is 456 kilo newton meter, so 456 into 10 to the 6 newton millimeter by 250 by 1.1. So this is becoming 2006.4 into 10 cube millimeter cube, okay.

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Handwritten calculations for ISLB 550 section properties:

$$\text{ISLB 550 @ } 0.846 \text{ kN/m}$$
$$\left\{ \begin{array}{l} Z_{pz} = 2228.16 \times 10^3 \text{ mm}^3 \\ Z_{ez} = 1933.2 \times 10^3 \text{ mm}^3 \end{array} \right\} \text{SP 6(i)}$$
$$\left\{ \begin{array}{l} D = 550 \text{ mm} \\ b_f = 190 \text{ mm} \\ t_w = 9.9 \text{ mm} \\ t_f = 15 \text{ mm} \\ R = 18 \text{ mm} \end{array} \right.$$
$$\begin{aligned} d &= 550 - 2(t_f + R) \\ &= 550 - 2(15 + 18) \\ &= 484 \text{ mm} \end{aligned}$$
$$I_{zz} = 53161.6 \times 10^4 \text{ mm}^4$$

So in step three we could find out the  $Z_p$  require and now we can find out a suitable section from sp6 and if we look into sp6 (i)(14:09) we can say that ISLB 550 at 0.846 kilo newton per meter, we can choose this section with  $Z_{pz}$  is equal to 2228.16 into 10 to the 3 millimeter cube. We need 2006 and we are providing 2228, right and  $Z_{ez}$  here will be 1933.2 into 10 to the 3. So these values are obtained from sp6 (i)(14:52), right and also we could find out  $D$  value as 550 the overall depth, width of flange 190 mm, thickness of the web 9.9 mm, thickness of flange 15 mm and root radius  $R$  will occupy an 18 mm, right.

So from this also I can find out the  $d$  value small  $d$  that will be 550 minus 2 into  $t_f$  plus  $R$  is equal to 550 minus 2 into  $t_f$  is 15 plus  $R$  18, so this is becoming 484 millimeter and also,  $I_{zz}$  value we can find from the sp6 (i)(14:52) that is 53161.6 into 10 to the 4 millimeter to the 4. So these are the data which will be require to calculate the design bending strength and other details, right.



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Handwritten calculations on a blue background:

$$\frac{b_f}{t_f} = \frac{15}{2.3} = 6.33 < 9.4$$
$$\frac{d}{t_w} = \frac{484}{9.9} = 48.9 < 84$$

Hence, plastic

Step 4

$$V_d = \frac{f_y}{\sqrt{3} \gamma_{m0}} \times D \cdot t_w = \frac{250}{\sqrt{3} \times 1.1} \times 550 \times 9.9$$
$$= 714.47 \text{ kN}$$
$$0.6 V_d = 0.6 \times 714.47 = 428.68 \text{ kN} > V = 202.5$$

Low shear

Also, I have to find out what is the section here, whether it is plastic, compact or semi-compact. So now here we know  $b_f$  by here  $b_f$  by 2 by  $t_f$  is equal to it will be  $b_f$  we found as 190 by 2 by 15 is the thickness of flange. So this is becoming 6.33 and this is less than 9.4. Similarly,  $d$  by  $t_w$  we are getting 484 by 9.9 will be 48.9 less than 84 okay. So from this I can say this is a plastic section. So if it is plastic section then I can find out the value accordingly right.

So now in step 4 also we have to find out that what is the shear capacity  $V_d$ , because before calculating the bending moment bending strength of the section, we need to know whether it is high shear or low shear, so for that we have to find out what is the design shear strength of the section and also we have to know whether it is plastic, compact or semi-compact according to that the beta  $b$  value will be calculated, right.

So here  $V_d$  will be  $F_y$  by root 3 gamma  $m_0$  into  $D$  into  $t_w$ , so I can put value that is  $D$  is 550 into  $t_w$  is 9.9, so these values are 714.47 kilo newton, right and  $0.6V_d$  is coming as 0.6 into 714.47 this is 428.68 kilo newton, which is more than the  $V$  and  $V$  value is coming 202.5 kilo newton, right. So what will be see that here the design shear strength is becoming 714 kilo newton and  $0.6V_d$  is coming 428 which is more than  $V$ , so it is low shear. So we can see this is as low shear, because shear force is coming less compared to the design shear capacity, okay.



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step 5

$$M_d = \frac{Z_p f_y}{\gamma_{m0}} = \frac{2228.16 \times 10^3 \times 250}{1.1}$$

$$= 506.4 \text{ kNm} > M \quad \text{OK}$$

$$\leq \frac{1.5 \times Z_e f_y}{\gamma_{m0}} = \frac{1.5 \times 1933.2 \times 10^3 \times 250}{1.1}$$

$$= 659 \text{ kNm}$$

step-6

$$\delta = \frac{w l^4}{8 E I} = \frac{27 \times 4500^4}{8 \times 2 \times 10^5 \times 53161.6 \times 10^4} \approx 14 \text{ mm}$$

So if it is low shear then I can find out the design bending moment according to design bending strength accordingly design bending strength accordingly? So in step 5, I can find out design capacity of the section design moment capacity of the section, it is  $M_d$ . So  $M_d$  will be simply  $Z_p F_y$  by  $\gamma_{m0}$ , so  $Z_p$  value we find earlier which is this  $F_y$  is 250 and  $\gamma_{m0}$  is 1.1. So this is 506.4 kilo newton and it has to be less than as it is a cantilever beam. So it has to be less than and equal to  $1.5 Z_e$  into  $F_y$  by  $\gamma_{m0}$  that is  $1.5 Z_e$  is 1933.2, 10 cube into 250 by 1.1, right that is 659 kilo newton. So  $M_d$  is this,  $M_d$  is less than this, so we can consider the design bending strength of the section as 506.4 kilo newton, right.

So next we will find out and this is more than the moment coming maximum moment coming, right so okay. Now in step 6, we can check for deflection. So check for deflection  $\delta$  we know that will be  $w l^4$  to the 4 by  $8 E I$ . This will be the maximum deflection for cantilever beam. So if I calculate  $w$  will be 27, because this will be under service load. So it will be 27 into 45 sorry,  $w$  will be 18 plus 12, this will be 30, 30 into 4500 whole to the 4 by 8,  $E$  is 2 into 10 to the 5 into  $I$  we found earlier that is 53161.6 into 10 to the 4, right. So this is coming around 14 mm right.

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Handwritten calculations on a blue background:

Allowable deflection  $\frac{L}{150} = \frac{4500}{150} = 30 > \delta$   
OK

Step 7  
Web buckling

$A_b = (b_1 + n_1) t_w$   
 $= (100 + 275) \times 9.9$   
 $= 3712.5 \text{ mm}^2$

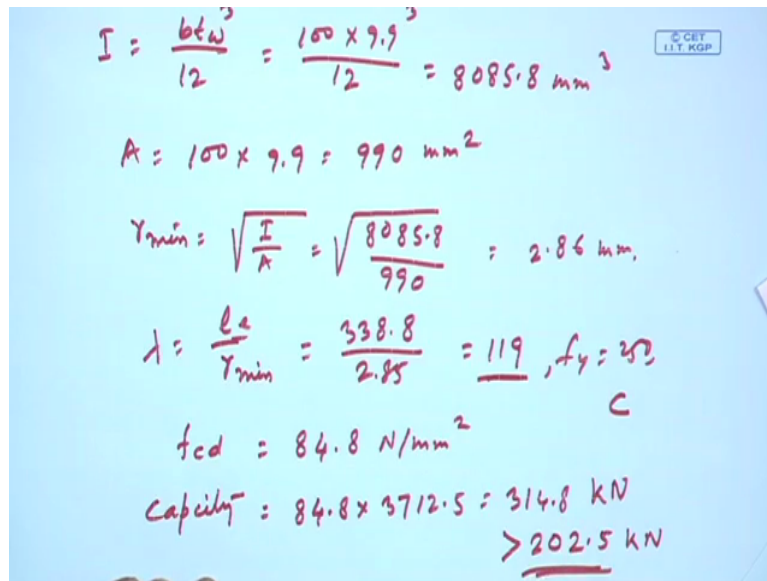
$b_1 = 100 \text{ mm}$   
 $n_1 = \frac{D}{2} = \frac{550}{2} = 275$

$l_e = 0.7 d = 0.7 \times 484 = 338.8 \text{ mm}$

An allowable deflection, we can find out allowable deflection, allowable deflection will be  $L$  by 150 that is 4500 by 150 that is 30 mm, so which is more than the actual deflection, so it is okay. So from deflection point of view also the section which is assumed is find.

Now we will go to next step that is step 7, which will be dealt with web buckling. So we have to see whether it is buckling means, its web is buckling or not. So the cross sectional area of the web for buckling that is  $A_b$  we can find out that is  $b_1$  plus  $n_1$  into  $t_w$ , because of the support only, it will be and  $b_1$  is, actually  $b_1$  is 100 okay,  $b_1$  is given as bearing length that is 100 mm and  $n_1$  will be  $D$  by 2 that will be 550 by 2, 275 so  $A_b$  we can find out that is 100 plus 275 into  $t_w$  as 9.9. So cross sectional area of web for buckling will become 3712.5 millimeter square and effective length of web  $l_e$  is equal to  $0.7d$  that will be  $0.7$  into  $d$  we calculate earlier 484. So this is becoming 338.8 millimeter. Now I can find out moment of inertia  $I$ .

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Handwritten calculations on a blue background:

$$I = \frac{bt^3}{12} = \frac{100 \times 9.9^3}{12} = 8085.8 \text{ mm}^3$$
$$A = 100 \times 9.9 = 990 \text{ mm}^2$$
$$r_{\min} = \sqrt{\frac{I}{A}} = \sqrt{\frac{8085.8}{990}} = 2.86 \text{ mm}$$
$$\lambda = \frac{l_e}{r_{\min}} = \frac{338.8}{2.85} = 119, f_y = 250$$

C

$$f_{cd} = 84.8 \text{ N/mm}^2$$
$$\text{Capacity} = 84.8 \times 3712.5 = 314.8 \text{ kN}$$

> 202.5 kN

So I will be, bt cube by 12, so b has been considered as 100, t is 9.9 by 12. So this is becoming 8086.8 millimeter to the cube and area we can find out this is as 100 into 9.9, so 990, right. So r minimum I can find out, r minimum will be I by A, so I is 8085.8 cross section area is 990, this is becoming 2.86 and lambda I can find out, l by r we locate it by r minimum, l effective we found 338.8 r minimum is 2.85, so this is becoming 119.

So allowable stress Fcd for lambda 119, Fy as 250 and buckling class as C. so from table 9C, I can find out the value of Fcd. So Fcd value is coming as 84.8 newton per millimeter square. So this is obtained from table 9C. So capacity of the section, I can find out, capacity buckling strength I can find out as Fcd value 84.8 into area 3712.5. So this is becoming 314.8 kilo newton and that is greater than the maximum shear which was coming to 202.5 kilo newton, right. So it is okay.

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The image shows a handwritten calculation on a light blue background. At the top left, 'step 8' is written in red and underlined. Below it, 'web crippling' is also written in red and underlined. The main calculation is for  $F_w$ , which is set equal to  $(b_1 + n_2) \times t_w \times f_y / \gamma_{m0}$ . The values are substituted:  $b_1 = 100$ ,  $n_2 = 82.5$ ,  $t_w = 9.9$ ,  $f_y = 250$ , and  $\gamma_{m0} = 1.1$ . The result is  $410.6 \text{ kN}$ . To the right of the main equation, the calculation for  $n_2$  is shown:  $n_2 = 2.5(R + t_f)$ , where  $R = 18$  and  $t_f = 15$ , resulting in  $n_2 = 82.5$ . At the bottom, the final comparison is shown:  $410.6 \text{ kN} > 202.5 \text{ kN}$ , with 'OK' written below it. A red pen is visible in the bottom right corner.

$$\begin{aligned} \text{step 8} \\ \text{web crippling} \\ F_w &= (b_1 + n_2) \times t_w \times f_y / \gamma_{m0} \\ &= (100 + 82.5) \times 9.9 \times \frac{250}{1.1} \\ &= 410.6 \text{ kN} > 202.5 \text{ kN} \\ &\quad \text{OK} \end{aligned} \quad \left| \begin{aligned} n_2 &= 2.5(R + t_f) \\ &= 2.5(18 + 15) \\ &= 82.5 \end{aligned} \right.$$

So the section is safe against web buckling. Now next step I will go that is step 8 web crippling. We have to check for web crippling. So for web crippling I have to find out the web crippling strength  $F_w$  is equal to  $b_1$  plus  $n_2$  into  $t_w$  into  $F_y$  by  $\gamma_{m0}$ , right. Here,  $n_2$  will be  $2.5$  into  $R$  plus  $t_f$ , because it will disperse with  $2.5$  slop  $2.5$  is to  $1$  slop. So  $2.5$  into  $R$  was  $18$ , thickness of flange was  $15$ . So this is becoming  $82.5$ . So  $b_1$  we have consider  $100$  mm length of bearing, so  $100$  mm plus  $82.5$  as  $n_2$  and thickness of web is  $9.9$  then  $250$  by  $\gamma_{m0}$ , for this is becoming  $410.6$  kilo newton and this is greater than  $202.5$  kilo newton. So the section is safe against web crippling right. So this is how we can find out the web crippling strength and we can see that the section is safe against web crippling.

So from this what we can see that we have to design the beam stage by stage. So the steps are discussed earlier, so according to these steps, we have gone through and we have checked every step that we are the assumed section is safe against moment, against shear, against deflection, against buckling and against crippling. So all the checks are satisfied, so the sections which are assumed is okay. So whatever I have done, I am quickly going through this the power point presentation so that you can keep note.

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**Solution:**

*Step 1: Calculation of load*  
Dead load =  $1.5 \times 18 = 27 \text{ kN/m}$   
Live load =  $1.5 \times 12 = 18 \text{ kN/m}$   
Total load =  $(27 + 18) = 45 \text{ kN/m}$

*Step 2: Calculation of BM and SF*  
$$BM = \frac{wl^2}{2} = \frac{45 \times 4.5^2}{2} = 456 \text{ kN-m}$$
$$SF = w \times l = 45 \times 4.5 = 202.5 \text{ kN}$$

*Step 3: Choosing a trial section*  
$$Z_{p, reqd} = \frac{M \times \gamma_{m0}}{f_y} = \frac{456 \times 10^6 \times 1.1}{250} = 2006.4 \times 10^3 \text{ mm}^3$$

So for example here, in first step we have calculated the load factor load which is coming 45 kilo newton per meter, 1.5 times dead load plus live load and then we have found the maximum bending moment, which will be  $wl^2$  square by 2 and maximum shear force which will be  $wl$  at the support. So maximum bending moment and maximum shear force are calculated in step 2. Then we have to choose a trial section so for that we can find out the plastic section modulus require, which is  $M$  by  $F_y$  by  $\gamma_{m0}$  and from that we can find that the  $Z_p$  require as 2006 into 10 cube millimeter cube. So this is the require of  $Z_p$ .

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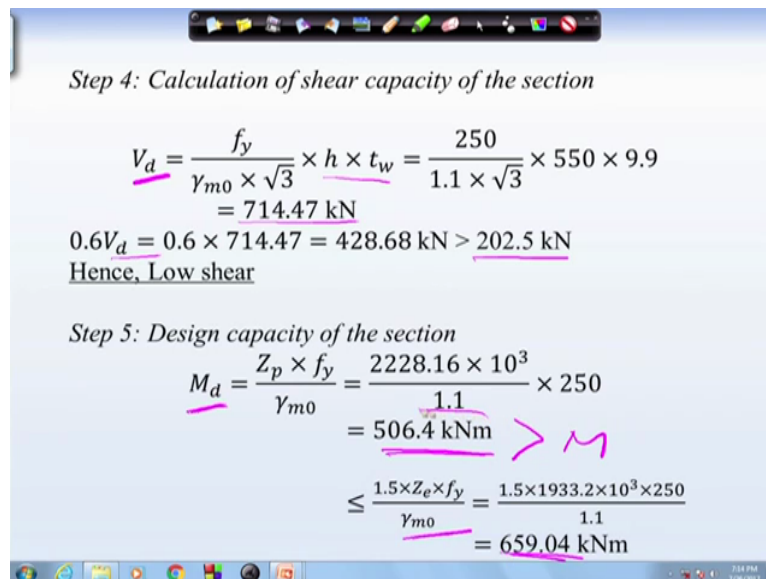
Let us select the section ISLB 550 @ 0.846 kN/m  
 $Z_{pz} = 2228.16 \times 10^3 \text{ mm}^3$   
 $Z_{ez} = 1933.2 \times 10^3 \text{ mm}^3$   
 $h = 550 \text{ mm}, b_f = 190 \text{ mm}, t_f = 15 \text{ mm}, t_w = 9.9 \text{ mm}, R = 18$   
 $d = 550 - 2 \times (15 + 18)$   
 $= 484 \text{ mm}$   
 $I_{zz} = 53161.6 \times 10^4 \text{ mm}^4$   
*Section classification*  
$$\frac{b_f/2}{t_f} = \frac{95}{15} = 6.33 < 9.4$$
$$\frac{d}{t_w} = \frac{484}{9.9} = 48.9 < 84$$

Hence, the section is plastic

And we will try to choose a section further its  $Z_p$  value becomes little higher than the requirement. So through sp6 we could find a section whose size is ISLB 550 and its  $Z_{pz}$  is

becoming slightly more than the requirement that is 2228.16 into 10 to the 3 millimeter cube and also we found the value of  $Z_{ez}$  then the overall depth, small  $d$  we can find and also we can find the  $t_f$ ,  $t_w$ , root radius and  $I_{zz}$ , right also we can find the section classification. So from section classification we can see, it is less than 9.4 and less than 84 as a result this section is a plastic. So the ISLB 550 when we are going to use, it will be a plastic section and its properties are given here.

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*Step 4: Calculation of shear capacity of the section*

$$V_d = \frac{f_y}{\gamma_{m0} \times \sqrt{3}} \times h \times t_w = \frac{250}{1.1 \times \sqrt{3}} \times 550 \times 9.9$$

$$= 714.47 \text{ kN}$$

$$0.6V_d = 0.6 \times 714.47 = 428.68 \text{ kN} > 202.5 \text{ kN}$$

Hence, Low shear

*Step 5: Design capacity of the section*

$$M_d = \frac{Z_p \times f_y}{\gamma_{m0}} = \frac{2228.16 \times 10^3}{1.1} \times 250$$

$$= 506.4 \text{ kNm} > M$$

$$\leq \frac{1.5 \times Z_e \times f_y}{\gamma_{m0}} = \frac{1.5 \times 1933.2 \times 10^3 \times 250}{1.1}$$

$$= 659.04 \text{ kNm}$$

So considering this property we can now find out the shear capacity of the section in step 4 that is  $V_d$  is equal to  $F_y$  by root 3  $\gamma_{m0}$  into (area) shear area which is  $h$  into  $t_w$ . So this value is coming 714 kilo newton and 0.6 times of that is greater than the developed shear. Therefore, it is low shear. So for low shear the design capacity of the sections can be calculate from this formula, where  $M_d$  is equal to  $Z_p$  into  $F_y$  by  $\gamma_{m0}$ .

So we can find out the  $M_d$  value as 506.4 kilo newton meter, which is greater than the develop moment  $w_l$  square by 2. So it is okay and then also we have to check whether it is less than this or not,  $1.5 Z_e F_y$  by  $\gamma_{m0}$ . There we can see that  $Z_e F_y$  by  $\gamma_{m0}$  is 659. So  $M_d$  value can be considered as this.

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Step 6: Check for deflection

$$\delta = \frac{wl^4}{8EI} = \frac{27 \times 4500^4}{8 \times 2 \times 10^5 \times 53161.6 \times 10^4} = 13 \text{ mm}$$

Allowable deflection =  $L/150 = 4500/150 = 30 \text{ mm}$   
 OK

Step 7: Web buckling

Cross sectional area of web for buckling  $A_b = (b_1 + n_1)t_w$   
 $b_1 = 100 \text{ mm}$   
 $n_1 = D/2 = 550/2 = 275 \text{ mm}$   
 $A_b = (100 + 275) \times 9.9$   
 $= 3712.5 \text{ mm}^2$

Effective length of the web =  $0.7 \times d = 0.7 \times 484 = 338.8 \text{ mm}$

$I = \frac{b \times t_w^3}{12} = \frac{100 \times 9.9^3}{12} = 8085.8 \text{ mm}^3$   
 $A = 100 \times 9.9 = 990 \text{ mm}^2$




Now we will go for deflection. So for deflection we will find out delta is equal to  $wl^4$  to the 4 by  $8EI$ ,  $w$  will be 30 into 4500 to the whole 4 by  $8EI$  then is value we have to check may be around 14 mm it will be and allowable deflection is coming 30 mm, which is much higher than this, right. So from deflection point of view, the (34:20) section is safe.

Then we will go for web buckling and add the bearing length was given 100 mm, so I can find out the cross section area of the web for buckling. So for that  $b_1$  we know and  $n_1$  because if this is bearing then it will be dispersed 45 degree up to neutral axis and this is  $d$  by 2, right which is  $n_1$ . So  $n_1$  is  $d$  by 2, is 275, so area of the web we can calculate and effective length of the web is  $0.7d$ , where  $d$  was calculated earlier, so we can find out  $I$  value,  $A$  value.



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$$r_{min} = \sqrt{\frac{8085.8}{990}} = 2.86 \text{ mm}$$

$$\lambda = \frac{l_{eff}}{r_{min}} = \frac{338.8}{2.85} = 119$$

Allowable stress  $f_{cd} = 84.8 \text{ N/mm}^2$   
 Capacity of the section =  $84.8 \times 3712.5 = 314.8 \text{ kN} > 202.5 \text{ kN}$   
 Hence, the section is safe against web buckling

Step 8 : Check for web crippling

$$F_w = \frac{(b_1 + n_2) \times t_w \times f_y}{\gamma_{m0}}$$

$n_2 = 2.5 (R + t_f) = 2.5(18 + 15) = 82.5 \text{ mm}$   
 $F_w = \frac{(100 + 82.5) \times 9.9 \times 250}{1.1} = 410.6 \text{ kN} > 202.5 \text{ kN}$   
 So the section is safe against web crippling

And then I can find out the  $r_{min}$  as 2.86 and then  $\lambda$ , right. Once  $\lambda$  is found we can find out the value of allowable stress  $F_{cd}$ , because corresponding to this  $\lambda$  and with buckling class C for  $F_y$  is equal to 250, the allowable stress is becoming 84.8 from table 9C and then capacity of the section can be found from the  $F_{cd}$  into  $A_v$  and it is becoming more than the shear force coming on the support. So the section is safe against web buckling and then we will go for web crippling.

So crippling strength also we can find out  $F_w$  is equal to  $b_1$  plus  $n_2$  into  $t_w$  into  $F_y$  by  $\gamma_{m0}$  and  $n_2$  will be calculated with a dispersion of 25 is to 1. So and it will be calculated if this is the flange thickness  $t_f$  and if this is the root radius then up to this, right. So this will be  $n_2$ . So  $n_2$  value is becoming 82.5 and then I can find out the crippling strength which is becoming 410 and is more than 202 kilo newton. So the section is safe against web crippling is. So this is how all the checks are performed and the size of the section which has been assumed is safe, right. So this is how we can design a section under low shear with its means which is means laterally supported beam means is it is laterally supported and low shear then how to design that has been demonstrated through this example, okay.