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# **Lecture – 18 Problems Solving on Black Board**

Very good afternoon to all of you. Today we are going to have fluid kinematics solving some of the problems on the blackboard.

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Looking that as I said it earliest we are again having same reference book starting from Cengel, Cimbala, F M White and Bidya Sagar Pani. So my sincere request to you to please look for the book of Cengel, Cimbala book which have which gives lot of illustrations to visualize the fluid flow problems because if would try to understand the fluid kinematics which is very interesting stuff subject.

Beside this solving the problems also we should look at how the flow behavior flow visualizes a technique that is what is very good illustrations are there in Cengel, Cimbala book. So please refer to Cengel, Cimbala book of fluid mechanics and other 2 books as we refer earlier case also. **(Refer Slide Time: 01:34)** 



Looking that again I will show it there it could be conduct a very small experiments which is called Heles apparatus. So where we can have an apparatus like these and we can create the streamline pattern, the streakline pattern and pathline patterns using these the small device which is called the Hele-Shaw apparatus. Like for examples if you can look it that if I have the obstruction structures like these and having the flow patterns like these you can see the streamline patterns what is going on near the structures and far away from the structures. So very easy to visualize the flow when you use the apparatus like Hele-Shaw apparatus.

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And if you look it very interesting flow visualizations are available in internet. Please refer to look at these flow visualization details, videos what these are available in the internets. Like for examples here it is showing a how the wake formation happens just behind of the cylinders when uniform flow is going on and how the wake formation happens is the very interesting phenomena.

Similar way you can have the oscillating plate and because of the oscillating plate how the streamline patterns are changing with respect to the time, the pathline, the streaks line all you can visualize using this type of video. So my sincere suggestions to you please visualize the flow visualizations by visiting this the YouTube sites you can see that how the flow patterns are changing it.

# **Flow Visualizations** Wake of cylinder visualized by surface powder Instantaneous streamlines for oscillating plate flow Source: Flow Visualization by S.J. KLINE, Stanford University; https://www.youtube.com/watch? **NUCHAKEGUY ICH**

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This is what the examples what earlier also we showed to you that if you have a triangular structures and how the way vortex sheddings are happening and it is quite interesting way if you see this unsteady patterns of waves vortex pattern. These are details obtained from the CFD solutions. So we can get it from the CFD solutions these type of vortex patterns and very interesting vortex patterns and that is what it shows that how these the average velocity line changes with that and with a comparison with experimental data.

So what I am to say that if you look at this advanced level of flow visualization technique and the experiment techniques what was available today we can have a very interesting flow problems. We can get the solutions. We can visualize the streamline, streaks line, the pressure distributions, the velocity distributions, the acceleration distributions all field we can see it. So with having these introductions levels, let us solve the 6 problems on the black.

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Before starting solving the blackboard applications, let me just have a recap that what we already discussed in the fluid kinematics is that you know that any velocity field we can define as 3 scalar components. The scalar component can have whether velocity scalar component can have whether positions and the time the independent part that is what the velocity distribution. Similar way the rate of the change of the velocity is accelerations but in terms of local accelerations and convective part we can define the acceleration terms.

Similar way when you have a motion of the fluid particles can have 4 type of conditions the motion and deformations like translations, rotations, linear strain, and the shear strain. We discussed more detail in the last class. So here I am just doing the recap for you to just look it this is the velocity distributions, the accelerations field and there could be the translations, rotations and the deformations like linear and shear strain.

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And if you look it if you have the rotations we can write the rotations quantity in terms of the velocity field. Similar way the shear strain components also we can write in terms of the velocity gradients and we have the vorticity measures what we derived very details we can compute the what could be the vorticity in different place. Look at that when you have the flow is incompressible flow that means when you have a volumetric strain equal to 0.

The delta dot product of the v velocity should be equal to 0. These the equations we were going to use more detail. When you are try to solve these simple problems what we are going to address on the blackboards. So the basically these are the recap the basic equations what we are going to use to solve the problems on the black.

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Let us start the first example 1.

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Example 1 says that what is the irrotational velocity field associated with the potential functions as given here. We have to find out the irrotational velocity field, does this flow field satisfy incompressible continuity equations? So it has 2 steps, the first steps were to find out what is the irrotational velocity field? So that is the points the already is given the flow is irrotational.

So that is the condition is given that means for this velocity potential functions which is the functions of if I write it is a function of x, y, z and the t the velocity potential function is the functions of the positions x, y, z and the t. If you can see these independent variables. So if you have the velocity potential functions which is function of x, y, z and t. Let me I just give a simple examples if I have the velocity potential function is a only 1 space functions and another time functions.

$$
\varphi = 3x^2 - 3x + 3y^2 + 16t^2 + 12zt
$$

$$
\varphi = \varphi(x, y, z, t)
$$

And if I plot this the functions could variables like this which is a functions of x and the t and to determine what could be the velocity? As the velocity potential function satisfy for this flow field we know from the definitions that the u will be a partial derivative of velocity potential functions and the v will be the partial derivative functions with respect to the y directions.

So these are the scalar component of v, u and the v the velocity functions what we can get it. So that is means if you look it the graphically what it indicates is that when you try to find out the velocity that means from the partial derivative directions the  $\varphi$  changes only the x directions when you do not consider other component that is what is of the negative of that is what is indicate for us the velocity field.

So looking that definitions let us come back to the problems which are quite easy problem for us to solve here that as we knew it the u is a the partial derivative of the  $\varphi$  with respect to x with negative sign because it is what it directions of that similar way the  $\varphi$  will have the negative signs of this. If I substitute that value that means if I just compute it that could be the partial derivative of  $\varphi$  with respect to x.

You can look it except these 2 terms I do not have a x term in here. So those terms becomes 0. So if you look it that way if you do a partial derivative of this I will get it -  $6x + 3$ , same way if I compute it the scalar velocity field in the u, y directions which will be the partial derivative of with respect to the y. If you look at these equations again what we can see it in these equations only the y is in this term.

$$
u = -\frac{\partial \varphi}{\partial x} = -6x + 3
$$

$$
\frac{\partial u}{\partial x} = -6
$$

$$
v = -\frac{\partial \varphi}{\partial y} = -6y
$$

$$
\frac{\partial v}{\partial x} = -6
$$

So others are can be considered as a constants. So if you look at that and do a partial derivative  $\varphi$ with respect to the y, you will get -6y. So these are 2 velocity field which we obtain from this case. Now if we look at it these are the velocity field that is what is the part number a what we are looking for irrotational field. Now the second part of the problem does the flow field satisfy the incompressible continuity equations.

Since the flow field what we are looking in a 2 dimensional the continuity equations for 2 dimensional flow field will have

$$
\frac{\partial u}{\partial x} + \frac{\partial v}{\partial x} = 0
$$

. So it is a quite interesting to look at that means we can again do a partial derivative with respect to 1 x. This is what also partial derivative with respect to y and substitute it. If it is satisfied it then we can say it satisfy the continuity equations.

$$
\frac{\partial u}{\partial x} + \frac{\partial v}{\partial x} = 0
$$
  
-6 - 6 = -12 \rightarrow not satisfied.

So if you look in this part if you do y del y del x then we will get it -6. Similar way del v/del y will also get -6. So substituting this value what will get it -6 -6 is -12. That means it is not satisfied. That is what is the answer for the second part okay.

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Let us solve this next example 2, in fluids the stream function is described at the  $\Psi$  functions.

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The stream functions as defined as

$$
\Psi = x^3 - 3xy^2
$$

determine the flow whether the flow is rotational or irrotational. If the flow is irrotational then indicate the correct value of velocity potentials. So the first part of the problem is that we have to prove whether the flow is irrotational or rotational. If again if I try to explaining you that now we have the stream functions which is functions of only the 2 dimensional x and y that is what could be a representing like a stream functions like this.

$$
\Psi=\Psi(x,y)
$$

And we try to find out whether the flow is rotational or irrotational. The questions comes is that to find out whether the flow is rotational or irrotational. What do you mean by that? That means we have to compute it the vorticity. If this vorticity becomes 0 that means we can say the flow is irrotational. So we have to find out the vorticity vector in this case it is only the 2 dimensional we need to have compute the vorticity vector in z directions okay.

$$
\zeta = \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y}\right)\vec{k}
$$

That means we need a vorticity vector of z direction components if that becomes 0 then that what indicate is the flow is irrotational that is the **fact** if not the flow is rotational. Let us compute the vorticity vector, the mathematically as we define the vorticity vectors as in the z directions is a functions of so it is a partial derivative of v scalar component with respect to y, scalar component of u with respect to y.

$$
\zeta = \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y}\right)\vec{k}
$$

This is the vorticity vector component in the z direction. So we are just looking it in the z direction vector vorticity components and only z directions. Now the problem is very simple that will do a partial derivative of the stream functions to compute it what will be the v and u once you know that we can compute it what will be the vorticity vectors. What is the relationship between the stream functions and the velocity scalar velocity components as by definitions?

What we know it the velocity component of u will be the partial derivative with respect to y directions? If you look at this u components will have either the partial derivative of stream functions y directions that what is give me the scalar component in x directions and the v is given is the partial derivative with x directions. The negative of that it is represent the v component that means the scalar component in the y directions.

$$
u = \frac{\partial \Psi}{\partial y} = -6xy
$$

$$
v = -\frac{\partial \Psi}{\partial v} = -3x^2 - 3y^2 = -3(x^2 - y^2)
$$

By substituting this value of the stream functions we can get it this is what  $-6xy$  and this is what minus of 3x square 3y square that is what will come out to be  $-3(x^2 - y^2)$ . Now as I know these u and the v component. I will just substitute here to compute it what is the vorticity vector in the z direction. This is very simple once I know the scalar component of even v which is a function of the partial derivative of the stream functions I once I get that things that what I can substitute here to compute it.

$$
\zeta = \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y}\right)
$$

$$
\zeta = -6x - (-6x) = 0 \rightarrow flow \text{ is irrotational}
$$

That is what you can compute do partial derivatives where we are not going a step by steps but what you can get it - 6 x minus of  $-6$  x that is equal to 0. That is what indicate this flow is irrotational that means that is what is the first part of the component is that we have proved that the flow is irrotational as the vorticity vector becomes 0 that the first components.

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So the second part of the component is that if the velocity potential functions is given to us okay the velocity potential function is given to us in these case we have

$$
\varphi = y^3 - 3xy^2
$$

We need to find out whether this is what also indicate it is a flow is irrotational that means the condition is irrotational flow conditions. That is what also we can get it the basic equations is for flow irrotational is the Laplace equations with respect to the velocity potential functions.

$$
\frac{\partial^2 \varphi}{\partial x^2} + \frac{\partial^2 \varphi}{\partial y^2} = 0
$$

That means as it is equal to 0. This is the Laplace functions with respect to the  $\varphi$  in the 2 dimensional form, the x and the y components. So that is what is indicating for us to substitute this the phi value and try to look it does it satisfy is equal to 0. If it is equal to 0 then flow is irrotational. When I substitute these value that means we can put the first components is that the which is if you just split like this you can easily find out what could be the values.

Like for examples if you put it these function

$$
\frac{\partial \varphi}{\partial y} = 3y^2 - 3x2y
$$

$$
\frac{\partial^2 \varphi}{\partial y^2} = 6y - 6x
$$

We can split like this and looking these terms, you can find out the first partial derivative will give us -3y square and second one if I do it which does not have a function of x then definitely it will give us the 0 value.

The same way if I compute the partial derivatives of the with respect to y that is what will give you the  $6y - 6x$ .

$$
\frac{\partial^2 \varphi}{\partial x^2} = \frac{\partial}{\partial x} \left( \frac{\partial \varphi}{\partial x} \right) = 0
$$

$$
\frac{\partial^2 \varphi}{\partial y^2} = \frac{\partial}{\partial y} \left( \frac{\partial \varphi}{\partial y} \right) = 6y - 6x
$$

So if I substitute these values the Laplace equations of phi x square here that what will get it by -6x. So it is not equal to 0.

$$
\frac{\partial^2 \varphi}{\partial x^2} + \frac{\partial^2 \varphi}{\partial y^2} = 6y - 6x \neq 0
$$

That means the flow is rotational and it does not satisfy the basic equations of the Laplace equation which is a continuity equation for the profit. The similar way if I go for the second components where another phi function is given.

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 $\varphi = -3x^2y$ 

So were to again just change the functions find out whether it satisfy the Laplace equations of the phi value. If it satisfy with the Laplace equations of the phi value then we can say that this is for the velocity potential functions for this process. If I substitute these once again like to compute the partial derivative of  $\varphi$  with x in a second partial derivative of the  $\varphi$  with respect to x that what will give us if I substitute these values that is what will give this for here is -6y.

$$
\frac{\partial^2 \varphi}{\partial x^2} = \frac{\partial}{\partial x} \left( \frac{\partial \varphi}{\partial x} \right) = -6y
$$

Similar way if I look it since it is only the first order y is there I can easily say it this what will becomes 0.

$$
\frac{\partial^2 \varphi}{\partial y^2} = \frac{\partial}{\partial y} \left( \frac{\partial \varphi}{\partial y} \right) = 0
$$

So if you look it this 2 term -6y and 0 definitely the Laplace equations after substituting this Laplace equations that what will not be equal to  $0$ , so it is not satisfied.

$$
\frac{\partial^2 \varphi}{\partial x^2} + \frac{\partial^2 \varphi}{\partial y^2} = -6y \neq 0
$$

This is what very simple examples just to look it by substituting this the velocity potential function that is satisfy the Laplace equations.

If satisfy the Laplace equations that means it is the velocity potential field. If not, then it is not a appropriate velocity field to define this flow of behaviours. So that what we have to look it for these keys.

(Refer Slide Time: 26:13)

Example 3  
\nFlow through a converging nozzle can be approximated by a one  
\nDimensional velocity distribution u=u(x). For the nozzle shown  
\nin the figure, assume the velocity varies linearly from u=V<sub>0</sub> at the  
\nentrance to u=3V<sub>0</sub> at the exit:  
\n
$$
u(x) = V_0 \left[1 + \frac{2x}{L}\right] \quad \frac{\partial u}{\partial x} = \frac{2V_0}{L}.
$$
\nEvaluate acceleration  $\frac{du}{dt}$  at the entrance and exit if V<sub>0</sub> is 10 ft/s and  
\nL= 1 ft.

Let us solve the third examples which is a very interesting problems. The flow through a converging nozzle as given in the figure can be approximated as 1 dimensional flow velocity distributions  $u = u(x)$  and from the nozzle shown in this figure and assume the velocity varies linearly at this point is  $u = V_0$ .

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And at the entrance points and the velocity at the exit point will be the  $3V_0$ . And the velocity distributions is given with respect to the x directions this is the about the nozzles. The velocity field is given as the

$$
u(x) = V_0 \left[ 1 + \frac{2x}{L} \right]
$$

This is what the velocity distributions given to us. What we have to need to compute it? What is the acceleration in the x directions? du/dt at the entrance point at this  $x = 0$  that is what the entrance and this is what of exit of the flow.

So if it is the conditions we need to compute it what will be the accelerations ax du/dt.

If the  $V_0 = 10 ft/sec$  and the length is 1 feet. Now if you look at these problems is very easy problems it may be has described as the converging nozzles giving the velocity increase of the velocity as the nozzle dimension is coming, decreasing trend. But in mathematically it is a very easy that we are defining it what could be the accelerations as the total derivative of u component with respect to the time.

And as given in the figure also I am just highlighting it this is the what 1 dimensional flow only x directions component what we have. That means the by definitions the accelerations and along the x directions accelerations the accelerations in the x directions can be written as very simple forms just taking the definitions that the accelerations will be total derivative of du/dt that is what will

represent as a partial derivative of time, the local accelerations component and convective acceleration component in the x directions that what were look into it.

$$
\frac{du}{dt} = \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z}
$$

$$
\frac{\partial u}{\partial t} = 0 \rightarrow steady flow
$$

$$
v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} = 0 \rightarrow one dimensional flow field
$$

But if you look at these problems what are the components can be neglected as it is a steady flow there is no time component on this. We can make it this is equal to 0 and since it is a 1 dimensional flow as it is a steady flow when is no time component in the velocity field and it is a 1 dimensional flow. These the components also becomes 0 as it is 1 dimensional flow field. That is what also with these components all becomes 0.

So it is quite easy we have to compute this part. So if you just substitute this value and do a partial derivative of this u with respect to x and if you substitute it will get it the du that by substitute it is

$$
\frac{du}{dt} = \frac{2V_0^2}{L} \left( 1 - \frac{2x}{L} \right)
$$

So it is very easy as you do a partial derivative with respect to x. If you do the partial derivative you can look it and substituting this the u value, you will get this value.

So we know this how this accelerations ax varies with respect to x and what is a functions in terms of V<sub>0</sub> and the L? So we have to find out what will be the velocity when  $x = 0$  and  $x = L$  and the V<sub>0</sub> is given to us. So using given data we can compute it the accelerations in the x directions du/dt at  $x = 0$ . I am just substitute it in here when  $x = 0$ ,

$$
\left. \frac{du}{dt} \right|_{x=0} = 200 \, ft/s^2
$$

value substituting the  $V_0$  and the L value that is what we will get it 200 feet per square.

The same way at exit point where  $x = L$  just substituting here,  $x = L$  value. We will get it du/dt = 600 feet per second square which is given is 3 times of  $V_0$ .

$$
\frac{du}{dt} = 600 \ ft/s^2
$$

So if you look at that we can compute this the velocity is increasing, the accelerations also increasing than that point that is what we are computing using that substitute of it.

So let me summarize these problems one of the easy problems only it has described the velocity field giving a converging nozzles and telling this the velocity at this point is equal to  $V_0$  and the velocity increases into 3 times of V<sub>0</sub> here and the velocity was described here and that what we are just substituting as computing the acceleration in the x directions only. That what we just substituting mathematically get it this value what will be the acceleration at the entrance point and the exit point. This is one of the easiest problems.

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So let us come to the example 4 which is indicates that a simple 2 dimensional flow as shown in this figure.

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It is a 2 dimensional flow patterns, the upper boundary is being a rectangular hyperbola is given an equation is equal to  $xy = k$ This upper part as given it that and assuming the scalar component of velocity fields are as given as the

$$
u = V_x = -Ax
$$

$$
v = V_y = Ay
$$

$$
V_z = 0
$$

as a 2 dimensional field and where the A is a constant. We have to determine the streamline pattern and also find the accelerations field.

The problems what we are looking at we have to determine now the streamline the pattern means we are looking at what could be the equations for that and also we are looking at what will be the accelerations field. This 2 the problems which is a very straightforward equation looking at that as we have given the velocity scalar components we have to find out the stream functions once you know the stream functions then we can derive what could be the accelerations field?

That means you can sketch the stream functions could be like this would come it as expanding it. So we can draw a streamlines like this. So we are looking at what is these functions? So defining this stream functions. Now if you look at the result part what we are looking at here that we need to put the basic equations of the streamline equations okay. The basic the definitions are the equations of the streamlines as the by definitions what we do it that

$$
\left. \frac{dy}{dx} \right|_{streamline} = \frac{V_y}{V_x} = \frac{-y}{x}
$$

this is what the definitions.

As you know it the when you take a tangential component of a streamlines that should be indicate in the velocity vector this is what the velocity vectors at this point and the tangential component are the parallel that is what the definitions of the streamlines. So we are defining that definitions in mathematically the slope of the streamline  $=Vy/Vx$ . So if I substitute this  $Vy/Vx$  you can just substituting these values we can get  $-y/x$ .

$$
\frac{V_y}{V_x} = \frac{-y}{x}
$$

So as we are looking at what could be the streamline functions. This is what we resize the streamline functions here see if I do integrations, the integrating the above equations we get that the value or we can write it very simple form is the

$$
ln y = ln x + C
$$

$$
xy = c
$$

That means again it is indicating it also will have a hyperbolic nature of the streamlined functions. That is what is define the boundary the vectoral hyperbola functions where it is falling for the streamline functions that what we are getting from the is the equations for streamline.

Now if you look it to compute the accelerations components. So basically we look at 2 accelerations component as there is no the time components. We can as the flow is steady which is very easy to compute now. We can compute x values as given as earlier du/dt total derivative of y with respect to time. That is what we will substitute as in this case this is what the v, u as this is the v components.

$$
a_x = \frac{du}{dt} = A^2 x
$$

$$
a_y = \frac{dv}{dt} = A^2 y
$$

So these components become 0. So you know it is value to that but it does not have a functions of y and z. So these components 0 and this component 0. So substituting this value will get its value is equal to  $A^2x$ . Similar way we compute ay which is dv/dt writing the same expressions instead of u we can write a scalar component in the y direction is V and if I substitute it I will get it  $A^2y$ .

That means the accelerations becomes a is the accelerations field in the 2 dimensional that what will be the  $A^2x$  i,  $A^2y$  i component that is what the answer for these problem.

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Let us solve the next examples that is example number 5, an incompressible flow around a circular cylinders of radius a as given in the figure is represented by a stream functions as given it here as

$$
Ur\sin\theta+\frac{Ua^2\sin\theta}{r}
$$

r is the radius as a variables and the total radius of the cylinders is equal to the a, a is a radius of this and U is here represent as the free stream velocity.

So we need to compute it first we have to prove it the Vr the radial velocities is equal to 0 along the circles when  $r = y$ . So we have to first find out the radial velocity Vr is it equal to 0 when  $r = a$ value and the tangential V theta component also we have to find out the value of theta where the magnitudes of the velocity vectors should equal to the free stream velocity components. So it is bit analytical way to compute it.

But if you look it the stream function is given to us where to compute the radial velocity and the tangential velocity. So the problem is very easy but only the problem is here it has given in terms of cylindrical coordinate system. So we have cylindrical coordinate systems that is what we are using it. So we have to write this the velocity the function in terms of the stream function for a cylindrical coordinate systems as the flow is also the flow is incompressible okay.

That is a basic definitions here. So in cylindrical coordinates or the polar coordinate systems we can define it the velocity field we can define the velocity scalar component as given like this that

$$
V_r = \frac{1}{r} \frac{\partial \Psi}{\partial \theta}
$$

$$
V_{\theta} = -\frac{\partial \Psi}{\partial r}
$$

So the please refer to a books to know what is the definitions of the relationship between velocity scalar components and the stream functions for a polar coordinate system which is very same way what we do for that.

But just telling if you look it the  $1/r$  component where it comes when you have partial derivation with respect to theta values. So as these problems is given to us we are just to find out whether the Vr the radial velocity component is equal to 0. So it is very easy things. Now let us compute it by getting substituting the  $\Psi$  functions. You can compute it that means I am substituting the stream functions has given it here is

$$
Ur\sin\theta+\frac{Ua^2\sin\theta}{r}
$$

So if you substitute but do a partial derivatives with respect to  $\theta$  and put it these value which really comes out to be

$$
-U\cos\theta\left(1-\frac{a^2}{r^2}\right)
$$

If you look at these functions when the r becomes a, no doubt the  $V_r$  will be 0. Look at this function if substitute  $a = r$  this component become 0. So  $V_r = 0$ . That is what is the first component what we prove it. Now looking for the second component that what will be the value of  $V_a$ ?

$$
r = a \Rightarrow V_r = 0
$$

That means with respect to r just substituting this stream functions partial derivative with r and negative of that which indicates the tangential velocity component along the theta locations that what will come it here is

$$
V_{\theta} = -\frac{\partial \Psi}{\partial r} = U \sin \theta \left( 1 + \frac{a^2}{r^2} \right)
$$

So when you have a becomes r that means the conditions when  $a = r$  value. So you get is

$$
V_{\theta} = 2 \sin \theta V
$$

That is the tangential velocity.

So r = a, the radial velocity 0 but the tangential velocity is equal to 2 Vsin  $\theta$  Now you are looking it at what  $\theta$  values we will have the resultant of the velocity vector is equal to the free stream velocity. This is quite easy for us that means we can as we know this  $V_r$  and  $V_\theta$  value we can find out total magnitudes of the velocity vectors and that what we will equate to the 0 and that the conditions at which theta value its gives a value equal to the resultant velocity magnitude is equal to this is the conditions what we looking it.

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If I just as I know it V and  $V_\theta$  and I can get the magnitudes and if I equate with U value, I will get

$$
|\vec{V}| = U
$$

$$
\Rightarrow \sin \theta = \pm \frac{1}{2}
$$

$$
\Rightarrow \theta = 30^{\circ}, 150^{\circ}, 210^{\circ}, 330^{\circ}
$$

So you can look it that you will have a  $\theta = 30^{\circ}$  after that 150 degrees, 210 and 330 degrees you will have the magnitudes of your resultant velocity is will be equal to the free streams velocity.

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Which is very interesting the figures what you can see on the blackboard a 2 dimensional dipole source at the origins produces a steady incompressible flow with the stream functions as it is given.

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The stream functions is given to us we have to find out the directions of motion of the fluid particles at the  $x = 6$  and  $y = 9$ . We need to know it what could be the directions of the fluid particles okay. So if we look at that, that means if it is a dipoles and the streamlines could be like these functions as you look at this function previous that means at a particular locations, we are looking it what could be the flow directions okay?

That means we have to find out if a streamlines are going through these what could be the functions of the streamline with respect to y and x, the slope components? If I know the slope component of that functions at that point that is what will give us the directions. So the first let us go for that we as know it that along the streamlines okay the  $d\Psi = 0$  because along the streamline the stream functions value is constant. It does not wait.

So  $d\Psi = 0$ . That is what is indicate for us if I substitute the

$$
d\Psi = 0
$$
  

$$
\Rightarrow \frac{(x^2 + y^2)dy - y(2xdy + 2ydy)}{(x^2 + y^2)^2} = 0
$$

That is what is equal to 0. So here we have a very basic if a  $\Psi$  is a function of x and y, the total derivative of that. That is what we can expand it and substitute for that you can get it this components okay.

Just to follow any mathematics books to know it how it is expressions is coming it. If I rearrange this term then I can just rearrangement of the terms we can get

$$
\frac{dy}{dx} = \frac{2xy}{x^2 - y^2} = -2.4
$$

$$
\Rightarrow \tan^{-1}(-2.4) = -67.4
$$

So it is giving it that when you have a x and y value the velocity vectors will have an angle minus of 67.4 where you can find out what will be the directions. Now the second components were showing it that what could be the sketching of the streamlines.

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The sketching of the streamlines if you can find out that if you look in these streamline functions and since it is a dipole that means there is a 2 source components are there and we are drawing this streamlines patterns like this. So it is should be related into the functions in terms of a circular equations. So considering that part what we define it that the streamlines is defined as a constant is defined as

$$
\Psi=2\frac{1}{2\lambda}
$$

2 lambda for our easy to write the circular equations for that.

That is the reasons 2 lambda is given. So what I am to say that before draw find out what could be the equations you should understand the flow things that what could be the conditions. In this case very easily can say that there will be the equation of the circles will be there to define these things. That is 2 reasons where define these the stream functions is 1/2 lambda values and if I substitute that value as

$$
\frac{y}{x^2 + y^2} = \frac{1}{2\lambda}
$$

$$
\Rightarrow [x^2 + (y - \lambda)^2 = \lambda^2]
$$

And these equations if I further simplified it which is the equations for a circles. We can write it like this form that is what is representing here the it is a circle of radius centred at 0 and the  $\lambda$ , as the  $\lambda$  will vary we will change it the circles different circles will get it. That is what is the solution for this. So with this let us complete this blackboard lectures. What we have prepared by solving the 6 problems. Thank you.