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**Lecture 20 - Source Model Problems**

So welcome to this module of Source Model and in this module we are going to discuss several problems which are helpful in the entire study.

(Refer Slide Time: 00:43)

Question1: Calculate the head loss using 2-K method developed due to pipe friction and fittings for the section shown in figure below. Details are as follows:

Pipe material: Cast Iron (4")

Pipe internal diameter: 10.23 cm

Pipe length: 60m

Fluid: Water

Density of fluid:  $1000 \text{ kg/m}^3$

Viscosity of fluid:  $8.9 \times 10^{-4} \text{ Pa}\cdot\text{s}$

Velocity of fluid: 5 m/s

Fittings:

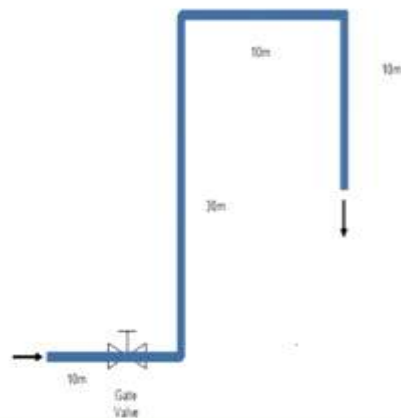
Elbows: 3 x  $90^\circ$  long radius ( $r/D = 1.5$ )

Gate Valve: 1 x Full line size,  $\beta = 1.0$



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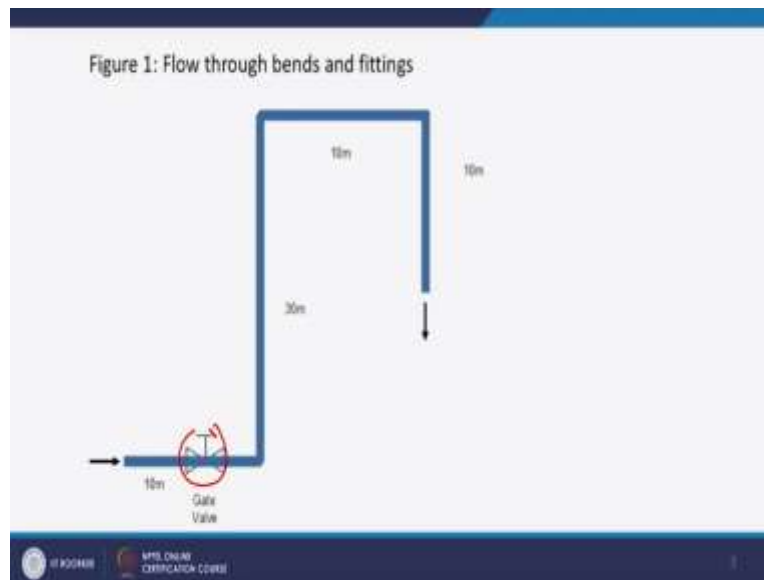
Figure 1: Flow through bends and fittings



So let us have the first problem. This problem here you need to calculate the head loss using 2-K which we have studied in the previous module, developed in due to pipe friction and fitting for the section as shown in the figure this one, this particular figure and details whatever details are required they are as follows: That pipe material which is having the cast iron 4 inch, the pipe internal diameter is given to you which is 10.23 centimeter, the pipe length is 60 meter. You are using the fluid as water, the density of the fluid is 1000 kg per meter cube.

Viscosity of the fluid is  $8.9 \times 10^{-4}$  Pascal Second, the velocity of the fluid is 5 meter per second. 2 fittings are there, elbow 3 x 90 degree long radius that is  $r$  by  $D$  is equal to 1.5 and a gate valve 1 into full line size and beta is equal to 1.

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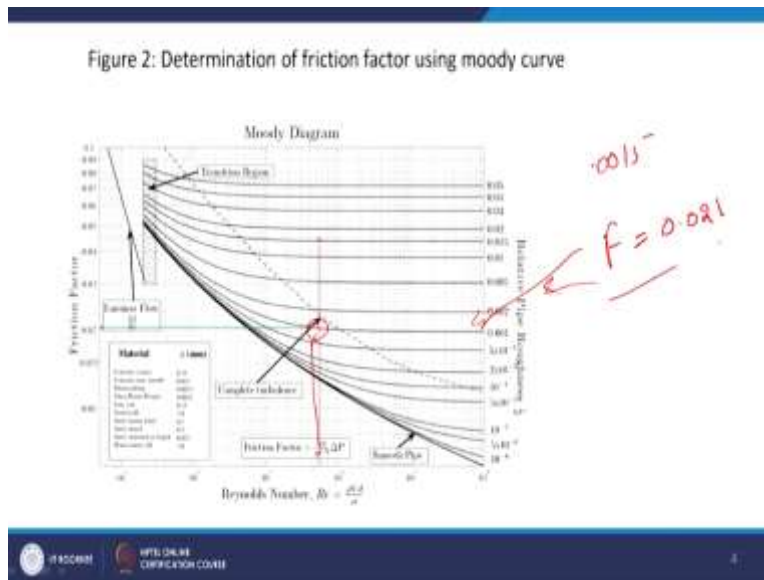
So this is the figure, here this is the 10 meter and then the gate valve is there, the height is 10 meter, then again the length is 10 meter and this 10 meter height. To assist you we are having the Moody's chart with us, so let us start with this particular problem, the solution of this particular problem. The value of  $K$  and  $K_0$  we can determine the table which was given earlier.

(Refer Slide Time: 02:18)

90° Elbow ( $r/D = 1.5$ )  $K_1 = 800$   $K_\infty = 0.20$  — (1)  
Gate valve  $K_1 = 300$   $K_\infty = 0.10$  — (2)  
Gate value  
 $\epsilon = 0.15 \text{ mm}$   
relative roughness  $= \frac{\epsilon}{D} = \frac{0.15 \text{ mm}}{102.3 \text{ mm}} = 0.0015$  — (3)  
Reynolds no  $Re = \frac{DV}{\nu} = \frac{DV\rho}{\mu}$   
 $= \frac{0.1023 \text{ m} \times 5 \text{ m/s} \times 1000 \text{ kg/m}^3}{8.9 \times 10^{-4} \text{ kg/m}\cdot\text{s}}$   
 $= 574719.1$   
 $\approx 5.75 \times 10^5$  — (4)

The 90 degree elbow  $r$  by  $D$  is equal to 1.5, so the  $K_1$  is 800 and  $K_\infty$  is 0.2 and gate valve, the  $K_1$  is 300 and  $K_\infty$  is 0.10. So absolute roughness for cast iron pipe,  $\epsilon$  is 0.15 mm and the relative roughness is  $\epsilon$  upon  $D$  that is 0.15 upon mm, so it comes out to be 0.0015. So it is equation 1, 2 and 3. The Reynolds number  $Re$ ,  $DV/\nu$ ,  $DV\rho/\mu$ , it is 0.1023 meter into 5 meter per second into 1000 kilogram per meter cube divided by 8.9 into 10 to the power minus 4 kilogram per, which is (57), 574719.1, nearly you can write 5.75 into  $10^5$ . Now we can determine the value of friction factor using Moody's curve, now here we are having the Moody's curve with us.

(Refer Slide Time: 04:27)



So we can determine the value of friction factor, so if we utilize the values which we have calculated earlier (the) this is your Reynolds number and this is the relative pipe roughness. Now with the help of these 2 factors we can calculate the friction factor. Now, here the relative roughness is 0.0015, so we can calculate from here and the Reynolds number is  $5.75 \times 10^5$  into the power 5, so the point of cross section is somewhere here and if we extrapolate the things to the friction factor it is coming out to be 0.021, now this is a very useful information through which we can proceed further.

(Refer Slide Time: 05:34)

$$K_f = \frac{K_1}{Re} + K_2 \left( \frac{H}{D} \right)^{1/5}$$

$$D = 0.1023 \text{ m}$$

$$K_f = \frac{800}{5.75 \times 10^5} + 0.2 \left( 1 + \frac{1}{\frac{10.23 \text{ cm}}{2.54 \text{ cm/inch}}} \right)$$

$$K_f = 139.13 \times 10^{-5} + 24965.79 \times 10^{-5}$$

$$K_f = 25,104.92 \times 10^{-5}$$

$$K_f = 0.251$$

$$K_f (\text{for } 3 \times 90^\circ \text{ elbow}) = 3 \times 0.251 = 0.753$$

Now, let us calculate the resistance coefficient for 90 degree elbow, the

$$K_f = \frac{K_1}{Re} + \left(1 + \frac{1}{D}\right)$$

now using  $K_1$  and  $K_\infty$  from the previous equation and the Reynolds number from equation number 4 and  $D$  is equal to 0.1023 meter, we can calculate the  $K_f$  that is 800 divided by 5.75 into  $10^5 + 0.2$ , 2.54 centimeter per inch.

Now always remember to convert the value of  $D$  into inch for determining the value of resistance and even in fact all calculation you must be consistent towards the unit.

Now  $K_f$  is 139.13 into  $10^5$  plus  $24965.79 \times 10^{-5}$

So,  $K_f$  is coming out to be 25104.92 into  $10^{-5}$ , now  $K_f$  is 0.251. Now for elbow of the similar geometry we can directly multiply the value of this particular value with 3, so  $K_f$  for 3 x 90 degree elbow which is equal to 3 into 0.251 which is coming out to be 0.753.

(Refer Slide Time: 07:28)

Gate Value  $K_1 = 300$  &  $K_\infty = 0.10$

$$K_f (\text{Gate Value}) = \frac{300}{5.75 \times 10^5} + 0.1 \left(1 + \frac{10.23 \text{ cm}}{2.54 \text{ cm}}\right)$$

$$= 52.17 \times 10^{-5} + 12482.89 \times 10^{-5}$$

$$= 12535.06 \times 10^{-5}$$

$$= 0.125$$

$$K_f = K_f (90^\circ \text{ Elbow}) + K_f (\text{Gate Value})$$

$$K_f = 0.753 + 0.125 = 0.878 \quad \text{--- (8)}$$

$$h_L = K_f \frac{V^2}{2g} = \frac{0.878 \times (5.796)^2}{2 \times 9.81 \text{ m/s}^2}$$

$$h_L = 1.12 \quad \text{--- (9)}$$

Similarly for gate value  $K_1$  is equal to 300 and  $K_\infty$  is 0.10 which is for gate valve, the remaining value will be the same, now  $K_f$  for gate value is equal to 300 on  $5.75 \times 10^5 + 0.1$ , 1 plus 10.23, 2.54 which is 52.17 into 10 to the power minus 5 plus 12482.89 into 10 to power, so 12535.06 into, or nearly 0.125.

So the total resistance coefficient due to the fitting will be  $K_f$  is equal to  $K_f$  90 degree elbow plus  $K_f$  gate valve that is  $K_f$  is equal to 0.753 plus 0.125, 0.878, now total head loss due to fitting can be calculated using this particular equation that is  $h_L$  is equal to  $K_f \frac{V^2}{2g}$  and this is 0.878 into 5 meter per second square divided by 2 into 9.81 meter per second square,  $h_L$  is 1.12.

(Refer Slide Time: 09:34)

head loss due to pipe friction

$$h_f = f \frac{L}{D} \frac{V^2}{2g}$$

$f = 0.021$   
 $L = 60 \text{ m (given)}$   
 $V = 5 \text{ m/s (given)}$   
 $D = 0.1023 \text{ m (given)}$

$$h_f = \frac{0.021 \times 60}{0.1023} \times \frac{5^2}{2 \times 9.81}$$

$h_f = 15.70 \text{ m}$

Total head loss =  $1.12 + 15.70$   
 $= 16.82 \text{ m}$  Ans

Now calculation for the head loss due to pipe friction, so  $f$  is equal to 0.021,  $h_f$  is  $f \frac{L}{D} \frac{V^2}{2g}$ ,  $L$  is 60 meter that is given,  $V$  is 5 meter per second again given,  $D$  is 0.1023 meter that is given, so  $h_f$  is equal to 0.021 into 60 and 0.1023 into 5, 9.81 meter per second square, so  $h_f$  is 15.70 meter.

So total head loss, 1.12 plus 15.70 which is 16.82 meter, now it is to be noted from the equation number 10 and 9 that the loss due to fitting is comparatively very less than the losses due to friction pipeline of 60 meter. So therefore for a large pipeline such as the crude oil pipeline which usually flows from countries to countries in several kilometers we can neglect the frictional loss due to fitting whereas for safety calculation these values are extremely important.

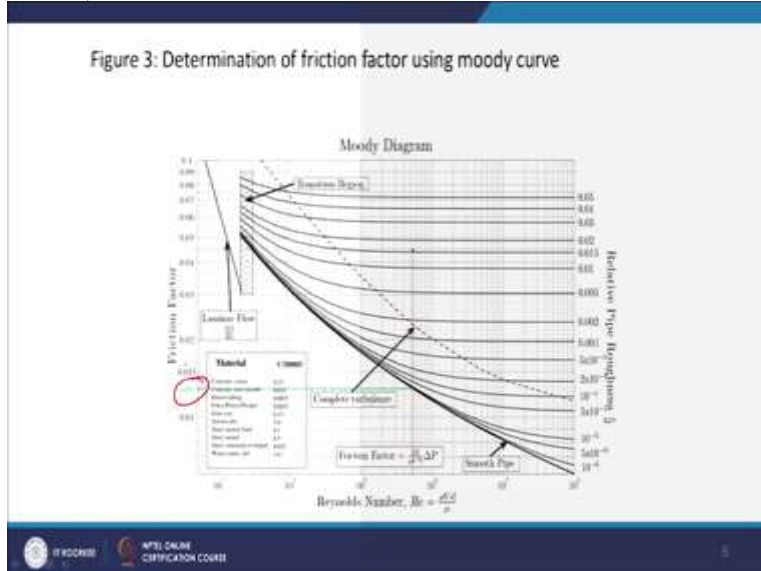
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Question 2: Calculate the head loss for smooth pipes such as glass or plastic using 2-K method for the same problem.

$$\begin{aligned}k_s &= 5.75 \times 10^{-5} \\f &= 0.013 \\h_f &= f \times \frac{L}{D} \times \frac{V^2}{2g} \\&= 0.013 \times \frac{60\text{ m}}{0.1023\text{ m}} \times \frac{(5.78\text{ m/s})^2}{2 \times 9.81\text{ m/s}^2} \\h_f &= 9.72\text{ m}\end{aligned}$$

(Refer Slide Time: 12:05)

Figure 3: Determination of friction factor using moody curve



Now, let us take the second problem that is you need to calculate the head loss for the smooth pipe such as glass or plastic using the 2K method for the same problem which we have discussed in the previous problem, so as all conditions are same for this problem except the pipe material low and hence the relative roughness, so we can directly calculate the value of the friction factor, now from  $Re$  which we have calculated in the previous equation  $Re$  is equal to  $5.75 \times 10^5$ .

Now with the help of Moody's chart which is given over here we can calculate because the relative pipe roughness is will be different because we have changed the configuration and Reynolds

number is same. So we can, if we extrapolate then we can have this value of  $f$  that is  $f$  is equal to 0.013, so hence losses due to the fitting will remain same. Only losses due to the friction pipe will reduce and can be determined with the help of this equation that is  $f$  into  $L$  upon  $D$  into  $V$  square upon  $2g$ , so this is comes out to be 0.13 into 60, 0.1023 meter into 5 meter per, into 2, into 9.81 meter per square, so  $h_f$  is 9.72 meter.

(Refer Slide Time: 13:10)

Question 3: How head loss will change for the problem statement 1, if we change the fluid from water to oil with density of 900 kg/m<sup>3</sup> and kinematic viscosity of 0.00001 m<sup>2</sup>/s?

$Re = \frac{Dv}{\nu} = \frac{0.1023 \times 5}{0.00001} = 51150$   
 $Re = 51,150 \approx 5.115 \times 10^4$   
 For Cast iron with  $\frac{\epsilon}{D} = 0.0015$  &  $Re = 5.115 \times 10^4$   
 $f = 0.026$   
 90° elbow with  $K = 0.9$   
 $K_f = \frac{K}{Re} + K_{el} \left(1 + \frac{1}{Re}\right)$   
 $= \frac{0.9}{51150} + 0.2 \left(1 + \frac{1}{51150}\right)$   
 $= 1.76 \times 10^{-5} + 0.2 \times 2.000194$   
 $K_f \approx 0.26$

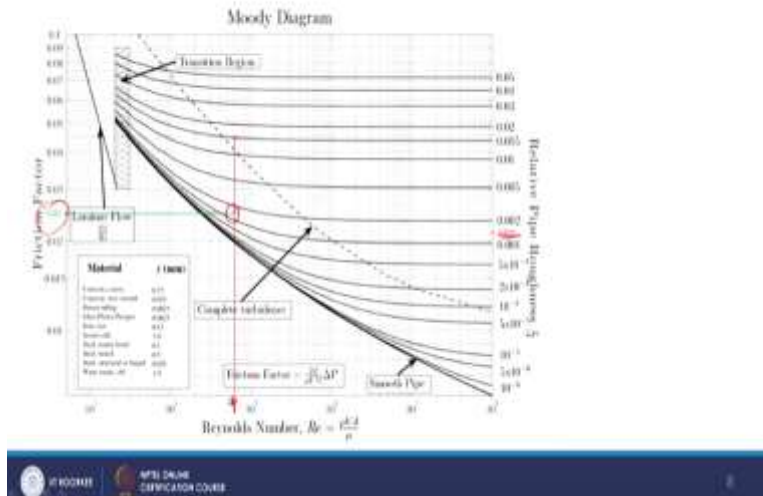
Now, let us have another problem, the problem number 3, the problem says that, “How head loss will change for the problem number 1, if we change the fluid from water to oil which is having the density of 900 kilogram meter cube and kinematic viscosity of 0.00001 meter square per second?” So now we have changed the fluid, so as a Reynolds number depends upon the density and viscosity of the fluid, so it will be changed and hence all the calculation will be changed accordingly.

So let us see that how Reynolds number is having its value 0.1023, we have taken it from problem number 1, 5 meter per second, 0.00001 meter square per second, so Reynolds number is coming out to be 51150 that is roughly 5.15 into 10 to the power 4, now for cast iron with  $\epsilon$  upon  $D$  is equal to 0.0015 and Reynolds number 5.115 into 10 to the power 4, now we have calculated the Reynolds number, so now next aspect is to calculate, to find out the Moody’s chart.

(Refer Slide Time: 14:40)



Figure 4: Determination of friction factor using moody curve



Now this is our Moody's chart, from here we can calculate because we know the roughness (fact) relative pipe roughness which is point 0.0015, here you can see this one and Reynolds number is somewhere in the zone of this one, so the cross section is this one and if we extrapolate the things and it is comes out to be 0.25. So if that is the friction factor is 0.25, now calculation for determining the resistance coefficient for 90 degree elbow with K<sub>1</sub> is equal to 800 and K<sub>infinity</sub> is equal to 0.2.

So K<sub>f</sub> is equal to K<sub>1</sub> upon Re plus 1 plus 1 upon D which is 800 upon 5.115 into 10 to the power 4 plus 0.21 plus 1 upon and 10.23 centimeter upon 2.54 cm. This is 156.4 into 10<sup>-4</sup> plus 2496.58 x 10<sup>-4</sup>, so K<sub>f</sub> is 0.265.

(Refer Slide Time: 16:20)

For 3 elbow  
 $K_f (3 \times 90^\circ \text{ elbow}) = 3 \times 0.265 = 0.795$  --- (3)

For Gate valves  $K_f = 300$  &  $K_{\infty} = 0.10$

$$K_f = \frac{300}{5.115 \times 10^4} + 0.10 \left( 1 + \frac{1}{\frac{10.23 \text{ cm}}{2.54 \text{ cm}}} \right)$$

$$K_f = 58.65 \times 10^{-4} + 1248.29 \times 10^{-4}$$

$$K_f (\text{gate valve}) = 0.131$$
 --- (4)

Total resistance coefficient  
 $K_f (\text{Total}) = 0.795 + 0.131$

$$K_f (\text{Total}) = 0.926$$

Now for 3 elbow, the  $K_f$  3 into 90 degree elbow which is 3 into 0.265, 0.795. For gate valve  $K_f$  is equal to 300 and  $K_{\infty}$  is 0.10. Now  $K_f$  is 300 upon 5.115 into 10 to the power 4 plus 0.10, 1 plus 1 upon 10.23 centimeter, 2.54. Now  $K_f$  is 58.65 into 10 to the power minus 4 plus 1248.29 into 10 to the power minus 4, so  $K_f$  that is gate valve 0.131.

So the total resistance coefficient  $K_f$  total is 0.795 plus 0.131 which is 0.926.

(Refer Slide Time: 18:05)

$$h_v = K_f \frac{v^2}{2g} = \frac{0.926 \times (5 \text{ m/s})^2}{2 \times 9.81 \text{ m/s}^2}$$

$$h_v = 1.180 \text{ m}$$
 --- (6)

$h_f = f \times \frac{L}{D} \times \frac{v^2}{2g}$  (head loss due to friction in pipe)

$$= 0.025 \times \frac{60 \text{ m}}{0.1023 \text{ m}} \times \frac{(5 \text{ m/s})^2}{2 \times 9.81 \text{ m/s}^2}$$

$$h_f = 18.68 \text{ m}$$
 --- (7)
$$\text{Total head loss} = 18.68 + 1.18 = 19.86 \text{ m}$$

Ans.

Now the head loss due to the fitting can be calculated as  $h_L$  is equal to  $K_f V^2$  upon  $2g$ , this is 0.926 we have calculated earlier into 5 divided by 2 into 9.81 meter per second square, so that is  $h_L$  is equal to 1.180 meter.

Now head loss due to friction in pipe can be calculated as  $h_f$  is equal to  $f$  into  $L$  upon  $D$  into  $V^2$  upon  $2g$ , this is the head loss due to friction in pipe which is 0.025 into 60 meter upon 0.1023 meter into 5 meter per second square upon 2 into 9.81 meter per second square.

So  $h_f$  is 18.68 meter, so that total head loss is 18.68 plus 1.18 is equal to 19.86 meter, this is what we required. Now it is to be noted that in above mentioned 3 problems if we did not consider the loss of head due to the kinetic energy change, so try to calculate the head loss for the problem number 3 using the case for a smooth pipe.

(Refer Slide Time: 20:00)

Question 4: Liquid in a tank is to be boiled through a 2-cm diameter nichrome coil electrical resistance heating device. Determine the maximum heat flux that can be attained in the nucleate boiling regime and the surface temperature of the heater surface for the case.

Given Data:

Properties of liquid at its boiling point (100 deg. C)

$\sigma = 0.0589 \text{ N/m}$

Density of liquid  $\rho_l = 957.9 \text{ kg/m}^3$


Density of vapour  $\rho_v = 0.6 \text{ kg/m}^3$

Prandlt No. for liquid  $Pr_l = 1.75$

Enthalpy of vaporization  $h_{fg} = 2257.0 \times 10^3 \text{ J/kg}$

Viscosity of the liquid  $\mu_l = 0.282 \times 10^{-3} \text{ Kg.m/s}$  ←

Specific heat of liquid  $C_{p,l} = 4217 \text{ J/kg. } ^\circ\text{C}$



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So the next problem is related to the pool boiling, now here is the problem that a liquid in a tank is to be boiled through a 2 centimeter diameter nichrome coil electrical resistance heating device. Now you need to determine the maximum heat flux that can be attained in the nucleate boiling regime and the surface temperature of the heater surface for the case. Now there are sufficient data given that is the properties of the liquid at its boiling point 100 degree Celsius.

Small sigma is equal to 0.0589 Newton per meter, density of the liquid  $\rho_l$  is 957.9 kilogram per meter cube, density of the vapor  $P V$  is equal to 0.6 kilogram per meter cube, Prandlt number for

liquid Pr<sub>1</sub> is 1.75, enthalpy of vaporization h<sub>fg</sub> is 2257.1 into 10 to the power 3 joule per kilogram, viscosity of the liquid mu<sub>f</sub> is 0.282 into 10 the power minus 3 kilogram meter per second, the specific heat of the liquid C<sub>p1</sub> is 4217 joule per kilogram degree Celsius.

(Refer Slide Time: 21:16)

Approx.  $C_{sf} = 0.006$  and  $n = 1.0$  for the boiling of liquid on nickel plated (Nichrome) surface

$L = r = 0.01 \text{ m}$

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And approximately  $C_{sf}$  is 0.006 and  $n$  is equal to 1.0 for the boiling of the liquid on nickel plated nichrome surface, so let us take the problem. This is the figure of the pool boiling here, this is the 2 centimeter electric supply, this is the supplied electric supply, 100 degree Celsius, now  $P$  is 1 atmosphere. Now the heating element in the case can be considered to be the short cylinder whose characteristic dimensions in its radius that is  $L$  is equal to  $r$  is equal to 0.01 meter. The dimensionless radius  $L^*$  and the constants here can be determined from the table which I am going to draw.

(Refer Slide Time: 22:24)

Heater Geometry	$C_0$	L	Range of $L^*$
Large horizontal flat heater	0.149	width or dia	$L^* > 27$
Small horizontal flat heater	18.9 K <sub>1</sub>	width or dia	$9 < L^* < 20$
Large horizontal cylinder	0.12	radius	$L^* > 1.2$
Small horizontal cylinder	$0.12 L^* - 0.45$	radius	$0.15 < L^* < 1.2$
Large Sphere	0.11	radius	$L^* > 4.26$
Small Sphere	$0.227 L^* - 0.45$	radius	$0.15 < L^* < 4.26$

$$K_1 = \frac{5}{9(L_1 - P_1)} A_{heater}$$

$$L^* = L \left( \frac{g(L_1 - P_1)}{\sigma} \right)^{1/2}$$

$$= 0.01 \left( \frac{9.81(9578 - 0.6)}{0.0589} \right)^{1/2} = 4.08$$

$$L^* > 1.2$$

Now here the heating, this is very important table, heater geometry L, range of L star, now here the large horizontal flat heater 0.149, or dia, small horizontal flat heater which is our case 18.9, large horizontal cylinder 0.12, in this case we will consider radius. This L star is greater than 1.2. Small horizontal cylinder 0.12 L star, radius 0.15, L star 1.2. Large sphere 0.11, 4.26 L star is greater than 4.26. Small 0.227 LS star, fine less than 4.2 size. So if we calculate the K1 which is  $g P L - P V$ , A of heater. L star is  $L g P L - P V$ , half which is 0.01, 9.81, 0.6 going to the power half which is, now if we compare with this table the L star is coming out to be the greater than 1.2.

(Refer Slide Time: 25:30)

Handwritten derivation for critical heat flux:

$$C_{cr} = 0.12$$

$$\dot{q}_{max} = C_{cr} h_{fg} \left[ \sigma g P_L^2 (P_L - P_V) \right]^{1/4}$$

$$= 0.12 (2257 \times 10^3) \left[ 0.0589 \times 9.8 \times (0.6)^2 (957.9 - 0.6) \right]^{1/4}$$

$$\dot{q}_{max} = 1017411 \text{ W/m}^2$$

$$\dot{q}_{nucleate} = h_{c,1} h_{fg} \left[ \frac{g(P_L - P_V)}{\sigma} \right]^{1/4} \left[ \frac{C_{FL}(T_s - T_{sat})}{C_{SF} h_{fg} \rho_{lc}} \right]^{3/4}$$

$$1017411 = (0.282 \times 10^3) \times (2257 \times 10^3) \times \left[ \frac{9.81(957.9 - 0.6)}{0.0589} \right]^{1/4} \times \left[ \frac{4217(T_s - 100)}{0.013(2257 \times 10^3)^{1.75}} \right]^{3/4}$$

Now which corresponds to  $C_r$  is equal to 1.2, then the maximum or critical heat flux is you can determine from the equation which we have discussed previously  $h_{fg} g P V^2 (P_L - P_V)$  to the power 1 by 4 which is 0.12, 2257 into 10 to the power 3, 0.0589 into 9.8 into 0.6 square into 957.9, 0.6 to the power 1 by 4. So  $Q_{max}$  is 1017411, so once our radiation which gives the nuclear boiling heat flux for a specified surface temperature can also be used to determine the surface temperature when the heat flux is given.

Now if you substitute the maximum heat flux into the following equation that is this one,  $\rho V, T S$  minus  $T$  and to the power 3, so it is 1017411 is equal to 0.282 into 10 to the power minus 3 into 2257 into 10 to the power 3 into 9.81, 957.9 minus 0.0589 to 1 by 2, multiplied by 4217  $T S$  minus 100 upon 0.013 into 2257 into 10 to the power 3, 1.75 whole cube.

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$$3.996 = \left[ \frac{4217 (T_s - 100)}{0.013 (2257 \times 10^3)^{1.75}} \right]^3$$
$$T_s = 100 + \frac{(3.996)^3 \times 0.013 \times 2257 \times 10^3 \times 1.75}{4217}$$

$T_s = 119.3^\circ\text{C}$  Ans.

So this is 3.996, 4217 T S minus 100, 0.013 2257 into 10 to the power 3 into 1.75 whole cube, so T S is equal to 100 plus 3.996 into 0.013 into 2257 into 10 to the power 3 into 1.75, 4217, so T S is coming out degree Celsius this is the desired result. So, by this way we have discussed 4 different problems related to the source model and I hope that these problems, because these problems are very elaborative I hope that these problems are helpful for understanding of this particular chapter. Thank you very much.