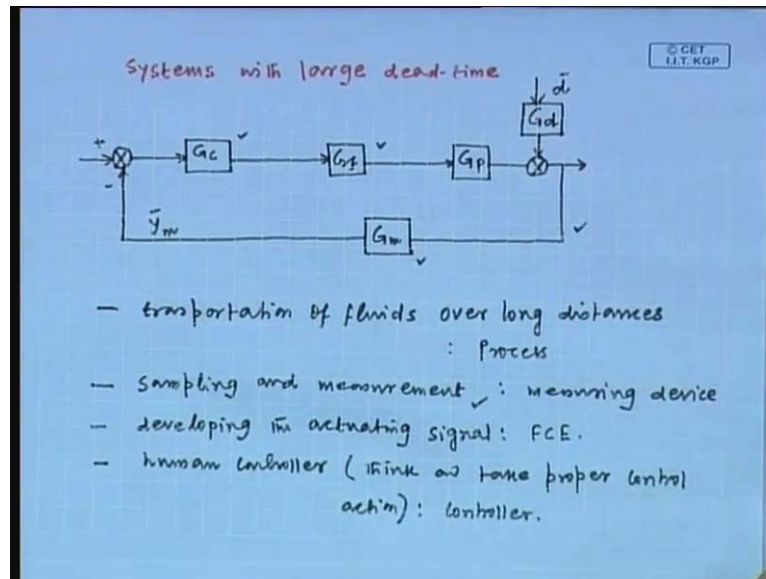


Process Control and Instrumentation
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Lecture - 30
Advanced Control Schemes

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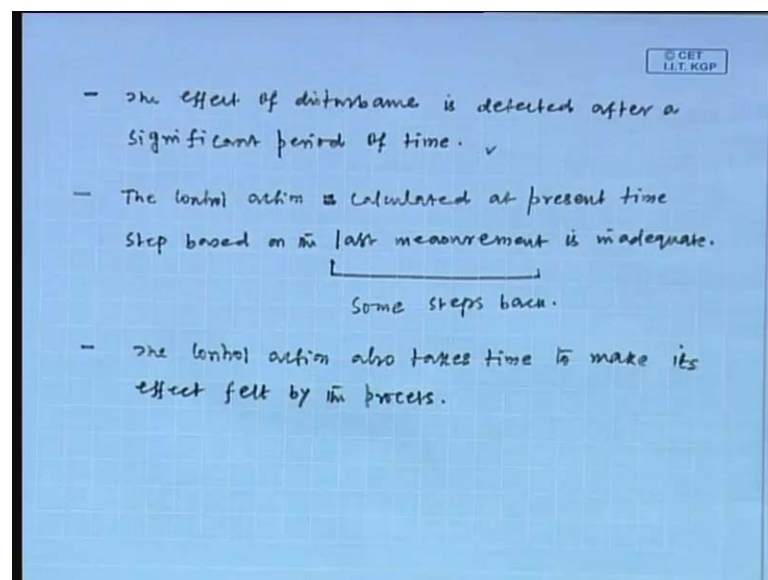
So, for we discussed a number of interesting topic under process control part and in the next phase we will discuss advanced control schemes under advanced control schemes today we will discuss this system with large dead time. So, our topic is systems with large dead time fine this under advanced control schemes now a closed loop process consist of a number of dynamic elements like the process the measuring device the controller the final control element a closed loop process includes a number of elements one is the controller the final control element the process, and the measuring device.

So, if we connect all these elements we obtain the close loop block diagram this is the close loop block diagram almost all the dynamic element may include time delays almost all the dynamic element may include time delays like time delay is involved in transportation of fluids over long distances see here controller final control element process and measuring device construct the close loop block diagram all this dynamic element may include time delays like time delay is involved in transportation of fluids transportation of fluids over long distances fine time delays like time delay is involved in transportation of fluids over long distances this is related to which element this is related

to the process time delay is involved in sampling. And measurement time delay is involved in sampling and measurement it is obvious in this close loop block diagram that the controller needs output information y that the controller needs output information y now that information we can get by collecting the sample and that and measuring that sample fine. So, that takes some time therefore, we can say that the time delay is involved in sampling and measurement of y .

So, this is related to which element measuring device this is related this is related to measuring device similarly time delay is involved in deviling the actuating signal time delay is involved in deviling the actuating signal and this is related to the final controlling element this is related to the final controlling element time delay is involved in human controller time delay is involved in human controller to thing and take proper control action to thing and take proper control action and; obviously, this related to the controller now in all of the situations noted above the feedback controller would provide unsatisfactory performance due to the presence of dead time in all of the above situations the feedback controller would provide unsatisfactory performance.

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Due to the presence of dead time and we will discuss few other reasons which are related to dead time like the effect disturbance is detected after a long period of time. So, the effect disturbance is detected after a significant period of time why the controller provide unsatisfactory performance, we are discussing the reasons first reason is the effect of

disturbance is detected after a significant period of time second reason is the controller calculated the control action at present time step based on the last measurement is not adequate next is the controller or the control action calculated at present time step based on the last measurement is in adequate see the controller calculates the the control action at present time step based on the last measurement originally, it is not last measurement due to the dead time this is actually the measurement some steps back. So, this originally not last measurement this is some steps back measured signal which is some steps back therefore, the control action is not adequate for the existing situation third one is the control action. Also takes time to make its effect felt by the process these are the reasons for which the feedback controller would provide unsatisfactory response.

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Example

$$G_{OL} = \frac{K_c e^{-t_d s}}{0.1s + 1} \quad \text{or}$$

t_d	$\omega_{CO} \checkmark$	$K_u \checkmark$
0.123	17 rad/min	2
0.236	10	1.4
0.535	5	1.1

With the increase of t_d

- ω_{CO} decreases: the closed-loop response is sensitive to lower-frequency periodic disturbances.

In the next we will take one example to observe the effect of dead time in the next we will consider one example to investigate the effects of dead time. Consider the open loop transfer function given as $K_c e^{-t_d s} / (0.1s + 1)$ this is the open loop transfer function which we considered earlier fine now t_d cross over frequency and ultimate gain if t_d is 0.123, as we considered earlier we can easily calculate the cross over frequency seventeen radian per minute if t_d is 0.123 minute then the cross over frequency is 17 radian per minute this we can calculate and the corresponding ultimate gain is 2 similarly if we increase t_d 0.236 then the cross over frequency is reduced to 10 radian per minute and ultimate gain is also reduced to 1.4, if we increase further t_d to suppose 0.535 then the cross over frequency is decreased to 5

and the ultimate gain is also decreased to 1.1, this datum. We obtain based on this open loop transfer function fine now it is very obvious that with the increase of t_d the cross over frequency and ultimate gain both decreased. So, with the increase of t_d the cross over frequency is decreases it means the close loop response is sensitive to lower frequency periodic disturbances it means the the close loop response is sensitive to lower frequency periodic disturbances this is the first observation. Secondly, with the increase of t_d ultimate gain decreases.

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0.236	10	1.4
0.535	5	1.1

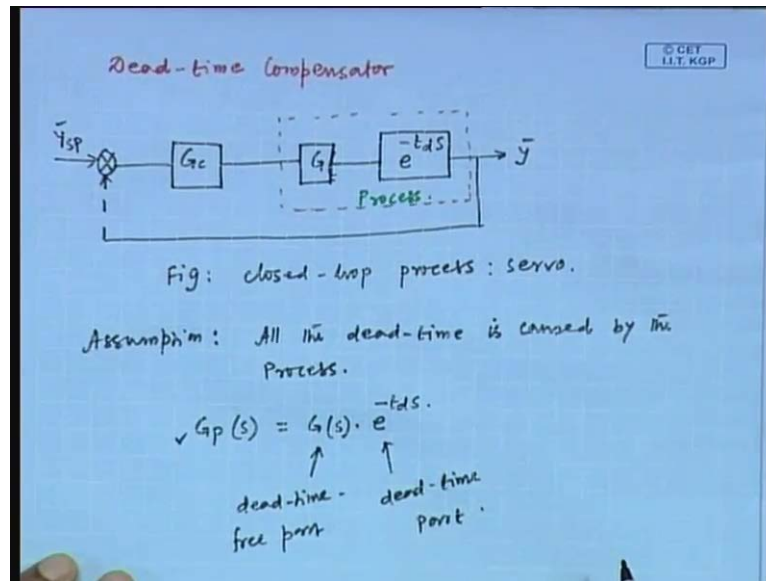
With the increase of t_d

- ω_{co} decreases: the closed-loop response is sensitive to lower-frequency periodic disturbances.
- K_u decreases: need to reduce K_c to avoid instabilities.
 - Sluggish.

That means, we are force to select K_c values low low values we have to select I mean we have to we need to reduce K_c to avoid instabilities fine because the ultimate gain is the maximum values we can take. So, if K_u decreases we have to select lower value. So, if we select lower K_c value then what happen? The response becomes shiggish. So, if we decrease K_c value then the close look response becomes sluggish. So, these are the effect of dead time. So, due to the presence of dead time we observe that the close loop response or the controller performance is degraded therefore, there is a need to devise a dead time compensative, fine.

So, our next topic is the derivation of dead time compensative dead time compensative for deriving the dead time compensator we will consider the simple close look block diagram the block diagram looks like this, this is the controller block we are assuming $G_f G_m$ both are one this is the process fine this the process this the process.

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So, this is the closed loop process for servo case fine now we assume here that all the dead time is caused by the process here we assume here that all the dead time is caused by the process fine usually we consider we we denote G_p to represent the process. So, here G_p is equal to G_s multiplied by exponential of minus $t_d s$ this is the dead time free part G_s is the dead time free part and this is the dead time part fine this is the transfer function of the process which we consider in this block diagram.

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$$G_{OL} = \frac{\bar{y}_m}{\bar{y}_{sp}} = G_p G_f G_m G_c \quad \checkmark$$

$$\checkmark \bar{y} = G_c [G_s e^{-tds}] \bar{y}_{sp} \quad \checkmark$$

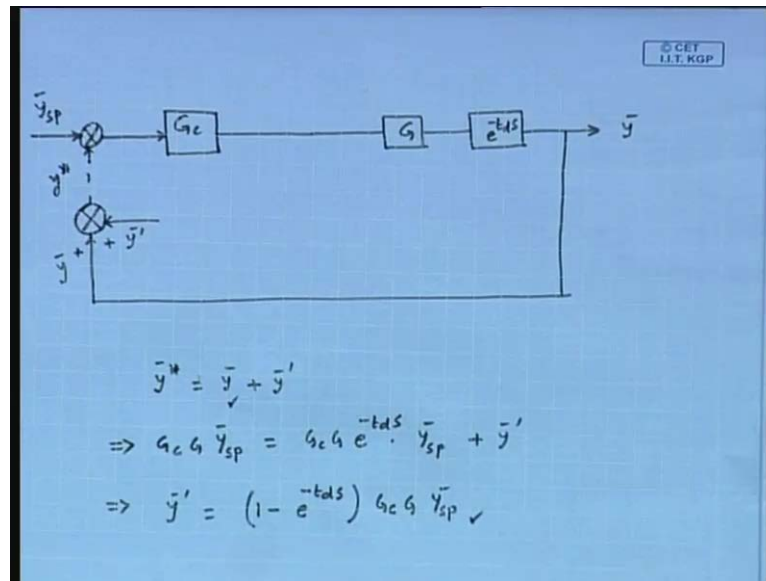
✓ Aim: Remove e^{-tds} from the input signal to the compensator. ✓

$$\bar{y}^* = G_c G_s \bar{y}_{sp} \quad \checkmark$$

Now, the open loop response we write $G_o l$ which is equal to y_m bar divided by $y_{set\ point}$ bar, and this is written as $G_p G_f G_m G_c$ multiplication of four individual transfer functions this is the generalized form of open loop transfer function fine we know all these individual transfer functions. We know the expression for all these transfer functions substituting those we obtain y_m bar here y_m bar and y bar both are identical because we are considered G_m equals one then we obtain $G_c G_s$ exponential of minus t_d $y_{set\ point}$ bar this is the expression for output response y , if we substitute all these individual transfer functions in this equation we obtain this now it is obvious that this is delayed by time t_d it is obvious from this expression that it is delayed by t_d minute. Now what is our aim basically this information y bar is feed back to the comparator our target is to give dead time free response to this comparator. So, that the controller is not effected fine. Now, if we consider this is the ideal response I mean if we represent anyways first I am just mentioning the aim aim is to remove this dead time part from the input signal to the comparator this is our aim aim is to remove this dead time part from the input signal to the comparator, fine.

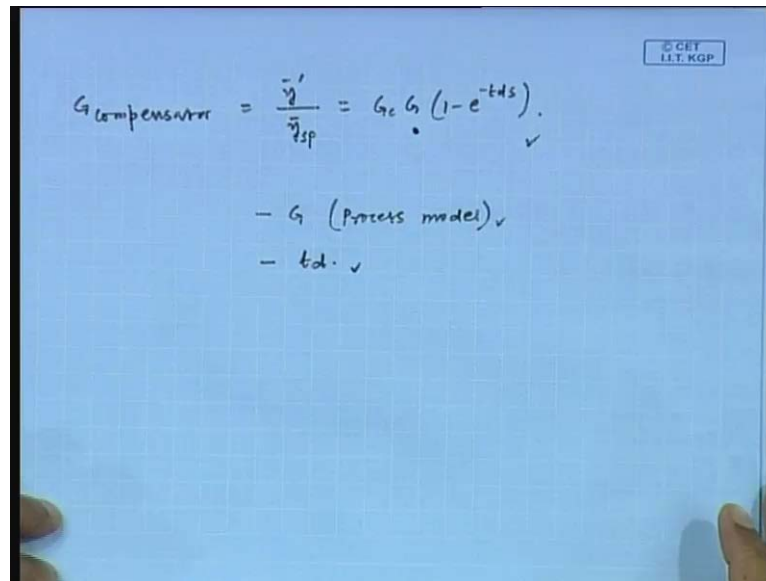
So, ideally the response should be $G_c G y_{set\ point}$ bar agree this response includes this dead time part we have to devise the comparator. So, that the input signal to the comparator does not include this dead time part. So, if we represent the ideal response by y^* then y^* should be $G_c G y_{set\ point}$. Now how we can obtain this y^* we can obtain this y^* this suppose y^* and the information which is coming from process to deduct measurement that is y bar we are just adding some thing with this y bar.

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So, that we can obtain y^* anyway I am just drawing this figure this is G_c this is our process that includes dead time part this \bar{y} this is comparator y_{sp} our target is to give y^* information to the comparator, but originally we getting \bar{y} fine, but we need to give dead time free response to this comparator. How that is possible suppose we adding another signal denoted by \bar{y}' with this \bar{y} to obtain y^* it is straight forward to calculate this \bar{y}' I mean y^* is basically if the summation of \bar{y} and \bar{y}' y^* is known to us that is $G_c G$ multiplied by y_{sp} y is $G_c G e^{-tds}$ multiplied by y_{sp} what will be the \bar{y}' that we calculate that will be if one minus exponential minus tds multiplied by $G_c G y_{sp}$. So, if we add this expression with the expression of \bar{y} then only we get dead time free response which can be feedback to the comparator fine.

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The image shows a handwritten equation on a blue grid background. The equation is $G_{\text{compensator}} = \frac{y'}{y_{sp}} = G_c G_p (1 - e^{-t_d s})$. Below the equation, there are two lines of text: "- G (Process model) ✓" and "- t.d. ✓". In the top right corner, there is a small logo that reads "© CEY I.I.T. KGP".

So, we obtain the block diagram for the compensator. The compensator output is y' and input is y_{sp} which is represented by $G_c G_p (1 - e^{-t_d s})$. Now in this expression G and t_d are involved. Now G is the process model and t_d is the dead time. So, t_d is the dead time. So, the dead time compensator depends on the accuracy of process model as well as the value of dead time. This dead time compensator depends on the process model represented by G . This also depends on the value of t_d . We can get perfect compensation only if G and t_d are exactly known. We can get perfect compensation if the process model and t_d value are exactly known, but in practice this is never known exactly.

So, they are known approximately. In practice, both the process model and t_d are known approximately. If that is the case, then we will represent the approximate model as G' and the approximate t_d as t_d' . We will represent the G' for the approximate model and t_d' for approximate dead time. Then what will be if y^* we know y^* is \bar{y} which is which is the actual process output plus y' which is the output of dead time compensator. \bar{y} is $G_c G_p (1 - e^{-t_d s})$.

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$$G_{\text{compensator}} = \frac{\bar{y}'}{\bar{y}_{sp}} = G_c G_p (1 - e^{-tds})$$

- G (Process model)
- td .

$$\bar{y}^* = \bar{y} + \bar{y}'$$

$$= [(G_c G_p e^{-tds}) + (1 - e^{-tds}) G_c G_p'] \bar{y}_{sp}$$

$$\bar{y}^* = G_c [G_p' + (G_p e^{-tds} - G_p' e^{-tds})] \bar{y}_{sp}$$

This \bar{y} and how much will be \bar{y}' if G and td are not exactly from this equation from this block \bar{y}' will be one minus exponential of minus td prime s multiplied by $G_c G_p'$ and this will be multiplied by \bar{y}_{sp} agree just in this expression of \bar{y}' we considered td prime and G_p' in place of td and G_p rearranging this we obtain G_c multiplied by G_p' plus G_p exponential of minus td s minus G_p' exponential of minus td prime s \bar{y}_{sp} rearranging we obtain this expression for \bar{y}^* .

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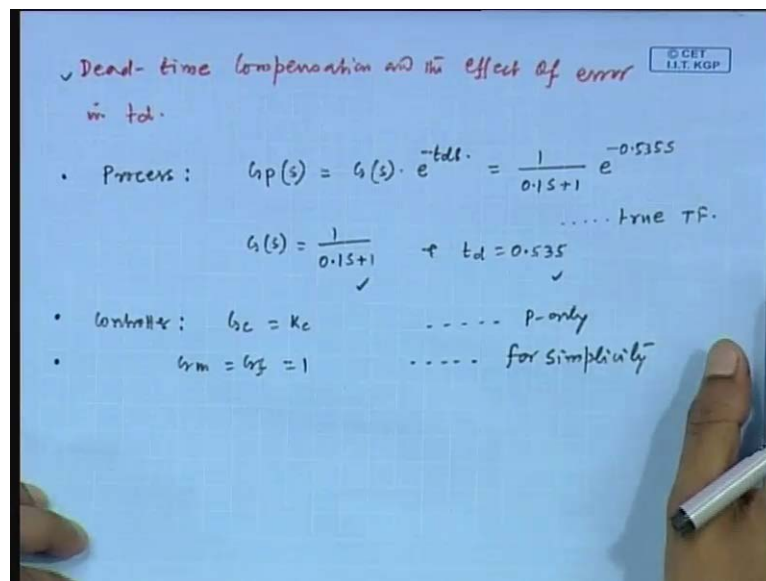
Remarks

- (1) $G = G'$ $td = td'$
- (2) $(G - G') + (td - td')$
- (3)
- (4)

So, what conclusion we can make based on these expression when we get perfect compensation we can get perfect compensation when G equal to and t_d equal to t_d prime I mean when the process model and dead time value exactly known then only we can get perfect dead time compensation the larger the modeling error that is G minus G prime and t_d minus t_d prime that less effective is the compensation fine third remark which one is more crucial

on the dead time compensation I mean error in G or error in t_d has sever effect on dead time compensation t_d because it is related to exponential term. So, this is more crucial to the dead time compensation because of the exponential function fourth remark see In this dead time compensator expression t_d is considered as constant quantity, but some time the t_d is vary with the time some time the dead time t_d may vary with time that is not taken into the account in the dead time compensator expression. So, that is you can say that is a draw back.

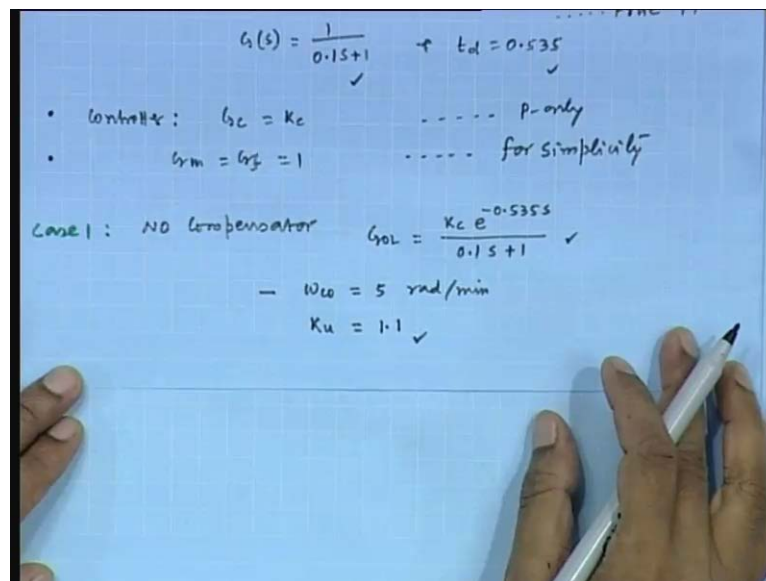
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Next we will discuss the dead time compensation and the effect of error in t_d next we will discuss the dead time compensation and the effect of error in t_d we observe that there may the possibility of the error in both G and t_d , but we are considering only error in t_d . So, for this purpose we are consider a process represented by G s exponential of minus t_d s which is again $0.1s$ plus 1 exponential minus 0.535 s fine to discuss this topic we are considering this trans verve function to represent the process see this the trans

transfer function of the true process I mean this is $G(s)$. So, this true transfer function of the process; obviously, $G(s)$ is 1 divided by 0.1 s plus one and t_d is 0.535 it is obvious that $G(s)$ is 1 divided by 0.1 s plus 1 and t_d is 0.535 we will consider P only controller. So, controller transfer function, we can write as K_c this P only controller we will consider here both G_m and G_f equal to one for simplicity, we are considering G_f G_m and G_f both are one.

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Now we will consider three different cases in the first case we will consider there is no compensator case 1 in case 1 we consider no compensator fine the open loop transfer function which we are considering is represented by K_c exponential minus 0.535 s divided by 0.1 s plus 1 fine this is given to us. So, based on this transfer function we can calculate the corresponding cross over frequency that is 5 radian per minute.

In fact, this problem we discussed previously and we obtain the cross over frequency is 5 radian per minute corresponding ultimate gain is 1.1 fine. So, what value for K_c we can select we have to select the K_c value lower than 1.1 fine this is the ultimate value at which we get sustain oscillation.

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$$G(s) = \frac{1}{0.1s+1} \quad \rightarrow \quad t_d = 0.535 \checkmark$$

- Controller: $G_c = K_c$ ----- P-only
- $G_m = G_f = 1$ ----- for simplicity

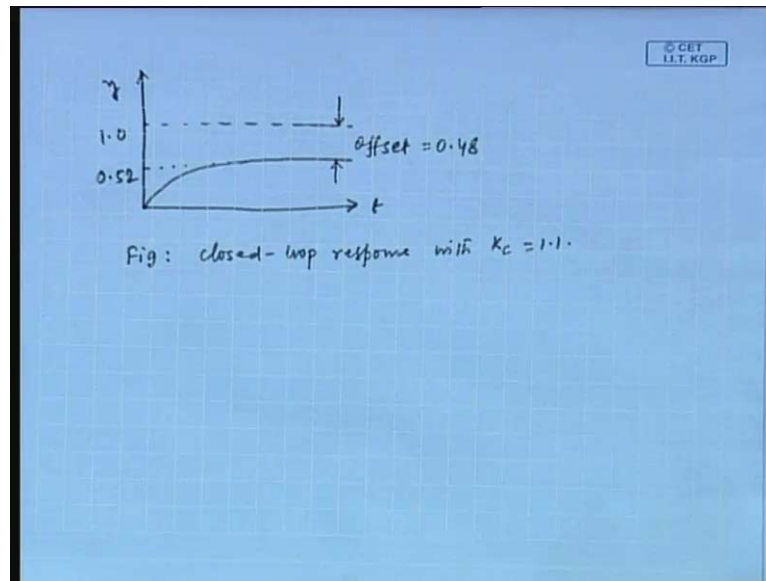
Case 1: NO Compensator $G_{OL} = \frac{K_c e^{-0.535s}}{0.1s+1} \checkmark$

- $\omega_{co} = 5 \text{ rad/min}$
- $K_u = 1.1 \checkmark$

- $K_c < 1.1 \checkmark$ (stable response)
- $\text{offset} = \frac{1}{1+k_p K_c} = \frac{1}{1+1 \cdot 1.1} = 0.48$

So, for stability if we K_c should be less than 1.1 for stable response, we have select K_c value lower than 1.1 if that is the case what will be the offset maximum holus we can select for K_c that is 1.1 fine at which we obtain sustain oscillation if that is the case what will be the offset the expression for offset for the example system we can derive as 1 divided by 1 plus $K_p K_c$ this is the expression for the offset. For the example system fine. So, if we substitute the values then we obtain 1 divided by 1 plus how much is K_p one how much is K_c , suppose we are considering 1.1 then it is 0.48 this is the large offset if we consider the maximum holus of K_c that is 1.1 then we obtain the offset 0.48 say this is time this y this is suppose one this is the offset which is equal to 0.48.

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That means, the corresponding value is 0.52 this is closed loop response with K_c equal to 1.1 fine. So, this case 1 if we do not use dead time compensator and if we use maximum permissible K_c value that is 1.1 we obtain offset 0.48.

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Next, we will consider case 2 and that is perfect compensation in case 2, we consider perfect compensation in case 2, we consider perfect compensator it is true when G' is G prime is equal to G perfect compensation is possible when G' prime is equal to G ; that means, process model is perfectly known. Secondly, this t_d' is t_d ; that means, the

dead time value is exactly known for the case of perfect compensation y^* by y set point is equal to $G_c G_I$ mean there will be not dead time part. So much is this $1/K_c$ divided by $0.1\text{ s} + 1$ agree there is no dead time part. So, for perfect compensation we get y^* by y set point equal to k_c divided by $0.1\text{ s} + 1$ now this is first order transfer function this is a transfer function of a first order system K_c by $0.1\text{ s} + 1$ is there any cross over frequency no because if we vary ω from 0 to infinity we obtain the variation ϕ from 0 to minus 90. So, there is no existence cross over frequency if there is no existence of cross over frequency then we can use any large value of K_c we can use any large value of K_c there will not be instability problem fine since there is no existence cross over frequency. So, we can use any large value of K_c suppose this value is hundred since there is no instability problem we are considering large value of K_c and that is 100.

If K_c is 100 what will be the offset 1 divided by $1 + K_p K_c$ that is 0.01 fine without using dead time compensator we got 0.48 we using perfect compensator we are getting insignificant offset this is insignificant offset fine, but we mentioned that in practice g and t_d they are not known exactly.

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Case 3: Imperfect Compensator

$K_p = 1$ } perfectly known.
 $\tau_p = 0.1$ }
 $G' = G = \frac{1}{0.1s + 1}$

Dead-time - not perfectly known.
 $t_d = 0.535$... time value
 $t_d' = 0.3$... approximated value

Stability ($K_c = 100$)

- Residual dead-time = $(0.535 - 0.3) = 0.235$
- Uncompensated dead time
 $W_{c0} = 10 \text{ rad/min}$
 $K_u = 1.4$

So, that will consider in third case in case three we consider imperfect compensator, imperfect compensator fine process gain for the example system is one time constant for the example system is 0.1 suppose these are perfectly known; that means, G prime equal

to G equal to 0.1 s plus 1 this is the K_p which is 1 this τ_p which is 0.1 the dead time, that is not perfectly known this is not perfectly known to us true value of this dead time 0.535 this is true value and approximate value is 0.3 fine original value is 0.535, but we have the design of the compensator based on this value of 0.3 now question is we are using K_c value 100, we know that our compensator is perfect question is what about stability when K_c equal to 100 what about stability when K_c equal to 100 original value is 0.535 we are using 0.3. So, how much is the residual dead time residual dead time is 0.535 minus 0.3 that is 0.235, that is residual dead time it is clear that the compensation is not perfect.

Now uncompensated dead time corresponds to ω_c not 10 radian per minute fine uncompensated dead time corresponds to K_u 1.4 at beginning of today class we got these values fine for the dead time of 0.235, we obtain cross over frequency 10 radian per minute and ultimate gain 1.4, but how much we are using what value of K_c we are using K_c equal to 100, but maximum I mean I mean limit is up to 1.4 what about stability unstable system becomes unstable fine. Now conclusion is be conservative in selecting the K_c value for the process having dead time be conservative for selecting the K_c value for the process having dead time.

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Dead-time - not perfectly known.
 $t_d = 0.535$... true value
 $t_d' = 0.3$... approximated value

Stability ($K_c = 100$),

- Residual dead-time = $(0.535 - 0.3) = 0.235$
- Uncompensated dead time
 $\omega_c = 10$ rad/min
 $K_u = 1.4$

- System becomes ~~unstable~~ unstable.

So, this is all about the smith predictor and dead time compensator. In the next class, we will discuss the another compensator that is inverse response compensative.

Thank you