

Process Control and Instrumentation
Prof. A. K. Jana
Department of Chemical Engineering
Indian Institute of Technology, Kharagpur

Lecture No. # 19
Feedback Control Schemes

In the last class we are discussing in the P control of a second order process, in the last class we are discussing of proportional control of a second order process we will continue the topic today.

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P Control of a Second-order Process: Servo problem.

Process: $G_p = \frac{K_p}{\tau^2 s^2 + 2\zeta\tau s + 1}$ Second-order process

Controller: $G_c = K_c$ P-only

$G_m = G_f = 1$

CLTF: $\bar{y} = \frac{G_c G_p G_f}{1 + G_c G_p G_f G_m} \bar{y}_{sp} + \frac{G_d}{1 + G_c G_p G_f G_m} \bar{d}$

$\Rightarrow \bar{y} = \frac{K_c K_p / (\tau^2 s^2 + 2\zeta\tau s + 1)}{1 + K_c K_p / (\tau^2 s^2 + 2\zeta\tau s + 1)} \cdot \bar{y}_{sp}$ Servo

$= \frac{K_c K_p}{\tau^2 s^2 + 2\zeta\tau s + (1 + K_c K_p)} \cdot \bar{y}_{sp}$

So, p control of a second order process and we will consider servo test, basically you want to see the closed loop response of a second order process under P only control. So, what are the different element of closed loop process, first element is the process that is basically open loop process. The open loop process with second order system has the transfer function G_p and that is equal to K_p divided by tau square s square plus 2 zeta tau is plus 1.

This is the transfer function of base a second order process, next element is the controller we are interested to observe the close loop performance under the P only control.

Accordingly, the controller equation has the form of G_c equals K_c , this is the transfer function of P only controller, for simplicity will consider G_m and G_f both are equal to unity.

The transfer function of the measuring device that is the G_m and the transfer function of the final control element that is G_f or equal to 1, now what is the expression general expression for closed loop transfer function. General expression of close loop transfer function is represented as \bar{y} equals $G_c G_p G_f$ divided by $1 + G_c G_p G_f G_m$ y set point bar plus G_d divided by $1 + G_c G_p G_f G_m d$ bar, this is the general form of closed loop transfer function.

Now, considering or substituting all the individual transfer function in this generalized form, we obtain \bar{y} equals $K_c K_p$ divided by $\tau^2 s^2 + 2\zeta\tau s + 1$ whole divided by $1 + K_c K_p$ by $\tau^2 s^2 + 2\zeta\tau s + 1$ multiplied by y set point bar. And this is the close loop transfer function for servo problem. Substituting all the individual transfer function $G_c G_p G_f G_m$ in the generalized form of this close loop transfer function, we obtain this form for the servo vocals. Now, if we simplify it than we get $K_c K_p$ divided by $t^2 s^2 + 2\zeta\tau s + 1 + K_c K_p$ multiplied by y set point bar.

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$$\Rightarrow \bar{y} = \frac{K_c K_p / (\tau^2 s^2 + 2\zeta\tau s + 1)}{1 + K_c K_p / (\tau^2 s^2 + 2\zeta\tau s + 1)} \cdot \bar{y}_{sp} \dots \text{Servo}$$

$$= \frac{K_c K_p}{\tau^2 s^2 + 2\zeta\tau s + (1 + K_c K_p)} \cdot \bar{y}_{sp}$$

$$\bar{y}(s) = \frac{K_p'}{(\tau')^2 s^2 + 2\zeta'\tau' s + 1} \cdot \bar{y}_{sp}$$

$$\tau' = \frac{\tau}{\sqrt{1 + K_p K_c}}, \quad \zeta' = \frac{\zeta}{\sqrt{1 + K_p K_c}}, \quad K_p' = \frac{K_p K_c}{1 + K_p K_c}$$

Notes

- ① No change of order.
- ② $\tau' < \tau$
 $\zeta' < \zeta$. more oscillatory. ✓

If we divide numerator and denominator by $1 + K_c K_p$, dividing numerator and the denominator by $1 + K_c K_p$, than we obtain $\bar{y}(s)$ equals K_p prime divided by τ

prime square s^2 plus $2\zeta\tau' s$ plus 1 multiplied by y set point. But, dividing numerator and the denominator $1 + K_c K_p$, we obtain this form higher τ' prime equals τ Divided by root over of $1 + K_p K_c$. ζ prime equals ζ divided by root over of $1 + K_p K_c$ and prime K_p prime equals $K_p K_c$ divided by $1 + K_p K_c$.

These are the expression with which happen used in this closed loop form, now you will conclude, so due to the inclusion of proportion controller with the second order process, the order does not change the order remains same. So, we will note down few points due to the inclusion of proportion controller there is no change of the order overall system, this is the first point.

Secondly, if we see the expression of τ' prime we can say that τ' prime less than τ , secondly ζ prime less than ζ it means overall response under P only control becomes more oscillatory, because damping factor is decreased. As a result the overall response under the p control becomes more oscillatory in the last class we discuss up to this.

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Offset

$$\bar{y} = \frac{K_p'}{(\tau')^2 s^2 + 2\zeta\tau' s + 1} \cdot \bar{y}_{sp} \dots \text{servo.}$$

$$\bar{y}_{sp} = \frac{A}{s} \checkmark$$

$$\bar{y} = \frac{K_p'}{(\tau')^2 s^2 + 2\zeta\tau' s + 1} \cdot \frac{A}{s}$$

$$\text{ultimate response} = \lim_{s \rightarrow 0} s \bar{y}(s) = \lim_{s \rightarrow 0} \frac{K_p' A}{(\tau')^2 s^2 + 2\zeta\tau' s + 1}$$

$$= K_p' A \checkmark$$

$$\text{Offset} = \text{new setpoint} - \text{ultimate value of the response}$$

$$= A - K_p' A = A(1 - K_p') = A \left(1 - \frac{K_p K_c}{1 + K_p K_c}\right)$$

$$= \frac{A}{1 + K_p K_c}$$

So, next will determine the offset, now out tools loop transfer function is y bar equals K_p prime divided by τ' prime s^2 plus $2\zeta\tau'$ prime s plus 1 y set point bar, this is the close loop transfer function we which we derived for servo case. Now, we will consider a step change in set point with magnitude A , we use A to determine the offset

introducing a step change for set point with magnitude A. So, considering this y set point equal A by s that close loop transfer function, becomes K_p prime divided by τ prime square s square plus 2 zeta prime τ prime s plus 1 multiply by A by s.

Now, according to final value theorem the ultimate response we determine as limit s tends to 0 s y bar s, so limit s tends to 0 K_p prime A divided by τ prime s square plus 2 zeta prime τ prime s plus 1, which is equal to K_p prime A. The ultimate response for the case of second order system under P only control with the introduction of set p change point with the magnitude A.

We obtain K_p prime A than what about of offset we know the offset equals new set point minus ultimate value of the response to new set point is A ultimate value is K_p prime A. , so A multiplied by 1 minus K_p prime. Now, we will substitute the expression of K_p prime, so A 1 minus K_p prime is $K_p K_c$ divided by 1 plus $K_p K_c$, what is the expression of K_p prime, K_p prime is $K_p K_c$ divided by 1 plus $K_p K_c$. So, the offset becomes A divided by 1 plus $K_p K_c$, this is the offset, now when the offset becomes 0 or when offset approach is to 0 ,when K_c is to infinity.

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$$\begin{aligned} \text{Offset} &= \text{new setpoint} - \text{ultimate value of the response} \\ &= A - K_p' A = A(1 - K_p') = A \left(1 - \frac{K_p K_c}{1 + K_p K_c}\right) \\ &= \frac{A}{1 + K_p K_c} \end{aligned}$$

$K_c \rightarrow \infty \Rightarrow \text{Offset} \rightarrow 0.$

$\checkmark \zeta' = \frac{\zeta}{\sqrt{1 + K_p K_c}}$ $K_c \downarrow \zeta' \uparrow$
 $\checkmark K_c \uparrow \zeta' \downarrow$ more oscillatory

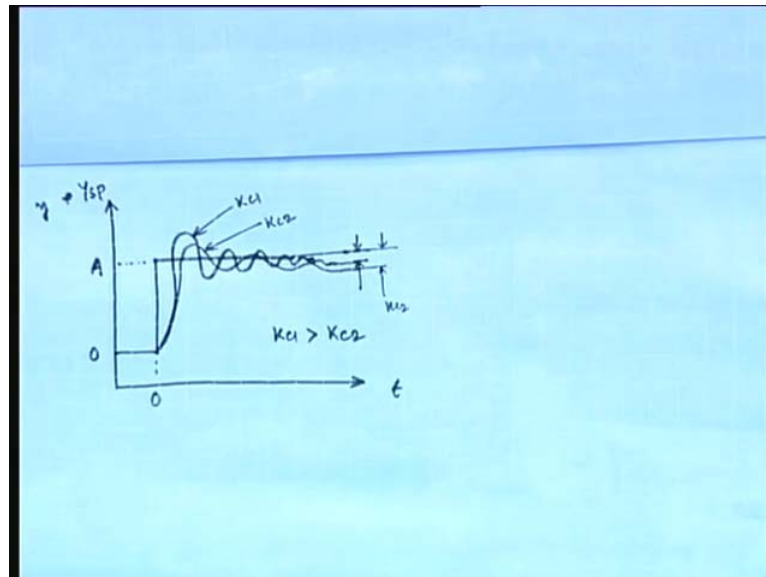
$\checkmark \text{Offset} = \frac{A}{1 + K_p K_c}$ \checkmark Larger the K_c value, smaller the offset.

When K_c tends to infinity, the offset approach is 0, but this is not through in practice due to some limitation of K_c value, now you will try to produce one plot to observe the dynamic behavior of the closed loop system. Now, we obtain the expression of zeta prime as zeta prime equals zeta divided by root over of 1 plus $K_p K_c$, this expression

we obtain. So, with the decrease of K_c , zeta prime increases; that means, with the increase of K_c zeta prime decreases.

So, if the increase of K_c value the overall response may be more oscillatory, so with the increase of K_c the overall response becomes more oscillatory. This is one point, another point is we know the offset equals A divided by $1 + K_p K_c$, the larger the K_c value the smaller the offset. From this expression we can say that the larger the K_c value the smaller the offset, but the considering this two points with the increase of K_c , the overall this becomes more oscillatory and another point is the larger the K_c the smaller the offset.

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We will try to produce the plot and in the close loop response keeping in mind this two points. So, we will produce y versus time y and y set point will consider and here time, now we consider a step change in set point with magnitude A , so this is magnitude A , initially it was say 0 and we are introducing step change a time equals 0.

Now, the close loop look response is like this, for a particular K_c that is suppose $K_c 1$ this is for another K_c suppose $K_c 2$, so here $K_c 1$ is greater than $K_c 2$ fine. So, for the last K_c that is $K_c 1$ we obtain more oscillatory response, one point an second point is for larger K_c the offset is relatively small and this is for the case of $K_c 2$ offset is large for small K_c .

So, above two points are taken into account in the development of this close loop response, in the next we will discuss the effect of integral control action. So, for we discussed the effect of proportion action on the overall response of a process, in the next we will discuss the effect of integral action on the overall response of a process.

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Effect of Integral Control Action: Servo problem.

Process: $G_p = \frac{K_p}{\tau_p s + 1}$

Controller: $G_c = \frac{K_c}{\tau_i s} \dots \dots$ I action.

$G_m = G_f = 1$

CLTF $\bar{y} = \frac{G_c G_p G_f}{1 + G_c G_p G_f G_m} \bar{y}_{sp} \dots \dots$ Servo

$G_c = K_c \left(1 + \frac{1}{\tau_i s}\right) = K_c + \frac{K_c}{\tau_i s}$

\uparrow P \uparrow I

$\bar{y} = \frac{\left(\frac{K_c}{\tau_i s}\right) \cdot \left[\frac{K_p}{(\tau_p s + 1)}\right]}{1 + \left(\frac{K_c}{\tau_i s}\right) \left[\frac{K_p}{(\tau_p s + 1)}\right]} \bar{y}_{sp}$

So, next topic is effect of integral control action and we will consider some servo problem only, the regulatory problem is left for the students. So, we will consider different elements and their expressions first, first element is the process and discuss to element integral control action we consider first order process. So, for the first order process is transfer function can be written as $G_p = \frac{K_p}{\tau_p s + 1}$, this is the transfer function of a first order process.

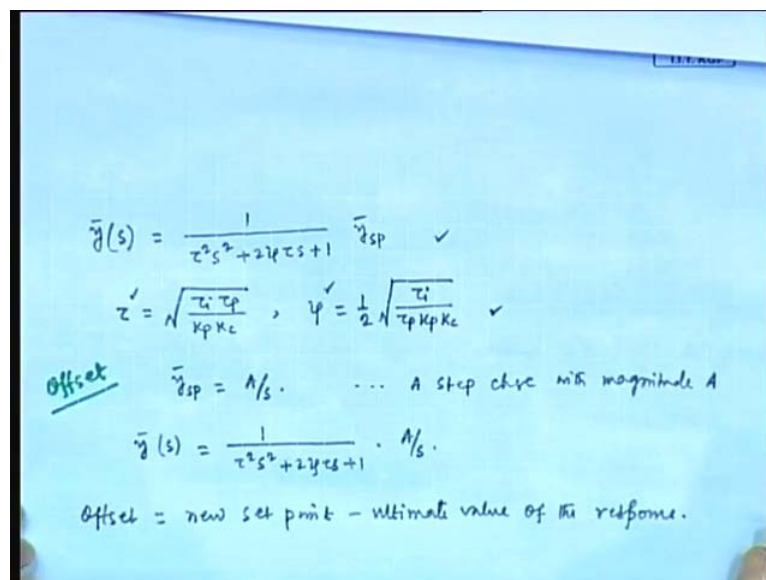
Next element is the controller, the controller is the integral controller what is the transfer function of integral controller, the transfer function of the integral controller is written as $G_c = \frac{K_c}{\tau_i s}$. For G_i control is the transfer function is K_c multiplied by $1 + \frac{1}{\tau_i s}$; that means, $K_c + \frac{K_c}{\tau_i s}$. So, this is the transfer function of P and this is the transfer function of I , so if we consider only the integral controller than the transfer function is $G_c = \frac{K_c}{\tau_i s}$.

And simplicity we will consider the transfer function of measuring device and the transfer function of final control element, both are unity; that means, G_m and G_f equal to 1. Now, we will try to develop the close loop transfer function substituting all this

individual transfer functions, in the next we will develop the closed loop transfer function.

The general form is $\bar{y}(s) = G_c G_p G_f \bar{y}_{sp}$ divided by $1 + G_c G_p G_f G_m$ \bar{y}_{sp} , now we will substitute the individual transfer function in this closed loop transfer function for the case of servo problem G_c is K_c divided by $\tau_i s$. This is $G_c G_p$ is K_p divided by $\tau_p s + 1$ and G_p is equal to 1 then $1 + K_c$ divided by $\tau_i s$ multiplied by K_p divided by $\tau_p s + 1$ and \bar{y}_{sp} .

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$$\bar{y}(s) = \frac{1}{\tau^2 s^2 + 2\zeta\tau s + 1} \bar{y}_{sp} \quad \checkmark$$

$$\zeta = \sqrt{\frac{\tau_i \tau_p}{K_p K_c}} \quad , \quad \tau = \frac{1}{2} \sqrt{\frac{\tau_i}{\tau_p K_p K_c}} \quad \checkmark$$

Offset $\bar{y}_{sp} = A/s$... A step chge with magnitude A

$$\bar{y}(s) = \frac{1}{\tau^2 s^2 + 2\zeta\tau s + 1} \cdot A/s$$

Offset = new set point - ultimate value of the response.

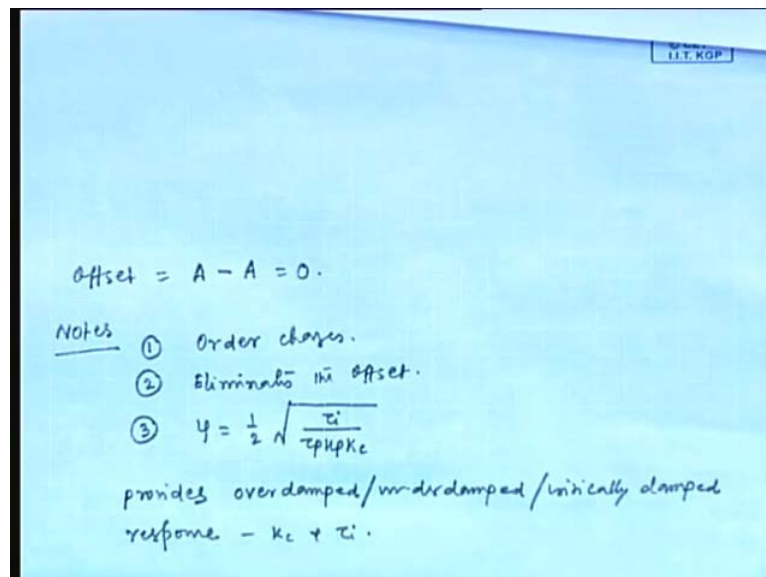
If we simplify this expression then we obtain $\bar{y}(s) = \frac{1}{\tau^2 s^2 + 2\zeta\tau s + 1} \bar{y}_{sp}$, simplifying we obtain this expression multiplied by \bar{y}_{sp} . First we substituted all the individual transfer function in generalized form of those transfer function, then after simplification we obtain this expression, where τ is root over of $\tau_i \tau_p$ divided by $K_p K_c$. Here, τ has this expression I mean τ has this relationship, similarly ζ is equal to half root over of τ_i divided by $\tau_p K_p K_c$.

So, these are the two expressions which have in consideration in the development of closed loop transfer function, you see found the co-relation is clear that the τ and ζ both depend on the value of K_c and τ_y . From these two co-relation, it is obvious that the τ and ζ both depend on the value of K_c and τ_y . Where, K_c and τ_y are the

tuning parameters, controller tuning parameters, in the next we will try to determine the offset.

Now, for the determining offset we will consider the step change in set point with magnitude A; that means, y set point bar is equal to A bar s. We are consider in step change with magnitude A, than the close loop transfer function becomes 1 divide y bar is equals 1 divide by $\tau^2 s^2 + 2 \zeta s + 1$ multiplied by A by s . Then offset is equal to new set point minus ultimate value of the response, we know this correlation offset equals minus ultimate value of the response.

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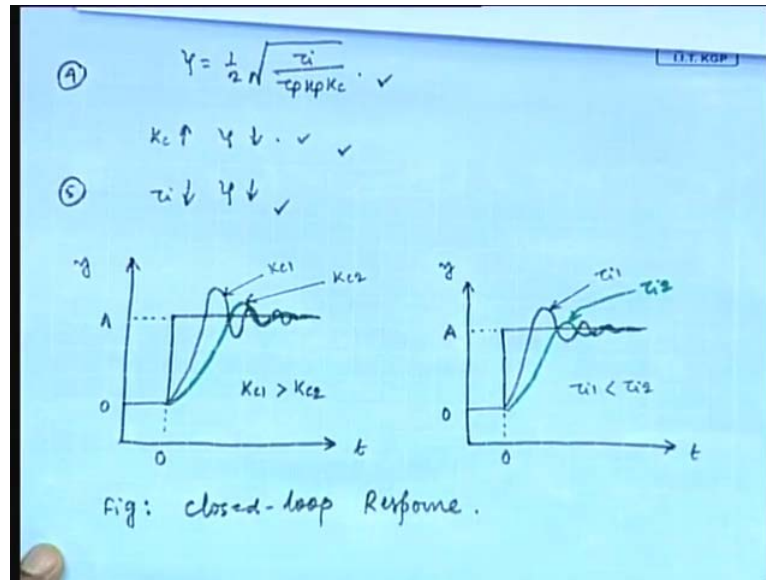


So, new set point is A, and ultimate value of the response is how much A, so offset becomes 0. So, we can say that the integral control action can eliminate the offset, it is very obvious from this that the integral control action can eliminate the offset. Now, we will note down some few important points, is any change of order due to the inclusion of integral action. Yes, so due to the integral action the order changes, the order of the overall system changes, due to the inclusion of integral action this is the first point.

Second point is we have seen that due to the inclusion of integral action the offset become 0, so second important point is the integral action can eliminate the offset, integral control action eliminates the offset. Third important point is as I mentioned zeta want depends on the value K_c and τ_i , if we see the expression of zeta we can say that zeta depends on the value of K_c and τ_i .

So, close loop system provides over damped or under damped, critically damped response depending on the value of K_c and τ_i , closed loop system provides over damped or under damped or critically damped response depending on the value of K_c and τ_i . So, the selection of K_c and τ_i is very important.

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Fourth point is if we again see the expression of zeta that is zeta is half root over of tau i divided by tau p K p K c, we can say that with the increase of K_c , zeta decreases. So, the over damped response may move to under damped response with large over suite and big ratio. If K_c is sufficiently large the over damped response may move to under damped response with large over suite and big ratio, in the same line we can conclude again that with the decrease of τ_i zeta decreases. You see the expression of zeta it is clear that with the decrease of τ_i zeta decreases.

So, if the τ_i is sufficiently small the over damped response may move to under damped response with large over large suite and big ratio. Now, we is to produce plots best on the common 4 and 5, in the fourth point we have mention that with the increase of K_c zeta decreases; that means, more oscillatory response.

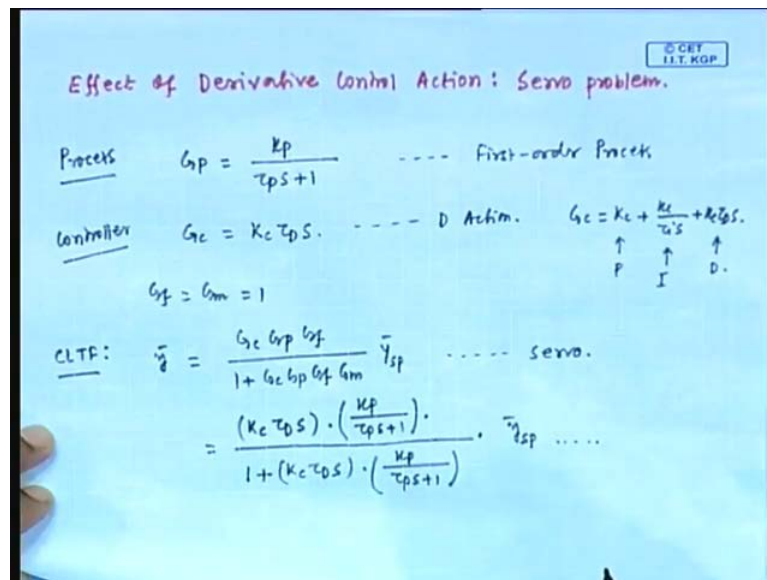
So, that we will reflect through this plot y verses time, a step change is introduced with magnitude A , this is suppose 0 and the step change is introduce that time equals 0. Now, this is the close loop response for a particular K_c value, suppose that is K_{c1} another close loop response we obtain for the value of K_{c2} . So, which one is greater K_{c1} or K_{c2} .

c 2, $K_c < 1$, because our fourth point is with the increase of K_c we get more oscillatory response. So, the $K_c < 1$ is larger than $K_c < 2$.

So, that is why we obtain more oscillatory response, but definitely here τ_i is kept constant we only varied the K_c value. Similarly, we will produce the close loop response based on 0.5 that is with the decrease of τ_i zeta decreases; that means, with decrease of τ_i the overall response become more oscillatory. So, we will try to produce y versus t , similarly a step change is introduced in set point with magnitude, suppose A at that time t equals 0. This is the close loop response for a particular τ_i value, suppose that τ_i is τ_{i1} , another response will we obtain one τ_{i2} .

So, which one is larger among this two τ_i ; that means, τ_{i1} is less than τ_{i2} , with the decrease of τ_i zeta decreases; that means, more oscillatory response. So, first one is the close loop response with the valuation of K_c and second one is the close loop response with the valuation of τ_i . So, these two points 4 and 5 clearly say that we need judiciously tune the values of K_c and τ_i . So, we discuss effect of proportion action and integral action and the close loop response, the third action is the derivative action.

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We will discuss that I mean discuss the next derivative action on close loop response, effect of derivative action, derivative control action and we will consider the servo problem. In the discussion of derivative action we will consider the first order process and servo problem, in a close loop block diagram the important element is process. The

transfer function of the process of a first order process can be written as G_p equals K_p divided by $\tau_p s + 1$, this is the transfer function of a first order process.

Next element is the controller recently we discuss derivative action, so the controller is only the derivative controller, what is the function of derivative control, $K_c \tau_p s$. This is derivative action, because the transfer function of P I D controller, we have written K_c K_c plus divided by $\tau_i s$ plus $K_c \tau_p s$. This is the transfer function of proportion action, this is the transfer of integral action and this is the transfer function derivative action.

And simplicity for will come consider the transfer function G_f and G_m both are equal to 1, now we will try to develop the close loop transfer function substituting of all these individual expressions. In the next will develop close loop transfer function which is express in generalized form as \bar{y} equals $G_c G_p G_f$ divided by $1 + G_c G_p G_f G_m$ y set point bar, this is generalized close loop transfer function for servo base.

Now, if we substitute the individual transfer functions, than we obtain G_c as $K_c \tau_D s$, next transfer function is G_p ; that means, K_p divided by $\tau_p s + 1$ and G_f is 1, whole divided by $1 + K_c \tau_D s$ multiplied by K_p divided by $\tau_p s + 1$ multiplied by y set point bar. Now, if we this is the close loop transfer function we obtain, substituting all individual transfer functions.

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The image shows handwritten mathematical work on a blue background. At the top, a partial equation is visible:
$$= \frac{K_p K_c \tau_D s}{1 + (K_c \tau_D s)} \cdot \left(\frac{K_p}{\tau_p s + 1} \right)$$

Below this, the closed-loop transfer function is derived:
$$\bar{y} = \frac{K_p K_c \tau_D s}{(\tau_p + K_p K_c \tau_D) s + 1} \cdot \bar{y}_{sp} \quad \dots \text{Final form.}$$

The effective time constant is defined as:
$$\tau_p' = (\tau_p + K_p K_c \tau_D) = \text{Effective time constant.}$$

Under the heading "Notes", there are five points:

- ① No change of order.
- ② $\tau_p' > \tau_p$. $K_p K_c \rightarrow$ positive
- ③ $K_c \uparrow \tau_p' \uparrow$. $\tau_D \rightarrow$ positive.
- ④ $\tau_D \uparrow \tau_p' \uparrow$; ⑤ Offset = A

Now, if we rearrange further than we obtain the final close loop transfer function as \bar{y} equals $K_p K_c \tau_D s$ divided by τ_p plus $K_p K_c \tau_D$ multiplied by $s + 1$. \bar{y} set point bar, this is the final form of close loop transfer function. Now, this is the time constant of the close loop transfer function, so this time we can say we can call as effective time constant denoted by τ_p' . So, τ_p' equals τ_p plus $K_p K_c \tau_D$, this is called as effective time constant we can call this time constant as effective time constant.

So, which one is larger τ_p or τ_p' , τ_p' now we will note down few important points, due to the inclusion of derivative action is there any change of order. No, the order of overall system does not change, we have taken first order process and overall processing is first order. So, there is no change of order of the overall system due to the inclusion of derivative function there is no change order of the overall system, this is the first point.

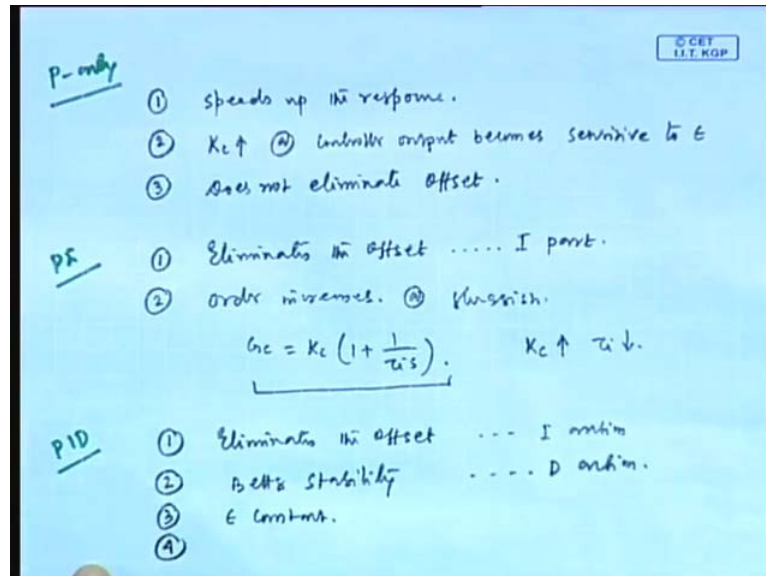
Second point is regarding the time constant, the time constant τ_p' is greater than τ_p , because $K_p K_c$ is positive and τ_D is also positive. $K_p K_c$ is positive multiplication of K_p and K_c that is positive and τ_D is time constant time constant is always positive. So, this is also positive therefore, τ_p' is greater than τ_p , so what it indicates the overall response becomes ((Refer Time: 47:34)).

Third point is as K_c increases, then what happens for τ_p' , if K_c increases then τ_p' also increases with the increase of K_c τ_p' increases; that means, the overall response becomes slower due to the increase of K_c . Similarly, what happens for τ_D , if τ_D increases the effective time constant increases. So, in the derivative action there are two tuning parameters involved, one is K_c and another one is τ_D . So, that is why we observe the effects of K_c as well as τ_D with the increase of K_c τ_p' increases and with the increase of τ_D also τ_p' increases; that means the overall response becomes slower due to the increase of K_c and τ_D .

Fifth point is related to the offset, can you calculate the offset with introducing step change point with set point magnitude A . What about offset? Offset is A , so fifth point is the derivative controller cannot eliminate offset, these are the five important points related to the close loop response and under derivative controller action. So, we have

discussed about three particular controller proportions one is P only, second one is P I and third one is P I D.

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Now we conclude these control actions one by one, so first one is P only controller, what you conclude the P only controller. It speeds up the response, if you see the response of over loop system and if you see the response of a process, same process under P only controller, we obtain first response. In case of process under P only controller, if we include a proportional controller with a process the overall response become faster than the open loop response. Therefore, we conclude that the P only controller speeds of the response.

Secondly, we can say that with the increase of K_c value, the proportional only controller output becomes more sensitive to error, with the increase of K_c value the P only controller output becomes more sensitive to the actuating error signal. This is the second conclusion on P only controller, third one is it cannot eliminate offset, it does not eliminate offset, these are the three important points related to P only controller.

Next is P I controller, most important point related to P I controller is that eliminate the offset, so which part eliminate the offset integral part integral action eliminates the offset. Second point is the order for P only controller there is no change of order, but P I controller order is increased. So, second point is order increases and as the order increases the overall response will sluggish.

Now, how we can improve this situation, due to the inclusion of an integral action with a proportional action, the overall response becomes sluggish. Now, we want to speed up the response, how we can do that either by increasing K_c and decreasing τ_i is indicated. We can speed up the response either by increasing K_c or by decreasing τ_i , if you see the controlled equation, a PI controller equation cleared that with the increase of K_c the overall system response becomes faster. But, if K_c is larger than the critical value or if τ_i is sufficiently small, the process has some instability problem.

So, if the K_c value is sufficiently large or τ_i is sufficiently small the instability problem may arise, for the case of a PI controller. Next is the PID controller, so first we can say that it eliminates the offset. A PID controller eliminates the offset and this is the definite effect of the integral action.

Now, for the case of a PI controller, we observe that with the increase of K_c value I mean when K_c is larger than a certain limit or τ_i is sufficiently small there is some instability problem, that can be improved by the addition of derivative action. So, PID control has better stability criteria and this is due to derivative action only. Third point is if error is constant, then there is no derivative action because $\frac{de}{dt}$ is zero, if error is constant then there is no derivative action. And last point is for a noisy signal almost 0 error, the derivative term needs to large control action although that is not required. So, these are the concluding remarks on three proportional controller actions.

Thank you.