

Process Control and Instrumentation
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Lecture - 13
Dynamic Behavior of Chemical Processes (Contd.)

(Refer Slide Time: 01:12)

Example of the noninteracting system

Tank 1

$$A_1 \frac{dh_1}{dt} = F_i - F_1$$

$$F_1 = \frac{h_1}{R_1}$$

$$A_1 \frac{dh_1}{dt} = F_i - \frac{h_1}{R_1}$$

$$\Rightarrow A_1 \frac{dh_1}{dt} + \frac{h_1}{R_1} = F_i$$

$$A_1 \frac{dh_1'}{dt} + \frac{h_1'}{R_1} = F_1' \quad \dots \text{in terms of deviation variables.}$$

$$\Rightarrow A_1 R_1 \frac{dh_1'}{dt} + h_1' = R_1 F_1' \quad \dots \text{Multiplying both sides by } R_1$$

Tank 2

$$A_2 \frac{dh_2}{dt} = F_1 - F_2$$

$$F_2 = \frac{h_2}{R_2}$$

$$A_2 \frac{dh_2}{dt} = F_1 - \frac{h_2}{R_2}$$

$$\Rightarrow A_2 \frac{dh_2}{dt} + \frac{h_2}{R_2} = F_1$$

Today, we will continue our discussion on dynamics of second order systems. And we will start today, with a physical example of noninteracting system. Example, of the noninteracting system it was told that if 2 first order systems or 2 first order tanks are connected in series the overall response is second order response. So, today we will take that example I mean, 2 first order systems those are first order tank systems connected in series.

So, this is tank 1, input to the tank is F_i cross sectional area is A_1 and height is h_1 output is F_1 and resistant to flow is suppose R_1 . This outlet from Tank 1 enters Tank 2, this is Tank 2 cross sectional area of this Tank 2 is A_2 and liquid height in the Tank is h_2 , output stream from Tank 2 is F_2 and suppose this resistance is R_2 this is a example of a noninteracting system.

Now, what are the modeling equations for these two Tanks? If we consider Tank 1 the modeling equation is $A_1 \frac{dh_1}{dt} = F_i - F_1$ that is F_i minus output flow rate that is F_1 . This is a modeling equation for Tank 1. Now, F_1 is again we can write by this

form h_1 divided by R_1 equals h_1 divided by R_1 driving force divided by resistant's to flow. If, we substitute this F_1 equals h_1 divided by R_1 in the modeling equation then, we have $A_1 \frac{dh_1}{dt}$ equals F_i minus h_1 divided by R_1 . Substituting the expression of F_1 we finally get this, rearranging this equation we have $A_1 \frac{dh_1}{dt}$ plus h_1 divided by R_1 equals F_i fine, rearranging we get this.

Now, if we write this equation in terms of deviation variables, then we obtain $A_1 \frac{dh_1'}{dt}$ plus h_1' divided by R_1 equals F_i' . This is modeling equation in terms of deviation variables. Fine now, if we multiplied both sides by R_1 then we get $A_1 R_1 \frac{dh_1'}{dt}$ plus h_1' equals $R_1 F_i'$, this equation we get by Multiplying both sides with R_1 fine.

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Handwritten mathematical derivation on a blue background:

$$A_1 \frac{dh_1'}{dt} + \frac{h_1'}{R_1} = F_i' \quad \dots \text{in terms of deviation variables.}$$

$$\Rightarrow A_1 R_1 \frac{dh_1'}{dt} + h_1' = R_1 F_i' \quad \dots \text{Multiplying both sides by } R_1$$

$$A_1 R_1 = \tau_{p1} \quad \text{and} \quad R_1 = K_{p1}$$

$$\tau_{p1} \frac{dh_1'}{dt} + h_1' = K_{p1} F_i'$$

$$G_1(s) = \frac{\bar{h}_1'(s)}{\bar{F}_i'(s)} = \frac{K_{p1}}{\tau_{p1}s + 1} \quad \dots \text{TF of } \bar{h}_1 \text{ } \# \text{ First Tank}$$

Now, you will assume that $A_1 R_1$ equals τ_{p1} and R_1 is equal to K_{p1} we assume $A_1 R_1$ equals τ_{p1} which is for the time constant First Tank and R_1 equals k_{p1} this is the gain of the first tank. Then, the equation modeling equation becomes $\tau_{p1} \frac{dh_1'}{dt}$ plus h_1' equal to this is $K_{p1} F_i'$ this is the first order differential equation of the output h_1 .

(Refer Slide Time: 09:39)

Tank-2

$$A_2 \frac{dh_2}{dt} = F_1 - F_2 \quad F_2 = \frac{h_2}{R_2}$$
$$\Rightarrow A_2 \frac{dh_2}{dt} = F_1 - \frac{h_2}{R_2}$$
$$\Rightarrow A_2 R_2 \frac{dh_2}{dt} + h_2 = R_2 F_1 \quad \checkmark$$

$A_2 R_2 \frac{dh_2'}{dt} + h_2' = R_2 F_1' \quad \dots$ In terms of deviation variables.

$A_2 R_2 = \tau_{p_2}$ and $R_2 = K_{p_2} \quad \checkmark$

$$\tau_{p_2} \frac{dh_2'}{dt} + h_2' = K_{p_2} F_1'$$

Now, if we take Laplace transform and rearrange then we get the transfer function of the First tank as \bar{h} is divided by F_1 is equal to K_{p1} divided by $\tau_{p1} s + 1$ this is a transfer function of the First tank that is tank 1. Now, you will represent this as $G_1(s)$ similarly we will consider tank 2 for tank 2 we can write the modeling equation as $A_2 \frac{dh_2}{dt} = F_1 - F_2$.

Now, F_2 is again the driving force divided by resistance to flow. So, $F_2 = \frac{h_2}{R_2}$ if you substitute that in this modeling equation we get $A_2 \frac{dh_2}{dt} = F_1 - \frac{h_2}{R_2}$. Multiplying both sides R_2 and rearranging we get $A_2 R_2 \frac{dh_2}{dt} + h_2 = R_2 F_1$ fine multiplying both sides by resistance R_2 and rearranging we get this equation.

Now, similarly we have to represent this equation in terms of deviation variables then we get $A_2 R_2 \frac{dh_2'}{dt} + h_2' = R_2 F_1'$. This is represented in terms of deviation variables fine. Now, we assume $A_2 R_2$ equals τ_{p2} and R_2 is equal to K_{p2} , we assume $A_2 R_2$ is equal to the time constant for tank 2 and R_2 as gain of the tank 2. Then we get $\tau_{p2} \frac{dh_2'}{dt} + h_2' = K_{p2} F_1'$.

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$$\checkmark F_1' = \frac{h_1'}{R_1} = \frac{h_1'}{K_{p1}}$$

$$G_2(s) = \frac{\bar{h}_2'(s)}{F_1'(s)} = \frac{K_{p2}}{\tau_{p2}s + 1} \quad \dots \text{TF of second tank.}$$

$$G_0(s) = \frac{\bar{h}_2'(s)}{F_1'(s)} = \left(\frac{\bar{h}_2'(s)}{F_1'(s)} \right) \cdot \frac{F_1'(s)}{F_1'(s)} = \left(\frac{\bar{h}_2'(s)}{F_1'(s)} \right) \cdot \left(\frac{h_1'(s)}{K_{p1} F_1'(s)} \right)$$

$$= G_{p2} \times G_1 = \frac{K_{p2}}{\tau_{p2}s + 1} \cdot \frac{1}{K_{p1}} \cdot \frac{K_{p1}}{\tau_{p1}s + 1}$$

$$G_0(s) = \frac{K_{p2}}{(\tau_{p1}s + 1)(\tau_{p2}s + 1)} \quad \dots \text{overall TF}$$

And taking Laplace transform finally, we get the transfer function of tank 2 as $G_2(s)$ is equal to $\bar{h}_2'(s)$ divided by $F_1'(s)$ equals K_{p2} divided by $\tau_{p2}s + 1$. This is the transfer function of the second tank that is tank 2. Now, if you see the schematic of the 2 tank system. We can say that the overall transfer function will be $\bar{h}_2'(s)$ divided by $F_1'(s)$. The overall transfer function of this 2 tank systems can be written $G_0(s)$ equals $\bar{h}_2'(s)$ this is a output of tank 2 divided by the input of input 2 tank 1 fine. This is a overall transfer function.

Now, we can write again the overall transfer function by this form, $\bar{h}_2'(s)$ divided by $F_1'(s)$ multiplied by $F_1'(s)$ divided by $F_1'(s)$ can we write this, the overall transfer function we can write in this form.

So, this is equal to $\bar{h}_2'(s)$ divided by $F_1'(s)$ this is nothing, but the transfer function of tank 2. And what is $F_1'(s)$ we know F_1' equals h_1' divided by R_1 , for the tank 1 we have consider this correlation. F_1' equals h_1' divided by R_1 that means, h_1' divided by K_{p1} . Can we write this?

Now, this relationship will substitute here, I mean F_1' F_1' prime this is not bar prime F_1' prime equals h_1' prime divided by K_{p1} . So, if we substitute here, h_1' prime s divided by $K_{p1} F_1'(s)$. Can we write this? There is no change of this term. Now, in place of F_1' prime we have substitute it h_1' prime divided by K_{p1} and then, F_1' prime remains there. So, the first term this 1 represents G_{p2} not G_{p2} G_2 and multiplied by

this which is G_1 divided by k_{p1} that means, k_{p2} divided by $\tau_{p2}s + 1$ multiplied by 1 divided by k_{p1} multiplied by k_{p1} divided by $\tau_{p1}s + 1$ agree. So, the overall transfer function $G(s)$ we can write as, k_{p2} divided by $\tau_{p1}s + 1$ multiplied by $\tau_{p2}s + 1$. This is a overall transfer function of the example non-interacting system.

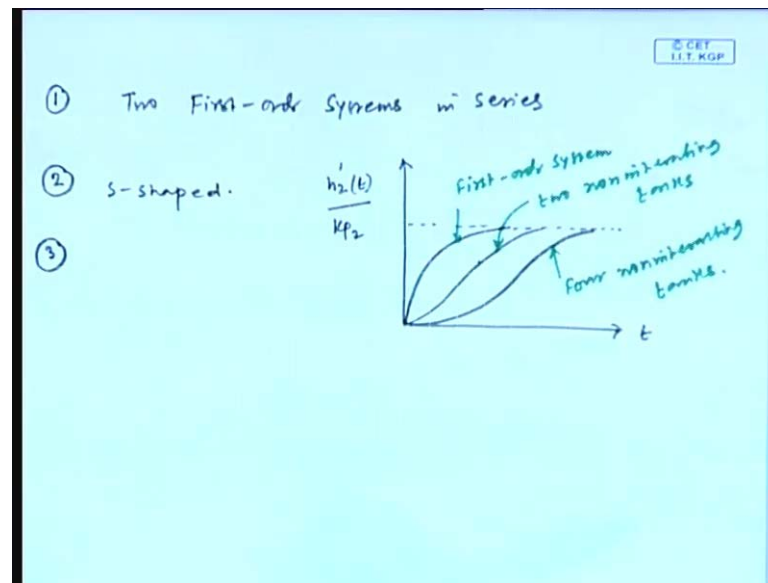
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$$\frac{\bar{h}_2'(s)}{\bar{f}_i'(s)} = \frac{k_{p2}}{(\tau_{p1}s + 1)(\tau_{p2}s + 1)} \quad \boxed{\bar{f}_i'(s) = \frac{1}{s}}$$

$$h_2'(t) = k_{p2} \left[1 + \frac{1}{\tau_{p2} - \tau_{p1}} (\tau_{p1} e^{-t/\tau_{p1}} - \tau_{p2} e^{-t/\tau_{p2}}) \right]$$

So, $G(s)$ is basically $\bar{h}_2'(s)$ divided by $\bar{f}_i'(s)$. We can write in this form now, you will introduce a unit step change in the input variable, considering a unit step change in F_i and taking inverse of Laplace transform we get the output in time domain as $h_2'(t) = k_{p2} \left[1 + \frac{1}{\tau_{p2} - \tau_{p1}} (\tau_{p1} e^{-t/\tau_{p1}} - \tau_{p2} e^{-t/\tau_{p2}}) \right]$ considering a unit step change in $\bar{f}_i'(s)$ I mean, $\bar{f}_i'(s) = \frac{1}{s}$ and taking inverse of Laplace transform we get the expression of output $h_2(t)$ in time domain like this fine.

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Now, you will conclude based on this discussion. So, First conclusion is that, by connecting to First-order systems in series. We get overall Second-order dynamics fine, by connecting to First-order system in series we obtain Second-order dynamics. So, by connecting to First-order systems in series the overall dynamics becomes Second-order dynamics this is a First conclusion. Second thing is we used to observe the behavior making a plot.

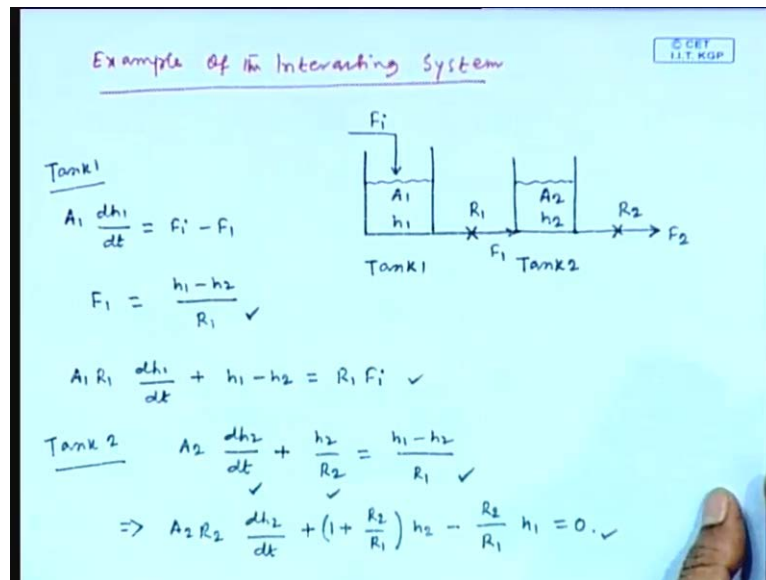
So, this is $h_2' t$ divided by K_{p2} versus time. The steady state is represented by the dotted line and the dynamics is like this, is for First-order system. This is the dynamics of First-order system, this is the dynamics of Second-order or 2 non-interacting 2 non-interacting tanks system and this is the dynamics of 4 non-interacting tanks fine.

For the First-order system this should be h by K_p for 2 non-interacting tanks this should be h_2 divided by K_{p2} or h_2' prime by K_{p2} for 4 non-interacting tanks this should be h_4 prime t divided by K_{p4} . So, this is the dynamic behavior of 3 systems. Now, what conclusion we can make initially you see it changes slowly for the 2 non-interacting tank systems initially it changes slowly then it fix up the speed, the behavior for second and higher order systems is s shaped, fine the behavior is s shaped. So, initially it changes slowly then its fix up the speed fine.

So, this is basically the over time response, and 3rd conclusion is as the number of capacities in series increases the delay in the initial response becomes more pronounced.

You see if you compared these two curves, the delay is more for the case of 4 non-interacting tank system. So, that is the 3rd conclusion. As the number of capacities in series increases the delay in the initial response becomes more pronounced this is the 3rd conclusion fine.

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So, this is the example physical example of a non-interacting tank system. In the next we will consider an interacting tank systems; we will connect 2 tanks in series. So, that they are interactive to each other. So, next example is, the Example of the Interacting System fine. For the case of interacting system similarly, you consider 2 tanks which are connected in series. This is tank 1 cross sectional area of A1 and h1 input is Fi resistant to flow is R1, this is Tank 2 cross-sectional area A2 height is h2. This flow rate is F1 and the stream which is coming out from Tank 2 has the flow rate of F2 and resistance is R2.

So, the modeling equation for Tank 1 we can write in this form $A_1 \frac{dh_1}{dt} = F_i - F_1$. What is F_1 here? What is the relationship of F_1 ? F_1 equals $\frac{h_1 - h_2}{R_1}$. Since h_1 is higher than h_2 that is why there is a flow. So, if we substitute this, then we obtain $A_1 R_1 \frac{dh_1}{dt} + h_1 - h_2 = R_1 F_i$. If we substitute the expression of F_1 in the modeling equation. Then we get this equation. Similarly, for Tank 2 the modeling equation we can write in this form $A_2 \frac{dh_2}{dt} + \frac{h_2}{R_2} = \frac{h_1 - h_2}{R_1}$. Can we write this, for Tank2 $A_2 \frac{dh_2}{dt} + \frac{h_2}{R_2} = \frac{h_1 - h_2}{R_1}$ fine. Now, these 2 modeling equation we need to write in terms of deviation

variables anyway before that, we can rearrange these equations and we obtained $A_1 R_1 \frac{dh_1}{dt} + h_1 - h_2 = R_1 F_i'$ for Tank 1. Similarly for Tank 2, we can write the modeling equation in terms of deviation variables as $A_2 R_2 \frac{dh_2}{dt} + (1 + \frac{R_2}{R_1}) h_2 - \frac{R_2}{R_1} h_1 = 0$, this is for Tank 2 fine.

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Handwritten mathematical derivation for two tanks. The equations are:

$$A_1 R_1 \frac{dh_1}{dt} + h_1 - h_2 = R_1 F_i' \quad \dots \text{Tank 1.}$$

$$A_2 R_2 \frac{dh_2}{dt} + \left(1 + \frac{R_2}{R_1}\right) h_2 - \frac{R_2}{R_1} h_1 = 0 \quad \dots \text{Tank 2.}$$

L- transform

$$\left. \begin{aligned} (A_1 R_1 s + 1) \bar{h}_1(s) - \bar{h}_2(s) &= R_1 \bar{F}_i'(s) \quad \dots \text{Tank 1.} \\ - \frac{R_2}{R_1} \bar{h}_1(s) + \left[A_2 R_2 s + \left(1 + \frac{R_2}{R_1}\right) \right] \bar{h}_2(s) &= 0 \quad \dots \text{Tank 2.} \end{aligned} \right\}$$

$$\bar{h}_1(s) = \frac{\tau_{p2} R_1 s + (R_1 + R_2)}{\tau_{p1} \tau_{p2} s^2 + (\tau_{p1} + \tau_{p2} + A_1 R_2) s + 1} \bar{F}_i'(s)$$

... Tank 1.

Now, we use to write these 2 equations in terms of deviation variables. So, for Tank 1 we can write the modeling equation in terms of deviation variables, and we obtained $A_1 R_1 \frac{dh_1}{dt} + h_1 - h_2 = R_1 F_i'$. This is for Tank 1. Similarly for Tank 2, we can write the modeling equation in terms of deviation variables as $A_2 R_2 \frac{dh_2}{dt} + (1 + \frac{R_2}{R_1}) h_2 - \frac{R_2}{R_1} h_1 = 0$, this is for Tank 2 fine.

Now, we will take Laplace transform of these 2 equations. Taking Laplace transform we get for, Tank 1 $A_1 R_1 s + 1, \bar{h}_1(s) - \bar{h}_2(s) = R_1 \bar{F}_i'(s)$. Taking Laplace transform for the case of Tank 1, I mean the modeling equation of Tank 1 we obtain this form. Similarly if we take the Laplace transform of this, we obtained $-\frac{R_2}{R_1} \bar{h}_1(s) + [A_2 R_2 s + (1 + \frac{R_2}{R_1})] \bar{h}_2(s) = 0$. If we take the Laplace transform for the modeling equation of Tank 2 we obtain this expression.

Now, we need to solve these 2 equations, to obtain the transfer function with respect to Tank 1. As well as Tank 2, solving these 2 equations we get the transfer function for Tank 1 as $\bar{h}_1(s) = \frac{\tau_{p2} R_1 s + (R_1 + R_2)}{\tau_{p1} \tau_{p2} s^2 + (\tau_{p1} + \tau_{p2} + A_1 R_2) s + 1} \bar{F}_i'(s)$.

square, plus tau p1, tau p2, plus A1 R2 multiplied by s plus 1 Fi bar prime s. This is for Tank 1 fine solving these 2 equations we obtain the transfer function for Tank 1 in this form.

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The image shows handwritten mathematical derivations on a light blue background. At the top, there is a partially visible equation: $\frac{h_1(s)}{R_1} = \frac{\tau_{p1} R_1 s + (R_1 + R_2)}{\tau_{p1} \tau_{p2} s^2 + (\tau_{p1} + \tau_{p2} + A_1 R_2) s + 1} \cdot \bar{F}_i'(s)$. Below this, the transfer function for Tank 1 is given as $\bar{h}_1'(s) = \frac{\tau_{p1} R_1 s + (R_1 + R_2)}{\tau_{p1} \tau_{p2} s^2 + (\tau_{p1} + \tau_{p2} + A_1 R_2) s + 1} \cdot \bar{F}_i'(s)$ with the note "... Tank 1." The next equation is $\bar{h}_2'(s) = \frac{R_2}{\tau_{p1} \tau_{p2} s^2 + (\tau_{p1} + \tau_{p2} + A_1 R_2) s + 1} \cdot \bar{F}_i'(s)$ with a checkmark and the note "... Interacting System." The final equation is $\bar{h}_2'(s) = \frac{\tau_{p2} \cdot \bar{F}_i'(s)}{(\tau_{p1} s + 1)(\tau_{p2} s + 1)} = \frac{R_2}{\tau_{p1} \tau_{p2} s^2 + (\tau_{p1} + \tau_{p2}) s + 1} \cdot \bar{F}_i'(s)$ with a checkmark and the note "... Noninteracting System." The term $A_1 R_2$ is circled in the first equation.

Similarly, for Tank 2 we obtain the transfer function as \bar{h}_2 bar prime s equals R_2 divided by τ_{p1} , τ_{p2} s square plus τ_{p1} plus τ_{p2} , plus $A_1 R_2$ s plus 1, \bar{F}_i bar prime s. This is a transfer function with respect to \bar{h}_2 fine. So, can say that this is the representation of overall transfer function because, overall transfer function we can represent by \bar{h}_2 bar prime s divided by \bar{F}_i bar prime s. So, this overall transfer function we obtain for the case of Interacting System. This overall transfer function we obtain for the case of Interacting System. What is the overall transfer function of the non-interacting system? For the non-interacting system we obtained, the overall transfer function in this form $k_p 2$ divided by τ_{p1} s plus 1 multiplied by τ_{p2} s plus 1, just we obtain before this discussion.

So, we can write this as R_2 divided by τ_{p1} τ_{p2} s square plus τ_{p1} plus τ_{p2} s plus 1 it will be multiplied with \bar{F}_i bar prime s. So, \bar{F}_i bar prime s this is the overall transfer function for non-interacting system. Can we compare these two overall transfer function, if we compare we see that 1 extra term is there for the case of interacting system. That is $A_1 R_2$ this term this is the additional term which is present in the overall transfer function of interacting tank system.

So, this term may be thought of as the interaction factor. These term $A_1 R_2$ may be thought of as a interaction factor fine. If we compare the 2 transfer functions we see that 1 additional term $A_1 R_2$ is present in the overall transfer function of interacting system, and these term may be thought of as the interaction factor.

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① second-order response.

②
$$p = \frac{-(\tau_1 + \tau_2 + A_1 R_2) \pm \sqrt{(\tau_1 + \tau_2 + A_1 R_2)^2 - 4\tau_1 \tau_2}}{2\tau_1 \tau_2}$$

$$(\tau_1 + \tau_2 + A_1 R_2)^2 - 4\tau_1 \tau_2 > 0.$$

$\zeta > 1$ overdamped.

Now, we will conclude based on this discussion for the case of interaction, Interacting System. What is the First conclusion? First conclusion is that the overall response is Second-order response this is the First conclusion. By connecting to First-order liquid tanks in series we obtain the overall response as Second-order response.

So, the overall response is Second-order response. Second conclusion in the second conclusion we use to first find the roots. What are the roots? There are 2 roots because, if you see the denominator that is if Second-order polynomial. That is it quadratic that is given in quadratic form. So, the roots we can write in this form, $\tau_1 + \tau_2 + A_1 R_2 \pm \sqrt{(\tau_1 + \tau_2 + A_1 R_2)^2 - 4\tau_1 \tau_2}$, divided by $2\tau_1 \tau_2$ fine there are 2 roots and they are represented by this form.

Now, $(\tau_1 + \tau_2 + A_1 R_2)^2 - 4\tau_1 \tau_2$ is greater than 0 fine. That means, 2 distinct real poles so, what about the overall response? The overall response is over damped response. So, we obtained zeta greater than 1 and the response of interacting capacities is always over damped.. So, these are about, the two tank

systems I mean, we observe that if 2 First-order systems are connected in series the overall response is Second-order response.

Another case we mentioned in the previous class that, Second-order system we can obtain by employing one controller around process, we mentioned 3 cases. 1 is by connecting to First-order systems in series, 1 is by employing 1 controller with a process, and in the 3rd case we consider that few systems are inherently higher order systems.

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Second-order dynamics : 1st First-order Process + Controller.

CV	MV
h	F _o

$A \frac{dh'}{dt} = F_i' - F_o'$ ✓ $h' = h - h_s$

controller eqn: $F_o = F_{os} + K_c h' + \frac{K_c}{\tau_i} \int h' \cdot dt + K_c \tau_D \frac{dh'}{dt}$

$\Rightarrow F_o' = K_c h' + \frac{K_c}{\tau_i} \int h' \cdot dt + K_c \tau_D \frac{dh'}{dt}$

$F_o' = F_o - F_{os}$

So, now we will consider the process having a controller. We use to observe the higher order dynamics. I mean Second-order dynamics for the case of a first-order process configured with a controller if a controller is employed for a first-order process what will be the overall dynamics that we use to observe.

So, we will first consider 1 example that is the liquid tank system. This is a liquid tank system, inlet flow rate is FI, and outlet flow rate is F naught. This is height cross-sectional area is A, control objective is to maintain the height of the liquid in the tank. So, for that we need to employ 1 level controller. So, our control objective is to maintain the height of liquid in the tank; that means the control variable is h and the corresponding manipulated variable is F naught fine, this is the control pair for this case.

Now, what is the modeling equation? Modeling equation in terms of deviation variables, we can write as A dh prime dt equals Fi prime minus F naught prime fine. Now since,

the controller is manipulating F_{naught} . So, we can correlate F_{naught} with the height, by a controller equation suppose a controller equation is given as F_{naught} equals F_{naught} s, plus $k_c h'$ plus k_c divided by τ_i integration of h' dt plus $k_c \tau_d$, dh' dt. This is the equation of a controller which we will discuss in the subsequent classes fine.

This is a controller equation which is correlating the manipulated variable F_{naught} with height; h' is h minus h_s fine. Now, we can write this equation as F_{naught}' equals $k_c h'$ plus k_c divided by τ_i integration h' dt plus $k_c \tau_d$, dh' dt, where F_{naught}' equals F_{naught} minus F_{naught} s. And in this equation $k_c \tau_i$ and τ_d these 3 are controller tuning parameters. In this controller equations $k_c \tau_i$ and τ_d are constant parameters fine. Now, we can substitute the expression of f_{naught}' in the modeling equation, by substituting the expression of F_{naught}' in the modeling equation.

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The image shows a whiteboard with handwritten mathematical derivations. The top part shows the controller equation in the time domain:

$$\Rightarrow F_o = k_c h' + \frac{k_c}{\tau_i} \int h' dt + k_c \tau_d \frac{dh'}{dt}$$

$$F_o' = F_o - F_o s$$

The middle part shows the Laplace transform of the controller equation:

$$A \frac{dh'}{dt} + k_c h' + \frac{k_c}{\tau_i} \int h' dt + k_c \tau_d \frac{dh'}{dt} = F_i'$$

$$(A + k_c \tau_d) s \bar{h}(s) + k_c \bar{h}(s) + \frac{k_c}{\tau_i s} \bar{h}(s) = \bar{F}_i'(s)$$

$$\Rightarrow \frac{\tau_i}{k_c} (A + k_c \tau_d) s^2 \bar{h}(s) + \tau_i s \bar{h}(s) + \bar{h}(s) = \frac{\tau_i s}{k_c} \bar{F}_i'(s)$$

$$\Rightarrow \tau^2 s^2 \bar{h}(s) + 2\tau s \bar{h}(s) + \bar{h}(s) = K_p s \bar{F}_i'(s)$$

The bottom part shows the final transfer function:

$$\Rightarrow \frac{\bar{h}(s)}{\bar{F}_i'(s)} = \frac{K_p s}{\tau^2 s^2 + 2\tau s + 1}$$

We obtain $A dh'$ dt plus $k_c h'$ plus k_c divided by τ_i integration h' dt plus $k_c \tau_d$, dh' dt equals F_i' . Substituting the expression of F_{naught}' in the modeling equation we obtain this. Now, taking Laplace transform we get, A plus $k_c \tau_d$ s \bar{h}' s, plus $k_c \bar{h}'$ s, plus k_c divided by τ_i s, \bar{h}' s equals \bar{F}_i' s fine. Taking Laplace transform we obtain this. So, we can write this

as τ_i divided by $k_c A + k_c \tau_d s^2$ $\bar{h}'(s)$, plus τ_i is $s \bar{h}'(s)$ plus $\bar{h}'(s)$ equals τ_i divided by $k_c F_i'(s)$.

Now, we can represent this as, $\tau^2 s^2 \bar{h}'(s) + 2\zeta\tau s \bar{h}'(s) + \bar{h}'(s) = k_p s F_i'(s)$. So, the transfer function can be represented by $\bar{h}'(s)$ divided by $F_i'(s)$ equals $k_p s$ divided by $\tau^2 s^2 + 2\zeta\tau s + 1$. The transfer function yields this form, $\bar{h}'(s)$ divided by $F_i'(s)$ equals $k_p s$ divided by $\tau^2 s^2 + 2\zeta\tau s + 1$.

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$$\Rightarrow \tau^2 s^2 \bar{h}'(s) + 2\zeta\tau s \bar{h}'(s) + \bar{h}'(s) = k_p s F_i'(s)$$

$$\Rightarrow \frac{\bar{h}'(s)}{F_i'(s)} = \frac{k_p s}{\tau^2 s^2 + 2\zeta\tau s + 1} \checkmark$$

$$k_p = \frac{\tau_i}{k_c} ; \quad \tau^2 = \frac{\tau_i}{k_c} (A + k_c \tau_d) ; \quad 2\zeta\tau = \tau_i$$

$$\zeta = \frac{1}{2} \sqrt{\frac{k_c \tau_i}{A + k_c \tau_d}} \checkmark$$

Classification based on ζ :

①	②	$\sqrt{\frac{k_c \tau_i}{A + k_c \tau_d}} < 2$	i.e. $\zeta < 1$	i.e. underdamped.
✓		$= 2$	$\zeta = 1$	critically damped
		> 2	$\zeta > 1$	overdamped.

Now, here k_p is equal to τ_i divided by k_c . Similarly τ^2 is equal to τ_i divided by $k_c A + k_c \tau_d$ and $2\zeta\tau$ is equal to τ_i . We have considered in the derivation of a transfer function these simplified forms. I mean this correlation has been used in the derivation of transfer function.

So, from these equations we obtain, ζ equals half root over of $k_c \tau_i$, divided by $A + k_c \tau_d$. These expressions for ζ we obtain from these 2 forms. Now we want to conclude. So, our first conclusion is that by employing a controller around a first-order system, we obtain overall second-order response. This is our first conclusion by employing a controller; around a first-order system we obtain overall second-order system. And in the second conclusion we need to know whether, it is over damped or critically damped or under damped response.

If, $\sqrt{kc \tau i}$ divided by $a + kc \tau d$ less than 2 then zeta is less than 1; That means, this is under damped response. If this is equal to 2; that means, zeta is equal to 1. So, it is critically damped fine similarly if this is greater than 2 then zeta is greater than 1 then it is the case of over damped response fine. So, our first conclusion is the overall response become second-order and it may be over damped may be under damped may be critically damped depending on the value of this zeta.