

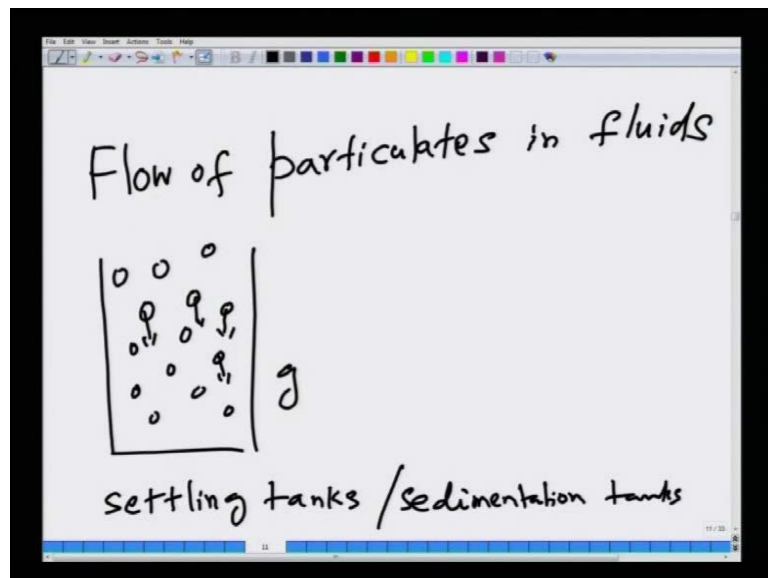
Fluid Mechanics
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Lecture No. # 36

Welcome to lecture number 36 on this NPTEL course on fluid mechanics for undergraduate students chemical of engineering.

Today we are going to discuss fluid mechanics, application of fluid mechanics to various chemical engineering processes. And one of the major topics in chemical engineering an application is the flow of particulates in fluids.

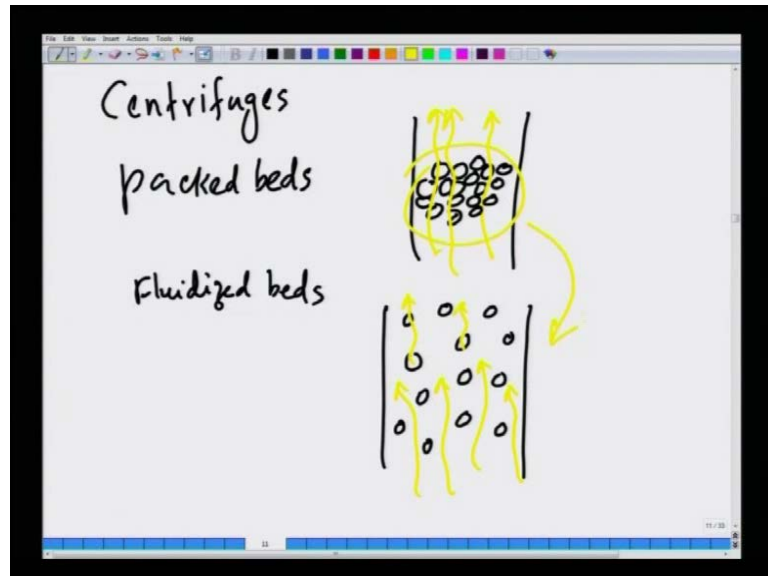
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In many chemical engineering applications for example, you have a system like channel in which you have many particles that are settling in a liquid. These are called settling tanks. So, here particles are moving and settling in a fluid due to gravity and this is used to separate particles of different sizes because of their difference in settling velocities as we will see. So, this one example there is settling tanks or sedimentation tanks as they are called in many chemical engineering applications. You will have sedimentation tanks

that are used to remove particulates from one liquid phase and then use the liquid phase for further processing in other unit operations.

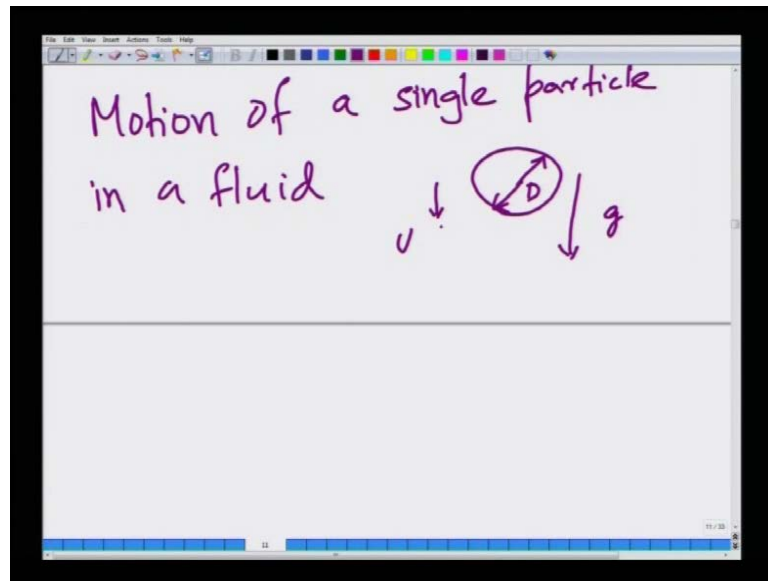
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And you also have centrifuges to separate particles of different densities, particles or droplets of different densities. So, here the driving force here again it is also settling, but, under the influence of centrifugal driving forces. And you can have other unit operations such as packed beds wherein you have a bed of particles which are in close contact and there is lot of gap in between the particles and fluid is flowing.

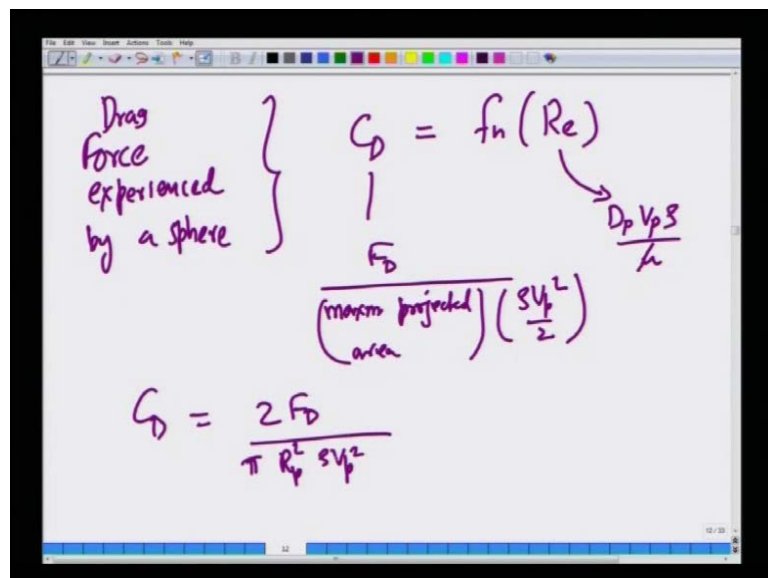
Fluid is flowing in the interstitial gap between these particles. And there are also other applications such as fluidized beds wherein you have the motion of fluids through a bed of particles. But, the particles themselves are no longer static, but, they are also in some sense fluid is suspended. So, they are also in a state of animated motion due to the fact that the drag force exerted by the fluid on the particles exceeds the weight of the particles. So, the bed of particles starts fluidizing after some velocity. So, all these are examples of motion of particles in a fluid. And in order to design these operations well in many chemical engineering operations unit operations you have to understand the fundamental mechanics of flow of particulates through a liquid.

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So, one of the simplest problem that we will first start with is flow or motion of a single particle in a fluid. (No audio from 03:24 to 03:34). So, essentially we will imagine that you have a spherical particle, rigid particle to begin of some radius or diameter D . And that is moving with a constant velocity under the influence of gravity. Let us say and we want to be able to predict what is the settling velocity of this particle; as a function of various parameters in the problem.

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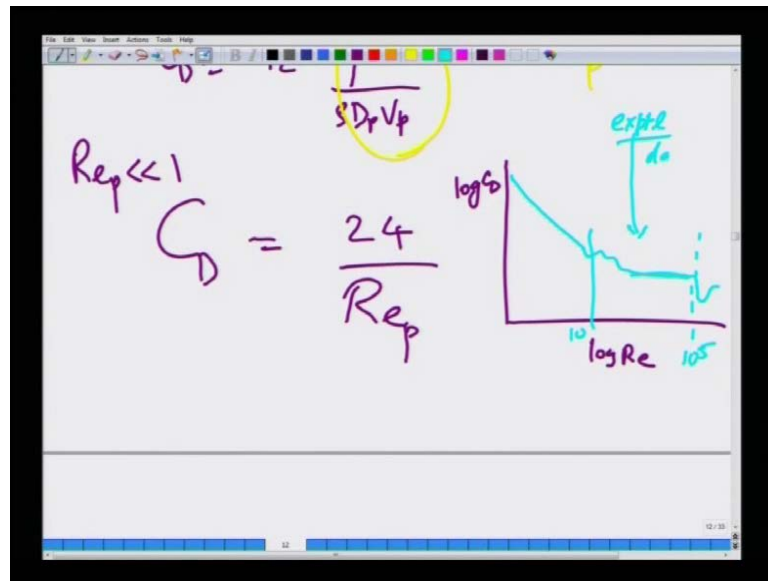
Now, we already know that the drag force experienced by a sphere is correlated in terms of non-dimensional groups C_D as a function of the Reynolds number, where C_D is nothing, but, $2 f D$ the force divided by maximum projected area times ρV_p square by 2. And then Reynolds number is of course defined as $D V_p \rho$ by μ where D V_p and V_p are the diameter and velocity of the particles. F_D is the drag force experienced by the particle. So, C_D is nothing, but, $2 f D$ by π or ρV_p square.

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The image shows a whiteboard with handwritten mathematical equations. The first equation is $C_D = \frac{2 \cdot 6 \pi R_p \eta V_p}{\pi R_p^2 5 V_p^2}$. The second equation is $C_D = 12 \frac{\eta}{5 R_p V_p}$ with $R_p = \frac{D_p}{2}$ written to the right. The third equation is $C_D = 12 \cdot 2 \frac{\eta}{5 D_p V_p}$, where the fraction $\frac{\eta}{5 D_p V_p}$ is circled in yellow and an arrow points to the label Re_p .

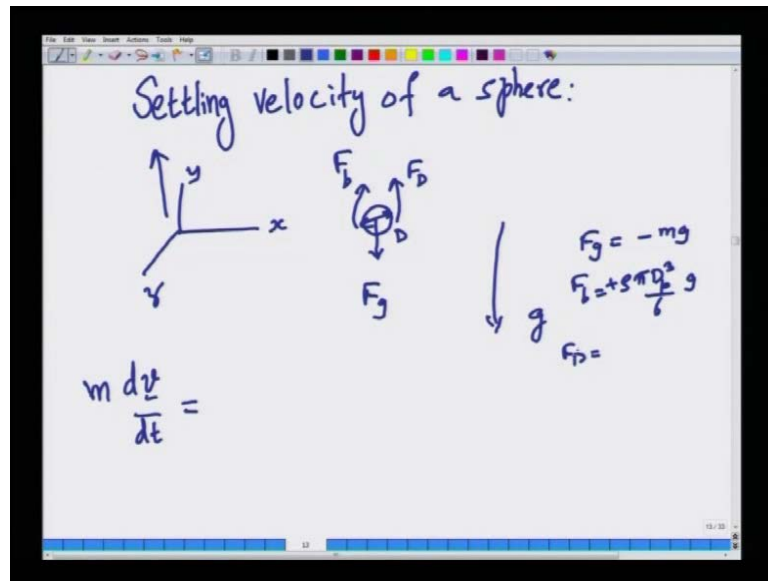
Now, if you substitute for Reynolds number small compared to one the drag force is given by the Stokes drag law $6 \pi R_p \eta V_p$. So, C_D is given by 2 times $6 \pi R_p \eta V_p$ divided by $\pi R_p^2 \rho V_p^2$. So, C_D for a spherical particle, at low Reynolds numbers given by, so, 2 times 6 is 12 and then the π cancels off. One V_p will cancel with one V_p and R_p will cancel with one R_p will give you 12 by 12 times η by $\rho R_p V_p$; R_p is D_p by 2. So, C_D is 12 times η 2 by $\rho D_p V_p$ and this quantity is the Reynolds number based on the particle diameter and velocity.

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So, the drag coefficient is given by 24 by divided by Re based on the particle for Reynolds number of the based on the particle much small compared to 1. Of course, at high Reynolds numbers we showed that we have $\log C_D \log Re$ plot and it will appear like this. For smaller Reynolds numbers is this constant. It is a straight line with slope minus 1, but, at higher Reynolds numbers of course, things are different and you will find that there is a reason of constant relatively constant drag coefficient. And then for Reynolds number of 10 to the 5 is about 10. You have transition from laminar flow to turbulence. So, we have this data. This is an experimental observation correlated in terms of drag coefficient and Reynolds number experimental data.

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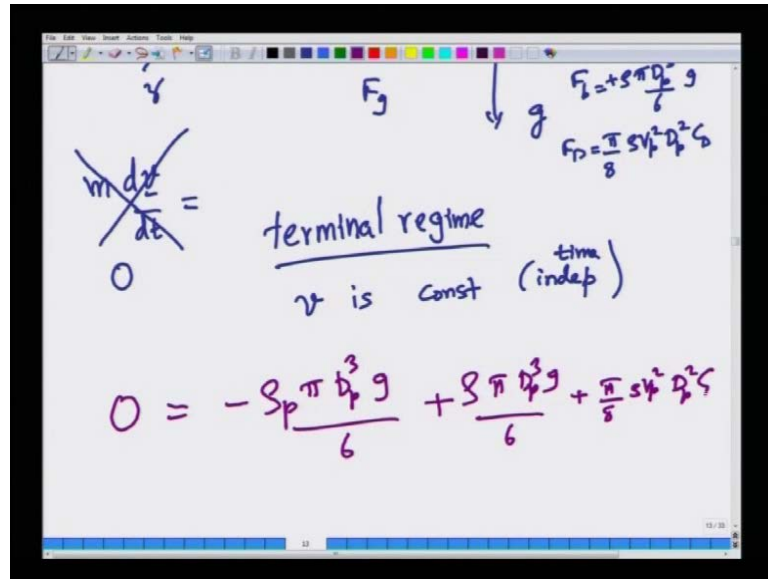
So, now we want to worry about settling velocity of a sphere. Imagine you have an infinite expanse of fluid and then you take a sphere and you drop it in the fluid in the liquid and there is gravity. After some time let us say you drop it with 0 velocity initially the velocity of the sphere is 0. After some time, initially, the sphere will accelerate because there is acceleration due to gravity acting on it, but, it is also being retracted by the fluid drag force. This is gravity force let us call it F_g . The gravity force that acts and then there is drag force that acts in the opposite direction. There is also buoyancy force that acts in the opposite direction.

So, these are the three forces. Now, initially, there will be an imbalance if you write force balance for the particle. So, the mass times rate of change of velocity acceleration of the particle is sum of all the forces. So, let us assume a coordinate frame x , y and z . So, the velocity mass times DV by Dt sum of all forces and the gravity force is in let us simply write the algebraic quantities of all the algebraic. So, let us use the sign convention the gravity force F_g is acting its minus mg buoyancy is plus the volume of the sphere let us assume the diameter of the sphere to be some D .

So, the buoyancy will act in the plus y direction because it is buoyancy acts in this direction the direction opposite to the motion of the particle. So, buoyancy will act in the plus direction which is $\rho \pi D^3/6 \cdot g$. This is plus then there is drag force which is given by it is also in the plus y direction. So, the drag coefficient is defined in

the following way. Let us look at the definition of the drag coefficient here. From here we can drag force which is nothing, but, $\pi R^2 \rho V^2 C_D$. Now if you write R as V_p divided by 2 then F_D will be simply π by 8 $\rho V_p^2 D^2 C_D$ times C_D of course, times C_D . So, I will write if F_D the drag force in terms of the drag coefficient.

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So, the drag force is π by 8 times $\rho V_p^2 D^2 C_D$. So, at steady state you will have no acceleration when the sphere has reached the terminal velocity. Sphere has reached a velocity such as all the forces balance its acceleration is zero. So, this is called the terminal regime, wherein the sphere velocity does not change with time the velocity of the sphere is constant that is independent of time.

So, you have to simply balance all the three forces and the three forces are simply zero is minus ρ_p by $D^3 \rho_p g$ by 6. This is the weight, ρ_p is the density of the particle times volume of the particle times g is the weight of the particle; it acts in the minus y direction times ρ_f times $\pi D^3 g$ divided by 6 plus π by 8 $\rho_f V_p^2 D^2 C_D$.

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The image shows a whiteboard with a digital drawing application interface. At the top, the force balance equation is written: $0 = -\frac{\rho_p \pi d_p^3 g}{6} + \frac{\rho \pi d_p^3 g}{6} + \frac{\pi}{8} s d_p^2 V^2 C_D$. Below this, the terminal velocity equation is boxed: $V_p = \sqrt{\frac{4}{3} \frac{(\rho_p - \rho)}{s} \frac{d_p g}{C_D}}$. Underneath the box, the text "Terminal velocity (settling)" is written in orange.

So, if I eliminate for V_p we will get V_p is equal to square root if I solve for this equation for V_p you will get 4 divided by 3 times ρ_p minus ρ divided by $\rho D_p g$ divided the drag coefficient. So, this is the terminal steady terminal velocity of a sphere. Expression for terminal settling velocity of settling of a sphere of diameter D_p which is moving on density ρ_p which is moving in a fluid of density ρ .

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The image shows a whiteboard with a digital drawing application interface. At the top, the text "Terminal velocity (settling)" is written in orange. Below this, the Reynolds number condition is written: $Re \ll 1: C_D = \frac{24}{Re_p}$. The terminal velocity equation is then written as: $V_p = \left[\frac{4}{3} \frac{(\rho_p - \rho)}{s} \frac{d_p g}{24} Re \right]^{1/2}$.

Now, of course, we have to look at different regimes when Reynolds numbers is small compared to one we know that C_D is 24 divided by Reynolds numbers based on particle

diameter. So, V_p becomes $\frac{4}{3} \rho_p - \rho_f$ divided by $\rho_f \mu$ times $D_p^3 g$ by 24 Reynolds number to the power half.

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The image shows a whiteboard with handwritten mathematical derivations. At the top, there are some numbers in brackets: $\left[\begin{matrix} 3 & 5 & 24 \\ & & 6 \end{matrix} \right]$. Below this, the settling velocity equation is written as $V_p = \left[\frac{1}{18} \frac{(\rho_p - \rho_f) D_p^3 g}{\rho_f \mu} \right]$. The 24 in the denominator and the 6 in the denominator are crossed out, and a 6 is written below the 24 . Below this, the equation is simplified to $V_p = \frac{(\rho_p - \rho_f) D_p^2 g}{18 \mu}$. To the right of this simplified equation, it is noted that $V_p \propto D_p^2$ and $Re_p \ll 1$.

So, V_p is nothing, but, or we can square the entire expression V_p square is nothing, but, we will get. So, you have $4/6$. So, you get 18 here you have 1 over 18 $\rho_p - \rho_f$ divided by $\rho_f \mu$ times $D_p^3 g$ times. We have substitute for Reynolds numbers is $D_p \rho_f V_p$ divided by μ .

Now, this fluid density will cancel with this density one factor of V_p will cancel with one factor of V_p here to give you V_p is nothing, but, $\rho_p - \rho_f D_p^2 g$ divided by 18μ . So, for low Reynolds numbers the settling velocity is given by this expression. So, notice that the settling velocity for low Reynolds numbers is proportional to square of D_p particle diameter. So, that is the key if everything remains constant. If you just worry about the variation of the settling velocity on the particle diameter it goes as the square of the particle diameter. Now, let us look at high Reynolds numbers.

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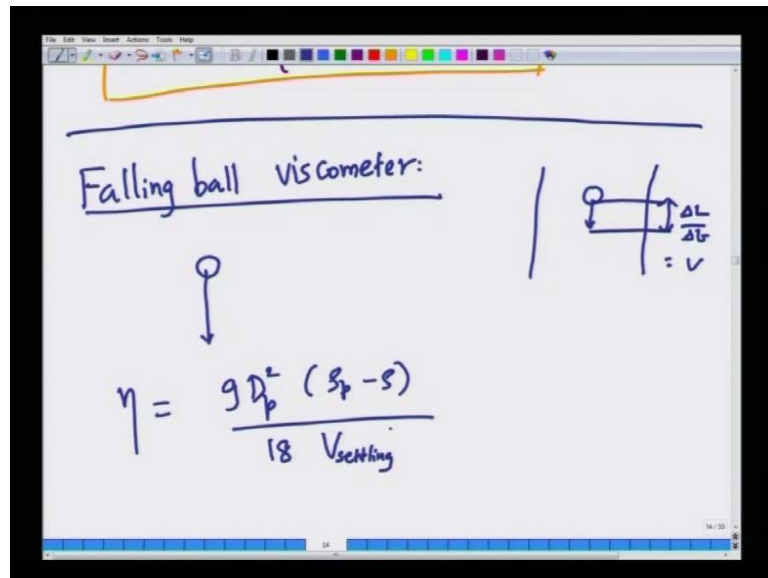
The whiteboard contains the following handwritten notes:

- At the top left, the number "18" is written and circled in yellow.
- In the center, a boxed equation shows the settling velocity: $V_p = \frac{(\rho_p - \rho) D_p^2 g}{18 \eta}$.
- To the right of this equation, two notes are circled in yellow: $V_p \propto D_p^2$ and $Re_p \ll 1$.
- Below the boxed equation, a note states: "When $10^3 < Re < 2 \times 10^5$ " with a yellow underline, and $C_D \approx 0.44$ below it.
- At the bottom, a boxed equation shows the settling velocity for the intermediate regime: $V_b \approx \left[\frac{3 D_p g (\rho_p - \rho)}{\rho} \right]^{1/2}$.
- To the right of this equation, two notes are circled in yellow: $V_p \propto D_p^{1/2}$ and $10^3 < Re < 10^5$.

Now, I told you that when Reynolds numbers is in between 1000 and twice 10 to the 5 there is a region of constant drag coefficient. If you look at this data here C_D is approximately constant. So, we are looking at this regime where C_D is approximately constant. It is a number and the number is approximately 0.44. So, then we have to merely substitute 0.44 in this expression for C_D and then we will get an expression for the settling velocity. And the settling velocity for this regime is approximately $3 D_p g (\rho_p - \rho) / \rho$ to the power half. Now, notice here that this is the settling velocity at the relatively high Reynolds numbers regime between 10 to the 3 and twice 10 to the 5, 10 to the 3 and 10 to the 5.

Notice that the settling velocity is proportional to diameter of the particle to the power half. So, there is a square root here also. So, the way in which the settling velocity depends on the diameter of the sphere is very different whether the Reynolds numbers is small or large. It goes as settling velocity goes as diameter of the particle squared at low Reynolds numbers while it goes diameters of the particle at half for high Reynolds numbers.

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Now, this expression can be used to measure the viscosity of a fluid and such a device is called the falling ball viscometer. Essentially the idea is you drop a sphere in a fluid and wait for it. You take a sphere and replace it in a liquid whose viscosity you want to find and wait for it to attain terminal velocity. Calculate the velocity by noting how much distance it travels over a particular time t and that will give you the terminal velocity settling velocity. So, the settling terminal velocity is measured in the experiment by simply measuring the time particles takes to move a certain distance and from there we can calculate the viscosity of the liquid in which it is settling through other parameters.

So, divided by so, merely inverting this expression. So, if the Reynolds number is low. So, we can pull θ here and V_p here and that is all we have done in this expression. And to get an expression for viscosity provided you know what is the diameter density of particle density of fluid and you measure the settling velocity; velocity of settling of the particle.

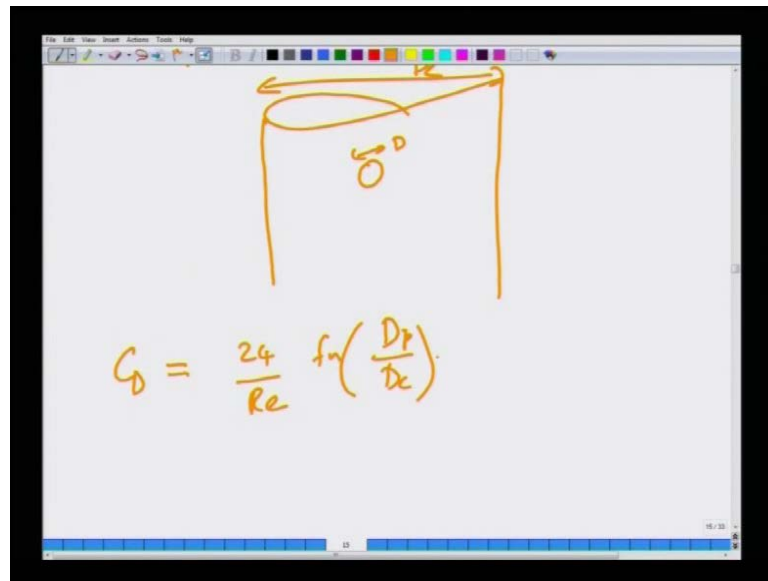
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The image shows a whiteboard with handwritten notes. At the top, the equation $\eta = \frac{9 D_p^2 (s_p - s)}{18 v_{\text{settling}}}$ is written, with $(Re_p < 1)$ to its right. A box is drawn around the equation, and an arrow points from the word "settling" in the denominator to the text "measure settling". Below this, the equation $F = 6\pi R_p v_p \eta$ is written, with "Stokes drag law" and " $Re_p \ll 1$ " written to its right. A vertical line with a downward arrow connects the two equations. At the bottom, the text "fluid is infinite in expanse." is written.

So, this is often used to measure the viscosity in many industrial settings because it is easy first of all to do such an experiment and this is assuming that Reynolds number is small. Because, you choose your particle dimensions such that the Reynolds number based on the particle is very small. Now, in the above when we used the Stokes drag law.

So, this is the Stokes drag law of valid at small values of particle Reynolds number. When we use the stroke drag law we are using, we are assuming that the fluid is infinite in expanse. That is the there are no surrounding walls. But in reality there will be surrounding walls.

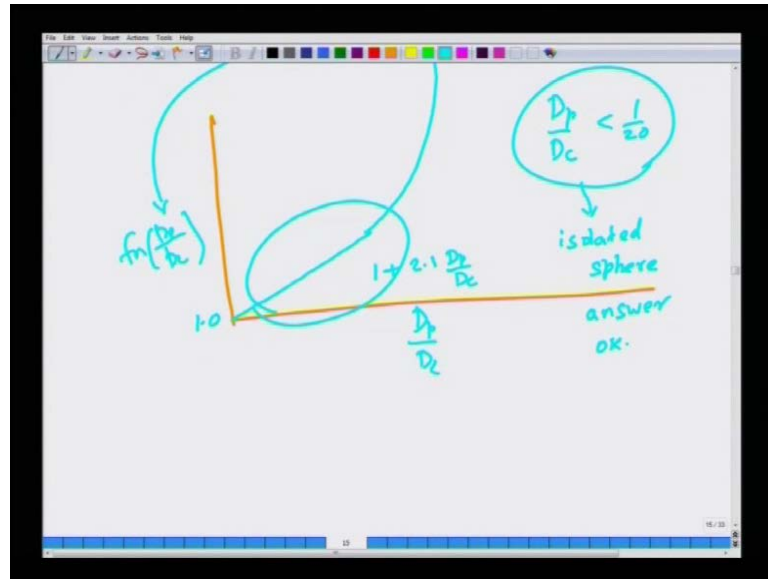
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Suppose you may do an experiment in a container and therefore, you have diameter of the particle D and then the diameter of the container let us say D_c . In principle you may expect that when the diameter of the particle is very small compared to the diameter of the cylinder in which it is flowing, it is moving. Then you would assume that the wall effect on the drag law drag law is negligible.

Because this drag law is valid only if the walls are infinitely away from the particle, but, in reality that is not the case because we do experiments often in laboratory where there are confining walls of cylinder or whichever container we are doing experiments. But, you can also correct for that. So, the drag coefficient is 24 by Re times some function of D_p by D_c , where people have found using experiments what this functional form is.

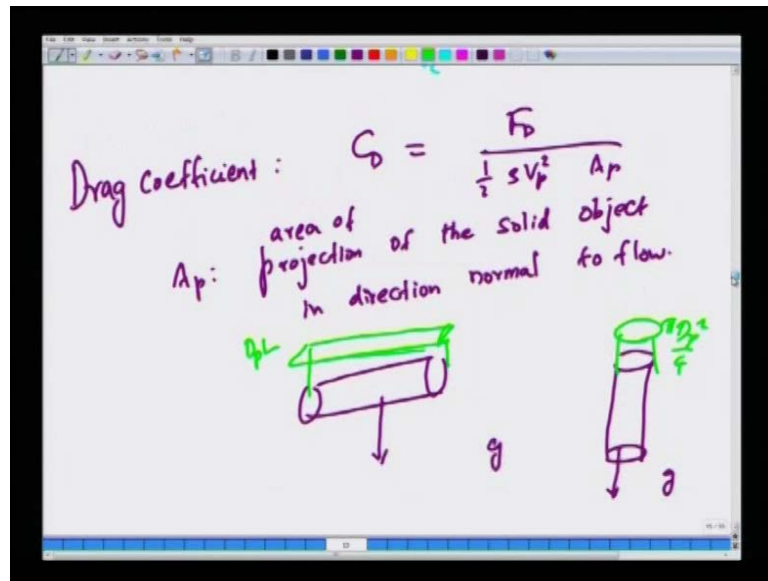
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So, initially it goes as 1 plus. It goes 1 plus this function is what I am plotting as a function of D_p over D_c . Initially it goes 1 plus 2 point one times D_p by D_c . So, when D_p equals D_c of course, we expect the function to go to 1 and then it starts deviating. So, the confining walls tend to increase the drag because there is additional dissipation in the problem compared to when you have no confining walls. So, typically when D_p by D_c is less than 1 over 20 then the isolated sphere results is pretty good; answer is pretty good. But, when the confining walls or when D_p by D_c is such that it is not as small then you will start seeing the effects of containing walls confining walls and this is given by this correction.

So, we have seen what is meant by settling of a sphere and we will we are going to now first understand that it is not just that drag coefficients are defined only for spheres. You can have in many applications particles of other shapes also.

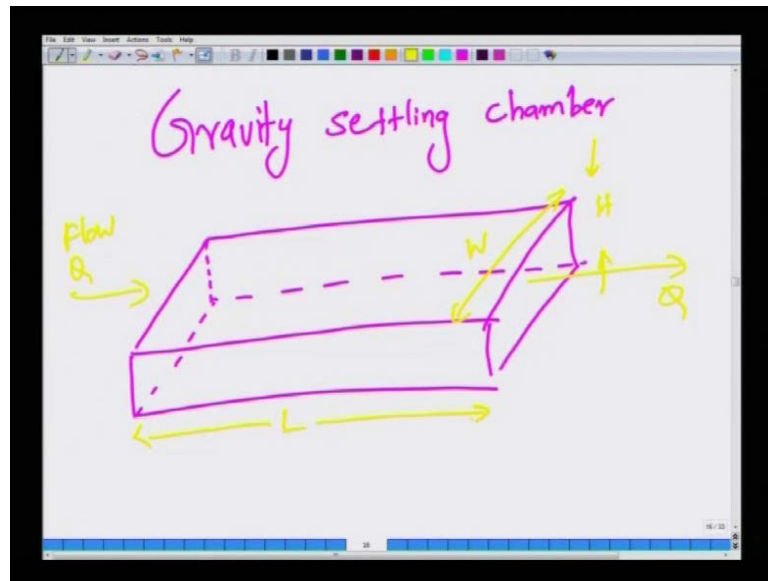
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So, a general definition of drag coefficient is simply C_D is the drag force divided by half $\rho V_p^2 A_p$ where A_p is the projection of the solid object on a plane normal to the direction of flow; Area of projection of the solid object in the direction normal to flow. So, for example, if you have a cylinder that is moving settling suppose, you have cylinder settling like this and you have cylinder settling like this and the under influence of gravity the projected area here is very different. Here the projected area is simply it is going to appear like a rectangle. Here the projected area is going to appear like a circle of radius πD_p^2 by 4 or whereas, over here the projection area is πD_p times l .

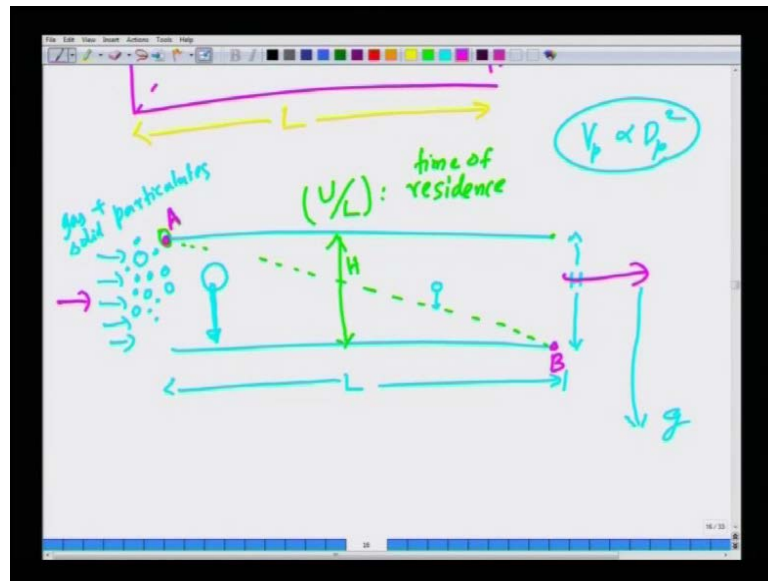
So, it is very different depending on sorry not there is no π here it is D_p times L because it is going to appear like rectangle of dimensions D_p times l . So, the drag coefficient is defined differently because the projected area is very different and similar drag loss. I mean that this how the drag coefficient depends on Reynolds number is empirically found using experiments for other geometry such as cylinders as well. But, those are matters of experimental detail which can be found in text books or those are just matter just empirical data which can found in text books. But, the idea is that you do have similar drag coefficient versus Reynolds number relations for other geometries as well.

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Now, I am going to do an application and the application is as follows. You want to have separation of particulates by virtue of the difference in their size for example. Suppose, you have how it is done is you see what is called a gravity settling chamber? (No audio from 23:14 to 23:23). Essentially, you have a box channel through which you have flow of a particulate. This is a suspension of it is this is like a suspension of some fly particulates like solid particles that are suspended in a fluid and the flow rate is Q . And width of the channel is W in this direction and height of the channel is H the gap in which particulate is flowing its flowing out in this direction and the length of the channel is l .

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So, let us assume W is very large. So, we can treat the problem as a 2 dimensional problem. So, you imagine you have a channel height is H and the length is L and you have a mixture of gas solid mixture gas plus tiny solid particulates that flow in. And we want to make sure when the gas when this mixture flows out we want to separate out the solid from the gas. And that happens by the virtue of the gravity which acts in this direction and settling. But, typically in industrial applications this in incoming mixture will have a size distribution of particles not all particles will have the same size. It will have particles of varying sizes.

So, but, we want to know. So, from settling from the basic idea of settling you will imagine that bigger particles will settle sooner compared to smaller particles. Because, the drag force sorry the let us let us look at this settling velocity the settling velocity at low Reynolds numbers is proportional to D_p^2 . So, the settling velocity is proportional to D_p^2 which is what we derived just now.

So, it is higher by particles with higher diameters are going to settle more quickly compared to particles with smaller diameter. So, you can ask the question suppose I start from here. What is the smallest particle that I can capture in a length L ? Suppose, you have a settling chamber of some length L and the gap with H what is the smallest particle that you can capture typically in a settling chamber. So, let us start with some position a here.

Now, if this particle has to be trapped over a distance L then it must settle or it must travel a distance of H within a time where wherein it resides in this settling chamber. That residence time is typically the average velocity divided by the length of the, this is typically the residence time; time of residence. A particle will approximately stay over a time of order u by L because it has to travel with an average velocity city u over a distance l . So, the time the particle spends in this chamber is u by l .

Now, within this time this particle which started out here must reach here as it moves. So, the particle will do this the trajectory of the particle will be something like this. So, if you consider a particle that is bigger than this particle that is anyway going to settle quickly. If you consider a particle that is smaller it is not going to settle. It is not going to settle within the chamber it is going to fly away with the flow. So, there is flow in this direction. So, which is that particle which will exactly start here at A and end here at B and that particle will be the cut off diameter which can be separated by this settling chamber.

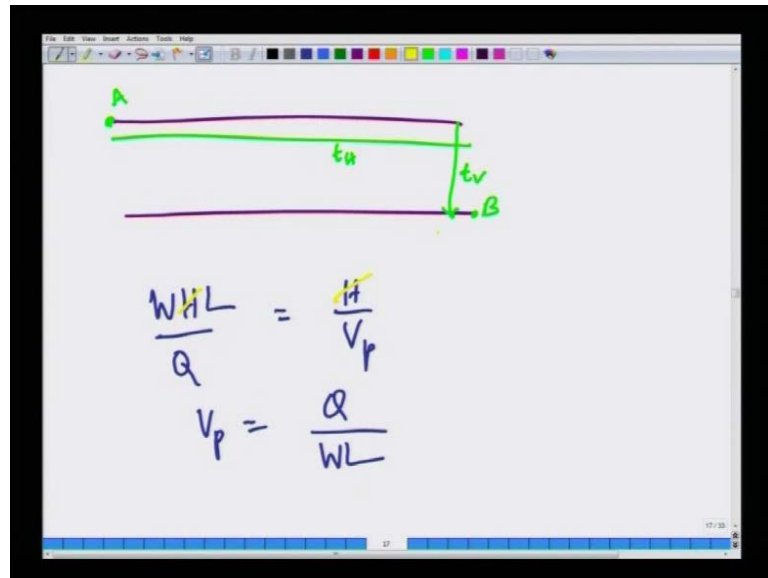
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The image shows a whiteboard with handwritten mathematical derivations. At the top, the horizontal velocity of the fluid is given as $V_f = \frac{Q}{WH}$. Below this, the time of residence is defined as $t_H = \frac{L}{V_f}$, which is then simplified to $t_H = \frac{WHL}{Q}$. A third equation states that if a particle falls with a settling velocity V_p , the time taken to settle over a distance H is $t_s = \frac{H}{V_p}$.

So, let us look at this analysis. The horizontal velocity of the fluid is simply average velocity of the flow is simply the volumetric flow rate divided by the area. Area is the cross section area of flow, which is width, which is into the board times the gap thickness h . A residence time of the particle the time, the particle typically spends in the chamber is simply L by V_f which is nothing, but, WHL by Q . Now, if the particle falls

with the settling velocity V_p the time it takes taken to settle over a distance H is simply H divided by V_p . This is let us call it t_v .

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Now, if you imagine that you wanted a particle which started at A and ends exactly at B. For such a particle the time of residence which is the time it spends over a length L must be exactly equal to the time of settling. So, t_H is the residence time velocity time. So, it must be equal to settling time. So, equate the two times we will have WHL divided by Q is nothing, but, H divided by V_p or V_p is Q divided by WL as H cancels from both sides.

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A whiteboard with a black border showing handwritten mathematical derivations. At the top, the equation $\frac{Q}{WL} = v_p$ is written. Below it, the equation $v_p = \frac{Q}{WL}$ is written. A yellow arrow points from this equation down to the text "Small particles $Re_p \ll 1$ ". Below the text, the equation $v_p = \frac{g D_p^2 (\rho_p - \rho)}{18 \eta}$ is written. A second yellow arrow points from the text "Small particles $Re_p \ll 1$ " to this equation.

Now, we have let us say, we have small particles and let us assume that Reynolds number of based on particle is small compared to 1. Then we will use the settling velocity that we just derived for such particles which is $g D p$ square ρp minus ρ divided by 18η . We are going to substitute this out here.

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A whiteboard with a black border showing handwritten mathematical derivations. The equation $\frac{g D_p^2 (\rho_p - \rho)}{18 \eta} = \frac{Q}{WL}$ is written. Below it, the equation $D_p = \left[\frac{18 \eta Q}{g WL (\rho_p - \rho)} \right]^{1/2}$ is written and enclosed in a yellow box. Two yellow arrows point from the bottom of the box to the terms $g WL (\rho_p - \rho)$ in the denominator.

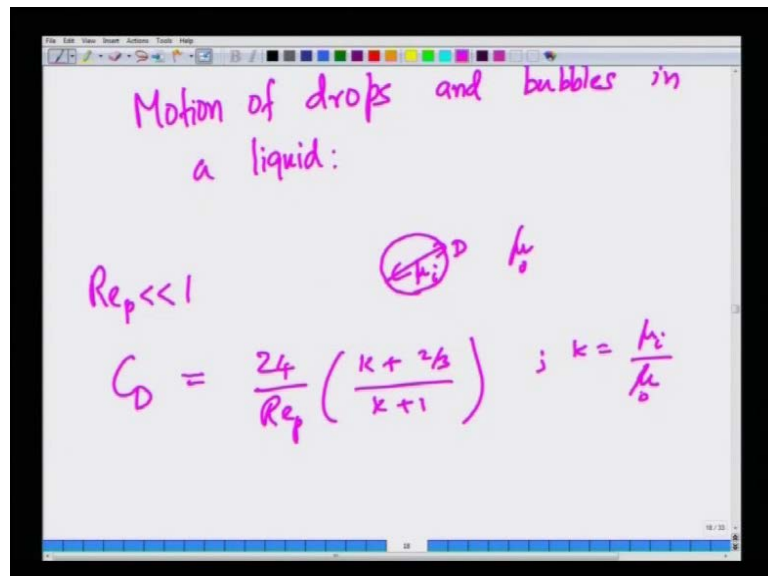
So, we will get $g D p$ square ρp minus ρ by 18η is Q divided $W L$ or $D p$ is nothing, but, $18 \eta Q$ divided by $g W L$ times ρp minus ρ the whole thing to the power half. Now, this is the smallest diameter typically that can be captured by in the

settling tank of this dimension. If Q is the volumetric flow rate of the mixture it has the viscosity of the liquid, g is the acceleration due to gravity, W is the width into the paper, L is the length of the channel, ρ_p is the density of the particles and ρ is the density of the fluid in which it is travelled it is moving. This is the diameter of the smallest particle that can that typically the diameter of the smallest particle.

Now, if a smaller particle it is in principle possible that a smaller particle than this D_p enters the chamber somewhat below A and it can be captured before B itself. But, that is something that is something that we can predict with certainty. So, this is typically because the incoming particles will come with distribution and each particle will enter the chamber at different vertical locations.

So, obviously, that can be smaller particles that can that can settle in the chamber, but, we can make with the following statement with certainly that no particle larger than D_p will settle within the chamber. Now, so far we have been discussing the settling and motion of rigid particles. There is also case in many chemical engineering applications where you have to motion of drops and bubbles in a liquid. (No audio from 31:59 to 32:09).

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Now, if you have a drop of some diameter D and let us say the drop viscosity is μ_i and the outside viscosity is μ_o . Then at low Reynolds numbers based on the drop small compared to 1 the drag coefficient is given by 24 divided by Re_p times k plus 2 divided

by 3 divided by k plus 1 where k is nothing, but, μ_i divided by μ_o . So, the ratio of inner fluid viscosity drops viscosity to the outer viscosity.

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$Re_p \ll 1$

$C_D = \frac{24}{Re_p} \left(\frac{k + \frac{2}{3}}{k + 1} \right)$; $k = \frac{\mu_i}{\mu_o}$

$\mu_i \ll \mu_o$: gas bubble
 $k \rightarrow 0$

$\mu_i \gg \mu_o$: rigid sphere
 $k \rightarrow \infty$

$C_D = \frac{16}{Re}$

$C_D \approx \frac{24}{Re}$

So, we have two limiting cases where μ_i is very small compared to μ_o you get the motion of an air bubble or a bubble, gas bubble. And you will get C_D in that limit. So, μ_i is less than μ_o k tends to 0, if, I put k tends to 0 in this case I will get 16 by Re . Now, if μ_i is very large compared to μ_o that is k tending to infinity you get a rigid object rigid sphere. So, C_D becomes 24 by Re . So, it does reduce to the well-known cases of the; it does reduce correctly to the result of 24 by Re when the viscosity of the inner fluid is very large compared to the viscosity of the outer fluid.

And we also have a new result that for the motion of the gas bubble suppose you have a gas bubble in water air bubble in water it is going to rise. What is the terminal rise velocity at low Reynolds numbers? Well we can use this relation and then find out the terminal rise velocity in a very similar way that we did for terminal settling velocity.

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The image shows a whiteboard with handwritten equations in purple ink. The first equation is $a_c = r\omega^2$. The second equation is $0 = r\omega^2 \rho_p \left(1 - \frac{\rho}{\rho_p}\right) - \frac{C_D S V^2 \rho_p}{2m}$. The final equation, enclosed in a yellow box, is $V_{\text{terminal}} = \omega \sqrt{\frac{2r(\rho_p - \rho)m}{\rho_p C_D S}}$.

Now, it does not just because of the driving force for settling is typically gravity in many applications, but, in many of in some application you do find the use of centrifugal driving forces that drive the motion of the particle in a fluid. So, the centrifugal happen forces happen whenever there is a rotation of fluid flow. So, whenever the direct of part of. So, when you have a suspension of a liquid in of solid in liquid and then if you want to separate these particles based on their sizes or densities. You can use gravity, but, gravity you cannot manipulate gravity because it is fixed on the surface earth.

So, if you were to use, if you want to speed up the process, if you want to have more control over the process, you could use centrifugal forces because these forces also appear as body forces. Because, this is like motion in an accelerating frame of reference it is a rotating frame of reference. So, you have fictitious forces which will act like body forces and that force proportional to $r \omega^2$ the acceleration centrifugal acceleration proportion to $r \omega^2$.

So, if you balance the centrifugal acceleration with the drag force you will get $0 = r \omega^2 \rho_p \left(1 - \frac{\rho}{\rho_p}\right) - \frac{C_D S V^2 \rho_p}{2m}$. This is the net driving, net force along the direction of the body force because you have to account for buoyancy minus the drag force which is opposing this. So, $C_D S V^2 \rho_p / 2m$. So, the terminal velocity in a centrifugal field is written as $\omega \sqrt{\frac{2r(\rho_p - \rho)m}{\rho_p C_D S}}$.

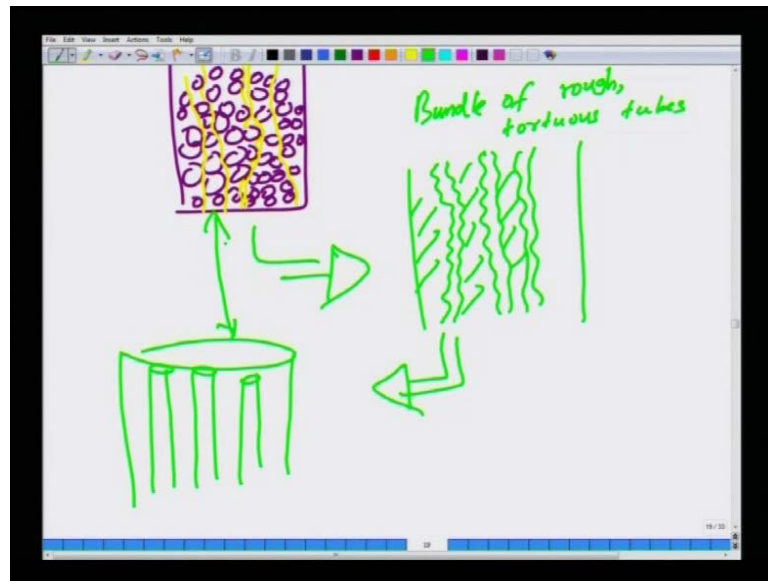
So, once you know what $C D$ is you can actually predict the terminal velocity in a settling of settling even in a centrifugal field. This is often used in you know you know if you have a dairy product milk is basically emulsion of droplets of one liquid in another. Suppose, you want to remove fat from the milk and since fat has different density compared to the other constituents of milk you can centrifuge the milk to separate the fat from the rest of the milk.

So, this is how the process skimming is done in a milk using centrifuges and there if you want to design those centrifuges you have to have clear cut idea of what is the terminal velocity. In order to design the centrifuge, what is the size of the centrifuge you need and so on? So, such calculations are very useful in the design of these unit operations.

So, the next topic that we are going to discuss is related to flow of particulates in a fluid or we are going to look at the flow of fluid through a bed of particles which is slightly opposite different from what we just did. We so far, we considered the motion of particles in a fluid now we are going to consider the motion of fluid in a bed of particles. Such beds or such operations are called packed beds in chemical engineering applications. The reason why packed beds are used in many unit of operations such as adsorption or even reaction is that the packing of a higher surface area per unit volume for gas liquid contacting.

So, typically you may want to conduct an absorption process in a packed bed because you want to contact essentially a gas phase with a liquid phase and transfer species across the interface between gas and liquid. And clearly the amount of I mean the extent the rate at which the transfer can happen will depend on how much area is available for the transfer. And if you provide a packing that essentially increases the contact area of mass transfer per unit of volume of the bed. So, because of this high surface areas high interfacial area of mass transfer that is available these are often used in chemical engineering applications.

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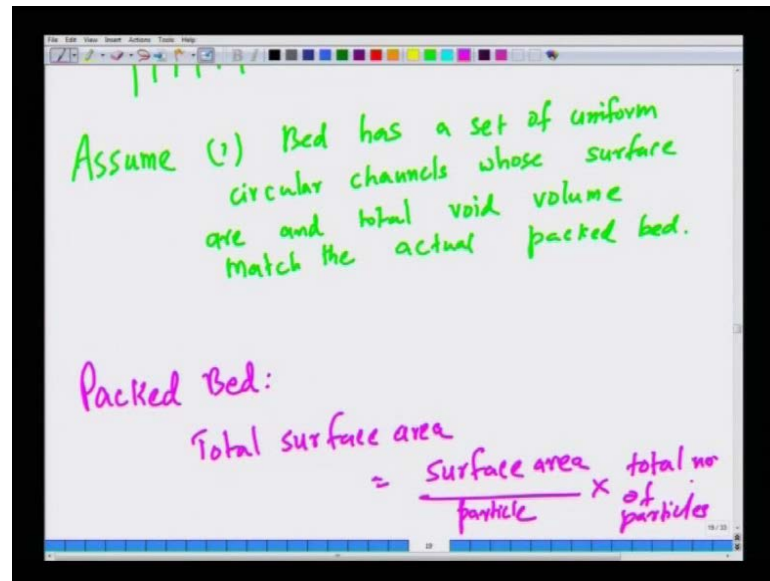
So, the next topic that we are going to do is packed beds. So, it is going to schematically appear like this you have a bed of particle and fluid is going to flow in this interstitial voids or interstitial spaces between these various particles. If you want to design a packed bed for a chemical engineering application, you want to know what is the pressure drop to make a fluid flow with some flow rate Q . So, this is the typical design question that comes up in designing packed beds. So, how are we going to do that? Now, if we were to draw a cartoon of the interstitial spaces, it may look like this.

So, and you can draw several such contours of interstitial spaces through which fluid is going to flow. So, like this and so on. So, there are going to be gaps through which fluid is going to flow and essentially the flow through a packed bed can be thought of as a flow through a bundle of corrugated tubes. So, you can imagine each path can be like a tube and there are several such paths. So, there is empty space within the tube and the rest is all solid filling. So, this is the model that we are going to use. A packed bed can be thought of as a bundle of rough and highly tortuous tubes of some complex curve cross section; rough tortuous tubes.

Now, we are going to assume further simplification since this problem is way too complex. We are going to assume this can be simplified as a bed of uniform circular tubes such that the surface area of this problem and the total void volume in this problem will match with that of the real packed bed.

So, that is the only connection that we are going to essentially think of flow through a packed bed as a flow through a bundle of tubes. Although, in reality these tubes are extremely complex in their cross section and they are very corrugated. But, we are going to assume this is going to look like a straight tube, but, we are going to make the connection between the model and reality by saying the following that.

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Assume that the bed has a set of uniform circular channels whose surface area and total void volume match the actual packed bed. So, this is the only connection we are going to make with the real packed bed. So, let us first look at the real packed bed. The real packed bed the total surface area is nothing, but, of particles is equal to the surface area per particle times the number of particles total number of particles. So, that is the total surface area available for the in the real packed bed.

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Volume fraction of particles

$$= \frac{\text{Volume of particles in bed}}{\text{total volume of bed}}$$
$$= \frac{\text{total no of particles} \times \text{volume of single particle}}{\text{total volume of bed}}$$

Now, the volume fraction of particles is nothing, but, volume of particles in the bed divided by the total volume of the bed. This is the volume fraction of particles it is essentially the amount of volume that is occupied by the particles divided by the total volume. This is nothing, but, the total number of particles times volume of one particle single particle divided by the total bed volume.

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porosity (ϵ) = $\frac{\text{void volume}}{\text{total volume}}$

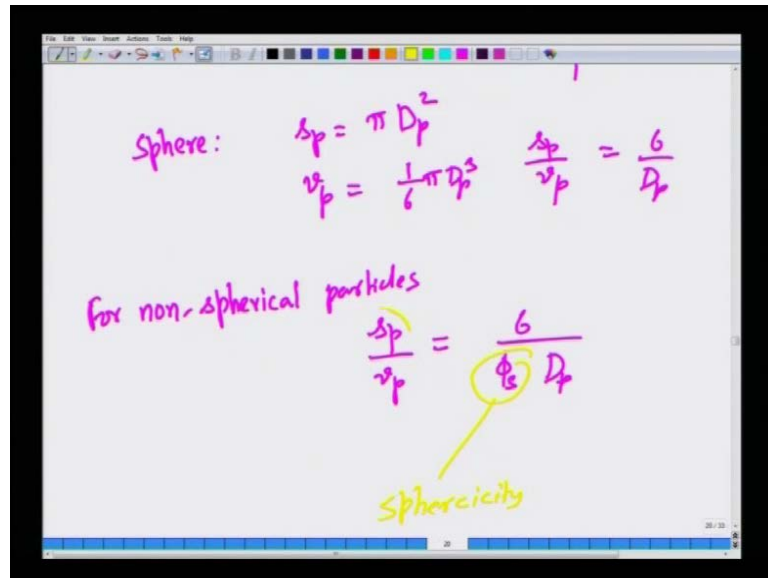
Volume fraction of particles: $(1 - \epsilon)$.

Surface to volume ratio of a single particle = $\frac{s_p}{v_p}$

Now, the bed porosity is defined by the symbol epsilon is essentially the void volume divided by total volume. So, the volume fraction of particles becomes 1 over 1 minus

epsilon. The volume fraction of particles becomes 1 minus epsilon because there only particles and void only 2. It is like of like two components system. If voids are occupying volume of epsilon then this can to occupy particles have to volume occupy the remaining volume. So, the fraction of that space is 1 minus epsilon.

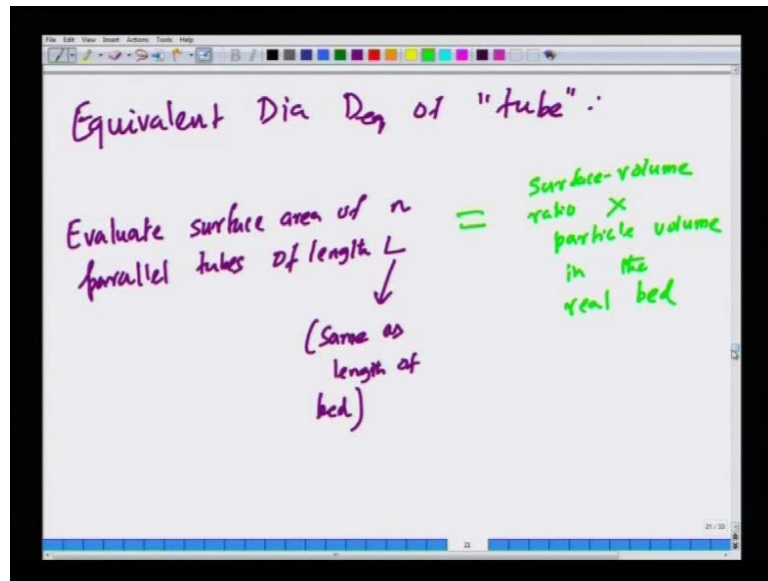
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So, the surface to volume ratio of a single particle is surface area of a particle divided by volume of a single particle. For a sphere, if the particle is a sphere s_p is nothing, but, πD_p^2 and v_p is nothing, but, $\frac{1}{6} \pi D_p^3$. So, s_p divided by v_p is $\frac{6}{D_p}$ for a sphere. For non-spherical particles because typically we will definitely have only non-spherical particles in practical applications s_p by v_p is written as $\frac{6}{\phi_s D_p}$ ϕ_s is called a Sphericity. So, it tells you about deviation from a spherical shape and ϕ_s for some spherical shape cause s_p by D_p is simply $\frac{6}{D_p}$ for non-spherical particles ϕ_s greater than 1. So, so this shows s_p by D_p is denoted by sphericity parameter.

So, you write the, so, you calculate how sphericity is calculated? You determine s_p by v_p for a non-spherical particle and then use this to determine sphericity. It tells you the deviation from the spherical shape.

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So, now we want to now find what is the equivalent diameter of the tube? (No audio from 46:38 to 46:46) through which we imagine fluid is flowing inside a packed bed in this bundle of tubes model. Now, first thing we will do is to evaluate the surface area of the tubes surface area of n parallel tubes of length L . So, here we are assuming that the length of the tube is same as the length of the bed same as the length of the bed which is a severe assumption. Because in reality the actual length of the flow of the particle sorry actual length of the fluid since the fluid is since, the tube is highly corrugated; it should certainly be more than l . But, we are now assuming because we do not know how what is the length of the actual flow path of a fluid inside this corrugated space.

So, we are going to assume the same as the length of the bed itself. So, this is an assumption. This is nothing, but, so, now first we evaluate this and then we equate this to this must be equal to the surface volume ratio times the particle volume in the actual bed in the real packed bed.

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① $n (\pi D_p L) = S_0 L (1 - \epsilon) \left(\frac{6}{\phi_s D_p} \right)$

② Void volume in the bed = Total volume of n channels

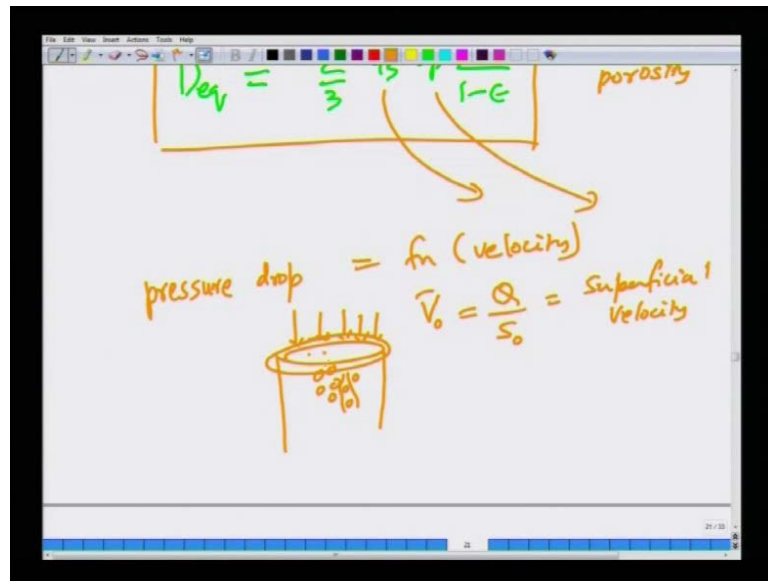
$S_0 L \epsilon = n \frac{1}{4} \pi D_p^2 L$

So, this is simple to calculate. So, number of tubes times pi D e Q time's l. This is surface area of a single tube through which fluid is moving and times n number of tubes is equal to. Therefore, the surface to volume ratio of a particle which is non-spherical is 6 by phi s D p times the particle volume in the bed particle volume in the bed is nothing, but, the cross section of the bed times the length of the bed times 1 minus epsilon.

This is the empty volume of the bed, but, 1 minus epsilon of this fraction 1 minus epsilon of this volume is occupied by the particles. So, we have to do that. So, this is the empty volume of the bed volume. This is particle volume fraction and this is the surface to volume ratio of a non-spherical particle if it is a sphere phi s is 1. This is one constraint that we have. The second constraint is that we have is that the void volume in the bed. The void volume in the bed is the same as the total volume of the n channels.

So, the void volume is simply s naught times L times epsilon. Because it is the empty bed volume and once you put in particles this is the void volume because epsilon is the void volume divided by the total volume. This is equal to n times pi times l. This is the cross section need of a single tube times its length times the number of such cha tube.

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So, by equating these two. So, D equivalent sorry b equivalent becomes therefore, $2 \text{ by } 3 \phi s$. So, if you look at this term you can get L will cancel out; L cancels out. So, you can get D equivalent in terms of other properties, other quantities. So, D equivalent becomes $2 \text{ by } 3 \text{ times } \phi s$ and also we have to eliminate for n by using this expression. So, essentially you can write instead of $s \text{ naught } L$ you substitute $n \text{ times } 1 \text{ by } 4 \pi D^2 L$ out here and then eliminate for D equivalent for D equivalent. So, we will get $2 \text{ by } 3 \phi s D p \text{ times } \epsilon \text{ by } 1 \text{ minus } \epsilon$.

So, this is the equivalent diameter of the various tubes based on the nature of the geometry of the particle the particle dimension and the porosity of the bed. So, we are trying to we have already gotten an expression for the equivalent diameter for these tubes through which we imagine fluid is flowing.

Now, the pressure drop depends on the average velocity in the channel; will be a function of the velocity in the channels in this various channels. The question is which velocity is to be used cause if you consider the bed. So, you have the cross section of the bed as $s \text{ naught}$ and volumetric flow rate that flows is q . So, this will be what is called the superficial velocity.

This is the superficial velocity of the fluid within the bed because once if you look at the actual velocity it is likely to be more than the superficial velocity. Because, the cross

section flow is not actually the entire thing there is only some part that is available for the flow. So, it has to be more. So, how do we do that? How to find the relation between the superficial velocity and the actual velocity? It is also called the empty tower velocity. If the tower was empty without any particles then what is the velocity that you would expect in the bed.

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The image shows a whiteboard with handwritten mathematical derivations. The equations are as follows:

$$Q = \bar{V}_0 S_0 = n \left(\frac{\pi D_{eq}^2}{4} \right) \bar{V}$$

$$\bar{V} = \frac{4 \bar{V}_0 S_0}{n \pi D_{eq}^2}$$

$$\bar{V} = \frac{\bar{V}_0}{\epsilon}$$

$$n \frac{\pi D_{eq}^2}{4} = S_0 \epsilon$$

$$\bar{V} > \bar{V}_0 \quad \epsilon < 1$$

So, how do you relate this to? So, Q naught is V naught times s naught. Now, that is simply equal to n times πD Equivalent Square by 4 times v . Because this is the volumetric flow rate in these n tubes each tube is of dimension of area these cross and there are n such tubes. This is the volumetric flow rate across the n tubes that we have modeled. So, V becomes therefore, V naught 4 V naught s naught divided by $n \pi D$ equivalent square.

Now, there is there is also this equation. You have this expression that relates n and s naught $L e$ epsilon. So, once you substitute that, but, n times πD equivalent squared by 4 is nothing, but, s naught times epsilon. So, V bar will become therefore, V naught by epsilon. So, we can see that V bar is greater than V naught because epsilon is always less than 1; epsilon is the porosity. So, this always is less than 1 and in the limit when epsilon is equal to 1 that is the whole bed is empty then we will get V equals V naught.

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The whiteboard shows three equations for the pressure drop per unit length, $\frac{\Delta p}{L}$:

$$\frac{\Delta p}{L} = 32 \frac{V_0}{\epsilon} \mu \frac{1}{D_{eq}^2}$$

$$\frac{\Delta p}{L} = 32 \frac{V_0}{\epsilon} \mu \frac{1}{\frac{4}{9} \phi_s^2 D_p^2 \epsilon^2} \frac{(1-\epsilon)^2}{\epsilon^2}$$

$$\frac{\Delta p}{L} = \frac{72 V_0 \mu}{\phi_s^2 D_p^2} \frac{(1-\epsilon)^2}{\epsilon^3}$$

So, now we know what is V in terms of the superficial velocity of the bed divided by porosity and we know what is D equivalent. So, we know what is the velocity and the diameter of the pipes. So, we can use suppose, the Reynolds number is low, Then we can use the laminar flow relation. So, Δp by L is $32 V \mu$ by D square it is $32 V \mu$ is nothing, but, V naught by ϵ times μ by 1 by instead of D we will put D equivalent square equivalent diameter and then substitute the expression for D equivalent.

So, you get Δp by L is nothing, but, $32 V$ naught by ϵ μ 1 over 1 minus ϵ squared by ϵ square 4 by $9 \phi_s$ square times D_p square just after substituting the expression for D equivalent that we just derived few minutes back. So, Δp by L becomes now after doing algebra the algebra here you will find that this is nothing, but, 72 times V naught μ divided by ϕ_s square D_p square times 1 minus ϵ whole square by ϵ cube.

Now, we will stop here and we will continue with this derivation in the next lecture where we will complete the derivation for pressure drop in packed bed both in the laminar regime as well as in the turbulent regime.