

Multiphase Micro fluidics
Prof. Raghvendra Gupta
Department of Chemical Engineering
Indian Institute of Technology, Guwahati

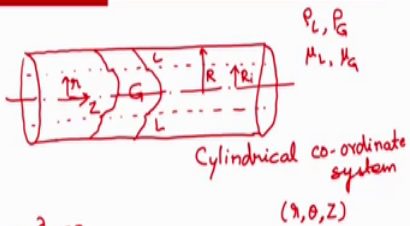
Lecture – 12
Smooth Interface Laminar-Laminar Annular Flow

Hello, in today's lecture, we will be looking at annular flow. In particular we will look at an ideal case of annular flow, where the interface is smooth and the film thickness or the size of the core, or the radius of the core is same throughout the channel. Ok?

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Ideal Annular Flow

➤ The flow configuration



Cylindrical co-ordinate system
(r, θ, z)

➤ Assumptions:

- Axisymmetric : $v_\theta = 0, \frac{\partial}{\partial \theta} = 0$
- Steady : $\frac{\partial}{\partial t} = 0$
- Laminar
- Fully-developed : $\frac{\partial v_z}{\partial z} = 0$

Incompressible flow (IT, Guwahati) ($P = \text{const.}$)

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So, let us draw a configuration of an ideal annular flow. We have a channel. Let us say this is a cylindrical channel and in this cylindrical channel, the interface is located somewhere here. So, this is phase 1. Let us consider gas liquid annular flow.

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Ideal Annular Flow


➤ Equation of continuity

$$\frac{\partial}{\partial t} + \frac{1}{r} \frac{\partial}{\partial r} (r u_r) + \frac{1}{r} \frac{\partial}{\partial \theta} (r u_\theta) + \frac{\partial}{\partial z} (r u_z) = 0$$

Steady state

axisymmetric

fully-developed flow



$$\frac{\partial}{\partial r} (r u_r) = 0$$

$$r u_r = \text{const.}$$

$$u_r = \frac{\text{const.}}{r}$$

at $r=0$ u_r finite $u_r = 0$

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 u_z - Non-Zero
 u_r, u_θ - zero
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So, gas is at the core and liquid is on the sides. Ok? And the radius of the channel is capital r and the radius of the interface, let us name it as I and we have properties of liquid ρ_l and ρ_g and μ_l and μ_g are the density and viscosities of gas and liquid. Now, this configuration, when we look at the flow regimes in gas liquid flow in micro channels, we often encounter annular flow, but this is really the annular flow that we see in gas liquid flow, in micro channels it is really a smooth interface annular flow.

We generally see the presence of waves on this annular flow, but to understand this, let us consider a case, where for which we can obtain an analytical solution and can get relationship, say for example, between pressure drop and film thickness or interfacial radius and the flow rate of the phases. So, this is the case that we are going to consider in this lecture..

There can be annular flow or what is termed as in general, core annular flow for in the reference of liquid flow. So, they are depending on the viscosities and densities of the two fluids plus hydrophobicity or hydrophilicity of the two liquids. One phase will be at the core and another phase will be these properties. We will determine which phase will be at the core and which phase will be near the wall. Ok?

So, we are looking at this. We will use the terms for ρ_l and ρ_g or μ_l and μ_g ; however, the same terms can be used with reference to liquid 1 and liquid 2. Ok? So, we will solve the mass conservation and momentum conservation equations and because this

is a cylindrical configuration, so we will prefer cylindrical coordinate system. So, where we have r , θ and z ? So, z is the direction, r the radial direction and when we say that the flow is Ax symmetric; that means, there is no flow in the angular direction and no gradients in the angular for azimuthally direction.

So; that means, v_θ is equal to 0 and $\frac{\partial}{\partial \theta}$ is 0. So, this is the cemetery configuration, ax symmetric configuration at steady state; that means, the $\frac{d}{dt}$ term is equal to 0 study, means as you will know that steady means the properties, the things, the configuration does not change with time. So, all the variables with respect to time are constant. So, that is why $\frac{d}{dt}$ term in the conservation equations will be 0, flow is laminar and we do not need any models for turbulence and we consider fully developed flow here.

So, when the flow is fully developed, we can say that $\frac{\partial v_z}{\partial z}$ is equal to 0; that means, the velocity profile, whatever the velocity profile be obtained at one axial location, a similar velocity profile will be obtained at another location. So, the corresponding difference Δv will be 0 or Δv_z will be 0. So, the flow is fully developed or that is the definition of a fully developed flow. Now, let us write the conservation equations. So, coming to the continuity equation, let us write a continuity equation in the cylindrical coordinates.

So, that is $\frac{\partial \rho}{\partial t} + \frac{1}{r} \frac{\partial}{\partial r} (r \rho v_r) + \frac{1}{r} \frac{\partial}{\partial \theta} (\rho v_\theta) + \frac{\partial}{\partial z} (\rho v_z)$ is equal to 0. Now, because this is at steady state, so, this term will be 0 and ax symmetric. So, this term will be 0. Now, what we have is $\frac{\partial v_z}{\partial z}$ is constant, because of fully developed flow. So, this is because the flow is ax symmetric and in this case, this says that fully developed flow. So, $\frac{\partial v_z}{\partial z}$ is 0 the flow. We consider in this case that the flow of both the phases is incompressible. So, we can add the assumption of incompressible flow. So, ρ is constant in these cases. So, that is why we can neglect this term.

Now, what we have from this, that $\frac{\partial}{\partial r} (r \rho v_r)$ is equal to 0 or v_r sorry, this is $\rho r v_r$, here, $\rho r v_r$ and; that means, $r v_r$ is equal to a constant and v_r is equal to constant over r , but v_r needs to be finite. So, that r is equal to 0, v_r is finite. So, the term v_r will be 0; that means, that is obvious from the flow configuration itself, because if the flow is happening in the radial direction; that means, the interface radius will

change, because some flow will happen along this direction and the radius interface radius will have to change to accommodate that flow.

So, because we do not have any such acceleration flow, so, the radial component of d will 0. So, the only component of velocity that we have non - 0 is, v_z non - 0 and v_r and v_θ , they are 0. Ok?.

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Ideal Annular Flow *gravity not considered*

➤ Momentum conservation equation:

➤ r- momentum:

$$\rho \left(\frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_r}{\partial \theta} + v_z \frac{\partial v_r}{\partial z} - \frac{v_\theta^2}{r} \right) = -\frac{\partial p}{\partial r} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial}{\partial r} (r v_r) \right) + \frac{1}{r^2} \frac{\partial^2 v_r}{\partial \theta^2} + \frac{\partial^2 v_r}{\partial z^2} - \frac{2}{r^2} \frac{\partial v_\theta}{\partial \theta} \right]$$

$\frac{\partial p}{\partial r} = 0 \Rightarrow p \neq p(r)$

➤ θ - momentum:

$$\rho \left(\frac{\partial v_\theta}{\partial t} + v_r \frac{\partial v_\theta}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_\theta}{\partial \theta} + v_z \frac{\partial v_\theta}{\partial z} + \frac{v_r v_\theta}{r} \right) = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial}{\partial r} (r v_\theta) \right) + \frac{1}{r^2} \frac{\partial^2 v_\theta}{\partial \theta^2} + \frac{\partial^2 v_\theta}{\partial z^2} + \frac{2}{r^2} \frac{\partial v_r}{\partial \theta} \right]$$

$p \neq p(\theta)$

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Now, let us come to the r momentum equation. So, for the cylindrical coordinate, the r momentum equation is $\rho \frac{dv_r}{dt} + v_r \frac{dv_r}{dr} + v_\theta \frac{dv_r}{r d\theta} + v_z \frac{dv_r}{dz} - \frac{v_\theta^2}{r} = -\frac{dp}{dr} + \mu \left[\frac{1}{r} \frac{d}{dr} \left(r \frac{dv_r}{dr} \right) + \frac{1}{r^2} \frac{d^2 v_r}{d\theta^2} + \frac{d^2 v_r}{dz^2} - \frac{2}{r^2} \frac{dv_\theta}{d\theta} \right]$. So, this term is 0, v_r is 0. So, this term is 0, v_θ is 0. So, this is 0, $\frac{dv_r}{dz}$ is 0. So, this is also 0 and v_θ is 0. So, this is 0 and because v_r is 0, so, this term is 0. This is again 0 because v_r and this is because v_θ is 0.

We do not consider gravity. So, we can say right gravity effect is not considered, if we take horizontal flow, then the flow will not be ax symmetric. If the gravity effect is considered, but remember that in micro channel, where the channel is sufficiently small, the effects of gravity are anyway negligible. So now, we can neglect or we can strike out the terms which are not relevant for this is at steady state. So, this term will be 0, v_r is 0. So, this term is 0, v_θ is 0. So, this is 0, $\frac{dv_r}{dz}$ is 0. So, this is also 0 and v_θ is 0. So, this is 0 and because v_r is 0, so, this term is 0. This is again 0 because v_r and this is because v_θ is 0.

So, what we have is $\frac{\partial p}{\partial r}$ is equal to 0; that means, p is not a function of r of pressure is same at a particular cross section in the radial direction. So, you take the pressure here and the pressure here, it is independent of the location, but remember for the two phases, what these equations we are writing general equation for phase 1 and phase two, but when we consider the boundary conditions, we will have a place of at the interface. So, you will discuss that, but for a phase inside one phase, pressure is not able to follow. Right?

Now, for theta momentum term, what we have is, $\rho \frac{\partial v_\theta}{\partial t} + v_r \frac{\partial v_\theta}{\partial r} + v_\theta \frac{\partial v_\theta}{\partial \theta} + v_z \frac{\partial v_\theta}{\partial z} = -\frac{1}{r} \frac{\partial p}{\partial \theta} + \mu \left(\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_\theta}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_\theta}{\partial \theta^2} + \frac{\partial^2 v_\theta}{\partial z^2} \right)$

So, in this again, we neglect the terms. This term is 0, v_θ is 0. So, this term is 0, v_θ is 0, v_θ is 0. So, all these terms are 0 and again, we will have this is 0, because v_r is 0. So, we will have that pressure is not a function of theta so; that means, pressure is, if it is, then it is a function of that only. So, let us look at the z momentum equation. Right?

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Ideal Annular Flow

➤ Momentum conservation equation:

➤ z- momentum:

$$\rho \left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + v_\theta \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right]$$

$$\mu \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) = -\frac{\partial p}{\partial z}$$

$$-\frac{\partial p}{\partial z} = B = -\frac{dp}{dz}$$

$$v_z = 0$$

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When we write the z momentum equation $\rho \frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + v_\theta \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z}$ is equal to minus del

$\mu \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial}{\partial \theta} \left(r^2 \frac{\partial v_z}{\partial \theta} \right) + \frac{\partial^2 v_z}{\partial z^2}$.

I have been very careful while writing these equations; I would request you to recheck these equations from a standard book. The book from which I have taken the equations is transport phenomena by Bird Stewart and light foot from the appendix B, you can see all those equations are in the chapters where the Navier Stokes equation have been derived. So, again $\frac{\partial v_z}{\partial t}$, because it is at a steady state and this term will be 0, because v_r is 0, this term will be 0 because v_θ is equal to 0 and this term is 0, because $\frac{\partial v_z}{\partial z}$ is 0.

Now, this is 0 because gradient in theta direction are 0 and this is 0 because flow is fully developed. So, $\frac{\partial v_z}{\partial z}$. It is minus $\frac{\partial v}{\partial z}$ is equal to $\mu \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right)$. So, we will, for clarity or for convenience, we will assume that minus $\frac{\partial p}{\partial z}$ is equal to b . Note, that this can be written. Now as minus $\frac{dp}{dz}$, because it does not depend, the pressure depends only on z . So, it is equal to the partial derivative is equal to the total derivative and the v_z , we will write as v only, now onwards because for our convenience. Ok?

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Ideal Annular Flow

➤ Boundary conditions:

at $r = R$ (wall) $v_L = 0$

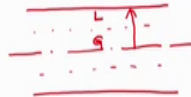
at $r = 0$ (axis) $v_a = \text{finite}$

at $r = R_i$ Interface

velocity continuity $v_L|_{R_i} = v_a|_{R_i}$

Shear stress continuity $\mu_L \frac{\partial v_L}{\partial z}|_{R_i} = \mu_a \frac{\partial v_a}{\partial z}|_{R_i}$

Normal stress $p_L - p_a = \frac{\sigma}{R_i} \quad R_i \neq R_i(z)$



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$\frac{dp_L}{dz} = \frac{dp_a}{dz} \Rightarrow p_L = p_a$

So, we can write this equation later on for both the phases. Let us look at the boundary condition. So, the configuration we look at again. So, this is our interface in the centre. So, at r is equal to capital R , which is general wall. We will have v of liquid, because this

is liquid phase and this is gas phase. So, v_L is equal to 0 at r is equal to 0, which is at the axes. We will have v_{gas} to be finite. So, if you use this condition or then, we will also have an r is equal to R_i which is at the interface..

There are two boundary conditions, the velocity continuity. So, v_{liquid} at R_i is equal to v_{gas} at R_i and we will also have shear stress continuity, $\mu_L \frac{\partial v_L}{\partial z}$ at R_i is equal to $\mu_G \frac{\partial v_G}{\partial z}$ at R_i . We will also have a boundary condition for normal stress and that will be a jump in pressure. So, we will have p_L minus p_G is equal to σ , κ and κ is 1 over R_i in this case. So, we can have σ over R_i , but R_i is constant throughout. So, we can say that $\frac{dp_L}{dz}$ minus $\frac{dp_G}{dz}$ is equal to 0, so; that means, because R_i is not a function of z .

So, that we can say the gradients in the both the phases are same. So, we can have B_L is equal to B_G from here.

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Ideal Annular Flow

➤ Integrate once: $\frac{\mu}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v}{\partial r} \right) = B \Rightarrow \frac{\partial}{\partial r} \left(r \frac{\partial v}{\partial r} \right) = \frac{Br}{\mu} \Rightarrow r \frac{\partial v}{\partial r} = \frac{Br^2}{2\mu} + C_1$
 $\frac{\partial v}{\partial r} = \frac{Br}{2\mu} + \frac{C_1}{r}$

➤ Use Shear stress continuity BC: $\mu_G \frac{\partial v_G}{\partial r} \Big|_{R_i} = \mu_L \frac{\partial v_L}{\partial r} \Big|_{R_i}$

$$\frac{BR_i}{2} + \frac{\mu_G C_{1G}}{R_i} = \frac{BR_i}{2} + \frac{\mu_L C_{1L}}{R_i}$$

$$\mu_G C_{1G} = \mu_L C_{1L}$$

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Now, the equation, if we look at this equation again, μ over m r $\frac{\partial}{\partial r}$ are μ over r , $\frac{\partial}{\partial r}$, $\frac{\partial v}{\partial r}$ is equal to minus $\frac{dp}{dz}$ or $\frac{dp}{dz}$, which we have named as B now. So, we will have, when we integrate this equation, we will get $\frac{\partial v}{\partial r}$, $r \frac{\partial v}{\partial r}$ over $\frac{\partial v}{\partial r}$ is equal to $\frac{BR}{\mu}$ which will give us Δ , sorry, this is not v , $\frac{\partial v}{\partial r}$ is equal to this, will be r and this will be equal to $\frac{BR^2}{4\mu}$.

So, we can write $\frac{dv}{dr}$ is $\frac{B}{4\mu} + \frac{C_1}{r}$. A constant of integration, let us say, this at C_1 . So, $\frac{dv}{dr}$ is equal to $\frac{B}{4\mu} + \frac{C_1}{r}$ and now, we can write the same equation for gas and liquid phases, using the shear stress continuity boundary condition at the interface, we will have $\mu_g \frac{dv_g}{dr}$ at R_i is equal to $\mu_l \frac{dv_l}{dr}$ at R_i . So, if we substitute here, we will have four constants, basically when we integrate this fellow.

So, we will have this term, written for two, sorry, term T is constants as C_1 and C_2 for two different things. So, when we multiply by μ , we can have $\frac{B}{4}$ and μ will cancel out plus μC_1 divided by R_i , similarly we can write $\frac{B}{4}$ plus μC_1 by R_i . So, $\frac{B}{4}$ by $\frac{B}{4}$ will cancel out and what we will have as the boundary condition, that $\mu_g C_1$ is equal to $\mu_l C_1$. Or from this boundary condition, we will get $\mu_g C_1$ is equal to $\mu_l C_1$. Ok?

So, we have a relationship between the two constants for two phases, C_1 and C_2 . Ok? Now, we can integrate this equation.

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Ideal Annular Flow

➤ Gas (core) phase velocity profile: $\frac{\partial v}{\partial r} = \frac{B}{2\mu} + \frac{C_1}{r}$

$$v = \frac{B r^2}{4\mu} + C_1 \ln r + C_2$$

at $r=0$, v_g = finite
 $C_1 = 0 \Rightarrow C_2 = 0$

$$v_g = \frac{B r^2}{4\mu} + C_2$$

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So, we will have $\frac{dv}{dr}$ is equal to $\frac{B}{4\mu} + \frac{C_1}{r}$, when we integrate, we get v is equal to, sorry, this is, this will be $\frac{B}{4\mu} r^2 + C_1 \ln r + C_2$. So, when we integrate this further, we will have $\frac{B}{4\mu} r^2 + C_1 \ln r + C_2$ and now, we can write the same equation for gas phase and liquid phase.

So, if we use the boundary condition for gas phase at the axis at r is equal to 0 v_G is finite or defined. So, then because $\ln 0$ is not defined for $C_1 G$ has to be 0 so; that means, we have B_G is equal to $B r^2$ by $4 \mu_L$ plus $C_2 G$. Ok fine! So, now, we need to find out the constant $C_2 G$. Ok?

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Ideal Annular Flow

➤ Liquid phase velocity profile: $v_L = \frac{B r^2}{4 \mu_L} + C_2 L$

at $r=R$, $v_L=0$ $0 = \frac{B R^2}{4 \mu_L} + C_2 L$

$C_2 L = -\frac{B R^2}{4 \mu_L}$

$v_L = \frac{B}{4 \mu_L} (r^2 - R^2)$

velocity continuity at interface $v_L|_{R_i} = v_G|_{R_i}$

$v_G = \frac{B}{4 \mu_G} (r^2 - R_i^2) + \frac{B}{4 \mu_L} (R_i^2 - R^2)$

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For liquid phase again, we can write the same velocity profile. So, if $C_1 G$ is equal to 0, this will make $C_1 L$ is equal to 0 too. So, we will have v_L is equal to $B r^2$ over $4 \mu_L$ plus $C_2 L$ and at r is equal to capital R , that is at the wall, we have a v_L is equal to 0. So, the substitute 0 is equal to B capital R square by $4 \mu_L$ plus $C_2 L$.

So, by subtracting these two equations, we can get the velocity profile or we can get rid of $C_2 L$ v_L is equal to B over $4 \mu_L$ small r square minus capital R square. Now, to obtain v_G , we will apply velocity continuity at the interface; that means, v_L and R_i is equal to v_G at R_i and we substitute that, we will get the liquid velocity, we have already got.

So, the gas velocity, we will get after substitution and manipulating the algebra, we will get v_G is equal to B over $4 \mu_G$ small r square minus capital R_i square plus B by $4 \mu_L$ into R_i square minus r square. So, I request you to go through this step and do the required algebra. So, we have got the velocity profile in the gas phase and velocity profile in the liquid phase for a smooth interface annular flow.

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Ideal Annular Flow

$$U_L = \frac{Q_L}{\pi R^2} = \frac{1}{\pi R^2} \int_{R_i}^R v_L 2\pi r dr = -\frac{BR^2}{8\mu_L} \left(1 - \frac{R_i^2}{R^2}\right) \quad \text{--- (A)}$$

$$U_G = \frac{Q_G}{\pi R^2} = \frac{1}{\pi R^2} \int_0^{R_i} v_G 2\pi r dr = -\frac{BR^2}{8\mu_L} \left[\left(\frac{\mu_L}{\mu_G} - 2\right) \left(\frac{R_i}{R}\right)^4 + 2 \frac{R_i^2}{R^2} \right] \quad \text{--- (B)}$$

Q_L, Q_G (or U_L, U_G) are generally known
superficial velocities

Solve (A) and (B) to obtain B, R_i

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Now, once we have got that, then, we can get the by integration, we can get the Q_L and Q_G . So, Q_L will be integrating $v_L 2\pi r dr$ from R_i to R . Q_G will be equal to 0 to R_i , sorry, this is v_L and v_G , will be equal to $v_G 2\pi r dr$ are integrated from 0 to R_i . Now, from this we can also find out the superficial velocities. So, when we do the substitution and the superficial velocity of the liquid phase will be Q_L over πR^2 .

So, 1 over πR^2 and U_G is equal to Q_G over πR^2 . So, 1 over πR^2 and by substituting the values and do the necessary integration, we will get U_G is equal to minus $B R^2$ over $8\mu_L$ into μ_L over μ_G minus 2 into R_i by R raised to the power 4 plus $2 R_i^2$ over R^2 .

Similarly, U_L can be written as minus $B R^2$ over $8\mu_L$ into 1 minus R_i^2 over R^2 . So, now, in general for any annular flow or for any experimental conditions, for in any experiment, what we will know is the gas and liquid flow rates, if we want to obtain the velocity profile; the problem for us for the velocity profiles are that, they are functions of B , which is pressure gradient and R_i which is interfacial radius. Now, we do not know both of these. So generally, what we have Q_L and Q_G or U_L U_G , these are the superficial velocities.

So, you can just note that U_L U_G are superficial velocity are generally known for a flow. So, by knowing this we have two equations and we can solve an equation, let us say, A and equation B to obtain B , which is pressure gradient and R_i .

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Ideal Annular Flow

$$U_L = -\frac{BR^2}{8\mu_L}(1 - \xi_i^2)^2$$

$$U_G = -\frac{BR^2}{8\mu_L} \left[\left(\frac{\mu_L}{\mu_G} - 2 \right) \xi_i^4 + 2\xi_i^2 \right]$$

Solve for pressure gradient and interface radius

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So, we can look at this in the ideal annular flow, these are the U_L and U_G , the two phase flow rates and the superficial velocities, which we have solved, just solved for and now, when we solve for pressure gradient and interfacial radius.

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Ideal Annular Flow

$$\xi_i^4 \left[\frac{1}{4\mu_L U_L} - \frac{1}{4\mu_G U_G} + \frac{1}{2\mu_L U_G} \right] - \xi_i^2 \left[\frac{1}{2\mu_L U_L} + \frac{1}{2\mu_L U_G} \right] + \frac{1}{4\mu_L U_L} = 0$$

$\xi_i = \frac{R_i}{R}$
 $\Rightarrow \xi_i^2 = ? \Rightarrow \xi_i = ?$ (ok ok)

$$B = \frac{8\mu_L U_L}{R^2(1 - \xi_i^2)^2}$$

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So, we can rearrange these two equations to in such a manner that it becomes a quadratic equation, in $\xi_i R^2$, ξ_i^2 where ξ_i is equal to R_i by capital R. Ok?

Sorry, this is not sigma, this is phi and so, ϕ_i is equal to R_i by R . So, once we solve that, we can obtain what is ϕ_i square and from that, we can obtain in the roots of it. Generally, with my experience, I have observed that one root is always greater than 1. So, we know that is unphysical. So, we will have to choose ϕ_i with the condition that ϕ_i is between 0 and 1. So, we can get what is ϕ_i and when we substitute back in the pressure gradient, we will get the pressure gradient.

So, basically by solving what we have done today, is solve the ideal annular flow equation starting with a basic Navier Stokes equation, we have neglected the terms the relevant terms in the R theta and z system of the equations and in the continuity equation and finally, after integrating we got the velocity profile, which had 4 constants; two constants for the liquid phase and two constants for the gas phase..

And then, we had four boundary conditions, one is at the axis, that the velocity gradient should be finite and L_N , it should be defined at the axes. Then, we had two boundary conditions at the interface, the interface continuity and shear stress continuity and one boundary condition at the wall that the liquid velocity at the wall is 0. So, by solving that we could obtain the velocity profiles in terms of the pressure gradient and the interfacial radius. Once, we have done that, then we integrate it and obtained the flow rates of the two phases or from the flow rates, we can also obtain the superficial velocities and from solve, by solving these two equations, we could get the pressure gradient and R_i or interfacial radius..

So that we can get the velocity profile for any given gas and liquid phase flow rates. Now, this ideal annular flow, there is lot of literature on the stability analysis non - linear and linear stability analysis of the annular flow, this is subject to a different interfacial instabilities.

So, if you remember the capillary instability, which we discussed in the previous class. So, in the capillary instability A_z disintegrates into smaller droplets. So, following that, the annular flow is at lower velocities, it becomes Slug Flow or Taylor Bubble Flow and there are generally, what we see in practice that, there are always waves present at the interface and that is because of probably at the Kelvin Helmholtz instability, which is that, when there are two parallel liquids flowing. There are waves generated or there are small perturbations, they grow up to become wave to make the interface maybe ok,

Thank you.