

Satellite Attitude Dynamics and Control
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Lecture - 50
Reaction Wheel / Gyrostat (Contd.)

Welcome to the 15th lecture. So, we have been discussing about the reaction wheels and in that context we have derived the linear momentum equation. Now we will go for the angular momentum equation and work out the details.

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Lecture - 50
Reaction wheel / Gyrostat

$\vec{r}_p \equiv \vec{r}_w$ = radius vector of the c.m. of the wheel measured from origin O of the body frame
 e_1, e_2, e_3 | \vec{J}_b = moment of inertia dyadic about point O

$$\vec{h}_b = \int d\vec{h} = \int \vec{r} \times d\vec{m}\vec{v} = \int \vec{r} \times (\vec{v}_0 + \vec{\omega} \times \vec{r}) dm$$

$$= \int \vec{r} dm \times \vec{v}_0 + \int \vec{r} \times (\vec{\omega} \times \vec{r}) dm$$

$$= m_b \vec{r}_w \times \vec{v}_0 + \vec{J}_b \cdot \vec{\omega}$$

So, we have, this is the universal system the body frame fixed here and wheel is located here this we have tagged as body and this we have tagged as wheel and this we have written as r_w , this we have written as r_p here.

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$m_b \vec{r}_B = \vec{a}_B$ (6)
 $m_w \vec{r}_P = \vec{a}_P$
 Total linear momentum
 $\vec{P} = \vec{P}_b + \vec{P}_w$
 $= (m_b \vec{v}_o + \vec{\omega} \times m_b \vec{r}_B) + (m_w \vec{v}_o + \vec{\omega} \times m_w \vec{r}_P)$
 $= (m_b + m_w) \vec{v}_o + \vec{\omega} \times (m_b \vec{r}_B + m_w \vec{r}_P)$
 $= m \vec{v}_o + \vec{\omega} \times m \vec{r}_{cm}$
 $\vec{r}_{cm} = \frac{m_b \vec{r}_B + m_w \vec{r}_P}{m_b + m_w}$ = C.M. of the whole system.
 $\vec{P} = m \vec{v}_o + \vec{\omega} \times m \vec{r}_{cm}$ (7)

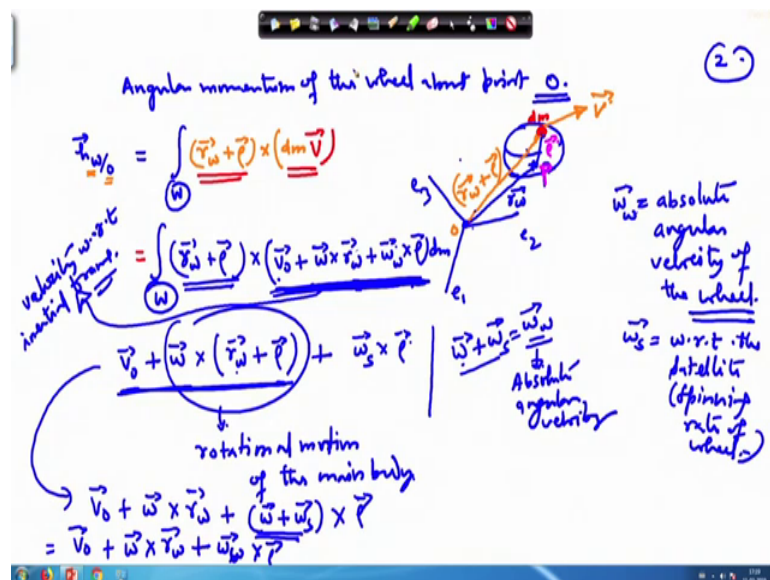
So r_p , r_p or r_w this point we have written as P. So, this is your basically r_p . you can show it by some other this is r_p and then your wheel is thick. So, this is not just a plate thin plate so we need to draw it appropriately. So, your wheel is thick. So, from this point to any other point so here $d m$ mass is there. And this vector we have written as the rho vector and r_p also we can write as r_w . So, this is the radius vector of the centre of mass of the wheel measured from origin o of the body frame e_1, e_2, e_3 . So, we are going to now calculate the angular momentum for these 2 entities; the main body and the wheel separately.

So, let us assume that this is a mass $d m$ on the main body and r is the radius vector to this point. So, what will be the angular momentum of this? If V is the velocity of this point and we can name this point, this point we are named as P. So, let us name this as q. So, this is an arbitrary point. So, r cross angular momentum is defined as r cross $d m$ times $d m V$. And if we integrate it; so, we get h and we integrate it over main body and what this quantity V is? V is, obviously V_0 plus ω cross r and this type of integration we have done numerous times. So, I hope that you are now by this time conversant with this, this integration over the body cross V_0 and plus r cross ω cross $r d m$ this is integration over the body. So, this gives us r times $d m$ is the location of the centre of mass of the so, this is m body plus r centre of mass of the body, cross V_0 and plus this quantity as we know this is the angular momentum, this part can be

written as in terms of moment of inertia ok. So, this is the moment of inertia dyadic, this is for the body dot omega.

So, I am not expanding and then working and doing so on. So, because we have done so many times that you should by now remember it. So, here J_b is the moment of inertia about the point o ok. So, we should write here also the moment of inertia dyadic about point O. And here we will write h_b . So, we should write here as a conclusion; h_b equal to $m \mathbf{r}_b \times \mathbf{V}_0$ plus $J_{\text{double bar } b} \cdot \omega$. This is our main result.

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Next we look into the angular momentum of the wheel of the wheel about point O. So, this we will write as angular momentum of the wheel with respect to point o or about point o. This tag is necessary in order to differentiate as we will see later on. So, that becomes we have to integrate it over the wheel ok. And any point in this wheel which is shown here as I am showing it by this one. So, that we have to draw radius vector from this point to this point.

So, I will do it on the next page say if this is your body frame e_1, e_2 and e_3 and somewhere your wheel is there. So, the centre of mass of the wheel to this location you have the r_{wheel} the radius vector as we are writing, r_{wheel} or r_p . This is the point P and then any elementary mass we are taking out here in this place and to this the radius vector is rho. So, we write it here on this side, rho is the corresponding radius vector this is point P ok. And this is the distance is r_{wheel} this is the centre of mass of the wheel

location plus rho this is your vector here. So, this mass is located by the vector r_w plus rho with respect to point o. So, angular momentum of the wheel about the point o this is given by r_w plus rho cross $d m$ times velocity of this point. So, let us say the velocity vector is directed here.

So, this is velocity of point let us term this as v_r say this is, finding some notation. Ok I will not name this. We will fall short of the notation. So, this is your point mass here which is $d m$. So, $d m v$, this is your linear momentum of this mass and r_w this is the radius vector to this point. So, this gives you the angular momentum. Now we need to expand this. So, r_w plus rho cross V equal to V_0 plus omega cross r_w . So, this is the velocity of this point V_0 and what will be the velocity of this point, because this body is rigid so, at what angular speed this body is rotating, so add to that and then at what rate this wheel is spinning? So, wheel angular speed is w , w , w , we are with we are writing here for this rho $d m$. So, this becomes your ok.

So, w is your absolute angular velocity of the wheel. This we can also understand from the point of view that V_0 plus omega cross r_w plus rho. So, this is this part arises because of the rotational motion of the, of the main body and then your satellite itself is spinning, this wheel that itself is a spinning ok. So, if we have the spinning rate of wheel we write as w_s ok; which is this is w_s is with respect to the satellite. This spinning rate of wheel; so, w_s cross rho. Now if we break it up this particular part. So, this becomes V_0 plus omega cross r_w r_w is where r_w is the distance to this point which we have already written, which appears here in this place also in this place and plus omega plus omega s cross rho we can write it like this.

So, this becomes V_0 plus omega cross r_w plus this quantity is absolute angular velocity. Omega is the angular velocity of the satellite to this if we add the spin of the wheel with respect to the satellite. So, that becomes the angular velocity of the wheel with respect to the initial frame ok. So, this becomes the absolute angular velocity. So, therefore, this becomes w cross rho. This is what we have written. So, this is the distance to this point and then the corresponding velocity in the inertial frame; only thing that we are taking about this point. So, this is the velocity in the with respect to the universal frame.

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$$\vec{h}_{w/o} = \int (\vec{r}_w + \vec{\rho}) \times (\vec{v}_0 + \vec{\omega} \times \vec{r}_w + \vec{\omega} \times \vec{\rho}) dm$$

$$= \vec{r}_w \times \vec{v}_0 \int dm + \vec{r}_w \times (\vec{\omega} \times \vec{r}_w) \int dm + \vec{r}_w \times (\vec{\omega} \times \vec{\rho}) \int dm$$

$$= \vec{r}_w \times \vec{v}_0 \int dm + \int \vec{\rho} dm \times (\vec{\omega} \times \vec{r}_w) + \int \vec{\rho} \times (\vec{\omega} \times \vec{\rho}) dm$$

Val. of the c.m. of the wheel (\vec{v}_0)

$\vec{h}_{w/o} = m \vec{r}_w \times \vec{v}_0 + \vec{J}_w \cdot \vec{\omega}$

$\vec{h}_{w/o} = \vec{r}_w \times \vec{p}_w + \vec{J}_w \cdot \vec{\omega}$

when $\vec{J}_w = \int \vec{\rho} \times (\vec{\omega} \times \vec{\rho}) = \text{Inertia dyadic of the wheel about point P}$

Velocity with respect to the; so, omega wheel with about point o; h about angular momentum of the wheel about point o this we have written as I will refresh on this page, $r w$ plus ρ cross V_0 plus ω cross $r w$ plus ρw dm . Now, for working out this we need to expand it and write the whole thing. So, one by one we can expand and work it out. So, the first one we get as $r w$ cross V_0 , ok this term with respect to this term we are expanding. You can say that this term is a not dependent on mass, this term is not dependent on wheel mass and all the terms here they are not dependent on wheel mass and therefore, they are taken outside the integration sign.

Now with respect to this ρ we are going to integrate. This ρ we are going to use it. So, with respect this ρ will be multiplying all the terms. So, this becomes ρdm cross V_0 , V_0 is not dependent on the integration over the wheel. So, this comes outside here ok. This is ρ here, notation we have used for this is no ρ double ρw , but ρ we have used. So, this part I have corrected here. So, ρ cross; so, total of 6 terms we are having. Now we can see your some of the terms that drop out because, we have chosen P as the centre of mass of the wheel. So, therefore, this term is 0, and from P we are measuring this ρ vector and therefore, this will drop out so this quantity is 0 here and then this quantity similarly this becomes 0; here in this place this quantity becomes equal to 0. So, these terms will simply drop out. And what we get here; $h w$ o this equal to $r w$ cross.

So, we can write here $m\mathbf{w} \times \mathbf{V}_0$ plus here again this integration is $m\mathbf{w}$ this term drops out, out this term drops out, this term also drops out. So, 3 terms to where in a sequence they are getting dropped out and the last term this term is nothing but where this is the quantity ρ cross inertia dyadic of the wheel about point P. So, had all this equations the, one equation we have got is into this is equation number 1 and this is equation number 2. We need to add it to get the total angular momentum.

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$\vec{E} \rightarrow$ unit dyadic
 $\vec{h} =$ total angular momentum
 $= \vec{h}_{b/o} + \vec{h}_{w/o}$
 $= m_b \vec{r}_b \times \vec{V}_0 + \vec{J}_b \cdot \vec{\omega} + m_w \vec{r}_w \times \vec{V}_0 + m_w \vec{r}_w \times (\vec{\omega} \times \vec{r}_w) + \vec{J}_w \cdot \vec{\omega}$
 $= (m_b \vec{r}_b + m_w \vec{r}_w) \times \vec{V}_0 + \vec{J}_b \cdot \vec{\omega} + m_w \vec{r}_w \times (\vec{\omega} \times \vec{r}_w) + \vec{J}_w \cdot \vec{\omega}$
 $= m \vec{r}_{cm} \times \vec{V}_0 + \vec{J}_b \cdot \vec{\omega} + \vec{J}_w \cdot \vec{\omega} + m_w \vec{r}_w \times (\vec{\omega} \times \vec{r}_w)$
 $= m \vec{r}_{cm} \times \vec{V}_0 + [\vec{J}_b \cdot \vec{\omega} + m_w (\vec{r}_w \times (\vec{\omega} \times \vec{r}_w))] + \vec{J}_w \cdot \vec{\omega}$
 $= C \times \vec{V}_0 + [\vec{J}_b \cdot \vec{\omega} + m_w (\vec{r}_w^2 \vec{E} - \vec{r}_w \vec{r}_w) \cdot \vec{\omega}] + \vec{J}_w \cdot \vec{\omega}$

So, h there is a nothing, but the total angular momentum and this can be written as h main body plus and this also is about the point o. So, maybe we can go back and put this point o. So, this is also about the point o, this is about the point o, and also the wheel about the point o. So, this is the total angular momentum we just need to add that and then put all these quantities here $m \mathbf{r}_b \times \mathbf{V}_0$. $m \mathbf{r}_b \times \mathbf{V}_0$ plus $\mathbf{J}_b \cdot \boldsymbol{\omega}$. Here this quantity we are going to write this currently at \mathbf{J} ok. So, here also we will write as \mathbf{J} . The reason will become clear once we advance. This is the inertia the dyadic of the wheel. So, this is for the body times $\boldsymbol{\omega}$. This is what they $\mathbf{J}_b \cdot \boldsymbol{\omega}$ and plus the other part $h_{w/o}$. So, we have to copy all this things $m \mathbf{r}_w \times \mathbf{V}_0$, $m \mathbf{r}_w \times \mathbf{V}_0$ plus $m \mathbf{r}_w \times m \mathbf{r}_w \times \boldsymbol{\omega} \times \mathbf{r}_w$ and plus the third term here $\mathbf{J}_w \cdot \boldsymbol{\omega}$ time. So, this 2 terms can be combined together, see if this derivation is bit longer if we have chosen the point o to be the centre of mass itself it would have been much shorter ok. But then it deprives us of the what happens if the point o does not coincide with the centre of mass and quite often this is the situation it is not always that your

organize at the centre of mass of the whole system. There may be difficulty with the fabrication of the system itself that you are not able to conceive your center of mass at that point from there you are doing the measurements.

So, in that case this equation what we are deriving here they will be applicable. This is the total mass m and then the centre of mass of the whole system. So, that we have written as we have used at notations from there this is the r_{cm} and ok. So,, this becomes m times r_{cm} . This is the, this gives you this quantity which is here $\mathbf{V}_0 + \boldsymbol{\omega} \times \mathbf{r}_{cm}$. This equation what we have written here, this we can little bit refine it and this will be required later on so, we can also rewrite this as $\mathbf{h}_w = m \mathbf{w}$, $m \mathbf{w}$ in these 2 places they are available. So, we can take this as common. So, $m \mathbf{w}$, $r \mathbf{w}$ is also a common. So, $r \mathbf{w}$ goes outside and $\mathbf{V}_0 + \boldsymbol{\omega} \times r \mathbf{w} + \mathbf{J} \text{ double bar } \mathbf{w}$ times $\boldsymbol{\omega}$. which we can re write as $m \mathbf{w}$, $r \mathbf{w}$, \mathbf{w} $r \mathbf{w}$ we keep it here $\mathbf{V}_0 + \boldsymbol{\omega} \times r \mathbf{w}$ this is the velocity of the centre of mass of the wheel. So, this is the velocity of the centre of mass of the wheel and therefore, we can write this quantity here as, velocity of the centre of mass of the wheel and together with we multiple π by $m \mathbf{w}$. So, the quantity $m \mathbf{w}$ times \mathbf{v}_w we will write as \mathbf{p}_w .

So, here we write this as \mathbf{p}_w , this $m \mathbf{w}$ times this quantity here this is your \mathbf{V}_w . So, $m \mathbf{w}$ times \mathbf{V}_w is \mathbf{p}_w . So, we are writing here and plus $\mathbf{J} \text{ double bar } \boldsymbol{\omega} \times \mathbf{w}$ ok. So, this expression will be especially helpful later on. So, here all here I am writing it finally, $r \mathbf{w} \times \mathbf{p}_w + \boldsymbol{\omega} \times \mathbf{w}$. We combined this term and this term and this particular term here m times r_{cm} we can write us for gravity we can write as C . So, this quantity you can see that this is the moment of inertia of the moment of inertia dyadic of the $\mathbf{J} \text{ double bar}$, $\mathbf{J} \text{ double bar}$. So, this is indicating moment of inertia dyadic of the main body, moment of inertia dyadica and to this your then adding this particular term.

So, this has the same configuration as say the r times $\boldsymbol{\omega} \times r \mathbf{dm}$. Once we write it like this so, this has got the same configuration. So, this also indicates moment of inertia times $\boldsymbol{\omega}$ but here it is for the here we have missed out that \mathbf{w} term this is \mathbf{w} subscript is there. So, you have the main body here. So, main body moment of inertia about this point and then this point mass also has moment of inertia about this point o . This point mass is what here the wheel whose mass is m_w . So, m_w has if it is concentrated here and then you are taking this. And so, we can write this as we can we will write up on the; we will have to expand it little bit and then write it. So, let us, so, here this becomes C

cross \mathbf{V}_0 where, \mathbf{C} we are defining as m times $\mathbf{r} \times \mathbf{c} \times m$ plus \mathbf{J} double bar dot this is body, ω plus m \mathbf{w} and this we can write as $\mathbf{r} \times \mathbf{w}$ a square \mathbf{E} double bar. Where \mathbf{E} double bar is the unit dyadic as per our earlier notation unit dyadic minus $\mathbf{r} \times \mathbf{w}$ dot ω plus.

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$\vec{\omega}_0 = \vec{\omega} + \vec{\omega}_s$
 Spin ang. vel. w.r.t. the main body.

$$\vec{h} = \vec{c} \times \vec{V}_0 + \vec{J}_b \cdot \vec{\omega} + m_w (\mathbf{r}_w^2 \mathbf{E} - \mathbf{r}_w \mathbf{r}_w) \cdot \vec{\omega} + \vec{J}_w \cdot \vec{\omega}_s$$

$$= \vec{c} \times \vec{V}_0 + [\vec{J}_b + m_w (\mathbf{r}_w^2 \mathbf{E} - \mathbf{r}_w \mathbf{r}_w)] \cdot \vec{\omega} + \vec{J}_w \cdot (\vec{\omega} + \vec{\omega}_s)$$

$$= \vec{c} \times \vec{V}_0 + [\vec{J}_b + \vec{J}_w + m_w (\mathbf{r}_w^2 \mathbf{E} - \mathbf{r}_w \mathbf{r}_w)] \cdot \vec{\omega} + \vec{J}_w \cdot \vec{\omega}_s$$

moment of inertia of the main body about point O
 moment of inertia of the wheel about point P (C.M. of the wheel)
 moment of inertia of the mass of the wheel concentrated at point P about O
 whole system of main body + wheel
 moment of inertia of the whole system about point O

So, this we have copied from the previous page. Now, we can write this as $\mathbf{C} \times \mathbf{V}_0$ plus \mathbf{J} double bar, and ω this is absolute angular velocity of the wheel. So, this will be equal to the angular velocity of the main body and with respect to this then the wheel is spinning. So, your ω this equal to ω plus ω_s ; so, this is spin rate with respect to the, ω_s is the spin rate where this is a spin velocity or the spin angular velocity, a spin angular velocity with respect to the main body. This term we can pick up from this place. This dot is here, and we can write \mathbf{J}_w and plus m_w times \mathbf{r}_w a square this is the unit dyadic minus \mathbf{r}_w ; this part is moment of inertia of the main body about point o and moment of inertia of the wheel about point P which is the centre of mass of the wheel, the wheel and then this acts as a point mass.

So, this is the inertia of the point mass. Inertia moment of inertia of the mass of the wheel concentrated at point P . The inertia of the mass of the wheel concentrated point P about o . So, this three together means you have say some body is there and the in the parallel axis theorem what we do? We have one body here and inside this there is another body. So, what we do that, first we find out the mass of the moment of inertia of the main body about this; say line then we find out the moment of inertia about this small

body about this line and then we consider this to be the point mass and then calculate the moment of inertia about this by writing $m \rho^2$.

And at the further of this is the main body b . So, we write as J_B and plus here it is a inertia about this line is suppose J_w double bar. So, we are adding this J_w double bar and then to this $m \rho^2$ you are adding. So, this part we have to put it in a vector format because it is in 3 dimension. So, this is in 3 dimension, 3 D. While here it is written only for 1 dimension ok. So, this is a simplified model. So, this is not applicable here, but this is the way so, this is something acting like a parallax theorem. So therefore, this part the quantity in the bracket is the moment of inertia of the whole body about the point o . So, this gets reduced to this implies h can be written as $C \times V_0$ plus J double bar dot ω plus. So, this is the moment of inertia of the whole system. Moment of inertia of the whole system about point o whole system means it is a whole system implies, this implies main body plus wheel. So, we will stop here and then we will continue in the next lecture.

Thank you very much.